

二〇〇三年答案解析

一、解：(1)、自由度 $3 \times 3 - 2 \times (2 + 2) - 1 = 0$ ，因此属于静定结构

(2)、由 D 点的平衡关系，得 $F_{NAD} = 0$

$$\sum F_x = 0 \quad F_1 \cos \theta + F_2 = 0$$

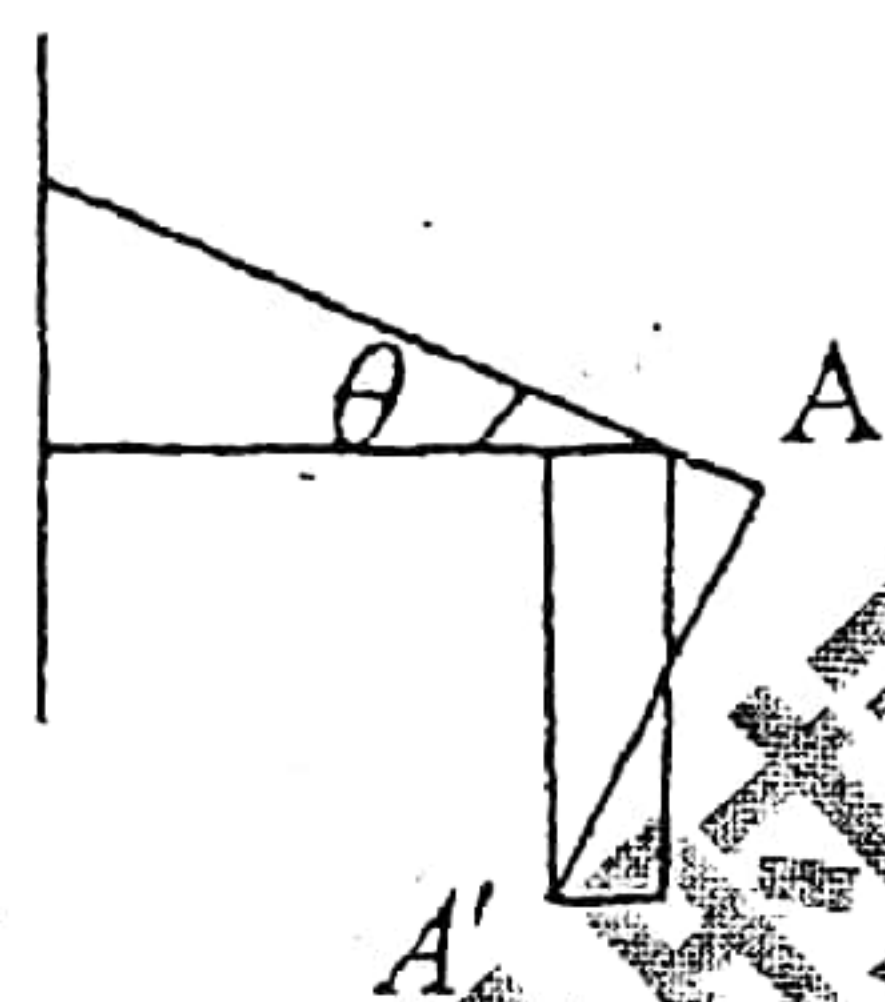
$$\sum F_y = 0 \quad F_1 \sin \theta - P = 0$$

$$\text{得 } F_1 = \frac{P}{\sin \theta}, \quad F_2 = -\frac{P \cos \theta}{\sin \theta}$$

$$(3)、\sigma_{AC} = \frac{F_1}{A_{AC}} = \frac{P}{A \sin \theta} \quad (\text{拉应力})$$

$$\sigma_{BA} = \frac{F_2}{A_{BA}} = -\frac{P \cot \theta}{2A} \quad (\text{压应力})$$

$$\sigma_{AD} = 0$$



(4)、

$$\Delta L_{AC} = \frac{\sigma_{AC} L_{AC}}{E} = \frac{PL}{A \sin \theta \cos \theta} \quad (\text{伸长})$$

$$\Delta L_{BA} = \frac{\sigma_{BA} L_{BA}}{E} = -\frac{PL \cot \theta}{2A} \quad (\text{压缩})$$

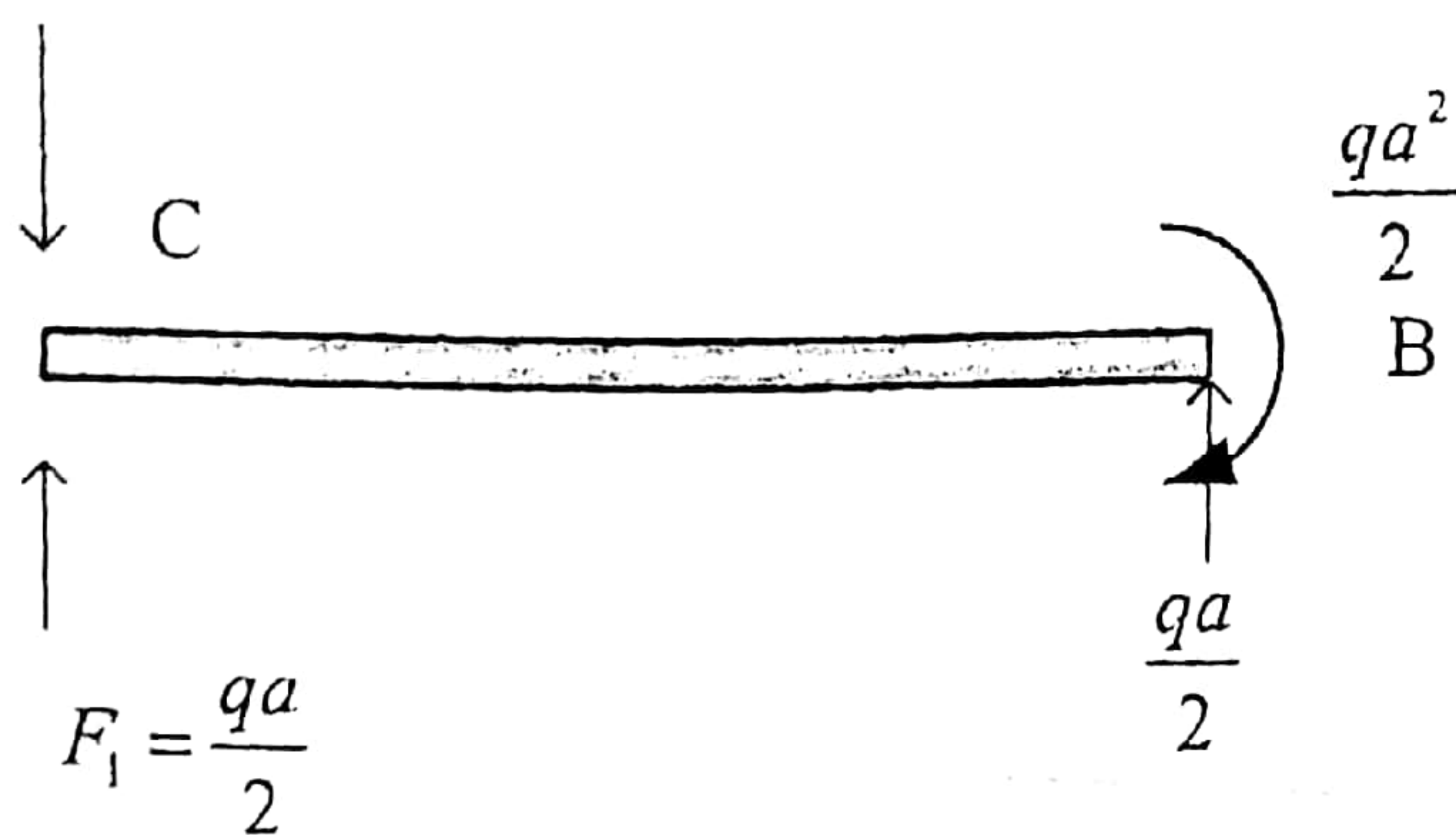
$$\Delta L_{AD} = 0$$

$$(5)、\Delta_{Ax} = \Delta L_{AB} = -\frac{PL \cot \theta}{2A} \quad (\leftarrow)$$

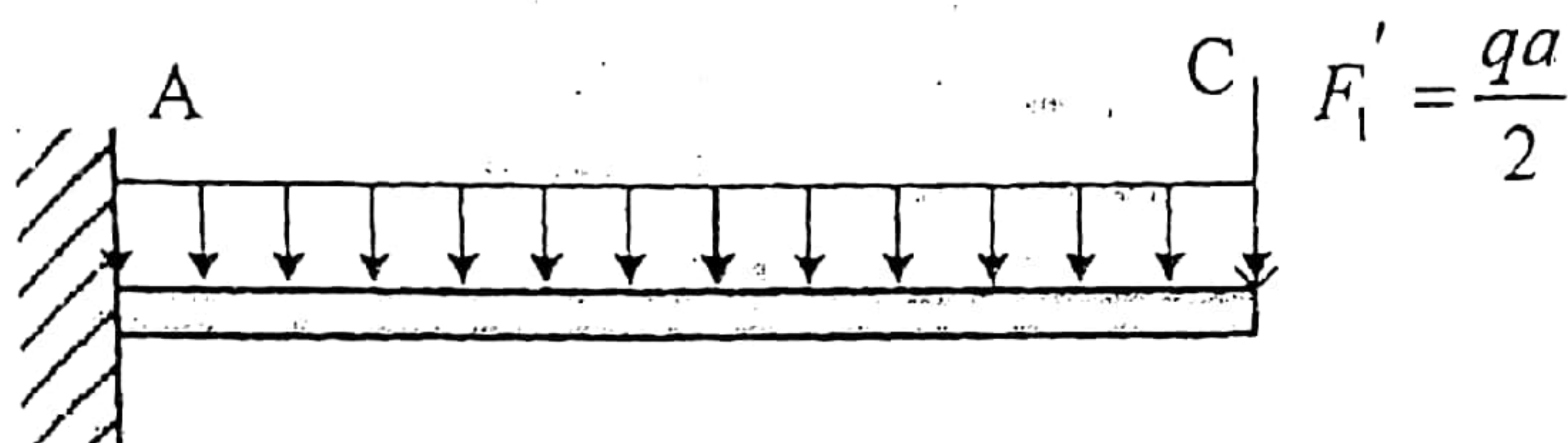
$$\Delta_{Ay} = \frac{\Delta L_{AB}}{\tan \theta} + \frac{\Delta L_{AC}}{\sin \theta} = \frac{PL(\cos^3 \theta + 2)}{2A \sin^2 \theta \cos \theta} \quad (\downarrow)$$

二、解：C 右部分

qa

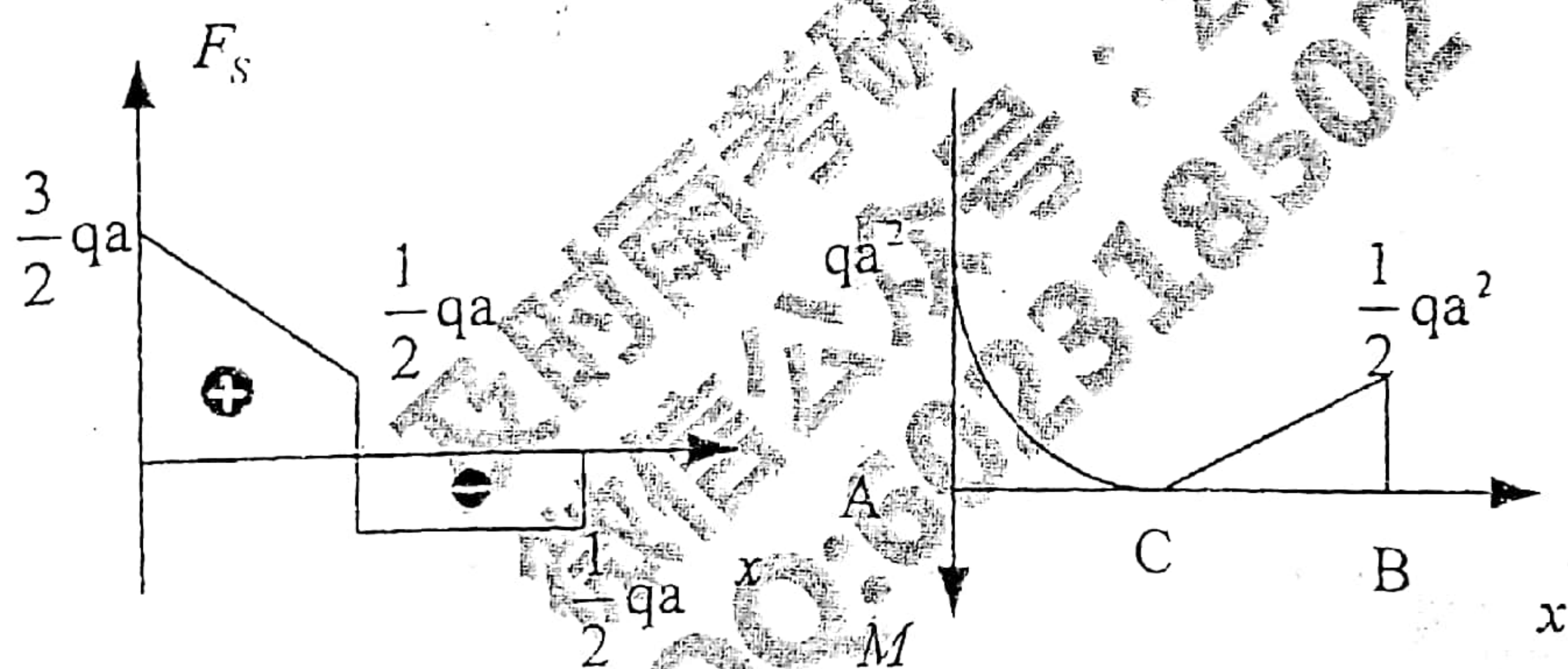


$$\sum M_C = 0, \quad F_B \cdot a - \frac{qa^2}{2} = 0, \quad F_B = \frac{qa}{2} (\uparrow)$$



$$\text{CA 段, } F_N = \frac{qa}{2} + qx \quad 0 \leq x \leq a$$

$$M(x) = -\frac{qa}{2}x - \frac{qx^2}{2} \quad 0 \leq x \leq a$$



$$\text{三、解: 易知, } F_A = F_B = \frac{qa}{2}, \quad F_s(x) = \frac{qL}{2} - qx \quad 0 \leq x \leq L$$

$$(1), \quad \tau_{\max} = \frac{F_{S,\max} S_z^*}{Ib} = \frac{3qL}{4bh}$$

$$(2), \quad M(x) = \frac{qLx}{2} - \frac{qx^2}{2} \quad 0 \leq x \leq L$$

$$M_{\max} = \frac{qL^2}{8}, \quad \sigma_{\max} = \frac{M(x)}{W} = \frac{\frac{qL^2}{8}}{\frac{1}{6}bh^2} = \frac{3qL^2}{4bh^2}$$

$$(3)、\text{下边缘}\varepsilon(x)=\frac{M(x)}{EI}=\frac{1}{E\frac{1}{6}bh^2}\left(\frac{1}{2}qLx-\frac{1}{2}qx^2\right)$$

$$\text{则}\Delta L=\int_0^L\varepsilon(x)dx=\int_0^L\frac{1}{E\frac{1}{6}bh^2}\left(\frac{1}{2}qLx-\frac{1}{2}qx^2\right)dx=\frac{qL^3}{2Ebh^2}$$

(4)、 $\sigma(x,y)=\frac{M(x)y}{I}$, 可知, 在非边界横截面上正应力沿高度成正比, 在两端边上为0

四、解: 取 σ_x 可取作为一个主应力, 大小为50MPa

$$(1)、\sigma_x=90MPa \quad \tau_{xy}=-40MPa \quad \sigma_y=30MPa$$

$$\left. \begin{array}{l} \sigma_{\max} \\ \sigma_{\min} \end{array} \right\} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} = \begin{cases} 110MPa \\ 10MPa \end{cases}$$

故 $\sigma_1=110MPa$, $\sigma_2=50MPa$, $\sigma_3=10MPa$

$$(2)、\tau_{\max}=\frac{\sigma_1-\sigma_3}{2}=50MPa$$

(3)、

$$\varepsilon_1=\frac{1}{E}[\sigma_1-\nu(\sigma_2+\sigma_3)]=\frac{1}{200\times 10^9}[110-0.3(50+10)]\times 10^6=4.6\times 10^{-4}$$

$$\varepsilon_2=\frac{1}{E}[\sigma_2-\nu(\sigma_1+\sigma_3)]=\frac{1}{200\times 10^9}[50-0.3(110+10)]\times 10^6=7\times 10^{-5}$$

$$\varepsilon_3=\frac{1}{E}[\sigma_3-\nu(\sigma_1+\sigma_2)]=\frac{1}{200\times 10^9}[10-0.3(50+110)]\times 10^6=-1.9\times 10^{-4}$$

$$(4)、\theta=\frac{1-2\nu}{E}(\sigma_1+\sigma_2+\sigma_3)=\frac{1-2\times 0.3}{200\times 10^9}(110+10+50)\times 10^6=3.4\times 10^{-4}$$

(5)、最大拉应力理论: $\sigma_{r1}=\sigma_1=110MPa$

最大伸长线应变理论: $\sigma_{r2}=\sigma_1-\nu(\sigma_2+\sigma_3)=92MPa$

最大剪应力理论: $\sigma_{r3}=\sigma_1-\sigma_3=100MPa$

形状改变能理论:

$$\sigma_{r4}=\sqrt{\frac{1}{2}[(\sigma_1-\sigma_2)^2+(\sigma_1-\sigma_3)^2+(\sigma_2-\sigma_3)^2]}=\sqrt{\frac{1}{2}(100^2+40^2+60^2)}\times 10^3=87.18MPa$$

五、解：(1)、 $I = \frac{1}{64} \pi d^4$ $A = \frac{1}{4} \pi d^2$

$$i = \sqrt{\frac{I}{A}} = \frac{d}{4} \quad \lambda_{CD} = \frac{\mu L}{i} = \frac{4a}{d} (\mu = 1)$$

$$(2)、F_{cr} = \frac{\pi^2 EI}{(\mu L)^2} = \frac{\pi^3 E d^4}{64 a^2}$$

故 $\frac{F_{cr}}{n_{st}} \geq \frac{[P]}{\sqrt{3}}$, 得 $[P] = \frac{\sqrt{3} \pi^3 E d^4}{192 a^2}$

(3)、受力分析, 可知, $F_{NAC} = F_{NAD} = F_{NBC} = F_{NBD} = \frac{P}{\sqrt{3}}$

$$F_{NCD} = -\frac{P}{\sqrt{3}}$$

(4)、在 AB 两端施加单位力, 则 $\bar{F}_{NAC} = \bar{F}_{NAD} = \bar{F}_{NBC} = \bar{F}_{NBD} = \frac{1}{\sqrt{3}}$

$$\bar{F}_{NCD} = -\frac{1}{\sqrt{3}}$$

$$\text{则 } \Delta_{AB} = \sum \frac{F_N \bar{F}_N L_i}{EA} = \frac{\frac{1}{3} Pa \times \frac{1}{\sqrt{3}}}{EA} \times 4 + \frac{\frac{1}{3} Pa \times \frac{-1}{\sqrt{3}}}{EA} = \frac{20 Pa}{3 \pi d^2 E}$$

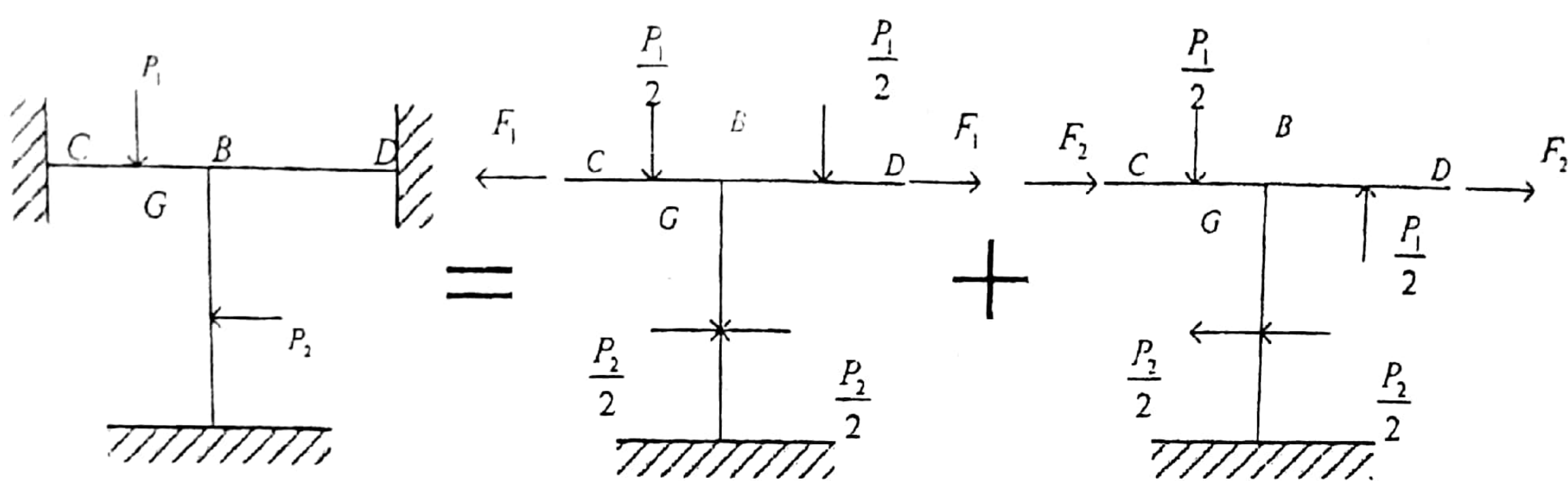
(5)、改变作用力 P 方向时, $F'_{NAC} = F'_{NAD} = F'_{NBC} = F'_{NBD} = -\frac{P}{\sqrt{3}}$

$$\bar{F}'_{NCD} = \frac{1}{\sqrt{3}}$$

对于 AC 杆, $F_{cr} = \frac{\pi^2 EI}{(\mu L)^2} = \frac{\pi^3 E d^4}{64 a^2}$

$$\frac{F_{cr}}{n_{st}} \leq \frac{[P]}{\sqrt{3}}, \quad P = \frac{\sqrt{3} \pi^3 E d^4}{192 a^2}, \quad \text{容许值未改变}$$

六、此题源自孙训方课本上的正反对称类型题 (下册 3-14)



原结构

图一

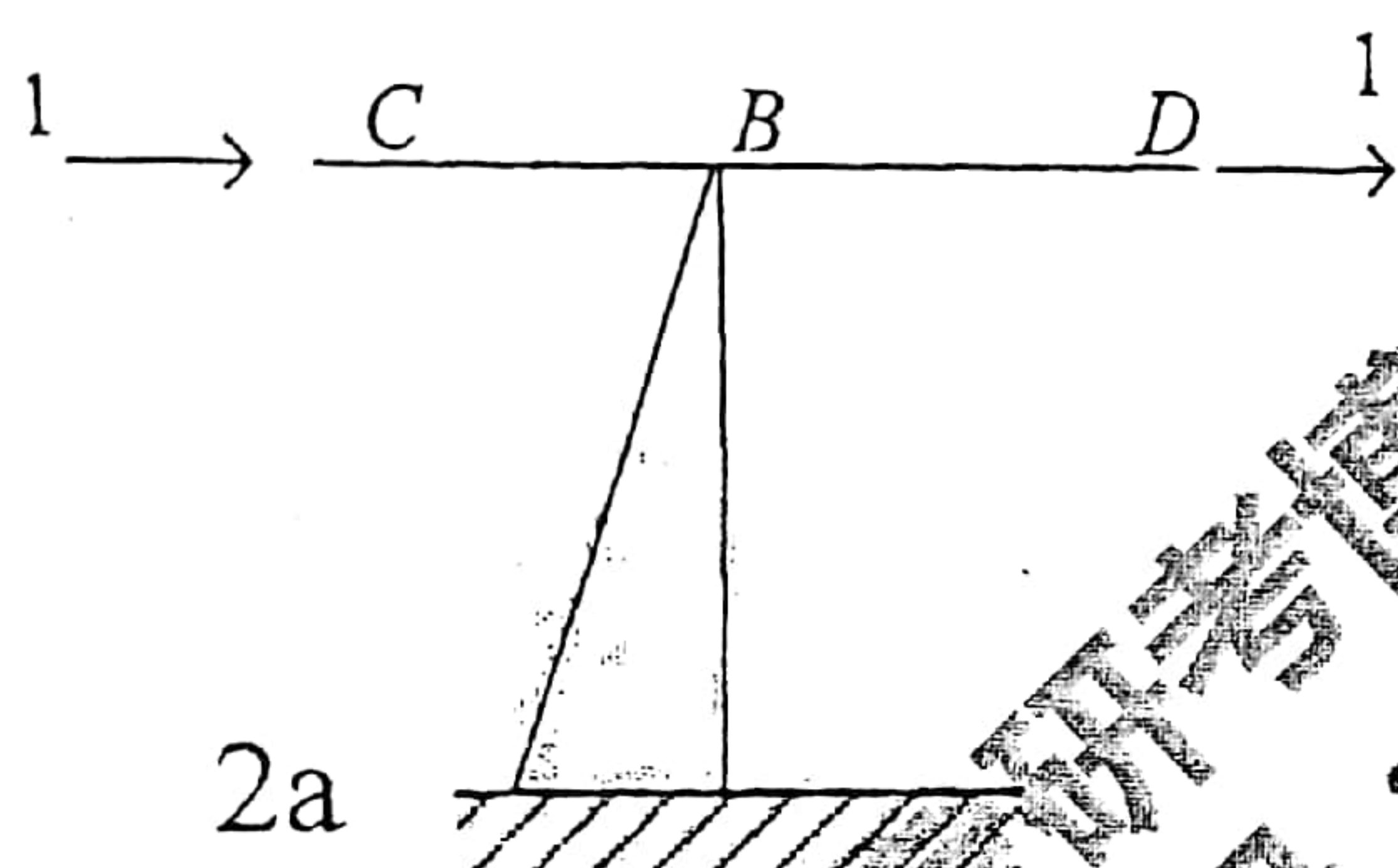
图二

对图一而言

$$\sum M_A = 0, \text{ 可知 } F_1 = 0$$

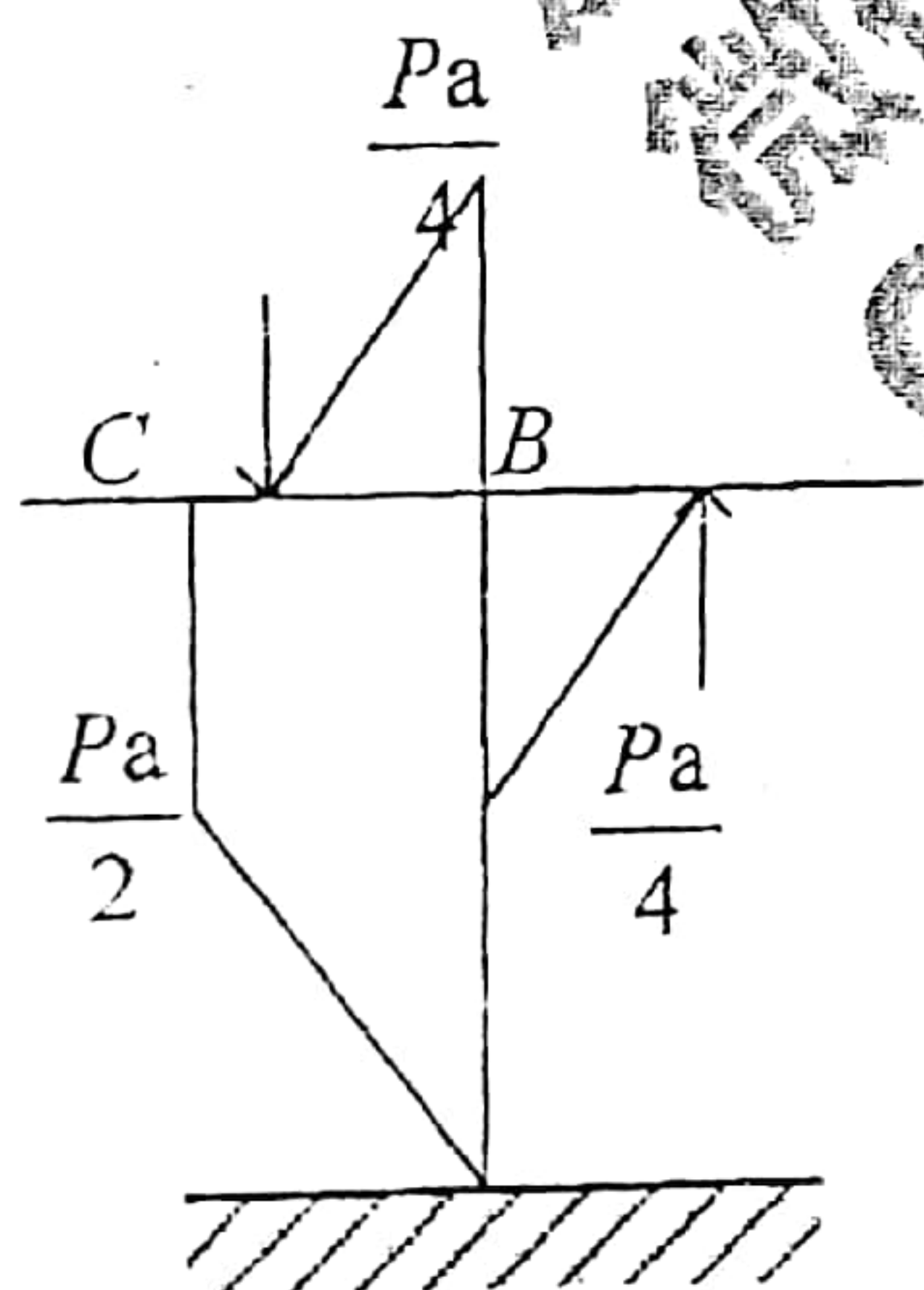
故求解约束力可用图二来做

用单位力法：在 $F_2=1$ 的作用下



$$\delta_{11} = \frac{1}{EI} \times \left(a \times 2a \times \frac{1}{2} \times \frac{2}{3} \times 2a \right) = \frac{4a^3}{3EI}$$

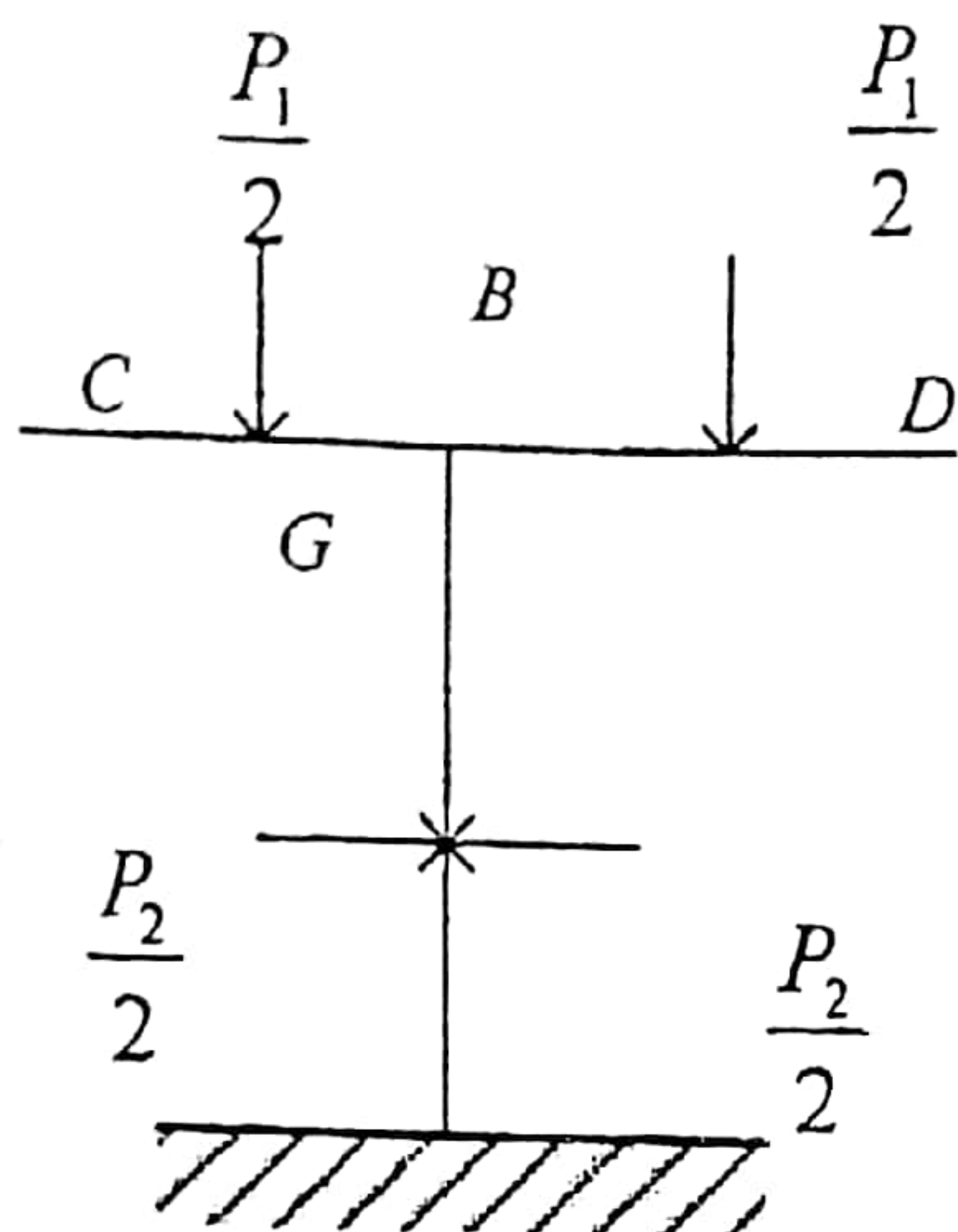
实际力作用下：



$$\Delta_{1P} = \int_0^a \frac{2x \frac{P}{2}}{EI} dx + \int_0^{2a} \frac{2x \left(Pa - \frac{P}{2} x \right)}{EI} dx = \frac{5Pa^3}{16EI}$$

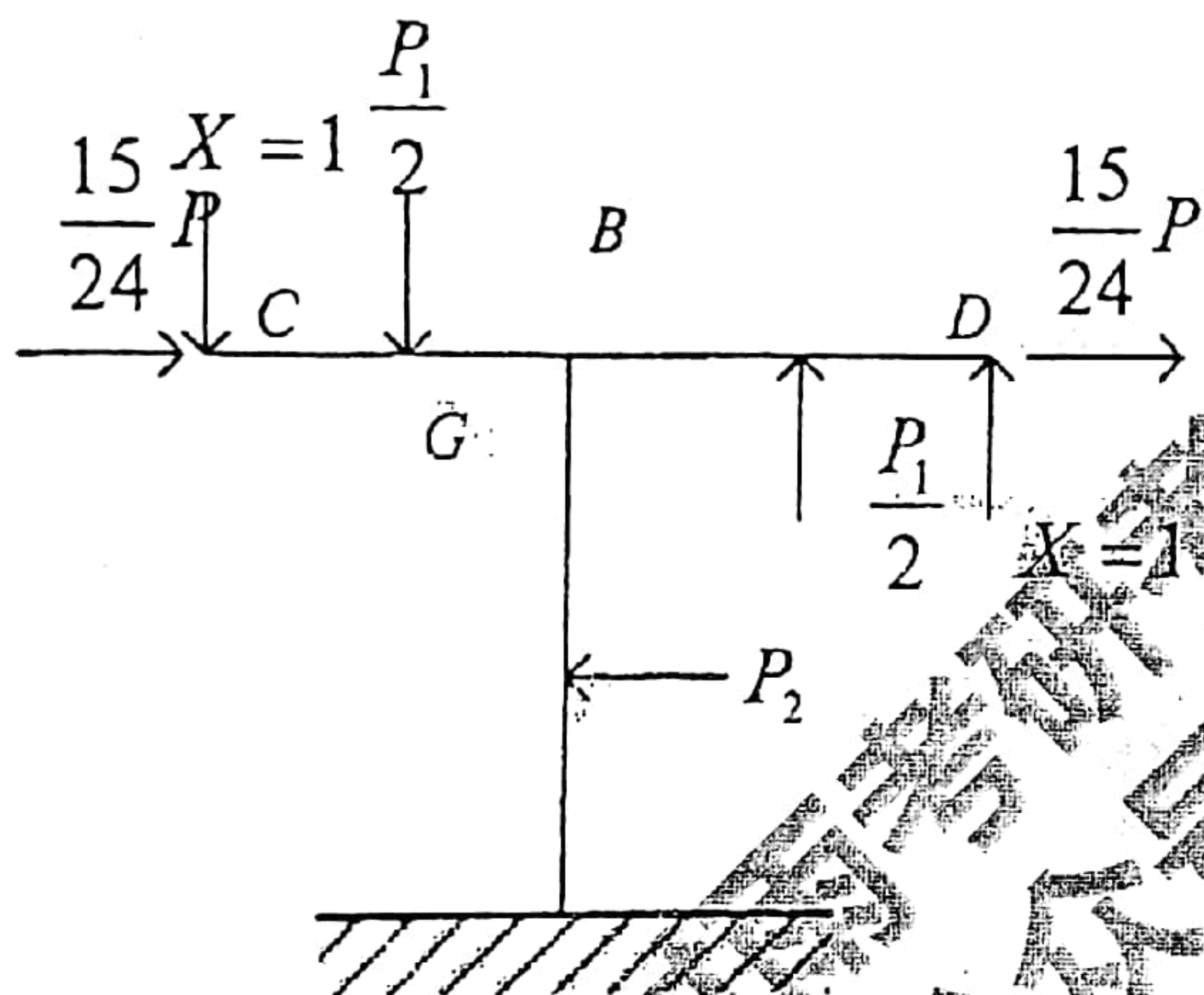
$$\text{故 } F_2 = -\frac{\Delta_{1P}}{\delta_{11}} = \frac{5}{8} P$$

(2)、在图一中



$$\omega_{c1} = \omega_{D1} = \frac{\frac{P}{2} \left(\frac{a}{2} \right)^3}{3EI} + \frac{\frac{P}{2} \left(\frac{a}{2} \right)^2}{2EI} \times \frac{a}{2} = \frac{5Pa^3}{64EI} (\downarrow)$$

在图二中，C、D 的竖直位移反向且相等



$$\text{故 } \omega_{c2} = \frac{21Pa}{256EI} (\downarrow) \quad \omega_{D2} = \frac{21Pa}{256EI} (\uparrow)$$

$$\text{综上, } \omega_c = \frac{5Pa}{64EI} + \frac{21Pa}{256EI} = \frac{41Pa}{256EI} \quad \omega_D = \frac{Pa}{256EI}$$

于是在 C、D 两端分别设一对方向相反的单位力 $X=1$

