

Super-Twisting Control with Quaternion Output Feedback for a 3-DOF Inertial Stabilization Platform

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Abstract—

I. INTRODUCTION

II. ISP MODELING

In this section, a procedure for deriving the kinematic and dynamic models of an ISP installed on a moving base.

A. Notations and Conventions

This section presents the notations and conventions used in this work. Let $\mathbb{N} = \{0, 1, 2, \dots\}$ denote the set of *natural numbers* and define $\tilde{\mathbb{N}} = \{\bar{0}, \bar{1}, \bar{2}, \dots\}$. Unless otherwise stated, $i, j, k \in \mathbb{N} \cup \tilde{\mathbb{N}} \cup \{s, c\}$.

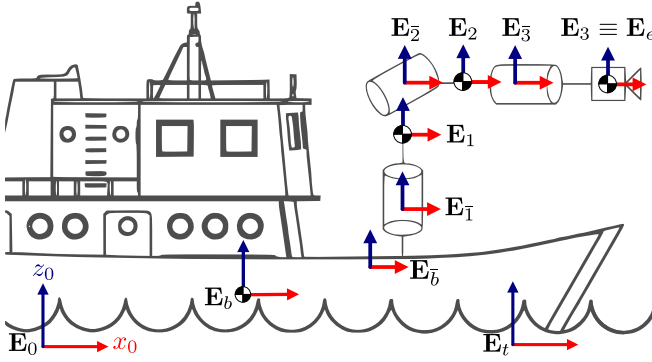


Fig. 1. Frame conventions for a 3-DOF ISP system installed on a vessel.

Considering Fig. 1, define the following:

- \mathbf{E}_w : world frame, arbitrarily located;
- \mathbf{E}_i : frame fixed on body i with origin on its center of gravity (CG) ($i \in \mathbb{N}_0$);
- \mathbf{E}_i : fixed on body i with origin on joint i axis ($i \in \mathbb{N}_0$);
- \mathbf{E}_c : camera frame, fixed on the last link;
- $x_i^k, y_i^k, z_i^k \in \mathbb{R}^3$: \mathbf{E}_i canonical unit vectors, written in \mathbf{E}_k ;
- $p_{ij}^k \in \mathbb{R}^3$: position vector from the origin of frame \mathbf{E}_i to the origin of \mathbf{E}_j , represented in \mathbf{E}_k ;
- $v_{ij}^k \in \mathbb{R}^3$: linear velocity from \mathbf{E}_i to \mathbf{E}_j , written in \mathbf{E}_k ;
- $\omega_{ij}^k \in \mathbb{R}^3$: angular velocity from \mathbf{E}_i to \mathbf{E}_j , written in \mathbf{E}_k ;
- $h_i^k \in \mathbb{R}^3$: unit vector defining the rotation axis of joint i , represented in \mathbf{E}_k ($i \in \mathbb{N}^*$);

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- $m_i \in \mathbb{R}$, $I_i^i \in \mathbb{R}^{3 \times 3}$: mass and inertia tensor of body i represented in \mathbf{E}_i ($i \in \mathbb{N}$);
- $\hat{\cdot}: \mathbb{R}^3 \rightarrow so(3)$ cross-product operator;
- $[v]^\alpha: \mathbb{R}^n \rightarrow \mathbb{R}^n$, where its elements are given by $\|v_i\|^\alpha \text{sgn}(v_i)$, with $v_i \in \mathbb{R}$ ($i = 1, \dots, n$) being the elements of $v \in \mathbb{R}^n$ and $\alpha \in \mathbb{R}$;
- $\mathbf{I}_n \in \mathbb{R}^{n \times n}$: identity matrix of dimension n .

Let body 0 be the vessel and bodies 1, 2, 3 be the ISP gimbals. Also, if a superscript is omitted, the vector is written in \mathbf{E}_w .

B. Quaternion-Based Kinematics

Let $R \in SO(3)$ be a *rotation matrix* describing the rotation from an arbitrary frame to another. Then, R is a diffeomorphism with respect to the projective space $\mathbb{RP}^3 = \{\|v\|^2 \leq \pi \mid v \in \mathbb{R}^3\}$. Therefore, each point $v \in \mathbb{RP}^3$ represent a 4-parameter representation for $SO(3)$ called the *angle-axis*, where the unitary vector on the direction of v represents the rotation axis and $\|v\|$ represents the corresponding rotation angle around that axis.

Remark 1. Note that \mathbb{RP}^3 covers $SO(3)$ twice, since any point on it actually represents the same rotation than the opposite point of the sphere.

This representation can be expressed by $v = \{\theta, n\}$, where $\theta \in \mathbb{R}$ is the angle of rotation around the unit axis vector $n \in \mathbb{R}^3$, $\|n\| = 1$. Another non-minimal representation is the *unit quaternion*. The set of *quaternions* \mathbb{H} is defined by:

$$\mathbb{H} := \{\eta + i\epsilon_1 + j\epsilon_2 + k\epsilon_3 \mid \eta, \epsilon_1, \epsilon_2, \epsilon_3 \in \mathbb{R}\}, \quad i^2 = j^2 = k^2 = ijk = -1. \quad (1)$$

A quaternion $Q \in \mathbb{H}$ can also be represented as the pair $Q := \{\eta, \epsilon\}$, where $\eta = \text{Re}(Q) \in \mathbb{R}$ represents the *real* part of the quaternion and $\epsilon = \text{Im}(Q) = [\epsilon_1 \ \epsilon_2 \ \epsilon_3]^T \in \mathbb{R}^3$ represents the vector part. The quaternion *conjugate* is given by $Q^* = \{\eta, -\epsilon\}$. One can also represent the quaternion in fully vector form by the notation $\bar{Q} = [\eta \ \epsilon_1 \ \epsilon_2 \ \epsilon_3]^T \in \mathbb{R}^4$.

Quaternions also form an algebraic *group* with respect to *multiplication*. Given two quaternions $Q_1 = \{\eta_1, \epsilon_1\}$ and $Q_2 = \{\eta_2, \epsilon_2\}$, their multiplication follow the rules established by definition (1), which results in:

$$Q_1 \circ Q_2 = \{\eta_1\eta_2 - \epsilon_1^T \epsilon_2, \eta_1\epsilon_2 + \eta_2\epsilon_1 + \epsilon_1 \times \epsilon_2\}. \quad (2)$$

Quaternion multiplication can also be performed as a linear transformation in \mathbb{R}^4 , by:

$$\overline{Q_1 \circ Q_2} = \mathbf{H}_+(Q_1) \overline{Q_2}, \quad (3)$$

$$= \mathbf{H}_-(Q_2) \overline{Q_1}, \quad (4)$$

where \mathbf{H}_+ , \mathbf{H}_- are Hamilton operators defined by:

$$\mathbf{H}_+(Q) = \begin{bmatrix} \eta & -\epsilon^\top \\ \epsilon & \eta \mathbf{I}_3 + \hat{\epsilon} \end{bmatrix}, \quad (5)$$

$$\mathbf{H}_-(Q) = \begin{bmatrix} \eta & -\epsilon^\top \\ \epsilon & \eta \mathbf{I}_3 - \hat{\epsilon} \end{bmatrix}. \quad (6)$$

The square of the quaternion *norm* is defined as the *scalar*:

$$\|Q\|^2 = Q \circ Q^* = \{\eta^2 + \epsilon^\top \epsilon, 0\}, \quad (7)$$

and its *inverse* is the quaternion Q^{-1} such that $Q \circ Q^{-1} = \{1, 0\}$, the *unitary* quaternion.

The set of *unit quaternions* $\mathbb{H}^* = \{Q \in \mathbb{R} : \|Q\| = 1\}$ can be used as a parametrization for orientation in the following way. For an element $p = \{\theta, n\} \in \mathbb{RP}$, define:

$$Q = \left\{ \cos\left(\frac{\theta}{2}\right), \sin\left(\frac{\theta}{2}\right) n \right\} \in \mathbb{H}^*. \quad (8)$$

The inverse of an unit quaternion is given by $Q^{-1} = Q^*$, which according to (8), clearly corresponds to the opposite rotation due to negative direction of the rotation axis n .

Let $r_0, r_1, \dots, r_n \in \mathbb{H}^*$ be the n absolute rotations between frames $\mathbf{E}_0, \mathbf{E}_1, \dots, \mathbf{E}_n$ and the world frame \mathbf{E}_w , and $r_{i+1}^i \in \mathbb{H}^*$ ($i = 1, 2, \dots, n-1$) represent the rotations from frame \mathbf{E}_i to \mathbf{E}_{i+1} . Then, since the unit quaternions form a group with respect to multiplication, then:

$$r_n = r_1 \circ r_2^1 \circ \dots \circ r_n^{n-1} \in \mathbb{H}^*. \quad (9)$$

Now, define the set of *pure* quaternions $\mathbb{H}_p = \{v \in \mathbb{H} : \text{Re}(v) = 0\}$. Note that any vector from \mathbb{R}_3 can be represented as the vector part of a corresponding element $v \in \mathbb{H}_p$. Besides, note that the following holds for $v, w \in \mathbb{H}_p$:

$$v \circ w = \{-\text{Im}(v)^\top \text{Im}(w), \text{Im}(v) \times \text{Im}(w)\}. \quad (10)$$

Let v^i and $v^j \in \mathbb{H}_p$ be representations for a vector \vec{v} in frames \mathbf{E}_i and \mathbf{E}_j , respectively, and r_j^i represents the rotation from \mathbf{E}_i to \mathbf{E}_j , with unitary axis $n_j^i \in \mathbb{R}^3$ and rotation angle θ_{ij} . Then, the following relation is valid:

$$v^i = (r_j^i) \circ v^j \circ (r_j^i)^* = \text{Ad}_{r_j^i} [v^j], \quad (11)$$

where $\text{Ad}_{r_j^i}[\cdot]$ is the adjoint operator. Note that, in vector algebra, $\text{Ad}_{r_j^i}$ represents the corresponding rotation matrix $R_{ij} \in \text{SO}(3)$ associated to the unit quaternion $r_j^i \in \mathbb{H}^*$. In terms of the components of r_j^i , this matrix is given by:

$$R_{ij} = n_j^i (n_j^i)^\top + s_{ij} \mathbf{S}(n_j^i) + c_{ij} (\mathbf{I}_3 - n_j^i (n_j^i)^\top), \quad (12)$$

where s_{ij} and c_{ij} are the sine and cosine functions of θ_{ij} . The rotation matrix corresponding to an absolute rotation $r_i \in \mathbb{H}^*$ is written with only one subscript, as $R_i \in \text{SO}(3)$.

Algorithm 1 (Kinematic Propagation). *The algorithm is initialized with the vessel configuration $p_{00} \in \mathbb{R}^3$ and $r_0 \in \mathbb{H}^*$. Then, varying index i from 0 to $n-1$, the configuration of each frame with respect to the vessel frame \mathbf{E}_0 can be computed by:*

$$p_{0,i+1} = p_{0,i} + R_i p_{i,i+1}^i, \quad (13)$$

$$\bar{r}_{i+1} = \mathbf{H}_+(r_i) \bar{r}_{i+1}^i. \quad (14)$$

with R_i computed from r_i in (12). The camera pose can be computed as $p_{0c} = p_{0n} + R_c p_{nc}^n$, $r_c = \mathbf{H}_+(r_n) \bar{r}_c^n$.

Let \vec{v}_i and $\vec{\omega}_i$ be the physical linear and angular velocities of \mathbf{E}_i . They are represented by $v_i^i \in \mathbb{R}^3$ and $\omega_i^i \in \mathbb{R}^3$ when written in its own body frame. Let $r_i = \{\eta_i, \epsilon_i\} \in \mathbb{H}^*$ be the absolute rotation of \mathbf{E}_i . The time-derivative of r_i can be related to ω_i^i by:

$$\dot{r}_i = \begin{bmatrix} \dot{\eta}_i \\ \dot{\epsilon}_i \end{bmatrix} = \frac{1}{2} \begin{bmatrix} -\epsilon_i^\top \\ \eta_i \mathbf{I}_3 + \hat{\epsilon}_i \end{bmatrix} \omega_i^i, \quad (15)$$

which is known as the *quaternion propagation* formula.

The vector $V_i^i = [(v_i^i)^\top (\omega_i^i)^\top]^\top \in \mathbb{R}^6$ is the *body velocity twist* associated to \mathbf{E}_i . Two body velocity twists associated to different frames $\mathbf{E}_i, \mathbf{E}_j$ located in the *same rigid-body* are related through the constant adjoint map $\text{Ad}_{g_{ij}} \in \mathbb{R}^{6 \times 6}$:

$$V_i^i = \text{Ad}_{g_{ij}} V_j^j, \quad \text{Ad}_{g_{ij}} = \begin{bmatrix} R_{ij} & \hat{p}_{ij} R_{ij} \\ 0 & R_{ij} \end{bmatrix}. \quad (16)$$

which has the property $\text{Ad}_{g_{ji}} = \text{Ad}_{g_{ij}}^{-1}$.

Now, given the body twists V_0^0 and \dot{V}_0^0 of frame \mathbf{E}_0 in the ship, it is possible to compute all velocities V_i^i and accelerations \dot{V}_i^i associated to each link ($i = 1, 2, 3$) by means of an iterative algorithm described below. It consists in propagating the body velocity/acceleration twists of each link frame \mathbf{E}_i through the system, obtaining $V_i^i, \dot{V}_i^i, i \in \{1, 2, 3\}$.

Algorithm 2 (Propagation of Velocities and Accelerations). *The algorithm is initialized with given V_0^0, \dot{V}_0^0 . Then, the velocities and accelerations are propagated upwards the kinematic chain from $i = 0$ until $i = n = 3$:*

$$V_i^i = \Omega_{i-1,i}^\top (\Phi_{i,i-1} V_{i-1}^{i-1} + H_i \dot{q}_i), \quad (17)$$

$$\dot{V}_i^i = \Omega_{i-1,i}^\top (\Phi_{i,i-1} \dot{V}_{i-1}^{i-1} + H_i \ddot{q}_i + A_i \dot{q}_i). \quad (18)$$

The velocity/acceleration twists of the camera are computed by $V_c^c = \text{Ad}_{g_{cn}} V_n^n, \dot{V}_c^c = \text{Ad}_{g_{cn}} \dot{V}_n^n$, with a constant transformation g_{cn} . The matrices in (17), (18) are given by:

$$\begin{aligned} \Phi_{i+1,i} &= \begin{bmatrix} \mathbf{I}_3 & -\hat{p}_{i,i+1} \\ 0 & \mathbf{I}_3 \end{bmatrix}, & \Phi_{i+1,i}^{-1} &= \begin{bmatrix} \mathbf{I}_3 & \hat{p}_{i,i+1} \\ 0 & \mathbf{I}_3 \end{bmatrix}, \\ H_{i+1}^\top &= [0^\top (h_{i+1}^i)^\top], & \Omega_{i,i+1} &= \begin{bmatrix} R_{i,i+1} & 0 \\ 0 & R_{i,i+1} \end{bmatrix}, \\ A_{i+1} &= \begin{bmatrix} (\hat{v}_{0i}^i + \widehat{\omega_{0i}^i p_{i,i+1}^i}) h_{i+1}^i \\ \widehat{\omega_{0i}^i} h_{i+1}^i \end{bmatrix}, \end{aligned}$$

where the rotation matrices $R_{i,i+1}$ are computed from $r_{i,i+1} = \{\cos(\frac{1}{2}q_{i+1}), h_{i+1}^i \sin(\frac{1}{2}q_{i+1})\} \in \mathbb{H}^*$ using (12).

Now, recall that ω_i^i can be written as the sum $\omega_i^i = \omega_0^i + \omega_{0,i}^i$ and can be expressed in terms of $q, \dot{q} \in \mathbb{R}^3$ by means of the *angular body link Jacobian* $J_{0i}^i(q, \Pi_g) \in \mathbb{R}^{3 \times 3}$ as $\omega_{0,i}^i = J_{0i}^i(q, \Pi_g) \dot{q}$:

$$\omega_i^i = J_{0i}^i(q, \Pi_g) \dot{q} + \omega_0^i, \quad (19)$$

where Π_g is the vector of *geometric* parameters of the ISP, containing combinations of components of the axes and distance vectors of each link frame. Note that the body link Jacobian $J_{0i}^i(q, \Pi_g)$ can be computed numerically from (17), and also that the last $3-i$ columns of J_{0i}^i are null, since the velocity of link i only depends on the previous joints.

These kinematic relations can be used to describe the dependance among vehicle, ISP and camera motion by applying the group operation of \mathbb{H}^* , equation (19) and its time-derivative, with $\mathbf{E}_i = \mathbf{E}_c$:

$$r_c = r_0 \circ r_c^0(q, \Pi_g), \quad (20)$$

$$\omega_c^c = J_{0c}^c(q, \Pi_g) \dot{q} + \omega_0^c, \quad (21)$$

$$\dot{\omega}_c^c = J_{0c}^c(q, \Pi_g) \dot{q} + \dot{J}_{0c}^c(q, \Pi_g) \dot{q} + \dot{\omega}_0^c. \quad (22)$$

C. Dynamic Equations for Vehicle Manipulator Systems

In [1], it is shown that the equations of motion for a VMS with respect to the vehicle CG frame \mathbf{E}_b can be written as:

$$M_{qq} \ddot{q} + C_{qq} \dot{q} + G_q + M_{qV} \dot{V}_0^0 + C_{qV}^b V_0^0 = \tau_q, \quad (23)$$

where $\tau_q \in \mathbb{R}^n$ is the vector of generalized forces acting on the robot joints, collocated with \dot{q} . Matrices $M_{qq}(q, \Pi_g, \Pi_d) \in \mathbb{R}^{3 \times 3}$ and $M_{qV}(q, \Pi_g, \Pi_d) \in \mathbb{R}^{3 \times 6}$ are mass matrices, $C_{qq}(q, \dot{q}, V_0^0, \Pi_g, \Pi_d) \in \mathbb{R}^{3 \times 3}$ and $C_{qV}(q, \dot{q}, V_0^0, \Pi_g, \Pi_d) \in \mathbb{R}^{3 \times 6}$ are Coriolis matrices and $G_q(q, r_0, \Pi_g, \Pi_d) \in \mathbb{R}^3$ is the gravity vector.

It is well known that the *Newton Euler* method is a computationally efficient algorithm that can be used to numerically solve the *inverse dynamics* problem for (23). Given $q, \dot{q}, \ddot{q}, r_0, V_0^0, \dot{V}_0^0$ and $g \in \mathbb{R}$, the *Newton-Euler* algorithm for the inverse dynamics is expressed by:

$$\tau_q = NE(q, \dot{q}, \ddot{q}, r_0, V_0^0, \dot{V}_0^0, g, \Pi), \quad (24)$$

where $\Pi^T = [\Pi_g^T \ \Pi_d^T]$ contains combinations of the geometric and dynamic parameters (masses and inertias).

The Newton-Euler algorithm is composed of two steps. The first one is the *propagation of velocities and accelerations* upwards the kinematic chain, summarized in Algorithm 2. The second one consists in solving the dynamic equations of motion for each rigid body in the system, starting from the n -th link and ending up on \mathbf{E}_b .

Algorithm 3 (Backward Propagation of Wrenches). *Solving the Newton-Euler equations for the contact body wrenches*

$F_i^i \in \mathbb{R}^6$ between the VMS bodies (links and vehicle), yields:

$$F_i^i = \Phi_{i+1,i}^T \Omega_{i,i+1} F_{i+1}^{i+1} + M_i \dot{V}_{0i}^i + B_i, \quad (25)$$

$$M_i = \begin{bmatrix} m_i \mathbf{I}_3 & -m_i \hat{p}_{ii}^i \\ m_i \hat{p}_{ii}^i & I_{ii}^i \end{bmatrix},$$

$$B_i = \begin{bmatrix} m_i \hat{\omega}_i^i (\hat{\omega}_i^i \hat{p}_{ii}^i + v_i^i) \\ m_i \hat{p}_{ii}^i \hat{\omega}_i^i v_i^i + \hat{\omega}_i^i I_{ii}^i \hat{\omega}_i^i \end{bmatrix},$$

where the parameters \hat{p}_{ii}^i , m_i and I_{ii}^i compose Π_d in (24).

These equations must be solved from $i = n = 3$ to $i = 1$, using the velocity and acceleration twists V_i^i and \dot{V}_i^i previously computed in Algorithm 2. Also, we set here $n+1 = c$ and we do not consider external wrenches acting on the camera frame \mathbf{E}_c , so that $F_{n+1}^{n+1} = 0$.

Finally, the joint torques can be computed projecting the wrenches acting on frames \mathbf{E}_i into their rotation axis by:

$$\tau_{q_i} = H_i^T \Omega_{i-1,i} F_i^i. \quad i = 1, \dots, n. \quad (26)$$

Note that (25) does not take into account the gravity forces acting on the links. The effect of gravity (in $-z_0$ direction) is introduced by modifying \dot{V}_i^i in (25) for each i -th link with:

$$\dot{V}_i^i \leftarrow \dot{V}_i^i - g \begin{bmatrix} R_i^T z_0 \\ 0 \end{bmatrix}. \quad (27)$$

This algorithm can be used to compute the terms and some matrices of (23) separately: the mass matrices M_{qq} , M_{qV} , the gravity vector G_q and the Coriolis term $C_{qq} \dot{q} + C_{qV} V_0^0$.

III. SUPER-TWISTING CONTROL

A. Super-Twisting Control with Full State Feedback

In this section, a novel second-order sliding mode (SOSM) controller based on super-twisting algorithm (STC) will be developed for the stabilization and tracking of the LOS.

Consider (23), rewritten here as:

$$M_{qq} \ddot{q} + \tau_d = \tau_q, \quad (28)$$

where $\tau_q \in \mathbb{R}^3$ is the vector of generalized forces acting on the robot joints, collocated with \dot{q} and $\tau_d = C_{qq} \dot{q} + G_q + M_{qV} \dot{V}_{0b}^b + C_{qV} V_{0b}^b \in \mathbb{R}^3$ is a disturbance vector. The dynamic model (28) can be rewritten as:

$$\dot{x}_1 = x_2, \\ \dot{x}_2 = M_{qq}^{-1}(x_1, \Pi) \tau_q + x_3(x_1, x_2, \Pi, t), \quad (29)$$

where the states $x_1 = q$, $x_2 = \dot{q}$ are the ISP joint angles and velocities and $x_3 = -M_{qq}^{-1}(x_1) \tau_d$ is a state-dependent disturbance.

Remark 2. Note that, under assumption of torque control $u(t) = \tau_q$, state-space model (29) is a double-integrator with a nonlinear high-frequency gain and a matched disturbance $x_3 \in \mathbb{R}^3$.

First, define the following *sliding surface*:

$$s_x = x_2 - \int_0^t w(\tau) d\tau. \quad (30)$$

where $w(\tau) \in \mathbb{R}^3$ is an arbitrary signal. Due to (29), the dynamics of the sliding variable s_x is given by:

$$\dot{s}_x = \dot{x}_2 - w(t) = M_{qq}^{-1} u + x_3 - w(t). \quad (31)$$

The *super-twisting control law* is defined as:

$$S_t(s, A, B) = A[s]^{1/2} + B \int_0^t \text{sgn}(s) d\tau, \quad (32)$$

where matrices $A, B > 0$. Using the inner control law

$$u(t) = M_{qq}(x_1, \Pi) [w(t) - S_t(s_x, \Lambda_1, \Lambda_2)], \quad (33)$$

where $\Lambda_1, \Lambda_2 > 0$, (31) becomes:

$$\begin{aligned} \dot{s}_x &= -\Lambda_1 [s_x]^{1/2} + w_x, \\ \dot{w}_x &= -\Lambda_2 [s_x]^0 + \dot{x}_3, \end{aligned} \quad (34)$$

which is identified as the STA and is finite-time stable for $\|\dot{x}_3\| < L_x$. According to [?], the STA guarantees SOSM in finite-time, which means that after a time $t > T_1 > 0$, $s_x = \dot{s}_x = 0$. Therefore, $\dot{x}_2 = w(t) \forall t > T_1$, even in the presence of the bounded disturbance x_3 .

Now, in a similar way than in (29), (15) and (22) can be rewritten as:

$$\dot{y}_1 = \frac{1}{2} \begin{bmatrix} -y_{12}^T \\ y_{11} \mathbf{I}_3 + \hat{y}_{12} \end{bmatrix} y_2, \quad (35)$$

$$\dot{y}_2 = J_{0c}^c(x_1, \Pi_g) \dot{x}_2 + y_3(x_1, x_2, \Pi_g, t). \quad (36)$$

where the state $y_1^T = \bar{r}_c^T = [y_{11} \ y_{12}^T]$ is the vector representation of the camera orientation $r_c \in \mathbb{H}^*$, with $y_{11} = \eta_c$ and $y_{12} = \epsilon_c$ being the scalar and vector components. State $y_2 = \omega_c^c$ is the camera angular velocity, while $y_3 = \hat{J}_{0c}^c \dot{q} + \dot{\omega}_0^c$ is another state-dependent disturbance.

Remark 3. Note that the state-space model (36) is a double integrator with a nonlinear high-frequency gain and a matched disturbance y_3 with respect to a control input \dot{x}_2 .

This structure strongly suggests the use of a *cascade controller* for both stabilization and tracking.

Given an orientation reference $r_{cd}(t) \in \mathbb{H}^*$ for the camera, it can be represented in vector form by $\bar{r}_{cd}^T(t) = y_{1d}^T(t) = [y_{11d}(t) \ y_{12d}^T(t)]$. The angular velocity of the camera is also given as $\omega_{cd}^c(t) = y_{2d}(t)$. The quaternion and angular velocity errors can be defined as:

$$e_c = r_{cd}(t) \circ r_c^*, \quad (37)$$

$$e_\omega = y_{2d}(t) - y_2. \quad (38)$$

Note that when $r_c = r_{cd}(t)$, the orientation error (37) is zero. Define another *sliding surface* as

$$s_y = e_\omega + K_c \text{Im}(e_c), \quad K_c > 0, \quad (39)$$

whose dynamics is given by

$$\dot{s}_y = \dot{y}_{2d} - J_{0c}^c(x_1) \dot{x}_2 - y_3 + K_c \psi(y_1, y_2, r_{cd}), \quad (40)$$

with $\psi(y_1, y_2, r_{cd}) = y_{11} \dot{y}_{12d} - 0.5 y_{12}^T y_2 y_{12d} - \dot{y}_{11d} y_{12} - \hat{y}_{12d} y_{12} - 0.5 y_{11d} (y_{11} \mathbf{I}_3 - \hat{y}_{12}) y_2 - 0.5 \hat{y}_{12d} (y_{11} \mathbf{I}_3 - \hat{y}_{12}) y_2$ and the derivatives of the reference are $\dot{y}_{11d} = -0.5 y_{12}^T y_{2d}$ and $\dot{y}_{12d} = 0.5(y_{11} \mathbf{I}_3 - \hat{y}_{12}) y_{2d}$.

Assumption 1. Here, suppose that the system (34) is already sliding after a finite time $T_1 > 0$. Therefore, $\dot{x}_2 = w(t)$. Under this assumption, (40) becomes

$$\dot{s}_y = \dot{y}_{2d}(t) - J_{0c}^c(x_1, \Pi_g) w(t) - y_3 + K_c \psi(y_1, y_2, r_{cd}).$$

Under Assumption 1, the *outer control law*:

$$w(t) = J_{0c}^c(x_1, \Pi)^{-1} [\dot{y}_{2d}(t) + K_c \psi + S_t(s_y, \Lambda_3, \Lambda_4)]. \quad (41)$$

can be applied to (40), yielding:

$$\begin{aligned} \dot{s}_y &= -\Lambda_3 [s_y]^{1/2} + w_y, \\ \dot{w}_y &= -\Lambda_4 \text{sign}(s_y) - \dot{y}_3, \end{aligned} \quad (42)$$

once again guaranteeing finite-time stabilization after a time $T_2 > 0$ and under the assumption of $\|\dot{y}_3\| < L_y$. It means that for all $t \geq T_2$, the system is sliding and therefore, it follows the nonlinear dynamics of the sliding variable (39), which is asymptotically stable according to [2].

Theorem 1.

Proof. □

B. HOSMO Observer for STC with Output Feedback

If all system states $q, \dot{q} \in \mathbb{R}^3$ are not available, but only the joint angles q , a second order differentiator can be used to estimate \dot{q} . It was introduced by Levant in his seminal work [1]. This system is a third-order observer based on the theory of HOSMs.

Defining the estimation error $e_{x_1} = x_1 - \hat{x}_1$, the HOSM observer for the second order nonlinear system (29) is:

$$\begin{aligned} \hat{\dot{x}}_1 &= k_1 [e_{x_1}]^{2/3} + \hat{x}_2, \\ \hat{\dot{x}}_2 &= k_2 [e_{x_1}]^{1/3} + \hat{x}_3 + M_{qq}^{-1}(x_1, \Pi) u, \\ \hat{\dot{x}}_3 &= k_3 \text{sgn}(e_{x_1}). \end{aligned} \quad (43)$$

Its already proven by literature [?] that after a finite time $T_{reach} > 0$, the states of the second-order differentiator (43) uniformly converge to $\hat{x}_1 = x_1$, $\hat{x}_2 = \dot{x}_2$ and $\hat{x}_3 = \ddot{x}_2$, given that $\|\dot{x}_3\| < L_x$.

Defining the *modified sliding surface*

$$\hat{s}_x = \hat{x}_2 - \int_0^t w(\tau) d\tau, \quad (44)$$

its dynamics is given by:

$$\dot{\hat{s}}_x = k_2 [e_{x_1}]^{1/3} + \hat{x}_3 + M_{qq}^{-1}(x_1, \Pi) u - w(t). \quad (45)$$

Modifying control law (33) to

$$u(t) = M_{qq}(x_1, \Pi) \left[w(t) - k_2 [e_{x_1}]^{1/3} - S_t(\hat{s}_x, \Lambda_1, \Lambda_2) \right], \quad (46)$$

the exact same STA than in (34) is obtained, but with s_x replaced by the modified sliding variable \hat{s}_x .

Remark 4. *Another HOSMO could be designed to estimate the camera angular velocity $\omega_c^c(t)$. However, usually the camera orientation $r_c(t)$ is obtained from an Inertial Measurement Unit (IMU), a device that combines measurements of gyroscopes (which measure angular velocity) and magnetometers (which measure magnetic fields), providing an accurate estimate for $r_c(t)$. Therefore, trustworthy direct measurements of $\omega_c^c(t)$ are usually already available.*

Theorem 2.

Proof.

□

IV. SIMULATION RESULTS

V. CONCLUSION AND FUTURE WORKS

REFERENCES

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