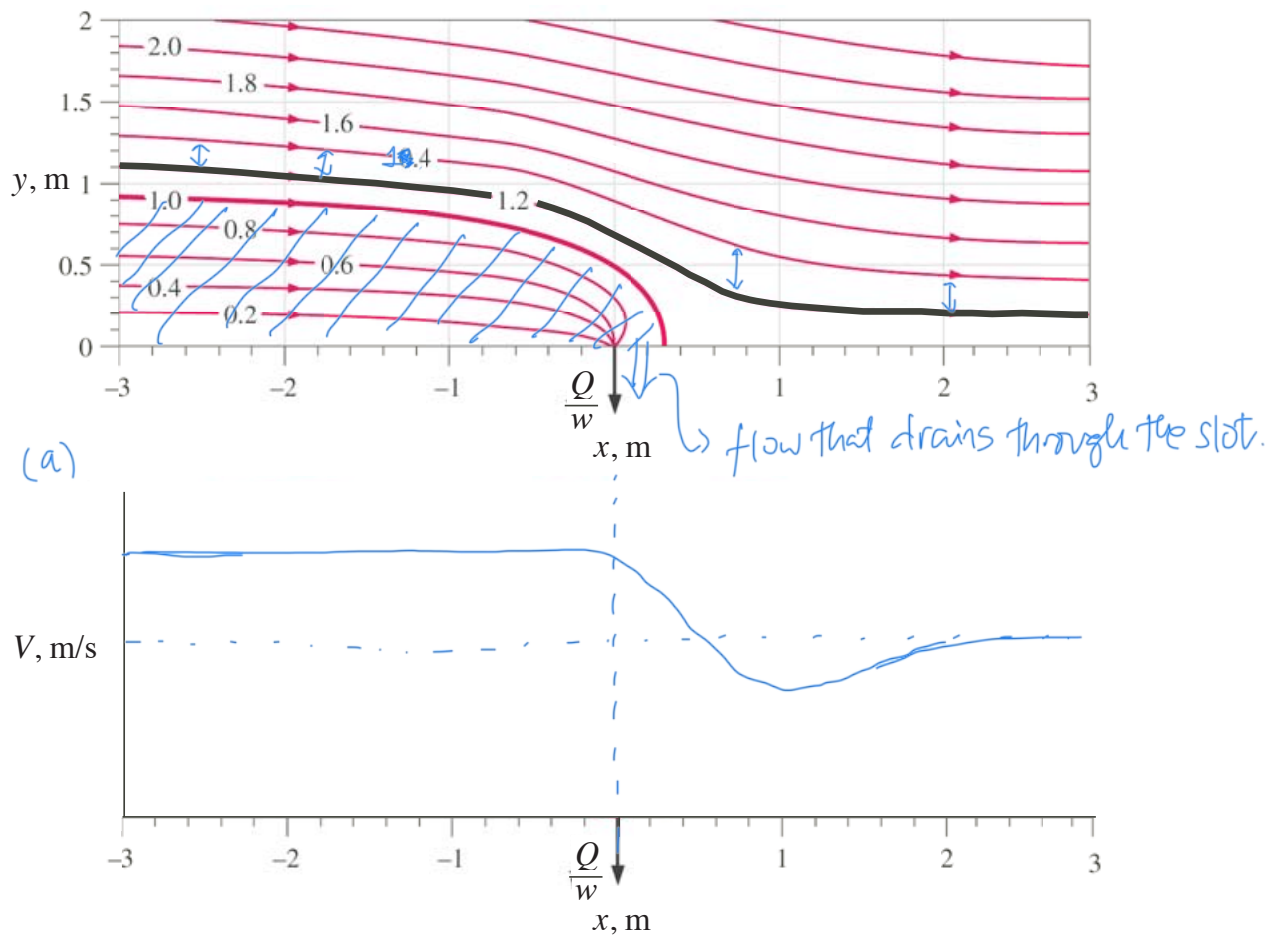


## In Class Practice Problem – Boundary Layer Theory (Fall 2025)

**P1.** Water drains through a narrow slot, located at  $x = 0$ ,  $y = 0$  on the bottom wall of a channel having a width of  $w = 2$  m. The streamline pattern and corresponding values of the streamfunction  $\psi$  are shown below.

- In the figure provided, plot the magnitude of the velocity  $V$  as a function of  $x$  along the streamline drawn in black. [Be sure to include the correct values of  $V$  on the  $y$ -axis]
- What is the volume flow rate  $Q$  through the slot? [Hint: assume the streamline along the bottom wall between  $-3 \leq x \leq 0.3$  has a value  $\psi = 0$ .]



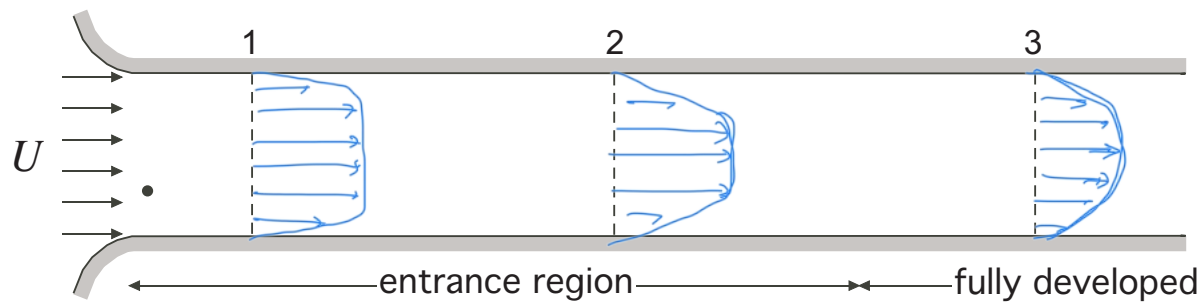
To estimate the velocity we use the fact that the volume flow rate between two streamlines is constant and equal to the difference in  $\psi$  values:

$$\text{@ } x = -3 : u = \frac{\partial \psi}{\partial y} \approx \frac{\Delta \psi}{\Delta y} = \frac{1.4 - 1.2}{0.15} = 1.33 \text{ m/s}$$

$$\text{@ } x = 3 : u = \frac{\partial \psi}{\partial y} \approx \frac{\Delta \psi}{\Delta y} = \frac{1.4 - 1.2}{0.4 - 0.2} = 1 \text{ m/s}$$

$$\text{(b) The volume flow rate } Q = \Delta \psi = 1 - 0 = 1 \text{ m}^3/\text{s}.$$

**P2.** Steady flow enters a two-dimensional channel with a uniform velocity  $U$ . Assume the pressure gradient  $\partial p / \partial x$  remains constant throughout the entire length of the channel.



- Draw (to-scale) the velocity profiles at locations 1, 2, and 3 on the figure provided.
- Write the simplified  $x$ - AND  $y$ -momentum equations in the *entrance region*.

$$\left. \begin{aligned} x\text{-mom: } u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} &= -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \frac{\partial^2 u}{\partial x^2} + \nu \frac{\partial^2 u}{\partial y^2} \\ y\text{-mom: } \frac{\partial p}{\partial y} &= 0 \end{aligned} \right\}$$

- Write the simplified  $x$ - AND  $y$ -momentum equations in the *fully developed* region.

$$\begin{aligned} x\text{-mom: } \frac{1}{\rho} \frac{\partial p}{\partial x} &= \nu \frac{\partial^2 u}{\partial y^2} \\ y\text{-mom: } \frac{\partial p}{\partial y} &= 0 \end{aligned}$$

- Comment on the magnitude of the *vertical* velocity in the entrance region compared to the fully developed region.

The vertical velocity is zero everywhere in the fully developed region. Alternatively in the entrance zone a small (but non-negligible) vertical velocity exists due to the growth of the BL.

- e. Write down the mathematical expression for the acceleration of the fluid particle marked by • in the figure.

$$\frac{Du}{Dt} = \cancel{\frac{\partial u}{\partial t}} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y}$$

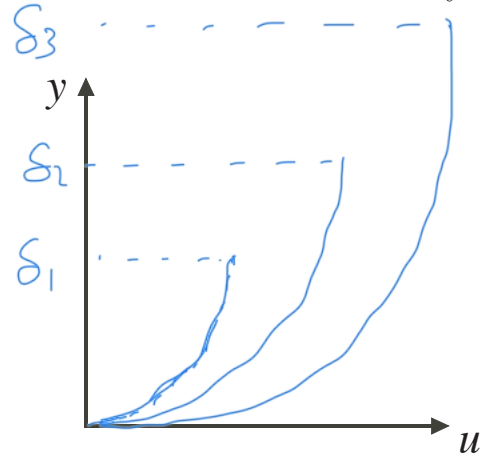
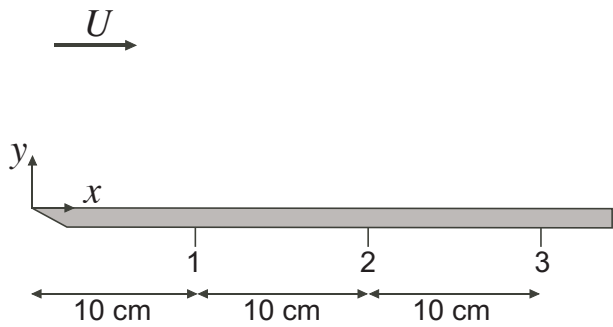
$$\frac{Dv}{Dt} = \cancel{\frac{\partial v}{\partial t}} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y}$$

- f. Describe what happens to the momentum of the fluid particle marked by • as it moves down the channel. [Be specific and use appropriate fluids terminology]

As the particle enters the channel, it will eventually enter the BL wherein its x-momentum will decrease and its y-momentum will increase. As it continues downstream, x-momentum will increase due to the pressure gradient.

In the fully developed region, momentum remains constant.

**P3.** Consider the steady flow over a flat plate as shown below. The freestream velocity is  $U = 5 \text{ m/s}$ .

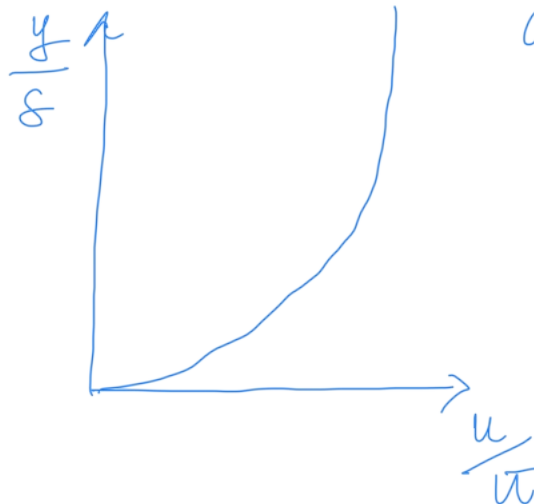


- a. Is the flow laminar at location 3 for the case of air? What about for the case of water? [Use calculations to support your answer. Note, the viscosities of air and water are  $1.5 \times 10^{-5} \text{ m}^2/\text{s}$  and  $1 \times 10^{-6} \text{ m}^2/\text{s}$ , respectively.]

$$\text{air: } Re = \frac{Ux}{\nu_a} = 1 \times 10^5 < Re_c \Rightarrow \text{Laminar}$$

$$\text{water: } Re = \frac{Ux}{\nu_w} = 1.5 \times 10^6 > Re_c \Rightarrow \text{Turbulent.}$$

- b. In the plot given, draw the *dimensional* velocity profiles at locations 1, 2, and 3 for the case of air.
- c. How would the figure change if you plotted  $u/U$  versus  $y/\delta$  instead, where  $\delta$  denotes the local boundary layer thickness?



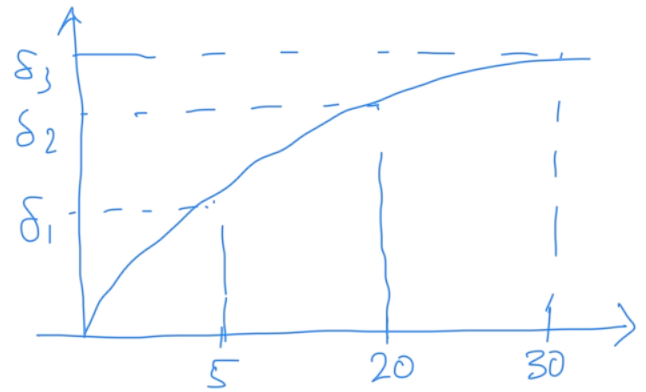
all profiles collapse.

- d. For the case of air, plot the boundary layer thickness  $\delta$  as a function of  $x$  over the range  $0 \leq x \leq 30$  cm.

$$\delta_1 = 5 / \sqrt{Re_x} = 2.7 \text{ mm}$$

$$\delta_2 = 3.9 \text{ mm}$$

$$\delta_3 = 4.7 \text{ mm}$$



- e. Consider flow through an octagonal pipe as shown in the figure. What is the increase in centerline velocity due to boundary layer growth over the pipe length  $L$ ? Assume a negligible pressure gradient, that the flow is uniform over the inflow cross section with a flow rate  $Q_0$  and an inflow velocity  $U_0$ , and that the wall boundary layers merge after some distance leading to a cylindrical inviscid core region. Using your answer and your own physical intuition, for  $L/D \approx 1$  is the impact of the wall boundary layers important in this flow?

