Lecture 11: Model-Reference Adaptive Systems

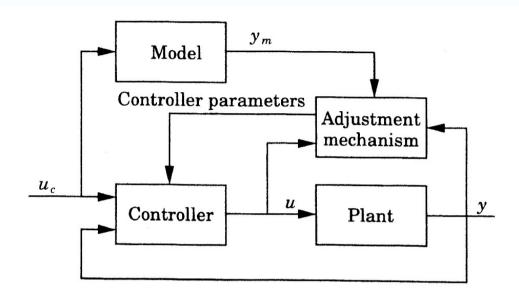


Figure 1.18 Block diagram of a model-reference adaptive system (MRAS).

Given:
$$y(t) = G_{\theta}(p) \, u(t), \qquad y_m(t) = G_m(p) \, u_c(t),$$

Find:
$$u(t)=-rac{S_{\hat{ heta}}(p)}{R_{\hat{ heta}}(p)}\,y(t)+rac{T_{\hat{ heta}}(p)}{R_{\hat{ heta}}(p)}\,u_c(t), \quad rac{d}{dt}\hat{ heta}=\ldots$$

Consider a stable single input single output (SISO) system

$$y(t) = k \cdot G(p) \Big(u(t) \Big)$$

where

- y(t) is the system output,
- G(s) is a known stable transfer function,
- u(t) is a control input,
- k is a constant unknown gain.

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- G(s) is a known stable transfer function,
- u(t) is a control input,
- k is a constant unknown gain.

The problem is find a controller $u(t) = \frac{T(p)}{R(p)} \frac{\mathbf{u_c(t)}}{\mathbf{v_c(t)}}$ to follow

$$y_m(t) = G_m(p) \left(\mathbf{u_c(t)} \right) = \mathbf{k_0} \cdot G(p) \left(\mathbf{u_c(t)} \right),$$

where k_0 is a given constant gain.

If k were known we can solve the problem

$$y(t) = k \cdot G(p) \left(\mathbf{u}(t) \right) \longrightarrow y_m(t) = \mathbf{k_0} \cdot G(p) \left(\mathbf{u_c}(t) \right)$$

using the simple proportional controller

$$u(t) = \theta \cdot u_c(t)$$

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$$\theta = \frac{k_0}{k}$$

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This is, indeed, works, because if θ is chosen as

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then

$$y(t) = k \cdot G(p) \left(\theta \cdot u_c(t)\right) = k \cdot G(p) \left(\frac{k_0}{k} \cdot u_c(t)\right) = y_m(t)$$

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Let us consider the error between the real and simulated outputs

$$e(t, \theta) = y(t) - y_m(t) = k \cdot G(p) \left(\theta(t) \cdot u_c(t)\right) - k_0 \cdot G(p) \left(u_c(t)\right)$$

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Let us update $\theta(t)$ so that $e(t, \theta)$ is getting smaller.

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Let us update $\theta(t)$ so that $e(t, \theta)$ is getting smaller.

Consider the loss function that measures the size of $e(t, \theta)$

$$\mathcal{J}(t,\theta) = |e(t,\theta)|^2$$

Its time derivative is given by (chain rule)

$$\left[rac{d}{dt}\mathcal{J} = \left[rac{\partial}{\partial t}\mathcal{J}
ight] + \left[rac{\partial}{\partial heta}\mathcal{J}
ight] \cdot rac{d}{dt} heta$$

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Consider the function that measures the size of $e(t, \theta)$

$$\mathcal{J}(t,\theta) = |e(t,\theta)|^2$$

Its time derivative should be made negative:

$$\frac{d}{dt}\mathcal{J} = \dots + \left[\frac{\partial}{\partial \theta}\mathcal{J}\right] \cdot \frac{d}{dt}\theta \quad \Rightarrow \quad \frac{d}{dt}\theta = -\gamma \cdot \left[\frac{\partial}{\partial \theta}\mathcal{J}\right]$$

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Consider the function that measures the size of $e(t, \theta)$

$$\mathcal{J}(t,\theta) = |e(t,\theta)|^2$$

Its time derivative should be made negative:

$$\left[rac{d}{dt} \mathcal{J} = \cdots + \left[2 \, e \, rac{\partial}{\partial heta} e
ight] \cdot rac{d}{dt} heta \quad \Rightarrow \quad \left[rac{d}{dt} heta = - \gamma \cdot \left[2 \, e \, rac{\partial}{\partial heta} e
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ight]$$

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Computing the partial derivative of e wrt θ we have

$$\frac{\partial}{\partial \theta} e = \frac{\partial}{\partial \theta} \left[k \cdot G(p) \left(\theta(t) \cdot u_c(t) \right) \right] = k \cdot G(p) \left(u_c(t) \right)
= \frac{k}{k_0} \cdot k_0 \cdot G(p) \left(u_c(t) \right) = \frac{k}{k_0} y_m(t)$$

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Then the update law for θ becomes

$$\left[rac{d}{dt} heta = -\gamma \cdot \left[2 \, e \, rac{\partial}{\partial heta} e
ight] = -\gamma_n \cdot y_m(t) \cdot e(t, heta)$$

where $\gamma_n>0$ is arbitrary since $\gamma_n=\gamma\,rac{k}{k_0}$ with arbitrary $\gamma>0$.

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Remark:
$$\mathcal{J}(\cdot) = |e(\cdot)| \Rightarrow \frac{d}{dt}\theta = -\gamma_n \cdot y_m(t) \cdot \mathrm{sign}[e(t,\theta)].$$

Suppose that

$$G(s) = \frac{1}{s+1}$$

and

$$k=1, \qquad k_0=2$$

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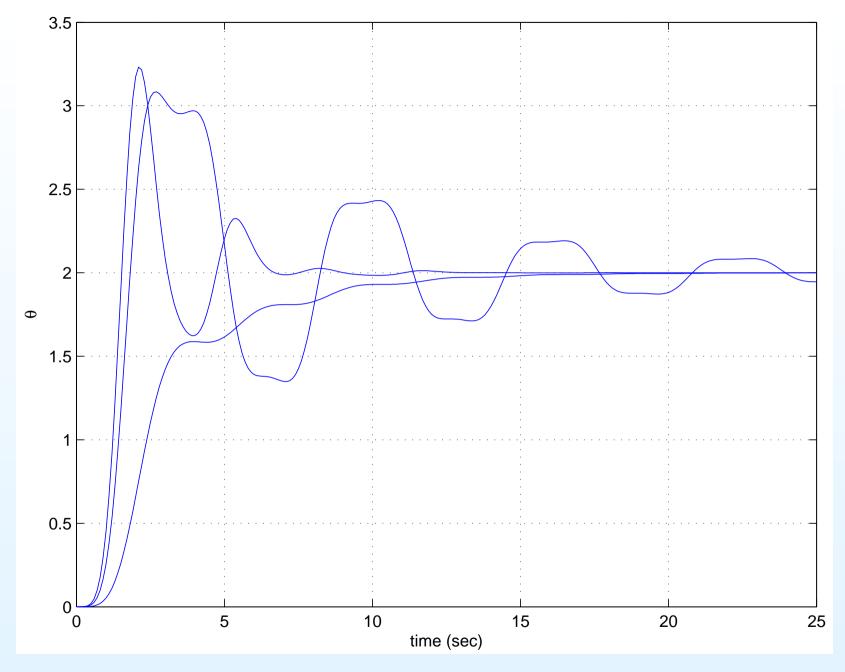
The update law for θ

$$\left|rac{d}{dt} heta = -\gamma\cdot\left[2\,e\,rac{\partial}{\partial heta}e
ight] = -\gamma_n\cdot y_m(t)\cdot e(t)$$

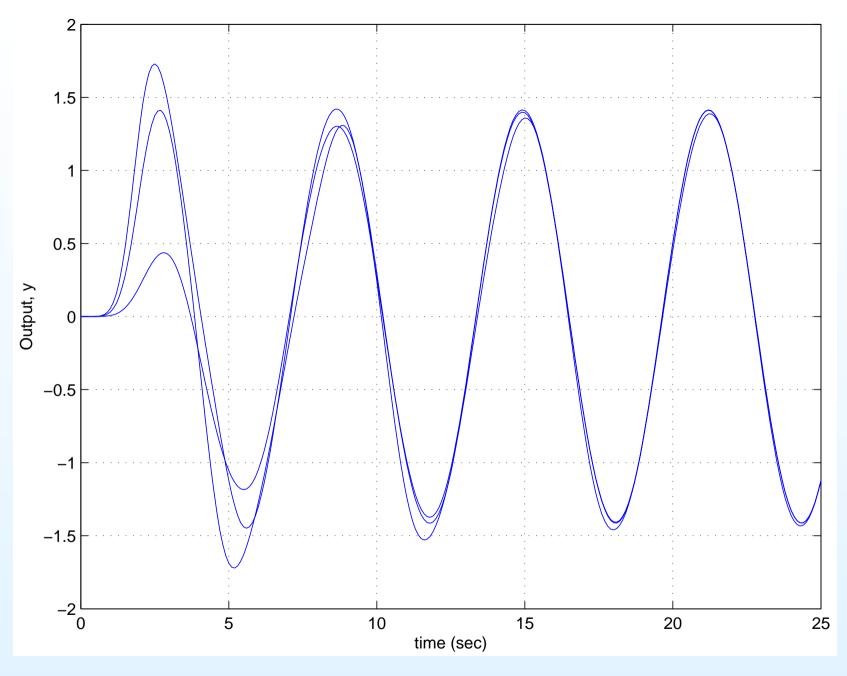
is simulated for various values of

$$\gamma = 0.5, \quad 2.5, \quad 4.5$$

with $u_c(t) = \sin t$.



The behavior of θ for various values of γ .



The behavior of y(t) for various values of γ .

Suppose that the system dynamics are

$$\frac{d}{dt}y = -ay + bu, \qquad y = \frac{b}{p+a}u$$

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While the desired dynamics for the closed loop system is

$$\frac{d}{dt}y_m = -a_m y_m + b_m u_c, \qquad y_m = \frac{b_m}{p + a_m} u_c$$

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The proportional controller that solves the problem is given by

$$u(t) = \frac{T(p)}{R(p)} \frac{\mathbf{u_c(t)}}{\mathbf{R}(p)} \frac{S(p)}{R(p)} y(t) = \theta_1 \frac{\mathbf{u_c(t)}}{\mathbf{u_c(t)}} - \theta_2 y(t)$$

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The proportional controller that solves the problem is given by

$$u(t) = \theta_1 \, \underline{u_c(t)} - \theta_2 \, y(t)$$

where the gains to ensure the desired system response are

$$heta_1= heta_1^0=rac{b_m}{b}, \qquad heta_2= heta_2^0=rac{a_m-a}{b}$$

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Introduce the error signal

$$e(t) = y(t) - y_m(t) = rac{b\, heta_1}{p+a+b\, heta_2}\, oldsymbol{u_c(t)} - rac{b_m}{p+a_m}\, oldsymbol{u_c}$$

Introduce the error signal

$$e(t) = y(t) - y_m(t) = \frac{b \theta_1}{p + a + b \theta_2} \frac{u_c(t)}{u_c(t)} - \frac{b_m}{p + a_m} \frac{u_c}{u_c}$$

Computing partial derivatives of $e(\cdot)$ w.r.t. θ_1 , θ_2 , we have

$$\frac{\partial e}{\partial \theta_1} = \frac{b}{p+a+b\theta_2} \mathbf{u_c(t)}$$

$$\frac{\partial e}{\partial \theta_2} = -\frac{b\theta_1 \cdot b}{(p+a+b\theta_2)^2} \mathbf{u_c(t)} = \frac{-b}{p+a+b\theta_2} y(t)$$

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Such formulas cannot be used for updating θ_1 and θ_2 values.

Indeed, constants a and b are not known!

Introduce the error signal

$$e(t) = y(t) - y_m(t) = \frac{b \theta_1}{p + a + b \theta_2} u_c(t) - \frac{b_m}{p + a_m} u_c$$

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Not much can be done, we will assume that we can initialize θ_2 around its nominal value

$$heta_2 pprox heta_2^0 = rac{a_m - a}{b}$$

Introduce the error signal

$$e(t) = y(t) - y_m(t) = \frac{b \theta_1}{p + a + b \theta_2} u_c(t) - \frac{b_m}{p + a_m} u_c$$

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Not much can be done, we will assume that we can initialize θ_2 around its nominal value

$$\theta_2 \approx \theta_2^0 = \frac{a_m - a}{b} \quad \Rightarrow \quad (a + b \, \theta_2) \approx a_m$$

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This results in the following relations

$$\begin{split} \frac{d}{dt}\theta_1 &= -\gamma \cdot e(t) \cdot \frac{\partial e(t)}{\partial \theta_1} \approx -\gamma \cdot e(t) \cdot \left(\frac{b}{s + a_m} u_c(t)\right) \\ &= -\gamma_n \cdot e(t) \cdot \left(\frac{a_m}{s + a_m} u_c(t)\right) \end{split}$$

This results in the following relations

$$egin{array}{lcl} rac{d}{dt} heta_1 &pprox & -\gamma_n\cdot e(t)\cdot \left(rac{a_m}{s+a_m}u_c(t)
ight) \\ rac{d}{dt} heta_2 &=& -\gamma\cdot e(t)\cdot rac{\partial e(t)}{\partial heta_2} pprox -\gamma\cdot e(t)\cdot \left(rac{-b}{s+a_m}y(t)
ight) \\ &=& \gamma_n\cdot e(t)\cdot \left(rac{a_m}{s+a_m}y(t)
ight) \end{array}$$

where

$$\gamma_n = \gamma rac{b}{a_m}$$

and should be positive!

This results in the following relations

$$\begin{array}{lcl} \frac{d}{dt}\theta_1 & \approx & -\gamma_n \cdot e(t) \cdot \left(\frac{a_m}{s+a_m} u_c(t)\right) \\ \\ \frac{d}{dt}\theta_2 & = & -\gamma \cdot e(t) \cdot \frac{\partial e(t)}{\partial \theta_2} \approx -\gamma \cdot e(t) \cdot \left(\frac{-b}{s+a_m} y(t)\right) \\ \\ & = & \gamma_n \cdot e(t) \cdot \left(\frac{a_m}{s+a_m} y(t)\right) \end{array}$$

where

$$\gamma_n = \gamma rac{b}{a_m}$$

and should be positive!

To implement this algorithm we need to know the sign of b!

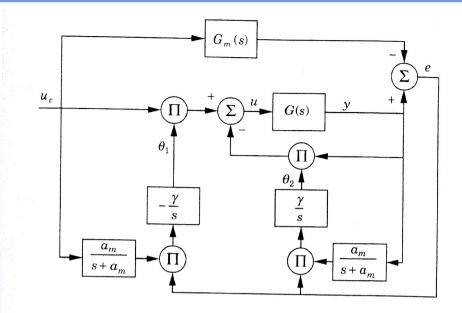


Figure 5.4 Block diagram of a model-reference controller for a first-order process.

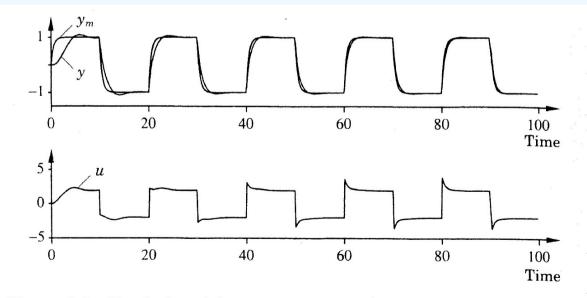


Figure 5.5 Simulation of the system in Example 5.2 using an MRAS. The parameter values are a = 1, b = 0.5, $a_m = b_m = 2$, and $\gamma = 1$.

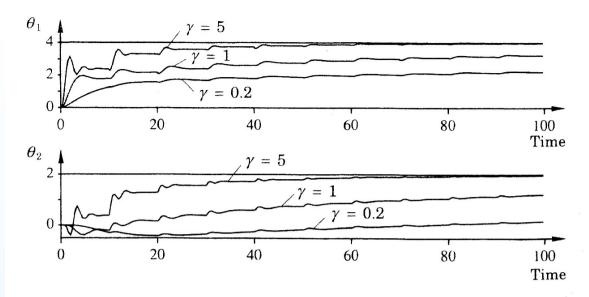


Figure 5.6 Controller parameters θ_1 and θ_2 for the system in Example 5.2 when $\gamma = 0.2$, 1 and 5.

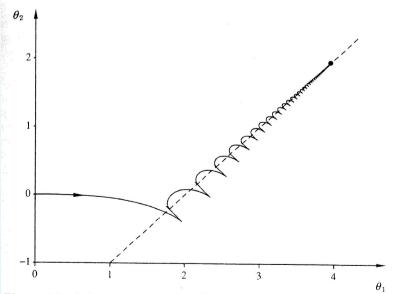


Figure 5.7 Relation between controller parameters θ_1 and θ_2 when the system in Example 5.2 is simulated for 500 time units. The dashed line shows the line $\theta_2 = \theta_1 - a/b$. The dot indicates the convergence point.

Consider the static system with unknown gain k

$$y(t) = k \cdot u(t), \qquad G(s) \equiv 1$$

and the problem of amplifying $u_c(t)$ so that we match

$$y_m(t) = k_0 \cdot u_c(t)$$

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$$y_m(t) = k_0 \cdot u_c(t)$$

With $u(t) = \theta u_c(t)$ introduce the error

$$e(t) = y(t) - y_m(t) = k \cdot \left(\theta u_c(t)\right) - k_0 \cdot u_c(t) = k \left(\theta - \theta^0\right) u_c(t)$$

with $\theta^0 = k_0/k$.

Consider the static system with unknown gain *k*

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with $\theta^0 = k_0/k$.

$$\frac{d}{dt}\theta(t) = -\gamma \cdot k^2 \cdot (\mathbf{u_c(t)})^2 \cdot (\theta(t) - \theta^0)$$

Consider the static system with unknown gain *k*

$$y(t) = k \cdot u(t), \qquad G(s) \equiv 1$$

and the problem of amplifying $u_c(t)$ so that we match

$$y_m(t) = k_0 \cdot u_c(t)$$

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with $\theta^0 = k_0/k$.

$$\frac{d}{dt} \left(\theta(t) - \theta^0 \right) = -\gamma_n \cdot k \cdot \left(\mathbf{u_c(t)} \right)^2 \cdot \left(\theta(t) - \theta^0 \right)$$

Consider the static system with unknown gain *k*

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with $\theta^0 = k_0/k$.

$$\left(\theta(t) - \theta^0\right) = \exp\left\{-\gamma_n \cdot k \cdot \int_0^t \left(\frac{u_c(t)}{u_c(t)}\right)^2 d\tau\right\} \cdot \left(\theta(0) - \theta^0\right)$$

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with $\theta^0 = k_0/k$.

$$e(t) = k \cdot \exp\left\{-\gamma_n \cdot k \cdot \int_0^t \left(u_c(t)\right)^2 d\tau\right\} \cdot \left(\theta(0) - \theta^0\right) \cdot u_c(t)$$

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For the system and model given by

$$y(t) = k \cdot u(t), \qquad y_m(t) = k_0 \cdot u_c(t)$$

we define $e(t) = y(t) - y_m(t)$ and take

$$u(t) = \theta(t) \frac{u_c(t)}{dt}, \qquad \frac{d}{dt}\theta(t) = -\gamma_n \cdot k \cdot (u_c(t))^2 \cdot (\theta(t) - \theta^0)$$

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we define $e(t) = y(t) - y_m(t)$ and take

$$u(t) = \theta(t) \, u_c(t), \qquad \frac{d}{dt} \theta(t) = -\gamma_n \cdot u_c(t) \cdot e(t)$$

As the result we obtain

$$\theta(t) = \theta^0 + \sigma(t), \qquad e(t) = k \cdot \sigma(t) \cdot u_c(t)$$

$$\sigma(t) = \exp\Bigl\{-\gamma_n \cdot k \cdot I_t\Bigr\}\Bigl(heta(0) - heta^0\Bigr), \qquad I_t = \int_0^t \left(rac{oldsymbol{u_c(t)}}{2}
ight)^2 d au$$

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As the result we obtain

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ight)^2 d au$$

If $\theta(0) \neq \theta^0$, for $e(t) \to 0$ as $t \to \infty$ we need:

$$\expigl(-\gamma_n\cdot k\cdot I_tigr) o 0$$
 or $oldsymbol{u_c(t)} o 0$

For the system and model given by

$$y(t) = k \cdot u(t), \qquad y_m(t) = k_0 \cdot u_c(t)$$

we define $e(t) = y(t) - y_m(t)$ and take

$$u(t) = \theta(t) \frac{u_c(t)}{dt}, \qquad \frac{d}{dt}\theta(t) = -\gamma_n \cdot u_c(t) \cdot e(t)$$

As the result we obtain

$$heta(t) = heta^0 + \sigma(t), \qquad e(t) = k \cdot \sigma(t) \cdot u_c(t)$$

$$\sigma(t) = \exp\Bigl\{-\gamma_n \cdot k \cdot I_t\Bigr\}\Bigl(heta(0) - heta^0\Bigr), \qquad I_t = \int_0^t \left(rac{oldsymbol{u_c(t)}}{oldsymbol{u_c(t)}}
ight)^2 d au$$

If $\theta(0) \neq \theta^0$, for $e(t) \to 0$ as $t \to \infty$ we need:

$$I_t = \int_0^t \left(u_{oldsymbol{c}}(t)
ight)^2 d au o \infty \qquad ext{or} \qquad u_{oldsymbol{c}}(t) o 0$$

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Consider again the problem with scaling the reference

$$y = k \cdot G(p) u$$
, $y_m = k_0 \cdot G(p) u_c$, $u = \theta u_c$

where $\theta(t)$ is determined by MIT rule:

$$rac{d}{dt} heta = -\gamma \cdot y_m \cdot e, \qquad e = y - y_m$$

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The equation for θ can be re-written as follows

$$rac{d}{dt} heta = -\gamma \cdot y_m \cdot \left(y - y_m
ight) = -\gamma \cdot y_m \cdot \left(k \cdot G(p) \, heta \, rac{\mathbf{u_c}}{\mathbf{u_c}} - y_m
ight)$$

Consider again the problem with scaling the reference

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Here

- the functions $y_m(t)$ and $u_c(t)$ are known!
- the range of the constant gain γ , for which the nominal value θ^0 (its stationary point) is stable, should be determined.

$$\frac{d}{dt}\theta(t) + \gamma \cdot k \cdot y_m(t) \cdot G(p) \left[\theta(t) \, \frac{u_c(t)}{u_c(t)}\right] = \gamma y_m^2(t)$$

In general the analysis of stability is difficult!

$$rac{d}{dt} heta(t) + \gamma \cdot k \cdot y_m(t) \cdot G(p) \Big[heta(t) \, rac{oldsymbol{u_c(t)}}{oldsymbol{u_c(t)}}\Big] = \gamma y_m^2(t)$$

In general the analysis of stability is difficult!

Consider the case when $y_m(t) \equiv y_m^o$, $u_c(t) = u_c^o$, then ODE

$$\frac{d}{dt}\theta(t) + \gamma \cdot k \cdot y_m^o \cdot \frac{\mathbf{u_c^0}}{\mathbf{c}} \cdot G(p) \Big[\theta(t) \Big] = \gamma (y_m^o)^2$$

is linear and time-invariant!

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is linear and time-invariant!

Stability is determined by the roots of the algebraic equation

$$s + \mu \cdot G(s) = 0, \qquad \mu = \gamma \cdot k \cdot y_m^o \cdot u_c^0$$

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is linear and time-invariant!

Stability is determined by the roots of the algebraic equation

$$s + \mu \cdot G(s) = 0, \qquad \mu = \gamma \cdot k \cdot y_m^o \cdot \mathbf{u_c^0}$$

Root locus analysis (variation of zeros with μ) can be used. A reasonable value for γ can be obtained from this analysis and might work for slowly varying signals.

Let, as in Example 5.1

$$G(s) = rac{1}{s+1}, \qquad k = 1, \qquad k_0 = 2$$

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$$G(s) = rac{1}{s+1}, \qquad k = 1, \qquad k_0 = 2$$

The characteristic equation

$$s + \mu \frac{1}{s+1} = 0 \quad \Leftrightarrow \quad s^2 + s + \mu = 0$$

has stable zeros if and only if

$$\mu = \gamma \cdot k \cdot y_m^o \cdot oldsymbol{u_c^0} = \gamma \cdot \left(k_0 \, G(0) \, oldsymbol{u_c^0}
ight) \cdot oldsymbol{u_c^0} = 2 \, \gamma \, (oldsymbol{u_c^0})^2 > 0$$

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ight) \cdot oldsymbol{u_c^0} = 2 \, \gamma \, (oldsymbol{u_c^0})^2 > 0$$

So, $\gamma > 0$ will work.

Note, however, that the transient depends on u_c^0 !

The relative damping is $\zeta = \frac{1}{2\sqrt{\mu}} = \frac{1}{2\sqrt{2\gamma}|\boldsymbol{u_c^0}|}$.

 $\mu pprox 1$ is reasonable \Leftarrow take $\gamma pprox 0.5$ for $u_c^0 pprox 1$ in average.

Consider the stable system with relative degree 2:

$$G(s) = \frac{1}{s^2 + a_1 s + a_2}, \qquad a_1 > 0, \qquad a_2 > 0$$

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The characteristic equation

$$s + \mu \frac{1}{s^2 + a_1 s + a_2} = 0 \quad \Leftrightarrow \quad s^3 + a_1 s^2 + a_2 s + \mu = 0$$

has stable zeros if and only if

$$\mu > 0$$
 and $a_1 a_2 > \mu = \gamma \cdot k \cdot y_m^o \cdot u_c^0$

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has stable zeros if and only if

$$\mu > 0$$
 and $a_1 a_2 > \mu = \gamma \cdot k \cdot y_m^o \cdot u_c^0$

Conclusion: with any choice of $\gamma > 0$, stability is lost for sufficiently large magnitudes of the reference signal $\boldsymbol{u_c^0}$!

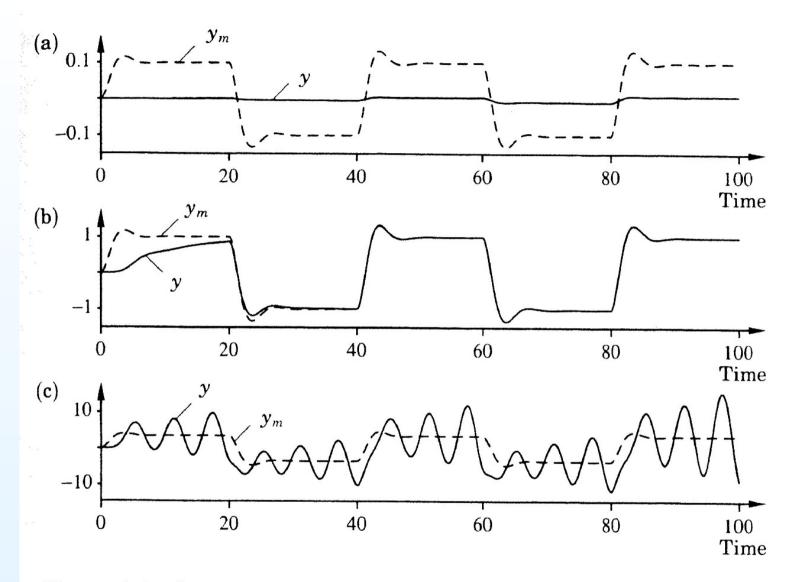


Figure 5.8 Simulation of the MRAS in Example 5.5. The command signal is a square wave with the amplitude (a) 0.1, (b) 1, and (c) 3.5. The model output y_m is a dashed line; the process output is a solid line. The following parameters are used: $k = a_1 = a_2 = \theta^0 = 1$, and $\gamma = 0.1$.

Normalized MIT rule

$$rac{d}{dt} heta = -\gamma \cdot e(t, heta) \cdot rac{\phi}{lpha + \phi^{{\scriptscriptstyle T}}\,\phi}, \qquad \phi = rac{\partial}{\partial heta}\,e(t, heta), \qquad lpha > 0$$

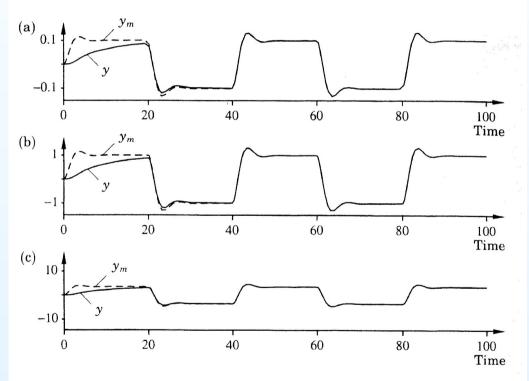


Figure 5.9 Simulation of the MRAS in Example 5.5 with the normalized MIT rule. The command signal is a square wave with the amplitude (a) 0.1, (b) 1, and (c) 3.5. Compare with Fig. 5.8. The model output y_m is a dashed line; the process output is a solid line. The parameters used are $k = a_1 = a_2 = \theta^0 = 1$, $\alpha = 0.001$, and $\gamma = 0.1$.

Next Lecture / Assignments:

Next meeting (May 24, 13:00-15:00, in A208Tekn): Lyapunov-based design.

Homework problem: The process and model are described by

$$G(s)=rac{1}{s}, \qquad G_m(s)=rac{2}{s+2}$$

For the control law

$$u(t) = \theta_1 \, u_c(t) - \theta_2 \, y(t)$$

design an MIT-like adaptation law such that

$$heta_i pprox - \left(\gamma_1 + \gamma_2 \, rac{1}{p}
ight) \, \left[e \, rac{\partial}{\partial heta_i} \, e
ight], \qquad i \in \{1,2\} \, .$$

Simulate the MRAS with various gains.

Consider $\gamma_{1,2} \in \{0,1,5\}$ and a unit square wave for $u_c(t)$. Compare performance for different combinations.