PHYSICAL REVIEW A

GENERAL PHYSICS

THIRD SERIES, VOL. 5, No. 2

FEBRUARY 1972

PHYSICAL REVIEW A

VOLUME 5, NUMBER 2

FEBRUARY 1972

Fresnel Drag in a Ring Laser: Measurement of the Dispersive Term

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(Received 23 February 1971)

The Fresnel drag has been measured in a triangular ring laser at a wavelength of 0.6328 μ . The drag coefficient in fused silica is $\alpha = 0.541 \pm 0.003$, while the theory, including the dispersion term, gives $\alpha_{\text{theor}} = 1 - (1/n^2) - \beta(\lambda/n) \, dn/d\lambda = 0.5423$ with $\beta = 1$. The coefficient 3 is thus determined as $\beta = 0.87 \pm 0.22$, which includes the classical value $\beta = 1$. Thus the magnitude of the drag in a ring laser is within the errors equal to that given in an inertial frame of reference (linear drag).

I. INTRODUCTION

The ring laser is an extremely sensitive instrument for measuring nonreciprocal phenomena in light propagation. As an example, a rotation of the ring produces a beat frequency which is large enough that the rotation of the earth can be detected in simple Sagnac-type arrangements.² The drag phenomenon of light in moving matter also introduces a nonreciprocity into the ring laser, and therefore produces a frequency difference between the two contracirculating beams. (Born noted^{2a} that "It is very difficult to test Fresnel's formula by means of experiments on the earth, because it requires that transparent substances be moved with extreme rapidity.") This has been demonstrated by Macek et al. with moving air, carbon tetrachloride, and fused silica, and recently by Moss et al. in a moving electron gas in InAs.

In this paper a precision measurement of the effect of drag in a ring laser has been attempted. The motivation for this experiment arises from the following facts: (i) The ring laser is a promising instrument for far more accurate drag measurements than could previously be achieved; (ii) if the linear drag is well understood, the transversal Fresnel drag can be attacked; (iii) the drag in a rotational arrangement need not be identical to that in a linear configuration as in the classic interferometer arrangements⁵; (iv) drag may provide a precision bias source in rotation-rate sensors; (v) kinematic effects in the field of general relativity may be investigated.⁶

II. THEORY OF DRAG IN INERTIAL FRAME

A medium with an index of refraction $n(\lambda)$ is moved with a velocity \vec{v} in an inertial frame of reference. A light beam passing through this medium with a velocity \vec{u} (with a magnitude c/n relative to the medium) possesses then a velocity \vec{u}_1 in the inertial frame, where \vec{u}_1 is given by \vec{v}_2

$$\vec{u}_1 = \frac{\vec{u} + \gamma \vec{v} + (\gamma - 1)(\vec{u} \cdot \vec{v} / \vec{v} \cdot \vec{v})\vec{v}}{\gamma[1 + (\vec{u} \cdot \vec{v} / c^2)]},$$
(1)

with

$$\gamma = (1 - \vec{\mathbf{v}} \cdot \vec{\mathbf{v}}/c^2)^{-1/2} \simeq 1 \qquad \text{if } |\vec{\mathbf{v}}| \ll c \ .$$

If the medium has a velocity component $\pm v_m$ parallel to the beam, then the magnitude v_1 of the beam velocity in its original direction \tilde{u} becomes in a first-order approximation (Fig. 1)

$$v_1 = |\vec{u}_1| = [c/n(\lambda')] \pm v_m[1 - 1/n^2(\lambda')] + O(v^2/c)$$
.

(1'

If there is a transversal component v_i of the medium, it will have an effect on v_i in the second-order term v^2/c^2 .

The wavelength λ' will be different from λ (medium at rest), because of the Doppler effect. If the index of refraction shows dispersion, then $n(\lambda')$

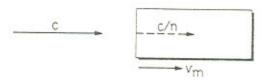


FIG. 1. Drag experiment in a medium with an index of refraction n and parallel velocities of light and medium.

 $\neq n(\lambda)$. In a first-order approximation, the Doppler effect is given by

$$\Delta \lambda / \lambda \simeq v_m / (c/n)$$
, (2)

and we may expand $n(\lambda')$,

$$n(\lambda') \approx n(\lambda) + \frac{dn}{d\lambda} \Delta \lambda \approx n + \frac{dn}{d\lambda} \frac{\lambda v_m n}{c}$$

= $n\left(1 + \frac{\lambda}{c} \frac{dn}{d\lambda} v_m\right)$. (3)

Applying Eq. (3) to Eq. (1'), we have as a firstorder approximation,

$$v_1 = \frac{c}{n} \pm \left(1 - \frac{1}{n^2} - \frac{\lambda}{n} \frac{dn}{d\lambda}\right) v_m = \frac{c}{n} \pm \alpha v_m, \quad (4)$$

where α is the drag coefficient.

For a vacuum, n=1, $dn/d\lambda=0$, there is no drag. For $n\gg 1$, $v_1=(c/n)\pm v_m$; i.e., superposition of the velocities holds, because in this case both velocities c/n and v_m are nonrelativistic.

The problem is thus to measure a small additional velocity $\pm \alpha v_m$, usually of the order of meters per second, in the presence of a very large velocity c/n, of the order of 10^8 m/sec. The ring laser presents itself as a proper instrument for this problem, since by virture of its operation it acts as a differential sensor [Eq. (11)], where the effect of c/n is exactly canceled by the two contracirculating beams, and the effect of αv_m is doubled, since the drag is a nonreciprocal phenomenon, and the resulting beat frequency between the two beams due to the drag is therefore doubled.

Recently, Post⁸ pointed out that in a rotational arrangement as given by a ring laser, the drag need not necessarily be identical with the one observed in an "open-end" experiment.⁵ In addition to that, the situation in a ring laser is complicated by the presence of the earth's rotation, which introduces an additional frequency difference through the Sagnac effect.

Table I gives a summary of the history of the drag coefficient. The experiments in Table I were all done with interferometric techniques. They are very delicate, since, for example, in water flowing at a velocity of 10 m/sec, the relative change of the velocity of light is $\alpha v_m/(c/n)$ $\simeq 2\times 10^{-8}$. This problem can be circumvented by the application of the ring laser, as the Sec. III will show.

III. RING LASER AS A SENSOR FOR NONRECIPROCAL PHENOMENA

A. Ring-Laser Operation

At this time, many of the aspects of ring lasers are well understood. Thus, only a very brief account of the basic theory of operation is given. For more extensive coverage of the theory we refer to papers such as Ref. 8.

To gain a basic understanding of the ring laser, it is sufficient to use two equations:

$$N\lambda = L$$
 (5)

(oscillation condition: N = integral number, $L = \text{optical path length for a round trip} = \int n \, ds$) and

$$\lambda f = c$$
 . (6)

If we keep the laser oscillating in a given axial mode (N = const), but vary L, we get through differentiation of Eqs. (5) and (6)

$$(\Delta \lambda/\lambda) = \Delta L/L = -\Delta f/f$$
, $N = \text{const}$. (7)

If we keep L = const, but switch to a neighboring axial mode, $\Delta N = 1$, we have

$$(\Delta \lambda/\lambda) = -\Delta N/N = -\lambda/L = -\Delta f_L/f$$
 or $\Delta f_L = c/L$

for L = const. (8)

Equation (8) has been used to determine the effective length L (see Sec. IV on experimental setup). Equation (7) is the basis for calculating the drag.

B. Drag in a Ring Laser

First, we determine the change of the effective length of the ring for one of the two contracirculating beams due to drag. The effective optical path length in the silica disk is ln (Fig. 3). Using Eq. (4), the change in the effective index of refraction can be expressed as a change in the velocity or

$$\Delta(ln)/ln = [v_1(v_m \neq 0) - v_1(v_m = 0)][v_1(v_m = 0)]^{-1}$$

= $[(c/n) \pm \alpha v_m - (c/n)](c/n)^{-1} = \pm n\alpha v_m/c$,

and the relative change of the total optical path is then

$$\frac{\Delta L}{L} = \frac{1}{L} \frac{\Delta (ln)}{ln} ln = \frac{\pm n^2 \alpha l v_m}{Lc} = -\frac{\Delta f}{f} . \qquad (10)$$

Equation (10) is derived under the assumption that the only change of effective length is due to the drag in l, and that Eq. (7) holds.

Finally, the beat frequency between the two contracirculating beams, Δf_D , is twice the frequency difference given by Eq. (10), and we have therefore

$$\Delta f_D = 2(\Delta f/f)f = 2n^2 \alpha lf v_m/Lc = 2n^2 \alpha lv_m/\lambda L. \quad (11)$$

Equation (11) was first given by Macek et al.³ It is a special case of the result for a general geometry of ring laser and dragging medium as derived by Post⁸

$$\Delta f_D = (2f/c) (\oint n^2 \alpha \vec{v} \cdot d\vec{r}) (\oint n dr)^{-1},$$
 (11')

where dr is a line element along the beam path.

Typical values in our experiments are n=1.46 (fused silica), $\alpha=0.54$, l=2 cm, $v_m=1$ m/sec, L=3 m, and $\lambda=0.63$ μ , so we expect a beat frequency of $\Delta f_B \simeq 24$ kHz, which is well above the lock-in frequency.

In the actual experiments, the silica disk is tilted at the Brewster angle with respect to the beam to avoid reflection losses. In this case we have (Appendix A)

$$v_m l = \vec{v}_t \cdot \vec{1} = \omega dx_0/n$$
, (12)

where $\omega = 2\pi f_m$ is the angular velocity of the silica disk, \vec{v}_t is the tangential velocity at the beam location, \vec{l} is the beam vector, \vec{d} is the thickness of the disk, x_0 is the horizontal displacment of the laser beam from the axis of rotation.

Under these conditions, the beat frequency due to drag in a rotating disk is

TABLE I. History of the drag coefficient.

Year	Investigator	Investigator Contribution		Comment	
1818	Fresnel	$\alpha_F = 1 - (1/n^2)$	Theor.	Correct result (without dispersion term), but wrong theory	
1851	Fizeau	$\alpha/\alpha_F = 1.14 \pm ?$	Expt.	In water; error probably larger than ±0.14	
1886	Michelson Morley	$\alpha/\alpha_F = 0.993 \pm 0.05$ $(\alpha/\alpha_L = 0.964 \pm 0.05)$	Expt.	In water	
1893	Lorentz	$\alpha_L = 1 - (1/n^2) - (\lambda/n) (dn/d\lambda)$	Theor.	Inclusion of dispersion term	
1907	Laue	Derivation of α_F by relativistic addition theorem	Theor.		
1914	Einstein	Explanation of dispersion term by Doppler effect	Theor.		
1914-1925	Zeeman	$\alpha/\alpha_L = 0.998 \pm 0.006$	Expt.	In water and fused silica at different wavelengths; con- firms dispersion term	

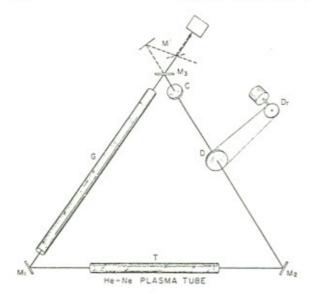


FIG. 2. Schematic of triangular ring laser with He-Ne plasma tube T (bottom arm), dragdisk D, compensator flat C (right arm), and an open glass tube G in the left arm to reduce unwanted Fresnel drag by moving air. The beam mixer M is behind the mirror M₃. The drive Dr for the drag disk is mounted off the granite support and coupled to the disk by a cotton thread "belt."

$$\Delta f_D = 2n\alpha \omega dx_0 / L\lambda . \qquad (13)$$

C. Rotation of Ring

Since the ring laser is fixed with the rotating earth, we have to account for the resulting Sagnactype effect. This is given by Post⁸ as

$$\Delta f_R = \frac{4\vec{\Omega} \cdot \vec{A}}{\lambda L} \left[1 - \left(\frac{\Omega r}{c} \right)^2 \right]^{-1} \simeq \frac{\Delta L}{\lambda} \frac{c}{L} \quad \text{if } \frac{\Omega r}{c} \ll 1,$$

where Ω_r is the tangential velocity of ring perimeter, $|\vec{\Omega}| = \Omega_0 \sin\theta$, $\Omega_0 = 2\pi/\text{day}$, θ is the degree of latitude, $|\vec{A}|$ is the area enclosed by the beam path.

Note that $\Delta L/\lambda$ represents the Sagnac effect as measured by the fringe shift. If multipled by the frequency $\Delta f_L = c/L$ [Eq. (8)], we obtain Δf_R . The frequency difference between neighboring modes, c/L, is in our experiments about 10^6 Hz. This accounts for the easy detectability of the Sagnac effect in a ring laser. The expected beat frequency due to the earth's rotation in the system present is

$$\Delta f_R = 43 \text{ Hz}$$
.

Thus, Δf_D is generally large compared to Δf_R . As a first approximation, we expect a total beat frequency Δf_B ,

$$\Delta f_B = \Delta f_D(\omega) + \Delta f_R(\Omega), \qquad (15)$$

assuming that no cross-coupling effects occur. In a plot of Δf_B vs ω , Δf_R will then produce a constant offset. The magnitude of this offset is used as a convenient means to check our equipment. The least-squares fits provide a weighted average of $\Delta f_{R,expt} = 33 \pm 11$ Hz, which proves that the equipment is accurate enough for drag experiments.

D. Evaluation of the Drag Coefficient

If the ring laser and rotating silica disk are properly adjusted, Eqs. (13) and (15) suggest a fit of

$$\Delta f_B = mf_m + b \tag{16}$$

to the data, where m and b are the parameters to be evaluated. α and Δf_R are then related to m and b by

$$\alpha = \lambda L (4\pi n dx_0)^{-1} m , \quad \Delta f_p = b . \qquad (17)$$

The fit also provides the internal rms errors Δm and Δb from the statistics of the measurements $f_{\rm m}$ and $\Delta f_{\rm B}$. Δm and Δb determine the quality of a given set of data and they enter into the final error analysis.

IV. EXPERIMENTAL SETUP

A short description of the arrangement is given. For more detailed information consult Ref. 10.

A. Ring Laser

The ring laser is built as a nearly equilateral triangle with sides of about 1 m. (see Fig. 2). We used two different sets of three dielectric coated concave mirrors in the experiments to realize the three requirements of ease of adjustment, large mode volume in the plasma tube, and small beam waist at the drag site. 11

The focal lengths were 3, 6, 6 m and 1, 1, 2 m with a specified peak reflectance of 99.9°% for normal incidence at 0.63 µ. Calculations of the reflectance at an incident angle of 30° show a small increase of 0.03% for the TE wave used. 12 The 3-6-6 m set has a larger mode volume in the plasma tube, but the 1-1-2 m set has a smaller waist at the drag site. Both sets are aligned with the same effort and resulting power. A rf-excited He-Ne plasma tube of 1-m length and 4-mm bore served to excite a laser beam with the polarization normal to the plane of the ring. An iris is used in the side containing the plasma tube to eliminate nonaxial modes. Up to five axial modes were observed. By suitable reduction of plasmatube gain and iris diameter, single-axial-mode operation is achieved, which is employed in the drag experiments. The two output beams at one corner mirror (the 3-m mirror or the 1-m mirror, respectively) are mixed in a simple beam mixer and detected in a photomultiplier. The block dia-

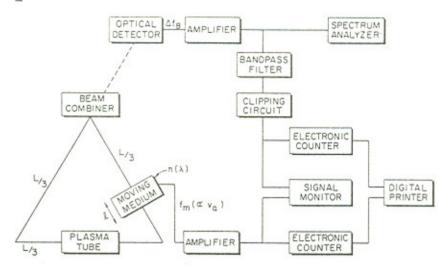


FIG. 3. Block diagram of ring laser and associated electronics.

gram, Fig. 3, shows the monitoring and counting of the beat frequency.

The ring laser and beam mixer is mounted on a 550-lb granite bloc which is vibrationally isolated by adjustable shock absorbers. The detecting, monitoring, and recording equipment was located on a different bench.

B. Optical Length of Ring

The length L was determined by observing the beat frequencies between four axial modes, while the ring was loaded with the drag disk and a compensator flat. The frequency separation between neighboring axial modes is determined as $\Delta f_{L1} = 92.0209 \pm 0.0006$ MHz. We obtained also $\frac{1}{2}\Delta f_{L2} = 92.0192 \pm 0.0007$ MHz and $\frac{1}{3}\Delta f_{L3} = 92.0199 \pm 0.0009$ MHz out of altogether 140 measurements. Mode pulling appeared to be sufficiently small. Thus the three data were averaged to give an optical length of the loaded ring laser [Eq. (8) with $c = 2.9979 \times 10^8$ m/sec] of L = 3.2589 m with a relative error smaller than 0.01%.

C. Drag in Gas

For initial adjustment of the system, a 90-cm flow tube with 5-mm bore and Brewster end windows is used. Some semiquantitative measurements with moderate flows of nitrogen gas were made, whereby beat frequencies above 10 kHz could easily be reached, but the unknown flow profiles and difficulties to measure total gas flows to better than 1%, prevented us from attaining the required accuracies. 13

D. Drag in Fused Silica

Setup. For precise measurements an arrangement used by Macek et al. is applied. A rotating disc is traversed by the beam such that beam and axis of rotation are not coplanar (Appendix A). The axis of rotation is normal to the disc surface. The beam enters the disc under the Brewster angle (Fig. 2) at a distance x_0 from the rotation center. The rotation center is in the plane of the ring. Under these conditions, Eq. (12) holds, and a knowledge of the quantities in Eq. (17) allows one to calculate α .

Rotating disk. We used a Homosil $\lambda/20$ disk of 2-in. diam, a specified index of refraction $n=1.4571\pm0.0001$, and a thickness $d=1.2772\pm0.0001$ cm. The normal or rotation axis was set to minimize reflection. The adjustment error of the two angles of the beam, namely the Brewster angle θ_B with respect to the normal of the disk and the angle θ_B with respect to a plane normal to the ring plane, is $\pm 0.1^\circ$. The disk is mounted in a bearing-mounted holder which can be displaced on a platform parallel to itself by an x-y vernier with an accuracy of 0.01 cm. The platform is adjustable to be able to set θ_B and θ_B .

The disk is rotated through a cotton thread belt from a 1.4-oz. in. speed-regulated motor at rates of 100 to 2500 rpm. The rotation angle is measured by counting the pulses in an optical detector which resulted from the chopping of light through 60 holes along the rim of the disk mount. The error of the rotation angle is thus less than 0.01% after 200 full revolutions. Note that a nonuniform rotation rate does not introduce errors as long as

TABLE II. Data of the drag disk.

Quantity	Symbol		Unit	absolute error
Thickness	d	1.2772	cm	1 ×10-1
Index of refraction	77	1.4571		1×10 ⁻⁴
Wavelength	λ	0.632815	TI.	<1×10 ⁻⁴
Brewster angle	0 8	Adjusted	deg	0.1
Normal angle	0,	Adjusted	deg	0.1
Rotation rate	100	100-2500	rpm	0.1
Beat frequency range	ΔV_{B}	1-50	kHz	0.01
Displacement	30	0.5-2.0	cm.	0.01

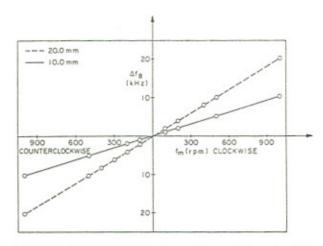


FIG. 4. Plot of beat frequency Δf vs rotation rate f_m for two displacements $x_0=1.00$ and 2.00 cm.

the gating period for the counting of the rotation angle and the counting of the beat frequency is synchronized, since the beat frequency is proportional to the frequency of rotation [Eq. (13)].

Table II gives a summary of the parameters and typical errors.

V. MEASUREMENTS

A. Statistics of a Run

In a given run with a nominal frequency of rotation and a displacement x_0 on the disk, the two frequencies were counted for a 10-sec period. Typically, 10 to 50 of such runs with identical parameters averaged to produce one data point $(f_m, \Delta f_B)$.

TABLE III. Beat frequency vs rotation rate with displacements $x_0 = 1.00$ and 2.00 cm.

	f_m (rpm)	Δf_B (Hz)	$\Delta f_B = \Delta f_B$, fitted (Hz)
$x_0 = 1.00 \text{ cm}$	- 998	-10176	22
	- 500	-5062	37
	-198	-1972	34
	-101	-1217	- 204
	+102	+1149	83
	+ 200	+2079	10
	+ 499	+ 5136	5
	- 998	+10254	13
$x_0 = 2.00 \text{ cm}$	-1002	-20 286	1
	- 499	-10107	- 27
	-399	-8063	- 12
	-300	-6031	11
	- 201	-4029	4
	-100	-1941	43
	+ 100	+ 2045	- 30
	+ 200	- 4113	9
	+ 400	-8166	3
	- 499	+10187	15
	- 1000	- 20 321	-17

As an example, 27 runs with the parameters x_0 = 2.00 cm and $f_m \simeq 1000$ rpm result in $\langle f_m \rangle_{av}$ = $(1.00233 \pm 0.00002) \times 10^3$ rpm, and $\langle \Delta f_B \rangle_{av}$ = $(2.02857 \pm 0.00013) \times 10^4$ Hz, with an rms deviation of $\delta f_m = \pm 0.12$ rpm and $\delta \Delta f_B = \pm 6.8$ Hz for one 10-sec run. The data were Gaussian distributed and showed no drift.

The data are arranged in two groups: (i) beat frequency Δf_B vs rotation rate f_m for a constant displacement x_0 , (ii) beat frequency Δf_B vs displacement x_0 for a constant rotation rate f_m .

B. Beat Frequency vs Rotation Rate

Figure 4 and Table III present data for $f_{\rm m}$ between 100 and 1000 rpm with x_0 = 1.00 and 2.00 cm. Negative values of $f_{\rm m}$ correspond to an arbitrarily defined counterclockwise rotation. A least-squares fit of a straight line to the data ¹⁴ gives Eq. (16)

at
$$x_0 = 1.00$$
 cm: $m = 613.4 \pm 1.2$, $b = 20 \pm 26$ Hz;
at $x_0 = 2.00$ cm: $m = 1216.0 \pm 2.8$, $b = 50 \pm 49$ Hz.

The third column in Table III lists the deviations of the measured points from the straight lines. They are more or less randomly distributed, indicating that there are no systematic trends within a set of runs, as would be the case if mode pulling were apparent. Also note that the results of m do not overlap (divide the second result by 2), which indicates that the statistical errors of a given fit are somewhat too small to account for fluctuations from one set of data to another.

C. Beat Frequency vs Displacement

Figure 5 presents beat frequency vs displacement for five different values of f_m . This plot is used

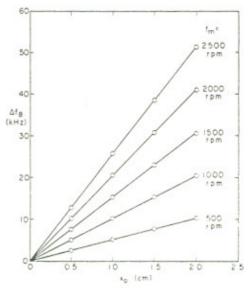


FIG. 5. Plot of beat frequency Δf vs displacement x_0 for five different rotation rates between 500 and 2500 rpm.

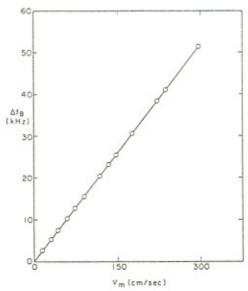


FIG. 6. Summarizing plot of beat frequency Δf vs drag velocity $|v_m|$. All data are plotted into the first quadrant.

to check whether there are any systematic deviations from the fit, which would disclose errors in determining the rotation center or errors in the vernier. No such deviations are observed.

D. Evaluation of Results and Discussion of Errors

Figure 6 contains all significant measurements during the investigation. The beat frequency is plotted against the drag velocity

$$v_m = 2\pi f_m x_0 \cos \theta_B , \qquad (18)$$

where $\cos\theta_B = (n^2 + 1)^{-1/2} = 0.5659$.

Table IV summarizes the significant measurements Δf_B vs f_m . The experimental value $\overline{\alpha}_{\rm expt}$ is

TABLE IV. Results of the least-squares fits of runs. $\Delta f_B \text{ vs } f_m \text{ with } x_0 = \text{parameter.}$

(cm)	m	Δm	b (Hz)	Δb (Hz)	α_i	$(\alpha_i - \alpha)/\alpha$ (%)
1.00	614.4	1.2	20	26	0.5416	-0.13
2.00	1216.0	2.8	50	49	0.5359	-1.18
1.81	1099.8	0.9	54	22	0.5356	-1.24
0.50	307.7	1.4	28	41	0.5428	+0.09
1.00	615.1	0.7	33	17	0.5425	+0.04
1.50	922.6	2.2	-14	62	0.5422	-0.02
2.00	1233.0	1.8	-13	51	0.5435	+0.02
All da	ita				0.5406	-0.31

calculated from Eq. (17) with the parameters from Table II. The comparison of $\overline{\alpha}_{\rm expt}$ with α from Eq. (4) gives

$$(\overline{\alpha}_{expt} - \alpha)/\alpha = -0.003 \pm 0.005$$
.

The error includes consistency errors as well as estimates of residual systematic errors.

Thus, there is no significant deviation of the drag coefficient from the theoretical coefficient for linear drag including the dispersion term.

As is apparent in Table II, the largest error is in x_0 . This error can be reduced by a considerable amount, if the beam waist is kept small enough, which can be accomplished by a judicious selection of mirror radii, mirror separations, and drag site. An xy-vernier with 1- μ accuracy would then measure exact differences in beam positions on the disc, even if the beam waist is much larger than 1 μ . The results may then be extracted with the help of Eq. (A5) (Appendix A).

A maladjustment of θ_B introduces also a first-order error (see Appendix B). Thus it is advis-

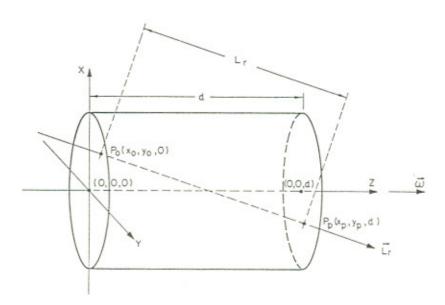


FIG. 7. Linear drag in a rotating right cylinder. The beam vector \vec{L}_r and the $\vec{\omega}$ axis are not coplanar. The xy plane of the coordinate system is in the front plane. L_r is the length of the beam path in the cylinder.

able to measure the angle θ_B , rather than to set it to achieve minimum reflection. The measurement of θ_B can be done with a considerably better accuracy than $\pm\,0.1^\circ$.

An over-all improvement by an order of magnitude in α appears feasible with this technique.

VI. CONCLUSION

The drag coefficient in silica measured in a ring laser appears to be equal to the drag coefficient as given by Lorentz [Eq. (4)] for a linear system. The experimental error is slightly smaller than that obtained in Ref. 5, but the limit of accuracy attainable with the ring laser is by far not yet reached. Expressing the present result in relation to the dispersive term only, we have

$$(\overline{\alpha}_{expt} - \alpha) [\alpha - 1 + (1/n^2)]^{-1} = 0.13 \pm 0.22$$
,

which means that the excess of the experimental drag coefficient over the amount $1 - (1/n^2)$ is within one-half of the rms error equal to the term $(\lambda/n) \times (dn/d\lambda)$.

ACKNOWLEDGMENTS

The interest and support of W. L. Hughes of the Electrical Engineering Department is gratefully acknowledged. We also thank H. Hall of the Physics machine shop and the staff of the Continental Oil Company, Ponca City, Okla. for technical assistance in setting up the ring laser and in computer problems.

APPENDIX A: DRAG IN A ROTATING CYLINDER

Figure 7 shows a right circular cylinder. A Cartesian coordinate system is located with the origin on the front surface at the intersection of the rotation axis. The rotation axis is identical with the z axis and normal to the surface. The beam shall enter at $F_0(x_0,\ y_0,\ 0)$ with directional angles ϕ , ξ , and ψ relative to the x, y, and z axes. Such a beam is described by a vector

$$\vec{L}_r = (x_0 + l \cos \phi) \vec{i} + (y_0 + l \cos \xi) \vec{j} + (l \cos \psi) \vec{k}$$
; (A1)

 \overline{i} , \overline{j} , and \overline{k} are unit vectors in the x, y, z direction and l is the length of vector from P_0 .

The velocity field \vec{v}_t at any point along \vec{L}_t is given by the vector product of $\vec{\omega}(0, 0, \omega)$ and $\vec{r}(x, y, 0)$:

$$\vec{v}_t = \vec{\omega} \times \vec{r} = -(\vec{y} \cdot \vec{i} - \vec{x} \cdot \vec{j})$$
. (A2)

Now we calculate the scalar product $v_m l$ throughout the cylinder:

$$v_m l = \int_{|\vec{L}_r|=0}^{|\vec{L}_r|=L_r} \vec{v}_t \cdot d\vec{L}_r = \omega L_r(x_0 \cos \xi - y_0 \cos \phi). \tag{A3}$$

Since

$$d = L_r \cos \psi$$
 (A4)

and

$$\cos^2\phi + \cos^2\xi + \cos^2\psi = 1 ,$$

we have

$$v_m l = \omega d(x_0 \cos \xi - y_0 \cos \phi)(1 - \cos^2 \phi - \cos^2 \xi)^{-1/2}$$
 (A5)

Thus, the drag is solely determined by the entrance point $P_0(x_0, y_0, 0)$ and the directional angles ξ and ϕ at P_0 .

Case 1. For the intended operation of the disk in this paper, we have $\xi = \theta_B$ with $\tan \theta_B = n$ ($\theta_B = B$ rewster angle) and $\phi = 90^\circ$ ($\theta_n = 0$). Equation (A5) gives in this case $v_m l = \omega dx_0/n$ [Eq. (12)]. Any displacement $y_0 \neq 0$ does not affect the drag, as long as the condition $\phi = 90^\circ$ prevails.

Case 2. In this case (coplanar beam and rotation axis) the drag is zero, as can be seen in the following way: The drag at any point is given through Eq. (A2) and Eq. (A3) as

$$v_{-}l = (\vec{\omega} \times \vec{r}) \cdot d\vec{L}_{r}$$
.

Since \vec{r} is in the plane set up by $\vec{\omega}$ and \vec{L}_{τ} , the velocity field $\vec{v}_t = \vec{\omega} \times \vec{r}$ is perpendicular to that plane at any point, and therefore $\vec{v}_t \cdot d\vec{L}_{\tau} = 0$.

APPENDIX B: ADJUSTMENT ERRORS IN THE ROTATING CYLINDER

We expand Eq. (A5) to first order in x, y, ϕ , ξ around the "ideal" conditions $x = x_0$, y = 0, $\phi = 90^\circ$, $\xi = \theta_B$, and get

$$\frac{\Delta(v_m l)}{v_m l} = \frac{\Delta x_0}{x_0} - \frac{n^2 + 1}{n} \Delta \xi ; \qquad (B1)$$

i.e., the only first-order errors are in the displacement x_0 and the Brewster angle θ_B . For example, an error of 1% in x_0 enters as 1% into the final result, and an error of $\Delta \xi = 0.1$ deg enters as 0.4%. The most careful attention has therefore to be given to the measurement of x_0 and θ_B .

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is, of course, not an inertial system either. Taking the next step by inclusion of the orbiting earth, we expect a diurnal peak-to-peak modulation of Δf_R of 0.13 Hz in the setup presented.

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CCl₄, respectively.

¹³In the fitting process, equal weight is given to all data points, and the f_m data are treated as exact, since the errors in Δf_B are usually large compared to δf_m .