

# Intermediate Fluid Mechanics

## Lecture 7: Basic Flow Fields

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August 26, 2025

# Chapter Overview

- 1 Chapter Objectives
- 2 Flow I: Stagnation Point flow
- 3 Flow II: Solid Body Rotation
- 4 Flow III: Linear Shear Flow
- 5 Flow IV: Vortex Line induced flow

# Lecture Objectives

In this lecture we will consider four different basic flow fields, namely

- Stagnation Point Flow,
- Solid Body Rotation,
- Linear Shear Flow,
- Vortex Line Induced Flow,

and investigate the kinematic behavior of the fluid particles in those flows.

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- ① Chapter Objectives
- ② Flow I: Stagnation Point flow
- ③ Flow II: Solid Body Rotation
- ④ Flow III: Linear Shear Flow
- ⑤ Flow IV: Vortex Line induced flow

# Flow I: Stagnation Point flow

Let's consider the example flow illustrated below,

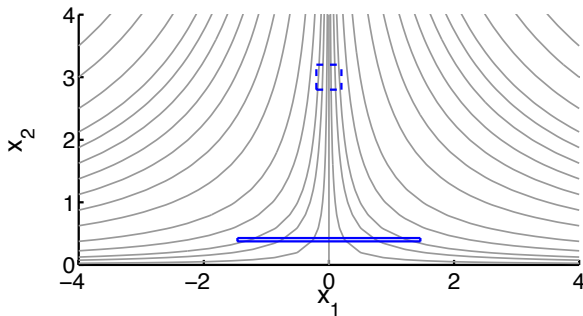


Figure: *Streamlines of a stagnation point flow. The square represents an initial fluid element, and the rectangle illustrates the same fluid element some time later.*

that is mathematically expressed as:

$$u_1 = c x_1, \quad u_2 = -c x_2, \quad u_3 = 0, \quad \text{with } c > 0. \quad (1)$$

# Flow I: Stagnation Point flow (continued ..)

To understand the kinematics of the fluid particles in this flow one needs to look at the strain-rate tensor  $e_{ij}$  and the spin tensor  $r_{ij}$ .

1) Let's start by first analyzing the **spin tensor**, recalling the definition of this one,

$$r_{ij} = -\frac{1}{2}\varepsilon_{ijk}\omega_k \quad \text{with} \quad \omega_k = \varepsilon_{ijk}\frac{\partial u_j}{\partial x_i}, \quad (2)$$

which in matrix form is written as,

$$r = \begin{pmatrix} 0 & -\frac{1}{2}\left(\frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2}\right) & \frac{1}{2}\left(\frac{\partial u_1}{\partial x_3} - \frac{\partial u_3}{\partial x_1}\right) \\ \frac{1}{2}\left(\frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2}\right) & 0 & -\frac{1}{2}\left(\frac{\partial u_3}{\partial x_2} - \frac{\partial u_2}{\partial x_3}\right) \\ -\frac{1}{2}\left(\frac{\partial u_1}{\partial x_3} - \frac{\partial u_3}{\partial x_1}\right) & \frac{1}{2}\left(\frac{\partial u_3}{\partial x_2} - \frac{\partial u_2}{\partial x_3}\right) & 0 \end{pmatrix}. \quad (3)$$

## Flow I: Stagnation Point flow (continued ..)

Since the tensor is anti-symmetric, one only needs to evaluate three of the terms,

$$r_{13} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_3} - \frac{\partial u_3}{\partial x_1} \right) = \frac{1}{2} \left[ \frac{\partial}{\partial x_3}(cx_1) - \frac{\partial}{\partial x_1}(0) \right] = 0 \quad (4)$$

$$r_{21} = \frac{1}{2} \left( \frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2} \right) = \frac{1}{2} \left[ \frac{\partial}{\partial x_1}(-cx_2) - \frac{\partial}{\partial x_2}(cx_1) \right] = 0 \quad (5)$$

$$r_{32} = \frac{1}{2} \left( \frac{\partial u_3}{\partial x_2} - \frac{\partial u_2}{\partial x_3} \right) = \frac{1}{2} \left[ \frac{\partial}{\partial x_2}(0) - \frac{\partial}{\partial x_3}(-cx_2) \right] = 0. \quad (6)$$

- In this type of flow the spin tensor  $r$  is equal to zero  $\implies$  **the flow is irrotational**.
- Physically, this means that the fluid particles are not rotating about their centroid.

## Flow I: Stagnation Point flow (continued ..)

2) Let's now analyze the strain-rate tensor, recalling the definition of this one in index notation,

$$e_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right), \quad (7)$$

and in matrix form,

$$e = \begin{pmatrix} \frac{\partial u_1}{\partial x_1} & \frac{1}{2} \left( \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) & \frac{1}{2} \left( \frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1} \right) \\ \frac{1}{2} \left( \frac{\partial u_2}{\partial x_1} + \frac{\partial u_1}{\partial x_2} \right) & \frac{\partial u_2}{\partial x_2} & \frac{1}{2} \left( \frac{\partial u_2}{\partial x_3} + \frac{\partial u_3}{\partial x_2} \right) \\ \frac{1}{2} \left( \frac{\partial u_3}{\partial x_1} + \frac{\partial u_1}{\partial x_3} \right) & \frac{1}{2} \left( \frac{\partial u_3}{\partial x_2} + \frac{\partial u_2}{\partial x_3} \right) & \frac{\partial u_3}{\partial x_3} \end{pmatrix}. \quad (8)$$



## Flow I: Stagnation Point flow (continued ..)

Since the tensor is symmetric, we need to evaluate six components, *i.e.* three diagonal and three off diagonal,

$$e_{11} = \frac{\partial u_1}{\partial x_1} = \frac{\partial}{\partial x_1}(cx_1) = c \quad (9)$$

$$e_{22} = \frac{\partial u_2}{\partial x_2} = \frac{\partial}{\partial x_2}(-cx_2) = -c \quad (10)$$

$$e_{33} = \frac{\partial u_3}{\partial x_3} = \frac{\partial}{\partial x_3}(0) = 0 \quad (11)$$

$$e_{12} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) = \frac{1}{2} \left( \frac{\partial}{\partial x_2}(cx_1) + \frac{\partial}{\partial x_1}(-cx_2) \right) = 0 \quad (12)$$

$$e_{13} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1} \right) = \frac{1}{2} \left( \frac{\partial}{\partial x_3}(cx_1) + \frac{\partial}{\partial x_1}(0) \right) = 0 \quad (13)$$

$$e_{23} = \frac{1}{2} \left( \frac{\partial u_2}{\partial x_3} + \frac{\partial u_3}{\partial x_2} \right) = \frac{1}{2} \left( \frac{\partial}{\partial x_3}(-cx_2) + \frac{\partial}{\partial x_2}(0) \right) = 0. \quad (14)$$

# Flow I: Stagnation Point flow (continued ..)

Therefore, in this case the strain-rate tensor is not null, but

$$e = \begin{pmatrix} c & 0 & 0 \\ 0 & -c & 0 \\ 0 & 0 & 0 \end{pmatrix}. \quad (15)$$

- Physically, this means that since  $e_{11} = c$ , line elements initially along the  $x_1$ -direction will elongate longitudinally in this direction.
- On the other hand, since  $e_{22} = -c$ , line elements initially along the  $x_2$ -direction will become compressed.
- Since all of the off-diagonal components are zero, the angle between material line elements never changes.
- Thus, two initially perpendicular line elements will remain perpendicular.

# Flow I: Stagnation Point flow (continued ..)

Finally, one can further check for the **dilation property** (*i.e.* compressibility).

- For this purpose one needs to check whether the volume of the fluid particles changes in time.
- Recall that the divergence of the velocity represents the dilation,

$$\frac{\partial u_i}{\partial x_i} = \frac{\partial u_1}{\partial x_1} + \frac{\partial u_2}{\partial x_2} + \frac{\partial u_3}{\partial x_3} = \frac{\partial}{\partial x_1}(c x_1) + \frac{\partial}{\partial x_2}(-c x_2) + \frac{\partial}{\partial x_3}(0) = 0. \quad (16)$$

$\Rightarrow$  the volume of the fluid particle cannot change.  $\Rightarrow$  This means that as material line elements elongate in the  $x_1$ -direction, line elements in the  $x_2$ -direction must proportionally compress.

+ Note: Realize that the flow is steady since the velocity  $\vec{u}$  is independent of time.

# Chapter Overview

- 1 Chapter Objectives
- 2 Flow I: Stagnation Point flow
- 3 Flow II: Solid Body Rotation
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## Flow II: Solid Body Rotation

Let's consider now the flow induced by a solid body rotation about an axis through the origin of the coordinate system as shown in the figure below,

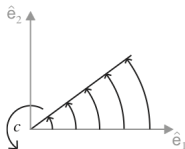


Figure: *Representation of the solid body rotation flow.*

This type of flow would occur, for example, in a large circular tube of water that was set spinning on a turn table with the axis of rotation through the center of the tube.

The velocity components for this flow are

$$u_1 = -c x_2, \quad (17)$$

$$u_2 = c x_1, \quad (18)$$

$$u_3 = 0, \quad \text{with } c > 0. \quad (19)$$

## Flow II: Solid Body Rotation (continued ...)

Proceeding similarly to what it was done for the stagnation point flow, one can easily compute the spin and strain-rate tensors.

1) For the **spin tensor** the off-diagonal terms are

$$r_{13} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_3} - \frac{\partial u_3}{\partial x_1} \right) = \frac{1}{2} \left[ \frac{\partial}{\partial x_3}(-cx_2) - \frac{\partial}{\partial x_1}(0) \right] = 0 \quad (20)$$

$$r_{21} = \frac{1}{2} \left( \frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2} \right) = \frac{1}{2} \left[ \frac{\partial}{\partial x_1}(cx_1) - \frac{\partial}{\partial x_2}(-cx_2) \right] = c \quad (21)$$

$$r_{32} = \frac{1}{2} \left( \frac{\partial u_3}{\partial x_2} - \frac{\partial u_2}{\partial x_3} \right) = \frac{1}{2} \left[ \frac{\partial}{\partial x_2}(0) - \frac{\partial}{\partial x_3}(cx_1) \right] = 0, \quad (22)$$

## Flow II: Solid Body Rotation (continued ...)

2) For the **strain-rate tensor** the six different terms are

$$e_{11} = \frac{\partial u_1}{\partial x_1} = \frac{\partial}{\partial x_1}(-cx_2) = 0 \quad (23)$$

$$e_{22} = \frac{\partial u_2}{\partial x_2} = \frac{\partial}{\partial x_2}(cx_1) = 0 \quad (24)$$

$$e_{33} = \frac{\partial u_3}{\partial x_3} = \frac{\partial}{\partial x_3}(0) = 0 \quad (25)$$

$$e_{12} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) = \frac{1}{2} \left( \frac{\partial}{\partial x_2}(-cx_2) + \frac{\partial}{\partial x_1}(cx_1) \right) = 0 \quad (26)$$

$$e_{13} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1} \right) = \frac{1}{2} \left( \frac{\partial}{\partial x_3}(-cx_2) + \frac{\partial}{\partial x_1}(0) \right) = 0 \quad (27)$$

$$e_{23} = \frac{1}{2} \left( \frac{\partial u_2}{\partial x_3} + \frac{\partial u_3}{\partial x_2} \right) = \frac{1}{2} \left( \frac{\partial}{\partial x_3}(cx_1) + \frac{\partial}{\partial x_2}(0) \right) = 0. \quad (28)$$

## Flow II: Solid Body Rotation (continued ...)

Therefore, in this type of flow, the matrix form the spin and strain-rate tensors are

$$r = \begin{pmatrix} 0 & -c & 0 \\ c & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix} \quad \text{and} \quad e = \begin{pmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix}. \quad (29)$$

- Physically, this means that there is neither longitudinal stretching of line elements, nor shear deformation in rigid body rotation.
- $\Rightarrow$  material elements maintain both their shape and size (i.e. a sphere elements of radius  $a$  remains a sphere of radius  $a$  and does not deform into an ellipsoid).
- $\Rightarrow$  the material element will rotate about the  $x_3$ -axis at an angular rotation rate of  $c$ .

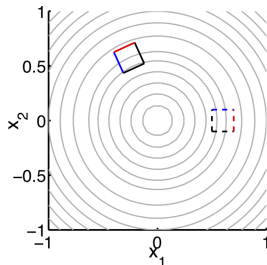


Figure: *Streamlines of a solid body rotation flow. The dotted square represents an initial fluid element that moves to a new location (solid lines). The gray circular lines illustrate the streamlines for this flow.*



# Chapter Overview

- 1 Chapter Objectives
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- 3 Flow II: Solid Body Rotation
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- 5 Flow IV: Vortex Line induced flow

# Flow III: Linear Shear Flow

- The two previous flows exemplified either strain-rate or rotation but not both.
- Now, we consider linear shear flow that incorporates both characteristics.

In this case, the velocity components can be of the form,

$$u_1 = c x_2, \quad (30)$$

$$u_2 = 0, \quad (31)$$

$$u_3 = 0, \quad \text{with } c > 0. \quad (32)$$

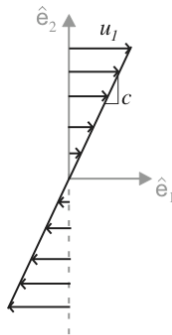


Figure: *Representation of a Linear Shear flow.*

## Flow III: Linear Shear Flow (continued ...)

Let's now compute once more the spin and strain-rate tensors.

1) For the **spin tensor** the off-diagonal terms are:

$$r_{13} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_3} - \frac{\partial u_3}{\partial x_1} \right) = \frac{1}{2} \left[ \frac{\partial}{\partial x_3}(cx_2) - \frac{\partial}{\partial x_1}(0) \right] = 0 \quad (33)$$

$$r_{21} = \frac{1}{2} \left( \frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2} \right) = \frac{1}{2} \left[ \frac{\partial}{\partial x_1}(0) - \frac{\partial}{\partial x_2}(cx_2) \right] = -\frac{c}{2} \quad (34)$$

$$r_{32} = \frac{1}{2} \left( \frac{\partial u_3}{\partial x_2} - \frac{\partial u_2}{\partial x_3} \right) = \frac{1}{2} \left[ \frac{\partial}{\partial x_2}(0) - \frac{\partial}{\partial x_3}(cx_1) \right] = 0, \quad (35)$$

## Flow III: Linear Shear Flow (continued ...)

2) For the **strain-rate tensor** the six different terms are

$$e_{11} = \frac{\partial u_1}{\partial x_1} = \frac{\partial}{\partial x_1}(cx_2) = 0 \quad (36)$$

$$e_{22} = \frac{\partial u_2}{\partial x_2} = \frac{\partial}{\partial x_2}(0) = 0 \quad (37)$$

$$e_{33} = \frac{\partial u_3}{\partial x_3} = \frac{\partial}{\partial x_3}(0) = 0 \quad (38)$$

$$e_{12} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) = \frac{1}{2} \left( \frac{\partial}{\partial x_2}(cx_2) + \frac{\partial}{\partial x_1}(0) \right) = \frac{c}{2} \quad (39)$$

$$e_{13} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1} \right) = \frac{1}{2} \left( \frac{\partial}{\partial x_3}(cx_2) + \frac{\partial}{\partial x_1}(0) \right) = 0 \quad (40)$$

$$e_{23} = \frac{1}{2} \left( \frac{\partial u_2}{\partial x_3} + \frac{\partial u_3}{\partial x_2} \right) = \frac{1}{2} \left( \frac{\partial}{\partial x_3}(0) + \frac{\partial}{\partial x_2}(0) \right) = 0. \quad (41)$$

## Flow III: Linear Shear Flow (continued ...)

Therefore, in this type of flow, the spin and strain-rate tensors are:

$$r = \begin{pmatrix} 0 & c/2 & 0 \\ -c/2 & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix} \quad \text{and} \quad e = \begin{pmatrix} 0 & c/2 & 0 \\ c/2 & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix}. \quad (42)$$

- In the linear shear flow, there are equal parts of shear-strain rate and rotation.
- Also, the diagonal components of  $e_{ij}$  are zero, so that one should not expect any longitudinal deformation of material.

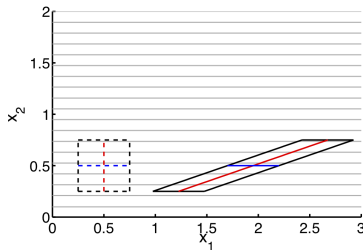


Figure: *Representation of the linear shear flow.*

One can decompose this motion into two parts, so that one is able to quantify it using the mathematical framework involving  $e_{ij}$  and  $r_{ij}$ .

## Flow III: Linear Shear Flow (continued ...)

The motion of the fluid particle in shear flow can be obtained through a shear deformation and an instantaneous rotation about the centroid of the particle.

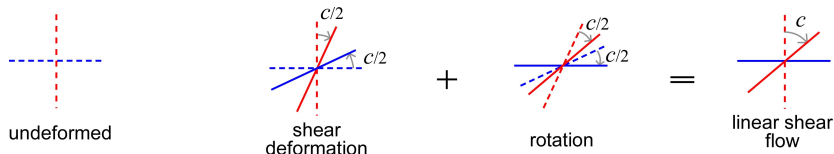


Figure: Pictorial representation of the strain-rate and rotation decomposition associated with a linear shear flow.

### Note

- While there is vorticity in this process, the fluid element does not continuously spin about its centroid.
- In rigid body motion the fluid element rotates all the way through  $360^\circ$ .
- In linear shear flow, one should think about the vorticity as the instantaneous angular velocity necessary to achieve the observed fluid motion.

# Chapter Overview

- ① Chapter Objectives
- ② Flow I: Stagnation Point flow
- ③ Flow II: Solid Body Rotation
- ④ Flow III: Linear Shear Flow
- ⑤ Flow IV: Vortex Line induced flow

## Flow IV: Vortex Line induced flow

Let's now consider the flow in the neighborhood of a single vortex line (similar to a streamline, a vortex line is a line that is everywhere tangent to the vorticity).

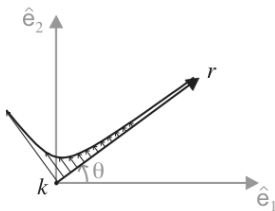


Figure: *The velocity field induced by the vortex line is independent of the polar angle  $\theta = \tan^{-1}(x_2/x_1)$*

The velocity components, assuming the vortex line passes through the origin of the coordinate system and is oriented along the  $\hat{e}_3$ -axis, are

$$u_1 = \frac{-k x_2}{x_1^2 + x_2^2}, \quad u_2 = \frac{k x_1}{x_1^2 + x_2^2}, \quad u_3 = 0. \quad (43)$$



## Flow IV: Vortex Line induced flow (continued ...)

In this case:

- $k$  denotes the strength of the vortex line, which is given by  $k = \omega \Delta S$ ,
- $\omega$  is the magnitude of the vorticity along the line
- $\Delta S$  is the infinitesimal cross sectional area of the vortex line.
- Since  $k$  remains constant along the vortex line (by definition), it can be seen that as  $\Delta S \rightarrow 0$  then  $\omega \rightarrow \infty$ .
- In this case, the origin is a singularity, so that the above velocity components are only valid for  $x_1^2 + x_2^2 \neq 0$ .

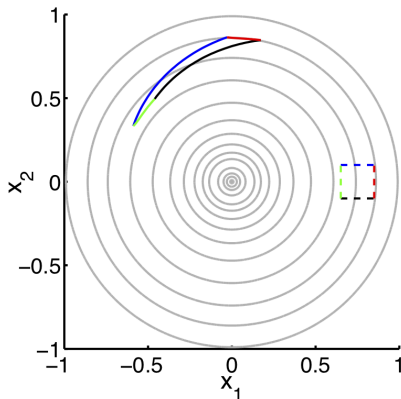


Figure: *Streamlines of steady vortex line flow and corresponding deformation of a fluid particle.*

## Flow IV: Vortex Line induced flow (continued ...)

If we now proceed similarly to what it was done for the previous flows, one can easily compute the spin and strain-rate tensors.

1) For the **spin tensor** the off-diagonal terms are

$$r_{13} = \frac{\omega_2}{2} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_3} - \frac{\partial u_3}{\partial x_1} \right) = 0 \quad (44)$$

$$r_{21} = \frac{\omega_3}{2} = \frac{1}{2} \left( \frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2} \right) = 0 \quad (45)$$

$$r_{32} = \frac{\omega_1}{2} = \frac{1}{2} \left( \frac{\partial u_3}{\partial x_2} - \frac{\partial u_2}{\partial x_3} \right) = 0, \quad (46)$$

## Flow IV: Vortex Line induced flow (continued ...)

and for the strain-rate tensor the six different terms are

$$e_{11} = \frac{\partial u_1}{\partial x_1} = \frac{\partial}{\partial x_1} \left( -kx_2(x_1^2 + x_2^2)^{-1} \right) = \frac{2kx_1x_2}{(x_1^2 + x_2^2)^2} \quad (47)$$

$$e_{22} = \frac{\partial u_2}{\partial x_2} = \frac{-2kx_1x_2}{(x_1^2 + x_2^2)^2} \quad (48)$$

$$e_{33} = \frac{\partial u_3}{\partial x_3} = 0 \quad (49)$$

$$e_{12} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) = \frac{k(x_2^2 - x_1^2)}{(x_1^2 + x_2^2)^2} \quad (50)$$

$$e_{13} = \frac{1}{2} \left( \frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1} \right) = 0 \quad (51)$$

$$e_{23} = \frac{1}{2} \left( \frac{\partial u_2}{\partial x_3} + \frac{\partial u_3}{\partial x_2} \right) = 0. \quad (52)$$

## Flow IV: Vortex Line induced flow (continued ...)

Therefore, in this type of flow, the matrix form the spin and strain-rate tensors are

$$r = \begin{pmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix} \quad \text{and} \quad e = \begin{pmatrix} \frac{2kx_1x_2}{(x_1^2+x_2^2)^2} & \frac{k(x_2^2-x_1^2)}{(x_1^2+x_2^2)^2} & 0 \\ \frac{k(x_2^2-x_1^2)}{(x_1^2+x_2^2)^2} & \frac{-2kx_1x_2}{(x_1^2+x_2^2)^2} & 0 \\ 0 & 0 & 0 \end{pmatrix}. \quad (53)$$

- Except for the origin, the flow is irrotational;
- However, the fluid particles follow circular paths.
- It can be seen from  $e_{11}$  and  $e_{22}$  that there is equal (but opposite) longitudinal stretch of material lines originally oriented along the  $\hat{e}_1$  &  $\hat{e}_2$  axes.
- One should also expect shear deformation since  $e_{12}$  (and  $e_{21}$ ) are non-zero, *i.e.* we expect the angle between line elements initially along the  $\hat{e}_1$  and  $\hat{e}_2$  axes to decrease.

## Flow IV: Vortex Line induced flow (continued ...)

Regarding the dilation of the flow,

$$\frac{\partial u_i}{\partial x_i} = \frac{\partial u_1}{\partial x_1} + \frac{\partial u_2}{\partial x_2} + \frac{\partial u_3}{\partial x_3} = 0, \quad (54)$$

and hence there is no change in the volume of the flow.

## Flow IV: Vortex Line induced flow (continued ...)

**Short-time Deformation:** To interpret  $e_{ij}$  and  $r_{ij}$  by looking at the motion of a material line, the time between the initial and deformed states must be small.

- From the analysis of  $e_{ij}$  one expects the top and bottom lines to elongate while the right and left lines shorten.
- On the side Figure it can be observed that initially the right line has a length  $l_r = 0.1$ ; at a short time later,  $l_r = 0.0943$ . For the bottom line,  $l_b = 0.1$ ; however a short time later this one has a length of  $l_b = 0.1122$ .
- It can also be observed that the angle between the bottom and right line has decreased from  $90^\circ$  to about  $66^\circ$  in the short time shown.

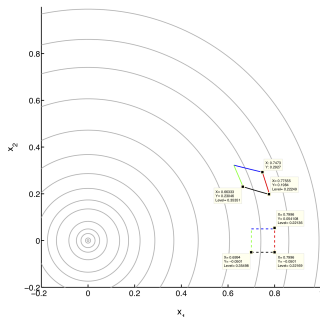


Figure: *Short-time deformation of a fluid element in the vortex line flow.*