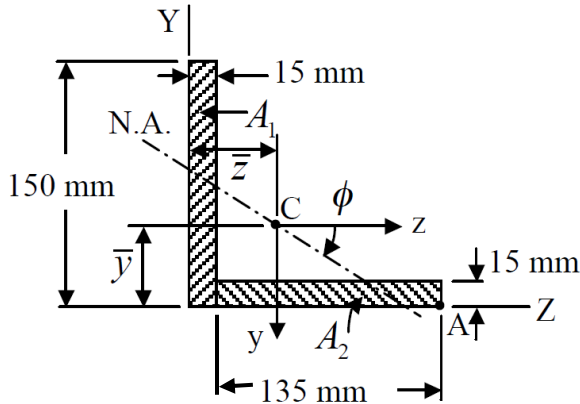
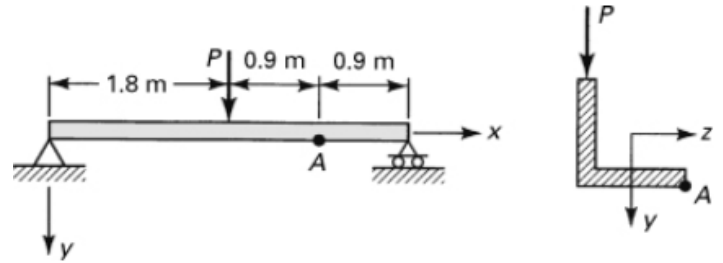


## Homework 8 Solutions

- 1) A simply supported beam constructed of a  $0.15 \times 0.15 \text{ m}$  angle is loaded by concentrated force  $P = 22.5 \text{ kN}$  at its midspan. Calculate stress  $\sigma_x$  at A and the orientation of the neutral axis. Neglect the effect of shear in bending and assume that beam twisting is prevented.



$$\bar{z} = \frac{A_1 z_1 + A_2 z_2}{A_1 + A_2} = \frac{(150 \times 15) 7.5 + (135 \times 15) [15 + (135/2)]}{150 \times 15 + 135 \times 15}$$

or  $\bar{z} = \bar{y} = 43 \text{ mm}$

Then,

$$I_y = \frac{1}{12} (150)(15)^3 + (150 \times 15)(35.5)^2 + \frac{1}{12} (15)(135)^3 + (135 \times 15)(39.5)^2$$

or  $I_y = I_z = 9.11(10^6) \text{ mm}^4$

$$I_{yz} = (150 \times 15)(-32)(-35.5) + (135 \times 15)(35.5)(39.5) = 5.4(10^6) \text{ mm}^4$$

We have the moment components:

$$M_y = 0, \quad M_z = -11.25(0.9) = -10.125 \text{ kN} \cdot \text{m}$$

Thus,

$$(\sigma_x)_A = \frac{-10125(5.4)(0.107) + 10125(9.11)(0.043)}{[(9.11)^2 - (5.4)^2](10^{-6})} = -35 \text{ MPa}$$

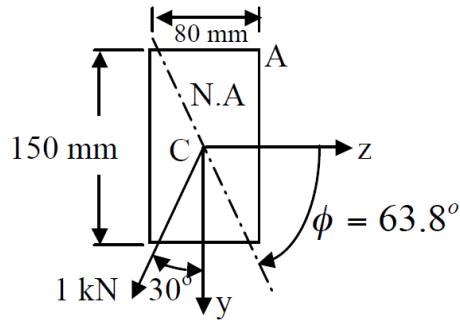
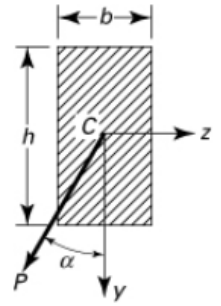
Equation (5.15) gives

$$\tan \phi = \frac{5.4}{9.11} = 0.593$$

or

$$\phi = 30.66^\circ$$

- 2) A wood cantilever beam with the cross section shown is subjected to an angled (as shown) load  $P$  at its free end. Determine (a) the orientation of the neutral axis; (b) the maximum bending stress. Given:  $P = 1 \text{ kN}$ ,  $\alpha = 30^\circ$ ,  $b = 80 \text{ mm}$ ,  $h = 150 \text{ mm}$ , and length  $L = 1.2 \text{ m}$ .



$$I_y = \frac{1}{12} h b^3 = \frac{1}{12} (150)(80)^3 = 6.4 \times 10^6 \text{ mm}^4$$

$$I_z = \frac{1}{12} b h^3 = \frac{1}{12} (80)(150)^3 = 22.5 \times 10^6 \text{ mm}^4$$

$$M_y = (P \sin \alpha)L = 600 \text{ N} \cdot \text{m}$$

$$M_z = (P \cos \alpha)L = 1,039.2 \text{ N} \cdot \text{m}$$

$$y_d = -75 \text{ mm} \quad z_d = 40 \text{ mm}$$

- ( a ) Equation (5.15) with  $I_{yz} = 0$  :

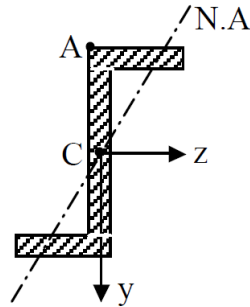
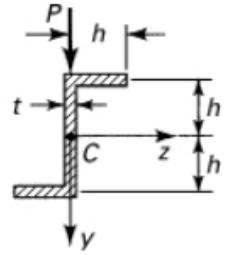
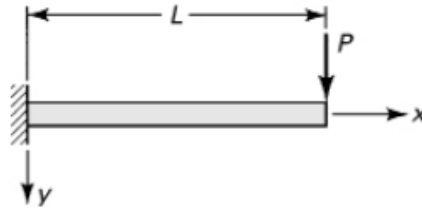
$$\tan \phi = \frac{I_z}{I_y} \frac{M_y}{M_z} = \frac{I_z}{I_y} \tan \alpha \quad \therefore \phi = 63.8^\circ$$

- ( b ) Thus, maximum tensile stress is at point A.

Equation (5.16) gives

$$\sigma_{\max} = \frac{600(0.04)}{6.4 \times 10^{-6}} - \frac{1,039.2(-0.075)}{22.5 \times 10^{-6}} = 7.21 \text{ MPa}$$

- 3) A cantilever beam has a Z section of uniform thickness for which  $I_y = \frac{2}{3}th^3$ ,  $I_z = \frac{8}{3}th^3$ , and  $I_{yz} = -th^3$ . Determine the maximum bending stress in the beam subjected to a load  $P$  at its free end.



We have  $M_y = 0$  and  $M_z = PL$

Equation (5.14) becomes

$$I_{yz}z = I_y y \quad \text{or} \quad -th^3 z = \frac{2}{3}th^3 y$$

or

$$y = -\frac{3z}{2}$$

Point A is the farthest from the N.A. Thus, with (In order to get the maximum stress, We consider point A, which is the farthest from the N.A.)

$$y_A = -h - \frac{t}{2} \quad \text{and} \quad z_A = -\frac{t}{2}$$

Equation (5.13) yields

$$(\sigma_x)_A = \frac{PL[-th^3(-t/2) - (2th^3/3)h^3(-h-t/2)]}{(2th^3/3)(8th^3/3) - (-th^3)^2}$$

or

$$(\sigma_x)_A = \frac{3PL(2.5t+2h)}{7th^3} = \sigma_{\max}$$

