

71.

a. With the speed controller configured as a proportional controller [ $K_{I_{SC}} = 0$  and  $G_{SC}(s) = K_{P_{SC}}$ ], the open-loop transfer function is:

$$G_{SC}(s)G_v(s) = \frac{0.11 K_{P_{SC}} (s + 0.6)}{s (s + 0.5173) + \frac{5}{s} (s + 0.6) \times (s + 0.01908)}.$$

Expanding the denominator of this transfer function, gives:  $D_G(s) = 6s^2 + 3.613s + 0.05724$ .

Solving for the roots shows that there are two open-loop poles:  $-0.5858$  and  $-0.0163$ . Thus, the open-loop transfer function may be re-written as:

$$G_{SC}(s)G_v(s) = \frac{0.11 K_{P_{SC}} (s + 0.6)}{6s^2 + 3.613s + 0.05724} = \frac{K_1 (s + 0.6)}{(s + 0.5858)(s + 0.0163)} = -1 \quad (1)$$

In this equation:  $K_1 = \frac{K_{P_{SC}} \times 0.11}{6} \quad (2)$

The following MATLAB M-file was written to plot the root locus for the system and to find the value of the proportional gain,  $K_I$ , at the breakaway or break-in points.

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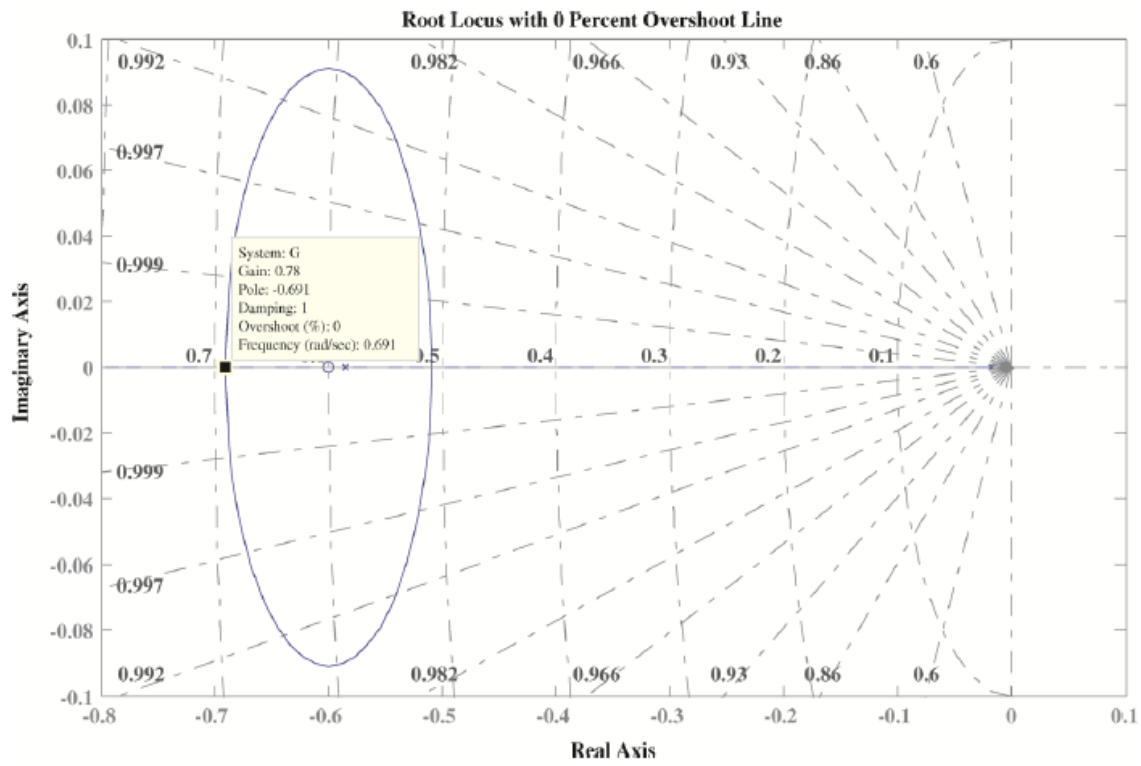
numg = [1 0.6];
deng = poly ([-0.0163 -0.5858]);
G = tf(numg, deng);
rlocus(G);
pos=(0);
z=-log(pos/100)/sqrt(pi^2+(log(pos/100))^2);
sgrid(z,0)
title(['Root Locus with ', num2str(pos) , ' Percent Overshoot
Line'])
[Kl,p]=rlocfind(G);
pause
T=feedback(Kl*G,1);      %T is the closed-loop TF of the system
T=minreal(T);
step(T);
axis ([0, 8, 0, 1]);
grid

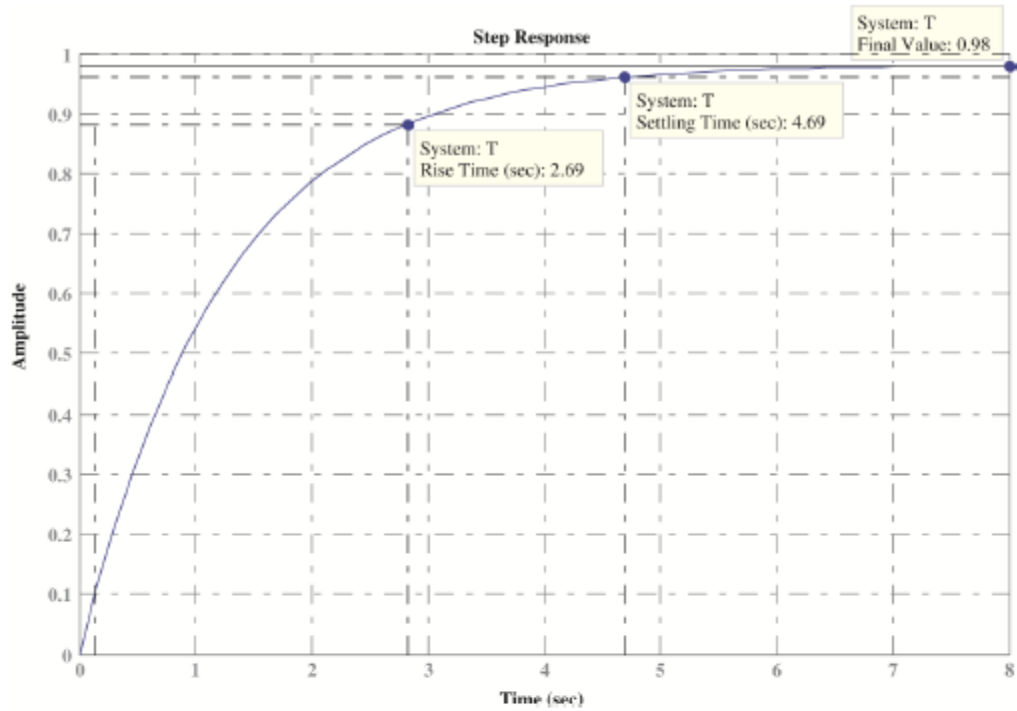
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The root locus shown below was obtained. Using MATLAB tools, the gain at the break-in point was found to be larger and, hence, would yield a faster closed-loop unit-step response. The following repeated real poles were found, which indicated that the step response is critically damped:  $p = -0.6910, -0.6910$ . These poles corresponded to:  $K_1 = 0.78$  (which corresponds to  $K_{p_{SC}} = 42.54$ ). The closed-loop transfer function,  $T(s)$ , was found to be:

$$T(s) = \frac{0.78s + 0.468}{s^2 + 1.382s + 0.4775}$$

Therefore, it was used to find the closed-loop transfer function of the system, to plot its unit-step response,  $c(t)$ , shown below, and to find the rise-time,  $T_r$ , and settling time,  $T_s$ .





As could be seen from the graph, these times are:

$$T_r = 2.69 \text{ sec and } T_s = 4.69 \text{ sec}$$

b. When integral action was added (with  $K_{ISC}/K_{psc} = 0.4$ ), the transfer function of the speed controller became:  $G_{SC}(s) = K_{psc} + \frac{K_{isc}}{s} = \frac{K_{psc}(s + 0.4)}{s}$  and the open-loop transfer function obtained was:

$$G_{SC}(s)G_v(s) = \frac{0.11K_{psc}(s + 0.6)(s + 0.4)}{s(6s^2 + 3.613s + 0.05724)} = \frac{K_1(s + 0.6)(s + 0.4)}{s(s + 0.5858)(s + 0.0163)} = -1$$

$$\text{Where } K_1 = \frac{0.11K_{psc}}{6} \text{ or } K_{psc} = \frac{6K_1}{0.11} = 54.5455 \times K_1$$

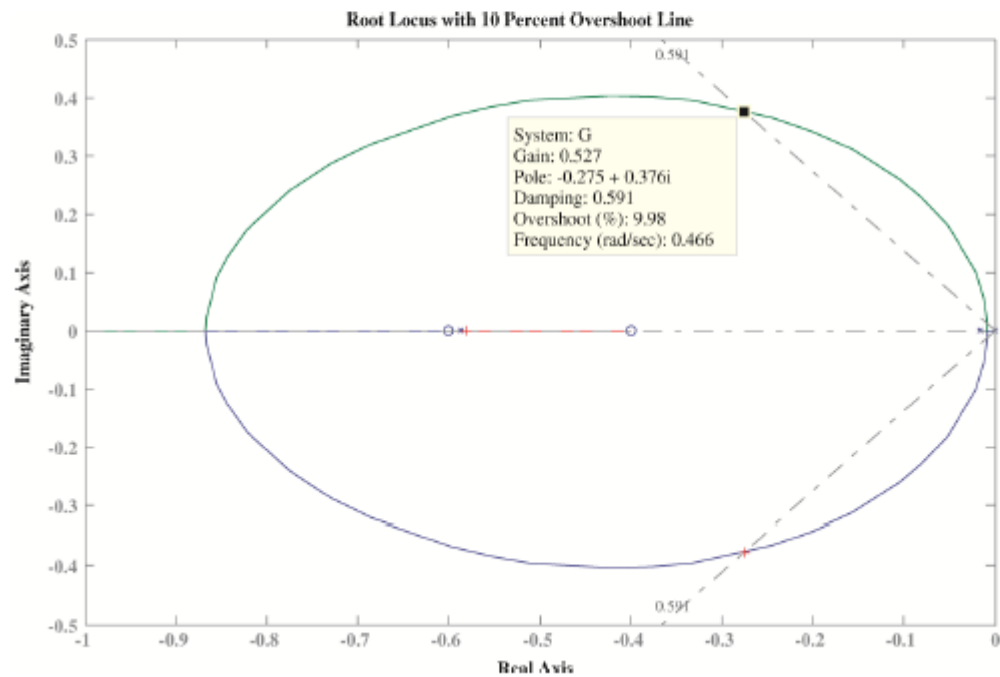
The following MATLAB M-file was written to plot the root locus for the system and to find the gain,  $K_1$ , which could result in a closed-loop unit-step response with 10% overshoot.

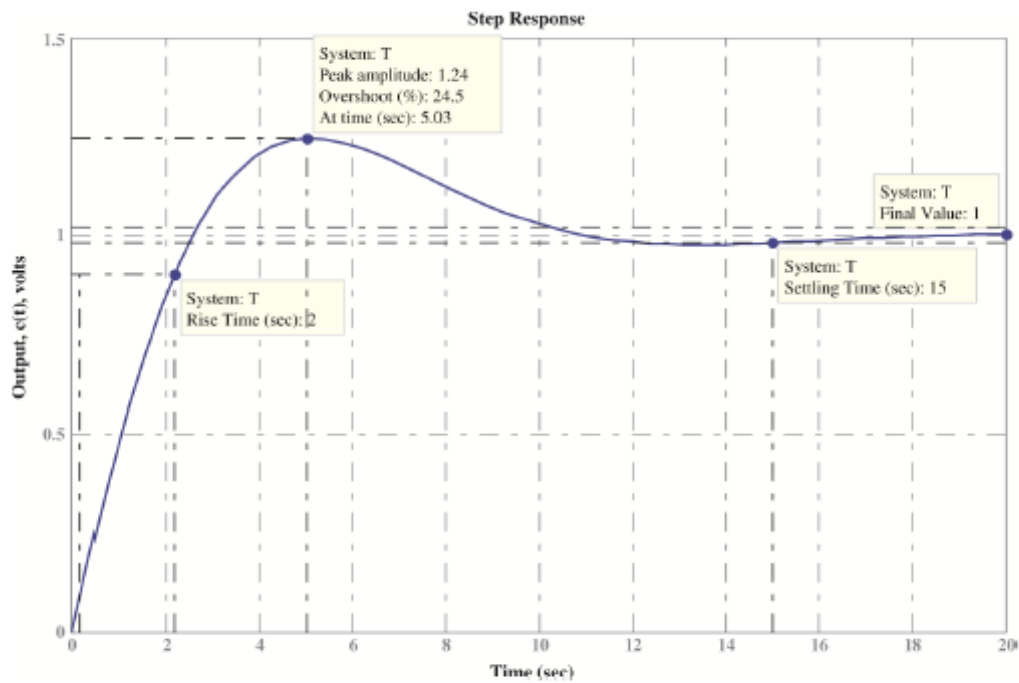
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numg = poly ([-0.4 -0.6]);
deng = poly ([0 -0.0163 -0.5858]);
G = tf(numg, deng);
rlocus(G);
pos=(10);
z=-log(pos/100)/sqrt(pi^2+(log(pos/100))^2);
axis ([-1, 0, -0.5, 0.5]);
sgrid(z,0)
title(['Root Locus with ', num2str(pos) , ' Percent Overshoot
Line'])
[K1,p]=rlocfind(G);
pause
T=feedback(K1*G,1);      %T is the closed-loop TF of the system
T=minreal(T);
step(T);
axis ([0, 20, 0, 1.5]);
grid
```

The root locus shown below was obtained. Using MATLAB tools, the gain at the point selected on the locus ( $-0.275 + j 0.376$ ) was found to be  $K_1 = 0.526$  (which corresponds to  $K_{r_{SC}} = 28.7$ ). The corresponding closed-loop transfer function,  $T(s)$ , is:

$$T(s) = \frac{0.526 s^2 + 0.526 s + 0.1262}{s^3 + 1.128 s^2 + 0.5355 s + 0.1262}$$

$T(s)$  has the closed-loop poles:  $p = -0.580, -0.275 \pm j 0.376$  and zeros at  $-0.4$  &  $-0.6$ . Thus, the complex conjugate poles are not dominant, and hence, the output response,  $c(t)$ , obtained using MATLAB, does not match that of a second-order underdamped system. Note also that the settling time,  $T_s = 15$  sec, the rise time,  $T_r = 2$  sec, the peak time,  $T_p = 5.03$  sec, and the overshoot is 24.5% (higher than the 10% corresponding to the dominant poles).





It should be mentioned that since we applied 1 volt-unit-step inputs (as compared to 4 volts in the Hybrid vehicle progressive problem in Chapter 5) in both parts (a) and (b) above, we should not be surprised that the final (steady-state) value of output voltage of the speed transducer was 1 volt, which corresponds to a change in car speed of only 5 km/hr.