

## Tutorial 10 Decoupling Control

**10.1** Consider the 2-input/2-output system described by

$$\dot{x} = \begin{bmatrix} -1 & 0 & 0 \\ 0 & -2 & 0 \\ 0 & 0 & -3 \end{bmatrix} x + \begin{bmatrix} 1 & 0 \\ 2 & 3 \\ -3 & -3 \end{bmatrix} u,$$

$$y = \begin{bmatrix} 1 & 0 & 0 \\ 1 & 1 & 1 \end{bmatrix} x.$$

Find the state-feedback decoupler.

**Solution:**

It is readily checked that

$$\begin{aligned} c_1^T A^0 B &= C_1^T B = \begin{pmatrix} 1 & 0 \end{pmatrix}, \quad \text{or} \quad \sigma_1 = 1; \\ c_2^T B &= \begin{pmatrix} 0 & 0 \end{pmatrix}, \\ c_2^T AB &= \begin{pmatrix} 4 & 3 \end{pmatrix}, \quad \text{or} \quad \sigma_2 = 2, \end{aligned}$$

so that

$$B^* = \begin{bmatrix} 1 & 0 \\ 4 & 3 \end{bmatrix}.$$

Since  $B^*$  is nonsingular, the system can be decoupled by state feedback  $u = -Kx + Fv$ . It is easily checked that

$$B^{*-1} = \frac{1}{3} \begin{bmatrix} 3 & 0 \\ -4 & 1 \end{bmatrix} = F,$$

Use the constructive procedure of Theorem 9.1, one finds that

$$K = B^{*-1} C^* = \begin{bmatrix} -1 & 0 & 0 \\ 1.67 & 1.33 & 3 \end{bmatrix}.$$

The resulting closed loop transfer function is

$$H(s) = \begin{bmatrix} \frac{1}{s} & 0 \\ 0 & \frac{1}{s^2} \end{bmatrix}.$$

Now that the system is decoupled further stabilization is easily done on the single loop basis.

**10.2** The linearized equations governing the equatorial motion of a satellite in a circular orbit is

$$\begin{aligned} \dot{x} &= Ax + Bu = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 3\omega^2 & 0 & 0 & 2\omega \\ 0 & 0 & 0 & 1 \\ 0 & -2\omega & 0 & 0 \end{bmatrix} x + \begin{bmatrix} 0 & 0 \\ 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{bmatrix} u, \\ y &= Cx = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} x. \end{aligned}$$

where the outputs and inputs are radial and tangential

$$y = \begin{bmatrix} r \\ \theta \end{bmatrix}, \quad u = \begin{bmatrix} u_r \\ u_\theta \end{bmatrix}.$$

These are coupled by the orbital dynamics, as indicated by the transfer function matrix

$$G(s) = \begin{bmatrix} \frac{1}{s^2 + \omega^2} & \frac{2\omega}{s(s^2 + \omega^2)} \\ \frac{-2\omega}{s(s^2 + \omega^2)} & \frac{s^2 - 3\omega^2}{s^2(s^2 + \omega^2)} \end{bmatrix}.$$

Find a state-feedback decoupler.

**Solution:**

To determine whether this system can be decoupled, we compute

$$\begin{aligned} c_1^T B &= [0 \quad 0], \\ c_1^T AB &= [1 \quad 0], \text{ or } \sigma_1 = 2. \end{aligned}$$

$$\begin{aligned} c_2^T B &= [0 \ 0], \\ c_2^T AB &= [0 \ 1], \text{ or } \sigma_2 = 2. \end{aligned}$$

We then have

$$B^* = \begin{bmatrix} c_1^T AB \\ c_2^T AB \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} = I,$$

is nonsingular. Thus decoupling can be achieved with

$$\begin{aligned} u(t) &= -Kx(t) + Fw(t) \\ &= -B^{*-1}C^*x(t) + B^{*-1}w(t), \\ &= -\begin{bmatrix} 3\omega^2 & 0 & 0 & 2\omega \\ 0 & -2\omega & 0 & 0 \end{bmatrix}x(t) + w(t) \end{aligned}$$

which results in the close-loop system

$$\dot{x} = (A - BK)x + BFw = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 \end{bmatrix}x + \begin{bmatrix} 0 & 0 \\ 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{bmatrix}w, \quad (1)$$

$$y = Cx = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix}x.$$

The transfer function matrix may easily be shown to be

$$H(s) = \begin{bmatrix} \frac{1}{s^2} & 0 \\ 0 & \frac{1}{s^2} \end{bmatrix}.$$

We design pole placement state feedback control for this decoupled process as follows

$$w = K_f x + \bar{w} = \begin{bmatrix} k_1 & k_2 & 0 & 0 \\ 0 & 0 & k_3 & k_4 \end{bmatrix}x + \begin{bmatrix} \bar{w}_r \\ \bar{w}_\theta \end{bmatrix},$$

resulting in the closed-loop system

$$\dot{x} = (A - BK + BK_f)x + B\bar{w} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ k_1 & k_2 & 0 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & k_3 & k_4 \end{bmatrix} x + \begin{bmatrix} 0 & 0 \\ 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{bmatrix} \bar{w},$$

$$y = Cx = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} x.$$

If we choose the feedback gains  $k_1 = k_3 = -1$  and  $k_2 = k_4 = -2$ , each of the two independent subsystems will be stable with a double pole at  $s = -1$ . The transfer function matrix may easily be shown to be

$$H(s) = \begin{bmatrix} \frac{1}{(s+1)^2} & 0 \\ 0 & \frac{1}{(s+1)^2} \end{bmatrix}.$$

This closed-loop system is not only stable (perturbations from the nominal trajectory will always decay to zero), but adjustments to  $r$  and  $\theta$  be made independently via the external inputs  $\bar{w}_r$  and  $\bar{w}_\theta$ .

**10.3** Design the controller in a unity output feedback configuration such that it decouples and stabilizes the plant

$$G(s) = \begin{bmatrix} \frac{1}{s+1} & \frac{1}{s+1} \\ \frac{1}{s+2} & \frac{2}{s+2} \end{bmatrix}.$$

**Solution:**

Write

$$G(s) = \begin{bmatrix} \frac{1}{s+1} & 0 \\ 0 & \frac{1}{s+2} \end{bmatrix} N_r = \begin{bmatrix} \frac{1}{s+1} & 0 \\ 0 & \frac{1}{s+2} \end{bmatrix} \begin{bmatrix} 1 & 1 \\ 1 & 2 \end{bmatrix}$$

Choose the decoupler

$$K_d = N_r^{-1} = \begin{bmatrix} 2 & -1 \\ -1 & 1 \end{bmatrix}$$

So that the decoupled plant,  $GK_d$ , is

$$GK_d = \begin{bmatrix} \frac{1}{s+1} & 0 \\ 0 & \frac{1}{s+2} \end{bmatrix}.$$

It can be easily verified that the closed loop is stable in the unity feedback configuration. So a simple gain matrix can decouple and stabilize the system at the same time.