

ME 3710

Homework 5

Due Tuesday February 22 at 11:59pm – upload to canvas

[6 problems –18 pts]

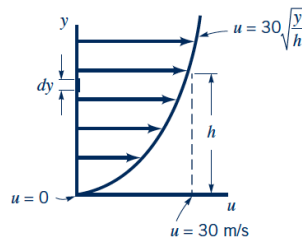
Note that for chapter 5 problems, the solution manual doesn't always go through all of the steps that we have gone through in class: 1) draw CV, 2) state Assumptions, 3) list Givens, 4) write down fundamental equations, 5) Simplify and solve. Please do this!

Solution 5.2

From the equation

$$\bar{V} = \frac{\int_A \rho \mathbf{V} \cdot \mathbf{n} dA}{\rho A} \text{ or with } \rho = \text{constant,}$$

$$\bar{V} = \frac{\int_A \mathbf{V} \cdot \mathbf{n} dA}{A}$$



Consider a unit depth normal to the x - y plane so that

$$A = 1 \times h = h \text{ and } dA = 1 \times dy = dy$$

Thus,

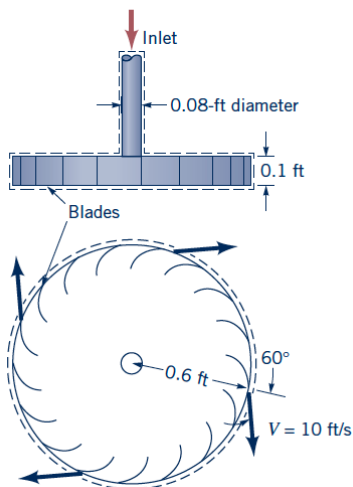
$$\bar{V} = \frac{\int u dA}{A} = \frac{\int_0^h u dy}{h} = \frac{\int_0^h 30 \sqrt{\frac{y}{h}} dy}{h} = \frac{30}{h^2} \int_0^h y^{\frac{1}{2}} dy = \frac{30}{h^2} y^{\frac{3}{2}} \left(\frac{2}{3} \right) \bigg|_{y=0}^{y=h} = \frac{2}{3} (30) \frac{\text{m}}{\text{s}}$$

or

$$\bar{V} = 20 \frac{\text{m}}{\text{s}}$$

Solution 5.4

Use the control volume container within the broken lines as shown in the sketch below.



From the conservation of mass principle $\dot{m}_{\text{inlet}} = \dot{m}_{\text{outlet}}$

$$\begin{aligned}\text{Also } \dot{m}_{\text{outlet}} &= \rho A_{\text{outlet}} V_{\text{outlet}} \cos 60^\circ = \rho 2\pi r_{\text{outlet}} h V_{\text{outlet}} \cos 60^\circ \\ &= \left(1.94 \frac{\text{slugs}}{\text{ft}^3}\right) 2\pi (0.6 \text{ ft}) (0.1 \text{ ft}) \left(10 \frac{\text{ft}}{\text{s}}\right) \cos 60^\circ \\ &= \underline{\underline{3.66 \frac{\text{slugs}}{\text{s}}}}\end{aligned}$$

Solution 5.13

For steady flow

$$\dot{m}_3 = \dot{m}_1 + \dot{m}_2$$

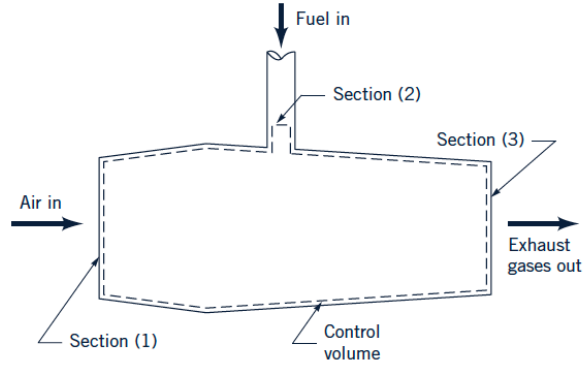
or

$$\rho_3 A_3 \bar{V}_3 = \dot{m}_1 + \dot{m}_2$$

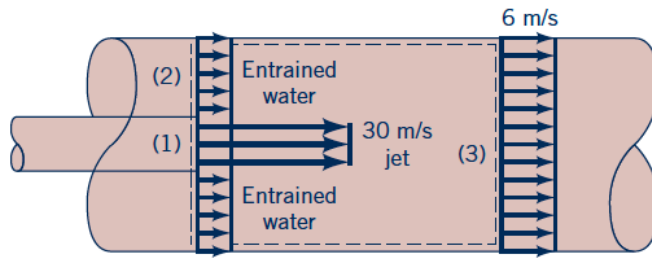
Thus

$$\rho_3 = \frac{\dot{m}_1 + \dot{m}_2}{A_3 \bar{V}_3} = \frac{65 \frac{\text{lbm}}{\text{s}} + 0.60 \frac{\text{lbm}}{\text{s}}}{(3.5 \text{ ft}^2) \left(1500 \frac{\text{ft}}{\text{s}}\right)}$$

$$\rho_3 = \underline{\underline{0.0125 \frac{\text{lbm}}{\text{ft}^3}}}$$



Solution 5.17



For steady incompressible flow through the control volume

$$Q_1 + Q_2 = Q_3$$

or

$$\bar{V}_1 A_1 + Q_2 = \bar{V}_3 A_3$$

Thus

$$Q_2 = \bar{V}_3 A_3 - \bar{V}_1 A_1 = \left[\left(6 \frac{\text{m}}{\text{s}} \right) (0.075 \text{ m}^2) - \left(30 \frac{\text{m}}{\text{s}} \right) (0.01 \text{ m}^2) \right] \left(1000 \frac{\text{kg}}{\text{m}^3} \right)$$

$$Q_2 = 150 \frac{\text{kg}}{\text{s}}$$

Solution 5.18

The mass flowrate is calculated with

$$\dot{m} = \int_0^R \rho u 2\pi r dr = 2\pi \rho \int_0^R u r dr$$

where

$$R = 3 \text{ in.}$$

$$\rho = 0.00238 \frac{\text{slug}}{\text{ft}^3}$$

$$u = \text{local axial velocity in } \frac{\text{ft}}{\text{s}}$$

$$r = \text{local radius in in.}$$

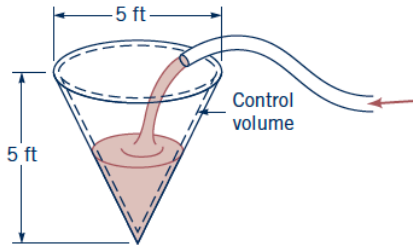
and $\int_0^R u r dr$ is evaluated numerically with the trapezoidal rule with unequal intervals.

The result is:

$$\dot{m} = 2\pi \left(0.00238 \frac{\text{slug}}{\text{ft}^3} \right) \int_{r=0}^{r=\frac{3}{12}\text{ft}} u \frac{\text{ft}}{\text{s}} (r \text{ ft}) (dr \text{ ft}) = 0.0114 \frac{\text{slugs}}{\text{s}}$$

Consider doing this problem in metric units. If you do the answer is: 0.0204 kg/s

Solution 5.27



From application of the conservation of mass principle to the control volume shown in the figure, we have

$$\frac{\partial}{\partial t} \int_{CV} \rho dV + \int_{CS} \rho \mathbf{V} \cdot \mathbf{n} dA = 0$$

For incompressible flow

$$\frac{\partial V}{\partial t} - Q = 0$$

or

$$\int_0^V dV = Q \int_0^t dt$$

Thus

$$t = \frac{V}{Q} = \frac{\pi D^2 h}{12 Q} = \frac{\pi (5 \text{ ft})^2 (5 \text{ ft}) \left(1728 \frac{\text{in.}^3}{\text{ft}^3} \right)}{(12) \left(20 \frac{\text{gal}}{\text{min}} \right) \left(231 \frac{\text{in.}^3}{\text{gal}} \right)}$$

and

$$\underline{\underline{t = 12.2 \text{ min}}}$$