

Supplementary Material - The Code Lists of Two Case Studies

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Supplementary Material A - Code List of the Bird Function Case Study

The bird function [1] is employed to construct the constraint in the case study in Section 5.1. The continuous constraint satisfaction problem is defined as:

Find all

$$\mathbf{X} = [x_1 \quad x_2], \quad (\text{A1})$$

within

$$-2\pi \leq x_1 \leq 2\pi, \quad (\text{A2})$$

$$-2\pi \leq x_2 \leq 2\pi, \quad (\text{A3})$$

subject to

$$g_1(\mathbf{X}) = (x_1 - x_2)^2 + \sin(x_1) \cdot e^{[1 - \cos(x_2)]^2} + \cos(x_2) \quad (\text{A4})$$

$$\cdot e^{[1 - \sin(x_1)]^2} \leq 0,$$

where x_1 and x_2 are two variables, and $g_1(x)$ is the only inequality constraint function.

The code list used to compute the bounds for the value of the bird function (i.e., the constraint function $g_1(x)$) based on interval arithmetic is provided as follows:

$$G_{1-1} = x_1 - x_2, \quad (\text{A5})$$

$$G_{1-2} = G_{1-1}^2, \quad (\text{A6})$$

$$G_{1-3} = \cos x_2, \quad (\text{A7})$$

$$G_{1-4} = 1 - G_{1-3}, \quad (\text{A8})$$

$$G_{1-5} = G_{1-4}^2, \quad (\text{A9})$$

$$G_{1-6} = e^{G_{1-5}}, \quad (\text{A10})$$

$$G_{1-7} = \sin x_1, \quad (\text{A11})$$

$$G_{1-8} = G_{1-7} \times G_{1-6}, \quad (\text{A12})$$

$$G_{1-9} = 1 - G_{1-8}, \quad (\text{A13})$$

$$G_{1-10} = G_{1-9}^2, \quad (\text{A14})$$

$$G_{1-11} = e^{G_{1-10}}, \quad (\text{A15})$$

$$G_{1-12} = G_{1-3} \times G_{1-11}, \quad (\text{A16})$$

$$G_{1-13} = G_{1-2} + G_{1-8}, \quad (\text{A17})$$

$$g_1 = G_{1-12} + G_{1-13}. \quad (\text{A18})$$

The code list defined by Eqs. (A5) - (A18) is used in the PITSA software. Notably, the code list of the bird function is not unique, and a different code list could be generated if necessary. Since subscripts and hyphens cannot be included in the symbolic names of variables in Python, G_{1-j} in Eqs. (A5) - (A18) is defined as $G1_j$ in the PITSA software, where $j \in \{1, 2, \dots, 13\}$. For example, G_{1-1} is defined as $G1_1$ in the PITSA software.

Supplementary Material B - Code Lists of Welded Beam Design Case Study

The welded beam design problem [2-4] is employed as the case study in Section 5.2. As illustrated in Figure B1, a rectangular beam is designed to be welded onto a primary structure. The design variables are defined as:

$$\mathbf{X} = [x_1 \quad x_2 \quad x_3 \quad x_4], \quad (\text{B1})$$

where x_1 is the weld height, x_2 is the length of the beam that is welded onto the primary structure, x_3 is the height of the beam, and x_4 is the thickness of the beam.

The allowable ranges of these design variables are defined as follows:

$$0.001 \text{ in} \leq x_1 \leq 1 \text{ in}, \quad (\text{B2})$$

$$0.001 \text{ in} \leq x_2 \leq 8 \text{ in}, \quad (\text{B3})$$

$$5 \text{ in} \leq x_3 \leq 30 \text{ in}, \quad (\text{B4})$$

$$0.001 \text{ in} \leq x_4 \leq 1 \text{ in}. \quad (\text{B5})$$

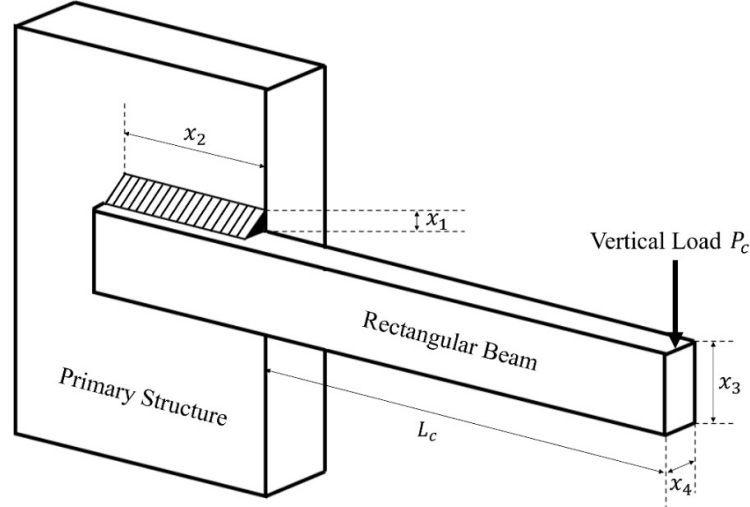


Fig. B1 Welded beam design problem

The design problem includes eight inequality constraints defined by Eqs. (6) - (23). Six constants involved in the welded beam design problem are defined in Table B1.

Table B1 Six constants involved in the welded beam design problem

	Definition	Value
P_c	The vertical load applied on the beam	6,000 pounds
L_c	Distance from load to the point of support	14 inches
E_c	Young's modulus of the beam material	30×10^6 psi
G_c	Shear modulus of the beam material	12×10^6 psi
S_1	Weld material cost per unit volume	0.10471 dollar/in ³
S_2	Beam material cost per unit volume	0.04811 dollar/in ³

1. The weld shear stress is less than or equal to 13,600 psi.

$$g_1(\mathbf{X}) = \tau(\mathbf{X}) - 13600 \leq 0, \quad (\text{B6})$$

$$\tau(\mathbf{X}) = \sqrt{(\tau')^2 + 2\tau'\tau''\frac{x_2}{2R} + (\tau'')^2}, \quad (\text{B7})$$

where τ' is the primary stress acting over the weld throat area defined by Eq. (B8), and τ'' is the secondary torsional stress defined by Eqs. (B9) - (B12).

$$\tau' = \frac{P_c}{\sqrt{2}x_1x_2}, \quad (\text{B8})$$

$$\tau'' = \frac{MR}{J}, \quad (\text{B9})$$

$$M = P_c \left(L_c + \frac{x_2}{2} \right), \quad (\text{B10})$$

$$R = \sqrt{\frac{x_2^2}{4} + \left(\frac{x_1 + x_3}{2} \right)^2}, \quad (\text{B11})$$

$$J = 2\sqrt{2}x_1x_2 \left[\frac{x_2^2}{12} + \left(\frac{x_1 + x_3}{2} \right)^2 \right]. \quad (\text{B12})$$

2. The normal stress in the beam is less than or equal to 30,000 psi.

$$g_2(\mathbf{X}) = \sigma(\mathbf{X}) - 30000 \leq 0, \quad (\text{B13})$$

$$\sigma(\mathbf{X}) = \frac{6P_cL_c}{x_4x_3^2}. \quad (\text{B14})$$

3. The weld height cannot exceed beam thickness.

$$g_3(\mathbf{X}) = x_1 - x_4 \leq 0. \quad (\text{B15})$$

4. The overall material cost should not exceed \$5.

$$g_4(\mathbf{X}) = S_1 x_1^2 x_2 + S_2 x_3 x_4 (L_c + x_2) - 5 \leq 0. \quad (\text{B16})$$

5. The weld height should be larger than or equal to 0.125 inches.

$$g_5(\mathbf{X}) = 0.125 - x_1 \leq 0. \quad (\text{B17})$$

6. The maximum allowable beam-end deflection is 0.25 inches.

$$g_6(\mathbf{X}) = \delta(\mathbf{X}) - 0.25 \leq 0, \quad (\text{B18})$$

$$\delta(\mathbf{X}) = \frac{4P_c L_c^3}{E_c x_3^3 x_4}. \quad (\text{B19})$$

7. The 6,000 lb vertical load does not cause the beam to buckle.

$$g_7(\mathbf{X}) = 6000 - B(\mathbf{X}) \leq 0, \quad (\text{B20})$$

where the beam buckling load, $B(\mathbf{X})$, is derived by

$$B(\mathbf{X}) = \frac{4.013 \sqrt{E_c G_c \frac{x_3^2 x_4^6}{36}}}{L_c^2} \left(1 - \frac{x_3}{2L_c} \sqrt{\frac{E_c}{4G_c}} \right). \quad (\text{B21})$$

8. The overall fabrication cost cannot exceed \$3.5.

$$g_8(\mathbf{X}) = F(\mathbf{X}) - 3.5 \leq 0, \quad (\text{B22})$$

$$F(\mathbf{X}) = (1 + S_1) x_1^2 x_2 + S_2 x_3 x_4 (L_c + x_2). \quad (\text{B23})$$

The code list used to compute the bounds for the value of each constraint function based on interval arithmetic is provided as follows.

Constraint function g_1 :

$$G_{1-1} = x_1 x_2, \quad (\text{B24})$$

$$t_1 = \frac{P_c}{\sqrt{2} G_{1-1}}, \quad (\text{B25})$$

$$G_{1-2} = 0.5 x_2, \quad (\text{B26})$$

$$G_{1-3} = G_{1-2} + L_c, \quad (\text{B27})$$

$$M = G_{1-3}P_c, \quad (\text{B28})$$

$$G_{1-4} = x_2^2, \quad (\text{B29})$$

$$G_{1-5} = 0.25G_{1-4}, \quad (\text{B30})$$

$$G_{1-6} = x_1 + x_3, \quad (\text{B31})$$

$$G_{1-7} = G_{1-6}^2, \quad (\text{B32})$$

$$G_{1-8} = 0.25G_{1-7}, \quad (\text{B33})$$

$$G_{1-9} = G_{1-5} + G_{1-8}, \quad (\text{B34})$$

$$R = \sqrt{G_{1-9}}, \quad (\text{B35})$$

$$G_{1-10} = x_2^2, \quad (\text{B36})$$

$$G_{1-11} = \frac{1}{12}G_{1-10}, \quad (\text{B37})$$

$$G_{1-12} = x_1x_2, \quad (\text{B38})$$

$$G_{1-13} = 2\sqrt{2}G_{1-12}, \quad (\text{B39})$$

$$G_{1-14} = G_{1-11} + G_{1-8}, \quad (\text{B40})$$

$$J = G_{1-14}G_{1-13}, \quad (\text{B41})$$

$$G_{1-15} = M \cdot R, \quad (\text{B42})$$

$$t_2 = \frac{G_{1-15}}{J}, \quad (\text{B43})$$

$$G_{1-16} = t_1^2, \quad (\text{B44})$$

$$G_{1-17} = t_2^2, \quad (\text{B45})$$

$$G_{1-18} = G_{1-16} + G_{1-17}, \quad (\text{B46})$$

$$G_{1-19} = t_1t_2, \quad (\text{B47})$$

$$G_{1-20} = \frac{x_2}{R}, \quad (\text{B48})$$

$$G_{1-21} = G_{1-19}G_{1-20}, \quad (\text{B49})$$

$$G_{1-22} = G_{1-18} + G_{1-21}, \quad (\text{B50})$$

$$t = \sqrt{G_{1-22}}, \quad (\text{B51})$$

$$g_1 = t - 13600. \quad (\text{B52})$$

Constraint function g_2 :

$$G_{2-1} = x_3^2, \quad (\text{B53})$$

$$G_{2-2} = x_4 G_{2-1}, \quad (\text{B54})$$

$$G_{2-3} = \frac{(6P_c L_c)}{G_{2-2}}, \quad (\text{B55})$$

$$g_2 = G_{2-3} - 30000. \quad (\text{B56})$$

Constraint function g_3 :

$$g_3 = x_1 - x_4. \quad (\text{B57})$$

Constraint function g_4 :

$$G_{4-1} = x_1^2, \quad (\text{B58})$$

$$G_{4-2} = G_{4-1} x_2, \quad (\text{B59})$$

$$G_{4-3} = S_1 G_{4-2}, \quad (\text{B60})$$

$$G_{4-4} = x_3 x_4, \quad (\text{B61})$$

$$G_{4-5} = S_2 G_{4-4}, \quad (\text{B62})$$

$$G_{4-6} = L_c + x_2, \quad (\text{B63})$$

$$G_{4-7} = G_{4-5} G_{4-6}, \quad (\text{B64})$$

$$G_{4-8} = G_{4-3} + G_{4-7}, \quad (\text{B65})$$

$$g_4 = G_{4-8} - 5. \quad (\text{B66})$$

Constraint function g_5 :

$$g_5 = 0.125 - x_1. \quad (\text{B67})$$

Constraint function g_6 :

$$G_{6-1} = x_3^3, \quad (\text{B68})$$

$$G_{6-2} = G_{6-1}x_4, \quad (\text{B69})$$

$$G_{6-3} = \frac{1}{G_{6-2}}, \quad (\text{B70})$$

$$G_{6-4} = \left(\frac{4P_c L_c^3}{E_c} \right) G_{6-3}, \quad (\text{B71})$$

$$g_6 = G_{6-4} - 0.25. \quad (\text{B72})$$

Constraint function g_7 :

$$G_{7-1} = x_3^2, \quad (\text{B73})$$

$$G_{7-2} = x_4^6, \quad (\text{B74})$$

$$G_{7-3} = G_{7-1}G_{7-2}, \quad (\text{B75})$$

$$G_{7-4} = \frac{G_{7-3}}{36}, \quad (\text{B76})$$

$$G_{7-5} = \sqrt{G_{7-4}}, \quad (\text{B77})$$

$$G_{7-6} = \left(\frac{4.013\sqrt{E_c G_c}}{L_c^2} \right) G_{7-5}, \quad (\text{B78})$$

$$G_{7-7} = \left(\frac{\sqrt{\frac{E_c}{4G_c}}}{2L_c} \right) x_3, \quad (\text{B79})$$

$$G_{7-8} = 1 - G_{7-7}, \quad (\text{B80})$$

$$G_{7-9} = G_{7-6}G_{7-8}, \quad (\text{B81})$$

$$g_7 = P_c - G_{7-9}. \quad (\text{B82})$$

Constraint function g_8 :

$$G_{8-1} = x_1^2, \quad (\text{B83})$$

$$G_{8-2} = (1 + S_1)x_2, \quad (\text{B84})$$

$$G_{8-3} = G_{8-1}G_{8-2}, \quad (\text{B85})$$

$$G_{8-4} = x_3x_4, \quad (\text{B86})$$

$$G_{8-5} = S_2 G_{8-4}, \quad (\text{B87})$$

$$G_{8-6} = x_2 + L_c, \quad (\text{B88})$$

$$G_{8-7} = G_{8-5} G_{8-6}, \quad (\text{B89})$$

$$G_{8-8} = G_{8-3} + G_{8-7}, \quad (\text{B90})$$

$$g_8 = G_{8-8} - 3.5. \quad (\text{B91})$$

The code lists defined by Eqs. (B24) - (B91) are used in the PITSA software. Notably, each of these code lists is not unique, and a different code list could be generated if necessary. Since subscripts and hyphens cannot be included in the symbolic names of variables in Python, G_{i-j} in Eqs. (B24) - (B91) is defined as G_{i_j} in the PITSA software, where $i \in \{1, 2, \dots, 8\}, j \in \{1, 2, \dots, 22\}$. For example, G_{1-1} is defined as G_{1_1} in the PITSA software.

References

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