

一、解: (1)、

$$\omega = \begin{cases} \frac{1}{EI}(ax^4 + Lx^3 + cx^2), & 0 \leq x \leq L \\ \frac{1}{EI}(dx^3 + cx^2 + ex), & L \leq x \leq 2L \end{cases}$$

$$\theta = \omega' = \begin{cases} \frac{1}{EI}(4ax^3 + 3Lx^2 + 2cx), & 0 \leq x \leq L \\ \frac{1}{EI}(3dx^2 + 2cx + e), & L \leq x \leq 2L \end{cases}$$

$$M(x) = -EI\omega'' = \begin{cases} -(12ax^2 + 6Lx + 2c), & 0 \leq x \leq L \\ -(6Lx + 2c), & L \leq x \leq 2L \end{cases}$$

$$(2)、F_s(x) = \frac{\partial M(x)}{\partial x} = \begin{cases} -(24ax + 6b), & 0 \leq x \leq L \\ -(6dL + 2c), & L \leq x \leq 2L \end{cases}$$

(3)、C 处存在集中外力偶, 则对于

$$\text{在 AC 段上: } M_1(L) = -(12aL^2 + 6bL + 2c)$$

$$\text{在 BC 段上: } M_2(L) = -(6dL + 2c)$$

因此  $M_1(L) \neq M_2(L)$ , 即

$$12aL^2 + 6bL + 2c \neq 0, \quad 2aL + b + d \neq 0$$

(4)、A 处存在外集中力, 则 AC 段:  $F_s(D) = -6b \neq 0$ , 即  $b \neq 0$

(5)、挠度:  $\omega_1(L) = \omega_2(L)$ , 即  $aL^4 + bL^3 + cL^2 = dL^3 + cL^2 + eL$ ,

$$\text{化简为 } aL^3 + (b - d)L^2 - e = 0$$

转角:  $\theta_1(L) = \theta_2(L)$ , 即:  $4aL^3 + 3bL^2 + 2cL = 3dL^2 + 2d + e$

$$\text{即: } 4aL^3 + 3(b - d)L^2 - e = 0$$

二、解: (1)、以  $0^\circ$  方向为 x 轴正方向,  $90^\circ$  方向为 y 轴正方向

$$\sigma_{45} = \frac{\sigma_0 + \sigma_{90}}{2} + \frac{\sigma_0 - \sigma_{90}}{2} \cos 90 - \tau_{xy} \sin 90$$

$$110 = \frac{180 + 100}{2} + \frac{180 - 100}{2} \times 0 - \tau_{xy}, \text{ 得 } \tau_{xy} = 30 \text{ MPa}$$

$$\left. \begin{array}{l} \sigma_{\max} \\ \sigma_{\min} \end{array} \right\} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left( \frac{\sigma_x - \sigma_y}{2} \right)^2 + \tau^2}$$

$$= \frac{180 + 100}{2} \pm \sqrt{\left( \frac{180 - 100}{2} \right)^2 + 20^2}$$

$$= \begin{cases} 190 \text{ MPa} \\ 90 \text{ MPa} \end{cases}$$

故  $\sigma_1 = 190 \text{ MPa}$ ,  $\sigma_2 = 90 \text{ MPa}$ ,  $\sigma_3 = 0$

(2)、最大切应力为:  $\tau_{\max} = \frac{\sigma_1 - \sigma_3}{2} = 95 \text{ MPa}$

$$\varepsilon_0 = \frac{1}{E} (\sigma_0 - \nu \sigma_{90}) = \frac{1}{200 \times 10^9} \times (-180 - 0.3 \times 100) \times 10^6 = 7.5 \times 10^{-4}$$

$$\varepsilon_{90} = \frac{1}{E} (\sigma_{90} - \nu \sigma_0) = -\frac{1}{200 \times 10^9} \times (100 - 0.3 \times 180) \times 10^6 = 2.3 \times 10^{-4}$$

$$\sigma_{135} = \frac{\sigma_0 + \sigma_{90}}{2} + \frac{\sigma_0 - \sigma_{90}}{2} \cos 270 - \tau_{xy} \sin 270 = 140 + 30 = 170 \text{ MPa}$$

$$\varepsilon_{45} = \frac{1}{E} (\sigma_{45} - \nu \sigma_{135}) = -\frac{1}{200 \times 10^9} \times (100 - 0.3 \times 170) \times 10^6 = 2.95 \times 10^{-4}$$

(4)、体积改变能密度:

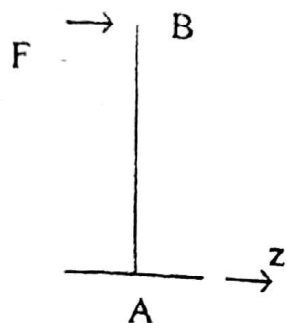
$$V_V = \frac{1 - 2\nu}{6E} (\sigma_1 + \sigma_2 + \sigma_3)^2 = \frac{1 - 2 \times 0.3}{6 \times 200 \times 10^9} \times (190 + 90 + 0)^2 \times 10^{12} = 2.61 \times 10^4 \text{ J/m}^3$$

形状改变能密度:

$$V_\sigma = \frac{1 + \nu}{6E} [(\sigma_1 - \sigma_2)^2 + (\sigma_1 - \sigma_3)^2 + (\sigma_2 - \sigma_3)^2]$$

$$= \frac{1 + 0.3}{6 \times 200 \times 10^9} \times [(190 - 90)^2 + (190 - 0)^2 + (90 - 0)^2] \times 10^{12} = 5.87 \times 10^4 \text{ J/m}^2$$

三、解: (1)、



$$F_y = F \sin \alpha = \frac{1}{2} F$$

$$F_z = F \cos \alpha = \frac{\sqrt{3}}{2} F$$

$$M_y = F_z a = \frac{\sqrt{3}}{2} Fa$$

$$M_z = F_y a = \frac{1}{2} Fa$$

$$(2)、I_y = \frac{\pi d^4}{64} - \frac{2b \cdot b^3}{12} = \frac{23b^4}{6}$$

$$I_z = \frac{\pi d^4}{64} - \frac{b \cdot (2b)^3}{12} = \frac{10b^4}{3}$$

中性轴方程为:  $\frac{M_y}{I_y} z + \frac{M_z}{I_z} y = 0$

代入数值,  $-\frac{\frac{\sqrt{3}}{2} Fa}{\frac{23b^4}{6}} z + \frac{\frac{1}{2} Fa}{\frac{10b^4}{3}} y = 0$

设  $\theta$  为中性轴与  $y$  轴的夹角, 则  $\tan \theta = \frac{z}{y} = \frac{23}{20\sqrt{3}}$ , 故  $\theta = \arctan \frac{23}{20\sqrt{3}}$

$$(3)、\sigma_{\max} = \frac{M_y}{I_y} \cdot \frac{d}{2} + \frac{M_z}{I_z} \cdot \frac{d}{2} = \frac{\frac{\sqrt{3}}{2} Fa}{\frac{23}{6} b^4} \cdot \frac{d}{2} + \frac{\frac{1}{2} Fa}{\frac{10}{3} b^4} \cdot \frac{d}{2} = \frac{Fa}{b^3 \pi^{\frac{1}{4}}} \left( \frac{6\sqrt{3}}{23} + \frac{3}{10} \right)$$

$$(4)、\sigma_{II} = \frac{M_y}{I_y} \cdot \frac{b}{2} + \frac{M_z}{I_z} \cdot b = \frac{\frac{\sqrt{3}}{2} Fa}{\frac{23}{6} b^4} \cdot \frac{b}{2} + \frac{\frac{1}{2} Fa}{\frac{10}{3} b^4} \cdot b = \frac{30\sqrt{3} + 69}{460} \frac{Fa}{b^3}$$

(5)、

$$\sigma_{\max} = \frac{M_y}{I_y} \cdot \frac{d}{2} + \frac{M_z}{I_z} \cdot \frac{d}{2} = \frac{Facos\alpha}{\frac{23}{6} b^3 \pi^{\frac{1}{4}}} \cdot \frac{1}{2} + \frac{Fasin\alpha}{\frac{10}{3} b^3 \pi^{\frac{1}{4}}} \cdot \frac{1}{2} = \frac{Fa}{b^3 \pi^{\frac{1}{4}}} \left( \frac{12}{23} \cos \alpha + \frac{3}{20} \sin \alpha \right)$$

当  $\sigma_z = \sigma_y$  时, 最大弯曲正应力取得极小值,

$$\text{故 } \frac{Facos\alpha}{\frac{23}{6} b^3 \pi^{\frac{1}{4}}} \cdot \frac{1}{2} = \frac{Fasin\alpha}{\frac{10}{3} b^3 \pi^{\frac{1}{4}}} \cdot \frac{1}{2}, \tan \alpha = \frac{20}{23}, \text{ 解得 } \alpha = \arctan \frac{20}{23}$$

四、解: (1)、 $M(x) = -Fx$

$$V_{\text{ab}} = \int_0^L \frac{M^2(x)}{2EI} dx = \int_0^L \frac{(Fx)^2}{2EI} dx = \frac{F^2 L^3}{4Eb^4}$$

(2)、截面为矩形, 故  $\alpha = \frac{6}{5}$

$$V_{\text{as}} = \int_0^L \alpha \frac{F^2(x)}{2GA} dx = \alpha \frac{F^2 L}{2GA} = \frac{3F^2 L}{8Eb^2}$$

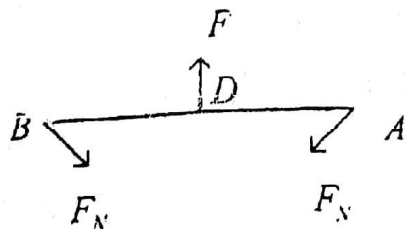
$$(3)、\frac{V_{\text{ab}}}{V_{\text{as}}} = \frac{\frac{F^2 L^3}{4Eb^4}}{\frac{3F^2 L}{8Eb^2}} = \frac{2}{3} \left( \frac{b}{L} \right)^2$$

$$\text{当 } \frac{b}{L} = \frac{1}{10} \text{ 时, } \frac{V_{\text{ab}}}{V_{\text{as}}} = \frac{3}{200}$$

$$\text{故 } V_{\text{as}} \text{ 所占百分比为 } \frac{\frac{3}{200}}{\frac{3}{200} + 1} = 1.48\%$$

五、请参照孙训服的课本第二册例 3-17

解: 正三角形的内力为超静定, 利用对称性, 截开 A、B 两点, 只存在对称的力。



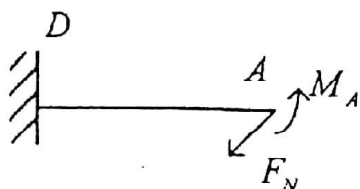
即只可能有轴力和弯矩, 由于 AB 段结构和荷载都关于 CD 轴对称, 故轴向截面

AB 上的轴力和弯矩相等，其轴力可由静力平衡方程求得，即  $\sum F_y = 0$ ，

$$2F_N \sin 60^\circ - F = 0, \quad F_N = \frac{F}{\sqrt{3}}$$

故仅有  $M_A$  未知，由于 A 截面的转角为 0，（反对称的位移为 0）

将结构化简为



$$M(x) = -\frac{F}{2}x + M_A$$

$$\text{故 } \theta = \int_0^a \frac{M_A - \frac{F}{2}x}{EI} dx = 0$$

$$\text{解得 } M_A = \frac{Fa}{4}$$

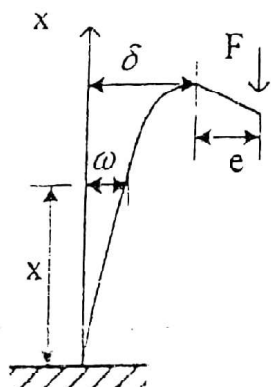
$$\text{所以 } M_D = M(a) = -\frac{Fa}{4}$$

$$\text{故 D 处的内力为 } M_D = -\frac{Fa}{4}$$

$$F_N = \frac{F}{2}$$

$$F_N = \frac{F}{\sqrt{3}} \cos 60^\circ = -\frac{\sqrt{3}}{6} F \quad (\text{压})$$

六、这道题严重被所谓的网络答案误导，而且在 15 年考研又考出类似的题，在刘鸿文的材料中绝对找不到，某些商家还声称指定用书只有刘鸿文。其实是孙训方下册，4-6 的课后题，4 章为《压杆稳定问题的进一步研究》，很多同学看到这里误以为超纲而没有仔细阅读，当年我考试也是被所谓的答案害惨了。



$$\text{解: } M(x) = -F(e + \delta - \omega)$$

$$EI\omega'' = -M(x) = F(e + \delta - \omega)$$

设  $\frac{F_{cr}}{EI} = k^2$ , 则  $\omega'' + k^2\omega = k^2(e + \delta)$

$$\omega = A\sin kx + B\cos kx + e + \delta = \delta$$

$$\omega' = Ak\cos kx - Bk\sin kx$$

当  $x = 0$  时,  $\omega = 0$  即  $B + e + \delta = 0$ ,  $B = -e - \delta$

当  $\omega' = 0$ , 即  $Ak = 0$

当  $x = L$  时,  $\omega = \delta$ , 即  $-(\delta + e)\cos kL + e + \delta = \delta$ ,

$$-(\delta + e) = \frac{e}{\cos kL}$$

$$\delta = \frac{1 - \cos kL}{\cos kL} e$$

故 B 截面的最大应力  $\sigma = \frac{F}{A} + \frac{M}{W}$

$$M = F(e + \delta) = \frac{Fe}{\cos kL}, \text{ 故 } \sigma = \frac{F}{A} + \frac{Fe}{\cos kL W}$$

临界力表达式  $\sigma_{st} = \frac{F}{A} + \frac{Fe}{W \cos\left(\sqrt{\frac{F}{EI}} L\right)} \leq [\sigma]$

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