

① The energy equation of compressible flow for adiabatic flow is derived below:-

Assumption:

1. Steady flow
2. One dimensional flow
3. Frictionless flow
4. Equation of state is applicable.

Bernoulli's equation;

$$\frac{p}{\rho} + \frac{V^2}{2} + z = \text{const}$$

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$$\Rightarrow \frac{dp}{\rho} + \frac{2VdV}{2} + 0 = 0$$

$$\Rightarrow \frac{dp}{\rho} + VdV = 0 \quad \text{--- ①}$$

$$\Rightarrow d\rho \left[\left(\frac{1}{\rho} \right) \cdot \frac{p}{\rho} \right] + VdV = 0$$

$$\Rightarrow \frac{p}{\rho} \int_1^2 \frac{d\rho}{\rho^{1/k}} + \int_1^2 VdV = 0$$

$$\Rightarrow \frac{p}{\rho} \left[\frac{\rho^{-\frac{1}{k}+1}}{-\frac{1}{k}+1} \right]_1^2 + \frac{1}{2}(V_2^2 - V_1^2) = 0$$

For adiabatic process

$$pV^k = \text{const}$$

$$\frac{p}{\rho^k} = \text{const}$$

$$\frac{p}{\rho^k} = \frac{p_1}{\rho_1^k}$$

$$\Rightarrow \frac{1}{\rho^k} = \frac{1}{\rho_1^k} \cdot \frac{p_1}{p}$$

$$\Rightarrow \frac{1}{\rho} = \left(\frac{1}{\rho_1} \right)^{1/k} \cdot \frac{p_1}{p}^{1/k}$$

$$\frac{1}{\rho} = \frac{1}{\rho_1} \left(\frac{p_1}{p} \right)^{1/k}$$

$$\Rightarrow \frac{p}{\rho} \left[\frac{p_1^{1/k}}{\rho_1^{1/k}} - \frac{p_2^{1/k}}{\rho_2^{1/k}} \right] + \frac{1}{2}(V_2^2 - V_1^2) = 0$$

$$\Rightarrow \frac{1}{2}(V_2^2 - V_1^2) = \frac{p}{\rho} \left[\frac{p_1^{1/k}}{\rho_1^{1/k}} - \frac{p_2^{1/k}}{\rho_2^{1/k}} \right]$$

$$= \frac{p}{\rho} \left[\frac{1}{\rho_1^{1/k}} \left(\frac{p_1}{p} \right)^{1/k} - \frac{1}{\rho_2^{1/k}} \left(\frac{p_2}{p} \right)^{1/k} \right]$$

$$= \frac{p}{\rho} \left[\frac{p_1^{1/k}}{\rho_1^{1/k}} - \frac{p_2^{1/k}}{\rho_2^{1/k}} \right] \cdot \frac{1}{p^{1/k}} \cdot \frac{p^{1/k}}{\rho^{1/k}}$$

$$= \frac{p}{\rho} \left[\frac{p_1^{1/k}}{\rho_1^{1/k}} - \frac{p_2^{1/k}}{\rho_2^{1/k}} \right]$$

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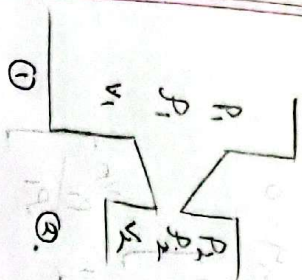
$$= \frac{p}{\rho} \left[\frac{p_1^{1/k}}{\rho_1^{1/k}} - \frac{p_2^{1/k}}{\rho_2^{1/k}} \right]$$

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$$= \frac{p}{\rho} \left[\frac{p_1^{1/k}}{\rho_1^{1/k}} - \frac{p_2^{1/k}}{\rho_2^{1/k}} \right]$$

$$\therefore \frac{1}{2}(V_2^2 - V_1^2) = \frac{p}{\rho} \left[\frac{p_1^{1/k}}{\rho_1^{1/k}} - \frac{p_2^{1/k}}{\rho_2^{1/k}} \right]$$

(2)



Assumptions:

1. Steady flow
2. One dimensional flow
3. Frictionless flow
4. Adiabatic process
5. Equation of state is applicable.

at point 2 max flow rate $\dot{m}_2 = A_2 \rho_2 v_2$ — (1)

if initial velocity $v_1 = 0$

Energy equation for adiabatic process

$$\frac{1}{2}(v_2^2 - v_1^2) = \frac{\kappa}{\kappa - 1} \frac{P_1}{\rho_1} \left[1 - \left(\frac{\rho_2}{\rho_1} \right)^{\kappa - 1/\kappa} \right]$$

$$\therefore v_2 = \sqrt{\frac{2\kappa}{\kappa - 1} \frac{P_1}{\rho_1} \left[1 - \left(\frac{\rho_2}{\rho_1} \right)^{\kappa - 1/\kappa} \right]} \quad \text{--- (II)}$$

Using equation (II) in equation (1) we have;

$$\dot{m}_2 = A_2 \rho_2 \sqrt{\frac{2\kappa}{\kappa - 1} \frac{P_1}{\rho_1} \left[1 - \left(\frac{\rho_2}{\rho_1} \right)^{\kappa - 1/\kappa} \right]} \quad \text{--- (III)}$$

$$\text{as } P/\rho^\kappa = \text{const}$$

$$\therefore \frac{P_1}{\rho_1^\kappa} = \frac{P_2}{\rho_2^\kappa}$$

$$\Rightarrow \rho_2^\kappa = \frac{P_2}{P_1} \rho_1^\kappa$$

$$\therefore \rho_2 = \left(\frac{P_2}{P_1} \right)^{1/\kappa} \rho_1 \quad \text{--- (IV)}$$

Using eqn (IV) in eqn (III)

$$\dot{m}_2 = A_2 \left(\frac{P_2}{P_1} \right)^{1/\kappa} \rho_1 \sqrt{\frac{2\kappa}{\kappa - 1} \frac{P_1}{\rho_1} \left[1 - \left(\frac{\rho_2}{\rho_1} \right)^{\kappa - 1/\kappa} \right]}$$

$$= A_2 \sqrt{\frac{2\kappa}{\kappa - 1} \frac{P_1}{\rho_1} \cdot \left(\frac{P_2}{P_1} \right)^{2/\kappa} \left[1 - \left(\frac{\rho_2}{\rho_1} \right)^{\kappa - 1/\kappa} \right]}$$

$$= A_2 \sqrt{\frac{2\kappa}{\kappa - 1} P_1 \rho_1 \left[\left(\frac{P_2}{P_1} \right)^{2/\kappa} - \left(\frac{P_2}{P_1} \right)^{2\kappa + \kappa - 1/\kappa} \right]}$$

$$= A_2 \sqrt{\frac{2\kappa}{\kappa - 1} P_1 \rho_1 \left[\left(\frac{P_2}{P_1} \right)^{2/\kappa} - \left(\frac{P_2}{P_1} \right)^{\frac{\kappa + 1}{\kappa}} \right]} \quad \text{--- (V) let } \frac{P_2}{P_1} = x$$

$$= A_2 \sqrt{\frac{2\kappa}{\kappa - 1} P_1 \rho_1 \left[x^{2/\kappa} - x^{\frac{\kappa + 1}{\kappa}} \right]}$$

$$\text{now, if } \frac{d\dot{m}_2}{dx} = 0$$

$$\Rightarrow \frac{d}{dx} \left[A_2 \sqrt{\frac{2\kappa}{\kappa - 1} P_1 \rho_1 \left[x^{2/\kappa} - x^{\frac{\kappa + 1}{\kappa}} \right]} \right] = 0$$

$$\Rightarrow A_2 \left(\frac{2\kappa}{\kappa - 1} P_1 \rho_1 \right)^{1/2} \left(x^{2/\kappa} - x^{\frac{\kappa + 1}{\kappa}} \right)^{-1/2} \left[\frac{2}{\kappa} x^{2/\kappa - 1} - \frac{\kappa + 1}{\kappa} x^{\frac{\kappa + 1}{\kappa} - 1} \right] = 0$$

$$\Rightarrow \frac{2}{\kappa} x^{2/\kappa - 1} - \frac{\kappa + 1}{\kappa} x^{1/\kappa} = 0$$

$$\Rightarrow \frac{2}{\kappa} x^{2/\kappa - 1} = \frac{\kappa + 1}{\kappa} x^{1/\kappa}$$

$$\Rightarrow \frac{x^{2/\kappa - 1}}{x^{1/\kappa}} = \frac{\kappa + 1}{2} x^{1/\kappa}$$

$$\Rightarrow x^{2/\kappa - 1 - 1/\kappa} = \frac{\kappa + 1}{2}$$

$$\Rightarrow 1 - k/k = \frac{k+1}{2}$$

$$\Rightarrow x = \left(\frac{k+1}{2}\right)^{k/k-1}$$

$$\Rightarrow x = \left(\frac{2}{k+1}\right)^{k/k-1}$$

$$\therefore \frac{P_2}{P_1} = \left(\frac{2}{k+1}\right)^{k/k-1} \quad \text{--- (6)}$$

For maximum max flow rate using eqn (5)

$$\dot{m}_1 = A_2 \sqrt{\frac{2k}{k-1} P_2} \left[\left(\frac{P_2}{P_1}\right)^{2/k} - \left(\frac{P_2}{P_1}\right)^{k+1/k} \right]$$

$$\dot{m}_2 = A_2 \sqrt{\frac{2k}{k-1} P_1} \left[\left(\frac{2}{k+1}\right)^{k/k-1} - \left(\frac{2}{k+1}\right)^{k/k+1} \right]$$

$$\dot{m}_1 = A_2 \sqrt{\frac{2k}{k-1} P_2} \sqrt{\left(\frac{2}{k+1}\right)^{2/k-1} - \left(\frac{2}{k+1}\right)^{k/k+1}}$$

$$= A_2 \sqrt{\frac{2k}{k-1} P_1} \cdot \left(\frac{2}{k+1}\right)^{k/k-1} \sqrt{\left(\frac{2}{k+1}\right)^{2/k-1} \left[1 - \left(\frac{2}{k+1}\right)^{k/k+1}\right]}$$

$$= A_2 \sqrt{\frac{2k}{k-1} P_1} \cdot \left(\frac{2}{k+1}\right)^{2/k-1} \sqrt{\left[1 - \left(\frac{2}{k+1}\right)^{k/k+1}\right]}$$

$$= A_2 \sqrt{\frac{2k}{k-1} P_1} \left(\frac{2}{k+1}\right)^{2/k-1} \left[\frac{k+1-2}{k+1} \right]^{1/k}$$

$$\dot{m}_1 = A_2 \left(\frac{2}{k+1}\right)^{2/k-1} \sqrt{\frac{2k}{k-1} P_1} \sqrt{\frac{k-1}{k+1}}$$

$$= A_2 \sqrt{2k P_1} \left(\frac{2}{k+1}\right)^{1/k-1} \cdot \frac{1}{\sqrt{k+1}} \cdot \frac{1}{\sqrt{k-1}}$$

$$= A_2 \sqrt{k P_1} \left(\frac{2}{k+1}\right)^{1/k-1} \cdot \left(\frac{2}{k+1}\right)^{1/k}$$

$$= A_2 \sqrt{k P_1} \left(\frac{2}{k+1}\right)^{1/k-1+1/2}$$

$$= A_2 \sqrt{k P_1} \left(\frac{2}{k+1}\right)^{\frac{k+1}{2(k-1)}}$$

$$\therefore \dot{m}_2 (\text{max}) = A_2 \sqrt{k P_1} \left(\frac{2}{k+1}\right)^{\frac{k+1}{2(k-1)}}$$

① The equation for relationship between mass velocity and Mach no. for compressible flow is given below:

Assumption:

① Steady flow

② One dimensional flow

③ Frictionless flow

④ adiabatic process

From Bernoulli's equation

$$\frac{dp}{\rho} + v dv = 0$$

$$\Rightarrow \frac{1}{\rho} K \rho \frac{dp}{\rho} + v dv = 0$$

$$\Rightarrow K \rho \frac{dp}{\rho} + v dv = 0$$

$$\Rightarrow \frac{dp}{\rho} + v dv = 0$$

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Mach no. $M = \frac{\text{velocity of fluid}}{\text{velocity of sound}}$

$$M = \frac{v}{c}$$

an adiabatic process

$$\frac{dp}{\rho} = \text{const}$$

Bulk modulus of elasticity

$$K = - \frac{dp}{dv} = \frac{dp}{d\rho}$$

$$K = K \rho$$

$$\frac{dp}{d\rho} = K \rho$$

$$-dp = K \rho \frac{d\rho}{\rho}$$

$$c^2 = \frac{K \rho}{\rho}$$

$$c^2 = \frac{K}{\rho}$$

$$\boxed{c^2 \rho = K}$$

an steady flow mass flow rate will be same.

$$\dot{m} = \rho A v = \text{const}$$

$$\Rightarrow \rho A v = \text{const}$$

$$\Rightarrow \log \rho + \log v + \log A = \log c$$

$$\Rightarrow \frac{d\rho}{\rho} + \frac{dv}{v} + \frac{dA}{A} = 0$$

$$\Rightarrow \frac{d\rho}{\rho} + \frac{dv}{v} - M^2 \frac{dv}{v} = 0$$

$$\Rightarrow \frac{d\rho}{\rho} + \frac{dv}{v} (1 - M^2) = 0$$

$$\Rightarrow \frac{d\rho}{\rho} = - \frac{dv}{v} (1 - M^2)$$

$$\Rightarrow \frac{d\rho}{\rho} = - \frac{dv}{v} (M^2 - 1)$$

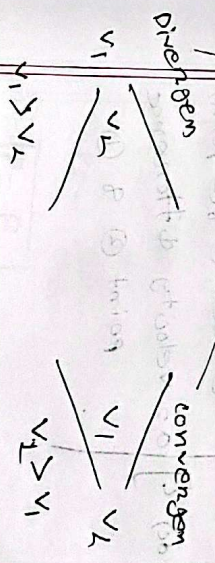
$$\Rightarrow \frac{d\rho}{\rho} = \frac{dv}{v} (M^2 - 1)$$

① If Subsonic flow $M < 1$

$$\therefore \frac{d\rho}{dv} = (-)ve$$

$$d\rho (+) \quad dv = -$$

$$d\rho (-) \quad dv = +$$



③ supersonic flow $M > 1$

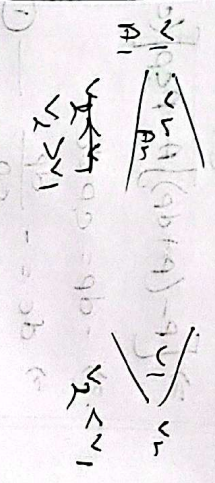
$$\therefore \frac{d\rho}{dv} = 0$$

④ If sonic flow $M = 1$

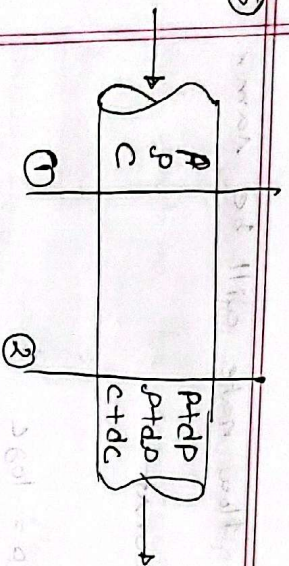
$$\therefore \frac{d\rho}{dv} = (+)ve$$

$$d\rho (+) \quad dv (+)$$

$$d\rho (-) \quad dv (-)$$



⑤



Assumptions:-

- ① Steady flow
- ② One dimensional flow
- ③ Frictionless flow
- ④ Adiabatic process.

As mass flow rate same we can write

$$\dot{m}_1 = \dot{m}_2$$

$$\Rightarrow A P c = A (P + dp) (c + dc) \quad \text{very small}$$

$$\Rightarrow = A (P c + P dc + c dp + dp dc)$$

$$\Rightarrow A P c \rightarrow A P c + A P dc + A c dp$$

$$\Rightarrow - P dc = c dp$$

$$\Rightarrow dc = -c \frac{dp}{P} \quad \text{--- ①}$$

We know $F = PA$

and from Newton's law of gravity

$$F = m a$$

$$\Rightarrow P A = m a$$

$$\Rightarrow [P - (P + dp)] A = A \rho c [c + dc - c]$$

$$\Rightarrow - dp = \rho c dc$$

$$\Rightarrow dc = - \frac{dp}{\rho c} \quad \text{--- ①}$$

P = pressure difference between ① & ② point
as velocity difference point ② & ①

⑥

For isentropic process also for adiabatic process.

$$P V^K = \text{const}$$

$$V^K dP + P \cdot K V^{K-1} dV = 0$$

$$\Rightarrow V^K [dP + \frac{P}{V} K dV] = 0$$

$$\Rightarrow dP = - K P \frac{dV}{V}$$

$$\Rightarrow K P = - \frac{dP}{dV}$$

$$\Rightarrow K P = E$$

For isothermal process

$$P V = \text{const}$$

$$P dV + dP \cdot V = 0$$

$$\Rightarrow P_2 = \frac{P_1 V_1}{V_2}$$

$$\Rightarrow P_2 = \frac{P_1}{\gamma}$$

$$\Rightarrow P = E$$

From equation ① & ②

$$-c \frac{dp}{P} = - \frac{dp}{\rho c}$$

$$\Rightarrow c^2 = \frac{dp}{dP}$$

$$\Rightarrow c^2 = \sqrt{\frac{dP}{dP}}$$

$$\boxed{c = \sqrt{\frac{E}{P}} \quad \text{or} \quad c^2 = \sqrt{\frac{K}{P}}}$$

Bulk modulus of elasticity

$$K = - \frac{dP}{dV/V} = \frac{dP}{dP/P} = E$$

$$\Rightarrow \frac{dP}{dP} = \frac{E}{P}$$

\therefore velocity of sound

$$c = \sqrt{\frac{E}{P}} = \sqrt{\frac{K P}{P}} = \sqrt{\frac{K}{P}}$$

$$c = \sqrt{\frac{K P}{P \rho}}$$

$$\boxed{c = \sqrt{\frac{K}{\rho}}}$$

if $P = \rho R T$

$$\boxed{c = \sqrt{\frac{P}{\rho}}}$$

$$\therefore c = \sqrt{R T}$$

Q) Prove that maximum velocity at the throat of the convergence divergence nozzle is equal to the velocity of sound.

From adiabatic energy equation;

$$\frac{1}{2}(v_2^2 - v_1^2) = \frac{k}{k-1} \frac{P}{\rho} \left[\frac{P_1}{P_1} - \frac{P_2}{P_2} \right]$$

$$= \frac{k}{k-1} \left[\frac{P_1}{P_1/RT_1} - \frac{P_2}{P_2/RT_2} \right]$$

$$= \frac{kR}{k-1} [T_1 - T_2]$$

$$= \frac{kR}{k-1} T_2 \left[\frac{T_1}{T_2} - 1 \right]$$

$$= \frac{kR}{k-1} T_2 \left[\left(\frac{P_2}{P_1} \right)^{\frac{1-k}{k}} - 1 \right]$$

$$= \frac{kR}{k-1} T_2 \left[\left(\frac{2}{k+1} \right)^{\frac{k}{k-1} \cdot \frac{1-k}{k}} - 1 \right]$$

$$= \frac{kR}{k-1} T_2 \left[\left(\frac{2}{k+1} \right)^{\frac{k}{k-1} \cdot \frac{-(k-1)}{k}} - 1 \right]$$

$$= \frac{kR}{k-1} T_2 \left[\left(\frac{2}{k+1} \right)^{-1} - 1 \right]$$

$$= \frac{kR}{k-1} T_2 \left[\frac{k+1}{2} - 1 \right]$$

$$= \frac{kR}{k-1} T_2 \cdot \frac{k-1}{2}$$

$$\frac{1}{2}(v_2^2 - v_1^2) = \frac{kRT_2}{2}$$

$$\therefore \frac{1}{2}v_2^2 = \frac{kRT_2}{2}$$

$$\therefore v_2 = \sqrt{kRT_2}$$

$$\boxed{v_2 = c}$$

proved.