

Solutions for Homework 4th.

Q1 解: 如图所示, 取 $Oxyz$ 坐标系固连于刚体, 刚体

(1) 定轴转动的自由度为 1, 取广义坐标为 φ . 考虑

刚体上任一质元 $dm(x, y, z)$, 其受达朗贝尔惯

性力为 $-\vec{a} dm = -[(\ddot{r} - r\dot{\varphi}^2)\hat{e}_r + (r\ddot{\varphi} + 2\dot{r}\dot{\varphi})\hat{e}_\theta] dm$

$\therefore \dot{r} = \ddot{r} = 0$, 而 $r = \sqrt{x^2 + y^2}$, \therefore 质点实际受 $\dot{\varphi}^2 \sqrt{x^2 + y^2} dm \hat{e}_r$ 及

$-\ddot{\varphi} \sqrt{x^2 + y^2} dm \hat{e}_\theta$ 两个惯性力, 只有后者做功不为零, 为

$$\delta W = -\ddot{\varphi} \sqrt{x^2 + y^2} dm \sqrt{x^2 + y^2} \delta \varphi = -\ddot{\varphi} (x^2 + y^2) dm \delta \varphi$$

设主动力对转轴之力矩为 M , 则根据力学普遍性方程有

$$[M - \int \ddot{\varphi} (x^2 + y^2) dm] \delta \varphi = 0$$

$$\Rightarrow M = \ddot{\varphi} \int (x^2 + y^2) dm = I_z \ddot{\varphi}, \text{ 即为所求。}$$

(2) 刚体定轴转动时各质点的位移可设想由绕过固定点 O 的任何轴的一个无限小转动引起, 表示为

$$\delta \vec{r}_i = \delta \vec{\theta} \times \vec{r}_i$$

则由力学普遍性方程, 得

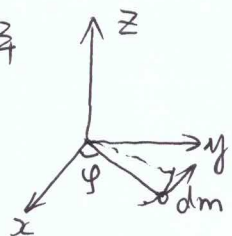
$$\sum_{i=1}^n (\vec{F}_i - m_i \ddot{\vec{r}}_i) \cdot \delta \vec{r}_i = \sum_{i=1}^n (\vec{F}_i - m_i \ddot{\vec{r}}_i) \cdot (\delta \vec{\theta} \times \vec{r}_i)$$

$$= \sum_{i=1}^n \delta \vec{\theta} \cdot \vec{r}_i \times (\vec{F}_i - m_i \ddot{\vec{r}}_i)$$

$$= \delta \vec{\theta} \cdot \left[\sum_{i=1}^n (\vec{r}_i \times \vec{F}_i) - \sum_{i=1}^n (\vec{r}_i \times m_i \ddot{\vec{r}}_i) \right]$$

$$= \delta \vec{\theta} \cdot \left\{ \vec{M} - \left[\frac{d}{dt} \sum_{i=1}^n (\vec{r}_i \times m_i \dot{\vec{r}}_i) - \sum_{i=1}^n \vec{r}_i \times m_i \dot{\vec{r}}_i \right] \right\}$$

$$= \delta \vec{\theta} \cdot \left(\vec{M} - \frac{d\vec{L}}{dt} \right) = 0 \quad \therefore \frac{d\vec{L}}{dt} = \vec{M} \text{ 即为所求。}$$



Q2 解: 选下杆质心的坐标 x_2, y_2 为广义坐标, 另选上杆位置 θ_1 及下杆位置 θ_2 为广义坐标, 由虚功原理:

$$m_1 g \delta x_1 + m_2 g \delta x_2 + F \delta y_3 = 0$$

用广义坐标:

$$m_1 g \delta \left(\frac{1}{2} l_1 \cos \theta_1 \right) + m_2 g \delta x_2 + F \delta \left(\frac{1}{2} l_2 \sin \theta_2 + y_2 \right) = 0$$

$$-\frac{1}{2} l_1 m_1 g \sin \theta_1 \delta \theta_1 + m_2 g \delta x_2 + \frac{1}{2} l_2 \cos \theta_2 F \delta \theta_2 + F \delta y_2 = 0 \quad (a)$$

对应于 δx_2 和 δy_2 的主动力分别是 $m_2 \vec{g}$ 和 \vec{F} , 由于广义坐标不独立, 受

$$\text{约束: } \begin{cases} f(x_2, y_2, \theta_1, \theta_2) = l_1 \cos \theta_1 + \frac{1}{2} l_2 \cos \theta_2 - x_2 = 0 \\ f(x_2, y_2, \theta_1, \theta_2) = l_1 \sin \theta_1 + \frac{1}{2} l_2 \sin \theta_2 - y_2 = 0 \end{cases}$$

$$\therefore -l_1 \sin \theta_1 \delta \theta_1 - \frac{1}{2} l_2 \sin \theta_2 \delta \theta_2 - \delta x_2 = 0 \quad (b)$$

$$l_1 \cos \theta_1 \delta \theta_1 + \frac{1}{2} l_2 \cos \theta_2 \delta \theta_2 - \delta y_2 = 0 \quad (c)$$

用 λ_1 遍乘 (b) 的各顶, λ_2 遍乘 (c) 的各顶, 并与 (a) 相加, 3 号

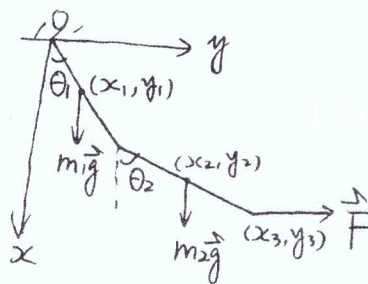
$$\left(-\frac{1}{2} m_1 g \sin \theta_1 + \lambda_1 \sin \theta_1 + \lambda_2 \cos \theta_1 \right) l_1 \delta \theta_1 + (m_2 g - \lambda_1) \delta x_2 +$$

$$\left(\frac{1}{2} \cos \theta_2 F - \frac{1}{2} \lambda_1 \sin \theta_2 + \frac{1}{2} \lambda_2 \cos \theta_2 \right) l_2 \delta \theta_2 + (F - \lambda_2) \delta y_2 = 0$$

令各系数分别为零, 得平衡方程, 其中对应 δx_2 和 δy_2 的两个

$$\text{平衡方程为 } \begin{cases} m_2 g - \lambda_1 = 0 \\ F - \lambda_2 = 0 \end{cases} \therefore R_p = \lambda_1 \text{ 对}$$

\therefore 下杆所受的 x 方向力为 $-m_2 \vec{g}$, y 方向力为 $-\vec{F}$.



Q3解: 小圆柱体作平面平行运动, 自由度为3个,

$$\text{又 } x_c^2 + y_c^2 = (R-r)^2, \quad r\psi = (R-r)\theta$$

$$\therefore s=1, \text{ 令 } q_1 = \theta$$

$$\therefore T = \frac{1}{2} m u_c^2 + \frac{1}{2} I_c \omega^2, \quad u_c = (R-r)\dot{\theta}$$

$$\dot{\psi} = \omega = \frac{R-r}{r} \dot{\theta}, \quad I_c = \frac{1}{2} m r^2$$

$$T = \frac{1}{2} m (R-r)^2 \dot{\theta}^2 + \frac{1}{4} m (R-r)^2 \dot{\theta}^2 = \frac{3}{4} m (R-r)^2 \dot{\theta}^2$$

$$\text{设 } \theta=0 \text{ 处的势能为零, 则 } V = mg(R-r)(1-\cos\theta)$$

$$L = T - V = \frac{3}{4} m (R-r)^2 \dot{\theta}^2 - mg(R-r)(1-\cos\theta)$$

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{\theta}} \right) = \frac{d}{dt} \left[\frac{3}{2} m (R-r)^2 \dot{\theta} \right] = \frac{3}{2} m (R-r)^2 \ddot{\theta}$$

$$\frac{\partial L}{\partial \theta} = -mg(R-r)\sin\theta, \text{ 得 } \frac{3}{2} m (R-r)^2 \ddot{\theta} + mg(R-r)\sin\theta = 0$$

$$\text{即 } \ddot{\theta} + \frac{2}{3} \frac{g}{(R-r)} \sin\theta = 0, \text{ 又对于小振动, } \sin\theta \approx \theta, \therefore$$

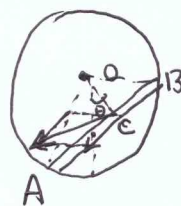
$$\text{方程为: } \ddot{\theta} + \frac{2g}{3(R-r)} \theta = 0, \text{ 得 } \omega = \sqrt{\frac{2g}{3(R-r)}}$$

解: 系统自由度为2, 取圆环中心O的坐标 x_0 及OC与垂直线的交角 θ 为广义坐标。圆环作纯滚动, 其转动角速度为 $\omega = \frac{\dot{x}_0}{r}$

$$\text{圆环动能 } T_2 = \frac{1}{2} m_2 \dot{x}_0^2 + \frac{1}{2} (m_2 r^2) \omega^2 = m_2 \dot{x}_0^2$$

$$\text{杆AB动能 } T_1 = \frac{1}{2} m_1 v_c^2 + \frac{1}{2} I_c \dot{\theta}^2$$

$$\text{其中 } v_c^2 = \dot{x}_0^2 + \frac{1}{2} r^2 \dot{\theta}^2 + \sqrt{2} r \dot{x}_0 \dot{\theta} \cos \theta, \quad I_c = \frac{1}{12} m_1 (\sqrt{2} r)^2$$



$$\therefore T_1 = \frac{1}{2} m_1 (\dot{x}_0^2 + \frac{1}{2} r^2 \dot{\theta}^2 + \sqrt{2} r \dot{x}_0 \dot{\theta} \cos \theta) + \frac{1}{12} m_1 r^2 \dot{\theta}^2$$

$$\text{则系统 } T = T_1 + T_2 = (m_2 + \frac{m_1}{2}) \dot{x}_0^2 + \frac{1}{3} m_1 r^2 \dot{\theta}^2 + \frac{\sqrt{2}}{2} m_1 r \dot{x}_0 \dot{\theta} \cos \theta$$

$$\text{系统势能 } V = -\frac{\sqrt{2}}{2} m_1 r g \cos \theta, \text{ 选O点的水平面为势能面}$$

$$\text{则拉氏函数为 } L = T - V$$

$$= (m_2 + \frac{m_1}{2}) \dot{x}_0^2 + \frac{1}{3} m_1 r^2 \dot{\theta}^2 + \frac{\sqrt{2}}{2} m_1 r \dot{x}_0 \dot{\theta} \cos \theta + \frac{\sqrt{2}}{2} m_1 r g \cos \theta$$

$$\text{代入拉格朗日方程得: } \begin{cases} (2m_2 + m_1) \ddot{x}_0 + \frac{\sqrt{2}}{2} m_1 r (\ddot{\theta} \cos \theta - \dot{\theta}^2 \sin \theta) = 0 \\ 4r \ddot{\theta} + 3\sqrt{2} \dot{x}_0 \cos \theta + 3\sqrt{2} g \sin \theta = 0 \end{cases}$$

若只考虑微振动则可略去 $\dot{\theta}^2$, 且 $\cos \theta \approx 1$, $\sin \theta \approx \theta$, 则

$$\begin{cases} (2m_2 + m_1) \ddot{x}_0 + \frac{\sqrt{2}}{2} m_1 r \ddot{\theta} = 0 \\ 4r \ddot{\theta} + 3\sqrt{2} \dot{x}_0 + 3\sqrt{2} g \theta = 0 \end{cases}$$

$$\text{消去 } x_0 \text{ 得到关于 } \theta \text{ 的方程: } \frac{8m_2 + m_1}{2m_2 + m_1} r \ddot{\theta} + 3\sqrt{2} g \theta = 0$$

$$\text{即 } \ddot{\theta} + 3\sqrt{2} \frac{g}{r} \frac{2m_2 + m_1}{8m_2 + m_1} \theta = 0$$

显然为一简谐振动, 则

$$\omega^2 = 3\sqrt{2} \frac{g}{r} \frac{2m_2 + m_1}{8m_2 + m_1}$$

解: 系统自由度为2, 取 $q_1 = \varphi_1$ 和 $q_2 = \varphi_2$

则 OA 和 OB 杆的质心速度分别为

$$u_1 = \frac{1}{2}l\dot{\varphi}_1, \quad u_2 = l\dot{\varphi}_1 + \frac{1}{2}l\dot{\varphi}_2$$

以广义坐标表示的系统动能为 $T = \frac{1}{2}I_0\dot{\varphi}_1^2 + \frac{1}{2}mu_2^2 + \frac{1}{2}I_{c2}\dot{\varphi}_2^2$

$$I_0 = \frac{1}{3}ml^2, \quad I_{c2} = \frac{1}{12}ml^2, \quad \therefore T = \frac{1}{2}ml^2\left(\frac{4}{3}\dot{\varphi}_1^2 + \dot{\varphi}_1\dot{\varphi}_2 + \frac{1}{3}\dot{\varphi}_2^2\right)$$

$$\text{系统所受广义冲量 } I_k = \sum_{i=1}^n \vec{I}_i \cdot \frac{\partial \vec{r}_i}{\partial \dot{q}_k} = \vec{I} \cdot \frac{\partial \vec{r}_B}{\partial \dot{q}_k} = I \frac{\partial x_B}{\partial \dot{q}_k}$$

如图所示 $x_B = l\sin\varphi_1 + l\sin\varphi_2$

$$\text{则 } I_{\varphi_1} = I \frac{\partial x_B}{\partial \dot{\varphi}_1} = I l \cos\varphi_1 \xrightarrow{\varphi_1 \rightarrow 0} I l$$

$$\text{同理, } I_{\varphi_2} = I l, \text{ 则代入 } \left[\frac{\partial T}{\partial \dot{q}_k} \right]_{t_2} - \left[\frac{\partial T}{\partial \dot{q}_k} \right]_{t_1} = I_k, \text{ 得}$$

$$\dot{\varphi}_1 = -\frac{6I}{7ml}, \quad \dot{\varphi}_2 = \frac{30I}{7ml}$$

