Analysis : Case 1

| Uniform load, w: | 10 N/mm |
|--------------------------------|---------|
| Length, L: | |
| Omit beam weight for this case | |

C1: Final Meshing

This mesh had two levels of refinement. For the larger blocks, the max element size was 16mm. For the smaller sections, the max element size was 8mm.

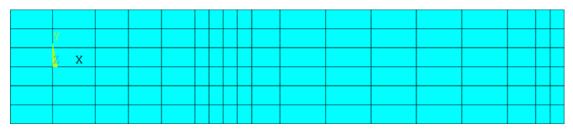


Fig. C1.1

C1 : Deflected shape of beam

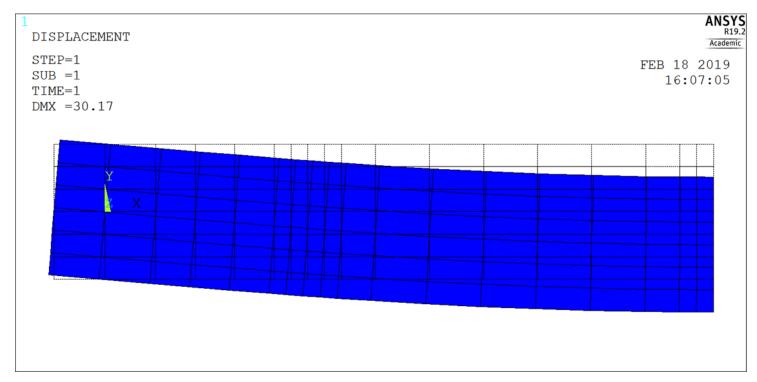


Fig. C1.2

C1 : Contour Plot of Normal Bending Stresses (σ_.)

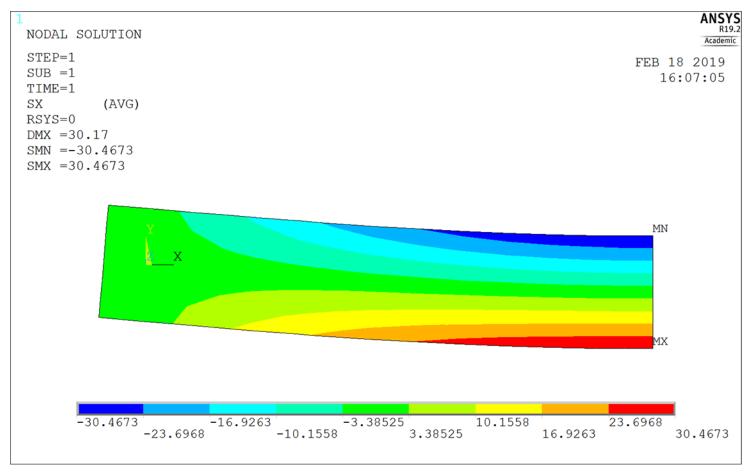


Fig. C1.3

C1: Vertical Deflection at midspan from ANSYS Model

In order to easily review the nodes at location Q and the bottom of Cut 1, I created components that would allow me to call them. Fig. C1.4 shows the list of the components which allows me to identify which label corresponds to the nodes in Fig. C1.5.

```
LIST SELECTED COMPONENTS
ENTITY TYPE = NODE

NAME TYPE SUBCOMPONENTS

NODE_CUT_1 NODE 167

NODE_Q NODE 19
```

Fig. C1.4

```
PRINT U
          NODAL SOLUTION PER NODE
 **** POST1 NODAL DEGREE OF FREEDOM LISTING *****
LOAD STEP=
               1 SUBSTEP=
 TIME=
          1.0000
                      LOAD CASE=
THE FOLLOWING DEGREE OF FREEDOM RESULTS ARE IN THE GLOBAL COORDINATE SYSTEM
  NODE
                     UY
    19
        -30.0155815879
   167 -15.2429198449
MAXIMUM ABSOLUTE VALUES
NODE
            19
        -30.0155815879
VALUE
```

Fig. C1.5

C1: Distribution of Bending Stress at Cut 1 and Cut 2

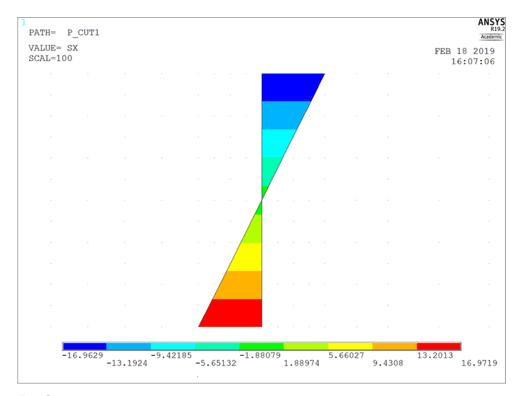


Fig. C1.6

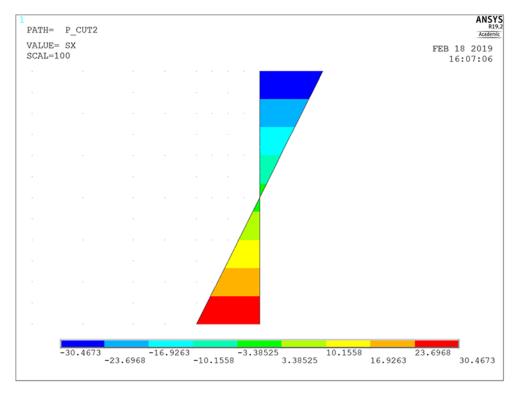


Fig. C1.7

C1 : Calculation of Deflection and Flexural Stress using Elementary Beam Theory

| Common Properties | | | | | | | | |
|-------------------------------|---|--------|------|--|--|--|--|--|
| Width | b | 20 | mm | | | | | |
| Depth | h | mm | | | | | | |
| Modulus of Elasticity | Е | 700 | N/mm | | | | | |
| Moment of Inertia = bh³/12 | I | 106667 | Pa | | | | | |
| Distance from N.A. | У | 20 | mm | | | | | |

| Case 1 Unique Properties | | | | | | | |
|--------------------------|-----|-------------|--------|--|--|--|--|
| Uniform Load | ω | | | | | | |
| Length | L | 360 | mm | | | | |
| Moment | h 4 | 4 162000 Nm | | | | | |
| $=\omega L^2/8$ | /VI | 162000 | NM | | | | |
| Deflection at Midspan | δ | 20.200 | | | | | |
| $= 5wL^4/384EI$ | 0 | 29.290 | mm | | | | |
| Flexural Stress | | 20.275 | D. | | | | |
| =My/I | σ | 30.375 | Pa | | | | |

C1: Comparison of Calcuations to ANSYS Results

| | Calculation | ANSYS | Accuracy |
|-----------------------|-------------|---------|----------|
| Deflection at Midspan | 29.290 | 30.0156 | 97.58% |
| Flexural Stress | 30.375 | 30.4673 | 99.70% |

Analysis : Case 2

| Uniform load, w: | 30 N/mm |
|--------------------------------|---------|
| Length, L: | |
| Omit beam weight for this case | |

C2: Final Meshing

This mesh had two levels of refinement. For the larger blocks, the max element size was 4mm. For the smaller sections, the max element size was 2mm.

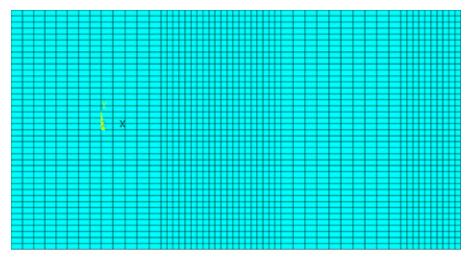


Fig. C2.1

C2: Deflected shape of beam

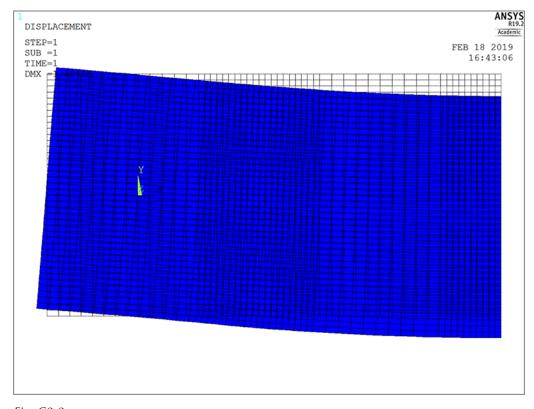


Fig. C2.2

C2 : Contour Plot of Normal Bending Stresses (σ_x)

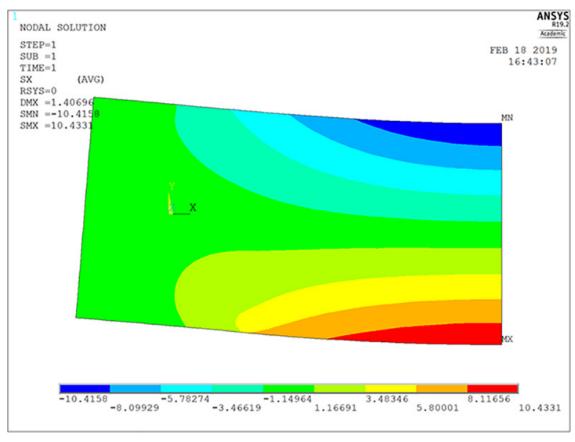


Fig. C2.3

C2: Vertical Deflection at midspan from ANSYS Model

Fig. C2.4 shows the list of the components which allows me to identify which label corresponds to the nodes in Fig. C2.5.

```
LIST SELECTED COMPONENTS
ENTITY TYPE = NODE

NAME TYPE SUBCOMPONENTS

NODE_CUT_1 NODE 2970

NODE_Q NODE 538
```

Fig. C2.4

```
PRINT U
          NODAL SOLUTION PER NODE
 **** POST1 NODAL DEGREE OF FREEDOM LISTING *****
 LOAD STEP=
               1 SUBSTEP=
                               1
          1.0000
                      LOAD CASE=
                                  0
 TIME=
 THE FOLLOWING DEGREE OF FREEDOM RESULTS ARE IN THE GLOBAL COORDINATE SYSTEM
   NODE
                     UY
        -1.34568286451
    538
   2970 -0.697286570642
MAXIMUM ABSOLUTE VALUES
NODE
           538
VALUE
        -1.34568286451
```

Fig. C2.5

C2: Distribution of Bending Stress at Cut 1 and Cut 2

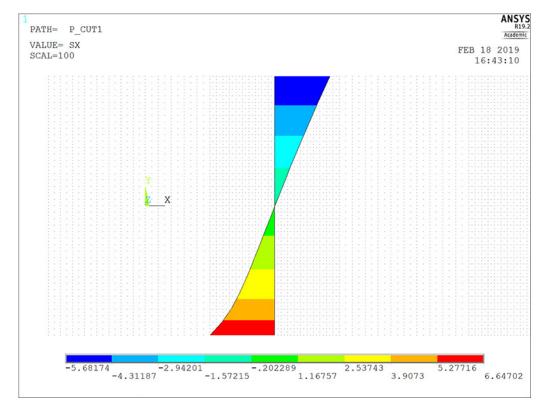


Fig. C2.6

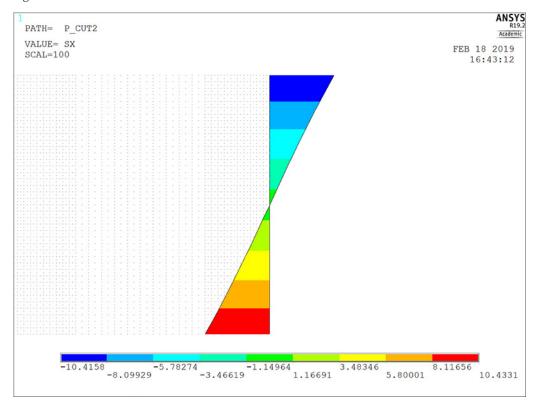


Fig. C2.7

C2: Calculation of Deflection and Flexural Stress using Elementary Beam Theory

| Common Properties | | | | | | | |
|-------------------------------|---|--------|------|--|--|--|--|
| Width | b | 20 | mm | | | | |
| Depth | h | 40 | mm | | | | |
| Modulus of Elasticity | Е | 700 | N/mm | | | | |
| Moment of Inertia = bh³/12 | I | 106667 | Pa | | | | |
| Distance from N.A. | У | 20 | mm | | | | |

| Case 1 Unique Properties | | | | | | | |
|--------------------------|-----|----------|--------|--|--|--|--|
| Uniform Load | ω | 30 | N/mm | | | | |
| Length | L | 120 | mm | | | | |
| Moment | М | 54000 Nm | | | | | |
| $=\omega L^2/8$ | 171 | 34000 | INIII | | | | |
| Deflection at Midspan | δ | 1.085 | mm | | | | |
| $= 5wL^4/384EI$ | | 1.005 | mm | | | | |
| Flexural Stress | σ | 10.125 | Da | | | | |
| =My/I | | 10.125 | Pa | | | | |

C2: Comparison of Calcuations to ANSYS Results

| | Calculation | ANSYS | Accuracy |
|-----------------------|-------------|---------|----------|
| Deflection at Midspan | 1.085 | 1.3457 | 80.61% |
| Flexural Stress | 10.125 | 10.4331 | 97.05% |

Analysis : Case 3

C3: Fundamental Natural Frequencies of the Beam System

Through four different substeps you are abble to see the different displacements and frequencies of the beam. Fig. C3.5 provides a table combining all of the numbers.

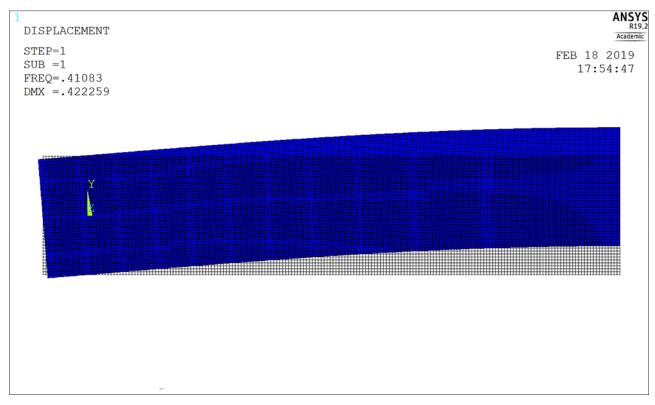


Fig. C3.1

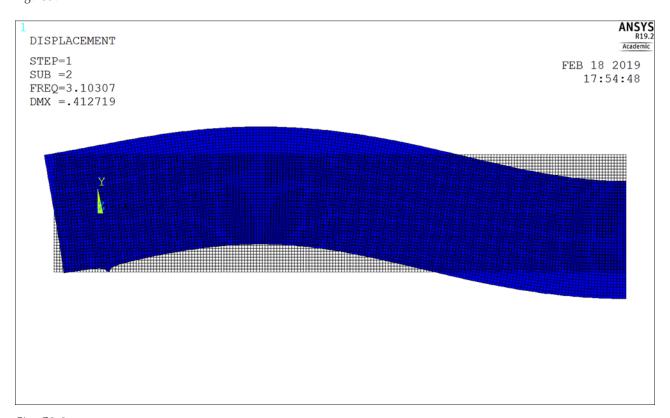


Fig. C3.2

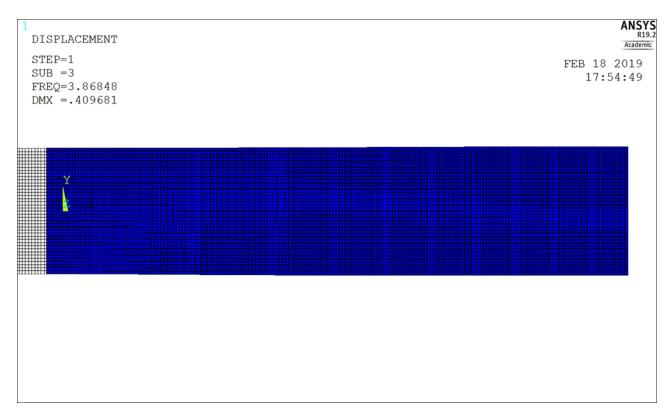


Fig. C3.3

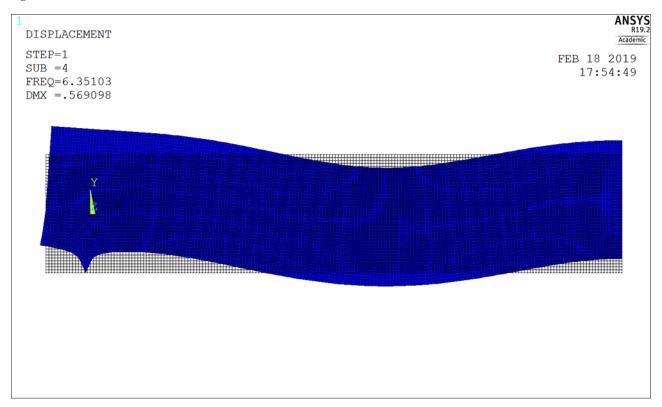


Fig. C3.4

***** INDEX OF DATA SETS ON RESULTS FILE *****

| SET | TIME/FREQ | LOAD STEP | SUBSTEP | CUMULATIVE |
|-----|-----------|-----------|---------|------------|
| 1 | 0.41083 | 1 | 1 | 1 |
| 2 | 3.1031 | 1 | 2 | 2 |
| 3 | 3.8685 | 1 | 3 | 3 |
| 4 | 1 6.3510 | 1 | 4 | 4 |

Fig. C3.5

Analysis: Case 4

| Uniform load, w: 7.5 N/mm | 1 |
|--------------------------------|---|
| Length, L: | |
| Omit beam weight for this case | |

C4: Bilinear Material Model with Kiematic Hardening

Fig. C4.1 is the preview of the relationships that will be used for modeling the loads and deflections.

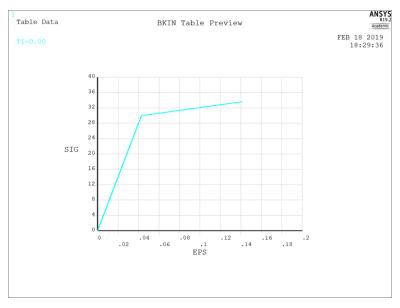


Fig. C4.1

C4: Deflected Shape of Beam

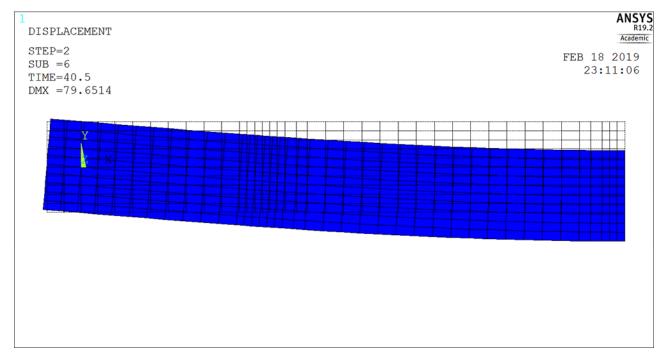


Fig. C4.2

C4 : Contour Plot of Plastic Strains (ϵ_x)

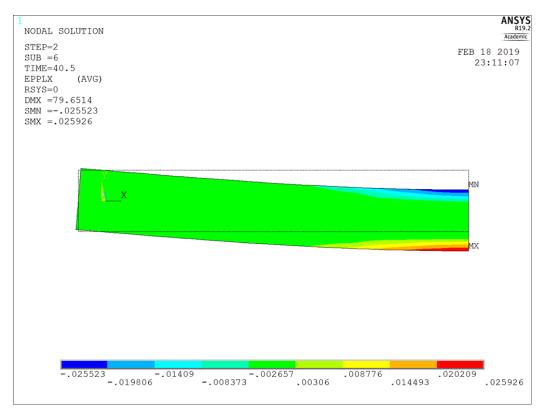


Fig. C4.3

C4: Distribution of Total Strain at Midspan

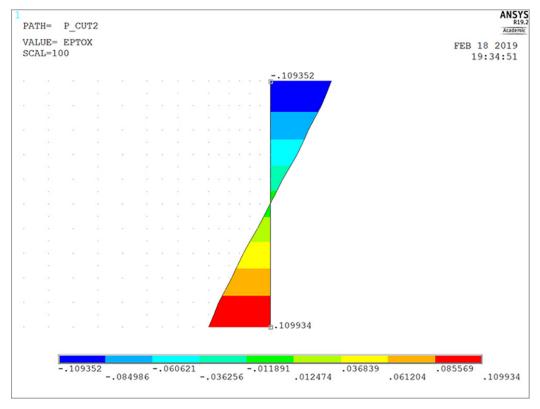


Fig. C4.4

Appendix: Convergence Studies - Case 1

Mesh 0

ALL=1

Mesh 1

LG=5, SM=1

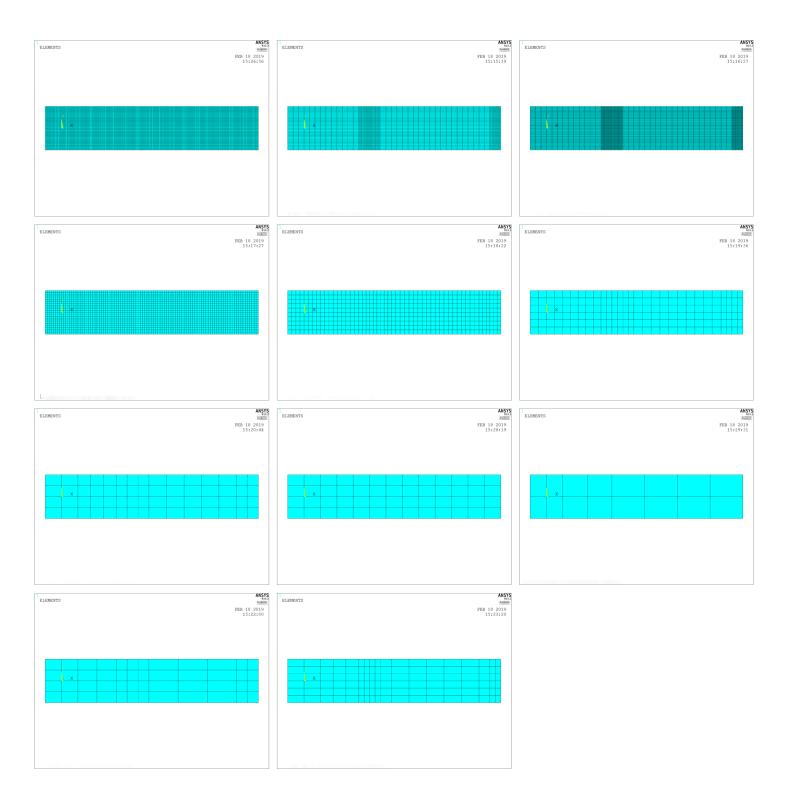
Case 1

The initial mesh that was used during the building of the model was with everything sized at 1mm. When you refine a mesh by level 1 it reduces the size by 1/2. I applied this methodology and just rebuilt the models each time and increasing or reducing the size by 1/2. Since It was too many nodes to refine smaller than 1, I created a multi-zone area for the model in order to refine around the cuts more. Once I determined that there was minimal impact by refining the mesh smaller, I decided to take a different approach and increase the mesh size until there was a substantial difference. After I found this new baseline, I then proceeded to reduce the mesh size again until there was less than .01% change.

Mesh 2

LG=5,SM=.5

| δ_{y} at Q | -30.01732 | -30.01716 | | -30.01702 | | | | | | | | | | | | | |
|---|-----------|------------|----|-----------|---------|-----------|-----------|-----------|----------|-----------|-----------|-----------|----|----------|---|---------|--|
| δ_{y} at Cut 1 | -15.24462 | -15.24446 | | -15.24433 | | | | | | | | | | | | | |
| δ_{max} | 30.1717 | 30.1716 | | 30.1715 | | | | | | | | | | | | | |
| σ_{max} | 30.4749 | 30.4749 | | 30.475 | | | | | | | | | | | | | |
| | | M1/M0 | | M2/M1 | | | | | | | | | | | | | |
| δ_{y} at Q Diff | | -0.00052% | - | 0.00046% | | | | | | | | | | | | | |
| δ_y at C1 Diff | | -0.00102% | - | 0.00090% | | | | | | | | | | | | | |
| $\delta_{	ext{max}}$ Diff | | -0.00033% | - | 0.00033% | | | | | | | | | | | | | |
| $\sigma_{\scriptscriptstyle{max}}$ Diff | | 0.00000% | (| 0.00033% | | | | | | | | | | | | | |
| Case 1 | Mesh 0 | Mesh 3 | | Mesh 4 | | Mesh 5 | | Mesh 6 | | Mesh 7 | | Mesh 8 | | | | | |
| | ALL=1 | LG=2,SM=2 | L | G=4,SM=4 | LC | G=8,SM=8 | LG: | =16,SM=16 | | ALL=16 | | ALL=32 | | | | | |
| $\delta_{_{y}}$ at Q | -30.01732 | -30.01732 | - | 30.01730 | -(| 30.01710 | -3 | 30.01594 | -3 | 30.01594 | -3 | 30.01011 | | | | | |
| δ_{y} at Cut 1 | -15.24462 | -15.24462 | - | 15.24461 | | -15.24444 | | -15.24444 | | -15.24444 | | 15.24343 | -1 | 15.24342 | - | 5.24018 | |
| δ_{max} | 30.1717 | 30.1717 | | 30.1717 | 30.1715 | | 30.1715 | | 30.1715 | | 30.1704 | | | 30.1704 | | 30.1645 | |
| σ_{max} | 30.4749 | 30.4744 | | 30.4722 | | 30.4673 | | 30.4668 | | 30.4863 | | 30.5469 | | | | | |
| | | M3/M0 | | M4/M3 | | M5/M4 | | M6/M5 | | M7/M6 | | M8/M7 | | | | | |
| $\delta_{_{_{\boldsymbol{y}}}}$ at Q Diff | | 0.00000% | - | 0.00005% | -(| 0.00069% | -C | 0.00386% | 0 | .00001% | -0 | -0.01942% | | | | | |
| $\delta_{_y}$ at C1 Diff | | 0.00000% | - | 0.00006% | -(| 0.00113% | -C | 0.00660% | -0 | 0.00007% | -0 | .02123% | | | | | |
| δ_{max} Diff | | 0.00000% | (| 0.00000% | -(| 0.00066% | -0.00365% | | 0.00000% | | -0.01956% | | | | | | |
| σ_{max} Diff | | -0.00164% | - | 0.00722% | -(| 0.01608% | -0 | 0.00164% | 0 | .06400% | (| .19878% | | | | | |
| Case 1 | Mesh 8 | Mesh 9 | | Mesh 10 | | | | | | | | | | | | | |
| | ALL=32 | LG=32, SM= | 16 | LG=16, SM | =8 | | | | | | | | | | | | |
| δ_{y} at Q | -30.01011 | -30.01593 | | -30.0155 | 8 | | | | | | | | | | | | |
| δ_{y} at Cut 1 | -15.24018 | -15.24342 | | -15.24292 | 2 | | | | | | | | | | | | |
| δ_{max} | 30.1645 | 30.1704 | | 30.17 | | | | | | | | | | | | | |
| σ_{max} | 30.5469 | 30.4666 | | 30.4673 | | | | | | | | | | | | | |
| | | M9/M8 | | M10/M9 | | | | | | | | | | | | | |
| δ_{y} at Q Diff | | 0.01940% | | -0.00118° | % | | | | | | | | | | | | |
| δ_y at C1 Diff | | 0.02124% | | -0.003279 | % | | | | | | | | | | | | |
| δ_{max} Diff | | 0.01956% | | -0.001339 | % | | | | | | | | | | | | |
| $\sigma_{\scriptscriptstyle{max}}$ Diff | | -0.26287% |) | 0.00230% | /o | | | | | | | | | | | | |

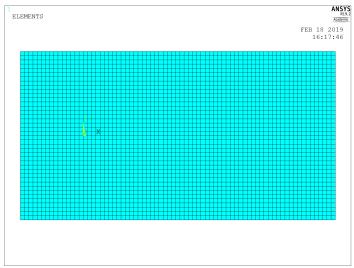


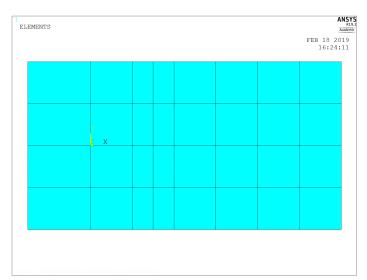
Appendix : Convergence Studies - Case 2

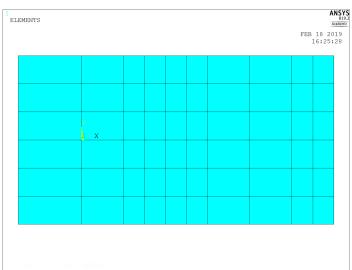
The first thing I checked with Case 2 was to see if the difference between the mesh used with Case 1 base-line and final was a comparable difference if the same two were used for Case 2. Unfortunately they were not close enough for me to accept. I stepped the outcome up to one step above what was used with Case 1, and then started refining from there. I ended up chosing Mesh 4, which now looking at the numbers I have determined it was not adequately descritized.

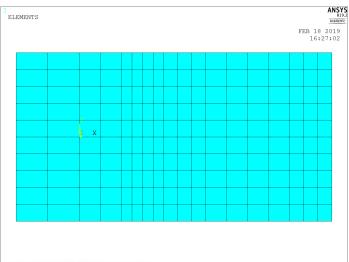
| Case 2 | Mesh 0 | Mesh 10 | | | | |
|----------------------------|----------|-------------|---------------|---------------------|--|--|
| | All=1 | LG=16, SM=8 | | | | |
| $\delta_{_{y}}$ at Q | -1.34568 | -1.34581 | | | | |
| δ_{y} at Cut 1 | -0.69729 | -0.69747 | | | | |
| δ_{max} | 1.40696 | 1.40708 | | | | |
| σ_{max} | 10.4331 | 10.4113 | | | | |
| | | M10/M0 | Case 1 M10/M0 | Difference C1 -> C2 | | |
| δ_y at Q Diff | | 0.00927% | -0.00578% | -0.00349% | | |
| δ_y at C1 Diff | | 0.02639% | -0.01114% | -0.01526% | | |
| δ_{max} Diff | | 0.00853% | -0.00563% | -0.00289% | | |
| $\sigma_{	ext{max}}$ Diff | | -0.20895% | -0.02494% | -0.18401% | | |

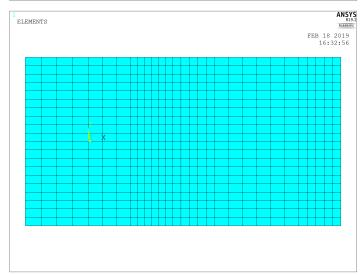
| Case 2 | Mesh 1 | Mesh 2 | Mesh 3 | Mesh 4 | Mesh 5 | |
|----------------------------|--------------|--------------|--------------|--------------|--------------|--|
| | LG=32, SM=16 | LG=16, SM=8 | LG=8, SM=4 | LG=4, SM=2 | LG=2, SM=1 | |
| δ_y at Q | -1.34569473 | -1.345807623 | -1.345680846 | -1.345682763 | -1.345682865 | |
| δ_y at Cut 1 | -0.6973392 | -0.697470642 | -0.697227077 | -0.697292698 | -0.697286571 | |
| δ_{max} | 1.40575 | 1.40708 | 1.40671 | 1.40696 | 1.40696 | |
| σ_{max} | 10.4083 | 10.4113 | 10.4256 | 10.4319 | 10.4331 | |
| | | M2/M1 | M3/M2 | M4/M3 | M5/M4 | |
| δ_y at Q Diff | | 0.00839% | -0.00942% | 0.00014% | 0.00001% | |
| δ_y at C1 Diff | | 0.01885% | -0.03492% | 0.00941% | -0.00088% | |
| δ_{max} Diff | | 0.09461% | -0.02630% | 0.01777% | 0.00000% | |
| $\sigma_{	ext{max}}$ Diff | | 0.02882% | 0.13735% | 0.06043% | 0.01150% | |

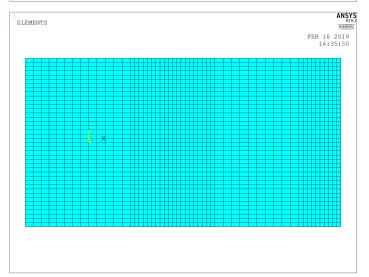








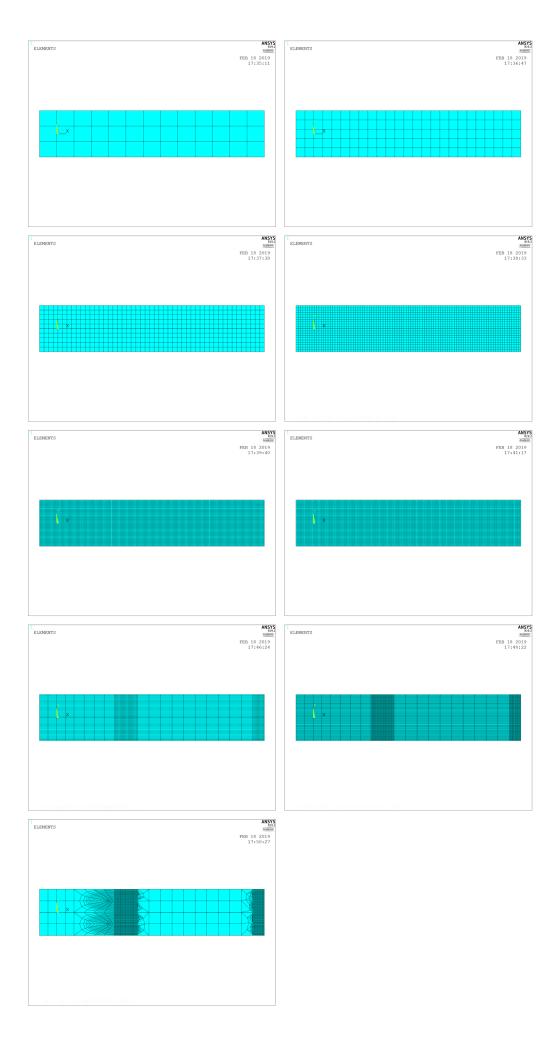




Appendix : Convergence Studies - Case 3

For Case 3 I did not try and compare to Case 1 differences since the measurements and procedures were too different. I started with the same baseline mesh as I did for Case 2. I decided to use substep 4 for the refinement since it was the last step. After a futile effort, I determined substep 4 was too volitile and switched to substep 1. While substep 1 still seem to be too volitile, I relaxed my tolerace and settled for the first refinement that was within 1%

| nement that wa | as within .1%. | | | | | | | | | | |
|---|----------------|---------------|-----------|--------------|-------------|--------------|--------------|---|----------|-----------------|--|
| Case 3 Sub=4 Mesh 1 | | Mesh 2 | | Mesh 3 | | Mesh 4 | | | Mesh 5 | | |
| | LG=32, SM=16 | LG=16, SM=8 | | LG=8, SM=4 | | LG= | LG=4, SM=2 | | G=2, SM= | =1 | |
| Frequency | 7.13497 | 6.99014 | | 6.87927 | | 6. | 6.64242 | | 6.3915 | | |
| δ_{max} | 0.41743 | 0.42480 | 0.424802 | | 0.422222 | | 0.452673 | | 0.555384 | | |
| | | M2/M1 | M2/M1 | | M3/M2 | | M4/M3 | | M5/M4 | | |
| Frequency Diff | | -2.02986 | % | -1.58609% | | -3.4 | -3.44295% | | 3.77754% | <mark>⁄0</mark> | |
| δ_{max} Diff | | 1.765319 | % | -0.607 | 34% | 7.21208% | | 2 | 2.689899 | <mark>%</mark> | |
| Case 3 Sub=4 | Mesh 1a | Mesh 2a | Me | esh 3a | Mesh | 4a | Mesh 5a | a | | | |
| _ | ALL=16 | ALL=8 | А | LL=4 | ALL= | =2 | ALL=1 | | | | |
| Frequency | 7.19015 | 7.08105 | 6.8 | 35946 | 6.806 | 75 | 6.35103 | } | | | |
| δ_{max} | 0.419254 | | 0.4 | 24583 | 0.423935 | | 0.56910 |) | | | |
| | | M2/M1 | М | 3/M2 | M4/N | Л3 | 13 M5/M4 | | | | |
| Frequency Diff | | 1.51735% -3.1 | | 2934% | -0.76843% | | -6.69512 | % | | | |
| δ_{max} Diff | | -0.10137% | 1.37383% | | -0.152 | 62% | 34.24181 | % | | | |
| Case 3 Sub=1 | Mesh 1b | Mesh 2b | Me | esh 3b | Mesh | 4b | Mesh 5k |) | Mesh | 6b | |
| | ALL=16 | ALL=8 | ALL=4 | | ALL= | LL=2 ALL= | | | LG=1, S | M=1 | |
| Frequency | 0.41226 | 0.414618 | 0.4 | 11565 | 0.411193 | | 0.41083 | | 0.410 | 83 | |
| δ_{max} | 0.422626 | 0.423305 | 0.4 | 23576 | 0.422 | 473 | 0.42226 | | 0.4222 | 259 | |
| | | M2/M1 | М | 3/M2 | M4/N | Л3 | M5/M4 | | | | |
| Frequency Diff | | 0.57197% | 0.2 | 4890% | -1.072 | 30% | -0.08828 | % | | | |
| δ_{max} Diff | | 0.16066% | 0.0 | 6402% -0.260 | | 40% | 0% -0.05065% | | | | |
| Case 3 Sub=1 | Mesh 6b | Mesh 7b | | Mesh 8b | | Mesh 9b Free | | | | | |
| | LG=1, SM=1 | LG=10, S | M=1 LG=10 | |), SM=.5 LG | | G=10, SM=.5 | | | | |
| Frequency | 0.41083 | 0.410613 | | 0.409739 | | | 0.412041 | | | | |
| δ_{max} | 0.422259 | 0.422167 | | 0.421793 | | | 0.422767 | | | | |
| | | M7/M | 6 M8/M | | B/M7 | M8/M9 | | | | | |
| Frequency Diff | | -0.05282% | | -0.21285% | | (| 0.56182% | | | | |
| $\delta_{\scriptscriptstyle{max}}$ Diff | | -0.02179% | | -0.08859% | | (| 0.23092% | | | | |



Appendix : Convergence Studies - Case 4

The first thing I checked with Case 4 was to see if the difference between the mesh used with Case 1 base-line and final was a comparable difference if the same two were used for Case 4. While it was very close, the difference was just over my ideal threshold of .01%. After refining the mesh once, it was within the tolerance.

| Case 4 | Mesh 0 | Mesh 10 | | | | | | | | |
|---|-------------|-------------|----|--------------|---------------------|--|---|---|---|--|
| | All=1 | LG=16, SM=8 | | | | | | | | |
| $\delta_{_{y}}$ at Q | -78.20845 | -78.20649 | | | | | | | | |
| δ_y at Cut 1 | -39.65608 | -39.65418 | | | | | | | | |
| δ_{max} | 78.4356 | 78.4336 | | | | | | | | |
| $\sigma_{	ext{max}}$ | 45.0832 | 45.0769 | | | | | | | | |
| | | M10/M0 | Ca | ase 1 M10/M0 | Difference C1 -> C2 | | | | | |
| δ_y at Q Diff | | -0.00250% | | -0.00578% | 0.00327% | | | | | |
| δ_y at C1 Diff | | -0.00479% | | -0.01114% | 0.00634% | | | | | |
| δ_{max} Diff | | -0.00255% | | -0.00563% | 0.00308% | | | | | |
| $\sigma_{\scriptscriptstyle{max}}$ Diff | | -0.01397% | | -0.02494% | 0.01096% | | | | | |
| | " | | | | 31 | | * | " | " | |
| Case 4 | Mesh 10 | Mesh 2 | | | | | | | | |
| | LG=16, SM=8 | LG=8, SM=4 | 4 | | | | | | | |
| δ_y at Q | -78.20649 | -78.20819 | | | | | | | | |
| δ_y at Cut 1 | -39.65418 | -39.65583 | | | | | | | | |

| Case 4 | Mesh 10 | Mesh 2 | | | |
|----------------------------|-------------|------------|--|--|--|
| | LG=16, SM=8 | LG=8, SM=4 | | | |
| δ_y at Q | -78.20649 | -78.20819 | | | |
| δ_{y} at Cut 1 | -39.65418 | -39.65583 | | | |
| δ_{max} | 78.4336 | 78.4353 | | | |
| σ_{max} | 45.0769 | 45.081 | | | |
| | | M10/M0 | | | |
| δ_y at Q Diff | | 0.00218% | | | |
| δ_y at C1 Diff | | 0.00416% | | | |
| δ_{max} Diff | | 0.00217% | | | |
| $\sigma_{	ext{max}}$ Diff | | 0.00910% | | | |

