

二〇〇八年答案解析

一、看到这幅图大家可以想到什么？

虽然官方说以刘鸿文教材为主，但刘鸿文版本的书弯矩图都是默认向上为正方向，而孙训方则相反，自己补脑，所以这俩门教材可以说是缺一不可。

解：(1)、二次函数 $M(x) = \frac{Px^2}{2}, 0 \leq x \leq a$

已知 (a, qa^2) 和 $(2a, -qa^2)$ ，则直线方程为

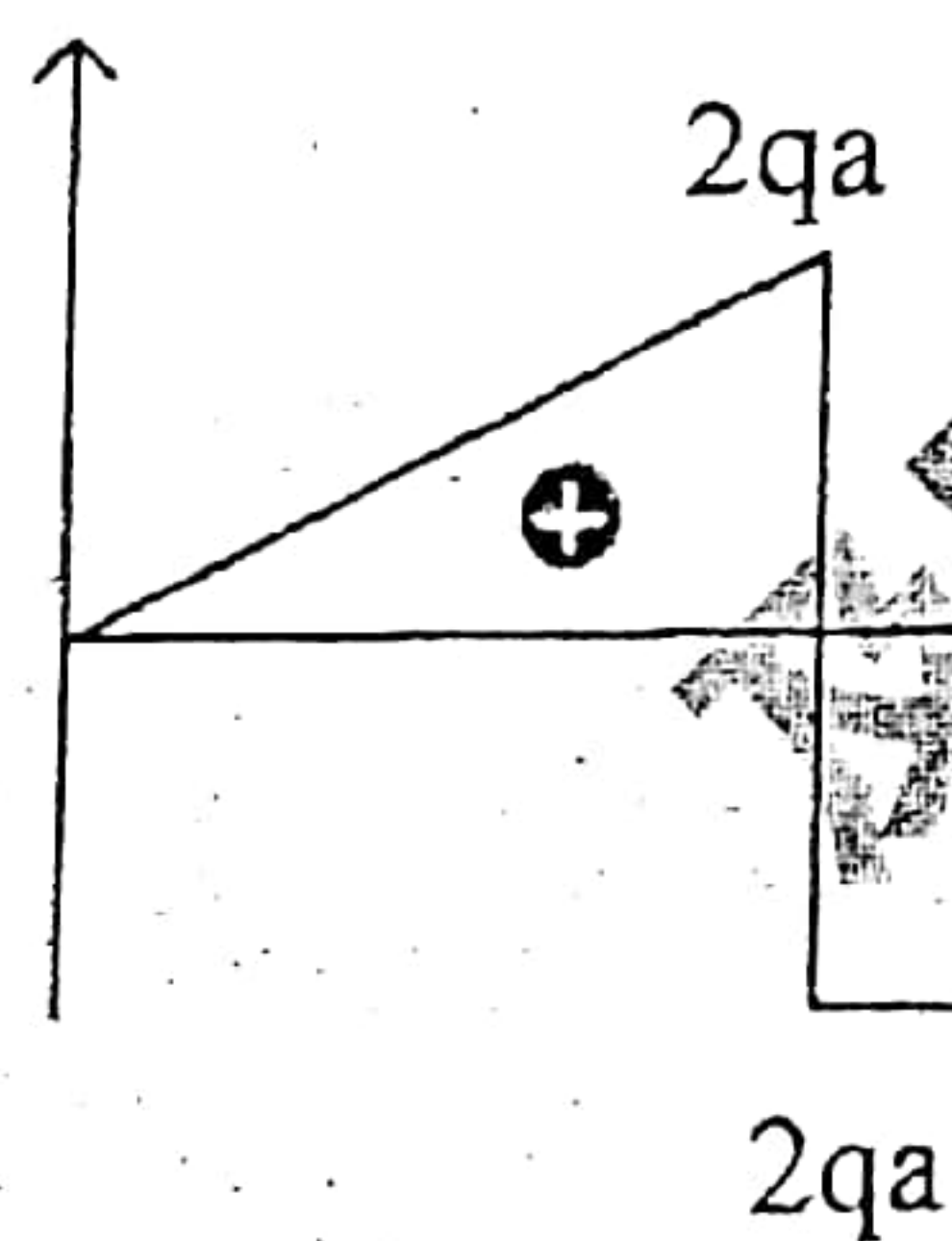
$$\frac{M - qa^2}{x - a} = \frac{qa^2 + qa^2}{a - 2a}, \quad M(x) = -2qa(x - a) + qa^2$$

$$\text{即 } M(x) = \begin{cases} qx^2 & 0 \leq x \leq a \\ -2qa(x - a) + qa^2 & a \leq x \leq 2a \end{cases}$$

$$(2)、F(x) = \frac{\partial M(x)}{\partial x} = \begin{cases} 2qx \\ -2qa \end{cases}$$

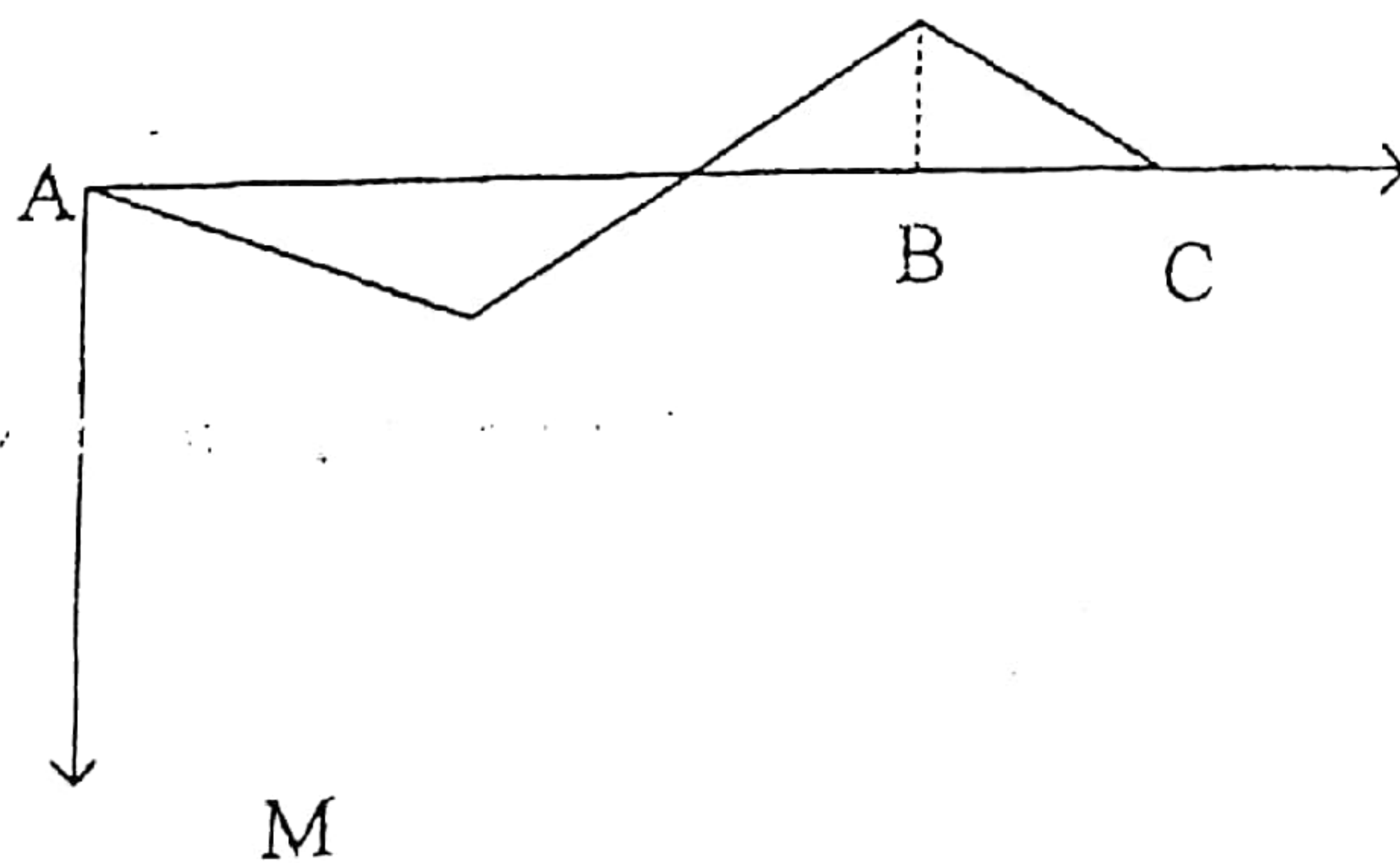
故

Fs图

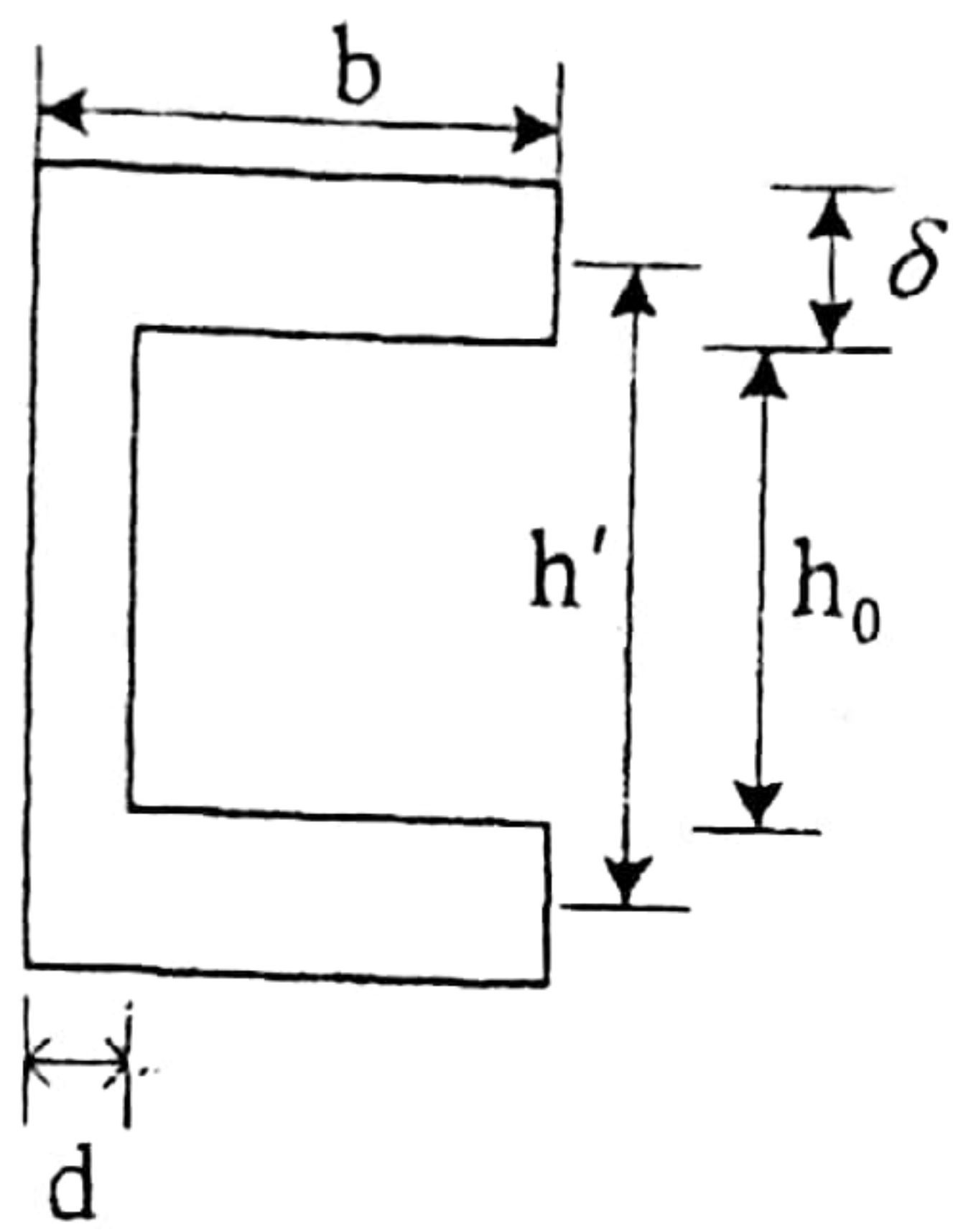


二、解、孙训方书上（第二册）的题

M图



(2)、

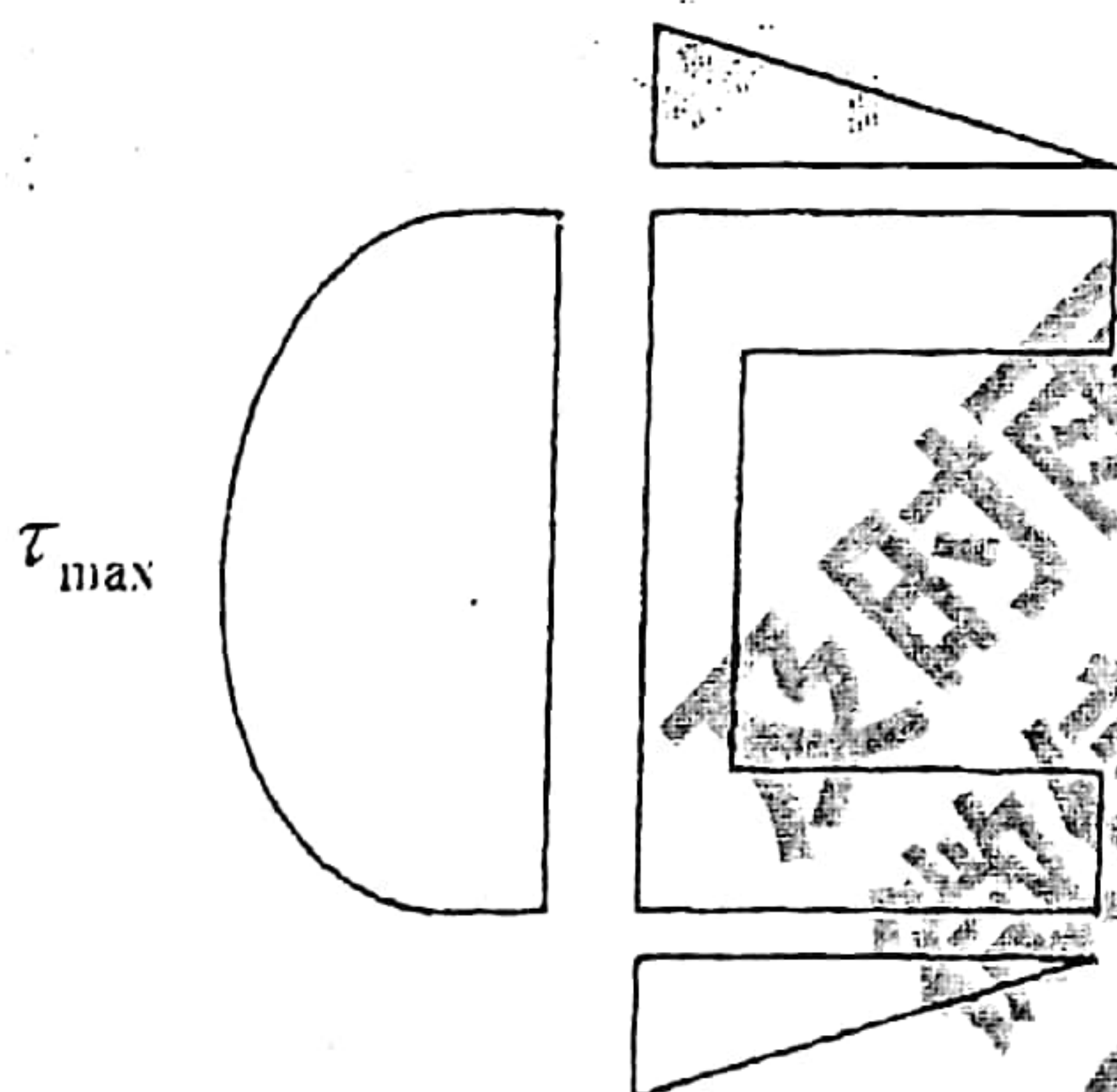


翼缘: $\tau = \frac{F_s h' \delta u}{2 I_z \delta} = \frac{F_s h' u}{2 I_z}$

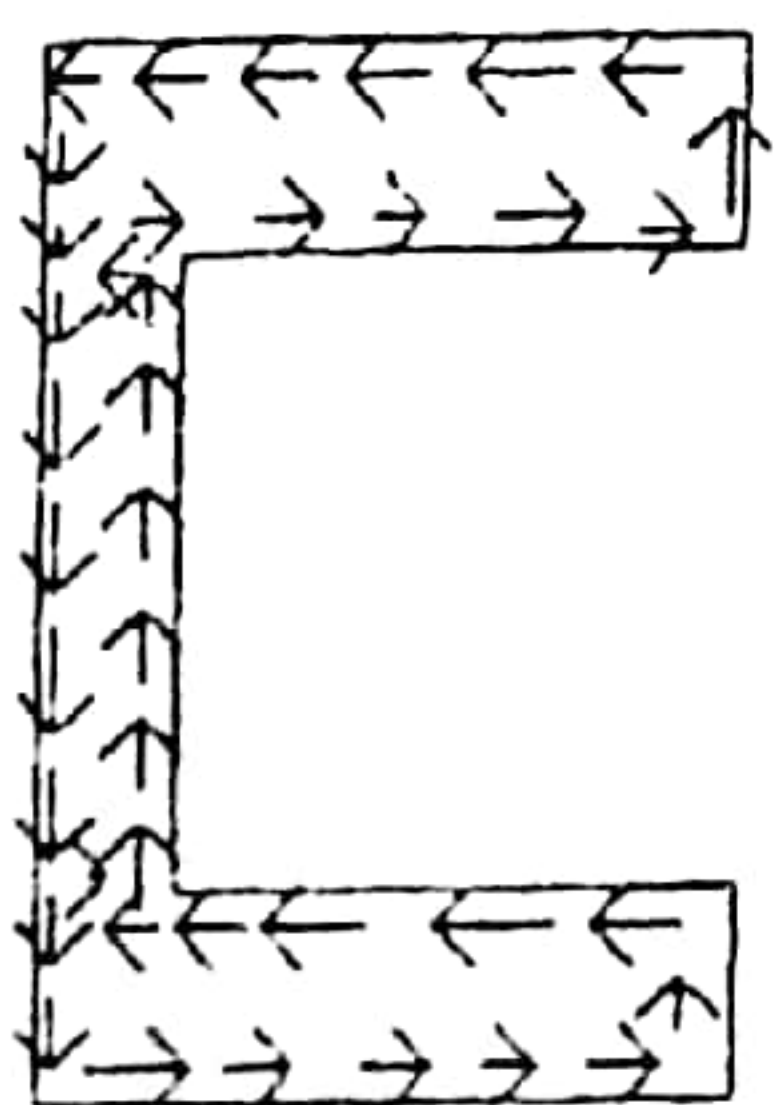
腹板: $\tau = \frac{F_s h' b}{2 I_z} + \frac{F_s}{I_z d} \left(\frac{d}{2} \left(\frac{h^2}{4} - y^2 \right) \right)$

$\tau_{max} = \frac{F_s h' b}{2 I_z}$

$\tau_{max} = \frac{F_s h' b}{2 I_z} + \frac{F_s h^2}{8 I_z}$



(3)、最大切应力位于腹杆中部



(箭头画的太丑了)

(4)、存在，腹杆中部点

三、解：(1)

$\epsilon_0 = \frac{1}{E} (\sigma_0 - \nu \sigma_{90})$

$$\varepsilon_{90} = \frac{1}{E}(\sigma_{90} - \nu\sigma_0)$$

$$\text{得到 } \sigma_0 = \frac{E(\varepsilon_0 + \nu\varepsilon_{90})}{1-\nu^2} = 82.64 \text{ MPa}$$

$$\sigma_{90} = \frac{E(\varepsilon_{90} + \nu\varepsilon_0)}{1-\nu^2} = 8.79 \text{ MPa}$$

$$\sigma_{45} = \frac{\sigma_0 + \sigma_{90}}{2} + \frac{\sigma_0 - \sigma_{90}}{2} \cos 90 - \tau_{xy} \sin 90$$

$$\sigma_{45} = \frac{\sigma_0 + \sigma_{90}}{2} + \frac{\sigma_0 - \sigma_{90}}{2} \cos 90 - \tau_{xy} \sin 90$$

$$\sigma_{135} = \frac{\sigma_0 + \sigma_{90}}{2} + \frac{\sigma_0 - \sigma_{90}}{2} \cos 270 - \tau_{xy} \sin 270$$

$$\varepsilon_{45} = \frac{1}{E}(\sigma_{45} - \nu\sigma_{135})$$

$$\text{联立上式, } \tau_{xy} = \frac{\sigma_0 + (1-\nu)\sigma_{90}}{2(1+\nu)} = 15.38 \text{ MPa}$$

$$\text{故 } \sigma_{45} = \frac{\sigma_0 + \sigma_{90}}{2} - \tau_{xy} = 61.11 \text{ MPa}$$

$$(2) \left. \begin{array}{l} \sigma_{\max} \\ \sigma_{\min} \end{array} \right\} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2} \right)^2 + \tau^2} = 45.72 \pm 40 = \begin{cases} 85.72 \text{ MPa} \\ 5.72 \text{ MPa} \end{cases}$$

$$\text{故 } \sigma_1 = 85.72 \text{ MPa}, \sigma_2 = 0, \sigma_3 = 5.72 \text{ MPa}$$

$$\tau_{\max} = \frac{\sigma_1 - \sigma_3}{2} = 42.86 \text{ MPa}$$

四、解：(1) $w, F_y = F \cos \alpha$

$$F_z = -F \sin \alpha$$

$$\text{梁的最大弯矩正应力 } \sigma_{\max} = \frac{M_y}{W_y} + \frac{M_z}{W_z} = \frac{6FL \sin \alpha}{hb^2} + \frac{6FL \cos \alpha}{bh^2}$$

$$(2)、\text{中性轴方程: } 0 = \frac{M_y z}{I_y} + \frac{M_z y}{I_z}$$

$$\text{即: } 0 = \frac{FL \sin \alpha}{\frac{1}{12}hb^3} - \frac{FL \cos \alpha}{\frac{1}{12}bh^3}$$

$$\text{化简得: } \frac{\sin \alpha}{b^2}z + \frac{\cos \alpha}{I_z}y = 0$$

(3)、上表面只有 σ_z , 侧面只有 σ_y

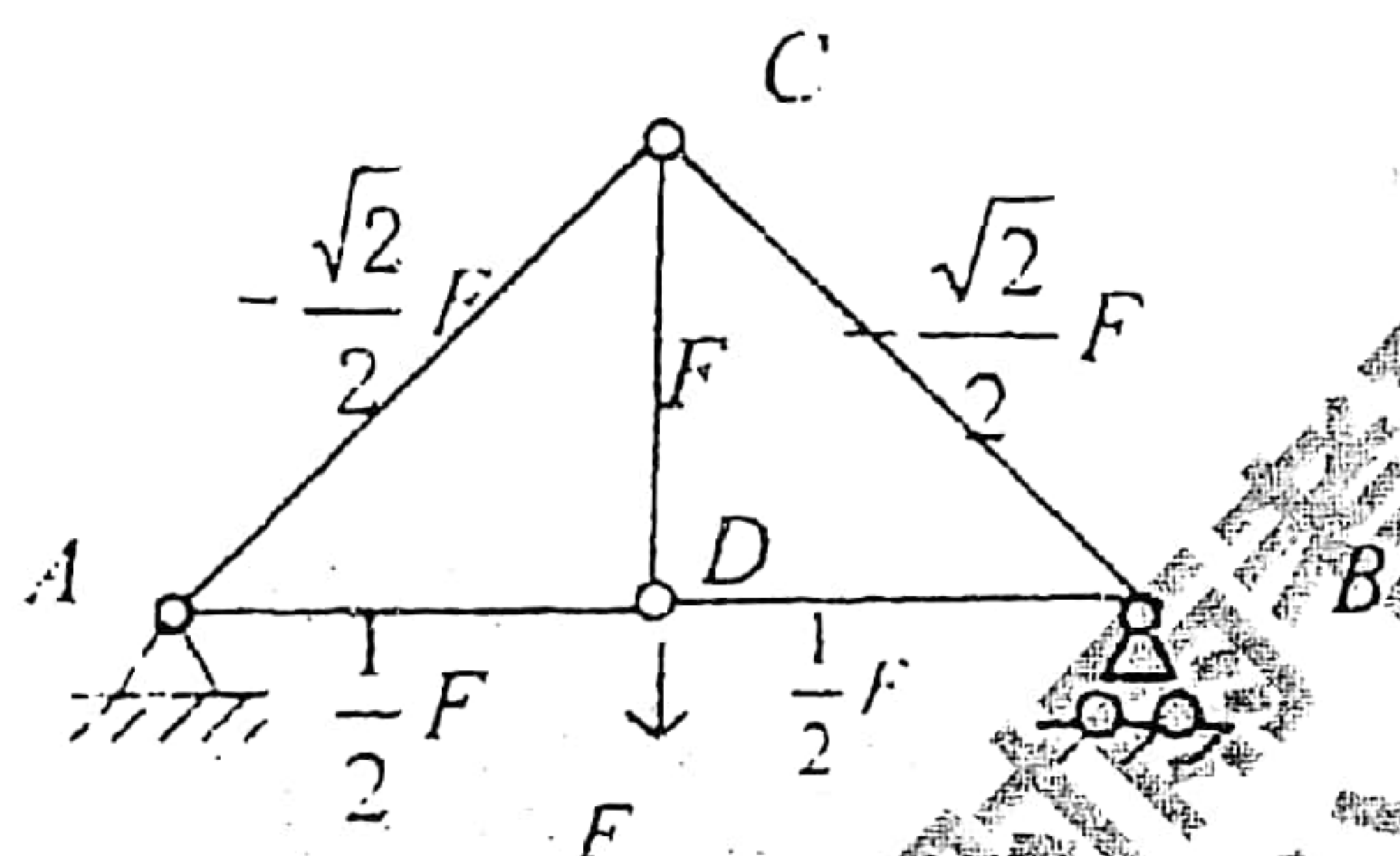
$$\text{故 } \varepsilon_1 = \frac{\sigma_z}{E} = \frac{1}{E} \cdot \frac{Mz}{W_z} = \frac{6FL \cos \alpha}{bh^2 E}$$

$$\varepsilon_2 = \frac{\sigma_y}{E} = \frac{1}{E} \cdot \frac{My}{W_y} = \frac{6FL \sin \alpha}{hb^2 E}$$

$$\text{联立上俩式, 可知 } F = \frac{bhE}{6L} \sqrt{(h\varepsilon_1)^2 + (b\varepsilon_2)^2}$$

$$\alpha = \arctan\left(\frac{b\varepsilon_2}{h\varepsilon_1}\right)$$

五、解：受力分析



$$F_{NAD} = F_{NDB} = \frac{F}{2}$$

$$F_{NCD} = F$$

$$F_{NAC} = F_{NBC} = -\frac{\sqrt{2}F}{2}$$

$$\text{由能量法: } \Delta_D = \int \frac{F_1}{EA} \frac{\partial F_1}{\partial F} ds = \frac{F}{EA} \cdot \frac{1}{2} \cdot 2 + \frac{\frac{\sqrt{2}}{2} \cdot \frac{\sqrt{2}}{2} \cdot F \cdot \sqrt{2}a}{EA} \times 2 + \frac{Fa}{EA} = \left(\frac{3}{2} + \sqrt{2}\right) \frac{Fa}{EA}$$

$$(2)、\frac{F_{NCD}}{F_{ND}} = 2, F_{NCD} \leq [\sigma] \cdot A_{CD}, F_{NAD} \leq [\sigma] \cdot A_{AD}, \text{故 } \frac{A_{CD}}{A_{AD}} = 2$$

$$(3)、AC \text{ 杆较小的惯性矩 } I_{\min} = \frac{2b \cdot b^3}{12} = \frac{b^4}{6}$$

$$i_{\min} = \sqrt{\frac{I_{\min}}{A}} = \frac{b}{2\sqrt{3}}$$

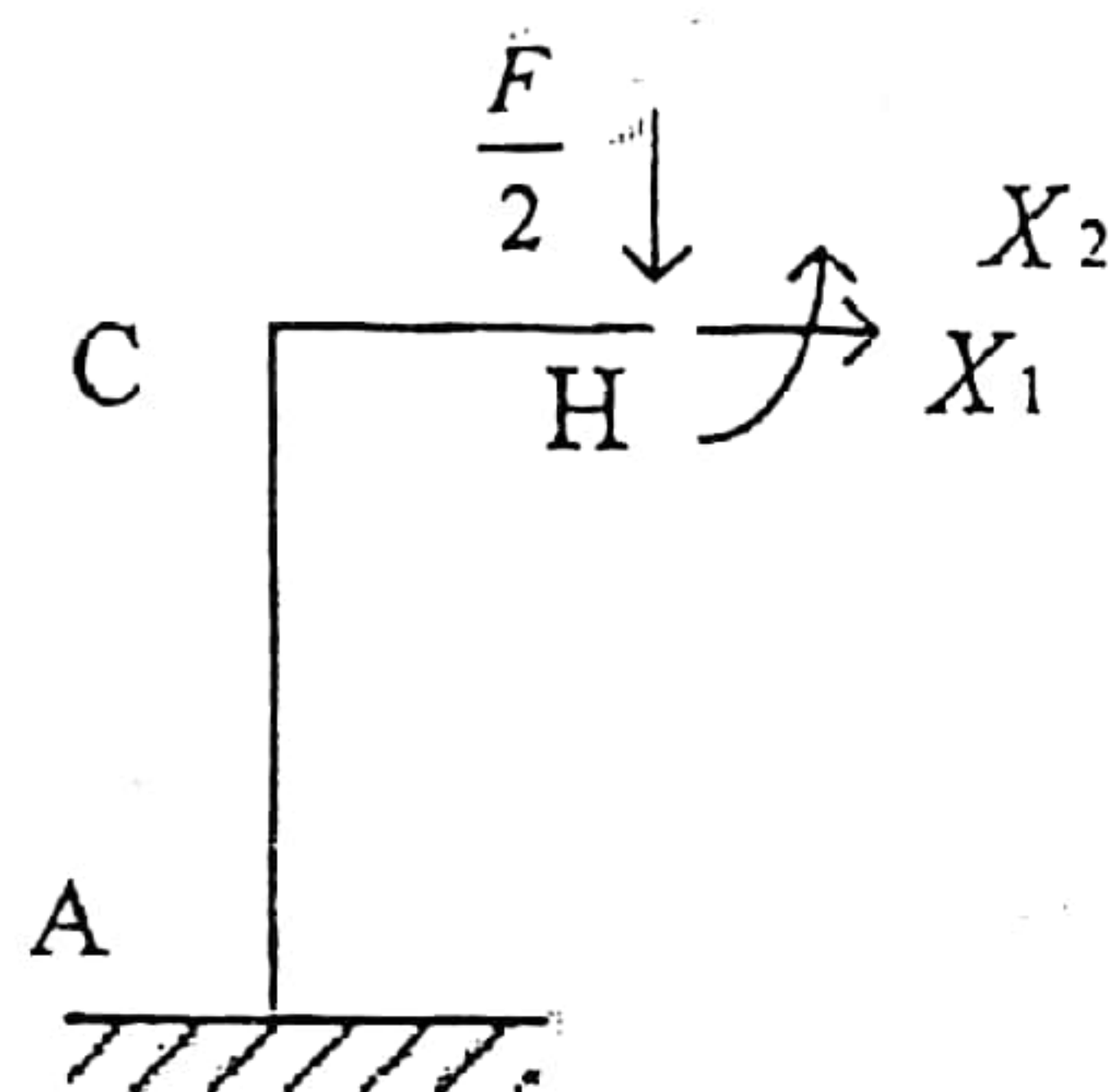
$$\lambda = \frac{\mu L}{i_{\min}} = \frac{2\sqrt{3}\sqrt{2}a}{b} = \frac{2\sqrt{6}a}{b} > \frac{4a}{b}$$

$$\text{属于大柔度杆件, } F_{cr} = \frac{\pi^2 EI}{(\mu L)^2} = \frac{\pi^2 E \frac{2b^4}{12}}{\sqrt{2}} = \frac{\pi^2 Eb^4}{12a^2}$$

六、解：结构与荷载左右对称，则对称截面 H 处只有存在正对称力，即 M 和 F_n

(1)、分析左半部分，则 BH 段的剪力 $F_s = \frac{F}{2}$

(2)、



$$\text{BH 段: } M(x_1) = -\frac{F}{2}x_1 + X_2, \quad \frac{\partial M(x_1)}{\partial X_1} = 0, \quad \frac{\partial M(x_1)}{\partial X_2} = 1$$

$$\text{BA 段: } M(x_2) = -\frac{F}{2}a + X_2 - X_1 \cdot x_2, \quad \frac{\partial M(x_2)}{\partial X_1} = -x_2, \quad \frac{\partial M(x_2)}{\partial X_2} = 1$$

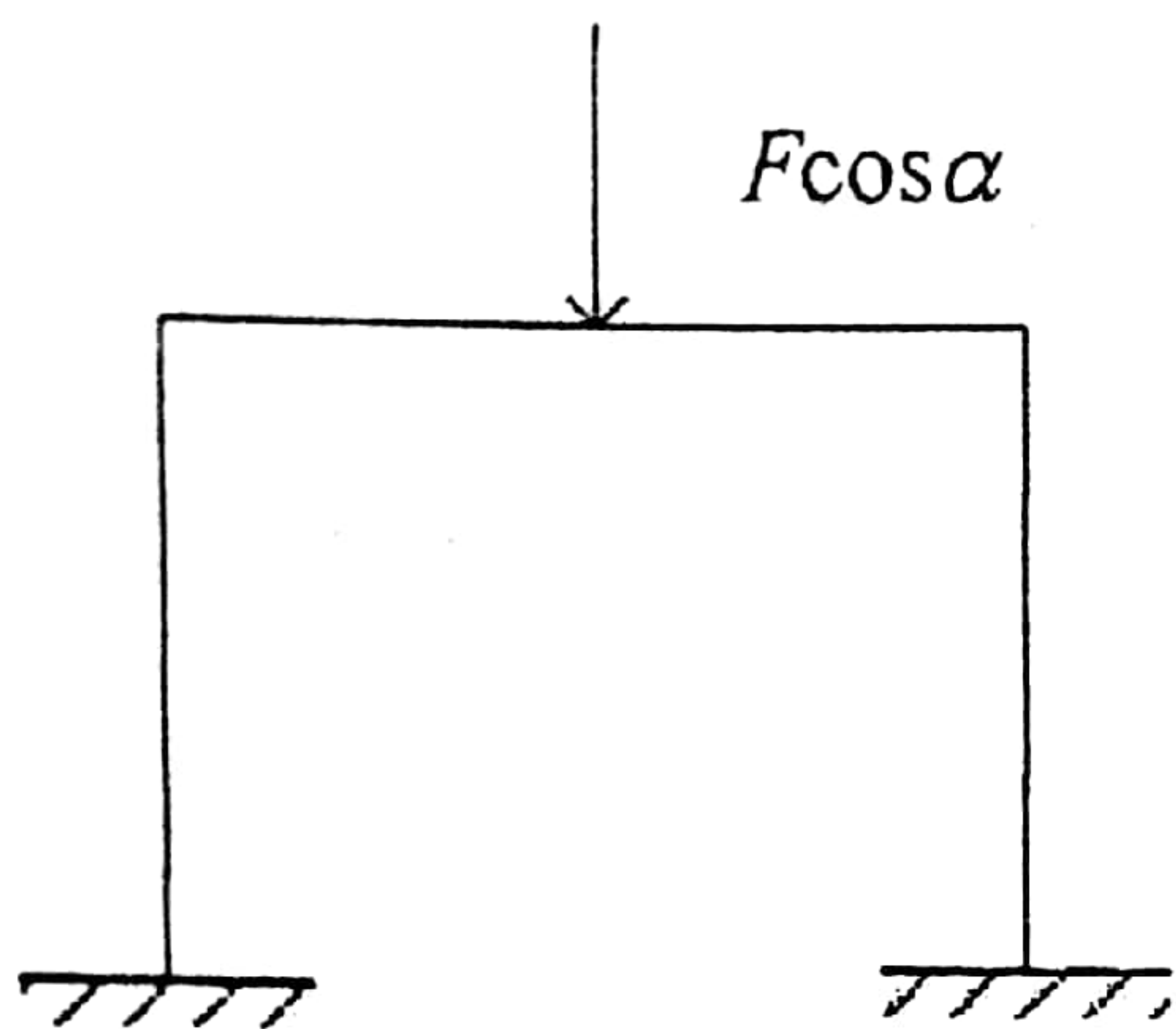
$$\begin{aligned} \text{正对称 } \theta_H &= \int_0^a \frac{-\frac{F}{2}x_1 + X_2}{EI} dx_1 + \int_0^{2a} \frac{-\frac{F}{2}a + X_2 - X_1 \cdot x_2}{EI} dx_2 \\ &= -\frac{5Fa^2}{4EI} + \frac{3X_2}{EI} - \frac{2X_1a}{EI} = 0 \end{aligned}$$

$$\begin{aligned} \Delta H_x &= \int_0^{2a} \frac{-\frac{F}{2}a + X_2 - X_1 \cdot x_2}{EI} \cdot x_2 dx_2 \\ &= \frac{4Fa^3}{4EI} - \frac{4X_2a^2}{2EI} + \frac{8X_1a^3}{3EI} = 0 \end{aligned}$$

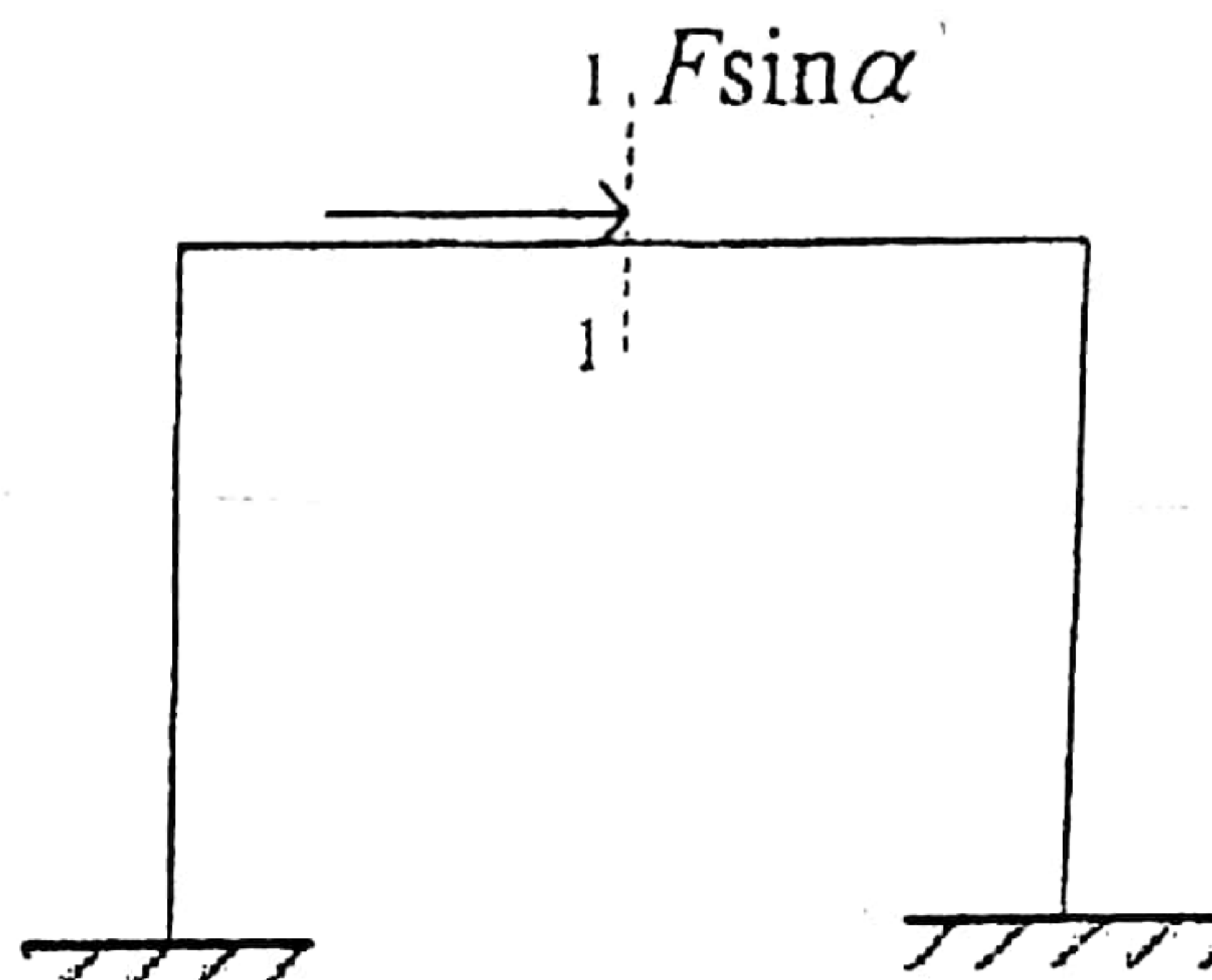
$$\text{解关于 } x_1, x_2 \text{ 的方程组, 得到 } \begin{cases} X_1 = -\frac{1}{8}F \\ X_2 = -\frac{1}{3}Fa \end{cases}$$

即 $F_N = -\frac{1}{8}F$ (受压), $M = \frac{1}{3}Fa(\downarrow)$

(3)、结构分为两个基本结构



图一



图二

其中，图一： $F_{N1} = -\frac{1}{8}F\cos\alpha$ （受压）

图二：结构反对称

$$F_{N_2} = \frac{1}{2} F \sin \alpha \quad (\text{受拉})$$

$$\text{故 } F_N = F_{N1} + F_{N2} = \frac{1}{2} F \sin \alpha - \frac{1}{8} F \cos \alpha$$

这道题的模式基本固定，仍是正反对称类型，同时能量法还没有上升至单位力法去做（这里提及单位力法建议掌握，因为他相比能量法有很多的优点）

二：结构反对称

$$F_{N2} = \frac{1}{2} F \sin \alpha \quad (\text{受拉})$$
$$F_1 + F_{N2} = \frac{1}{2} F \sin \alpha - \frac{1}{8} F \cos \alpha$$

模式基本固定，仍是正反对称类型，同时能量法还涉及单位力法建议掌握，因为他相比能量法有很