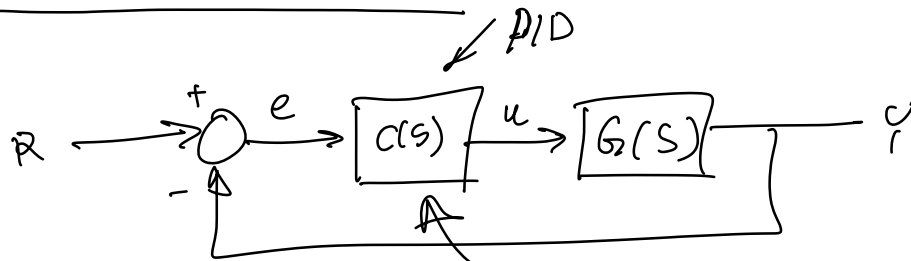
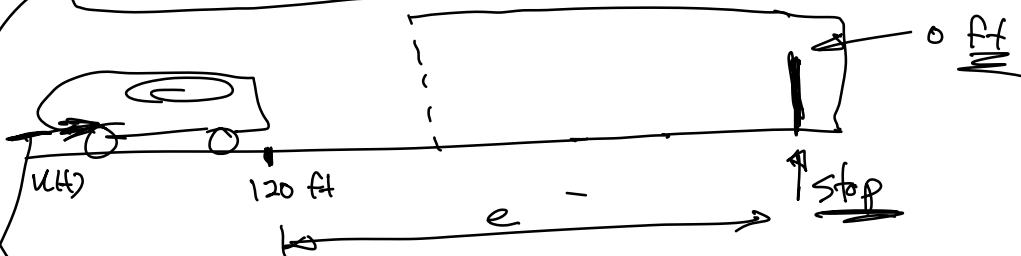


Classical PID control



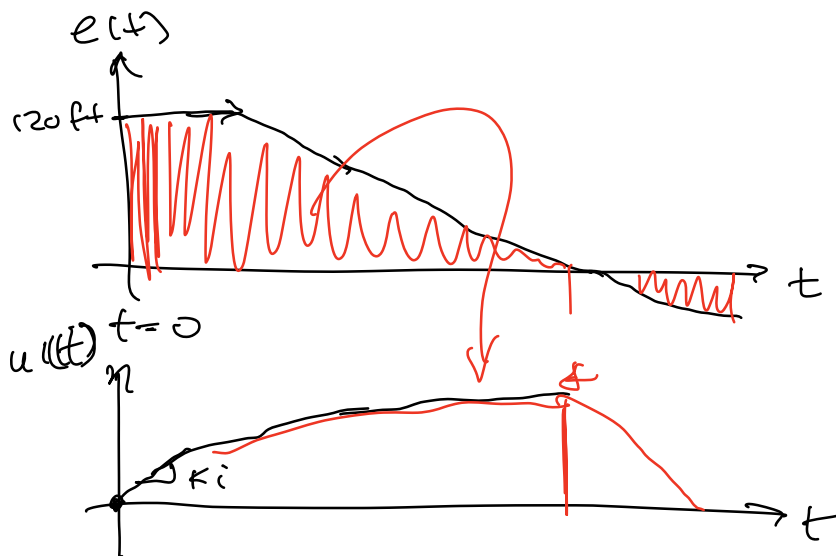
PID: proportional + Integral + derivative.

$$\frac{U(s)}{E(s)} = K_P + K_I/s + K_D s$$



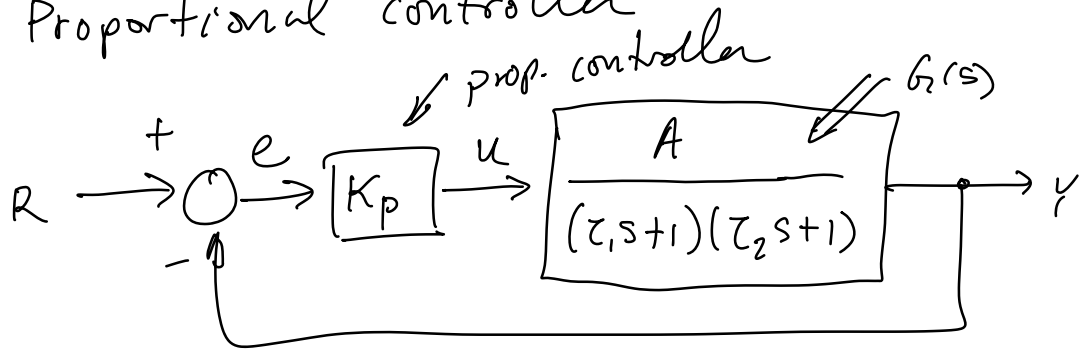
$$U(s) = K_P e(s) + \frac{K_I}{s} e(s) + K_D s e(s)$$

$$u(t) = K_P e(t) + K_I \int e(t) dt + K_D \dot{e}(t)$$



PID: closer look

1) Proportional controller



$$\frac{u(s)}{e(s)} = K_P \quad \text{prop. controller}$$

$$\frac{Y}{R} = \frac{K_P G_2(s)}{1 + K_P G_2(s)} = \frac{K_P \left(\frac{A}{(\tau_1 s + 1)(\tau_2 s + 1)} \right)}{\underbrace{1 + K_P \left(\frac{A}{(\tau_1 s + 1)(\tau_2 s + 1)} \right)}_{\text{poles of loop}}}$$

characteristic eq:

$$1 + K_P u(s) = 1 + K_P \frac{A}{(\tau_1 s + 1)(\tau_2 s + 1)} = 0$$

$$(\tau_1 s + 1)(\tau_2 s + 1) + K_p A = 0$$

$$\tau_1 \tau_2 s^2 + (\tau_1 + \tau_2) s + A K_p + 1 = 0$$

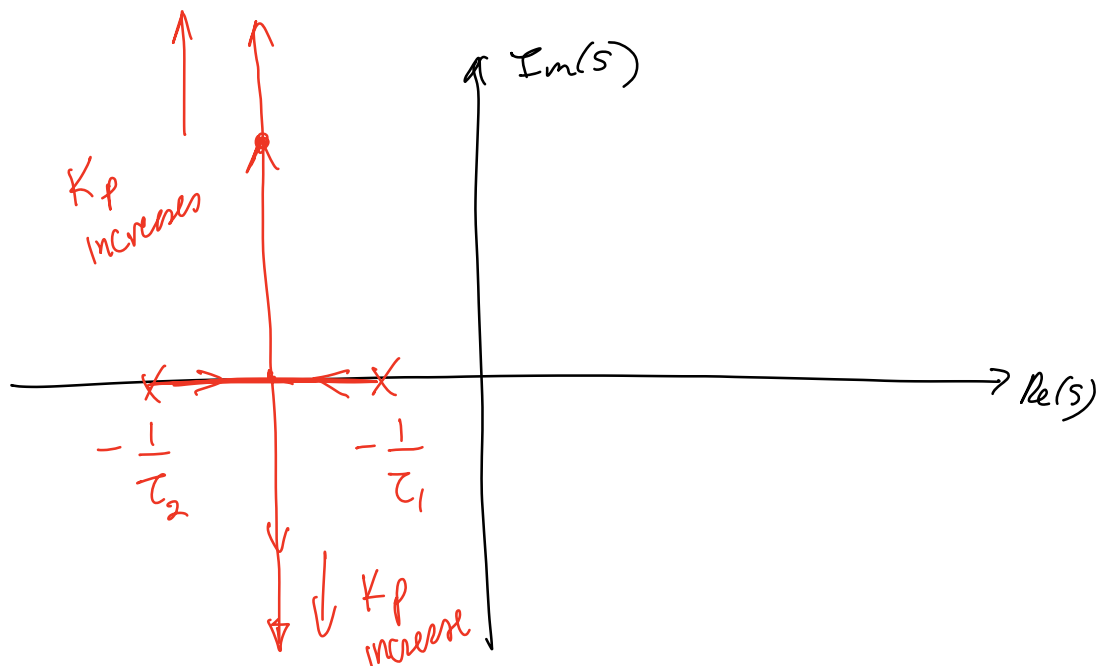
quadratic eq \Rightarrow 2 poles!

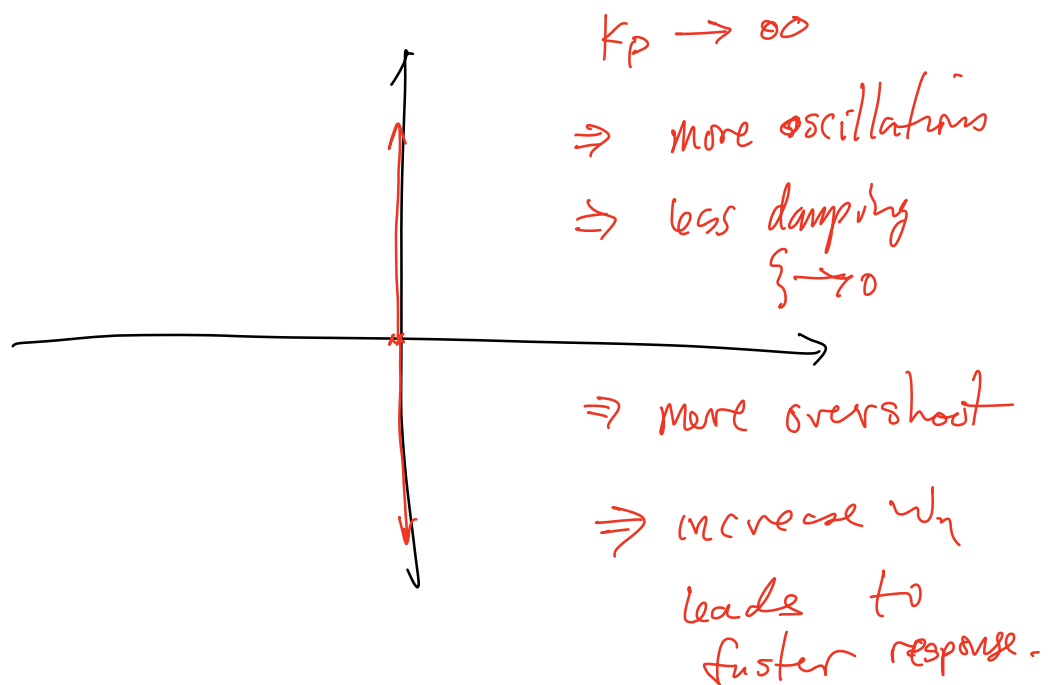
$K_p = 0$

$$s = -\frac{1}{\tau_1} \quad \text{and} \quad s = -\frac{1}{\tau_2}$$

when $K_p > 0$

$$s_{1,2} = \frac{-(\tau_1 + \tau_2) \pm \sqrt{(\tau_1 + \tau_2)^2 - 4\tau_1 \tau_2 (1 + A K_p)}}{2\tau_1 \tau_2}$$





Integral Control

Primarily used to deal w/ steady-state error, but can result in poor transient behavior and worse, can destabilize the system.

$$C(s) = \frac{U(s)}{E(s)} = \frac{K_i}{s}$$

Take back to
time domain.

$$\Rightarrow u(t) = K_i \int e(t) dt$$

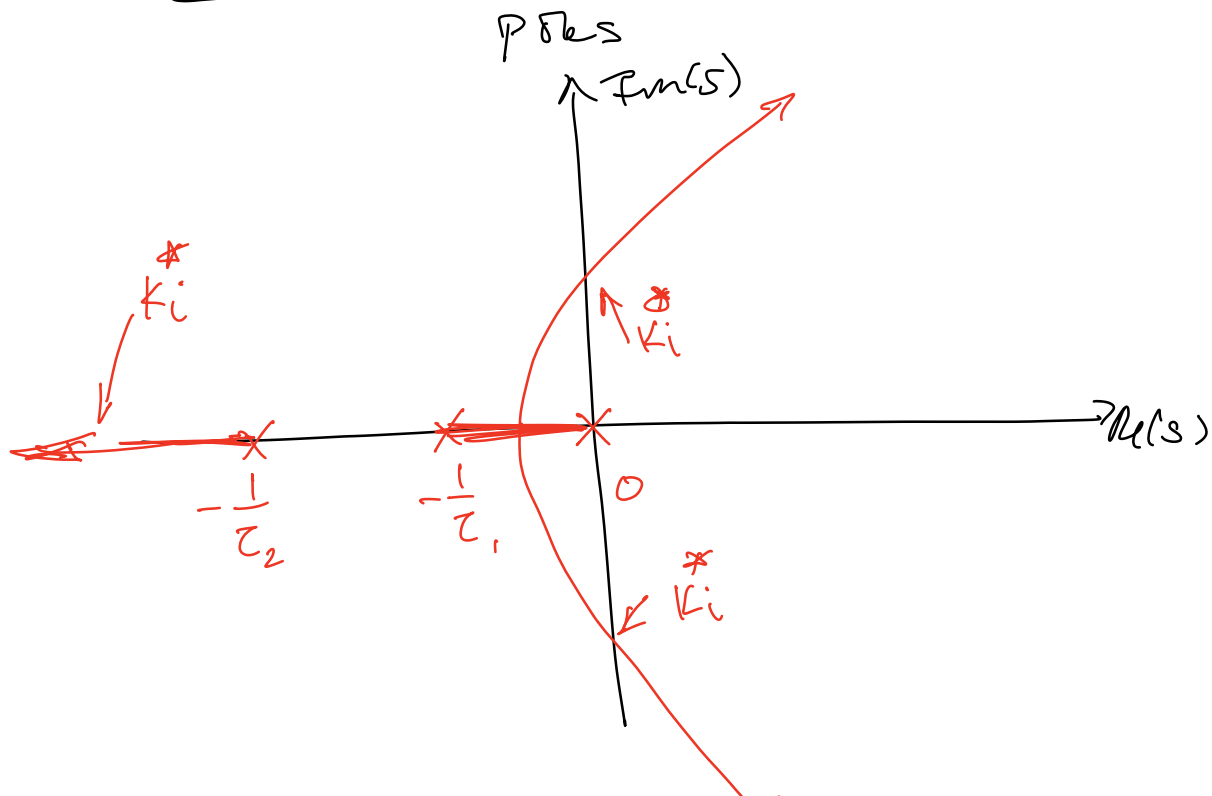
$$G(s) = \frac{A}{(\tau_1 s + 1)(\tau_2 s + 1)} \quad C(s) = \frac{K_i}{s}$$

closed-loop poles:

$$1 + C(s)G(s) = 1 + \frac{K_i}{s} \left(\frac{A}{(s\tau_1 + 1)(s\tau_2 + 1)} \right) = 0$$

$$\Rightarrow s(\tau_1 s + 1)(\tau_2 s + 1) + K_i A = 0$$

$$\Rightarrow \tau_1 \tau_2 s^3 + (\tau_1 + \tau_2)s^2 + s + K_i A = 0$$



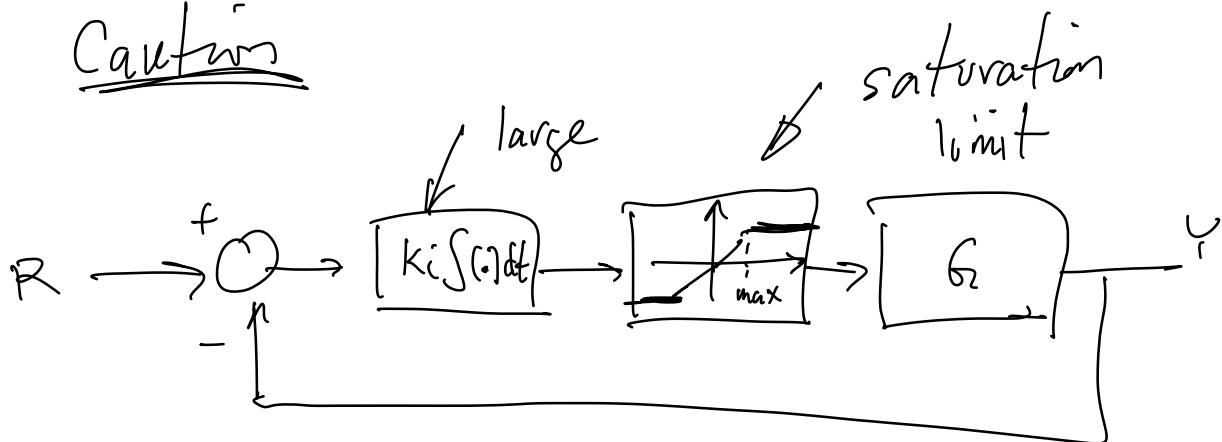
✓

Integral control:

$$K_i \rightarrow \infty$$

- ξ decrease
- Slower response
- Can go unstable!
- Decrease S.S. error

Cautions



Integrator wind-up — use saturation limit to cap the max control effort so you don't cause additional problems/instability.

Derivative

$$C(s) = K_d s + K_p \quad (\text{prop. + derivative})$$

$$G(s) = \frac{A}{(\tau_1 s + 1)(\tau_2 s + 1)}$$

$$\frac{Y(s)}{R(s)} = \frac{A(K + K_d s)}{\underbrace{\tau_1 \tau_2 s^2 + (\tau_1 + \tau_2)s + 1}_{\text{denominator}} + AK_p + AK_d s}$$

characteristic eq. for loop system:

$$\tau_1 \tau_2 \overset{2}{s}^2 + (\tau_1 + \tau_2 + AK_d) \overset{1}{s} + (1 + AK_p)$$

2nd order

$$s^2 + 2\zeta \omega_n s + \omega_n^2$$

$$A \left[s^2 + \left(\frac{\tau_1 + \tau_2 + AK_d}{\tau_1 \tau_2} \right) s + \frac{1 + K_p A}{\tau_1 \tau_2} \right]$$

Knobs: K_p , K_d

if I adjust

$K_d \rightarrow$ damping

$K_p \rightarrow \omega_n$

