```
clear all
clc
%TNPUTS
                                                                                                                                                                                                                                                                                                          906548745
L=22; %length of the bar
br=8.75/12; hei=27/12;
num=22; %number of elements
E=(2*10^6)*12^2;
\alpha = 1500;
%processed quantities
h=L/num; %size of element
I=(br)*(hei^3)*(1/12); %Second Moment of Inertia
%% DERIVING SHAPE FUNCTIONS
syms s
N10(s) = (1/4) * (2-3*s+s^3); N20(s) = (1/4) * (2+3*s-s^3);
N11(s) = (1/4) * (1-s-s^2+s^3); N21(s) = (1/4) * (-1-s+s^2+s^3);
N10 2=diff(diff(N10))/(h/2)^2;N20 2=diff(diff(N20))/(h/2)^2;
N11 2 = diff(diff(N11))/(h/2)^2; N21 2 = diff(diff(N21))/(h/2)^2;
a = (h/2) * (N10 2)^2, b = (h/2) * (N10 2) * (N20 2), c = (h/2) * (N20 2)^2
d=(h/2)*N10 2*N11 2; e=(h/2)*N10 2*N21 2; f=(h/2)*N20 2*N11 2; q=(h/2)*N20 2*N21 2;
h1=(h/2)*N11 2*N10 2; i=(h/2)*N11 2*N20 2;j=(h/2)*N21_2*N10_2; k=(h/2)*N21_2*N20_2;
1=(h/2)*N11 2^2; m=(h/2)*N11 2*N21 2; n=(h/2)*N21 2^2;
%% Element Matrices
 \texttt{K11 00} = \mathtt{int}(\mathtt{a}, -1, 1) \texttt{; K12 00} = \mathtt{int}(\mathtt{b}, -1, 1) \texttt{;} \qquad \texttt{K11 01} = \mathtt{int}(\mathtt{d}, -1, 1) \texttt{; K12 01} = \mathtt{int}(\mathtt{e}, -1, 1) \texttt{;} 
                                                                  K22 \ 00=int(c,-1,1); \ K21 \ 01=int(f,-1,1); K22 \ 01=int(g,-1,1);
K21 00=K12 00;
 \texttt{K11 10} = \texttt{int(h1,-1,1); K12 10} = \texttt{int(i,-1,1);} \\  \texttt{K11 11} = \texttt{int(1,-1,1); K12 11} = \texttt{int(m,-1,1);} \\  \texttt{K11 10} = \texttt{int(h1,-1,1); K12 11} = \texttt{int(m,-1,1);} \\  \texttt{K11 10} = \texttt{int(h1,-1,1); K12 10} = \texttt{int(m,-1,1);} \\  \texttt{K11 11} = \texttt{int(h1,-1,1); K12 11} = \texttt{int(m,-1,1);} \\  \texttt{K11 11} = \texttt{int(h1,-1,1); K12 11} = \texttt{int(m,-1,1);} \\  \texttt{K11 11} = \texttt{int(h1,-1,1); K12 11} = \texttt{int(m,-1,1);} \\  \texttt{K11 11} = \texttt{int(h1,-1,1); K12 11} = \texttt{int(m,-1,1);} \\  \texttt{K11 11} = \texttt{int(h1,-1,1);} \\  \texttt{K11 11} = \texttt{int(h1,-1,h1);} \\  \texttt{K11
K21 10=int(j,-1,1); K22 10=int(k,-1,1); K21 11=K12 11;
                                                                                                                                                                                                                        K22 11=int(n,-1,1);
A= [K11 00,K12 00;K21 00,K22 00]; B= [K11 01,K12 01;K21 01,K22 01];
C= [K11 10,K12 10;K21 10,K22 10]; D= [K11 11,K12 11;K21 11,K22 11];
%% Global Matrix
K00=zeros(num+1); K01=zeros(num+1); K10=zeros(num+1); K11=zeros(num+1);
for i=1:num
                          for r=1:2
                                         for c=1:2
                                                       K00(r+1*(i-1),c+1*(i-1))=A(r,c)+K00(r+1*(i-1),c+1*(i-1));
                                                       K01(r+1*(i-1),c+1*(i-1))=B(r,c)+K01(r+1*(i-1),c+1*(i-1));
                                                       K10(r+1*(i-1),c+1*(i-1))=C(r,c)+K10(r+1*(i-1),c+1*(i-1));
                                                       K11(r+1*(i-1),c+1*(i-1)) = D(r,c) + K11(r+1*(i-1),c+1*(i-1));
                                         end
                          end
end
K FH=cat(2,K00,K01); K SH=cat(2,K10,K11); K G=cat(1,K FH,K SH)*(E*I)
%% LOAD VECTOR:
F10 = -q*\left(\text{int}\left(\text{N10}, -1, 1\right)\right) * \left(\text{h/2}\right); \\ F20 = -q*\left(\text{int}\left(\text{N20}, -1, 1\right)\right) * \left(\text{h/2}\right); \\ F11 = -q*\left(\text{int}\left(\text{N11}, -1, 1\right)\right) * \left(\text{h/2}\right); \\ F21 = -q*\left(\text{int}\left(\text{N20}, -1, 1\right)\right) * \left(\text{h/2}\right); \\ F21 = -q*\left(\text{int}\left(\text{N20}, -1, 1\right)\right) * \left(\text{h/2}\right); \\ F31 = -q*\left(\text{int}\left(\text{N20}, -1, 1\right)\right) * \left(\text{int}\left(\text{N20}, -1, 1\right)\right) * \left(
 ( \verb"int(N21,-1,1)") * (h/2); F\_FH= zeros (num+1,1); F\_SH= zeros (num+1,1); \\
               for i=1:num
                   F FH(i) = F FH(i) + F10; F SH(i) = F SH(i) + F11;
                  F_FH(i+1)=F20;
                                                                                          F SH(i+1) = F21;
          end
F G=cat(1,F FH,F SH)
 for i=1:2*num+2
               K G(1,i)=0; K G(num+1,i)=0;
K G(1,1)=1; K G(num+1,num+1)=1; F G(1,1)=0; F G(num+1,1)=0;
d=inv(K G)*F G;
                   disp=zeros(num+1,1);
                   for i=1:num+1
                                disp(i)=d(i);
                   end
                   slope=zeros(num+1,1);
                   for i=1:num+1
                                 slope(i) = d(i+num+1);
                  end
v=1:num+1;
x=0:h:L;
disp=disp*12;
u an(s)=piecewise(0 \le s \le 22,((-q/24)*s^4+(16500)*((s^3)/6)-(665500)*s)/(E*I));u an=u an*12;
plot(x,disp) hold on, fplot(u an) hold off , legend('Approximate','Analytical'),
xlabel('distance from the left end'),ylabel('Displacement(in)')
```

```
M=zeros(num,1);
for i=1:num
    M(i)=disp(i)*N10_2(0)+disp(i+1)*N20_2(0)+slope(i)*N11_2(0)+slope(i+1)*N21_2(0);
end
M=M*E*I;figure(2)
z=h/2:h:(2*num-1)*h/2;scatter(z,M)
title('Moment vs distance from left end')
ylabel('Moment (lb*ft)')xlabel('distance(ft)')
```

(a) Considered 22 elements for this problem

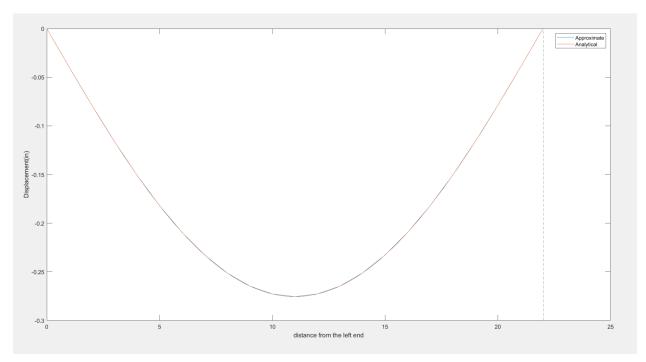


Fig.1.Approximate and Analytical Solution of Displacement Profile

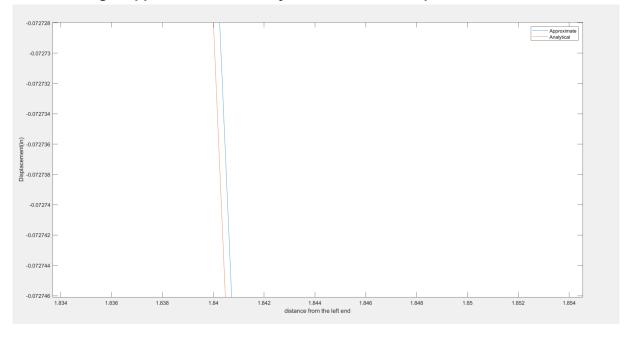
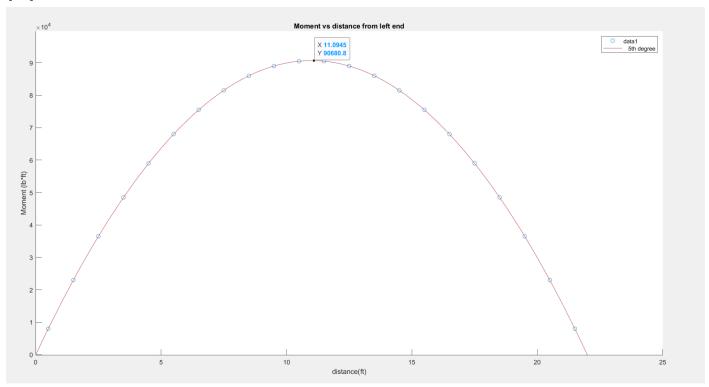


Fig.2.Zoomed image of Analytical and Approximate Solutions"

(b)



(c).

The maximum Tensile and compressive Stress occur at the point where the bending moment is the maximum, depending upon the load direction we can assess at which point at that particular cross section it is tensile and compressive.

```
Max bending Stress= (maximum Bending Moment)*(distance of the farthest layer)
/(Second Moment of Inertia)
```

```
\sigma=(Mx*y/I)
\sigma=(90700*12*27/2/I)
```

 σ =1.0235 x 10³ lbs/in (Max Tensile Str) at X=11ft on the bottom layer σ =-1.0235 x 10³ lbs/in (Max Compressive Stress) at X=11ft on the Upper layer

(d).

```
%% ERROR FUNCTION%X=(0.5*(1-s)*x(i)+0.5*(1+s)*x(i+1));
disp=disp/12;u_an=u_an/12;
Num=0;
for i=1:num
Num=Num+int((disp(i)*N10+disp(i+1)*N20+slope(i)*N11+slope(i+1)*N21-((-q/24)*(0.5*(1-s)*x(i)+0.5*(1+s)*x(i+1))^3)/6)-(665500)*(0.5*(1-s)*x(i)+0.5*(1+s)*x(i+1)))/(E*I))^2,-1,1)*(h/2);
end
for i=1:num
Num=Num+int((diff(disp(i)*N10+disp(i+1)*N20+slope(i)*N11+slope(i+1)*N21-((-q/24)*(0.5*(1-s)*x(i)+0.5*(1+s)*x(i+1)))^3)/6)-(665500)*(0.5*(1-s)*x(i)+0.5*(1+s)*x(i+1))^3)/6)-(665500)*(0.5*(1-s)*x(i)+0.5*(1+s)*x(i+1)))/(E*I)))^2,-1,1)*(2/h);
end
Den=int(u_an^2,0,22)+int((diff(u_an))^2,0,L);
Err=((Num)/(Den))^0.5;
Err=log(Err);ErrVpa=vpa(Err,6)
```

ErrVpa =-12.8088