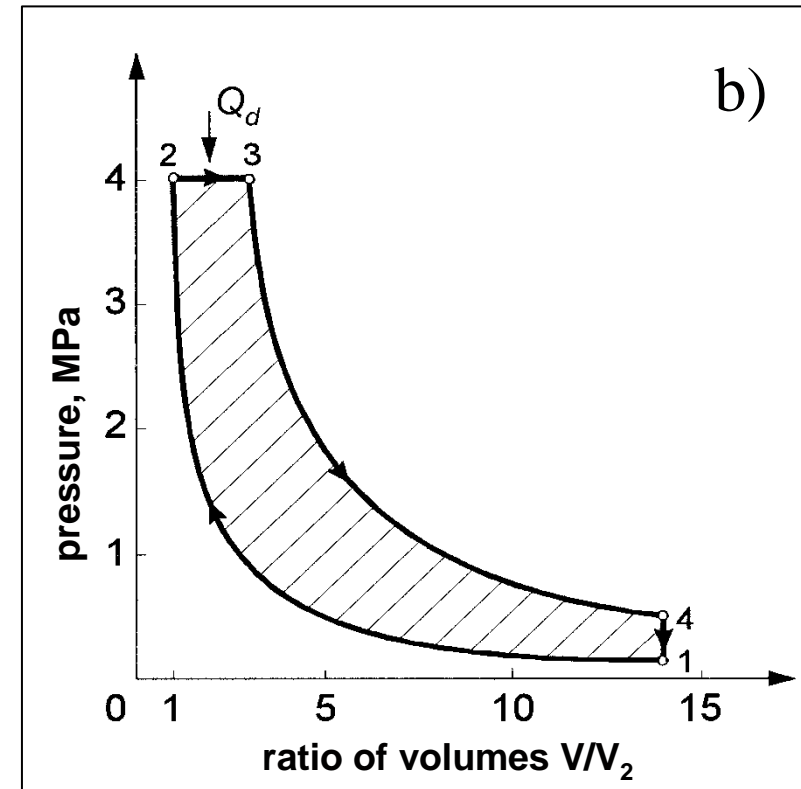
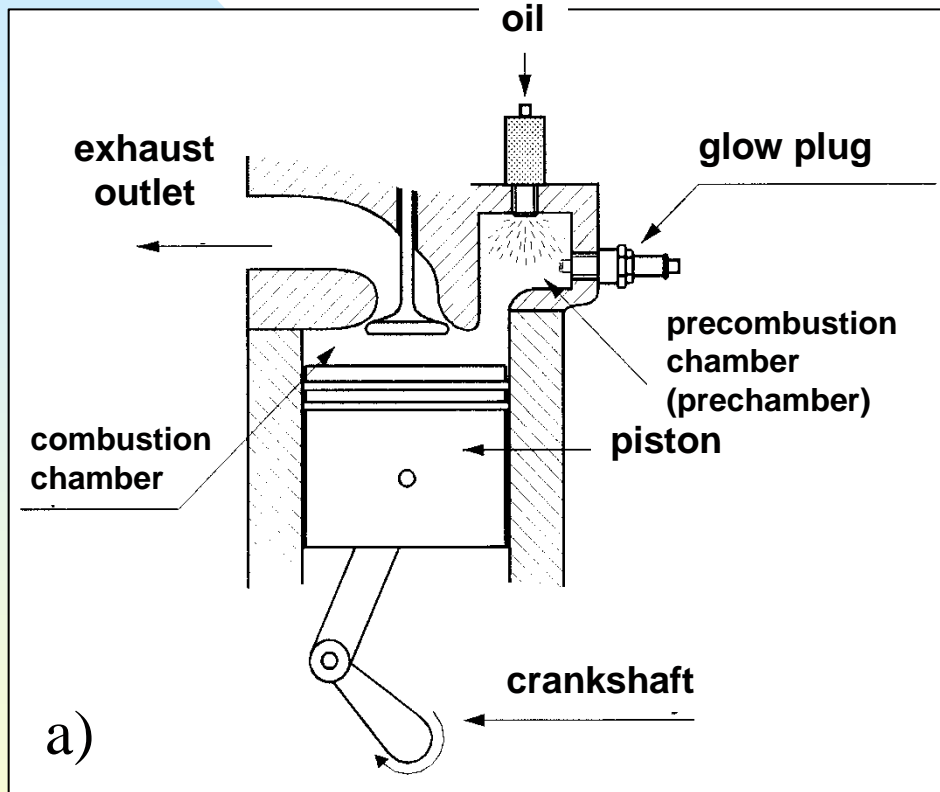




CI engine combustion

(CI - compressed – ignition)

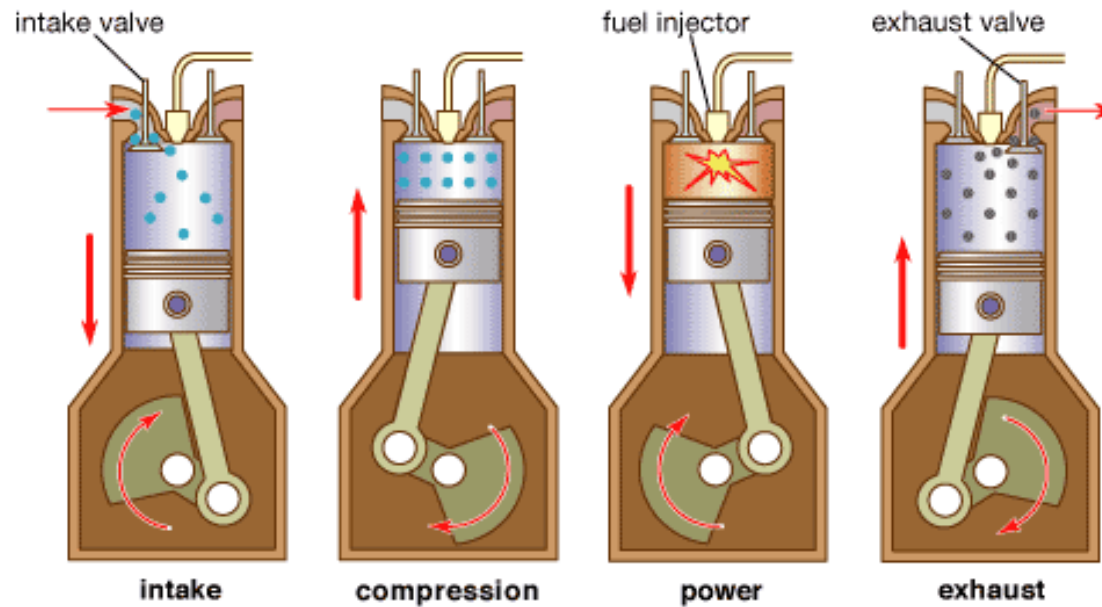
CI (Diesel) engine



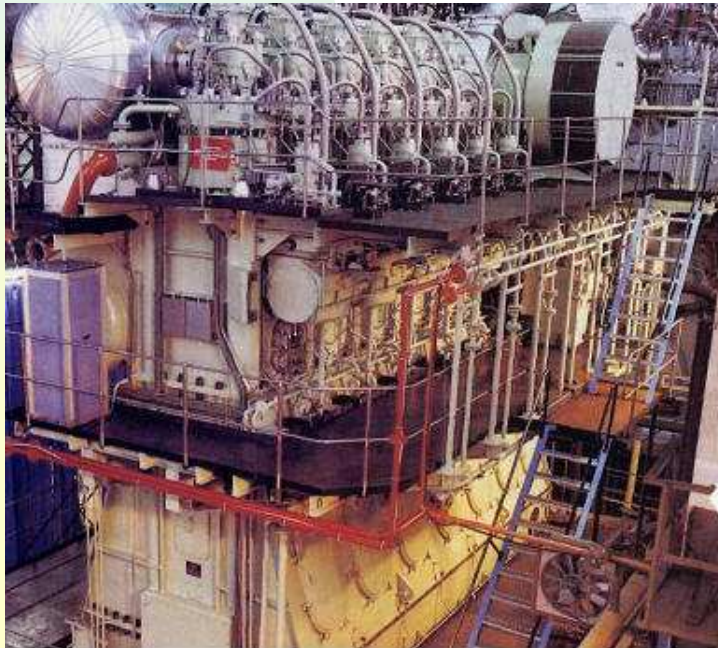
a) scheme of CI engine,

b) Diesel cycle

Diesel engines 4 and 2 - stroke



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Industrial Diesel engine

Selected parameters of CI engines

Compression ratio ε in Diesel engine

from 14:1 to 24:1

Passenger cars 21:1- 24:1

Trucks: 15:1-19:1

Compression pressure: 3-5 MPa

Combustion pressure: 5-8 MPa

Engine speed: 3000-5000 revolutions per minute



CI ENGINE COMBUSTION

Major processes in the combustion mechanism

I. Physical processes

1. Oil is injected into the combustion chamber
2. The oil jet is atomised to droplets
3. Droplets undergo evaporation.
4. Vapours are mixed with hot air and combustible mixture is formed.

II. Chemical processes

1. Low-temperature oxidation of hydrocarbons and alkylperoxy radicals formation
2. Cold flames
3. Blue flames
4. Hot flames

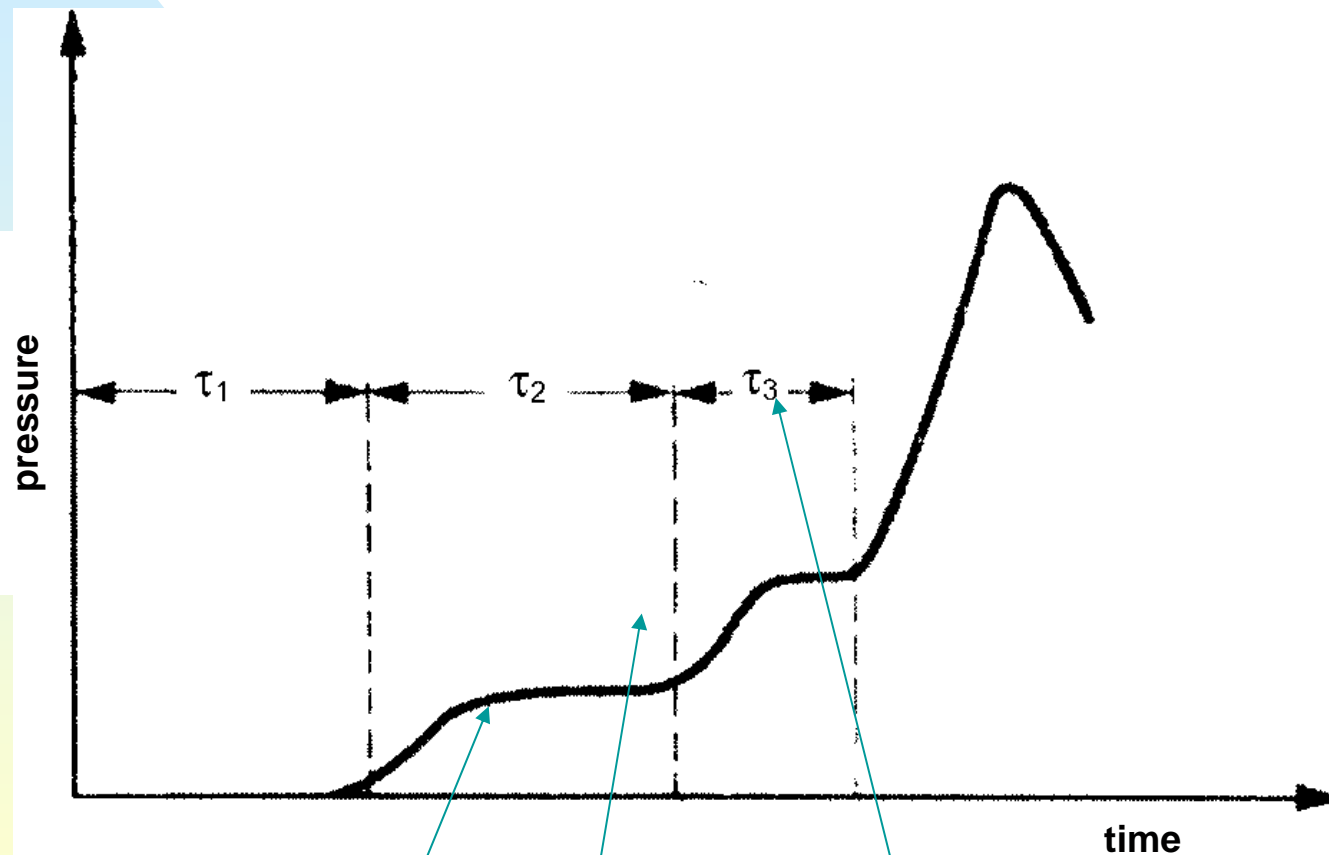
Major stages of combustion in CI engines

- I. Induction ignition period
- II. Kinetic combustion period
- III. Diffusion combustion period
- IV. Reburning period

Ignition delay period

- I. Oil is injected when the temperature of air reaches the **temperature of selfignition** (approx. 250 C).
- II. However, there is some delay of ignition – **induction ignition delay**.
- III. Induction ignition delay includes **physical and chemical delay periods (0.7-3 ms)**.
- IV. **The physical induction ignition period** is measured from the moment of oil injection to the moment of formation of combustible oil and air mixture.
- V. The chemical induction ignition period is measured to the moment of pressure indication.

Main combustion stages



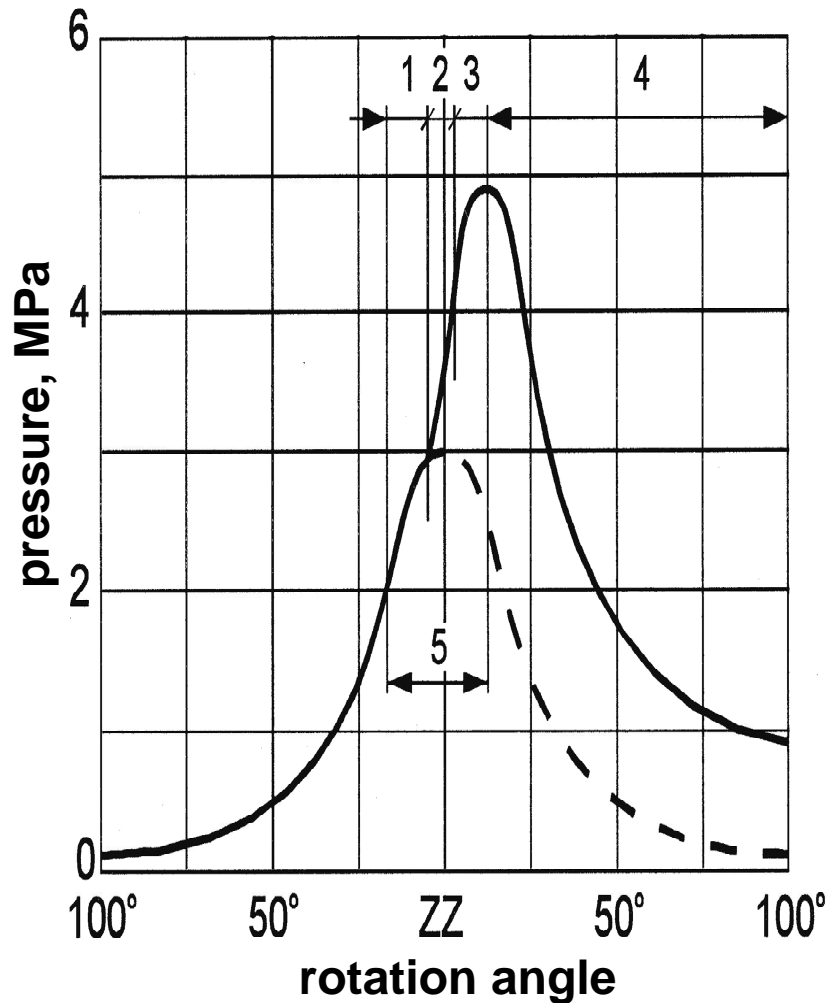
Flames: cold, blue, hot-kinetic, hot-diffusion

Selfignition temperature of selected fuels

Fuel	Temperature, °C
Diesel oil	233 (LC=41), 230 (LC=55), 225 (LC=55)
Gasoline	440-470
Ethyl alcohol	558, 426, 365
Methyl alcohol	574, 470, 464, 385
Methane	632, 537, 540
Propane	493, 450, 466, 504
n-Butane	408, 543, 477
Hydrogen	572, 400

Higher density, lower temperature of selfignition

Pressure record vs angle of shaft rotation angle



—— with combustion,

- - - without combustion

1 – induction ignition period,

2 – kinetic combustion period,

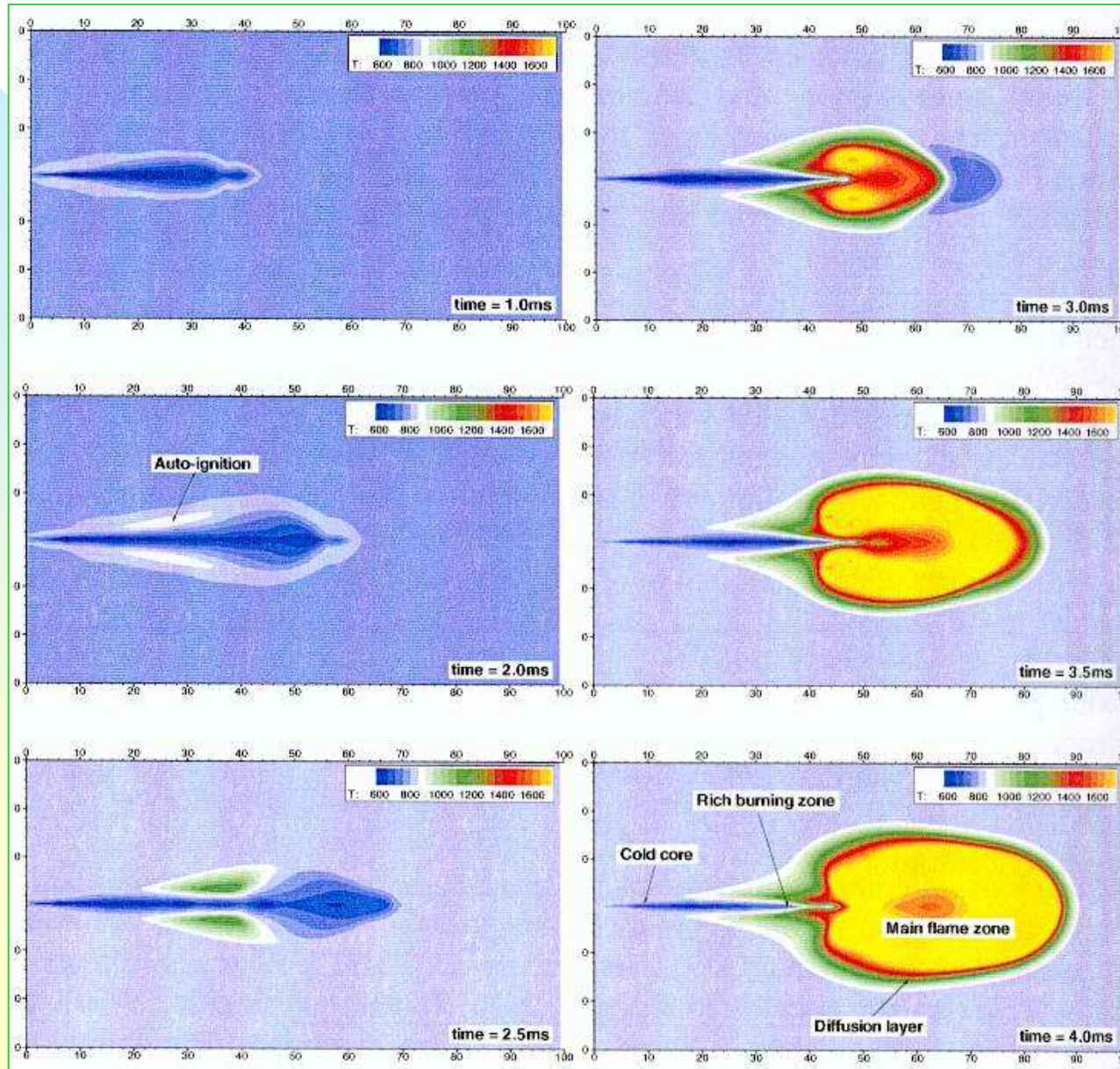
3 – diffusion combustion period,

4 – reburning period,

5 – injection period,

ZZ – upper position of piston

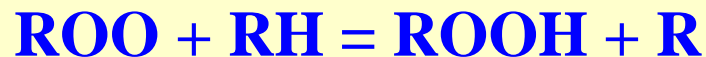
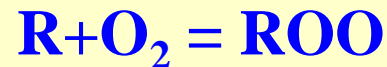
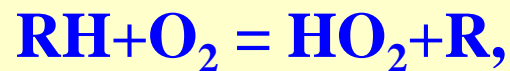
Flame Formation



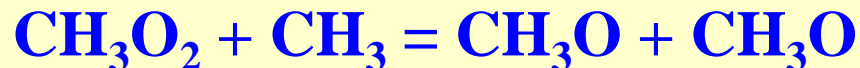
Phase of low-temperature reaction

Chemical processes

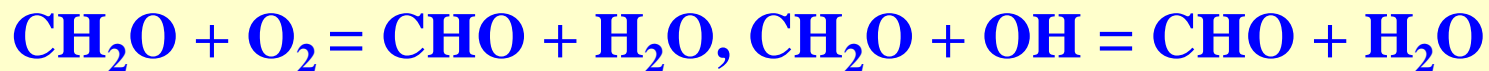
1. Formation of hydroperoxides:



2. Formaldehyde formation (cold flames):



3. Formaldehyde destruction:



4. Hot ignition and hot flames: $\text{CHO} + \text{M} = \text{CO} + \text{H} + \text{M}$

Hot combustion phase

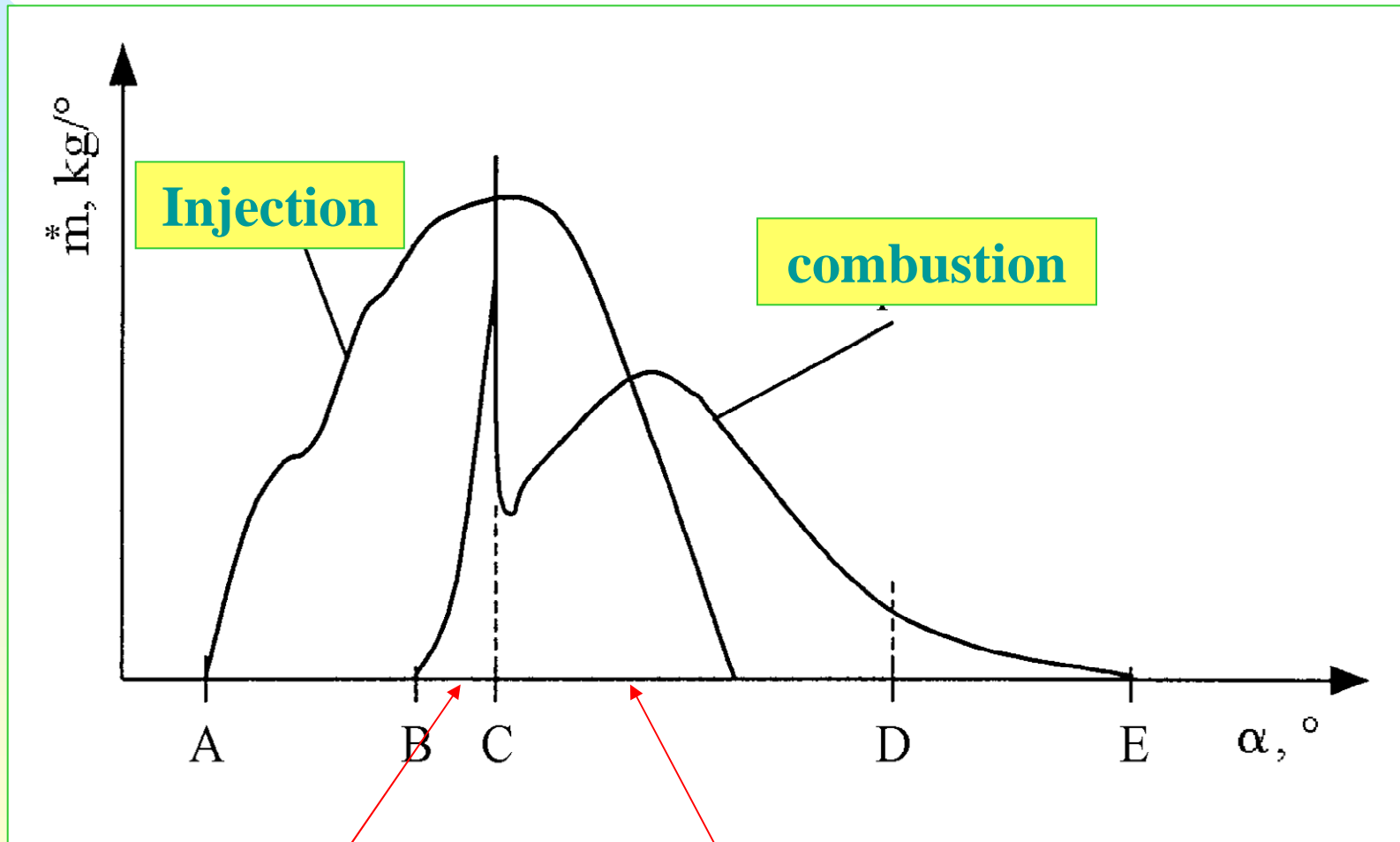
I. Kinetic phase

1. Vapour and air mixture undergoes ignition
2. Flame is formed
3. Charge is rapidly burned

II. Diffusion phase

1. Single droplets of oil undergo ignition
2. Diffusional flame is formed around the droplet
3. Droplets are burnt in diffusion regime
4. Reburning is in kinetic regime.

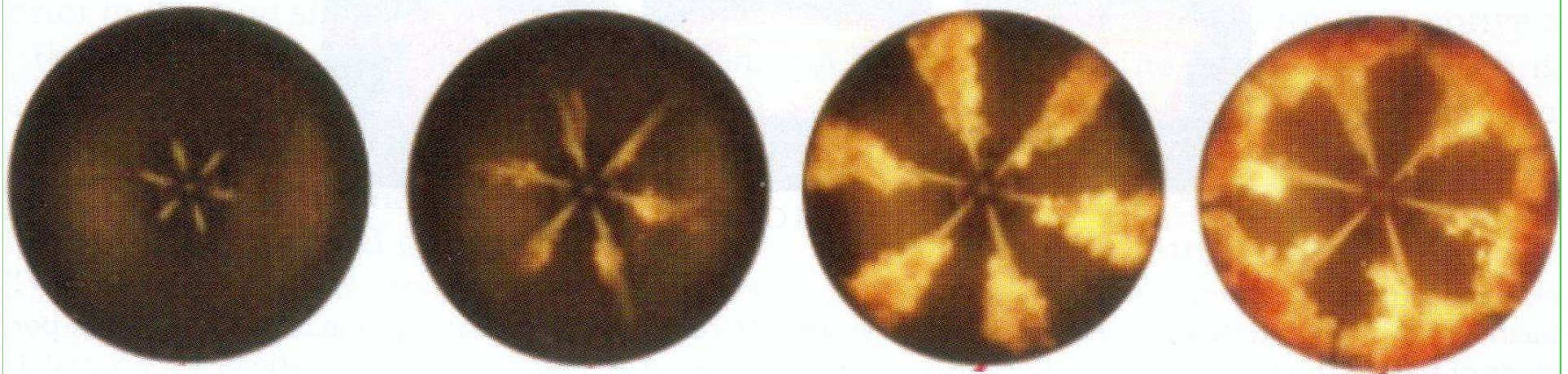
Combustion development in SC engines



Kinetic combustion

Diffusion combustion

Flame formation in CI engine





CI ENGINE COMBUSTION SYSTEMS

Types of Diesel engines

1. Low- speed engines $(n < 1500)$
2. Medium-speed engines $(n = 1500-3000)$
3. High-speed engines $(n = 3000-5000)$

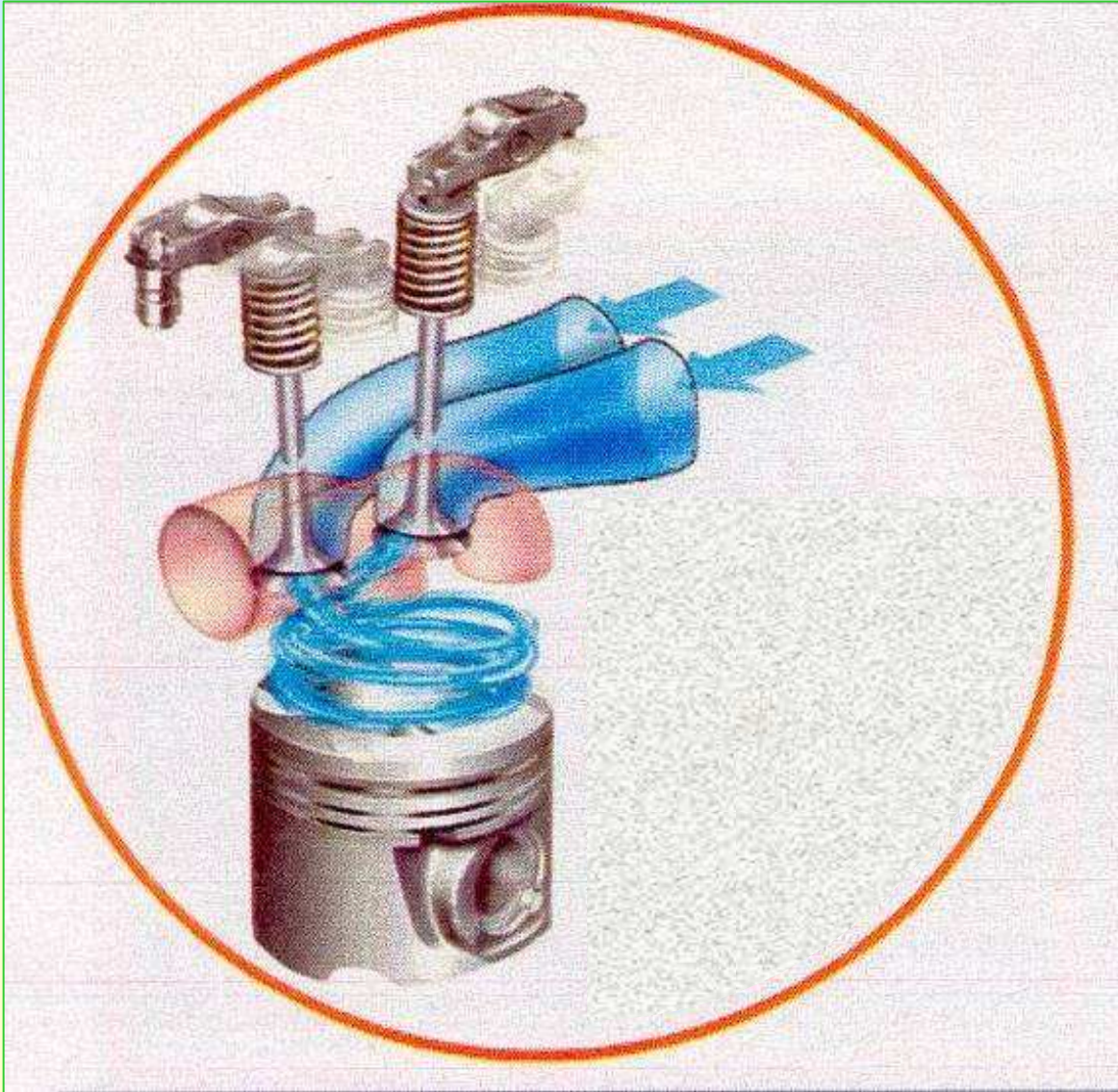
Combustion chambers in low- and medium speed CI engines ($n < 1500$)

1. Shallow, swirl-less combustion chambers.
2. Direct, multi-jet fuel injection.

Medium-speed CI engines ($n = 1500-3000$)

1. Deep combustion chambers with intensive swirl of charge.
2. Direct injection of atomised fuel.

Intensive swirl of charge ($n = 1500-3000$)



swirl
makes
the
mixture
(charge)
uniform

High-speed CI engines

(n = 3000-5000)

1. Prechambers (sectional combustion chambers).
2. Indirect injection of fuel into the prechamber.

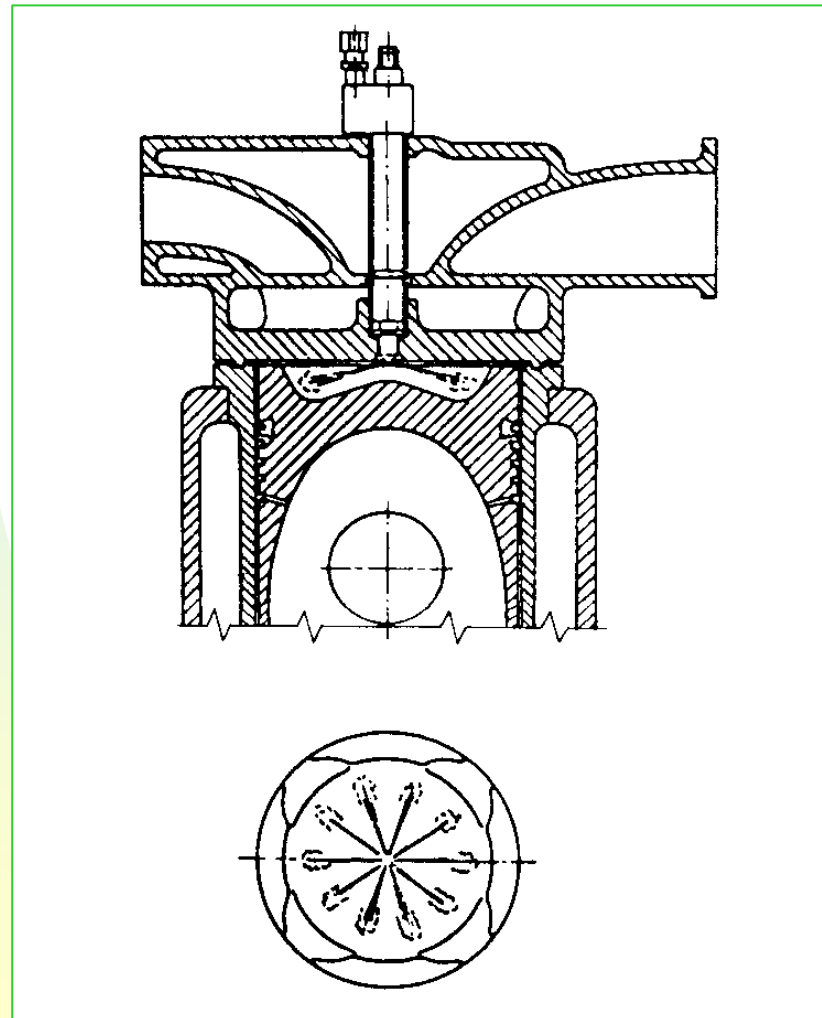


COMBUSTION CHAMBERS OF CI ENGINES

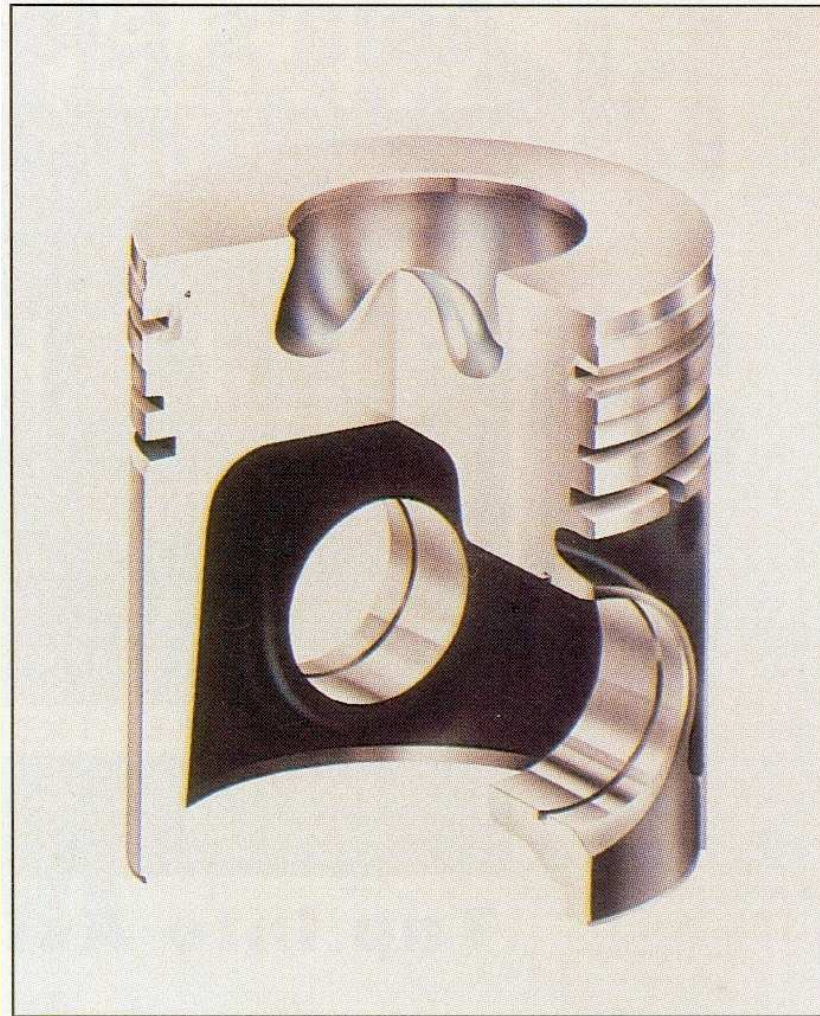
Combustion chambers of CI engines

1. *Direct injection* (DI) chambers.
2. *Indirect injection* (IDI) chambers.

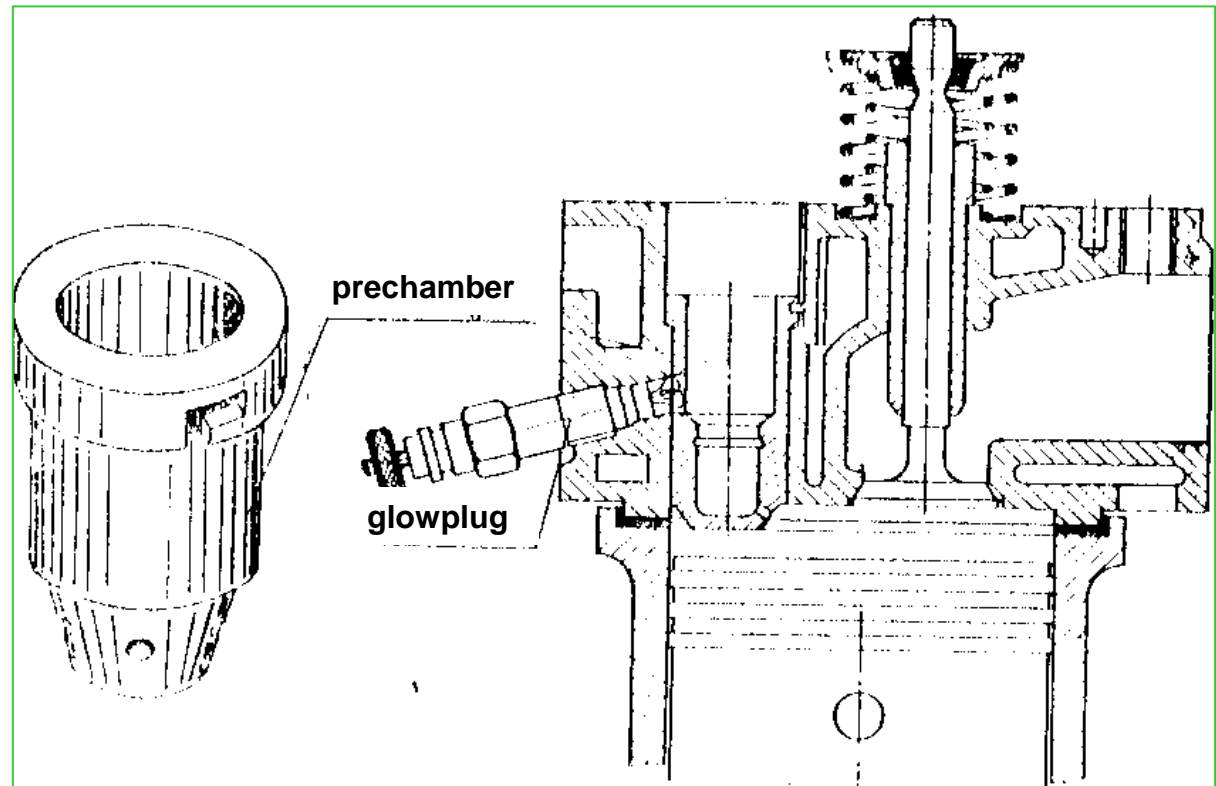
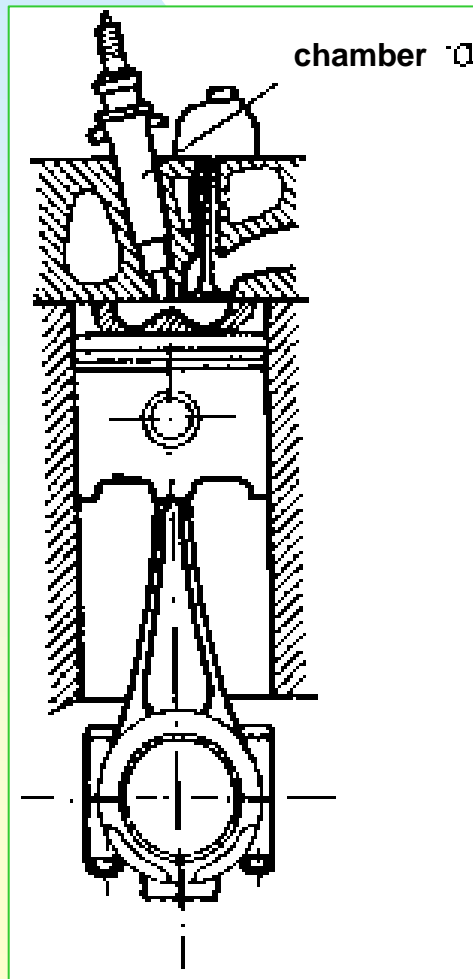
Combustion chambers of *direct injection* engines (DI)



Piston of DI engine (*direct injection*)

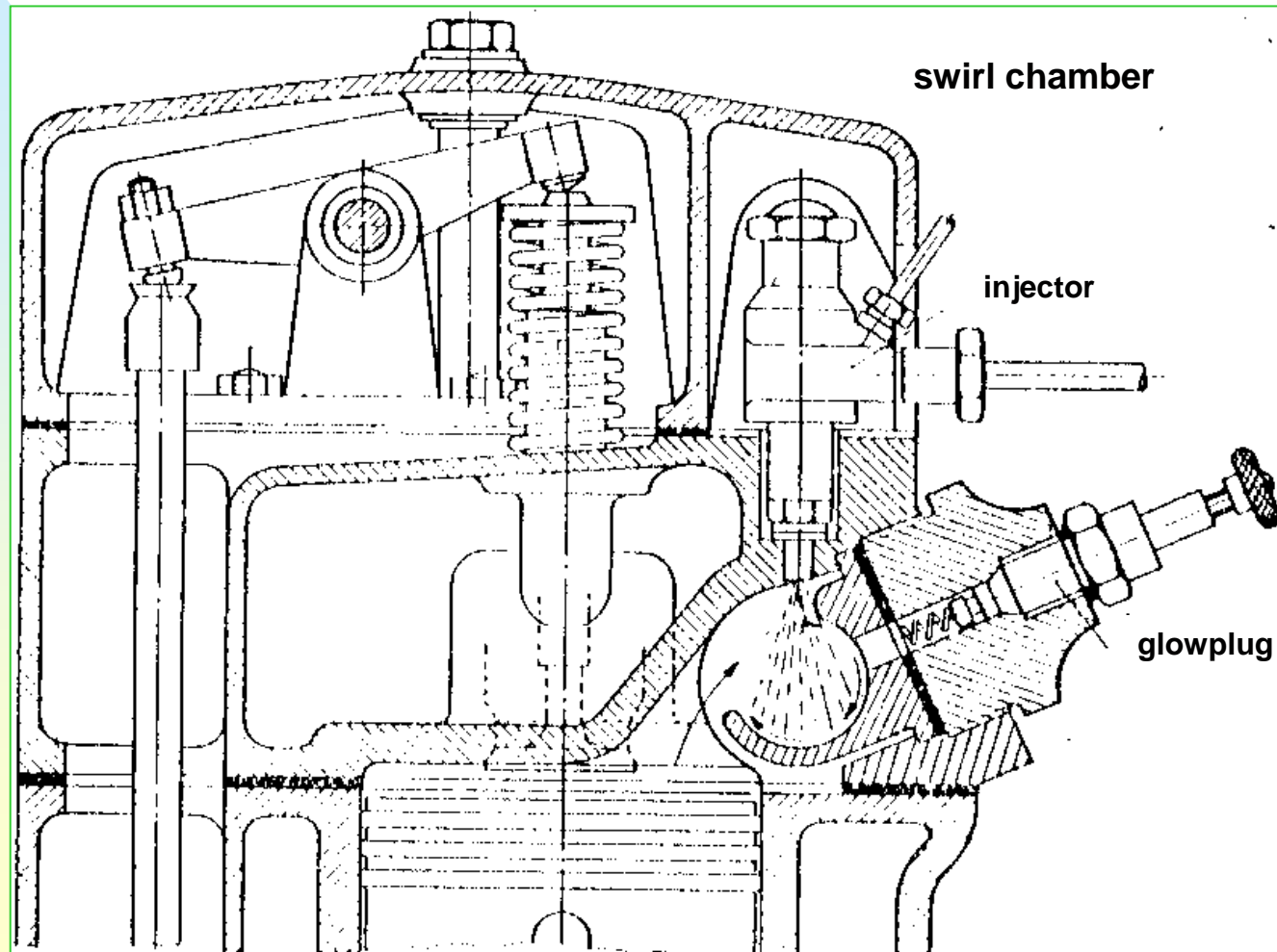


Combustion chamber of *indirect injection* (IDI)



prechamber

Combustion chamber of *indirect injection* (IDI)



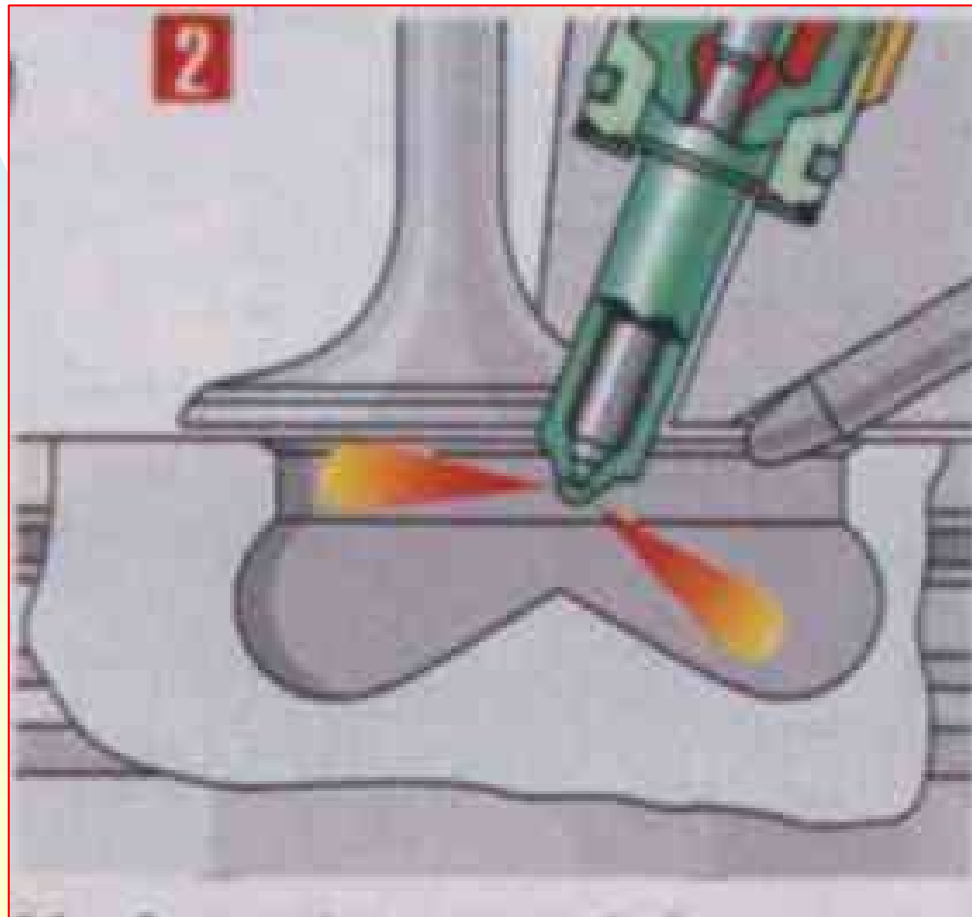


CI ENGINE FUEL INJECTION SYSTEMS

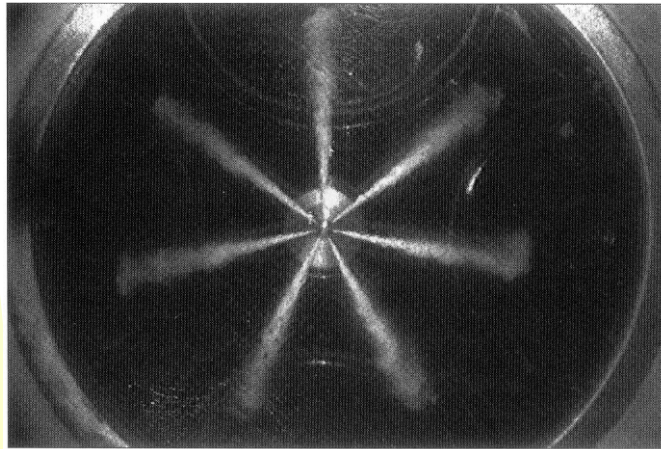
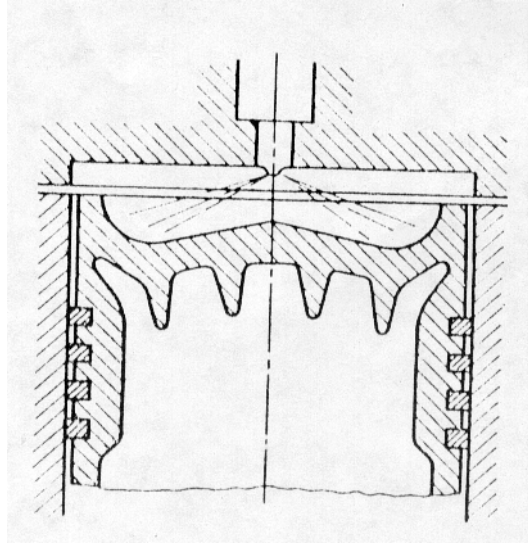
Injection systems of CI engines

- 1. Direct injection systems DI.*
- 2. Indirect injection IDI.*

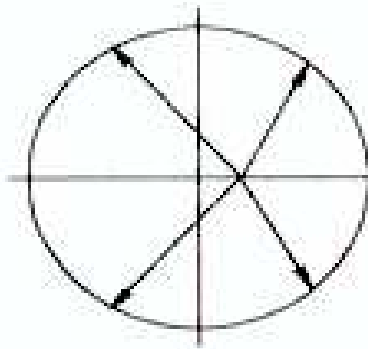
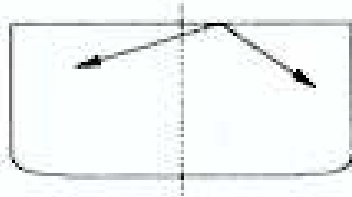
Direct injection DI



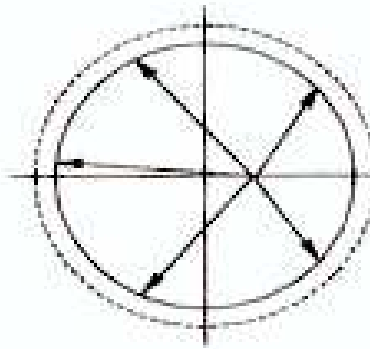
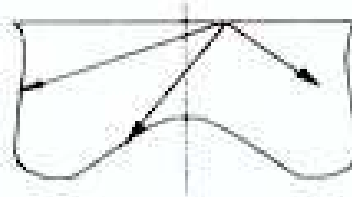
DI (*direct injection*)



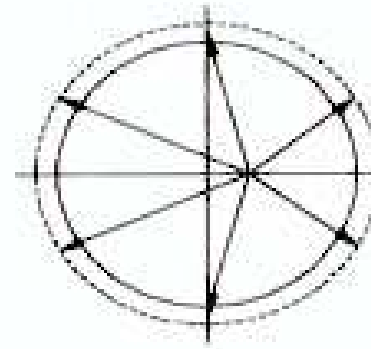
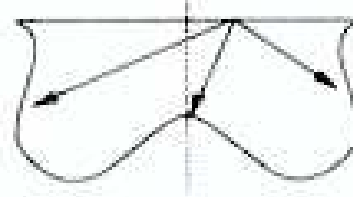
Direct injection systems DI



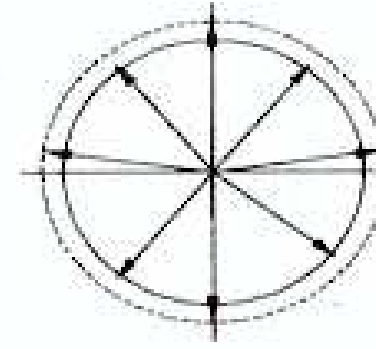
**4-holes
injector**



**5-holes
injector**



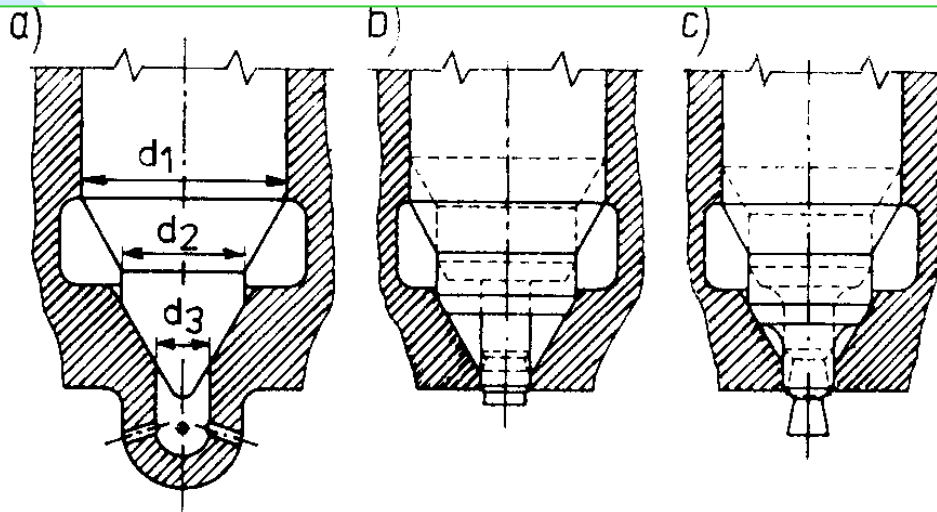
**6-holes
injector**



**8-holes
injector**

Shape of combustion chamber in piston

Diesel oil injectors

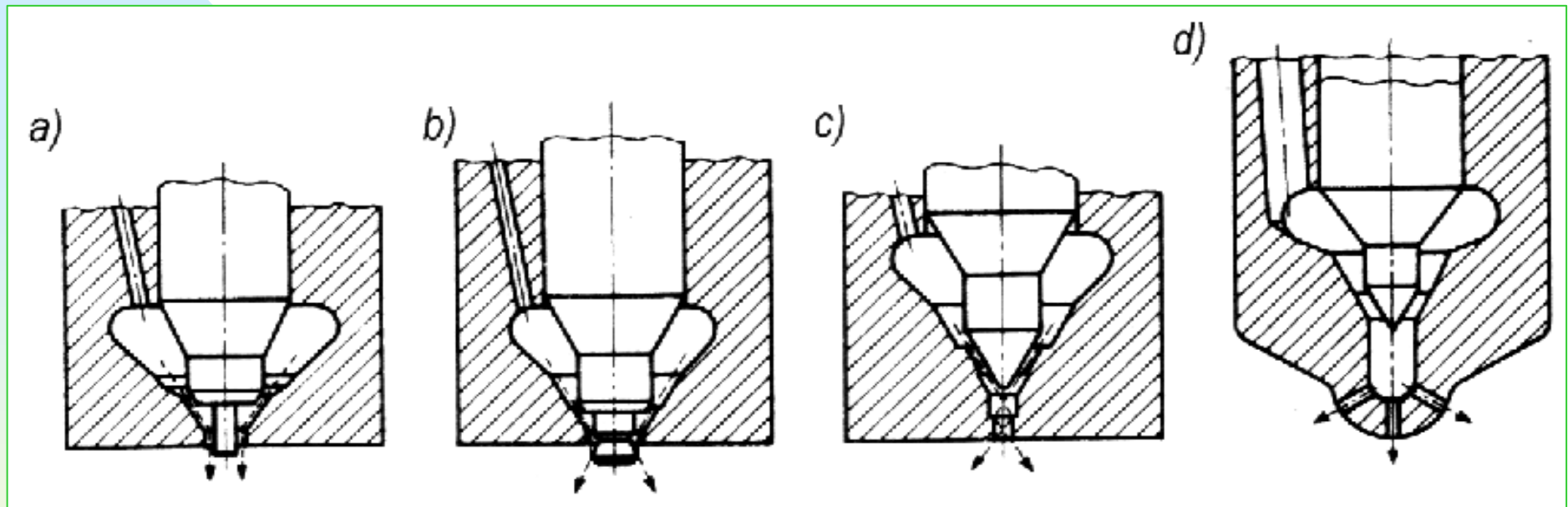


Rys. 11.7. Typowe końcówki wtryskiwaczy do silników Diesla: a) wielootworkowego, b) standardowego czopikowego, c) czopikowego dławiącego [2]

Typical injector nozzels

- multihole
- standard with needle
- chocking with needle

Types of Diesel oil nozzles



Types of injector

a) with cylindrically ended needle, b) with conically ended needle, c) single-hole, d) multi-hole

Typy rozpylaczy:

a) czopikowy z cylindrycznym zakończeniem iglicy, b) czopikowy ze stożkowym zakończeniem iglicy, c) jednootworowy, d) wielootworowy

DI and IDI nozzles



Delphi DI injectors

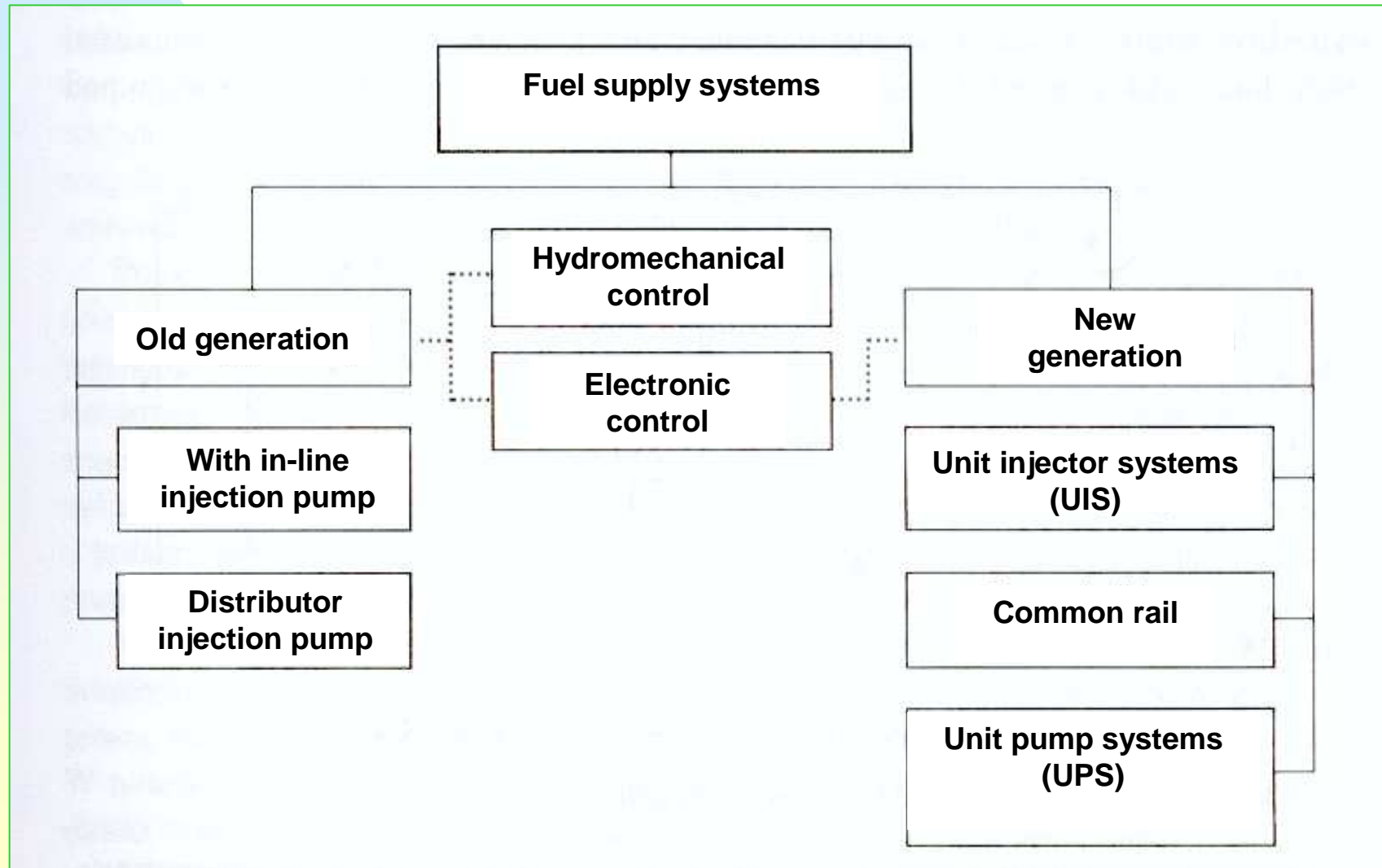


Delphi IDI injectors
with needle



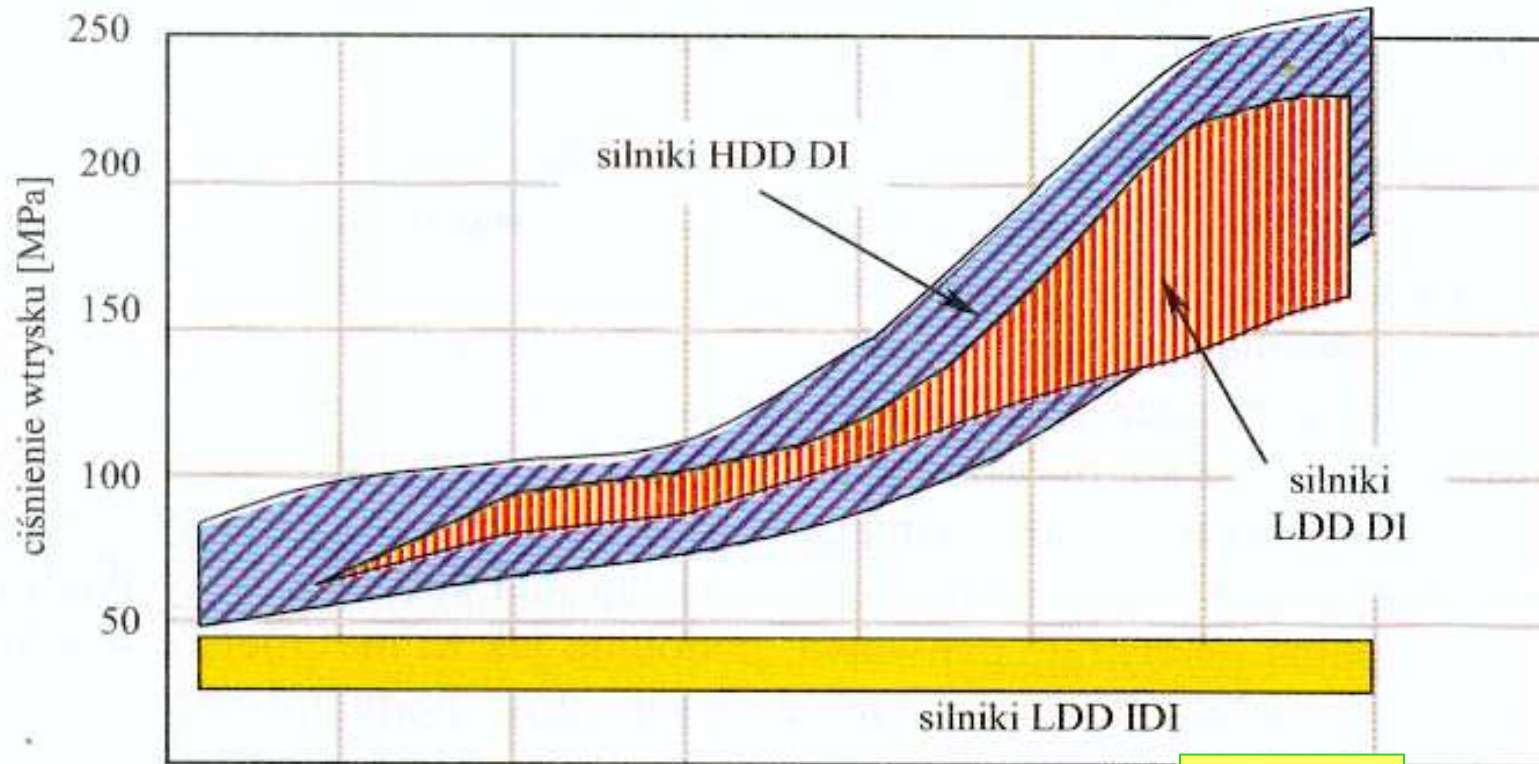
FUEL SUPPLY SYSTEMS IN CI ENGINES

Types of fuel supply systems in CI engines



Change of injection pressure of fuel in CI engines

**P
r
e
s
s
u
r
e**



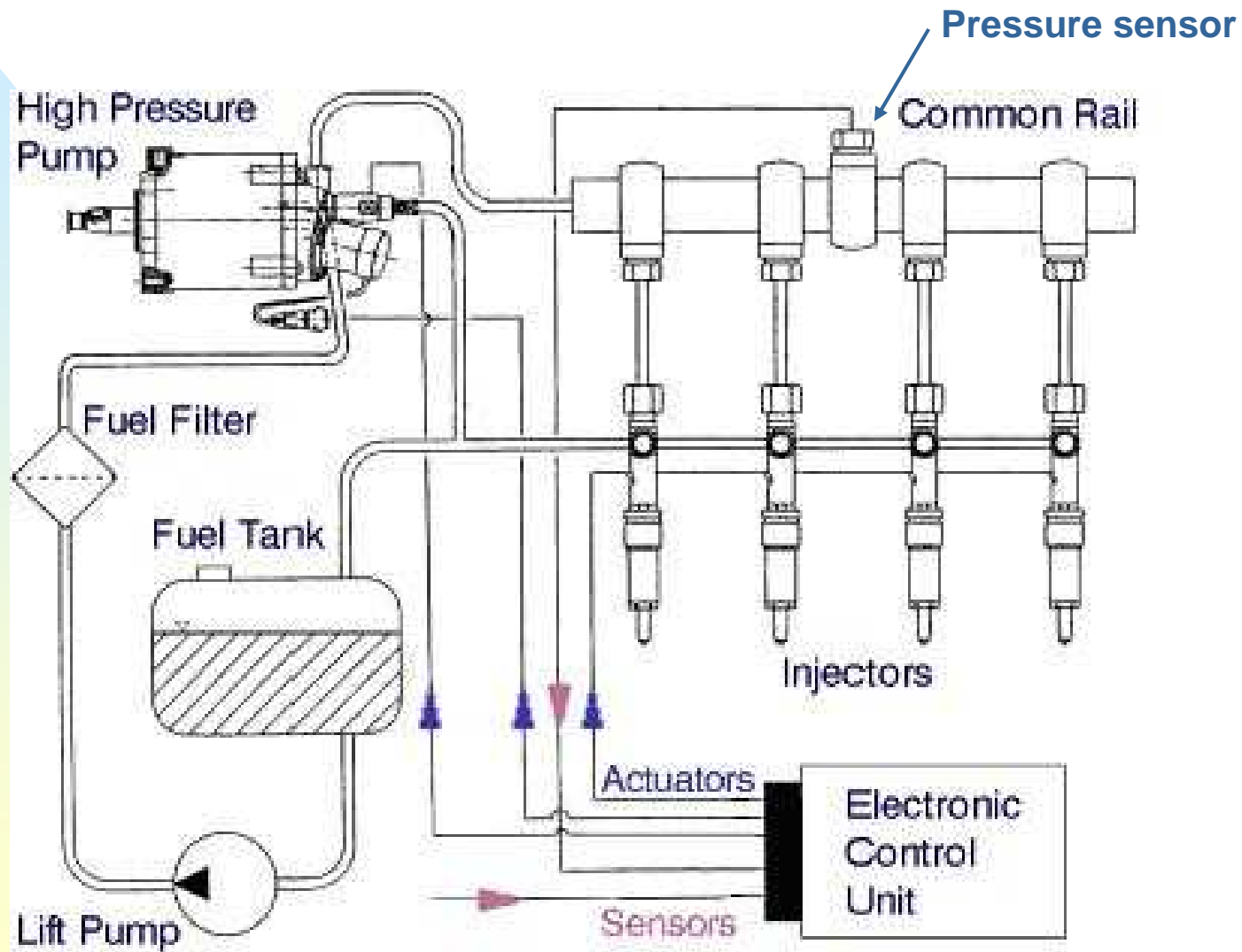
Time

HDD- heavy duty Diesel, LDD- light duty Diesel

Current Diesel oil injection systems

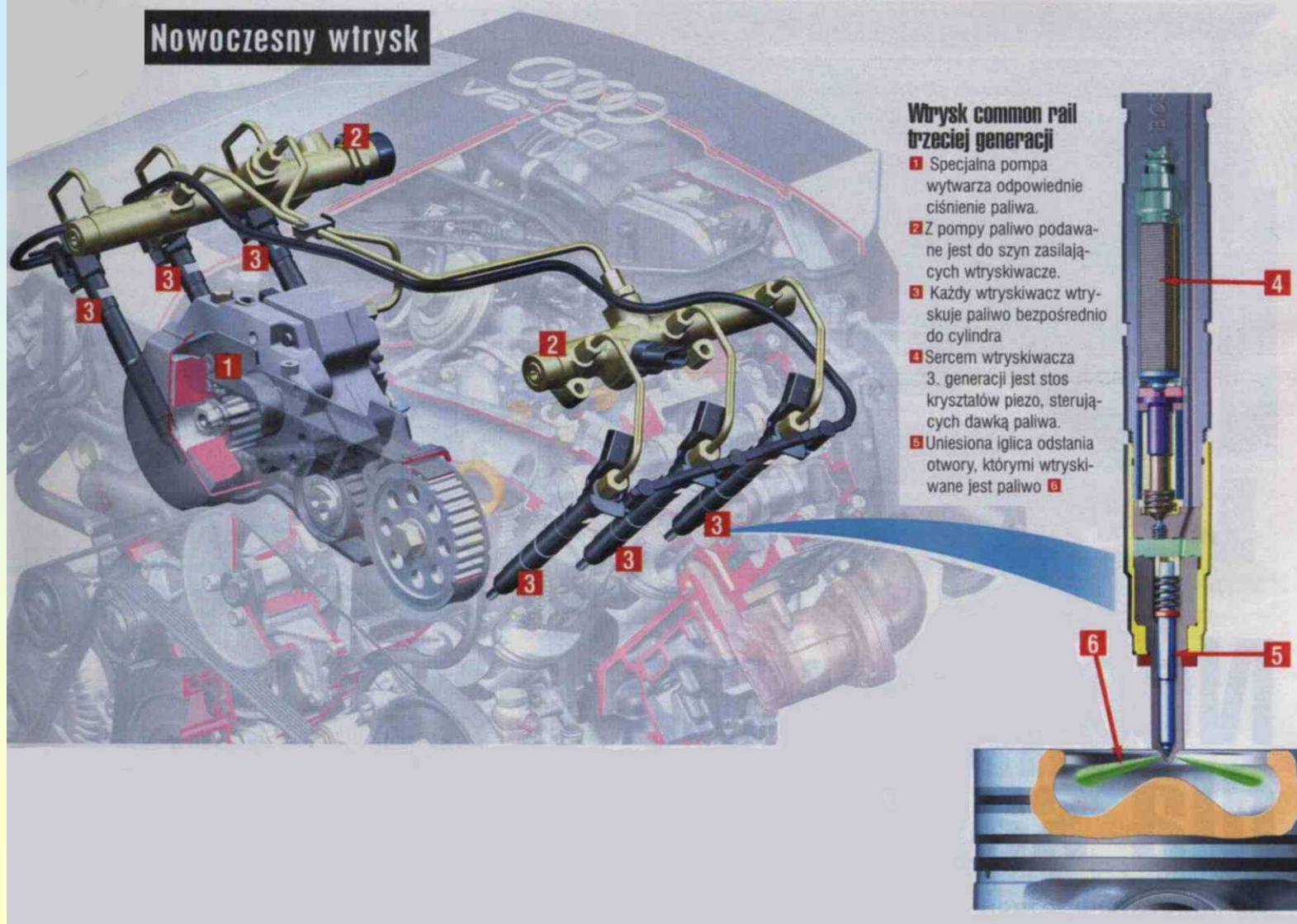
- 1. Common rail system.*
2. Pump-injectors.

Scheme of common rail system

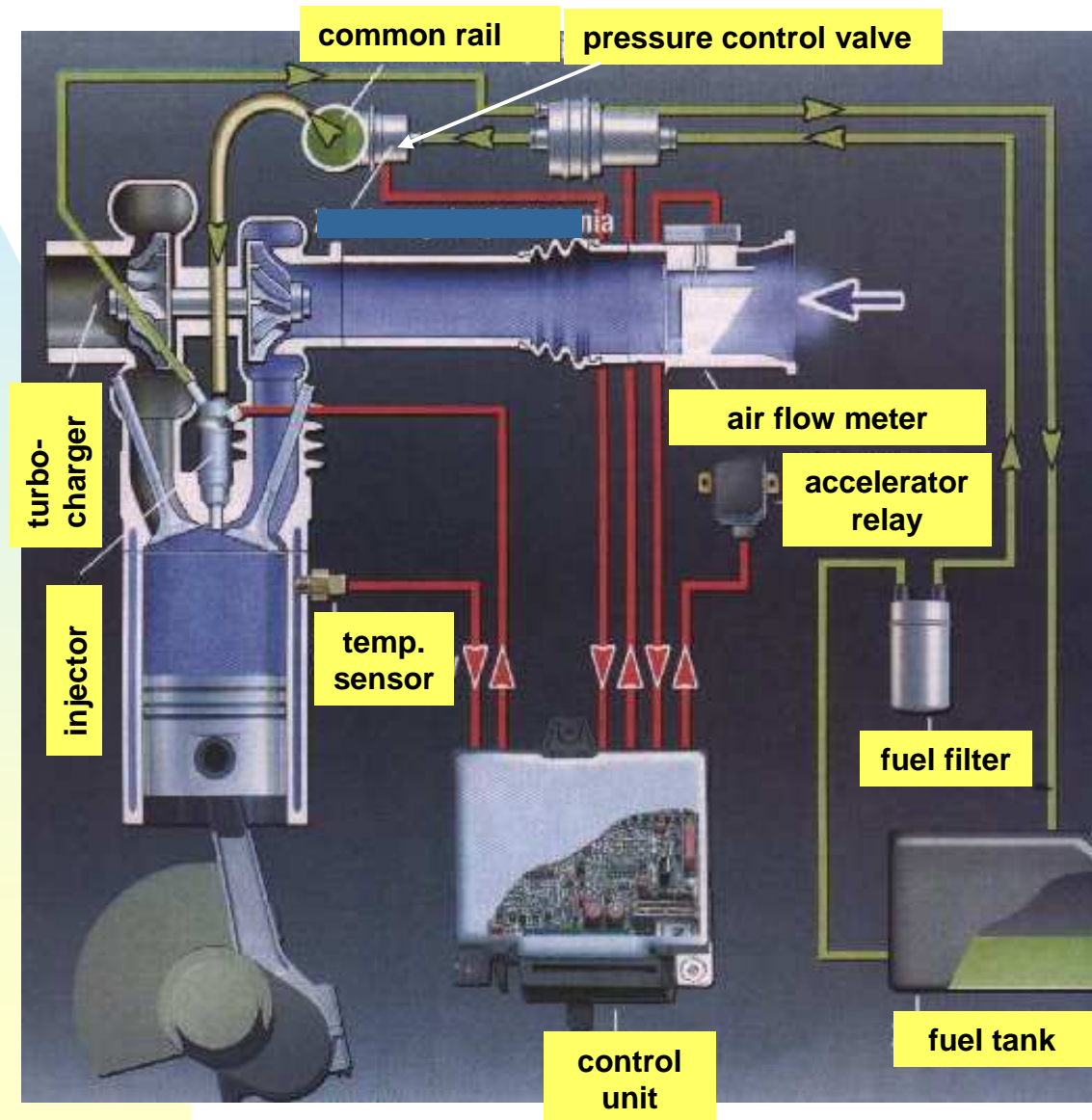


COMMON-RAIL

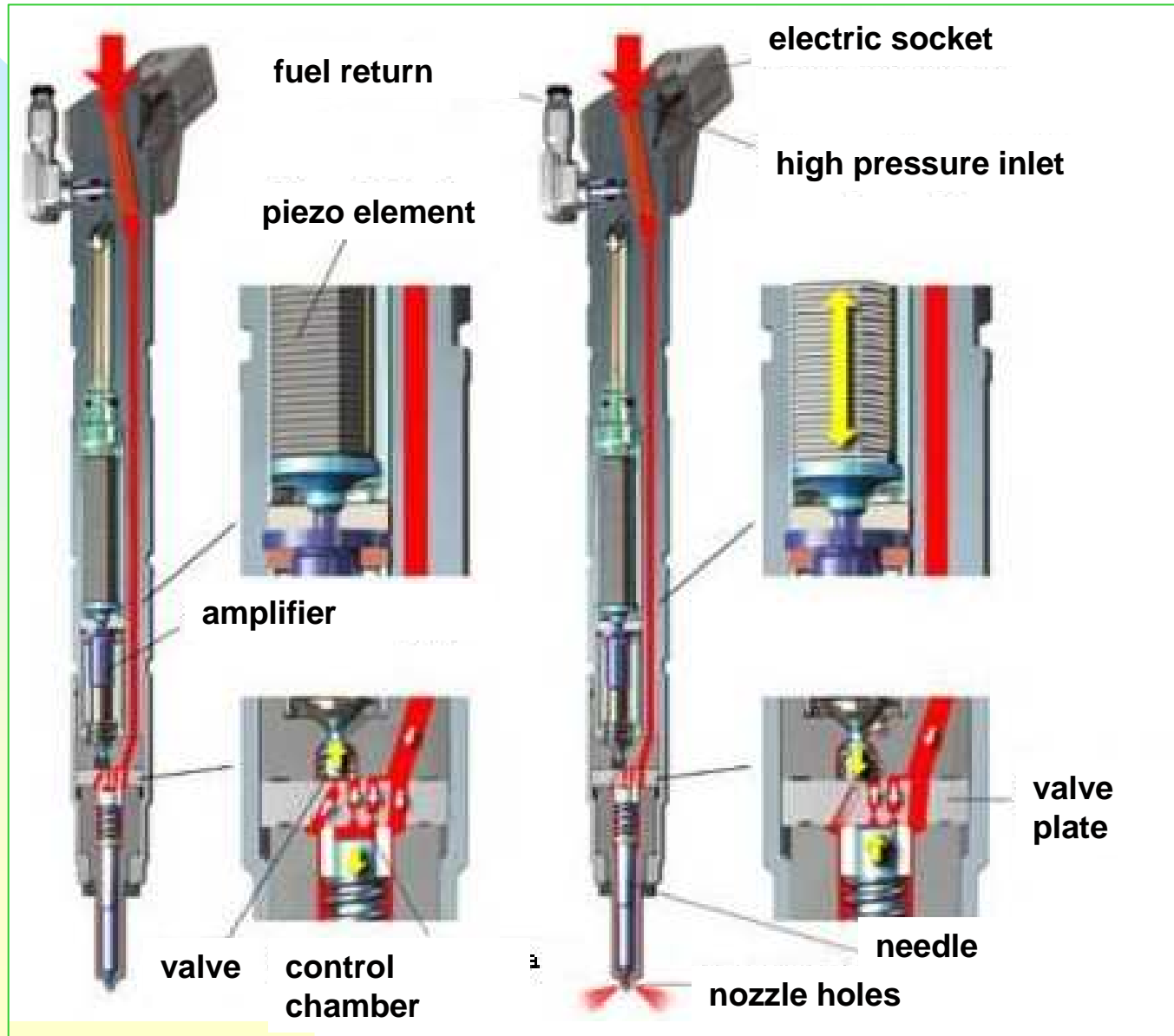
Nowoczesny wtrysk



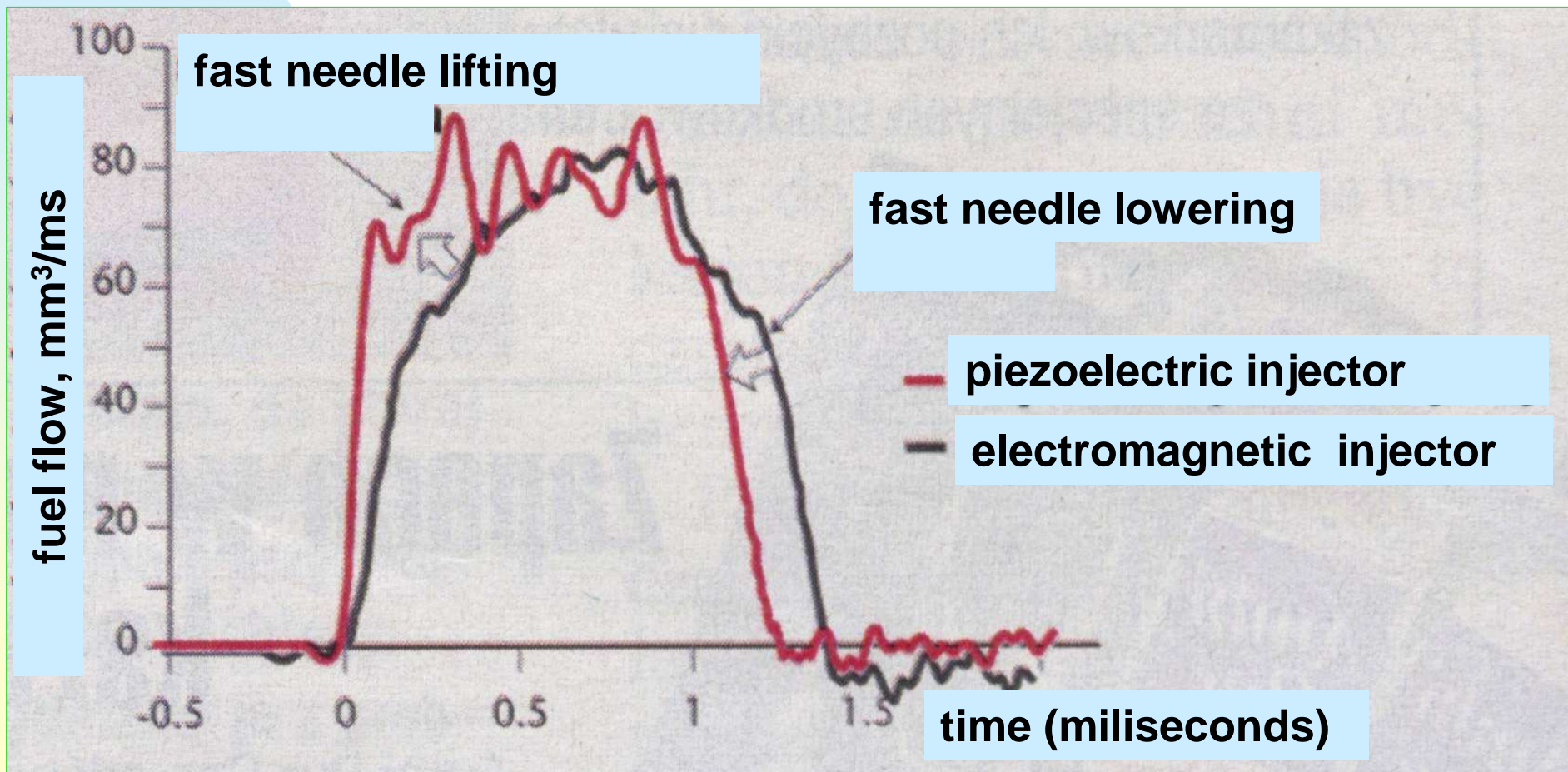
Scheme of CI engine with common-rail



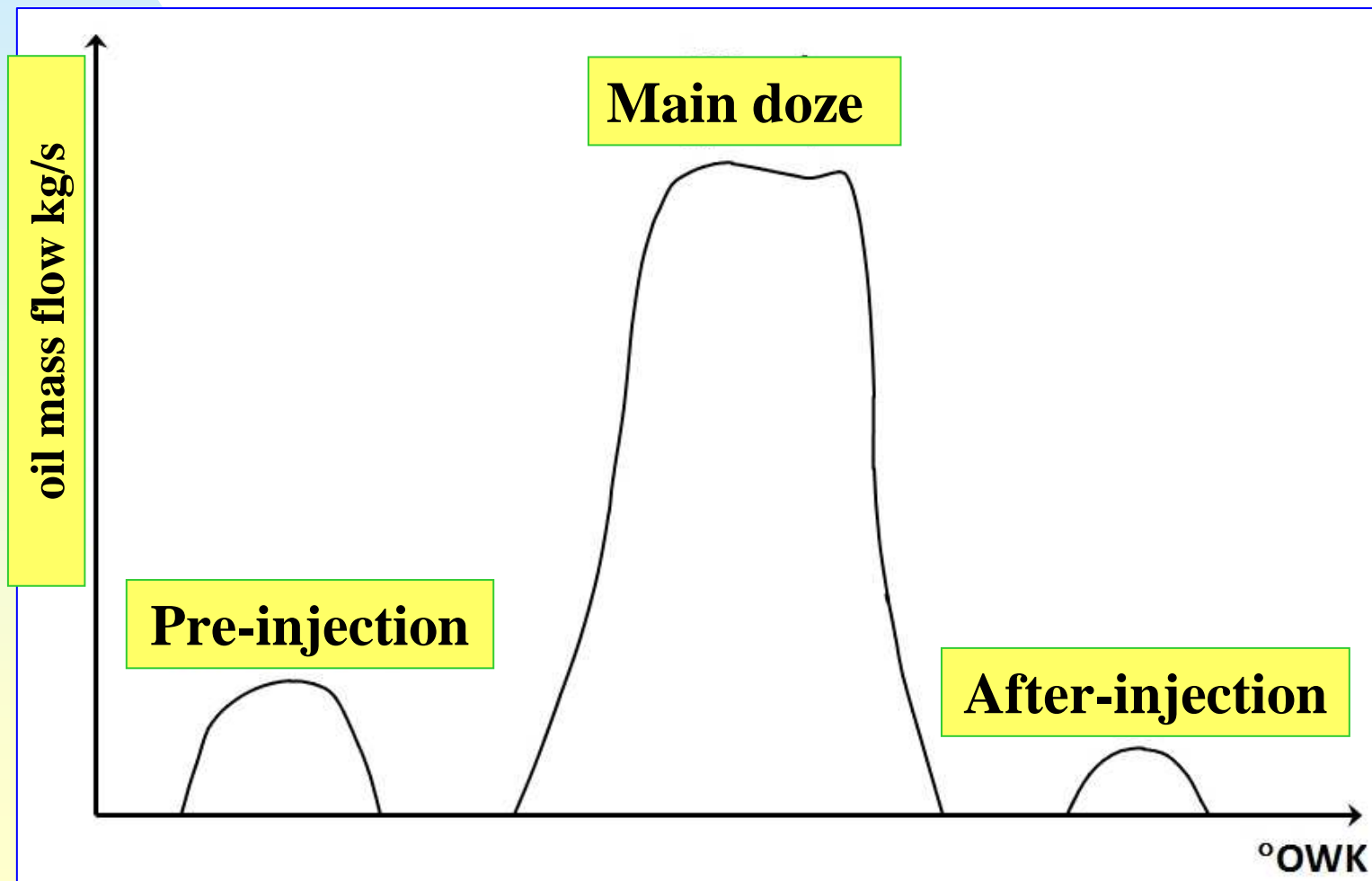
Piezoelectric Diesel oil injector



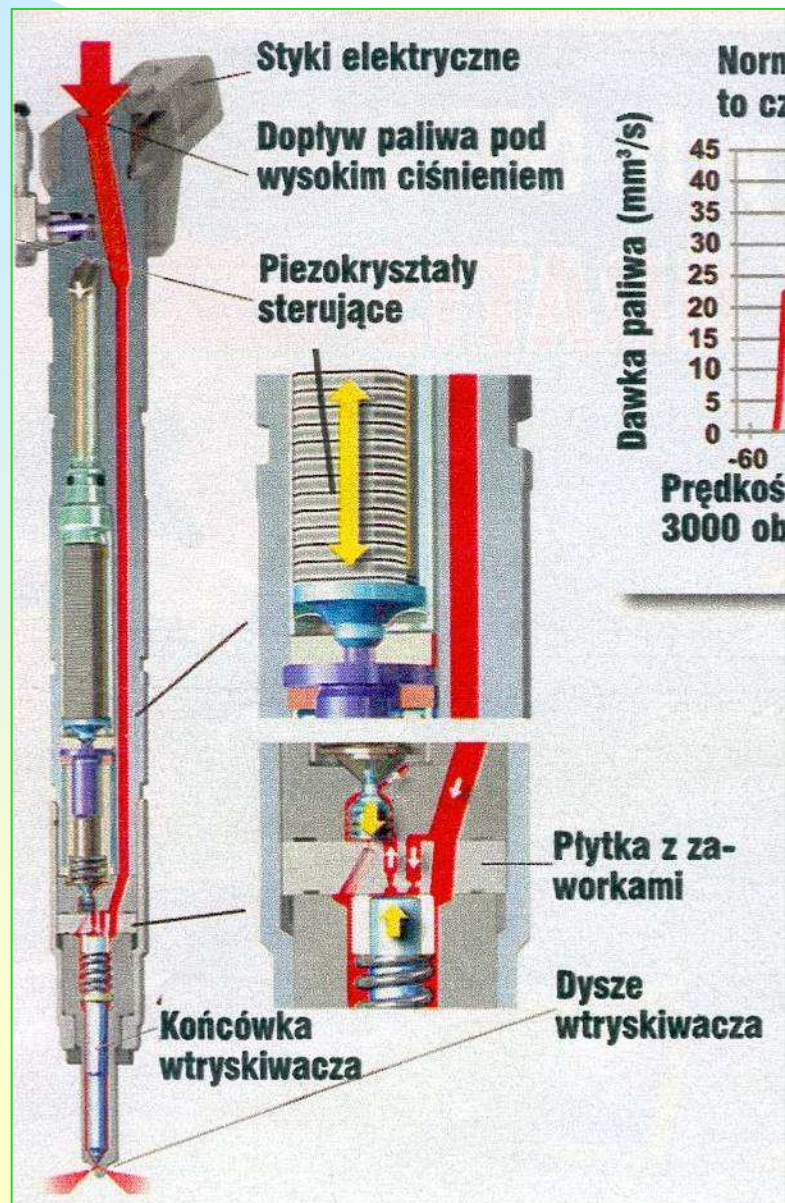
Comparison of piezoelectric and electromagnetic Diesel oil injectors



Divided Diesel oil injection

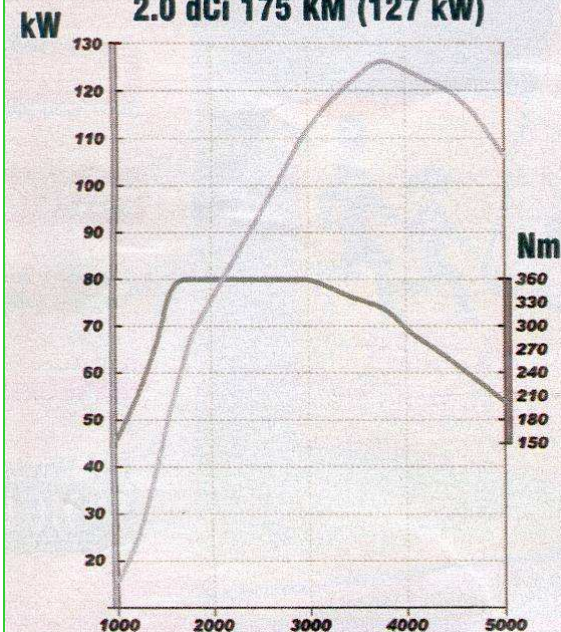


Common rail

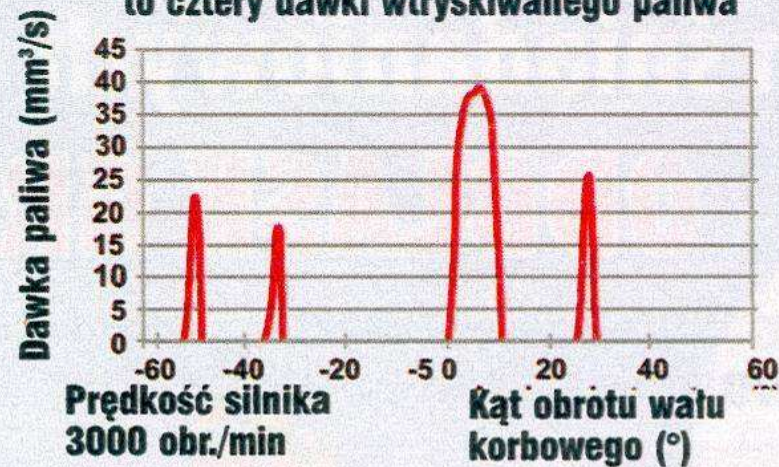


DWULITROWY SILNIK M9R rozwija moc 175 KM już przy 3750 obr./min

2.0 dCi 175 KM (127 kW)

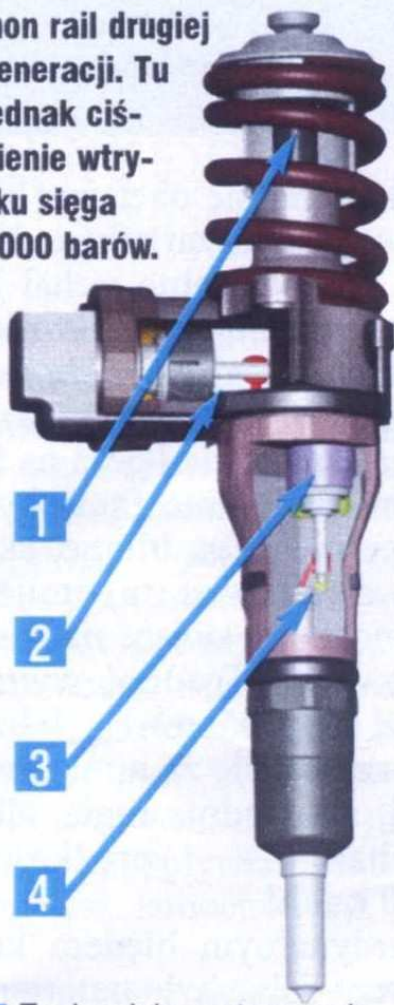


Normalna praca (bez wypalania sadzy) to cztery dawki wtryskiwanego paliwa



Pompowtryski 3. generacji

Dwa zawory magnetyczne pozwalają na szybkie podzielenie cyklu nawet na pięć wtrysków, podobnie jak w układach common rail drugiej generacji. Tu jednak ciśnienie wtrysku sięga 2000 barów.



Piezoelectric pump injector

- 1 Tradycyjnie, wytworzenie ciśnienia możliwe jest dzięki napędowi odbieranemu od krzywki wałka rozrządu.
- 2 Ten zawór magnetyczny steruje wzrostem i spadkiem ciśnienia.
- 3 Drugi zawór magnetyczny steruje otwieraniem i zamykaniem rozpylacza.
- 4 Element sterujący, tak jak w układzie common rail.



AIR SUPPLY

Air inflow systems of CI engine

1. In CI engines only air flows through the inlet system.
2. There is no throttle in the CI inlet system
3. The engine speed is controlled by the oil injection system.

Stoichiometry of combustion of CI engines

The stoichiometric ration in CI engines is greater than in SI engines because the charge is produced directly in the combustion chamber:

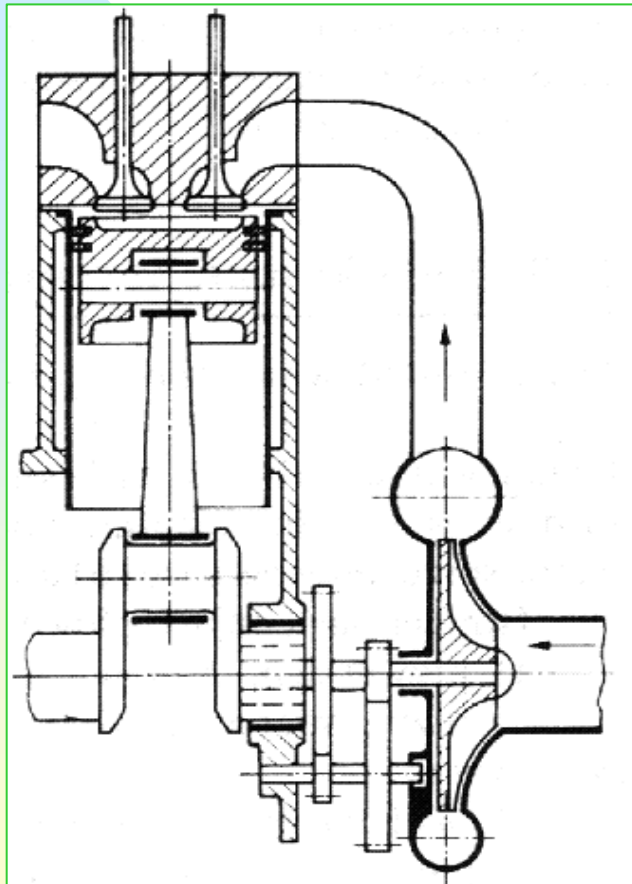
a) atmospheric charged engines: $\lambda = 1.3-1.4$

b) supercharged engines: $\lambda > 2$

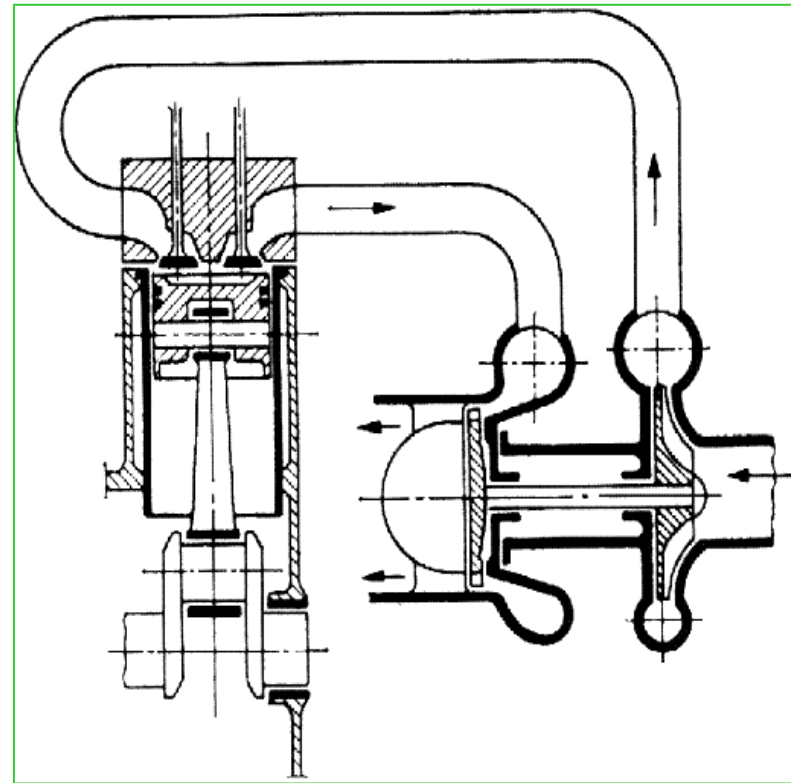
Supercharged CI engines

- Supercharge means to supply the cylinder with the charge of greater density (compressed air).
- Supercharge allow to increase of engine's power without increase it its capacity.
- Types of supercharge:
 - mechanical air compression,
 - gas turbine air compression.
- Compression pressure
 - low-pressure compression: < 0.15 MPa,
 - high pressure compression: from 0.3 to 0.5 MPa.

Types of air compression systems



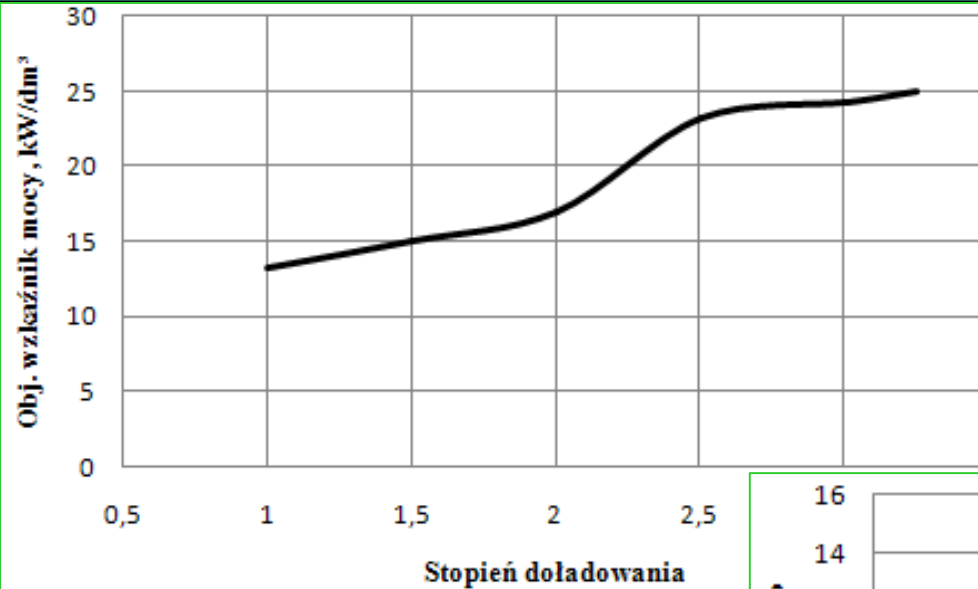
a) mechanical



a) gas turbine

Effect of supercharge on power and soot emission CI engines

Power per volume kW/dm³



Compression ratio

Fumes opacity, %



Compression ratio

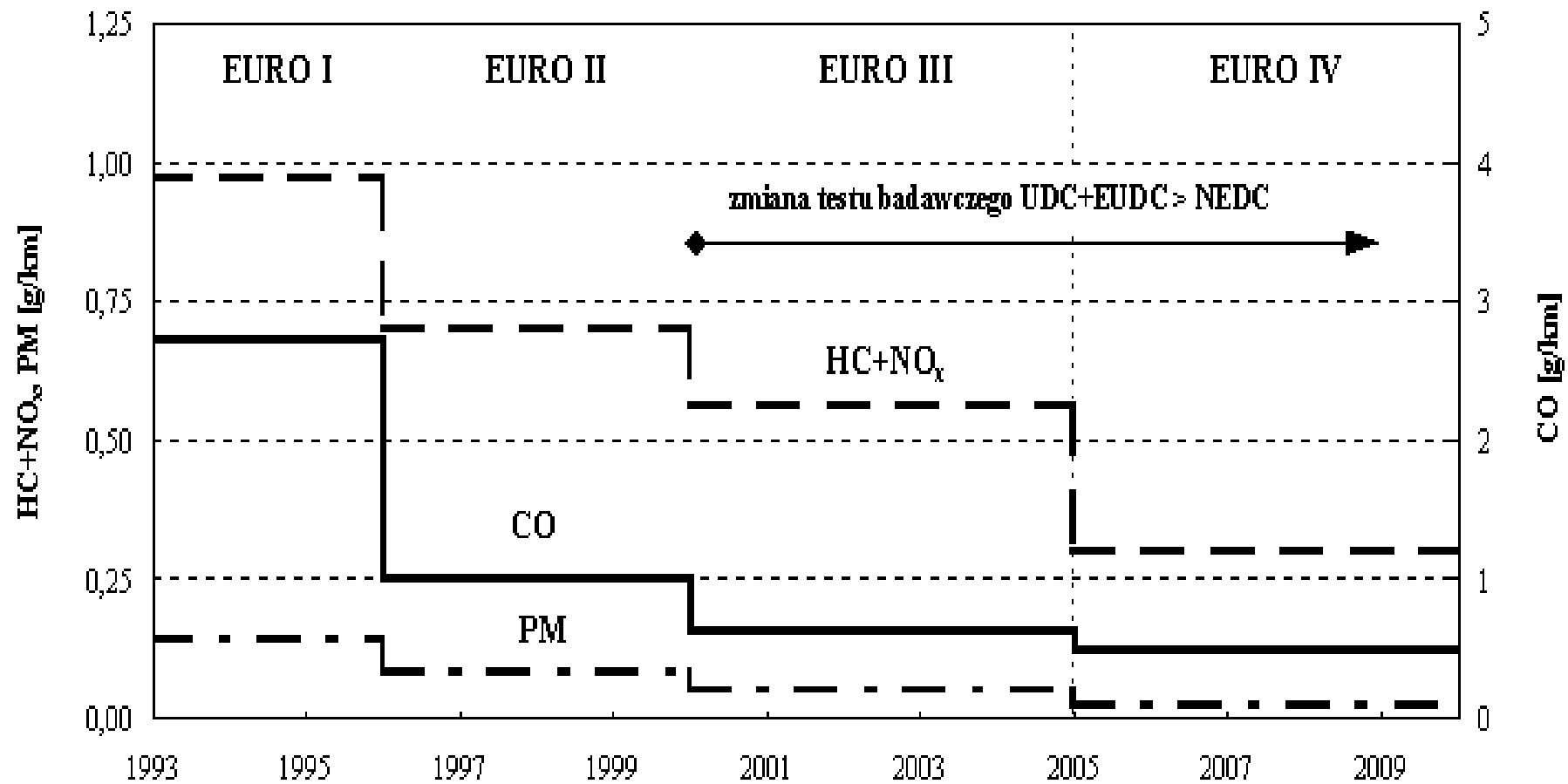


EMISSIONS OF POLLUTANT FROM CI ENGINES

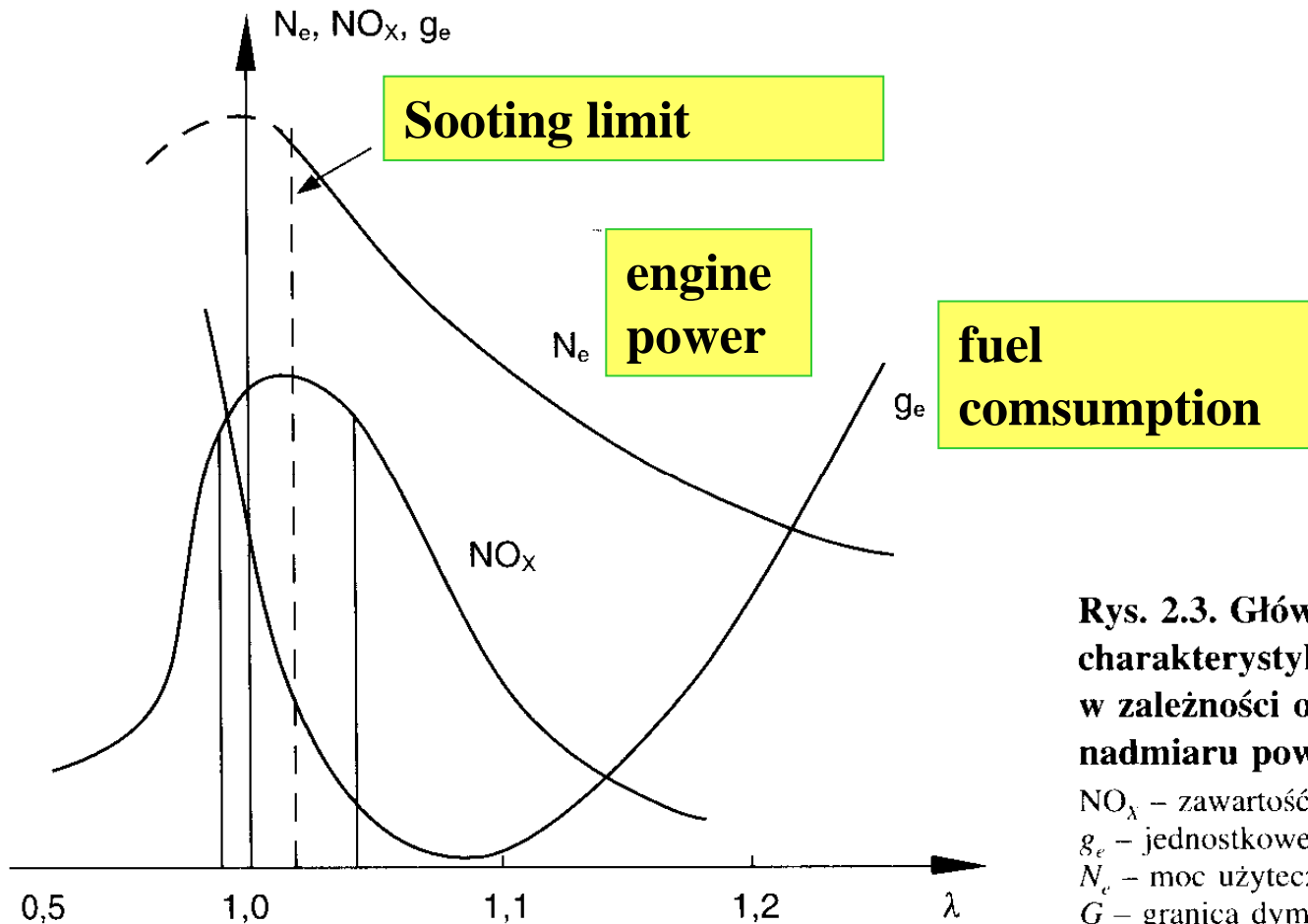
Content of combustion gases from Diesel engines

FLUE GAS			
Non-toxic components		Toxic components	
Water	H ₂ O	Carbon monoxide	CO
Carbon dioxide	CO ₂	Hydrocarbons	HC
Nitrogen	N ₂	Nitrogen oxides	NO _x
Hydrogen	H ₂	Aldehydes	CHO
Oxygen	O ₂	Solids	PM
Noble gases		Others	SO _x , Pb

Limits pollutant emissions from CI engines



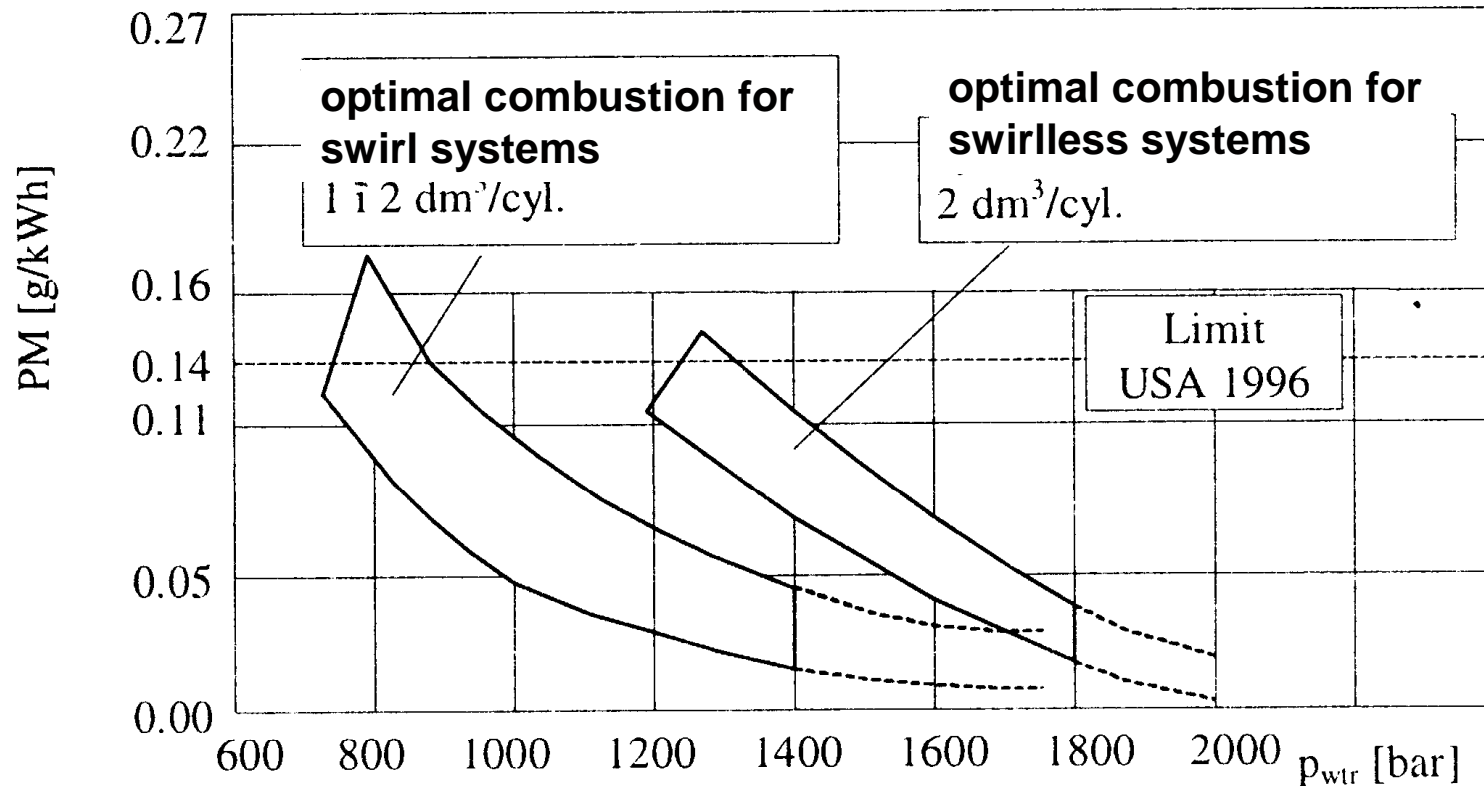
Effect of stoichiometry on pollution emissions from CI engines



Rys. 2.3. Główne charakterystyki silnika ZS w zależności od współczynnika nadmiaru powietrza λ

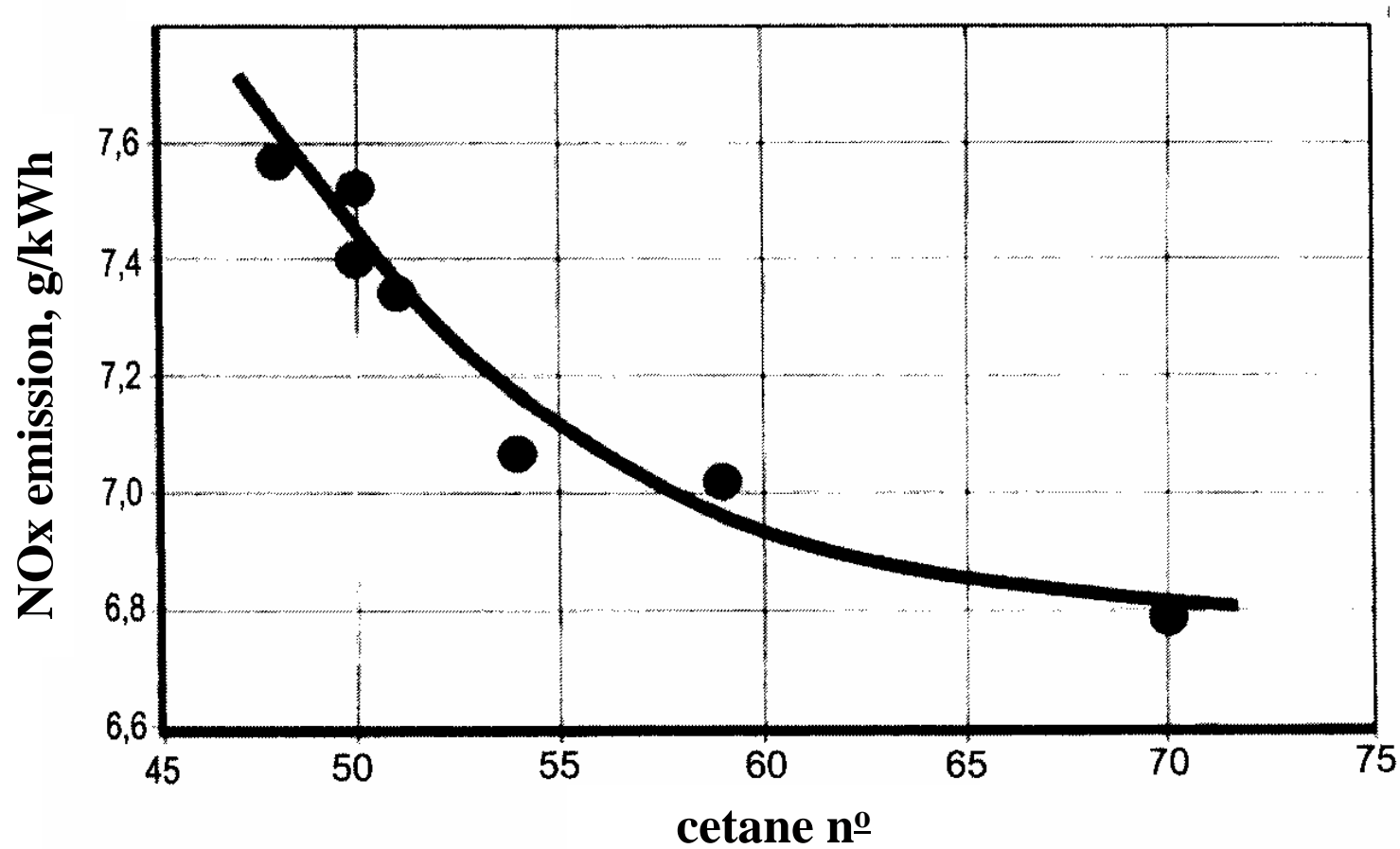
NO_x – zawartość w spalinach,
 g_e – jednostkowe zużycie paliwa,
 N_e – moc użyteczna silnika,
 G – granica dymienia

Effect of injection pressure on soot emission from CI engines



Rys. 39. Redukcja emisji cząstek stałych PM przez zwiększenie ciśnienia wtrysku dla silników HDD [50]

NO_x emission vs. Cetane No.



Soot removal from combustion gases

1. Soot particles are removed from combustion gases in reburning filters.
2. Soot filters are installed in outlet system of an engine.
3. Two types of soot filters are in use:
 - *SMF (Sintered Metal Filter)*, made of sintered metal with electric heaters inside,
 - *DPF (Diesel Particulate Filter)* which is composed of a preliminary catalyst and a main filter made of carbides.

SMF (Sintered Metal Filter)

Combustion gases flow through the filter (size of pores - $10\text{ }\mu\text{m}$), which stops soot particles. Temperature and pressure sensors start up the system of filter regeneration. The electric heaters burn soot collected in the filter with the efficiency better than 95%. Advantage of this type of soot filters is relatively low hydraulic drag for combustion gases.

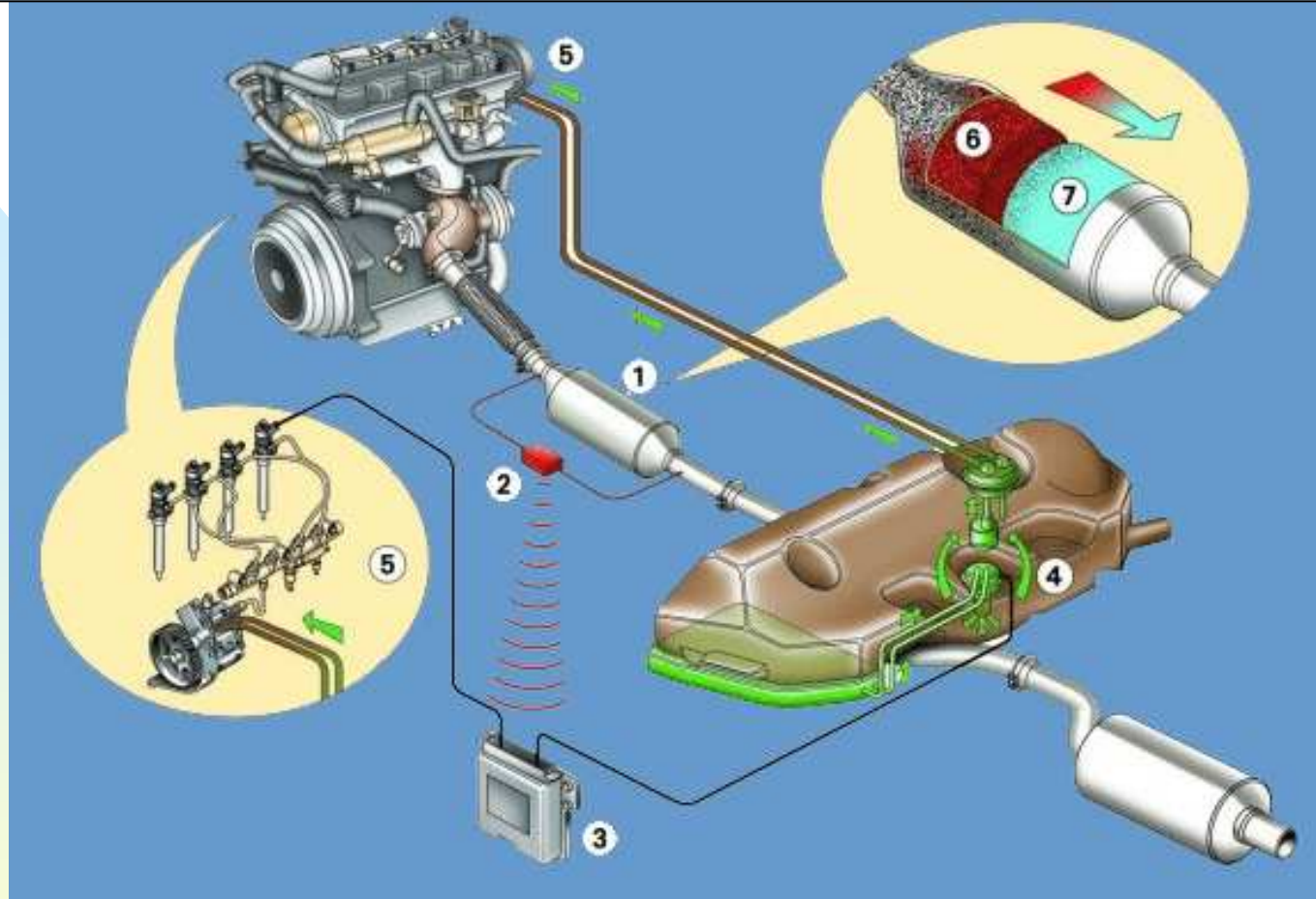


DPF (Diesel Particulate Filter)

The DPF filter is composed of a preliminary catalyst of combustion gases and main filter of particulate matter made of carbides. Soot particles collected in the filter are ignited by hot combustion gases. Because temperature of combustion gases in the outlet collector is in the range of $150\div 200\text{ }^{\circ}\text{C}$, and the ignition temperature of soot is approx. $550\text{ }^{\circ}\text{C}$, temperature of combustion gases is increased by burning of additional sample of oil during the decompression stage.

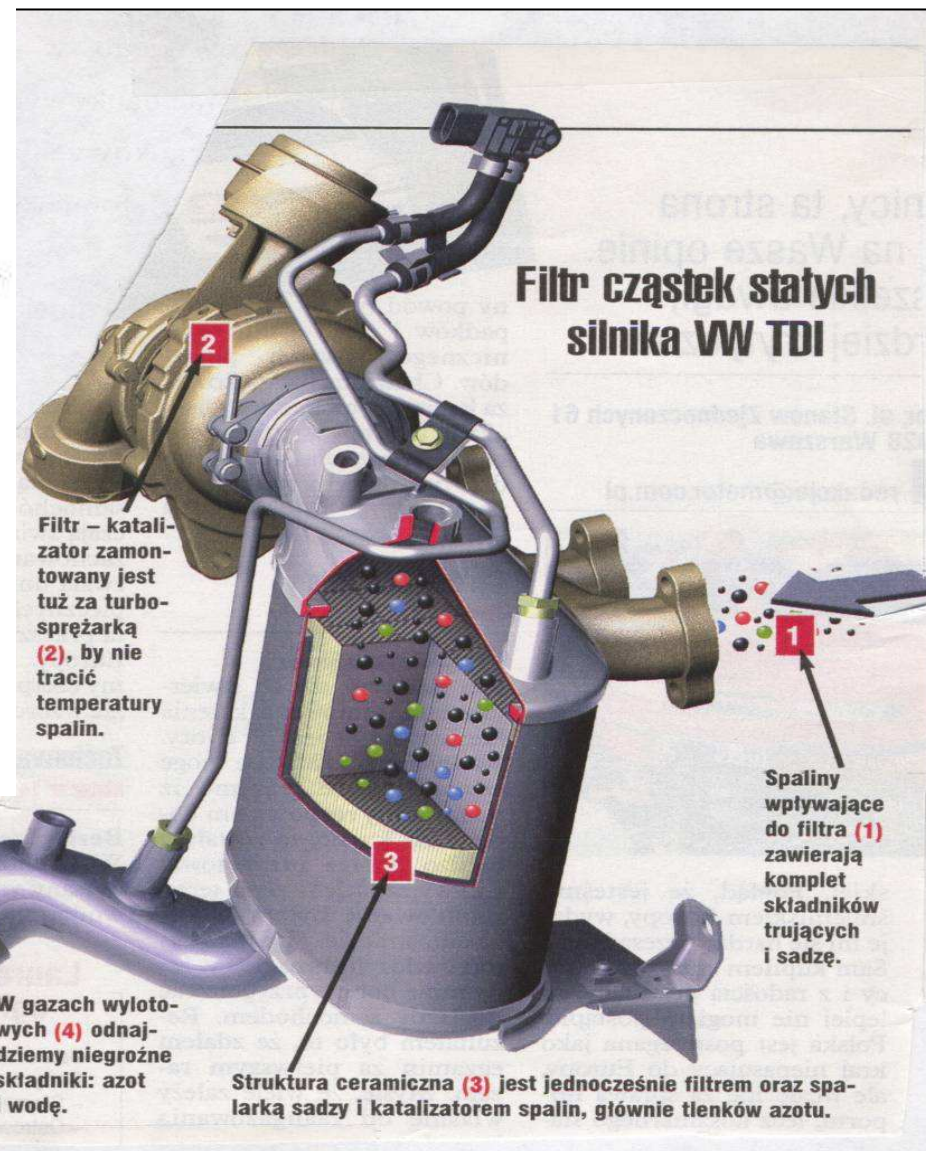
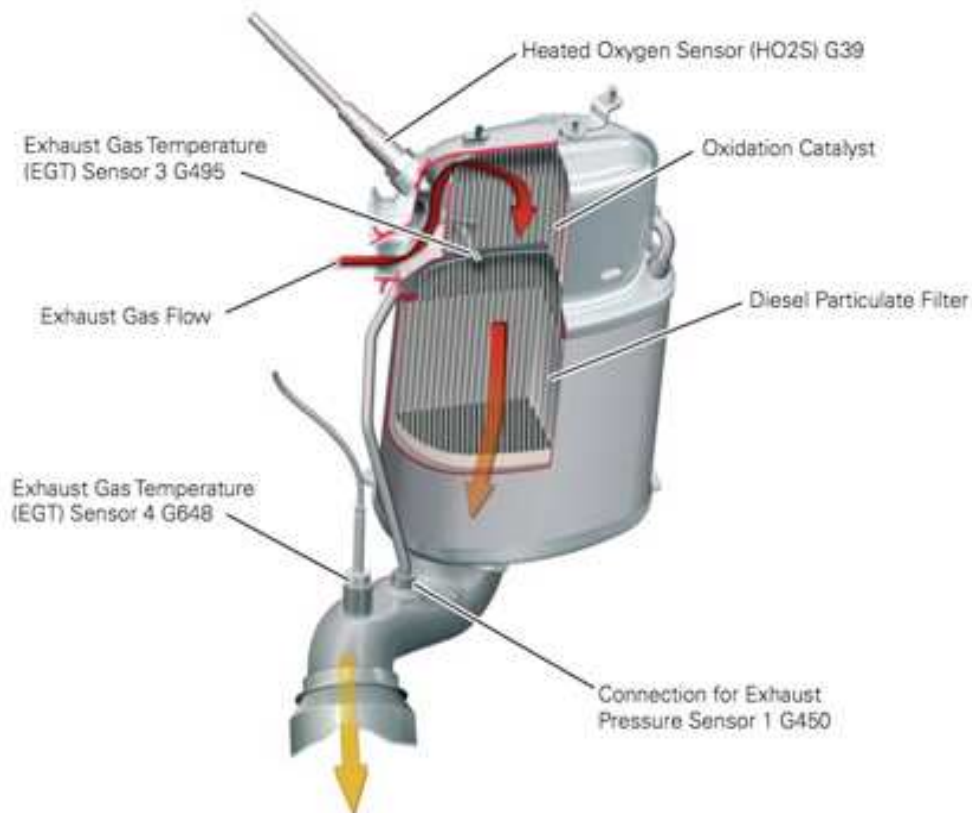


Scheme of soot particles removal



1. Particulate 'Filter and Pre-catalyst' filter assembly
2. Temperature and pressure sensors
3. Engine ECU
4. Injection of additives into the fuel in the main tank if necessary
5. Specific information sent to injector head when post-combustion needed
6. Pre-catalyser
7. Particulate filter

Catalysts in cleaning of combustion gases





DIESEL FUELS

Basic parameters of Diesel fuels

1. Colour and general look
2. Density at the temperature 15 °C
3. Cetane No.
4. Viscosity at the temperature 40 °C
5. Low-temperature parameters :
 - temperature of become hazy
 - temperature of cold filter blocking
 - temperature of flow
6. Sulphur content
7. Solid impurities content
8. Water content
9. Lubricating ability
10. Corrosivity

Basic parameters of Diesel fuels

11. Resistance to oxidation
12. Coking ability (Conradson no.)
13. Ashing
14. Content of aromatic hydrocarbons
15. Content of PAH
16. Temperature of ignition
17. Acidity no.
18. Washing properties
19. Content of FAME, ethanol, methanol and microorganisms
20. Content of metals (Zn, Cu, Mn, Ca, Na i in.)

Properties of Diesel oil

Parameter	Unit	Value	
		Minimum	Maximum
Cetane No.	-	51,0	-
Density at 15 °C	kg/m ³	820	845
Content of PAH	% mass.	-	11
Content of sulphur	mg/kg	-	50
Temperature of selfignition	°C	> 55	-
Content of ash	% mass.	-	0,01
Content of water	mg/kg	-	200
Viscosity at 40 °C	mm ² /s	2,00	4,50
Content of FAME	% vol.	-	5

Wg. PN EN 590

World Chart of Fuels (1998)

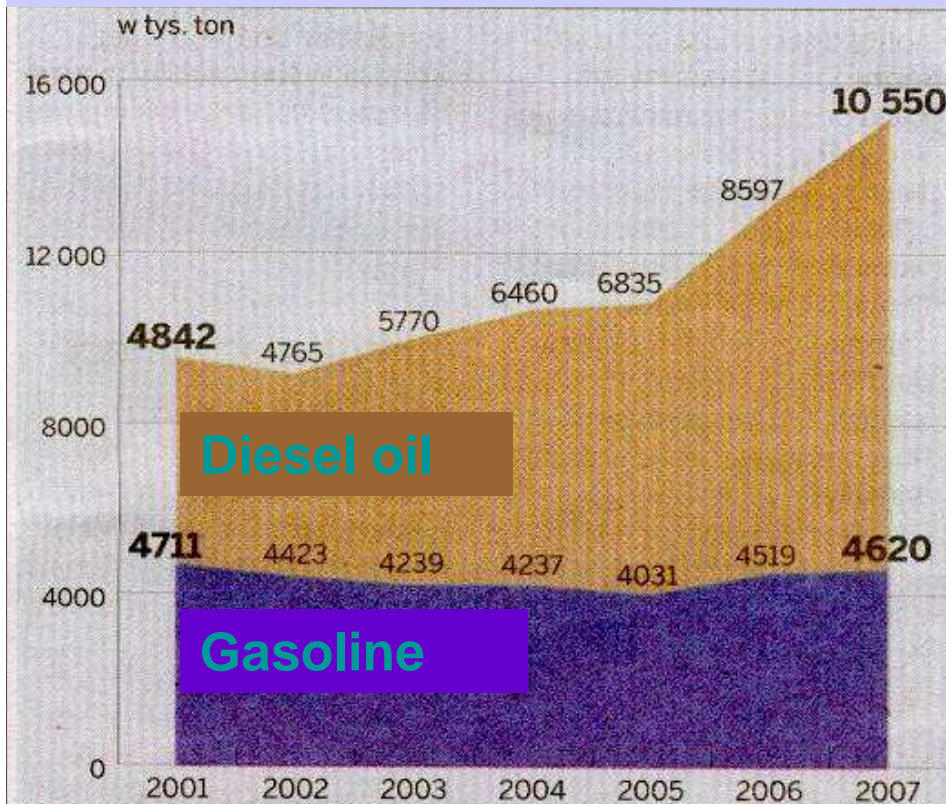
I. Categories of Diesel oils: 1 – 4

II. Changes with increase of category:

1. Fourth category: sulphur-less
2. Reduction of aromatic hydrocarbons content
3. No ethanol and methanol in Diesel oil
4. Content of FAME limited to 5%

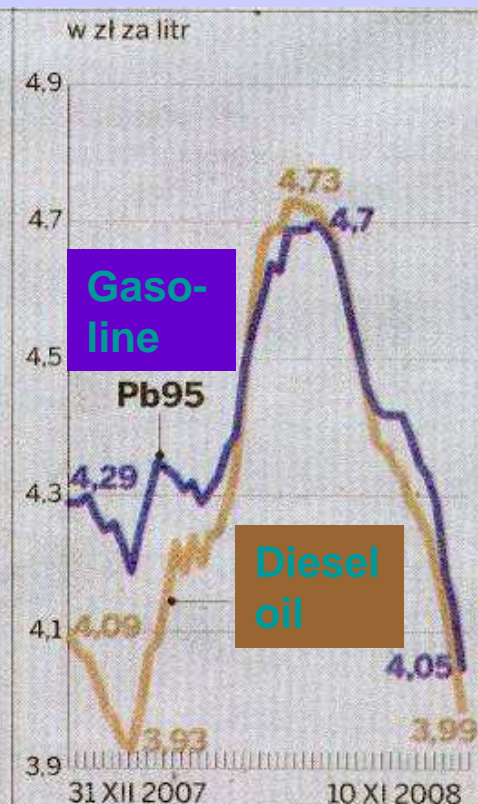
Diesel oil market in Poland

Consumption of fuels in Poland
(x 1000 t)

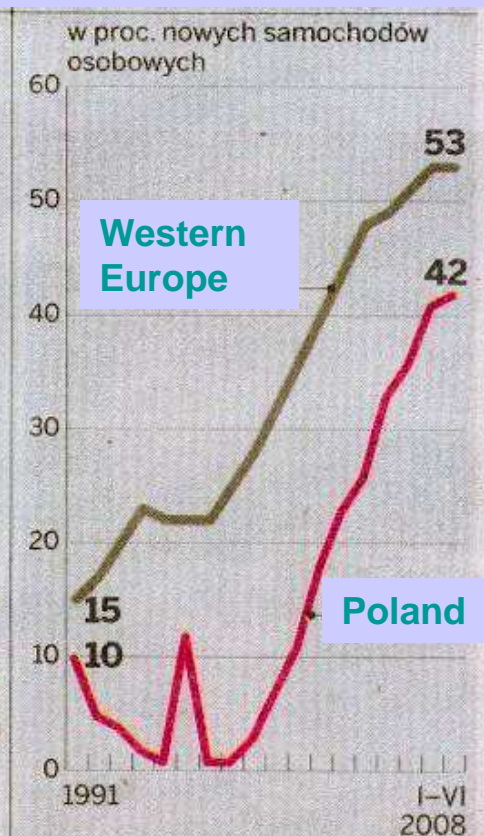


ŹRÓDŁO: E-PETROL, SAMAR, ACEA

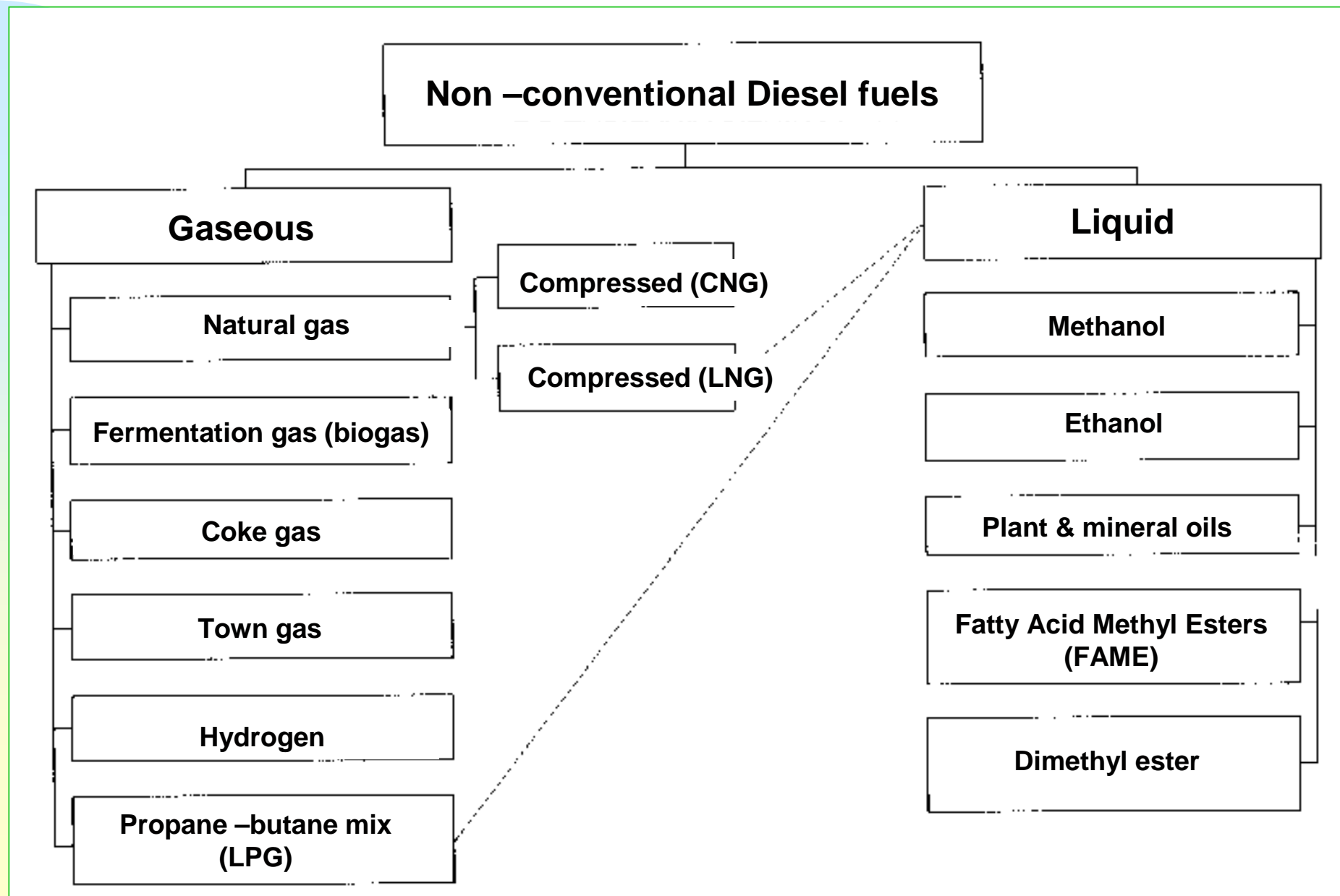
Prices of fuels in Poland
(zł / l)



Cars with Diesel engine
(% of new cars)



Non-conventional Diesel fuels



Alcohols and esters as CI fuels

Alcohols

- Methyl alcohol CH_3OH LC = 3-5
- Ethyl alcohol $\text{C}_2\text{H}_5\text{OH}$ LC = 5-8

Esters

- Dimethyl ester CH_3OCH_3 LC = 55-60
- Diethyl ester $\text{C}_2\text{H}_5\text{OC}_2\text{H}_5$

Gaseous CI engine fuels

Possible gaseous fuels:

a) LPG group:

- Propane
- Butane

b) NG group (stationary CI engines)

- LNG
- CNG

Because of low LC no. Gaseous fuels require for ignition:

a) Outer ignition source (e.g. electric spark)

b) Igniting pilot dose of Diesel oil