Chapter 4 Homework, 7th edition, Fall 2016

4.

Using voltage division,
$$\frac{V_C(s)}{V_i(s)} = \frac{1/RC}{s + \frac{1}{RC}} = \frac{1}{s+1}$$
. Since $V_i(s) = \frac{5}{s}$

$$V_C(s) = \frac{5}{s} \left(\frac{1}{s+1} \right) = \frac{5}{s} - \frac{5}{s+1}$$
.

Therefore: $v_c(t) = 5 - 5e^{-t}$.

Also,
$$T = \frac{1}{1} = 1 \sec; T_r = \frac{2.2}{1} = 2.2 \sec; T_s = \frac{4}{1} = 4 \sec$$
.

25.

a. Adding impedances $(5s^2 + 2s + 20)X(s) = F(s)$. So the transfer function is

$$\frac{X(s)}{F(s)} = \frac{1}{5s^2 + 2s + 20} = \frac{0.2}{s^2 + 0.4s + 4}.$$

b. Follows that $\omega_n^2 = 4$, or $\omega_n = 2$. $2\zeta\omega_n = 0.4$, so $\xi = 0.1$. %OS = $100e^{-\frac{\zeta\pi}{\sqrt{1-\zeta^2}}} = 72.93\%$. $T_s = \frac{4}{\xi\omega_n} = 20$

sec. $T_p = \frac{\pi}{\omega_n \sqrt{1 - \xi^2}} = 1.58$ sec. To obtain the rise time Figure 4.16 is used $\omega_n T_r = 1.104$, or

$$T_r = 0.552$$
 sec. The dc gain of the system is $c_{max} = \frac{0.2}{4} = 0.05$.

35. (a) Does not meet the x5 rule (page 184): 4 < 5*(3)

Residue is not negligible (section 4.8):

$$0.1 * \sqrt{0.001976^2 + 0.0005427^2} = 0.0002$$
 (using 10% as cutoff)

2nd order approximation is not valid

(b) Does not meet the x5 rule: 8 < 5*(6)

Residue is not negligible (section 4.8):

$$0.1*\sqrt{0.007647^2+0.01309^2}=0.0015\,$$
 (using 10% as cutoff)

2nd order approximation is not valid

(c) Does not meet the x5 rule (5.1 < 5*2)

Residue is negligible

$$0.1 * \sqrt{0.009990^2 + 0.001942^2} = 0.0022$$
 (using 10% as cutoff)

0.00018 < 0.0022 (actually is less than 1%)

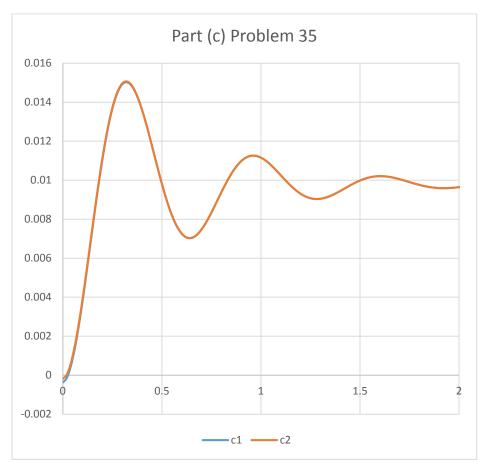
From the oscillatory part of c(t), the dominant poles are $s=-2\pm j9.796$

The time constant is the reciprocal of the magnitude of the real part, $\frac{1}{2}$ second. The settling time is four time constants: 4*(1/2) = 2 seconds = T_s . The damped frequency is 9.796 rad/s. The peak time is one half of the period: $T_p = \pi/\omega_d = 0.3207$ seconds = T_p .

To find damping constant, use $\zeta \omega_n = 2$ and $\omega_n \sqrt{1-\zeta^2} = 9.796$. Rearranging yields

$$\zeta = \sqrt{2^2/(2^2 + 9.796^2)} = 0.200$$

Then %OS =
$$e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}}$$
x100 = 52.7%



Approximate curve in matlab is:

$$c2 = 0.009804-0.009990*exp(-2*t).*cos(9.796*t) -0.001942 *exp(-2*t).*sin(9.796*t);$$

(d) Exactly meets the x5 rule: 10 = 5*2

That term also has a smaller residue, but only about five times smaller, so it does not meet the 10% cutoff.

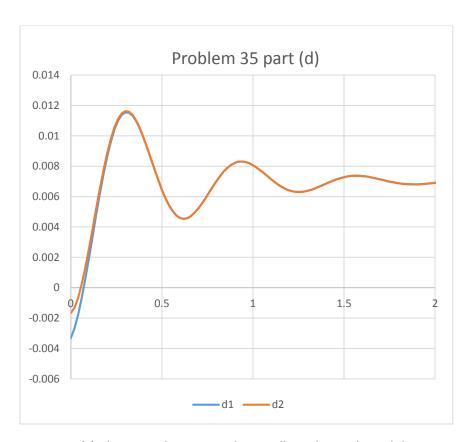
From the oscillatory part of c(t), the dominant poles are $s=-2\pm j9.951$

The time constant is the reciprocal of the magnitude of the real part, $\frac{1}{2}$ second. The settling time is four time constants: 4*(1/2) = 2 seconds = T_s . The damped frequency is 9.951 rad/s. The peak time is one half of the period: $T_p = \pi/\omega_d = 0.3157$ seconds = T_p .

To find damping constant, use $\zeta \omega_n = 2$ and $\omega_n \sqrt{1-\zeta^2} = 9.951$. Rearranging yields

$$\zeta = \sqrt{2^2/(2^2 + 9.951^2)} = 0.197$$

Then %OS =
$$e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}}$$
x100 = 53.1%



As in part (c), dropping the term with a small residue and rapid decay rate.

$$X(s) = (sI - A)^{-1} (x(0) + Bu(t)) = \begin{bmatrix} s+3 & 0 \\ 1 & s+1 \end{bmatrix}^{-1} \begin{bmatrix} 2 \\ 0 \end{bmatrix} + \begin{bmatrix} 2 \\ 1 \end{bmatrix} \frac{1}{s}$$

$$= \begin{bmatrix} \frac{1}{s+3} & 0 \\ -\frac{1}{(s+1)(s+3)} & \frac{1}{s+1} \end{bmatrix} \begin{bmatrix} \frac{2s+2}{s} \\ \frac{1}{s} \end{bmatrix} = \begin{bmatrix} \frac{2(s+1)}{s(s+3)} \\ -\frac{s-1}{s(s+1)(s+3)} \end{bmatrix}$$

$$Y(s) = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} \frac{2(s+1)}{s(s+3)} \\ -\frac{s-1}{s(s+1)(s+3)} \end{bmatrix} = \frac{2(s+1)}{s(s+3)} = \frac{2}{3}\frac{1}{s} + \frac{5}{3}\frac{1}{s+3}$$

Obtaining the inverse Laplace transform

$$y(t) = \frac{2}{3} + \frac{5}{3}e^{-3t}$$

Let the rotation of the shaft with gear N_2 be $\theta_L(s)$. Assuming that all rotating load has been reflected to the N_2 shaft, $\left(J_{eqL}s^2 + D_{eqL}s + K\right)\theta_L(s) + F(s)r = T_{eq}(s)$, where F(s) is the force from the translational system, r=2 is the radius of the rotational member, J_{eqL} is the equivalent inertia at the N_2 shaft, and D_{eqL} is the equivalent damping at the N_2 shaft. Since $J_{eqL} = 1(2)^2 + 1 = 5$ and $D_{eqL} = 1(2)^2 = 4$, the equation of motion becomes, $\left(5s^2 + 4s + K\right)\theta_L(s) + 2F(s) = T_{eq}(s)$. For the translational system $(Ms^2 + s)X(s) = F(s)$. Substituting F(s) into the rotational equation of motion, $\left(5s^2 + 4s + K\right)\theta_L(s) + \left(Ms^2 + s\right)2X(s) = T_{eq}(s)$. But,

 $\theta_L(s) = \frac{X(s)}{r} = \frac{X(s)}{2}$ and $T_{eq}(s) = 2T(s)$. Substituting these quantities in the equation above yields $\left((5+4M)s^2+8s+K\right)\frac{X(s)}{4} = T(s)$.

Thus, the transfer function is
$$\frac{X(s)}{T(s)} = \frac{4/(5+4M)}{s^2 + \frac{8}{(5+4M)}s + \frac{K}{(5+4M)}}.$$

Now,
$$T_s = 20 = \frac{4}{R_e} = \frac{4}{\frac{8}{2(5+4M)}} = (5+4M)$$
. Hence, $M = 15/4$.

For 16% overshoot, $\zeta = 0.504$ from Eq. (4.39).

Therefore, $2\zeta\omega_n = \frac{8}{(5+4M)} = 0.4$. Solving for ω_n yields $\omega_n = 0.3968$.

But,
$$\omega_n = \sqrt{\frac{K}{(5+4M)}} = \sqrt{\frac{K}{20}} = 0.3968$$
. Thus, $K = 3.15$.

85.

Substituting $\Delta F(s) = \frac{2650}{s}$ into the transfer function and solving for $\Delta V(s)$ gives:

$$\Delta V(s) = \frac{\Delta F(s)}{1908 \cdot s} = \frac{2650}{s(1908 \cdot s + 10)} = \frac{A}{s} + \frac{B}{(1908 \cdot s + 10)}$$

Here:
$$A = \frac{2650}{(1908 \cdot s + 10)} \Big|_{s=0} = 265 \text{ and } B = \frac{2650}{s} \Big|_{s=-\frac{1}{190.8}} = -505,620$$

Substituting we have:

$$\Delta V(s) = \frac{265}{s} - \frac{505620}{(1908 \cdot s + 10)} = 265 \left(\frac{1}{s} - \frac{1}{(s + 5.24 \times 10^{-3})} \right)$$

Taking the inverse Laplace transform, we have:

$$\Delta v(t) = 265(1 - e^{-5.24 \times 10^{-3}t}) \cdot u(t)$$
, in m/s

b.

>> s=tf('s');

$$>> G=1/(1908*s+10);$$

>> t=0:0.1:1000;

$$>> y1=2650*step(G,t);$$

>> plot(t,y1,t,y2)

>> xlabel('sec')

>> ylabel('m/s')

