

## Response Versus Pole Location

From the Laplace transform of ODE, we get a transfer function:

$$H(s) = \frac{n(s)}{d(s)}$$
      g zeros      output  
              : poles      input

to find then:

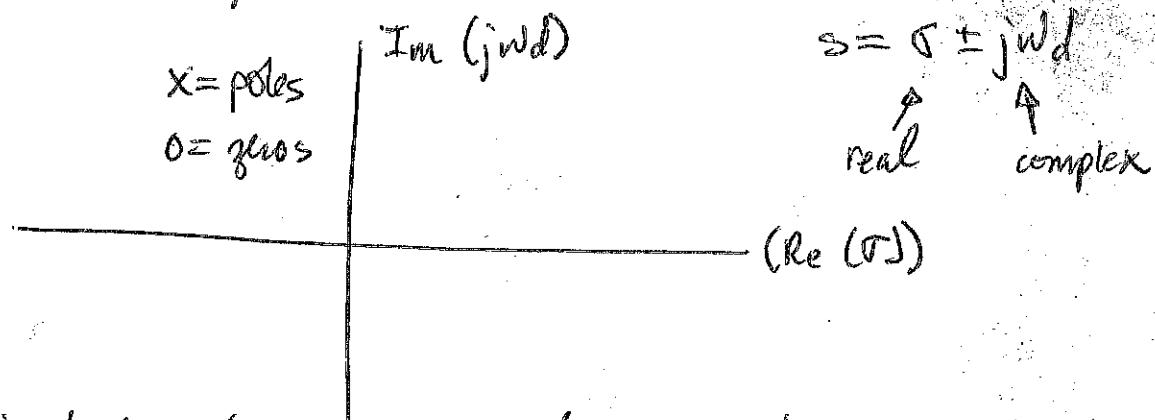
$$n(s) = 0$$

$$d(s) = 0$$

These poles and zeros describe the system.

Recall that impulse response  $\Rightarrow$  natural response thus we can identify the time histories with pole location in the s-plane.

s-plane: A graphical tool for plotting the zeros and poles thus giving us a pictorial view of the systems response:



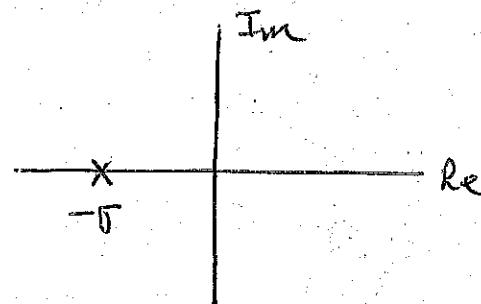
Now let's look at some examples of system response versus pole location

Ex: (Real Poles)

$$H(s) = \frac{1}{s+\tau} = \text{Laplace T.F. of impulse (1st order) Response}$$

from Table A-2 (#7)

$$h(t) = e^{-\tau t}$$



when  $\tau > 0$ ,  $s < 0 \Rightarrow$  stable system

when  $\tau < 0$ ,  $s > 0 \Rightarrow$  unstable system

$\tau = 0$ ,  $s > 0 \Rightarrow$  marginally stable system

(small perturbation can make it unstable).

Since this is a first order response:

$$\tau = \frac{1}{K} \text{ (time constant)}$$

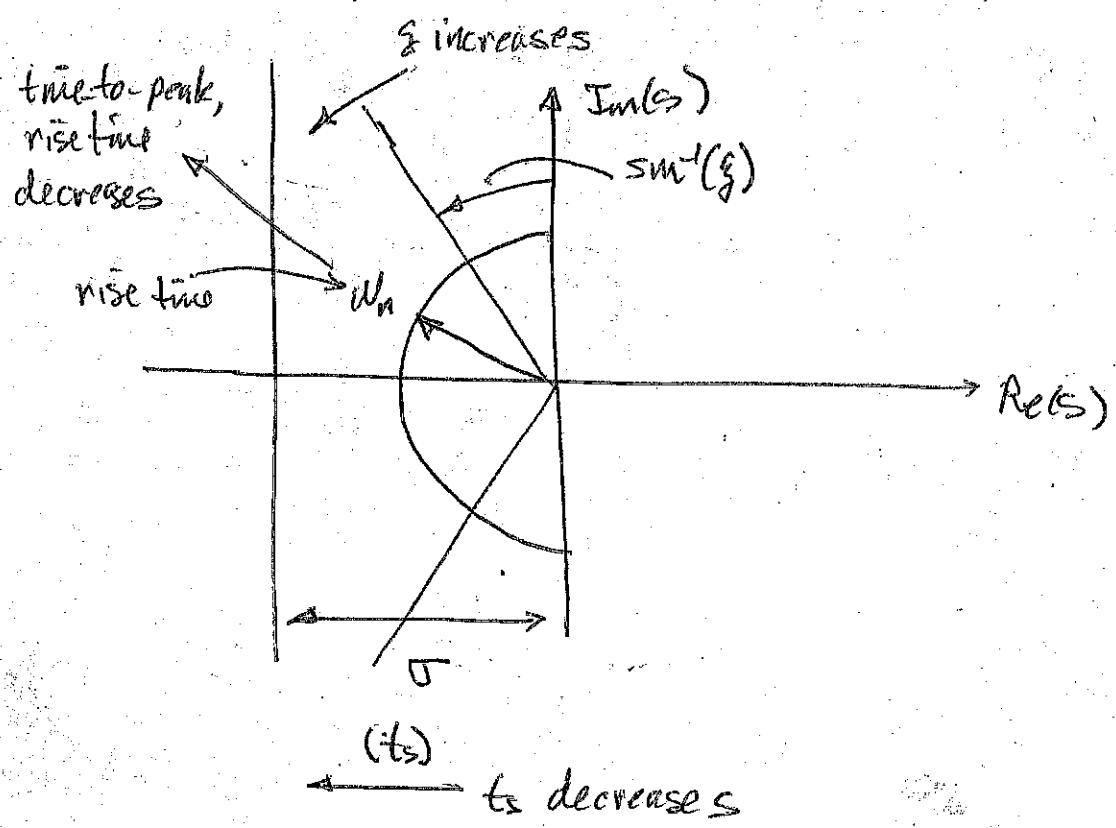
-Time it takes to reach 63% of final value.

$\tau$  characterizes the exponential behavior of system.

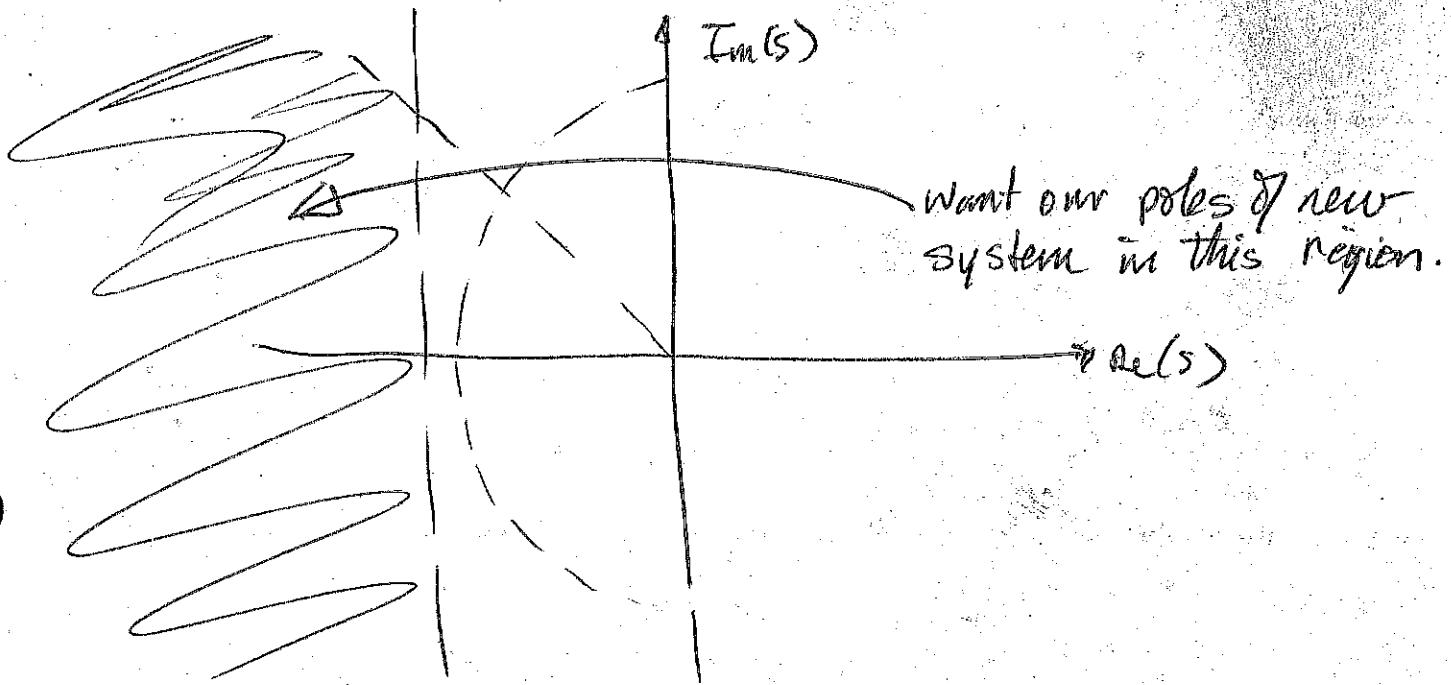
Larger  $\tau$  means the decay is faster  $\Rightarrow$  faster pole

smaller  $\tau$  mean slower decay and is usually the dominate behavior of a system.

How do they map into the s-plane?



Therefore, we can use these mapping to design a controller, ex:



## Effects of zeros and additional Poles on 2<sup>nd</sup> order systems

In general, for a 2<sup>nd</sup> order system in the form:

$$H(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

the transient response parameters are approximated by

$$\text{rise time: } t_r \approx \frac{1.8}{\omega_n}$$

$$\text{settling time: } t_s \approx \frac{4}{\zeta\omega_n} = \frac{4}{\zeta\omega_n} \quad (2\%)$$

$$\text{time-to-peak: } t_p = \frac{\pi}{\omega_d} = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}}$$

$$\%OS = 100 e^{-\frac{\pi}{1-\zeta^2}}$$

But these approximations only apply to 2<sup>nd</sup> order systems w/ no finite zeros.

But what happens if we add more poles and zeros to the system?

\* For some cases, ~~the~~ the systems with more poles/zeros can be approximated as a 2<sup>nd</sup> order system if it has two dominant complex poles.

### Adding Poles:

Take for example adding a pole (real) to a 2<sup>nd</sup> order system:

$$\frac{Y(s)}{U(s)} = \frac{B(s + \xi w_n) + C w_d}{(s + \xi w_n)^2 + w_d^2} + \frac{D}{s + \alpha}$$

Now consider a step input  $U(s) = \frac{1}{s}$

$$\Rightarrow Y(s) = \frac{A}{s} + \frac{B(s + \xi w_n) + C w_d}{(s + \xi w_n)^2 + w_d^2} + \frac{D}{s + \alpha}$$

$$\Rightarrow Y(t) = A u(t) + e^{-\xi w_n t} (B \cos w_d t + C \sin w_d t) + D e^{-\alpha t}$$

Then if  $\alpha$  is far away, the contribution of  $D e^{-\alpha t}$  term is small because rate of decay for large  $\alpha$  is fast.

But if  $\alpha$  is close to poles of the second order system i.e.

$$s = -\xi w_n \pm j w_n \sqrt{1 - \xi^2}$$

In general, if poles are at least 5-10 times away from 2<sup>nd</sup> order poles, then we can approximate the response as 2<sup>nd</sup> order.

## Adding zeros

Consider adding an zero to a 2nd order system:

$$H(s) = \frac{Y(s)}{U(s)} = \frac{as+b}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$Y(s) = \frac{as}{s^2 + 2\zeta\omega_n s + \omega_n^2} + \frac{b}{s^2 + 2\zeta\omega_n s + \omega_n^2} + \frac{c}{s}$$

$$Y(s) = saH_1(s) + bH_2(s) + H_3(s)$$

↑  
step input response

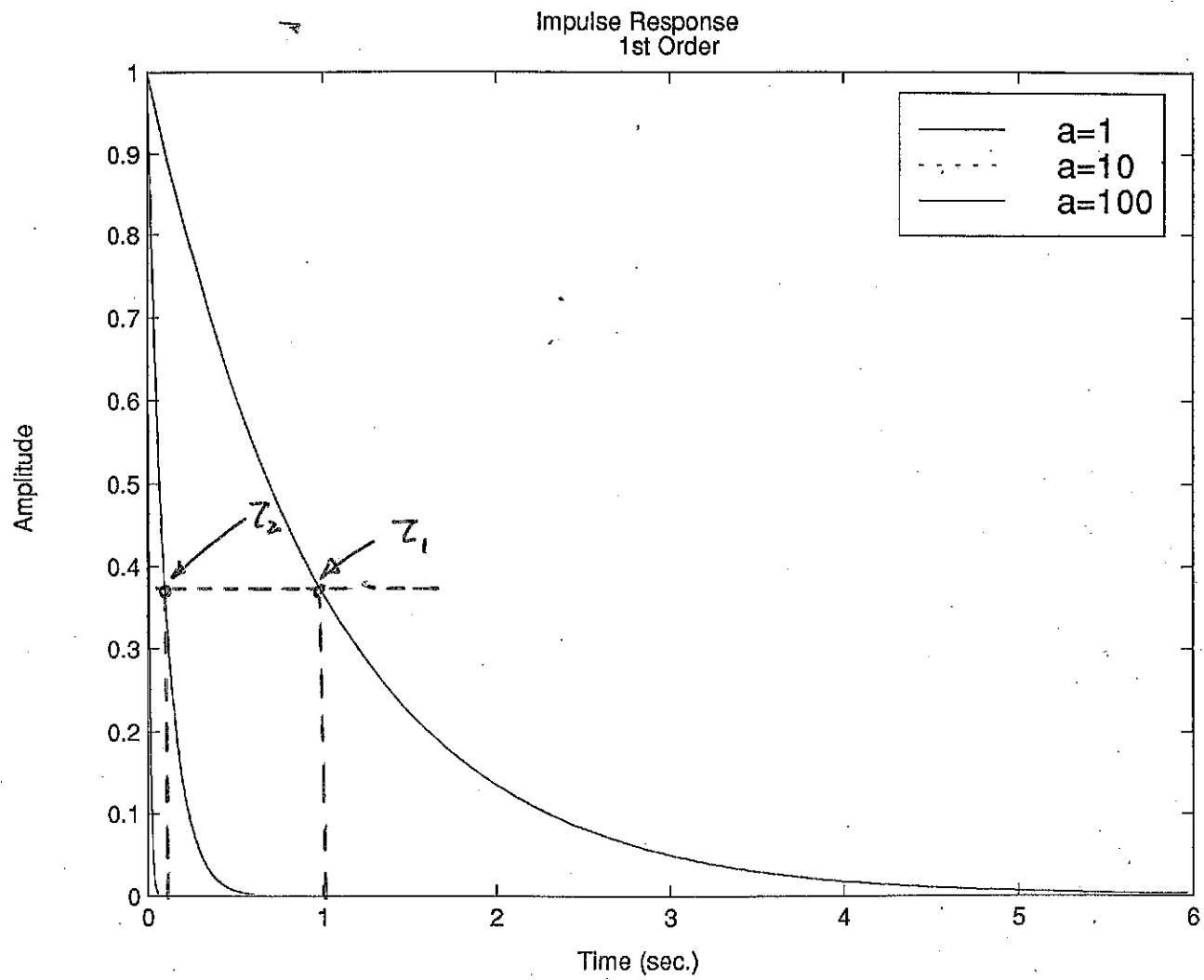
if  $b > 0 \Rightarrow$  LHP zero, therefore the response  
of  $H_1(s) > 0 \Rightarrow saH_1(s) > 0$

so, depending upon the magnitude of  $b$ ,  
the response of a zero will add to the  
system.

if  $b < 0 \Rightarrow$  RHP zero, therefore  $H_1(s) < 0$   
and the system will respond in negative  
direction. Non minimum phase systems.

$$H(s) = \frac{1}{s+a}$$

a=1, 10, 100



### Matlab Impulse Function

```
>>n=[1];
>>d=[1 a];
>>impulse(n,d);
```

## Complex Roots

- Come in complex conjugate pairs:

$$s = \sigma \pm j\omega_d$$

$$\Rightarrow d(s) = (s + \sigma + j\omega_d)(s + \sigma - j\omega_d)$$

$$d(s) = (s + \sigma)^2 + \omega_d^2$$

For a second order system, general form is

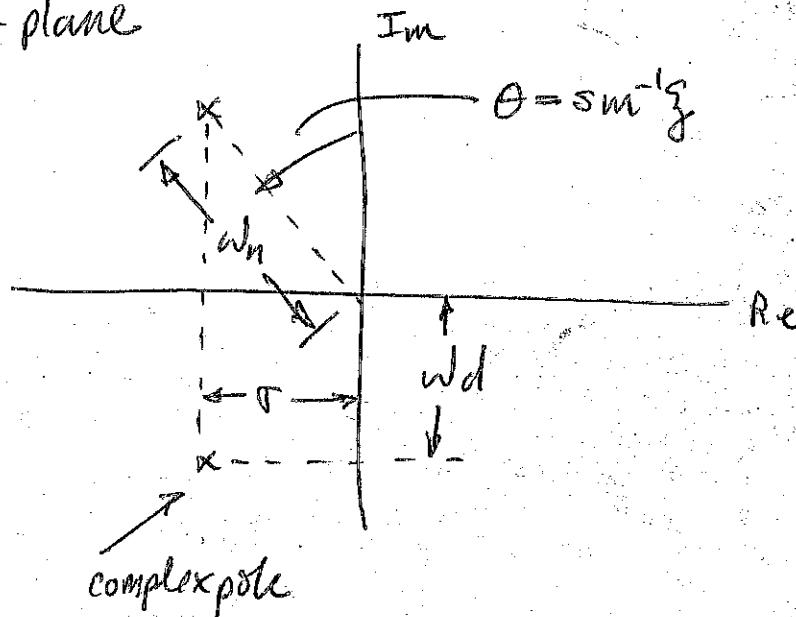
$$H(s) = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

Comparing  $d(s)$  w/  $H(s)$ 's denominator, we see that

$$\sigma = -\xi\omega_n \quad \omega_d = \omega_n \sqrt{1 - \xi^2}$$

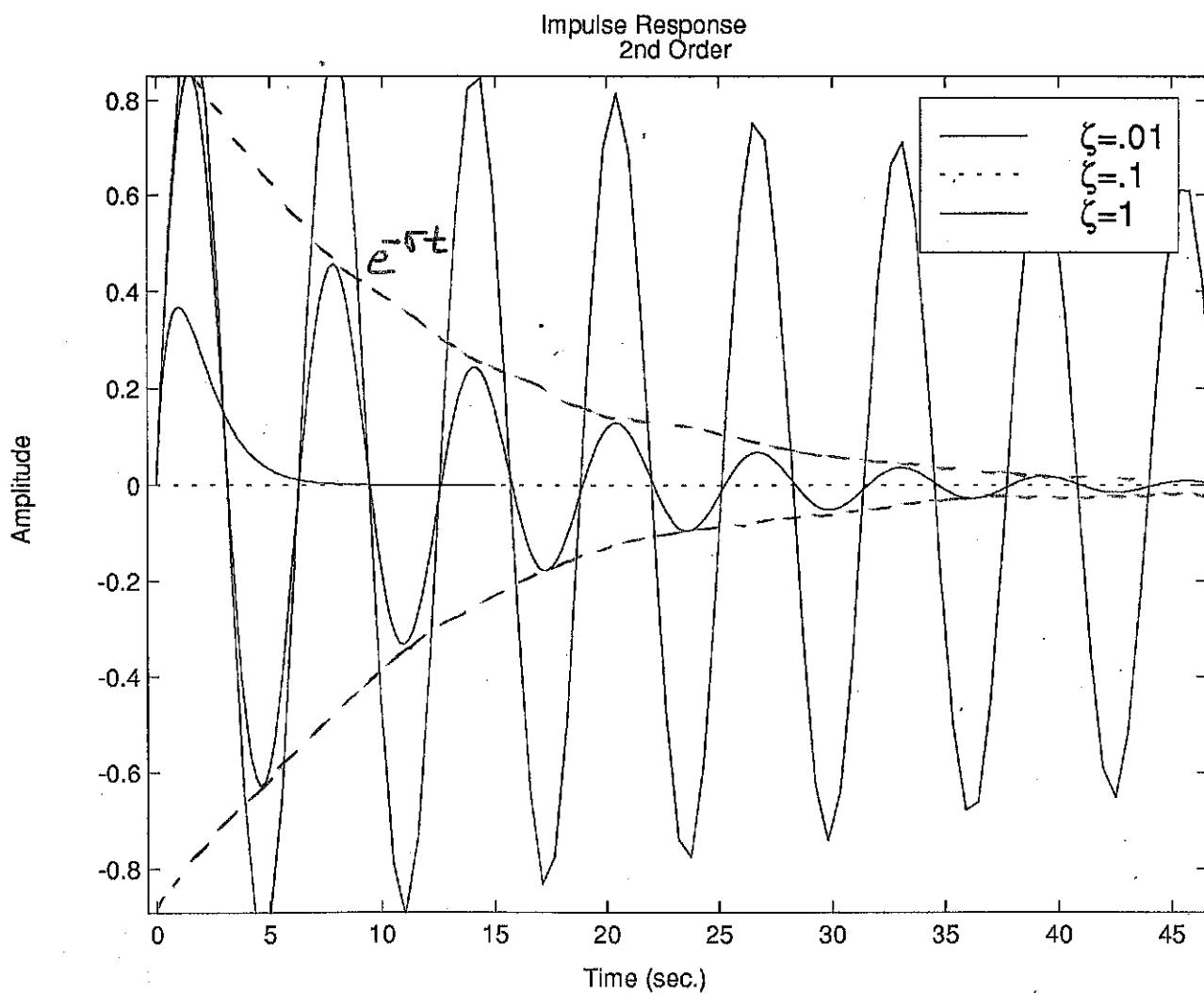
$\xi$  = Damping ratio     $\omega_d$  = damped natural frequency

- then in s-plane



$$H(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$\omega_n = 1.0$$



### Matlab Impulse Function

```
>>n=[wn^2];
>>d=[1 2*z*wn wn^2];
>>impulse(n,d);
```

stop!!  
6/3

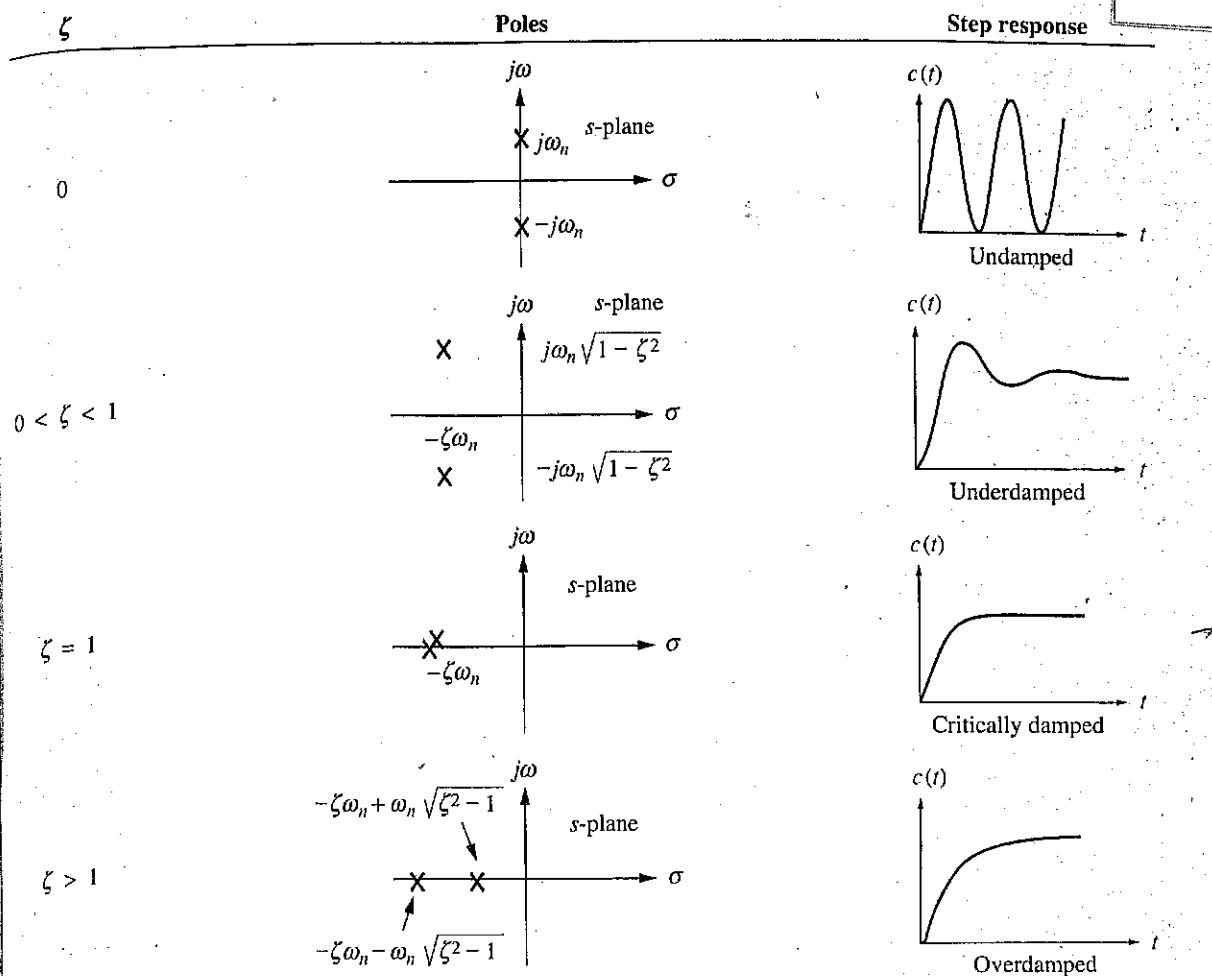


Figure 4.11 Second-order response as a function of damping ratio

Fig  
Systems for Example  
4.4