Modified Four Tank System Nonlinear State Estimation

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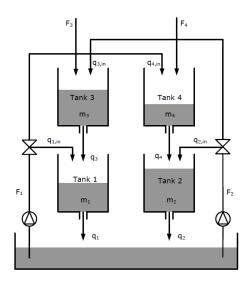
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Overview

- Illustration
- Nonlinear Model
- Simulation
- State Estimation

Illustration: Modified Four Tank System



Nonlinear Model - Simulation and Control Models

Simulation Model:

The simulation of the modified four tank system (MFTS) is governed by a stochastic differential equation (SDE) model

$$dx_S(t) = f_S(t, x_S(t), u(t), d_S(t), \theta_S)dt + \sigma_S(t, x_S(t), u(t), d_S(t), \theta_S)d\omega_S(t),$$

$$y(t_k) = g_S(t, x_S(t), u(t), d_S(t), \theta_S) + v_S(t_k),$$

where the process noise $\omega_S(t)$ denotes a standard Wiener process and $\mathbf{v}_S(t_k) \sim \mathcal{N}\left(0, R_S\right)$ denotes the normally distributed measurement noise.

Control Model:

The control model for the MFTS is similarly described by an SDE model

$$dx(t) = f(t, x(t), u(t), d(t), \theta)dt + \sigma(t, x(t), u(t), d(t), \theta)d\omega(t),$$

$$y_{C}(t_{k}) = g(t, x(t), u(t), d(t), \theta) + v(t_{k}),$$

where the process noise $\omega(t)$ denotes a standard Wiener process and $v(t_k) \sim \mathcal{N}(0,R)$ denotes the normally distributed measurement noise.



Simulation Model - Drift

The drift term is computed as

$$f_{S}(t,x_{S},u(t),d_{S}(t),\theta_{S}) = \begin{bmatrix} f_{1}(t,x_{S},u(t),d_{S}(t),\theta_{S}) \\ f_{2}(t,x_{S},u(t),d_{S}(t),\theta_{S}) \\ f_{3}(t,x_{S},u(t),d_{S}(t),\theta_{S}) \\ f_{4}(t,x_{S},u(t),d_{S}(t),\theta_{S}) \end{bmatrix} = \begin{bmatrix} \rho(q_{in,1}(t)+q_{3}(t)-q_{1}(t)) \\ \rho(q_{in,2}(t)+q_{4}(t)-q_{2}(t)) \\ \rho(q_{in,3}(t)-q_{3}(t)) \\ \rho(q_{in,4}(t)-q_{4}(t)) \end{bmatrix},$$

where

$$\begin{bmatrix} q_{in,1}(t) \\ q_{in,2}(t) \\ q_{in,3}(t) \\ q_{in,4}(t) \end{bmatrix} = \begin{bmatrix} \gamma_1 F_1(t) \\ \gamma_2 F_2(t) \\ (1-\gamma_2) F_2(t) + F_3(t) \\ (1-\gamma_1) F_1(t) + F_4(t) \end{bmatrix}, \qquad \begin{bmatrix} q_1(t) \\ q_2(t) \\ q_3(t) \\ q_4(t) \end{bmatrix} = \begin{bmatrix} a_1 \sqrt{2gH_1(t)} \\ a_2 \sqrt{2gH_2(t)} \\ a_3 \sqrt{2gH_3(t)} \\ a_4 \sqrt{2gH_4(t)} \end{bmatrix},$$

where $H_i(t) = m_i(t)/(\rho A_i)$ for $i \in \{1, 2, 3, 4\}$, a_i and A_i for $i \in \{1, 2, 3, 4\}$ are the cross-sectional areas of the drainage pipes and cyllindrical tanks respectively, and γ_i for $i \in \{1, 2\}$ are the split-values of the in-flow pipes.

Simulation Model - Diffusion

The process noise will be added to the disturbances of the system, such that the in-flows, $q_{in}(t)$, are described by

$$\begin{bmatrix} q_{in,1}(t) \\ q_{in,2}(t) \\ q_{in,3}(t) \\ q_{in,4}(t) \end{bmatrix} dt = \begin{bmatrix} \gamma_1 F_1(t) \\ \gamma_2 F_2(t) \\ (1-\gamma_2) F_2(t) \\ (1-\gamma_1) F_1(t) \end{bmatrix} dt + \begin{bmatrix} 0 \\ 0 \\ F_3(t) \\ F_4(t) \end{bmatrix} dt + \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ \rho \sigma_1 & 0 \\ 0 & \rho \sigma_2 \end{bmatrix} \begin{bmatrix} d\omega_1(t) \\ d\omega_2(t) \end{bmatrix},$$

which leads to the diffusion model

$$\sigma_{S}(t, x_{S}(t), u(t), d_{S}(t), \theta_{S}) = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ \rho \sigma_{1} & 0 \\ 0 & \rho \sigma_{2} \end{bmatrix}.$$



Simulation Model - Full Model

The full simulation model thus becomes

$$dx_{S}(t) = f_{S}(t, x_{S}, u(t), d_{S}(t), \theta_{S})dt + \sigma_{S}(t, x_{S}(t), u(t), d_{S}(t), \theta_{S})d\omega_{S}(t)$$

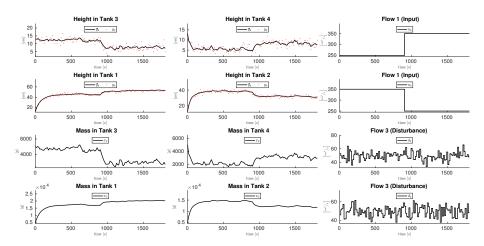
$$= \begin{bmatrix} \rho(q_{in,1}(t) + q_{3}(t) - q_{1}(t)) \\ \rho(q_{in,2}(t) + q_{4}(t) - q_{2}(t)) \\ \rho(q_{in,3}(t) - q_{3}(t)) \\ \rho(q_{in,4}(t) - q_{4}(t)) \end{bmatrix} dt + \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ \rho\sigma_{1} & 0 \\ 0 & \rho\sigma_{2} \end{bmatrix} d\omega_{S}(t)$$

where

$$\begin{bmatrix} q_{in,1}(t) \\ q_{in,2}(t) \\ q_{in,3}(t) \\ q_{in,4}(t) \end{bmatrix} = \begin{bmatrix} \gamma_1 F_1(t) \\ \gamma_2 F_2(t) \\ (1-\gamma_2) F_2(t) + F_3(t) \\ (1-\gamma_1) F_1(t) + F_4(t) \end{bmatrix}, \qquad \begin{bmatrix} q_1(t) \\ q_2(t) \\ q_3(t) \\ q_4(t) \end{bmatrix} = \begin{bmatrix} a_1 \sqrt{2gH_1(t)} \\ a_2 \sqrt{2gH_2(t)} \\ a_3 \sqrt{2gH_3(t)} \\ a_4 \sqrt{2gH_4(t)} \end{bmatrix},$$

where $H_i(t) = m_i(t)/(\rho A_i)$ for $i \in \{1, 2, 3, 4\}$, a_i and A_i for $i \in \{1, 2, 3, 4\}$ are the cross-sectional areas of the drainage pipes and cyllindrical tanks respectively, and γ_i for $i \in \{1, 2\}$ are the split-values of the in-flow pipes.

Closed-loop Simulation



Control Model - Drift

In the control model, we augment the state vector with the disturbances for the state estimation algorithm to reject them

$$f(t,x,u(t),d(t),\theta) = \begin{bmatrix} f_1(t,x,u(t),d(t),\theta) \\ f_2(t,x,u(t),d(t),\theta) \\ f_3(t,x,u(t),d(t),\theta) \\ f_4(t,x,u(t),d(t),\theta) \\ f_5(t,x,u(t),d(t),\theta) \\ f_6(t,x,u(t),d(t),\theta) \end{bmatrix} = \begin{bmatrix} \rho(q_{in,1}(t)+q_3(t)-q_1(t)) \\ \rho(q_{in,2}(t)+q_4(t)-q_2(t)) \\ \rho(q_{in,3}(t)-q_3(t)) \\ \rho(q_{in,4}(t)-q_4(t)) \\ \lambda_1(\bar{F}_3(t)-\bar{F}_3(t)) \\ \lambda_2(\bar{F}_4(t)-\bar{F}_4(t)) \end{bmatrix},$$

where

$$\begin{bmatrix} q_{in,1}(t) \\ q_{in,2}(t) \\ q_{in,3}(t) \\ q_{in,4}(t) \end{bmatrix} = \begin{bmatrix} \gamma_1 F_1(t) \\ \gamma_2 F_2(t) \\ (1-\gamma_2) F_2(t) + F_3(t) \\ (1-\gamma_1) F_1(t) + F_4(t) \end{bmatrix}, \qquad \begin{bmatrix} q_1(t) \\ q_2(t) \\ q_3(t) \\ q_4(t) \end{bmatrix} = \begin{bmatrix} a_1 \sqrt{2gH_1(t)} \\ a_2 \sqrt{2gH_2(t)} \\ a_3 \sqrt{2gH_3(t)} \\ a_4 \sqrt{2gH_4(t)} \end{bmatrix},$$

where $H_i(t) = \mathbf{m}_i(t)/(\rho A_i)$ for $i \in \{1, 2, 3, 4\}$, a_i and A_i for $i \in \{1, 2, 3, 4\}$ are the cross-sectional areas of the drainage pipes and cyllindrical tanks respectively, and γ_i for $i \in \{1, 2\}$ are the split-values of the in-flow pipes.

Control Model - Diffusion

In the control model, we include the disturbances as states for the state estimator to reject the disturbances. We model the disturbances as SDEs on the form

$$d\mathbf{F}_3(t) = \lambda_1 \left(\bar{F}_3(t) - \mathbf{F}_3(t) \right) dt + \sigma_1 d\omega_1(t),$$

$$d\mathbf{F}_4(t) = \lambda_2 \left(\bar{F}_4(t) - \mathbf{F}_4(t) \right) dt + \sigma_2 d\omega_2(t).$$

which leads to the diffusion model

$$\sigma(t, \mathbf{x}(t), u(t), d(t), \theta) = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ \sigma_1 & 0 \\ 0 & \sigma_2 \end{bmatrix}.$$

Control Model - Full Model

The full simulation model thus becomes

$$dx(t) = f(t, x, u(t), d(t), \theta)dt + \sigma(t, x(t), u(t), d(t), \theta)d\omega(t)$$

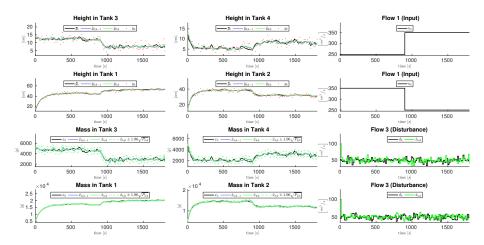
$$= \begin{bmatrix} \rho(q_{in,1}(t) + q_3(t) - q_1(t)) \\ \rho(q_{in,2}(t) + q_4(t) - q_2(t)) \\ \rho(q_{in,3}(t) - q_3(t)) \\ \rho(q_{in,4}(t) - q_4(t)) \\ \lambda_1(\bar{F}_3(t) - F_3(t)) \\ \lambda_2(\bar{F}_4(t) - F_4(t)) \end{bmatrix} dt + \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ \sigma_1 & 0 \\ 0 & \sigma_2 \end{bmatrix} d\omega(t)$$

where

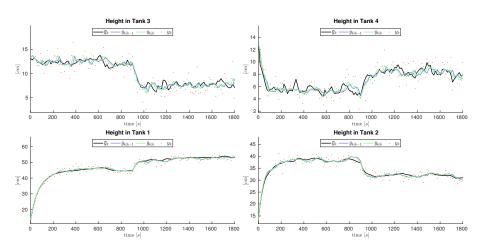
$$\begin{bmatrix} q_{in,1}(t) \\ q_{in,2}(t) \\ q_{in,3}(t) \\ q_{in,4}(t) \end{bmatrix} = \begin{bmatrix} \gamma_1 F_1(t) \\ \gamma_2 F_2(t) \\ (1-\gamma_2) F_2(t) + F_3(t) \\ (1-\gamma_1) F_1(t) + F_4(t) \end{bmatrix}, \qquad \begin{bmatrix} q_1(t) \\ q_2(t) \\ q_3(t) \\ q_4(t) \end{bmatrix} = \begin{bmatrix} a_1 \sqrt{2gH_1(t)} \\ a_2 \sqrt{2gH_2(t)} \\ a_3 \sqrt{2gH_3(t)} \\ a_4 \sqrt{2gH_4(t)} \end{bmatrix},$$

where $H_i(t) = m_i(t)/(\rho A_i)$ for $i \in \{1, 2, 3, 4\}$, a_i and A_i for $i \in \{1, 2, 3, 4\}$ are the cross-sectional areas of the drainage pipes and cyllindrical tanks respectively, and γ_i for $i \in \{1, 2\}$ are the split-values of the in-flow pipes.

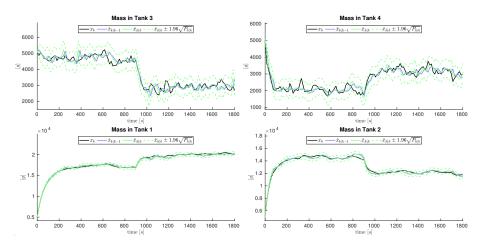
Closed-loop Simulation - State Estimation



State Estimation - Measurements



State Estimation - States



State Estimation - Inputs/Disturbances

