

二〇〇二年答案解析

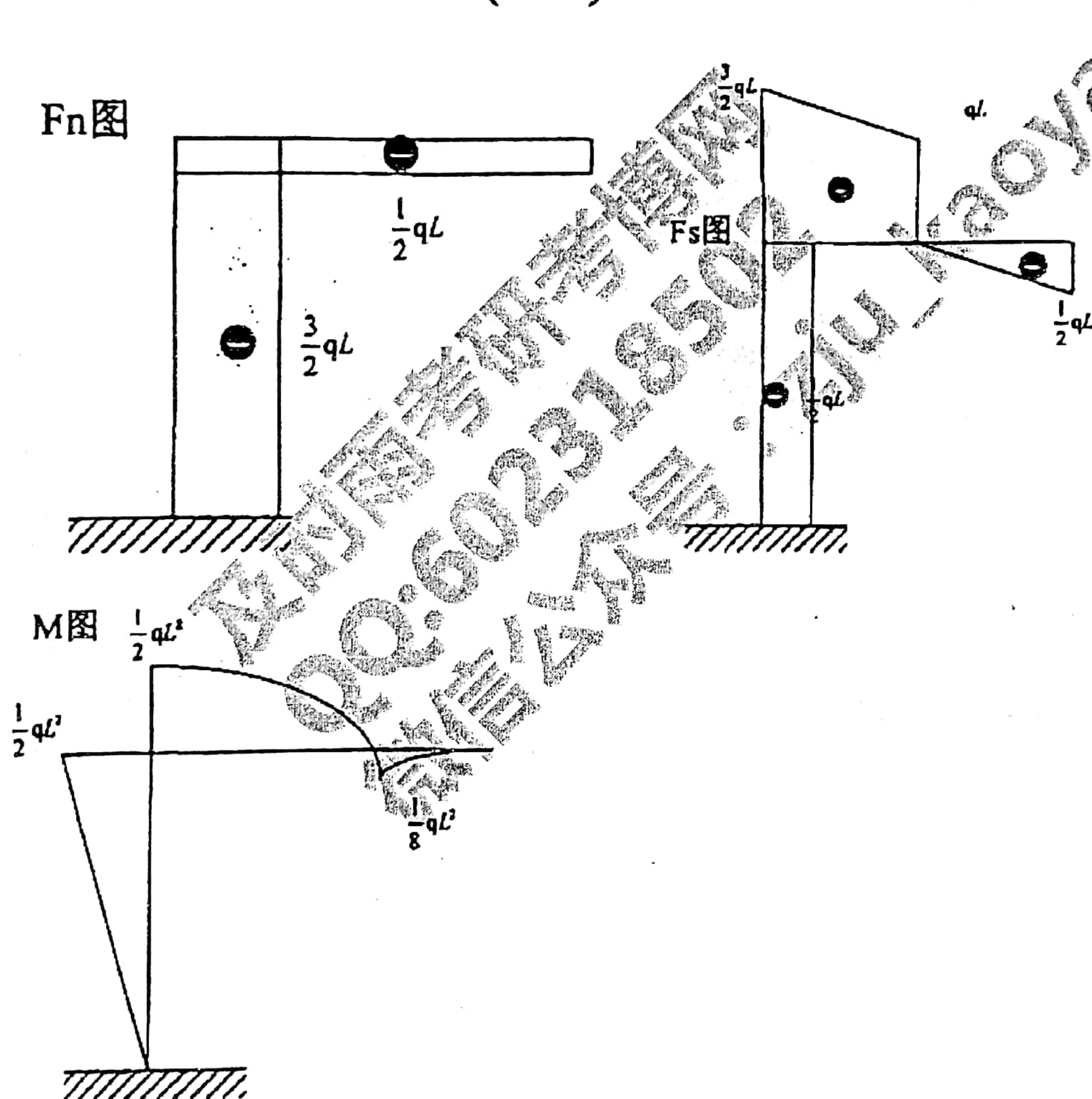
一、解: $\sum M_A = 0, -\frac{1}{2}qL^2 - \frac{1}{2}qL^2 + \frac{qL^2}{2} + F_C L = 0 \quad F_C = \frac{qL}{2} (\uparrow)$

$$\sum F_y = 0 \quad F_{Ay} = \frac{3}{2}qL (\uparrow)$$

$$\sum F_x = 0 \quad F_{Ax} = \frac{1}{2}qL (\rightarrow)$$

AB 段 $M(x) = \frac{qLx}{2} - \frac{qx^2}{2}$

BC 段 $M(x) = \frac{qLx}{2} - \frac{qx^2}{2} - qL\left(x - \frac{L}{2}\right) = -\frac{1}{2}qx^2 + \frac{1}{2}qL^2 - \frac{qLx}{2}$



二、解: (1)、a:

$$\left. \begin{matrix} \sigma_{\max} \\ \sigma_{\min} \end{matrix} \right\} = \frac{3\sigma + \sigma}{2} \pm \sqrt{\left(\frac{3\sigma - \sigma}{2}\right)^2 + (\sqrt{3}\sigma)^2} = \begin{cases} 4\sigma \\ 0 \end{cases}$$

$$\sigma_1 = 4\sigma, \quad \sigma_2 = \sigma_3 = 0$$

(2)、b:

$$\left. \begin{array}{l} \sigma_{\max} \\ \sigma_{\min} \end{array} \right\} = \frac{-\sqrt{3}\sigma + \sqrt{3}\sigma}{2} \pm \sqrt{\left(\frac{-\sqrt{3}\sigma - \sqrt{3}\sigma}{2} \right)^2 + \sigma^2} = \begin{cases} 2\sigma \\ -2\sigma \end{cases}$$

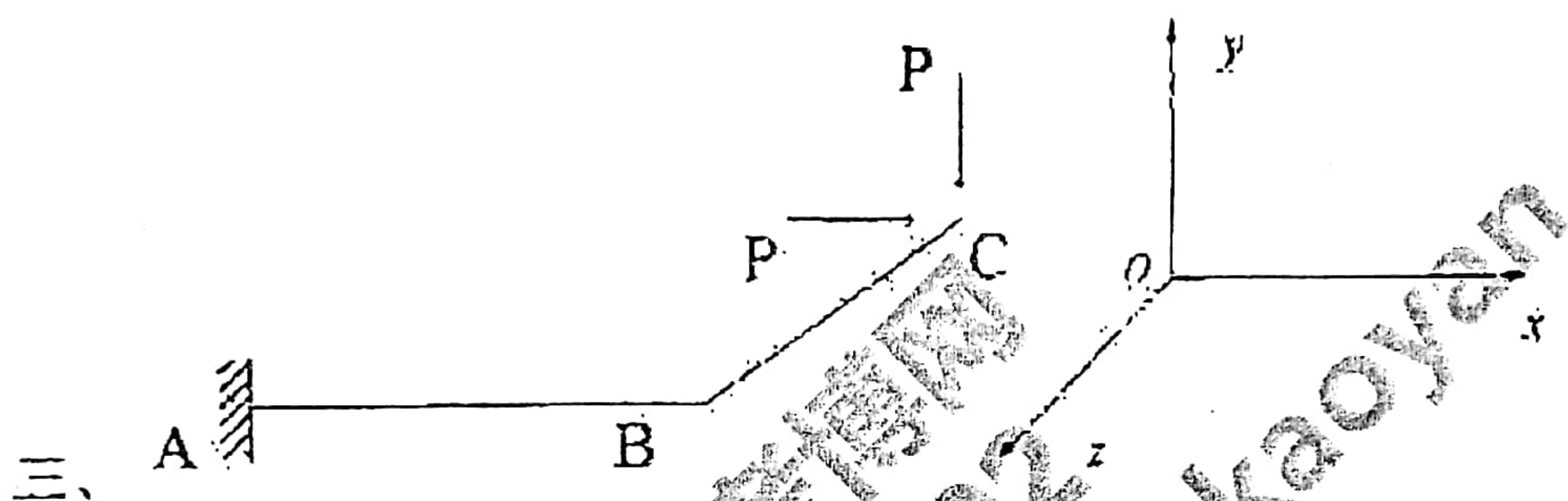
$$\sigma_1 = 2\sigma, \quad \sigma_2 = 0, \quad \sigma_3 = -2\sigma$$

$$(2) \quad a: \quad \varepsilon_{\max} = \frac{\sigma_1 - \sigma_2}{2} = 2\sigma$$

$$b: \quad \varepsilon_{\max} = \frac{\sigma_1 - \sigma_3}{2} = 2\sigma$$

(3)、a 单元体处于单轴应力状态

b 单元体处于纯剪切应力状态



(1)、BC 杆 B 端: $M_{B,x} = PL$ $M_{B,y} = PL$

$$\sigma = \frac{\sqrt{M_{B,x}^2 + M_{B,y}^2}}{W} = \frac{32\sqrt{2}PL}{\pi d^3}$$

$$\sigma_{1,3} = \sigma = \frac{32\sqrt{2}PL}{\pi d^3}$$

(2)、AB 杆的 B 截面 $M_{B,y} = PL$

$$T = PL$$

$$\sigma = \frac{M_{B,y}}{W} = \frac{32PL}{\pi d^3} \quad \tau = \frac{T}{W_p} = \frac{16PL}{\pi d^3}$$

$$\sigma_{1,3} = \sqrt{\sigma^2 + 4\tau^2} = \frac{32\sqrt{2}PL}{\pi d^3}$$

(3)、AB 杆的 A 截面

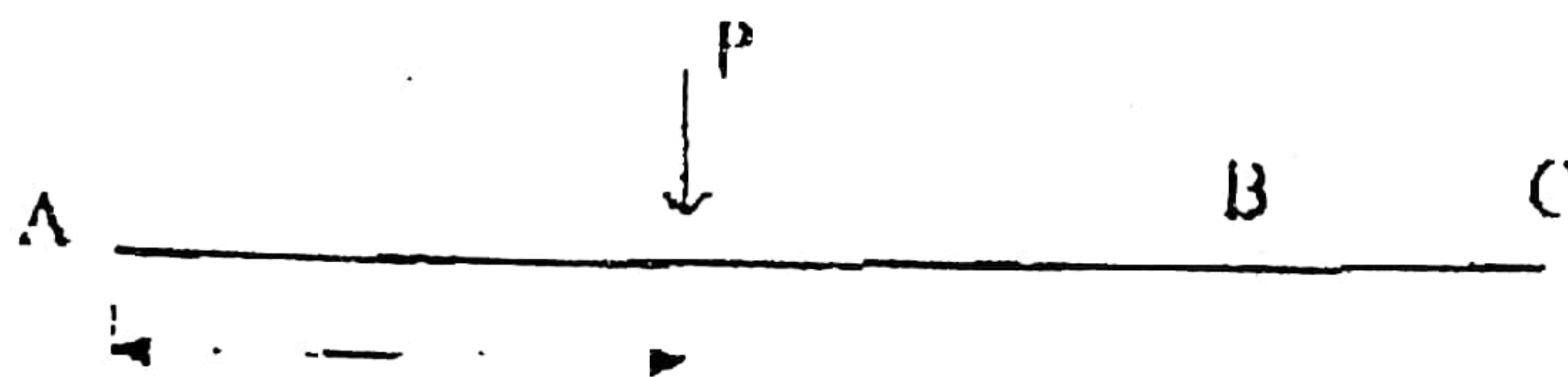
$$M_{A,x} = PL \quad M_{A,y} = PL$$

$$\sigma = \frac{\sqrt{M_{A,x}^2 + M_{A,y}^2}}{W} = \frac{32\sqrt{2}PL}{\pi d^3}$$

$$T = PL \quad \tau = \frac{T}{W_t} = \frac{16PL}{\pi d^3}$$

$$\sigma_{\max} = \sqrt{\sigma^2 + 4\tau^2} = \frac{32\sqrt{3}PL}{\pi d^3}$$

四、解：当 P 在 A、B 之间运动时

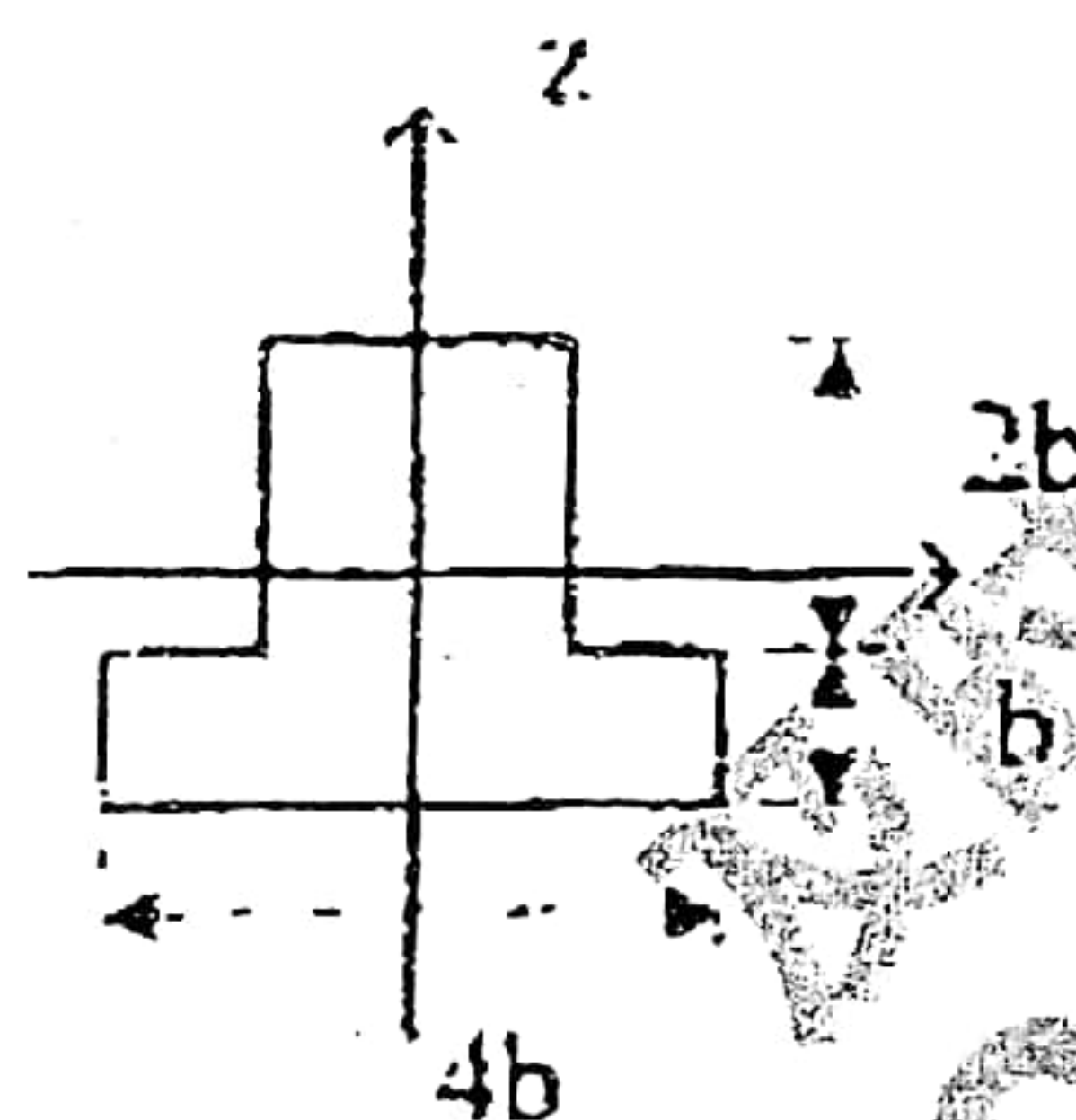


$$\sum M_A = 0, \quad F_B = \frac{Px}{6L}, \quad F_C = \frac{P(6L-x)}{6L}$$

$$\text{故 } M(x) = \frac{P(6L-x)}{6L}x \quad 0 \leq x \leq L$$

$$\frac{\partial M(x)}{\partial x} = 0, \quad x = 3L$$

$$M_{\max} = M(3L) = \frac{P(3L)^2}{6L} = \frac{3PL}{2}$$



$$\bar{y} = \frac{2b \times 2b \times 2b + b \times 4b \times 0.5b}{2b \times 2b + b \times 4b} = \frac{5}{4}b$$

$$I_{yy} = \frac{2b(2b)^3}{12} + \left(\frac{3}{4}b\right)^2 \cdot 4b^2 + \frac{4bb^3}{12} + \left(\frac{3}{4}b\right)^2 \cdot 4b^2 = \frac{37b^4}{6}$$

$$\sigma_{\max 1} = \frac{M_{\max} \frac{5}{4}b}{I_{yy}} = \frac{\frac{3}{2}PL \times \frac{5}{4}b}{\frac{37}{6}b^4} = \frac{15}{8} \times \frac{6PL}{37b^3} = \frac{45PL}{148b^3} \leq [\sigma_1] = 2[\sigma_2]$$

$$\sigma_{\max 2} = \frac{M_{\max} \frac{7}{4}b}{I_{yy}} = \frac{\frac{3}{2}PL \times \frac{7}{4}b}{\frac{37}{6}b^4} = \frac{63PL}{148b^3} \leq [\sigma_2]$$

$$\text{所以, } P \leq \frac{148b^3[\sigma_2]}{63PL}$$

当 P 在 BC 运动时, C 处弯矩最大, $M_{B, \max} = PL$

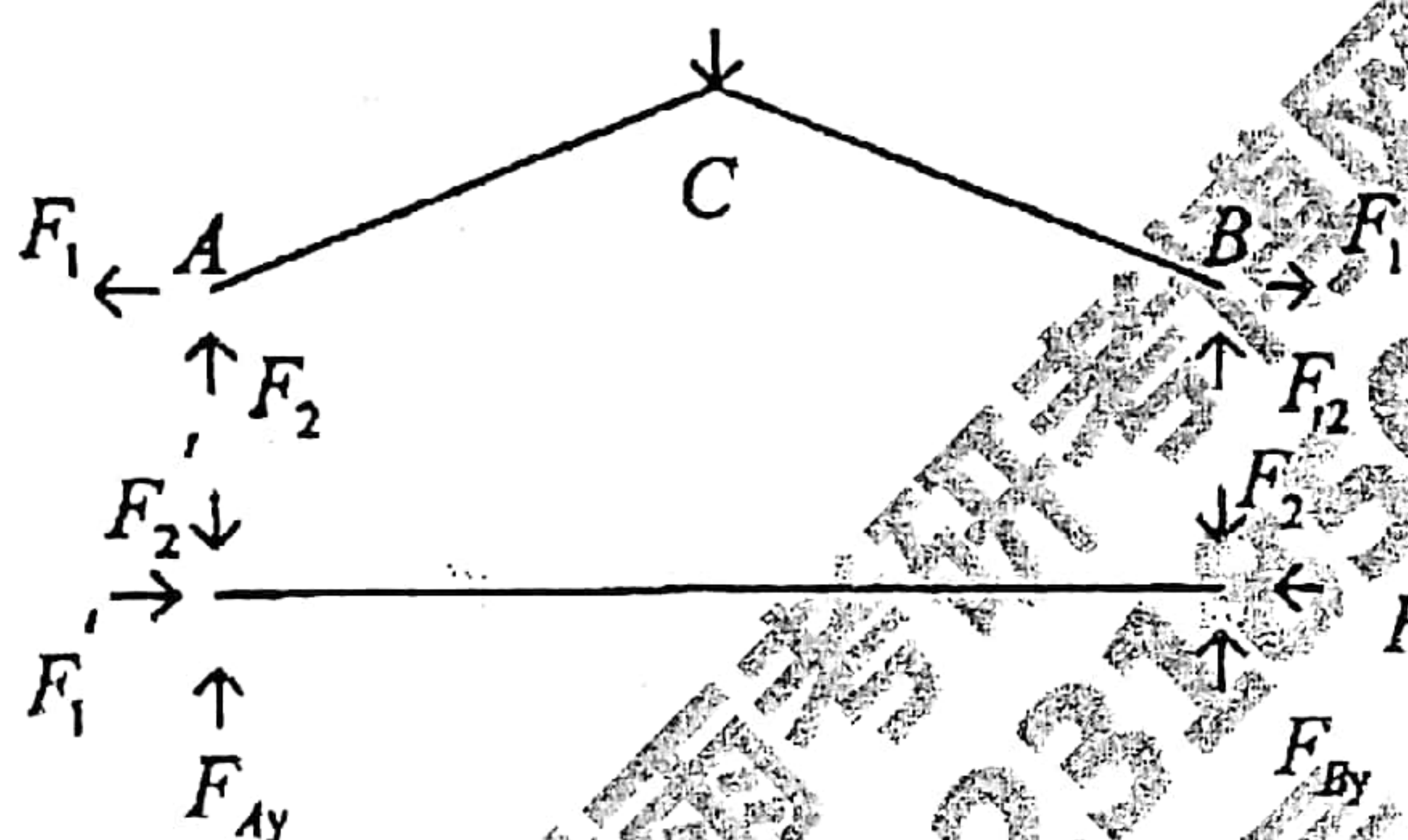
$$\sigma_{\max, c} = \frac{PL \times \frac{5}{4}b}{\frac{37}{6}b^4} = \frac{15PL}{74b^3} \leq [\sigma_c]$$

$$\sigma_{\max, l} = \frac{PL \times \frac{7}{4}b}{\frac{37}{6}b^4} = \frac{35PL}{74b^3} \leq 2[\sigma_c]$$

$$\text{所以 } P \leq \frac{148b^3[\sigma_c]}{35PL}$$

$$\text{综上, } P \leq \frac{148b^3[\sigma_c]}{63PL}$$

五、解: (1)、



$$\text{所以 } F_2 = F_2' = \frac{P}{2}$$

$$F_{Ay} = F_{By} = \frac{P}{2}$$

$$\text{有因为 } \Delta_{AB} = \Delta_{AB}' \quad \Delta_{AB} = 0$$

$$\text{再求 } \Delta_{AB}, \text{ AB 杆弯矩 } M(x) = F_1 \cos 45^\circ x + F_2 \sin 45^\circ x \quad 0 \leq x \leq L$$

$$\text{故 } \Delta_{AB} = 2 \int_0^L \frac{\left(F_1 \frac{\sqrt{2}}{2} x + F_2 \frac{\sqrt{2}}{2} x \right) \frac{\sqrt{2}}{2} x}{EI} dx = \frac{F_1 L^3}{3EI} + \frac{PL^3}{6EI} = 0$$

$$\text{则有 } F_1 = -\frac{P}{2} (\rightarrow)$$

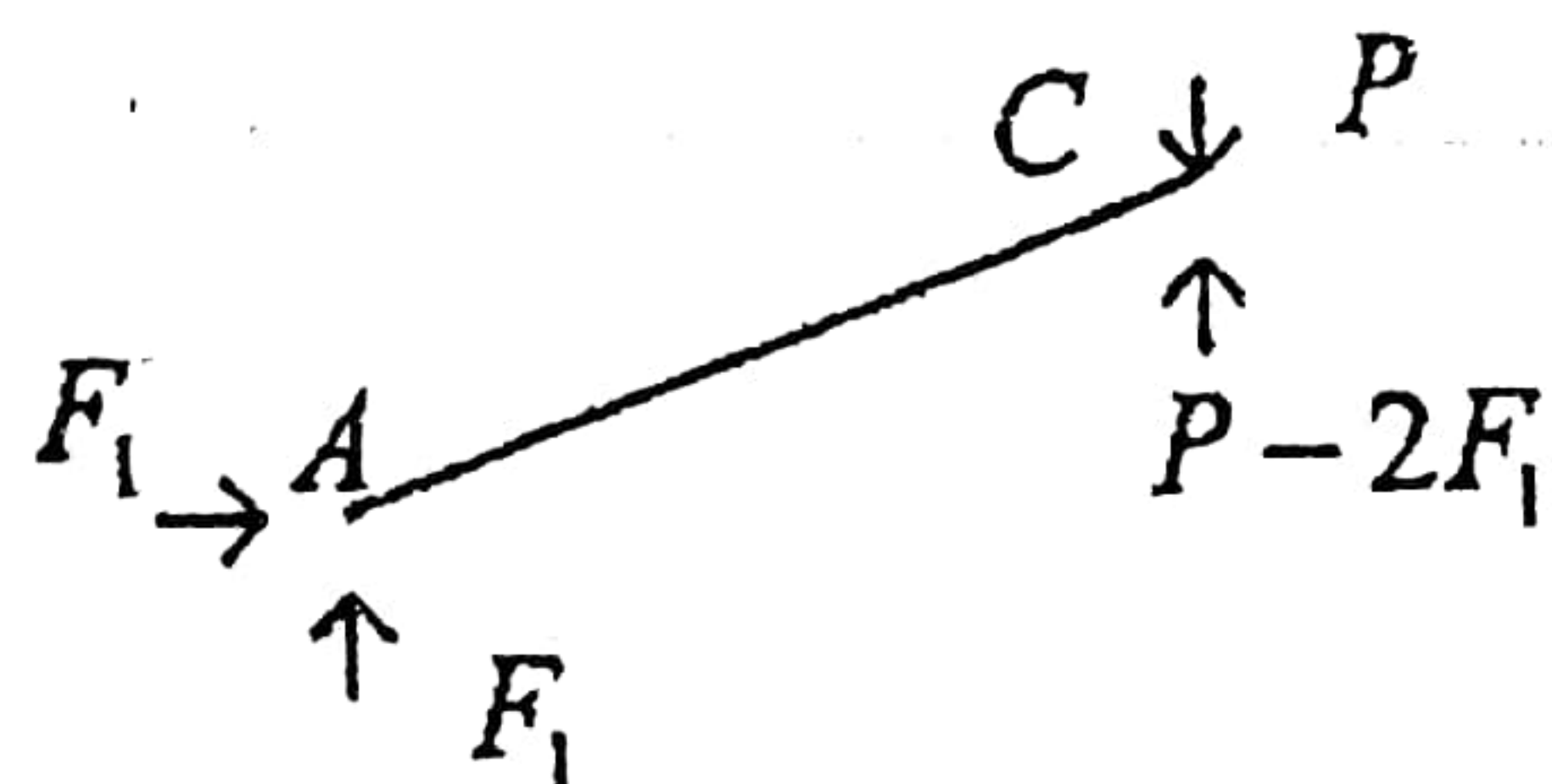
(2)、 ΔC 和 BC

$$M(x) = 0$$

$$F_{NAC} = -\frac{\sqrt{2}}{2} P \text{ (压应力)}$$

$$\text{所以 } \Delta_P = \frac{\bar{F}_{NAC} F_{NAC} L}{EA} \times 2 = \frac{PL}{EA}$$

(3)、 $\delta = \frac{PL}{2EA} < \omega_C$, 故 C 与水平支座接触



易知 $M(x) = 0$, 故 $F_x = F_y = F_1$, 则水平支座 $F = P - 2F_1$

$$F_N = -\sqrt{2}F_1$$

$$V_c = \frac{2F_1^2 L}{2EA} \times 2 = \frac{2F_1^2 L}{EA} \quad \delta = \frac{PL}{2EA}$$

$$\text{故 } \frac{\partial V_c}{\partial P} = \frac{PL}{2EA}$$

$$V_c = \frac{P^2 L}{4EA} = \frac{2F_1^2 L}{EA}$$

$$\text{因此, } F_1 = \frac{1}{2\sqrt{2}} P(\uparrow)$$