

UNIVERSITY OF CALIFORNIA AT BERKELEY  
 Department of Mechanical Engineering  
 ME233 Advanced Control Systems II  
 Spring 2012

## Homework #7

Assigned: Apr. 3 (Tu)  
 Due: Apr. 10 (Tu)

The second ME233 midterm will be held on Thursday, April 12th. The exam will be closed book and notes, but you are allowed to bring 4 double-sided sheets (i.e. 8 pages) of hand-written notes on 8.5"  $\times$  11" paper and a calculator. The midterm will focus on the material covered in Lectures 9–16, which includes the material in this assignment.

- Figure 1 shows the feedback interconnection for a system with a disturbance observer. When implementing the disturbance observer, we only implement the portion that

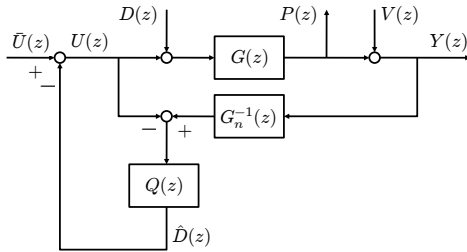


Figure 1: Disturbance Observer Structure

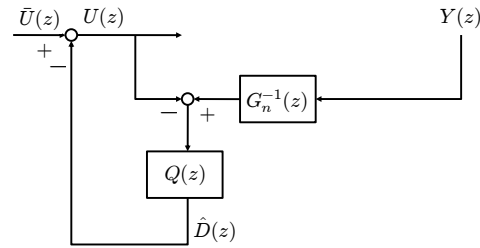


Figure 2: Disturbance Observer—Controller Only

generates  $U(z)$  from  $\bar{U}(z)$  and  $Y(z)$ , as shown in Fig. 2.

- Find the transfer function from  $Y(z)$  and  $\bar{U}(z)$  to  $U(z)$  in Fig. 2.
- Suppose that  $G_n^{-1}$  is proper. In this case, it is valid to choose  $Q(z) = \alpha \in \mathcal{R}$ . Based on your answer from the previous part, note that the block diagram in Fig. 2 is not well-posed when  $\alpha = 1$ . Does there exist  $\alpha \in \mathcal{R}$  such that the closed-loop transfer function from  $D(z)$  to  $P(z)$  is zero? If not, is there a limit to how small we can make the transfer function from  $D(z)$  to  $P(z)$ ?

2. Consider the feedback system in Fig. 3 where  $u(k)$  and  $d(k)$  are respectively the

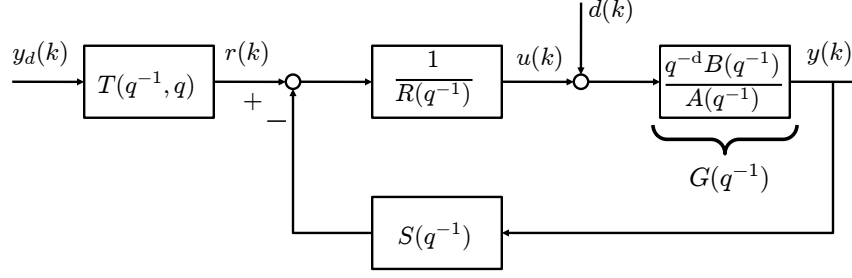


Figure 3: Feedback System

control and disturbance plant inputs,  $y_d(k)$  is the reference model's output, and  $r(k)$  is the reference input to the feedback block.

The control objective is to reject the persistent deterministic disturbance  $d(k)$ , place the feedback closed-loop poles, and track the desired output  $y_d(k)$ .

In order to help you verify your solutions of the Diophantine equation (also known as the Bezout equation), I have uploaded the MATLAB file `bezout.m`, which solves this equation. However, I advise you to solve the Diophantine equations in this problem by hand, so that you gain an understanding of what is involved in the solution of this type of equation.

- (a) The plant transfer function  $G(z)$  is derived from a continuous time transfer function  $G(s)$  that is preceded by a zero-order hold and followed by a sampler, and is given by

$$G(z) = \frac{\bar{B}(z)}{\bar{A}(z)} = (1 - z^{-1}) \mathcal{Z} \left\{ \mathcal{L}^{-1} \left( \frac{G(s)}{s} \right) \right\},$$

where

$$G(s) = \frac{1}{s(s+1)}$$

and the sampling time is  $T = 0.5$  seconds.

Calculate the plant polynomials  $A(q^{-1}) = 1 + a_1q^{-1} + a_2q^{-2}$ ,  $B(q^{-1}) = b_o + b_1q^{-1}$  and pure delay time  $d$ . You can use the MATLAB function `c2d` for this purpose.

- (b) The tracking control objective is to follow the reference signal  $y_d(k)$ , which is the output of the reference model

$$A_m(q^{-1})y_d(k) = q^{-d}B_m(q^{-1})u_d(k). \quad (1)$$

Select the coefficients of the second order polynomial  $A_m(q^{-1}) = 1 + a_{m1}q^{-1} + a_{m2}q^{-2}$ , so that the reference model has a natural frequency of 1 rad/sec and a damping ratio of 0.707.

**Hint:** Remember that, since  $z = e^{sT}$ , we can calculate the discrete time poles by  $p_d = e^{p_c T}$ , where  $p_d$  is the discrete time pole,  $p_c$  is the continuous time pole and  $T$  is the sampling time.

- (c) Letting  $B_m(q^{-1}) = b_{mo}$ , select  $b_{mo}$  so that the reference model has unity static gain<sup>1</sup>.
- (d) Choose the coefficients of the closed-loop system characteristic polynomial (after pole-zero cancelation)

$$A'_c(q^{-1}) = 1 + a'_{c1} q^{-1} + a'_{c2} q^{-2}$$

so that the closed-loop feedback dynamics from  $r(k)$  to  $y(k)$  behaves as a second-order system with a natural frequency of 2 rad/sec and a damping ratio of 0.5.

- (e) Design the control system under the following specifications and assumptions:
  - i. The closed loop system characteristic polynomial (before pole-zero cancelation) is given by

$$A_c(q^{-1}) = A'_c(q^{-1})B^s(q^{-1})$$

where

$$B^s(q^{-1}) = \frac{1}{b_o} B(q^{-1}), \quad B^u(q^{-1}) = b_o,$$

and  $b_o$  is the leading coefficient of  $B(q^{-1})$ . This means that none of the plant zeros will be canceled by the feedback system.

- ii. Assume that  $d(k) = 0$ . This means that the disturbance annihilating polynomial is selected to be  $A_d(q^{-1}) = 1$ .
- iii. The feedforward compensator  $T(q^{-1}, q)$  must be selected so that perfect tracking is achieved under a zero initial state for both the plant and the reference model.
- (f) Do a computer simulation of the control system designed in problem 2e when  $y_d(-1) = y_d(0) = y(-1) = y(0) = 0$  and

$$u_d(k) = [u_s(k) - 2u_s(k - 25)] + [2u_s(k - 50) - 2u_s(k - 75)] \quad (2)$$

$$d(k) = 0.5u_s(k - 40) \quad (3)$$

where  $u_s(j)$  is the unit step function, i.e.

$$u_s(j) = \begin{cases} 0 & j < 0 \\ 1 & j \geq 0 \end{cases}$$

Plot  $u_d(k)$ ,  $y_d(k)$ ,  $y(k)$  and  $u(k)$ .

- (g) Design the control system under the same specifications in problem 2e, except that assume now that  $d(k) = d(k - 1)$ .

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<sup>1</sup>i.e. if  $\lim_{k \rightarrow \infty} u_d(k) = u_{ss}$  then  $\lim_{k \rightarrow \infty} y_d(k) = u_{ss}$ .

- (h) Do a computer simulation of the control system designed in problem 2g under the conditions described in problem 2f. Plot  $u_d(k)$ ,  $y_d(k)$ ,  $y(k)$  and  $u(k)$ .
- (i) Design the control system under the following specifications and assumptions:
- i. The closed loop system characteristic polynomial satisfies

$$A_c(q^{-1}) = A'_c(q^{-1})B^s(q^{-1})$$

where

$$B^s(q^{-1}) = 1, \quad B^u(q^{-1}) = B(q^{-1}),$$

This means that all plant zeros will not be canceled by the feedback system.

- ii. Assume that  $d(k) = d(k - 1)$ .
  - iii. The feedforward compensator  $T(q^{-1}, q)$  is designed using the zero-phase error tracking control approach.
- (j) Do a computer simulation of the control system designed in problem 2i under the conditions described in problem 2f. Plot  $u_d(k)$ ,  $y_d(k)$ ,  $y(k)$  and  $u(k)$ .
- (k) Discuss the outcome of the simulation results. In particular
- Comment on the effectiveness of the zero-phase feedforward control technique.
  - Compare the control effort  $u(k)$  when the zeros are canceled vs when the zeros are not canceled.