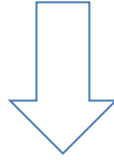


General Physics

- Classical Mechanics
 1. Newtonian Mechanics
 - Applied to particles, rigid bodies, Elastic systems, and fluids
 2. Hamiltonian Mechanics
- Electromagnetism
- Thermodynamics/ Statistical Mechanics
- Quantum Mechanics
- Special Relativity



Central Elements of Studies for Physics Students

- Classical Mechanics
- Electrodynamics
- Statistical Mechanics
- Quantum Mechanics

Physics

Physical
(body)

Mental
(mind)



Sound,
sight,
touch,
smell,
taste



Intuition



Instruments + Math

Measurements

+

Rigorous Analysis

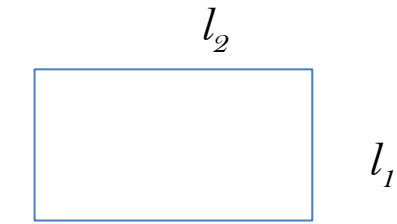


Physical Sciences

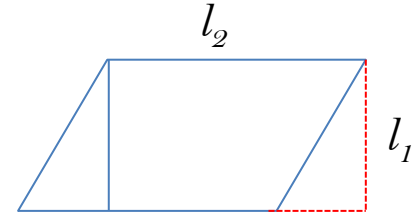
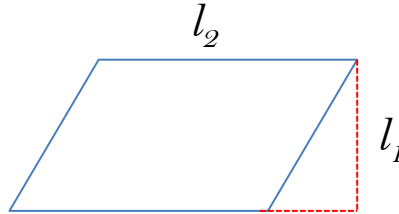
Measurements

- Physical Quantities
 - Base Quantities Base Units (SI or MKSA)
 - Length m
 - Mass kg
 - Time s
 - Current A
 - Derived Quantities
 - Area, Volume, Velocity, Momentum, etc.

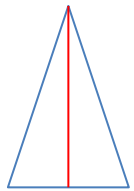
Derived Quantities e.g. Area



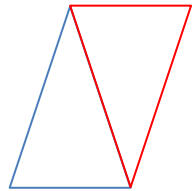
$$area = l_1 \times l_2$$



$$area = l_1 \times l_2$$

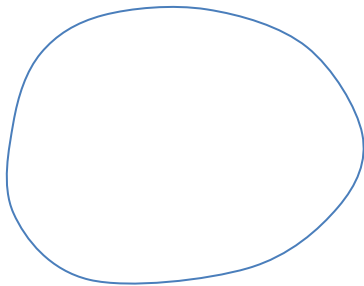


l_1

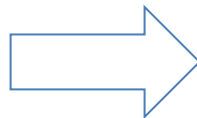


$$area = \frac{1}{2} l_1 \times l_2$$

l_2



$area = ?$



Integration

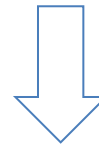
Derived Quantities e.g. Velocity

1. Motion in one dimension with a constant speed

Velocity at any time t $v(t) = \frac{\Delta x}{\Delta t}$

2. Motion in one dimension with a varying speed

Velocity at time t $v(t) = ?$



Differentiation

Differentiation Integration

Calculus

$f(x)$ x : the variable f : a function of x

Difference $\Delta x \Rightarrow \Delta f = f(x + \Delta x) - f(x)$

Note: Δx and Δf are finite (i.e. not infinite and not infinitesimal).

$\Delta x \rightarrow 0 \Rightarrow$ differentials dx , $df = f(x + dx) - f(x)$

Note: dx and df are infinitesimal.

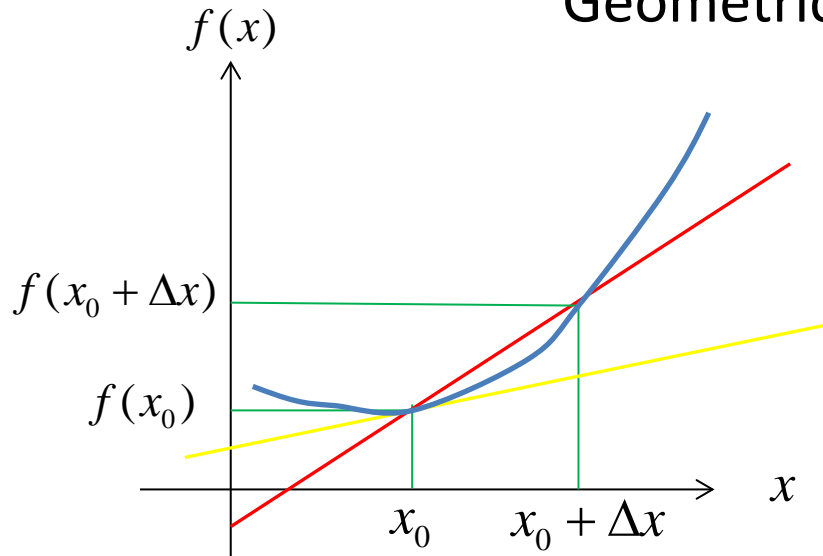
However, $\frac{df}{dx} = \frac{f(x + dx) - f(x)}{dx} = \lim_{\Delta x \rightarrow 0} \frac{f(x + \Delta x) - f(x)}{\Delta x}$ can be finite.

$f'(x) = \frac{df}{dx}$ is called the first derivative of f .

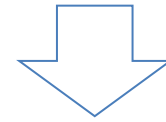
"To obtain derivatives of f " is called "to differentiate f ".

$\frac{d}{dx}$: differentiation operator

Geometrical Meaning of Differentiation



$$\lim_{\Delta x \rightarrow 0} \frac{f(x_0 + \Delta x) - f(x_0)}{\Delta x} = \frac{f(x_0 + dx) - f(x_0)}{dx}$$



Slope of the curve at x_0

Some Examples of Differentiation:

$$f(x) = 3x^2 + 2x + 1 \Rightarrow \frac{df(x)}{dx} = 6x + 2$$

$$f(x) = \sin x \Rightarrow \frac{df(x)}{dx} = \cos x$$

$$f(x) = \cos x \Rightarrow \frac{df(x)}{dx} = -\sin x$$

$$f(x) = \exp(x) = e^x \Rightarrow \frac{df(x)}{dx} = \exp(x)$$

$$f(x) = \ln x \Rightarrow \frac{df(x)}{dx} = \frac{1}{x}$$

Some Rules of Differentiation:

$$1. f(x) = a_1 f_1(x) + a_2 f_2(x) \Rightarrow \frac{df(x)}{dx} = a_1 \frac{df_1(x)}{dx} + a_2 \frac{df_2(x)}{dx}$$

$$e.g. \frac{d}{dx}[3\sin x + 5\cos x] = 3\cos x - 5\sin x$$

$$2. f(x) = f_1(x) \cdot f_2(x) \Rightarrow \frac{df(x)}{dx} = f_2(x) \frac{df_1(x)}{dx} + f_1(x) \frac{df_2(x)}{dx}$$

$$e.g. \frac{d}{dx}[x^2 - 1] = \frac{d}{dx}[(x+1)(x-1)] = (x-1) \frac{d}{dx}[x+1] + (x+1) \frac{d}{dx}[x-1] = 2x$$

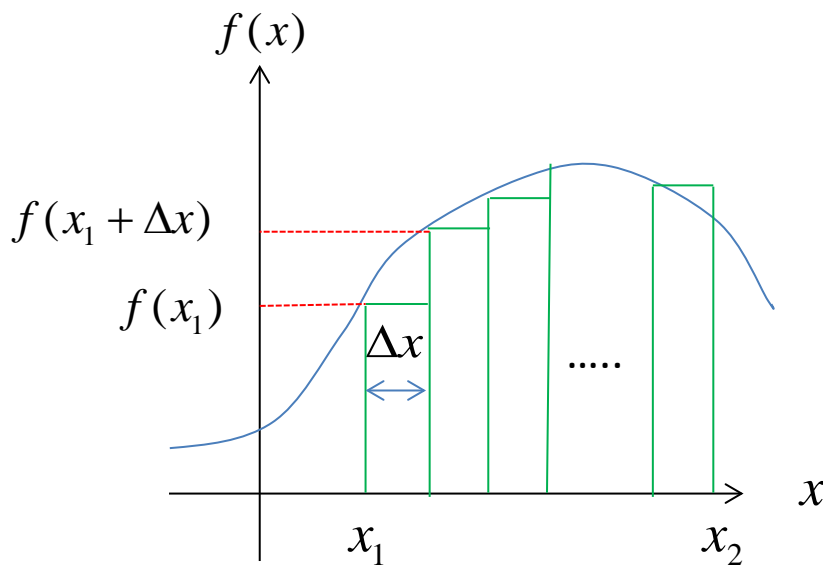
$$3. f(x) = f_1(f_2(x)) \Rightarrow \frac{df(x)}{dx} = \left. \frac{df_1(y)}{dy} \right|_{y=f_2(x)} \cdot \frac{df_2(x)}{dx}$$

$$e.g. \frac{d}{dx}[12x^2 + 12x + 5] = \frac{d}{dx}[3(2x+1)^2 + 2]$$

$$= \frac{d}{dy}[3y^2 + 2] \Big|_{y=2x+1} \cdot \frac{d}{dx}[2x+1] = 24x + 12$$

Integration { Definite Integration Indefinite Integration

Geometrical Meaning of Definite Integration



$$\text{let } \Delta x = \frac{(x_2 - x_1)}{n} \quad , \quad S = \sum_{i=0}^{n-1} f(x_1 + i \cdot \Delta x) \cdot \Delta x$$

$\Rightarrow S$ is the total area in the rectangles

If $n \rightarrow \infty \Rightarrow \Delta x \rightarrow dx$

$$\sum_{i=0}^{n-1} \rightarrow \int_{x_1}^{x_2}$$

$$\Rightarrow S = \int_{x_1}^{x_2} f(x) dx$$

is the area under the curve from x_1 to x_2 .

Indefinite Integration is the counter-action of differentiation

$$\int \frac{d}{dx} f(x) dx = f(x) + c$$

c : an arbitrary constant

Note : If $g(x) = \frac{df(x)}{dx}$ then the definite integral of $g(x)$ between x_1 and x_2

$$\begin{aligned} \int_{x_1}^{x_2} g(x) dx &= \int_{x_1}^{x_2} \frac{df(x)}{dx} dx = \int_{x_1}^{x_2} \frac{f(x+dx) - f(x)}{dx} dx \\ &= [f(x_1+dx) - f(x_1)] + [f(x_1+2dx) - f(x_1+dx)] + \cdots + [f(x_2) - f(x_2-dx)] \\ &= f(x_2) - f(x_1) \end{aligned}$$



$$\int_{x_1}^{x_2} f(x) dx = \int f(x) dx \Big|_{x=x_2} - \int f(x) dx \Big|_{x=x_1}$$

To calculate the area under a curve $f(x)$:

1. calculate the indefinite integral of $f(x)$ $g(x) = \int f(x)dx$
2. The definite integral $\int_{x_1}^{x_2} f(x)dx = g(x_2) - g(x_1)$

e.g.

$$1. f(x) = 6x + 2 \quad \int f(x)dx = 3x^2 + 2x + c$$

$$\Rightarrow \int_0^1 f(x)dx = (3 + 2 + c) - (c) = 5$$

$$2. f(x) = 3\exp(x) \quad \int f(x)dx = 3\exp(x) + c$$

$$\Rightarrow \int_0^1 f(x)dx = [3\exp(1) + c] - [3\exp(0) + c] = 3(e - 1)$$

$$3. f(x) = \frac{2}{x} \quad \int f(x)dx = 2\ln x + c$$

$$\Rightarrow \int_2^3 f(x)dx = [2\ln 3 + c] - [2\ln 2 + c] = 2(\ln 3 - \ln 2) = 2\ln\left(\frac{3}{2}\right)$$

Some Examples of Using Calculus to solve physical problems

I. Motion with constant speed in one dimension

$$\frac{dx}{dt} = v \text{ (a constant)} \Rightarrow dx = v dt$$

$$\Rightarrow \int dx = \int v dt \Rightarrow x = vt + c$$

$$\text{Let } x(0) = x_0 \Rightarrow c = x_0$$

$$\Rightarrow x(t) = vt + x_0$$

II. Motion with constant acceleration in one dimension

$$\frac{dv}{dt} = a \text{ (a constant)} \Rightarrow dv = a dt$$

$$\Rightarrow \int dv = \int a dt \Rightarrow v = at + c$$

$$\text{Let } v(0) = v_0 \Rightarrow c = v_0 \Rightarrow v(t) = at + v_0$$

$$\frac{dx}{dt} = \boxed{v(t) = at + v_0} \Rightarrow dx = (at + v_0) dt$$

$$\int dx = \int (at + v_0) dt \Rightarrow x = \frac{1}{2} at^2 + v_0 t + c$$

$$\text{Let } x(0) = x_0 \Rightarrow c = x_0$$

$$\Rightarrow \boxed{x(t) = \frac{1}{2} at^2 + v_0 t + x_0}$$

Physical Quantities

Scalars → magnitude only

(one number)

Vectors → magnitude and direction

(more than one number)

e.g.

Scalars : mass, temperature...

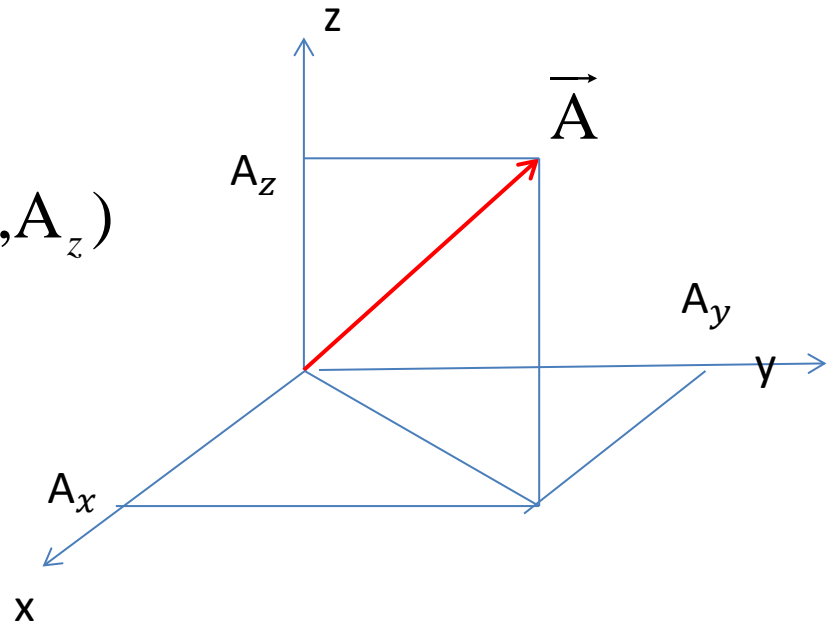
Vectors: position, displacement, velocity, acceleration, force...

Vectors

$$\vec{A} = A_x \hat{i} + A_y \hat{j} + A_z \hat{k} = (A_x, A_y, A_z)$$

in Cartesian Coordinates

$$|\vec{A}| = (A_x^2 + A_y^2 + A_z^2)^{\frac{1}{2}}$$



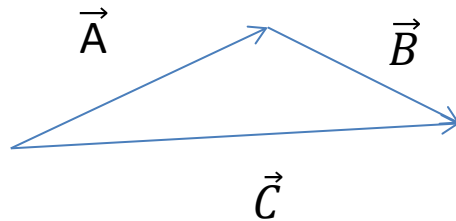
$$\vec{A} = A_x \hat{i} + A_y \hat{j} + A_z \hat{k} = (A_x, A_y, A_z)$$

$$\vec{B} = B_x \hat{i} + B_y \hat{j} + B_z \hat{k} = (B_x, B_y, B_z)$$

$$\vec{C} = C_x \hat{i} + C_y \hat{j} + C_z \hat{k} = (C_x, C_y, C_z)$$

Addition of Vectors

$$\vec{A} + \vec{B} = \vec{C} \Rightarrow C_x = A_x + B_x, \quad C_y = A_y + B_y, \quad C_z = A_z + B_z$$

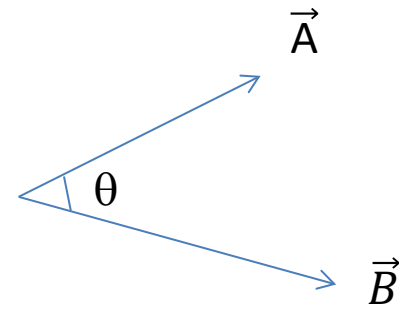


Multiplication by a scalar

$$a\vec{A} = \vec{B} \Rightarrow B_x = aA_x, \quad B_y = aA_y, \quad B_z = aA_z, \quad |\vec{B}| = a|\vec{A}|$$

Multiplication by a vector (dot product)

$$\vec{A} \cdot \vec{B} = c \quad c = |\vec{A}| |\vec{B}| \cos \theta$$



Multiplication by a vector (cross product)

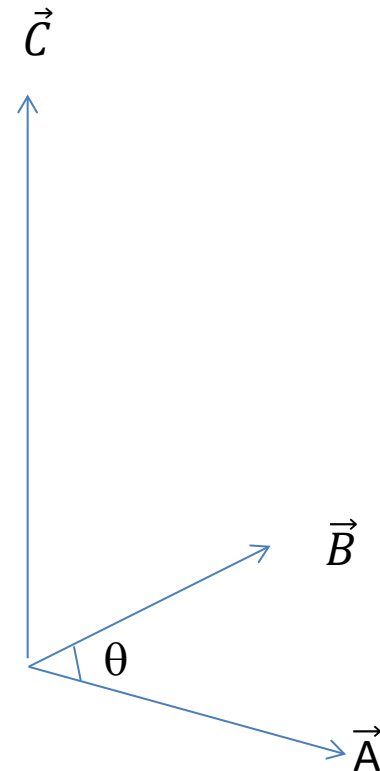
$$\vec{A} \times \vec{B} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ A_x & A_y & A_z \\ B_x & B_y & B_z \end{vmatrix} = \vec{C}$$

$$C_x = A_y B_z - A_z B_y$$

$$C_y = A_z B_x - A_x B_z$$

$$C_z = A_x B_y - A_y B_x$$

$$|\vec{C}| = |\vec{A}| |\vec{B}| \sin \theta$$



Note

$$1. \vec{A} \cdot \vec{B} = \vec{B} \cdot \vec{A}$$

$$2. \vec{A} \times \vec{B} = -\vec{B} \times \vec{A}$$

$$3. \text{ if } \vec{A} \perp \vec{B} \Rightarrow \vec{A} \cdot \vec{B} = 0$$

$$4. \text{ if } \vec{A} \parallel \vec{B} \Rightarrow \vec{A} \times \vec{B} = 0$$

$$5. \frac{d\vec{A}}{dt} = \hat{i} \frac{dA_x}{dt} + \hat{j} \frac{dA_y}{dt} + \hat{k} \frac{dA_z}{dt}$$

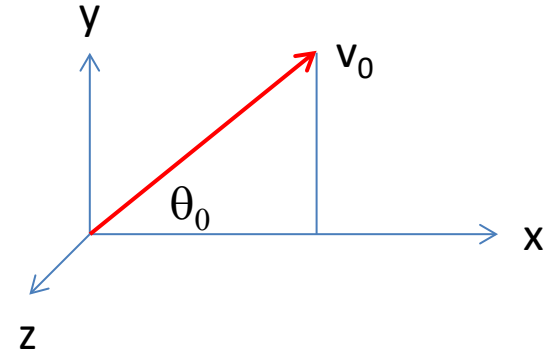
$$6. \int \vec{A} dt = \hat{i} \int A_x dt + \hat{j} \int A_y dt + \hat{k} \int A_z dt$$

Some Examples of Using Vectors and Calculus to solve physical problems

I. Projectile Motion

$$\vec{a} = -g\hat{j} = (0, -g, 0)$$

$$\vec{v}(0) = \vec{v}_0 = v_0 \cos \theta_0 \hat{i} + v_0 \sin \theta_0 \hat{j} = (v_0 \cos \theta_0, v_0 \sin \theta_0, 0)$$



$$\vec{a} = \frac{d\vec{v}}{dt} \Rightarrow \begin{cases} \frac{dv_x}{dt} = 0 \\ \frac{dv_y}{dt} = -g \\ \frac{dv_z}{dt} = 0 \end{cases} \Rightarrow \begin{cases} \int dv_x = 0 \\ \int dv_y = -\int g dt \\ \int dv_z = 0 \end{cases} \Rightarrow \begin{cases} v_x(t) + c_1 = 0 \\ v_y(t) = -gt + c_2 \\ v_z(t) + c_3 = 0 \end{cases} \Rightarrow \begin{cases} c_1 = -v_0 \cos \theta_0 \\ c_2 = v_0 \sin \theta_0 \\ c_3 = 0 \end{cases} \Rightarrow \begin{cases} v_x(t) = v_0 \cos \theta_0 \\ v_y(t) = -gt + v_0 \sin \theta_0 \\ v_z(t) = 0 \end{cases}$$

$$\vec{r}(t) = (x(t), y(t), z(t))$$

$$\vec{r}(0) = (x(0), y(0), z(0)) = (0, 0, 0)$$

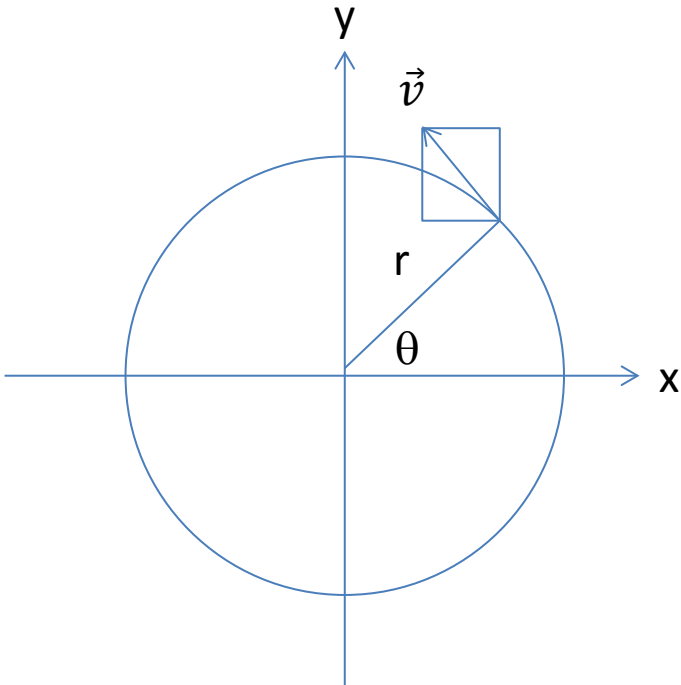
$$\vec{v}(t) = \frac{d\vec{r}}{dt} \Rightarrow \begin{cases} \frac{dx}{dt} = v_0 \cos \theta_0 \\ \frac{dy}{dt} = -gt + v_0 \sin \theta_0 \\ \frac{dz}{dt} = 0 \end{cases} \Rightarrow \begin{cases} x(t) = (v_0 \cos \theta_0)t \\ y(t) = -\frac{1}{2}gt^2 + (v_0 \sin \theta_0)t \\ z(t) = 0 \end{cases}$$

Note: $t = (v_0 \cos \theta_0)^{-1}x$

$$\Rightarrow y = (\tan \theta_0)x + \left(-\frac{g}{2(v_0 \cos \theta_0)^2}\right)x^2$$

A parabola

II. Uniform Circular Motion



$$\vec{v} = (-v \sin \theta) \hat{i} + (v \cos \theta) \hat{j} = \left(-v \frac{y}{r}\right) \hat{i} + \left(v \frac{x}{r}\right) \hat{j}$$

$$\vec{a} = \frac{d\vec{v}}{dt} = \frac{v}{r} \left(-\frac{dy}{dt}\right) \hat{i} + \frac{v}{r} \left(\frac{dx}{dt}\right) \hat{j} = \frac{v}{r} (-v_y \hat{i} + v_x \hat{j})$$

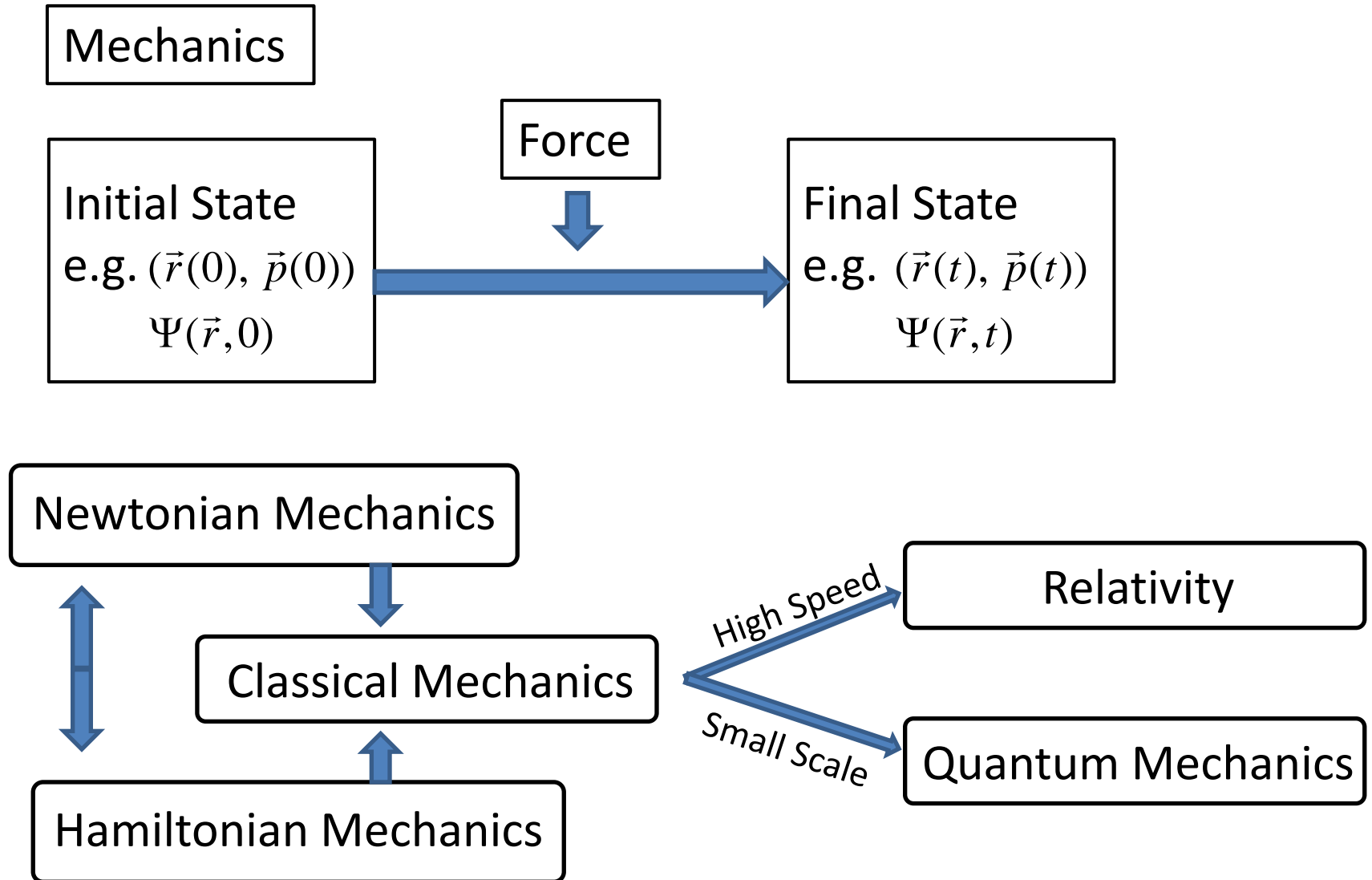
$$|\vec{a}| = \frac{v}{r} [(-v_y)^2 + (v_x)^2]^{\frac{1}{2}} = \frac{v^2}{r}$$

Note
 v_x is negative

\vec{a} has a magnitude of $\frac{v^2}{r}$ and a direction pointing to the center

Centripetal Acceleration

Chapter 5 Force and Motion



Newtonian Mechanics

Newton's Laws of Motion

I. Newton's 1st law :

$$\text{If } \vec{F}_{net} = 0 \text{ then } \frac{d\vec{v}}{dt} = 0$$

\vec{v} is indepent of time.

i.e. The object is at rest or moving with a constant velocity.

II. Newton's 2nd Law:

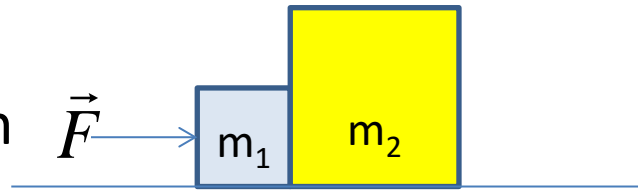
$$\vec{F}_{net} = m\vec{a}$$

III. Newton's 3rd Law:

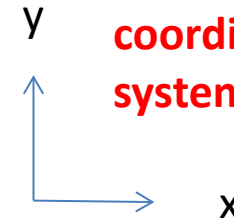
$$\vec{F}_{AB} = -\vec{F}_{BA}$$

Examples:

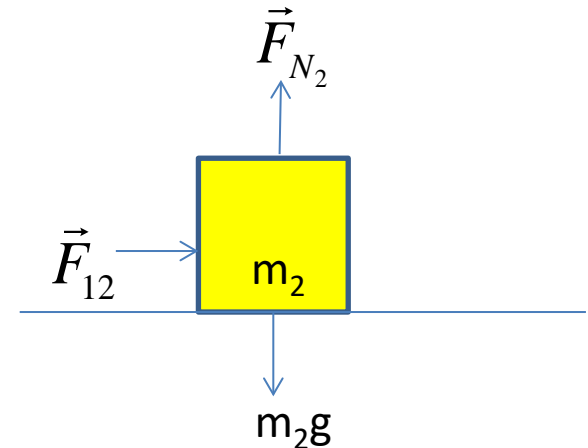
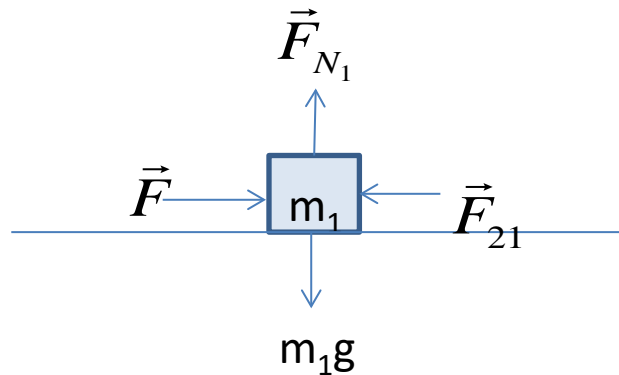
1. Find the forces between m_1 and m_2 .



1. Select a coordinate system



2. Draw Force Diagrams



3. Apply Newton's Laws

2nd law: in the x-direction $F - F_{21} = m_1 a$
in the y-direction $F_{N_1} - m_1 g = m_1 \times 0 = 0$

$F_{12} = m_2 a$
 $F_{N_2} - m_2 g = m_2 \times 0 = 0$

3rd law: $F_{12} = F_{21}$

$$\Rightarrow a = \frac{F}{m_1 + m_2}, \quad F_{12} = F_{21} = \frac{m_2 F}{m_1 + m_2}$$

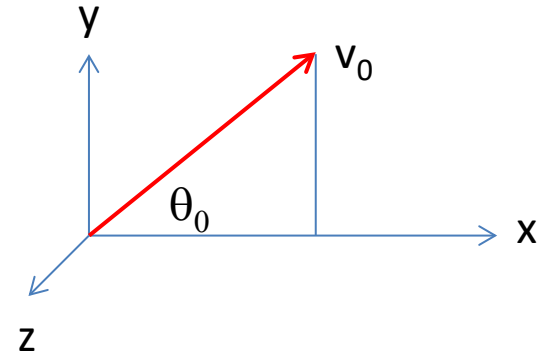
II. Projectile Motion

Initial State $(\vec{r}(0), \vec{p}(0))$

Force $\vec{F}_{net} = -mg\hat{j} = (0, -mg, 0)$

$$\vec{r}(0) = (x(0), y(0), z(0)) = (0, 0, 0)$$

$$\vec{p}(0) = m\vec{v}(0) = (mv_0 \cos \theta_0, mv_0 \sin \theta_0, 0)$$



Newton's 2nd Law: $\vec{F}_{ext} = m\vec{a} \Rightarrow (0, -mg, 0) = (m \frac{dv_x}{dt}, m \frac{dv_y}{dt}, m \frac{dv_z}{dt})$

$$\Rightarrow \begin{cases} \frac{dv_x}{dt} = 0 \\ \frac{dv_y}{dt} = -g \\ \frac{dv_z}{dt} = 0 \end{cases} \Rightarrow \begin{cases} \int dv_x = 0 \\ \int dv_y = -\int g dt \\ \int dv_z = 0 \end{cases} \Rightarrow \begin{cases} v_x(t) + c_1 = 0 \\ v_y(t) = -gt + c_2 \\ v_z(t) + c_3 = 0 \end{cases}$$

Since $\vec{v}(0) = \frac{\vec{p}(0)}{m} = (v_0 \cos \theta_0, v_0 \sin \theta_0, 0)$

$$\Rightarrow \begin{cases} v_x(t) = v_0 \cos \theta_0 \\ v_y(t) = -gt + v_0 \sin \theta_0 \\ v_z(t) = 0 \end{cases}$$

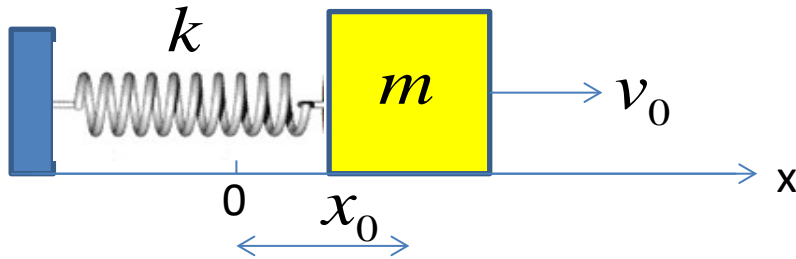
$$\vec{v}(t) = \frac{d\vec{r}}{dt} \Rightarrow \begin{cases} \frac{dx}{dt} = v_0 \cos \theta_0 \\ \frac{dy}{dt} = -gt + v_0 \sin \theta_0, \text{ and } \vec{r}(0) = (0, 0, 0) \\ \frac{dz}{dt} = 0 \end{cases} \Rightarrow \begin{cases} x(t) = (v_0 \cos \theta_0)t \\ y(t) = -\frac{1}{2}gt^2 + (v_0 \sin \theta_0)t \\ z(t) = 0 \end{cases}$$

Final State $(\vec{r}(t), \vec{p}(t))$

$$\vec{r}(t) = ((v_0 \cos \theta_0)t, -\frac{1}{2}gt^2 + (v_0 \sin \theta_0)t, 0)$$

$$\vec{p}(t) = (mv_0 \cos \theta_0, -mgt + mv_0 \sin \theta_0, 0)$$

III. Simple Harmonic Motion



Initial State $(x(0), p(0)) = (x_0, mv_0)$

Force $F_{net} = -kx$

$$\text{Newton's 2nd Law: } F_{ext} = ma = m \frac{d}{dt} \left(\frac{dx}{dt} \right) = m \frac{d^2 x}{dt^2} \Rightarrow m \frac{d^2 x}{dt^2} = -kx \Rightarrow \frac{d^2 x}{dt^2} + \frac{k}{m} x = 0$$

$$\text{Let } \omega^2 = \frac{k}{m}, \text{ we have } \boxed{\frac{d^2 x}{dt^2} + \omega^2 x = 0} \text{ (a second-order linear differential equation)}$$

Note:

The solutions of a second-order linear homogeneous differential equation

$$a \frac{d^2 f(x)}{dx^2} + b \frac{df(x)}{dx} + cf(x) = 0$$

form a 2 dimensional linear space (set of functions).

Any linear combination $a_1 f_1(x) + a_2 f_2(x)$ of solutions $f_1(x)$ and $f_2(x)$ is also a solution.

If $f_1(x)$ and $f_2(x)$ are linearly independent solutions, then the general solution is given by

$f(x) = a_1 f_1(x) + a_2 f_2(x)$, where a_1 and a_2 are arbitrary constants.

To find two independent solutions, try $x(t) = e^{\alpha t}$.

$$\frac{d^2}{dt^2} e^{\alpha t} + \omega^2 e^{\alpha t} = 0 \Rightarrow \alpha \frac{d}{dt} e^{\alpha t} + \omega^2 e^{\alpha t} = 0 \Rightarrow \alpha^2 e^{\alpha t} + \omega^2 e^{\alpha t} = 0$$

$$\Rightarrow \alpha^2 + \omega^2 = 0 \Rightarrow \alpha = \pm i\omega$$

\Rightarrow We have two independent solutions $x_1(t) = e^{i\omega t}$, $x_2(t) = e^{-i\omega t}$

And the general solution is $x(t) = c_1 e^{i\omega t} + c_2 e^{-i\omega t}$

$$\Rightarrow v(t) = \frac{dx}{dt} = i\omega c_1 e^{i\omega t} - i\omega c_2 e^{-i\omega t}$$

Initial conditions $x(0) = x_0$, $v(0) = v_0$

$$\Rightarrow c_1 + c_2 = x_0$$

$$c_1 - c_2 = \frac{v_0}{i\omega} = -i \frac{v_0}{\omega}$$

$$\Rightarrow c_1 = \frac{x_0}{2} - i \frac{v_0}{2\omega} = \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp\left[-i \tan^{-1}\left(\frac{v_0}{x_0 \omega}\right)\right]$$

$$c_2 = \frac{x_0}{2} + i \frac{v_0}{2\omega} = \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp\left[i \tan^{-1}\left(\frac{v_0}{x_0 \omega}\right)\right]$$

Note:

$$e^{i\theta} = \cos \theta + i \sin \theta$$

$$\text{let } \cos \theta = \frac{x_0 / 2}{\sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}}},$$

$$\sin \theta = \frac{v_0 / 2\omega}{\sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}}}$$

$$x(t) = \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp[-i \tan^{-1}(\frac{v_0}{x_0\omega})] e^{i\omega t} + \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp[i \tan^{-1}(\frac{v_0}{x_0\omega})] e^{-i\omega t}$$

$$= \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \left\{ \exp[i(\omega t - \tan^{-1}(\frac{v_0}{x_0\omega}))] + \exp[-i(\omega t - \tan^{-1}(\frac{v_0}{x_0\omega}))] \right\}$$

$$= \sqrt{x_0^2 + \frac{v_0^2}{\omega^2}} \cos[\omega t - \tan^{-1}(\frac{v_0}{x_0\omega})]$$

$$v(t) = i\omega \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp[-i \tan^{-1}(\frac{v_0}{x_0\omega})] e^{i\omega t} - i\omega \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp[i \tan^{-1}(\frac{v_0}{x_0\omega})] e^{-i\omega t}$$

$$= -\omega \sqrt{x_0^2 + \frac{v_0^2}{\omega^2}} \sin[\omega t - \tan^{-1}(\frac{v_0}{x_0\omega})]$$

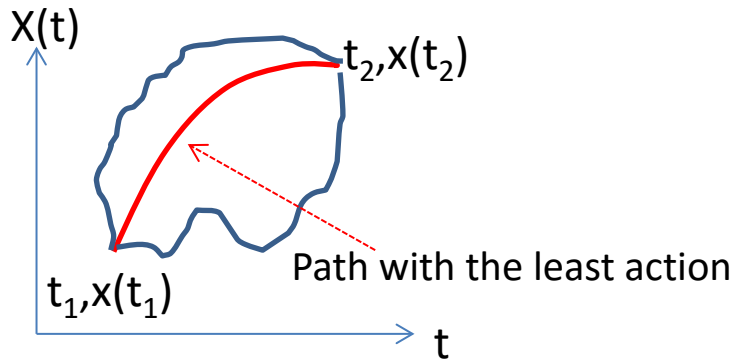
$$\text{Let } x_m = \sqrt{x_0^2 + \frac{v_0^2}{\omega^2}}, \quad \phi = -\tan^{-1}(\frac{v_0}{x_0\omega})$$

The final state: $[x(t), p(t)] = [x_m \cos(\omega t + \phi), -m\omega x_m \sin(\omega t + \phi)]$

Hamiltonian Mechanics

Hamilton's Principle:

Of all possible paths between two known points a dynamical system passes in the coordinate vs. time plot, the system takes the one that minimizes the action.



$$\text{Action } S = \int_{t_1}^{t_2} L\{x(t), \dot{x}(t)\} dt, \text{ where } \dot{x}(t) = \frac{dx}{dt}$$

$$\text{Lagrangian } L\{x(t), \dot{x}(t)\} = T(\dot{x}) - U(x)$$

T:kinetic energy, U:potential energy.

Some mathematical tools

1. If $f(x)$ has an extremum (maximum or minimum) at x_0 , then $\left. \frac{df}{dx} \right|_{x=x_0} = 0$

2. Partial differentiation for a multiple-variable function $f(x, y)$

$$\frac{\partial f}{\partial x} = \frac{f(x+dx, y) - f(x, y)}{dx}; \quad \frac{\partial f}{\partial y} = \frac{f(x, y+dy) - f(x, y)}{dy}$$

$$\text{If } x = x(u, v) \text{ and } y = y(u, v) \text{ then } \frac{\partial f}{\partial u} = \frac{\partial f}{\partial x} \frac{\partial x}{\partial u} + \frac{\partial f}{\partial y} \frac{\partial y}{\partial u}$$

Let $x(t)$ be the path that gives a minimum for the action S .

All paths connecting the two points $[t_1, x(t_1)]$ and $[t_2, x(t_2)]$ can be written as $x(\alpha, t) = x(0, t) + \alpha\eta(t)$, where $\eta(t)$ is any function satisfying $\eta(t_1) = \eta(t_2) = 0$.

$$S(\alpha) = \int_{t_1}^{t_2} L\{x(\alpha, t), \dot{x}(\alpha, t)\} dt$$

Since $x(t)$ is the path that gives a minimum for the action S , we have $\left. \frac{dS}{d\alpha} \right|_{\alpha=0} = 0$.

$$\begin{aligned} \frac{dS}{d\alpha} &= \frac{d}{d\alpha} \left[\int_{t_1}^{t_2} L\{x(\alpha, t), \dot{x}(\alpha, t)\} dt \right] = \int_{t_1}^{t_2} \frac{\partial L\{x(\alpha, t), \dot{x}(\alpha, t)\}}{\partial \alpha} dt \\ &= \int_{t_1}^{t_2} \left(\frac{\partial L}{\partial x} \frac{\partial x}{\partial \alpha} + \frac{\partial L}{\partial \dot{x}} \frac{\partial \dot{x}}{\partial \alpha} \right) dt \end{aligned}$$

Since $x(\alpha, t) = x(0, t) + \alpha\eta(t)$, we have $\frac{\partial x}{\partial \alpha} = \eta(t)$ and $\frac{\partial \dot{x}}{\partial \alpha} = \dot{\eta}(t) = \frac{d\eta}{dt}$.

$$\Rightarrow \frac{dS}{d\alpha} = \int_{t_1}^{t_2} \frac{\partial L}{\partial x} \eta(t) dt + \int_{t_1}^{t_2} \frac{\partial L}{\partial \dot{x}} \frac{d\eta}{dt} dt$$

Noting $\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{x}} \eta(t) \right) = \eta(t) \frac{d}{dt} \frac{\partial L}{\partial \dot{x}} + \frac{\partial L}{\partial \dot{x}} \frac{d\eta}{dt}$

$$\Rightarrow \int_{t_1}^{t_2} \frac{\partial L}{\partial \dot{x}} \frac{d\eta}{dt} dt = \left. \frac{\partial L}{\partial \dot{x}} \eta(t) \right|_{t_1}^{t_2} - \int_{t_1}^{t_2} \eta(t) \frac{d}{dt} \frac{\partial L}{\partial \dot{x}} dt$$

$$= - \int_{t_1}^{t_2} \eta(t) \frac{d}{dt} \frac{\partial L}{\partial \dot{x}} dt$$

$$\Rightarrow \frac{dS}{d\alpha} = \int_{t_1}^{t_2} \left[\eta(t) \left(\frac{\partial L}{\partial x} - \frac{d}{dt} \frac{\partial L}{\partial \dot{x}} \right) \right] dt, \text{ where } x = x(\alpha, t).$$

$$\left. \frac{dS}{d\alpha} \right|_{\alpha=0} = 0 \text{ for all } \eta(t) \Rightarrow \frac{\partial L}{\partial x} - \frac{d}{dt} \frac{\partial L}{\partial \dot{x}} = 0, \text{ where } x = x(0, t) = x(t).$$

$$\frac{\partial L}{\partial x} - \frac{d}{dt} \frac{\partial L}{\partial \dot{x}} = 0 \text{ is the Lagrange equation of motion.}$$

Note $L\{x(t), \dot{x}(t)\} = T(\dot{x}) - U(x) = \frac{1}{2}m\dot{x}^2 - U(x)$

$$\Rightarrow \frac{\partial L}{\partial \dot{x}} = m\dot{x} = p$$

and the Lagrangian equation $\frac{\partial L}{\partial x} - \frac{d}{dt} \frac{\partial L}{\partial \dot{x}} = 0$

$$\Rightarrow \frac{\partial L}{\partial x} = \frac{d}{dt} \frac{\partial L}{\partial \dot{x}} = \frac{d}{dt} p = \dot{p}$$

Define the Hamiltonian $H = p\dot{x} - L = T + U = \frac{p^2}{2m} + U(x)$

$$\Rightarrow \frac{\partial H}{\partial p} = \frac{p}{m} = \dot{x}; \quad \frac{\partial H}{\partial x} = \frac{\partial}{\partial x}(p\dot{x} - L) = -\frac{\partial L}{\partial x} = -\dot{p} \text{ (by Lagrangian equation)}$$

$$\frac{\partial H}{\partial p} = \dot{x}$$

$$\frac{\partial H}{\partial x} = -\dot{p}$$

are the Hamilton's equations of motion.

Note $\frac{\partial H}{\partial x} = \frac{dU}{dx} = -F$

$$\frac{\partial H}{\partial x} = -\dot{p} \Rightarrow F = m \frac{d^2 x}{dt^2} = ma$$

This reproduces Newton's 2nd law!

Chapter 6 Force and Motion II

Fundamental Forces:

Gravitational

Electromagnetic

Weak

Strong

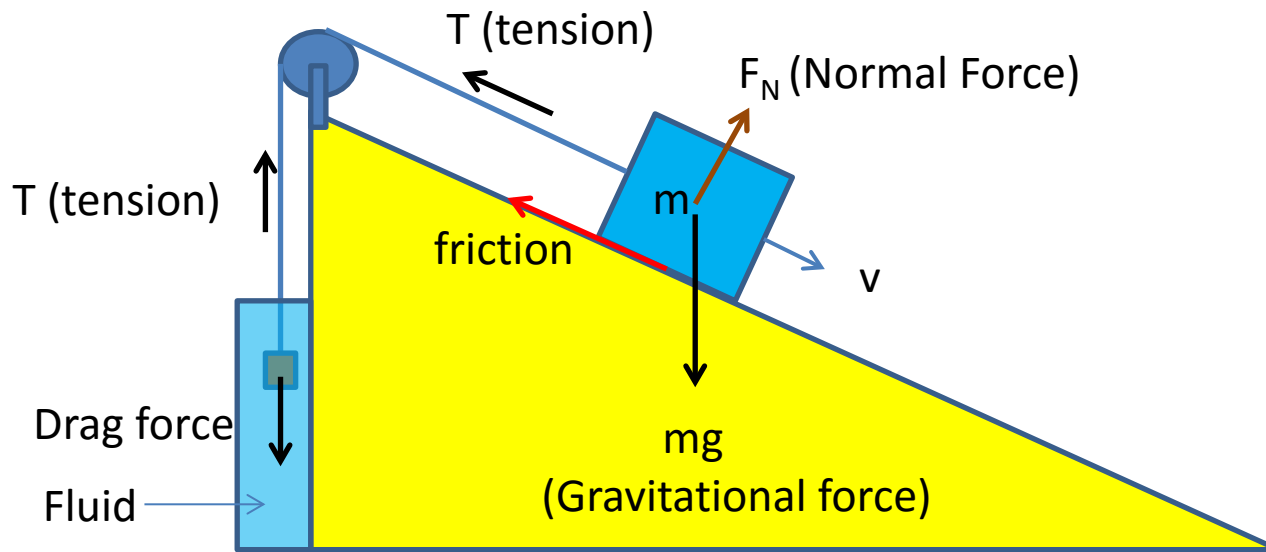
Some forces in the macroscopic world

$$-mg, \quad -\frac{GMm}{r^2}$$

Coulomb

Normal Force,
Tension,
spring force,
Friction,
drag force,

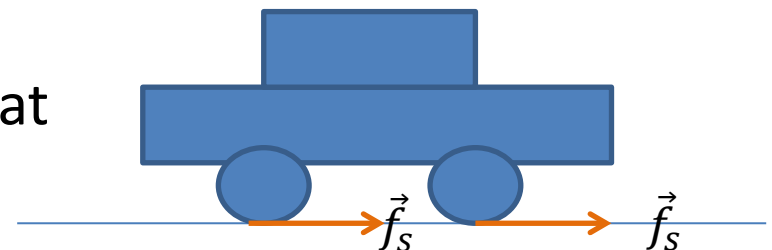
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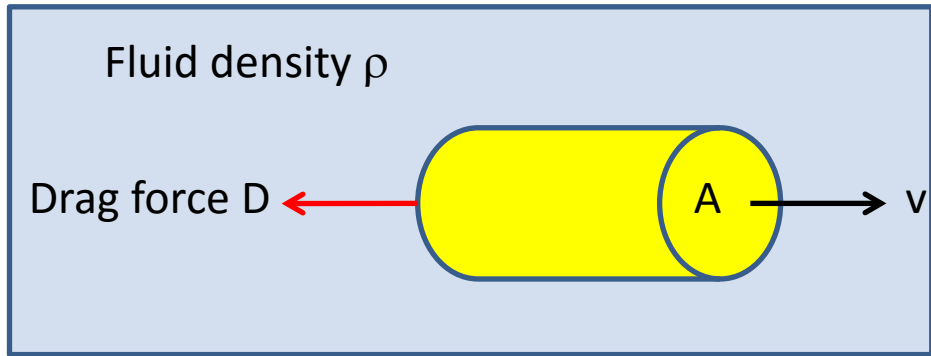


Friction force { Static frictional force \vec{f}_s ; $f_s \leq \overset{\text{Coefficient of static friction}}{\mu_s} \overset{\text{Normal force}}{F_N}$

Kinetic frictional force \vec{f}_k ; $f_k = \overset{\text{Coefficient of kinetic friction}}{\mu_k} F_N$

Note: Without spinning the wheels, the car is subject to STATIC friction that cause it to accelerate.



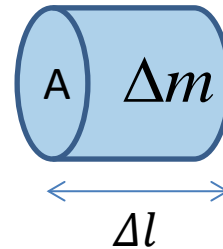


$$\text{Drag force } D = \frac{1}{2} C \rho A v^2 ;$$

C : drag coefficient
(typically 0.4~1.0)

Note: Consider the fluid of mass Δm and volume $A\Delta l$ that lies in the course of the object.

$$\frac{1}{2} \rho A v^2 = \frac{1}{2} \frac{\Delta m}{A\Delta l} A v^2 = \frac{\frac{1}{2} \Delta m v^2}{\Delta l}$$



$\frac{1}{2} \Delta m v^2$ is the kinetic energy of the fluid seen by the object.

If a fraction C of such energy is used to do work on the object

by drag force D , then $C \frac{1}{2} \Delta m v^2 = D \Delta l$.

$$\text{We have } D = \frac{1}{2} C \frac{\Delta m}{\Delta l} v^2 = \frac{1}{2} C \rho A v^2$$

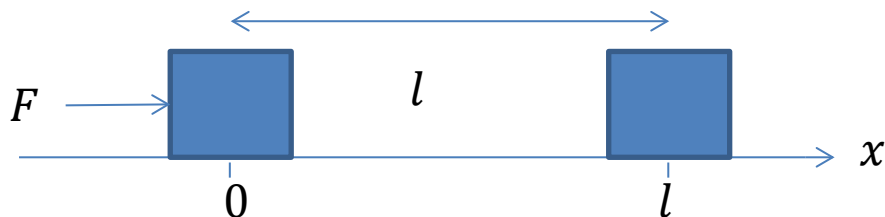
$$\text{Terminal Speed } v_t: F_g = \frac{1}{2} C \rho A v_t^2$$

$$\Rightarrow v_t = \sqrt{\frac{2F_g}{C\rho A}}$$

Chapter 7 Kinetic Energy and Work

Definition of Work:

$$W = F \times l$$

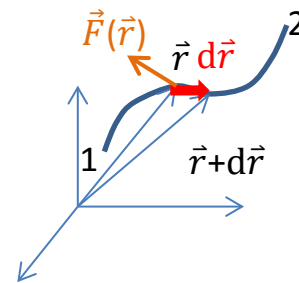


1. When \vec{F} is not parallel to \vec{l} :

$$W = \vec{F} \cdot \vec{l}$$

2. When F is a function of x :

$$W = \int_0^l F dx$$



$$dW = \vec{F} \cdot d\vec{r}$$

$$= (F_x, F_y, F_z) \cdot (dx, dy, dz)$$

$$= F_x dx + F_y dy + F_z dz$$

$$W_{12} = \int_1^2 \vec{F} \cdot d\vec{r}$$

$$= \int_{x_1}^{x_2} F_x dx + \int_{y_1}^{y_2} F_y dy + \int_{z_1}^{z_2} F_z dz$$

Newton's 2nd Law:

$$\text{i) in 1-D } F_{net} = ma = m \frac{dv}{dt}$$

$$\begin{aligned} \Rightarrow W_{net} &= \int_{x_1}^{x_2} F_{net} dx = \int_{x_1}^{x_2} m \frac{dv}{dt} dx = \int_{v_1}^{v_2} m \frac{dx}{dt} dv = m \int_{v_1}^{v_2} v dv \\ &= \frac{1}{2} mv^2 \Big|_{v_1}^{v_2} = \frac{1}{2} mv_2^2 - \frac{1}{2} mv_1^2 \end{aligned}$$

Define the kinetic energy $K = \frac{1}{2} mv^2$

$$\Rightarrow W_{net} = K_2 - K_1 = \Delta K$$

i.e. The work W_{net} done by the net force on an object is equal to the kinetic-energy increase ΔK for that object.

(Work-Kinetic Energy theorem)

$$\text{ii) in 3-D } \vec{F}_{net} = m\vec{a} = m \frac{d\vec{v}}{dt} \Rightarrow (F_{net,x}, F_{net,y}, F_{net,z}) = (m \frac{dv_x}{dt}, m \frac{dv_y}{dt}, m \frac{dv_z}{dt})$$

$$\Rightarrow W_{net} = \int_1^2 \vec{F}_{net} \cdot d\vec{r} = \int_{x_1}^{x_2} F_{net,x} dx + \int_{y_1}^{y_2} F_{net,y} dy + \int_{z_1}^{z_2} F_{net,z} dz$$

$$= \int_{x_1}^{x_2} m \frac{dv_x}{dt} dx + \int_{y_1}^{y_2} m \frac{dv_y}{dt} dy + \int_{z_1}^{z_2} m \frac{dv_z}{dt} dz$$

$$= \int_{v_{x,1}}^{v_{x,2}} m \frac{dx}{dt} dv_x + \int_{v_{y,1}}^{v_{y,2}} m \frac{dy}{dt} dv_y + \int_{v_{z,1}}^{v_{z,2}} m \frac{dz}{dt} dv_z$$

$$= m \int_{v_{x,1}}^{v_{x,2}} v_x dv_x + m \int_{v_{y,1}}^{v_{y,2}} v_y dv_y + m \int_{v_{z,1}}^{v_{z,2}} v_z dv_z = \frac{1}{2} m v_x^2 \Big|_{v_{x,1}}^{v_{x,2}} + \frac{1}{2} m v_y^2 \Big|_{v_{y,1}}^{v_{y,2}} + \frac{1}{2} m v_z^2 \Big|_{v_{z,1}}^{v_{z,2}}$$

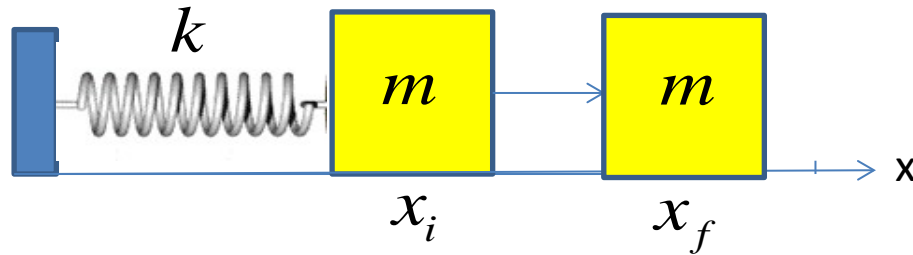
$$= (\frac{1}{2} m v_{x,2}^2 - \frac{1}{2} m v_{x,1}^2) + (\frac{1}{2} m v_{y,2}^2 - \frac{1}{2} m v_{y,1}^2) + (\frac{1}{2} m v_{z,2}^2 - \frac{1}{2} m v_{z,1}^2)$$

$$= \frac{1}{2} m (v_{x,2}^2 + v_{y,2}^2 + v_{z,2}^2) - \frac{1}{2} m (v_{x,1}^2 + v_{y,1}^2 + v_{z,1}^2) = \frac{1}{2} m v_2^2 - \frac{1}{2} m v_1^2$$

$$= K_2 - K_1 = \Delta K$$

$$\Rightarrow \textcolor{red}{W}_{net} = \textcolor{red}{\Delta K} \quad \text{Work-Kinetic Energy theorem}$$

Example: Work done by a spring force



$$F_{net} = F_s = -kx$$

$$W_s = \int_{x_i}^{x_f} F_s dx = \int_{x_i}^{x_f} (-kx) dx = -\frac{1}{2} kx^2 \Big|_{x_i}^{x_f} = \frac{1}{2} kx_i^2 - \frac{1}{2} kx_f^2$$

If $x_i = 0$ and $v_f = 0$, what is v_i ?

$$x_i = 0 \Rightarrow W_s = -\frac{1}{2} kx_f^2$$

By work-kinetic energy theorem,

$$W_s = \Delta K = K_f - K_i = \frac{1}{2} mv_f^2 - \frac{1}{2} mv_i^2$$

$$v_f = 0 \Rightarrow W_s = -\frac{1}{2} mv_i^2$$

$$\Rightarrow -\frac{1}{2} kx_f^2 = -\frac{1}{2} mv_i^2 \Rightarrow v_i = \sqrt{\frac{k}{m}} x_f$$

Note:

Recall that, for a simple spring-and-mass system,

Initial state $[x(0), p(0)] = (x_0, mv_0)$

Final state: $[x(t), p(t)] = [x_m \cos(\omega t + \phi), -m\omega x_m \sin(\omega t + \phi)]$

, where $x_m = \sqrt{x_0^2 + \frac{v_0^2}{\omega^2}}$, $\phi = -\tan^{-1}(\frac{v_0}{x_0\omega})$, $\omega = \sqrt{\frac{k}{m}}$

$$x_i = x_0 = 0 \Rightarrow x_m = \frac{v_0}{\omega} = \frac{v_i}{\omega}$$

$$v_f = 0 \Rightarrow \sin(\omega t + \phi) = 0 \Rightarrow \cos(\omega t + \phi) = 1 \Rightarrow x_f = x_m$$

$$\Rightarrow v_i = \omega x_m = \omega x_f = \sqrt{\frac{k}{m}} x_f$$

Power

$$\text{Power } P = \frac{dW}{dt} = \frac{\vec{F} \cdot d\vec{r}}{dt} = \vec{F} \cdot \frac{d\vec{r}}{dt} = \vec{F} \cdot \vec{v}$$

Chapter 8 Potential Energy and Conservation of Energy

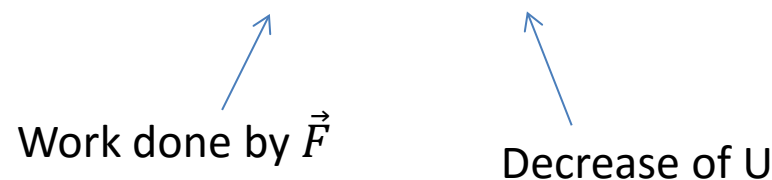
Force

Conservative force e.g. gravitational force, spring force, etc.

Nonconservative force e.g. frictional force, drag force, etc.

For a conservative force \vec{F} , there exists a potential energy function $U(\vec{r})$ such that

$$dW = -dU$$



Work done by \vec{F} Decrease of U

(i) In 1-D $dW = Fdx \Rightarrow Fdx = -dU \Rightarrow$ $F = -\frac{dU}{dx}$

(a) $U(x) = \int dU = -\int Fdx$

(b) $W_{12} = \int_{x_1}^{x_2} Fdx = -\int_{x_1}^{x_2} \frac{dU}{dx} dx = -\int_{U(x_1)}^{U(x_2)} dU = -[U(x_2) - U(x_1)] = -\Delta U$

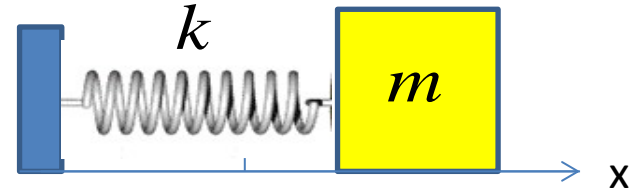
Example: Elastic potential energy

(a) $F = -kx$

$$U(x) = \int dU = -\int F dx = \int kx dx = \frac{1}{2} kx^2 + C$$

Let $U(0) = 0 \Rightarrow C = 0$ (The reference point is set at $x = 0$)

$$\Rightarrow U(x) = \frac{1}{2} kx^2$$



(b)

$$\left\{ \begin{array}{l} W_{12} = \int_{x_1}^{x_2} F dx = -\int_{x_1}^{x_2} kx dx = -\frac{1}{2} kx^2 \Big|_{x_1}^{x_2} = \frac{1}{2} kx_1^2 - \frac{1}{2} kx_2^2 \\ U(x) = \frac{1}{2} kx^2 \Rightarrow \Delta U = U(x_2) - U(x_1) = \frac{1}{2} kx_2^2 - \frac{1}{2} kx_1^2 \end{array} \right.$$

$$\Rightarrow W_{12} = -\Delta U$$

(ii) In 3-D $dW = \vec{F} \cdot d\vec{r} = (F_x, F_y, F_z) \cdot (dx, dy, dz) = F_x dx + F_y dy + F_z dz$

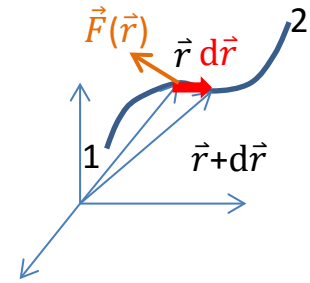
Note: $df(x, y, z) = f(x + dx, y + dy, z + dz) - f(x, y, z) = \frac{\partial f}{\partial x} dx + \frac{\partial f}{\partial y} dy + \frac{\partial f}{\partial z} dz$

Define $\nabla = \hat{i} \frac{\partial}{\partial x} + \hat{j} \frac{\partial}{\partial y} + \hat{k} \frac{\partial}{\partial z} = \left(\frac{\partial}{\partial x}, \frac{\partial}{\partial y}, \frac{\partial}{\partial z} \right) \Rightarrow df = \left(\frac{\partial f}{\partial x}, \frac{\partial f}{\partial y}, \frac{\partial f}{\partial z} \right) \cdot (dx, dy, dz) = \nabla f \cdot d\vec{r}$

$\Rightarrow dU = \nabla U \cdot d\vec{r}$

$dW = \vec{F} \cdot d\vec{r}$

$\Rightarrow dW = -dU \Rightarrow \vec{F} = -\nabla U$



(a) $dU = -dW = -\vec{F} \cdot d\vec{r} \Rightarrow U(\vec{r}) = \int dU = -\int \vec{F} \cdot d\vec{r}$

(b) $W_{12} = \int_1^2 \vec{F} \cdot d\vec{r} = -\int_1^2 \nabla U \cdot d\vec{r} = -\int_1^2 \left(\frac{\partial U}{\partial x} dx + \frac{\partial U}{\partial y} dy + \frac{\partial U}{\partial z} dz \right)$

$= -\int_1^2 dU = -[U(\vec{r}_2) - U(\vec{r}_1)] = -\Delta U$

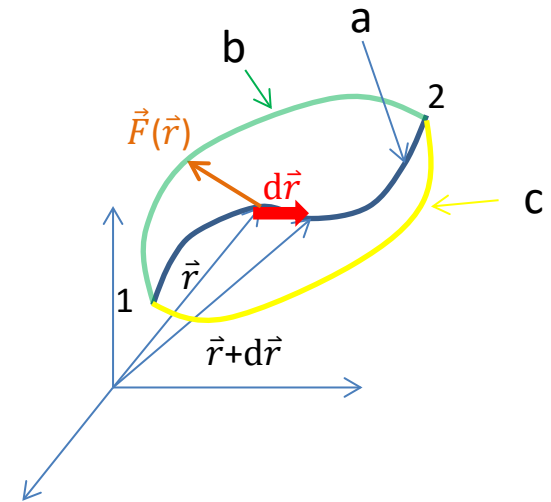
Note:

$$1. W_a = \int_a \vec{F} \cdot d\vec{r} = -[U(\vec{r}_2) - U(\vec{r}_1)] = -\Delta U$$

$$W_b = \int_b \vec{F} \cdot d\vec{r} = -[U(\vec{r}_2) - U(\vec{r}_1)] = -\Delta U$$

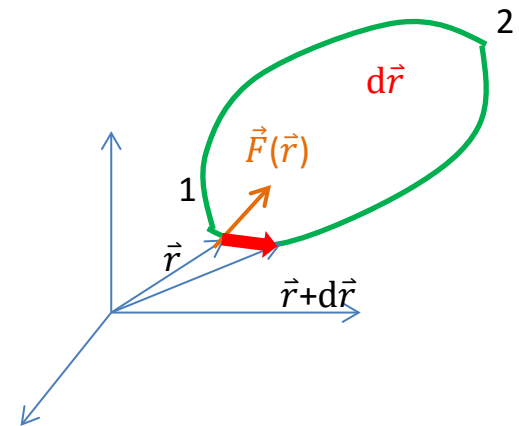
$$W_c = \int_c \vec{F} \cdot d\vec{r} = -[U(\vec{r}_2) - U(\vec{r}_1)] = -\Delta U$$

Work done by a conservative force on a particle moving from point 1 to point 2 is independent of the path the particle takes between the two points.



$$2. W = \oint \vec{F} \cdot d\vec{r} = W_{12} + W_{21} = W_{12} - (-W_{12}) = 0$$

Work done by a conservative force on a particle moving around any closed path is zero.



Example: Gravitational potential energy

(a) $\vec{F} = (0, -mg, 0)$

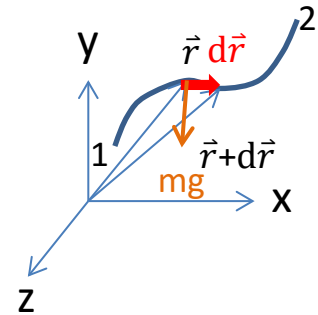
$$U(x, y, z) = \int dU = -\int \vec{F} \cdot d\vec{r} = -\int (0, -mg, 0) \cdot (dx, dy, dz) = \int mg dy = mgy + C$$

Let $U(x_i, y_i, z_i) = 0 \Rightarrow C = -mgy_i$ (The reference point is set at $y = y_i$)

$$\Rightarrow U(x, y, z) = mg(y - y_i)$$

(b)

$$\left\{ \begin{array}{l} W_{12} = \int_1^2 \vec{F} \cdot d\vec{r} = -\int_{y_1}^{y_2} mg dy = -mgy \Big|_{y_1}^{y_2} = mg(y_1 - y_2) \\ U(x, y, z) = mg(y - y_i) \Rightarrow \Delta U = U(x_2, y_2, z_2) - U(x_1, y_1, z_1) \\ \qquad \qquad \qquad = mg(y_2 - y_i) - mg(y_1 - y_i) \\ \qquad \qquad \qquad = mg(y_2 - y_1) \end{array} \right.$$



$$\Rightarrow W_{12} = -\Delta U$$

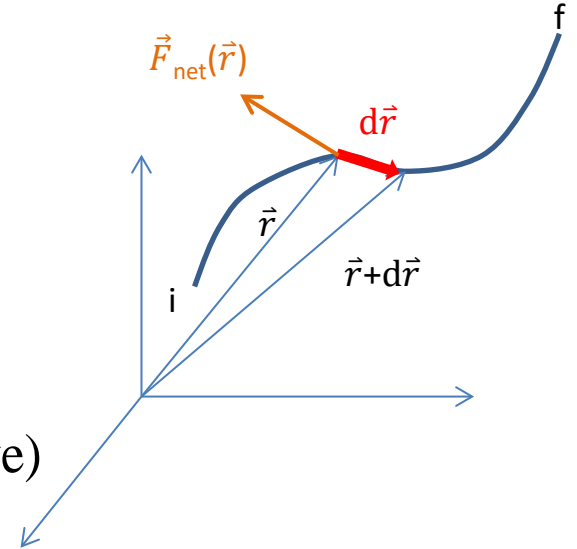
Conservation of Mechanical Energy

Work-Kinetic Energy Theorem: $W_{net} = \Delta K = K_f - K_i$

If \vec{F}_{net} is conservative $\Rightarrow W_{net} = -\Delta U = U_i - U_f$

$$\Rightarrow K_f - K_i = U_i - U_f$$

$$\Rightarrow K_f + U_f = K_i + U_i \text{ (when the net force is conservative)}$$



Define Mechanical Energy $E_{mec} = K + U$

$\Rightarrow E_{mec,f} = E_{mec,i}$ Conservation of Mechanical Energy

(when the net force is conservative)

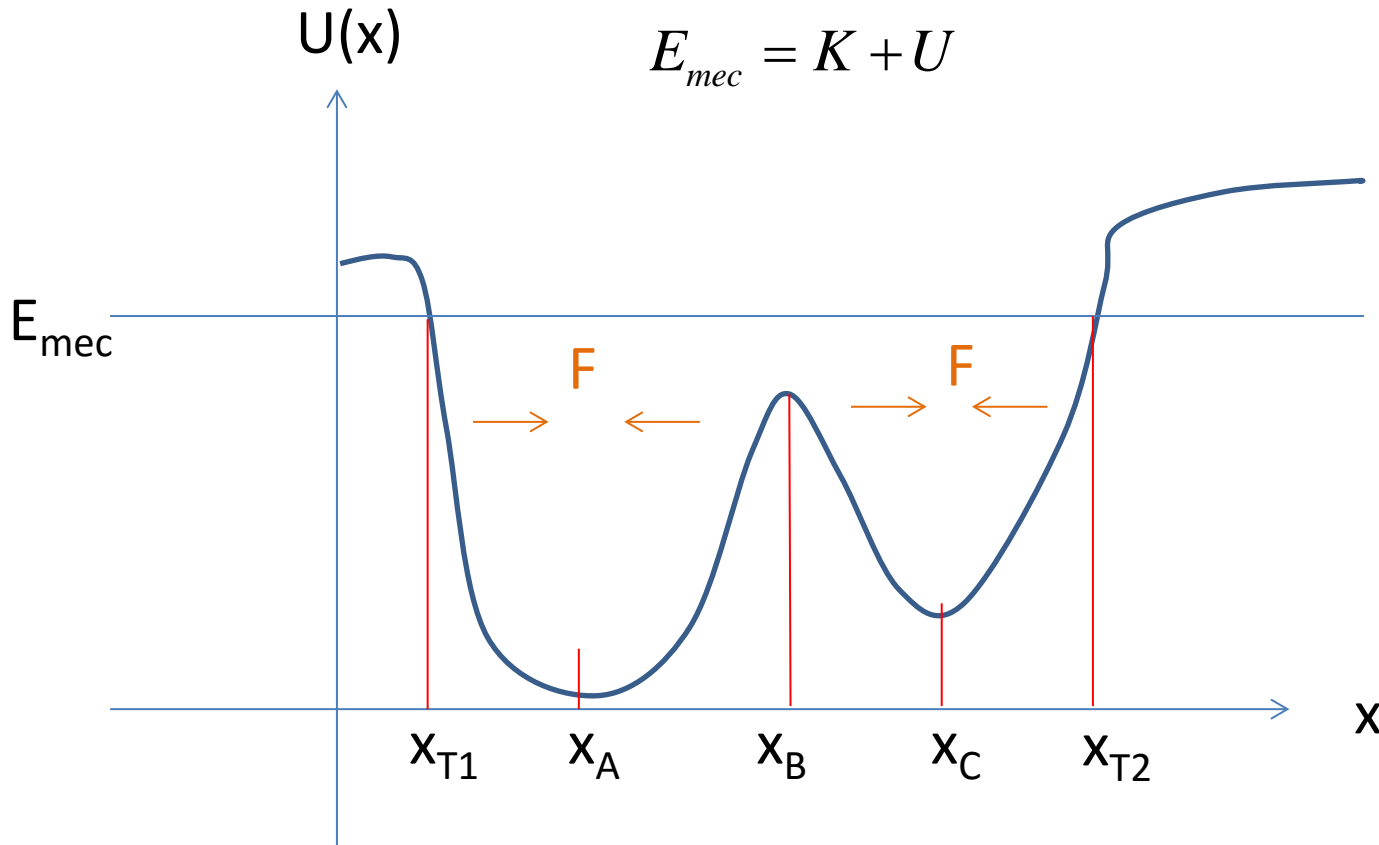
Note: U is determined by the force and the reference point through $\vec{F} = -\nabla U$.

$K = \frac{1}{2}mv^2$ is determined by the force and the initial velocity through $\vec{F} = m\frac{d\vec{v}}{dt}$.

Potential Energy Curve

$$F = -\frac{dU}{dx}$$

$$E_{mec} = K + U$$



x_{T1}, x_{T2} Turning Points;
 x_A, x_C Stable Equilibrium Points
 x_B Unstable Equilibrium Point

Chapter 9 Center of Mass and Linear Momentum

Newton's 2nd Law for each particle

$$\vec{F}_1 = \vec{F}_{ext,1} + \sum_{i \neq 1} \vec{F}_{i1} = m_1 \vec{a}_1$$

$$\vec{F}_2 = \vec{F}_{ext,2} + \sum_{i \neq 2} \vec{F}_{i2} = m_2 \vec{a}_2$$

\vdots

$$\vec{F}_n = \vec{F}_{ext,n} + \sum_{i \neq n} \vec{F}_{in} = m_n \vec{a}_n$$

$$\vec{F} = \sum_i \vec{F}_i = \sum_i \vec{F}_{ext,i} + \sum_i \sum_{j \neq i} \vec{F}_{ji} = \sum_i m_i \vec{a}_i$$

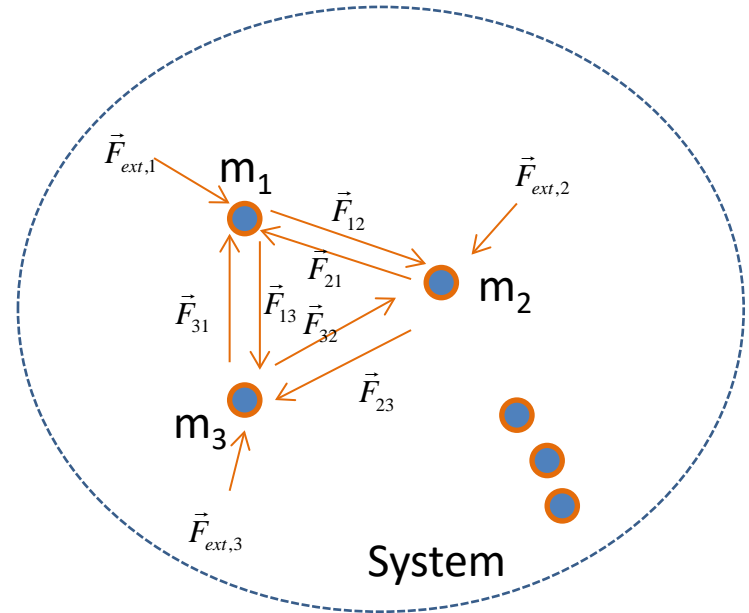
Newton's 3rd Law $\Rightarrow \vec{F}_{ji} = -\vec{F}_{ij} \Rightarrow \sum_i \sum_{j \neq i} \vec{F}_{ji} = 0$

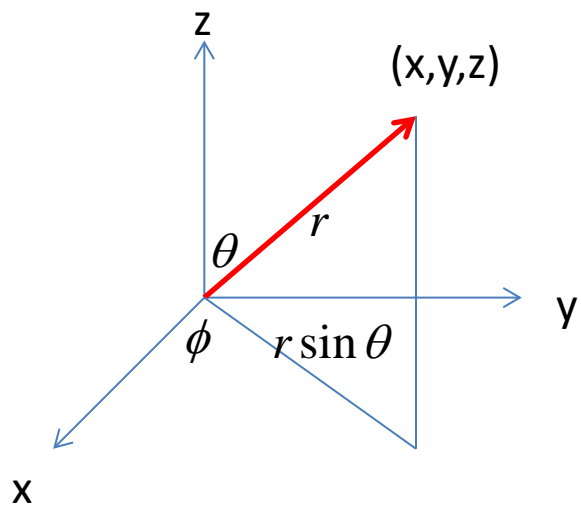
$$\Rightarrow \vec{F} = \sum_i \vec{F}_{ext,i} = \sum_i m_i \vec{a}_i = \sum_i m_i \frac{d^2 \vec{r}_i}{dt^2} = \sum_i \frac{d^2}{dt^2} (m_i \vec{r}_i) = \frac{d^2}{dt^2} \sum_i (m_i \vec{r}_i)$$

Let the total mass $\sum_i m_i = M \Rightarrow \vec{F} = M \frac{d^2}{dt^2} \left[\frac{1}{M} \sum_i (m_i \vec{r}_i) \right]$

Define center of mass $\vec{r}_{COM} = \frac{1}{M} \sum_i (m_i \vec{r}_i) \Rightarrow \vec{a}_{COM} = \frac{d^2 \vec{r}_{COM}}{dt^2} = \frac{d^2}{dt^2} \left[\frac{1}{M} \sum_i (m_i \vec{r}_i) \right]$

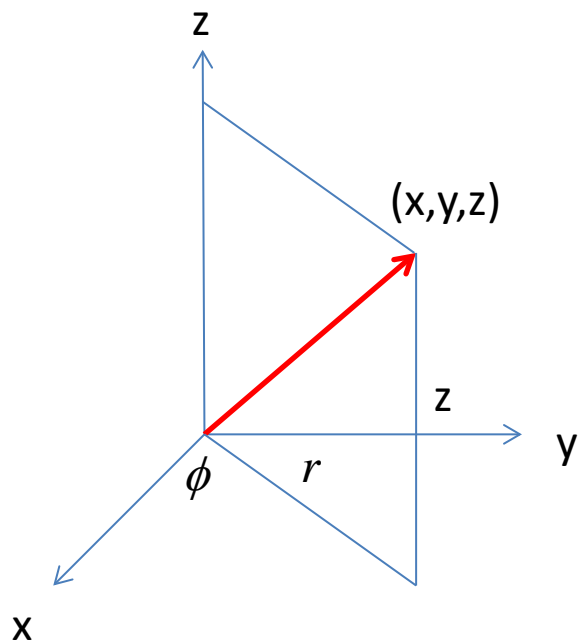
$\Rightarrow \vec{F} = M \vec{a}_{COM}$ Note: For continuous mass distribution $\vec{r}_{COM} = \frac{1}{M} \int_V \rho \vec{r} dV$; $M = \int_V \rho dV$





Spherical coordinates (r, θ, ϕ)

$$\begin{cases} x = r \sin \theta \cos \phi \\ y = r \sin \theta \sin \phi \\ z = r \cos \theta \end{cases}$$



Cylindrical coordinates (r, ϕ, z)

$$\begin{cases} x = r \cos \phi \\ y = r \sin \phi \\ z = z \end{cases}$$

Note:

$dV = dxdydz$ in Cartesian coordinates (x, y, z)

$dV = (dr)(r d\theta)(r \sin \theta d\phi) = r^2 \sin \theta dr d\theta d\phi$ in spherical coordinates (r, θ, ϕ)

$dV = (dr)(r d\phi)(dz) = r dr d\phi dz$ in cylindrical coordinates (r, ϕ, z)

Examples:

I. The volume of a cube of side length a

$$V = \int_V dxdydz = \int_0^a \int_0^a \int_0^a dxdydz = \int_0^a \left[\int_0^a \left(\int_0^a dx \right) dy \right] dz = a^3$$

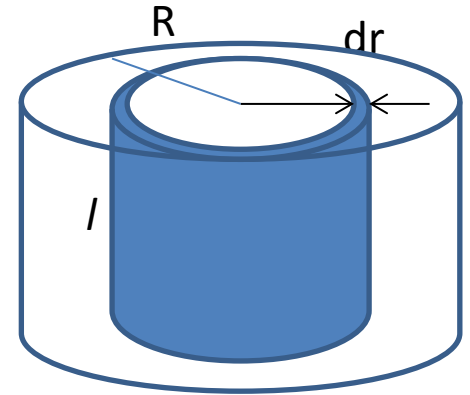
II. The volume of a sphere of radius R

$$\begin{aligned} V &= \int_V r^2 \sin \theta dr d\theta d\phi = \int_0^{2\pi} \int_0^\pi \int_0^R r^2 \sin \theta dr d\theta d\phi = \int_0^{2\pi} \left[\int_0^\pi \sin \theta \left(\int_0^R r^2 dr \right) d\theta \right] d\phi \\ &= \frac{R^3}{3} \int_0^{2\pi} \left[\int_0^\pi \sin \theta d\theta \right] d\phi = \frac{2R^3}{3} \int_0^{2\pi} d\phi = \frac{4\pi R^3}{3} \end{aligned}$$

III. The volume of a cylinder of radius R and a length l

$$(i) V = \int_0^l \int_0^{2\pi} \int_0^R r dr d\phi dz = \int_0^l \left[\int_0^{2\pi} \left(\int_0^R r dr \right) d\phi \right] dz = \int_0^l \left[\int_0^{2\pi} \frac{1}{2} R^2 d\phi \right] dz = \int_0^l \pi R^2 dz = \pi R^2 l$$

$$(ii) dV = 2\pi r \times l \times dr; V = \int_V dV = \int_0^R 2\pi r l dr = 2\pi l \int_0^R r dr = 2\pi l \frac{1}{2} R^2 = \pi R^2 l$$



Note: Area of the cone:

$$(i) Area = \pi \left(\sqrt{R^2 + h^2} \right)^2 \times \frac{\sqrt{R^2 + h^2}}{2\pi} = \pi R \sqrt{R^2 + h^2}$$

$$(ii) \tan \theta = \frac{R}{h}; \quad \cos \theta = \frac{h}{\sqrt{R^2 + h^2}}$$

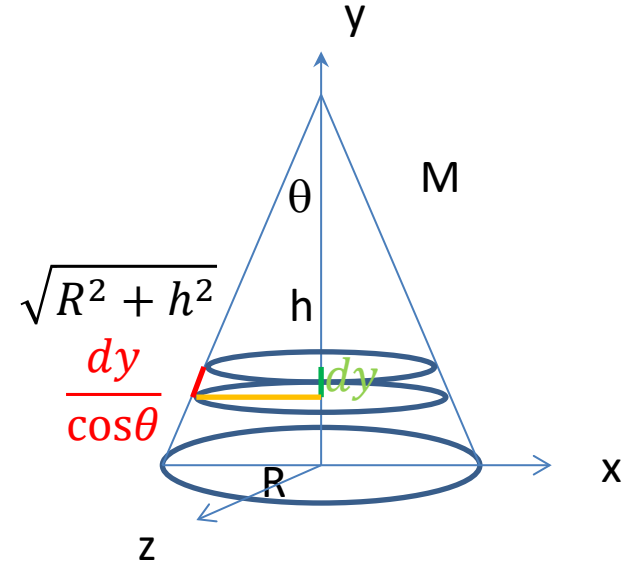
$$dA = 2\pi \times (h - y) \tan \theta \times \frac{dy}{\cos \theta} = \frac{2\pi R}{h^2} \sqrt{R^2 + h^2} (h - y) dy$$

$$Area = \int dA = \frac{2\pi R}{h^2} \sqrt{R^2 + h^2} \int_0^h (h - y) dy = \frac{2\pi R}{h^2} \sqrt{R^2 + h^2} \left(\frac{h^2}{2} \right) = \pi R \sqrt{R^2 + h^2}$$

Center of Mass

$$\sigma = \frac{M}{\pi R \sqrt{R^2 + h^2}}; \quad \text{By symmetry } x_{COM} = z_{COM} = 0.$$

$$\begin{aligned} y_{COM} &= \frac{1}{M} \int_0^h y \sigma dA = \frac{1}{M} \int_0^h y \frac{M}{\pi R \sqrt{R^2 + h^2}} \frac{2\pi R}{h^2} \sqrt{R^2 + h^2} (h - y) dy = \frac{2}{h^2} \int_0^h (hy - y^2) dy \\ &= \frac{2}{h^2} \times \frac{h^3}{6} = \frac{1}{3} h \Rightarrow \text{Center of Mass} = \left(0, \frac{1}{3} h, 0 \right) \end{aligned}$$



Define Linear Momentum $\vec{p} = m\vec{v} = m \frac{d\vec{r}}{dt}$

Newton's 2nd Law $\vec{F}_{net} = m\vec{a} = m \frac{d\vec{v}}{dt} = \frac{d}{dt}(m\vec{v}) = \frac{d\vec{p}}{dt}$

If $\vec{F}_{net} = 0 \Rightarrow \frac{d\vec{p}}{dt} = 0 \Rightarrow$ The momentum \vec{p} is constant.

Consider a system of n particles

$$\vec{P} = \sum_{i=1}^n \vec{p}_i = \sum_{i=1}^n m_i \vec{v}_i = \sum_{i=1}^n m_i \frac{d\vec{r}_i}{dt} = \frac{d}{dt} \left(\sum_{i=1}^n m_i \vec{r}_i \right) = M \frac{d}{dt} \left(\frac{1}{M} \sum_{i=1}^n m_i \vec{r}_i \right) = M \frac{d\vec{r}_{COM}}{dt} = M\vec{v}_{COM}$$

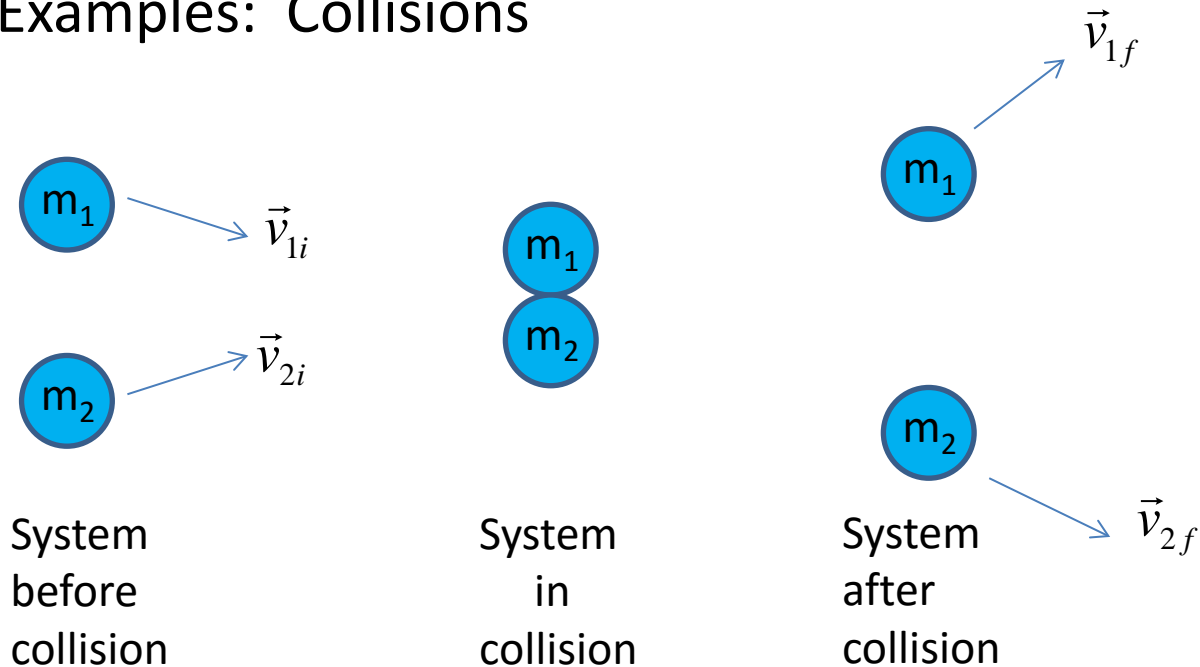
$$\text{Since } \vec{F} = \sum_i \vec{F}_{ext,i} = M\vec{a}_{COM}, \quad \sum_i \vec{F}_{ext,i} = M \frac{d\vec{v}_{COM}}{dt} = \frac{d}{dt} (M\vec{v}_{COM}) = \frac{d\vec{P}}{dt} = \frac{d}{dt} \left(\sum_{i=1}^n \vec{p}_i \right)$$

$$\sum_i \vec{F}_{ext,i} = \frac{d}{dt} \left(\sum_{i=1}^n \vec{p}_i \right)$$

\Rightarrow In a system of particles, if the sum of external force $\sum_i \vec{F}_{ext,i}$ is zero

then $\frac{d}{dt} \left(\sum_{i=1}^n \vec{p}_i \right)$ is zero and the sum of momentum $\sum_{i=1}^n \vec{p}_i$ is constant.

Examples: Collisions



No external force is involved

$$\sum_i \vec{F}_{ext,i} = 0 \Rightarrow \frac{d}{dt}(\sum_i \vec{p}_i) = 0 \Rightarrow \sum_i \vec{p}_i \text{ is constant}$$

(Note $\vec{P} = \sum_{i=1}^n \vec{p}_i = M\vec{v}_{COM}$; So $\sum_i \vec{p}_i$ is constant $\Rightarrow \vec{v}_{COM}$ is constant)

Therefore, we have

$$m_1 \vec{v}_{1i} + m_2 \vec{v}_{2i} = m_1 \vec{v}_{1f} + m_2 \vec{v}_{2f} \quad \text{Momentum Conservation}$$

Special Cases:

1. Elastic Collisions \Rightarrow Total kinetic energy is conserved

$$\frac{1}{2}m_1v_{1i}^2 + \frac{1}{2}m_2v_{2i}^2 = \frac{1}{2}m_1v_{1f}^2 + \frac{1}{2}m_2v_{2f}^2 \quad \text{Kinetic energy conservation}$$

2. Completely Inelastic Collisions \Rightarrow Two bodies merge

$$\vec{v}_{2f} = \vec{v}_{1f}$$

Typical Problem: Given $m_1, m_2, \vec{v}_{1i}, \vec{v}_{2i} \Rightarrow$ Find \vec{v}_{1f} and \vec{v}_{2f}

1. Elastic collisions in one dimension

$$\left. \begin{aligned} m_1v_{1i} + m_2v_{2i} &= m_1v_{1f} + m_2v_{2f} \\ \frac{1}{2}m_1v_{1i}^2 + \frac{1}{2}m_2v_{2i}^2 &= \frac{1}{2}m_1v_{1f}^2 + \frac{1}{2}m_2v_{2f}^2 \end{aligned} \right\} \begin{aligned} v_{1f} &= \frac{m_1 - m_2}{m_1 + m_2}v_{1i} + \frac{2m_2}{m_1 + m_2}v_{2i} \\ v_{2f} &= \frac{2m_1}{m_1 + m_2}v_{1i} + \frac{m_2 - m_1}{m_1 + m_2}v_{2i} \end{aligned}$$

2. Completely inelastic collisions in one dimension

$$\left. \begin{aligned} m_1v_{1i} + m_2v_{2i} &= m_1v_{1f} + m_2v_{2f} \\ v_{1f} &= v_{2f} = v_f \end{aligned} \right\} v_f = \frac{m_1v_{1i} + m_2v_{2i}}{m_1 + m_2}$$

Note: In a 1-D elastic collision,

$$v_{1f} = \frac{m_1 - m_2}{m_1 + m_2} v_{1i} + \frac{2m_2}{m_1 + m_2} v_{2i} \quad ; \quad v_{2f} = \frac{2m_1}{m_1 + m_2} v_{1i} + \frac{m_2 - m_1}{m_1 + m_2} v_{2i}$$

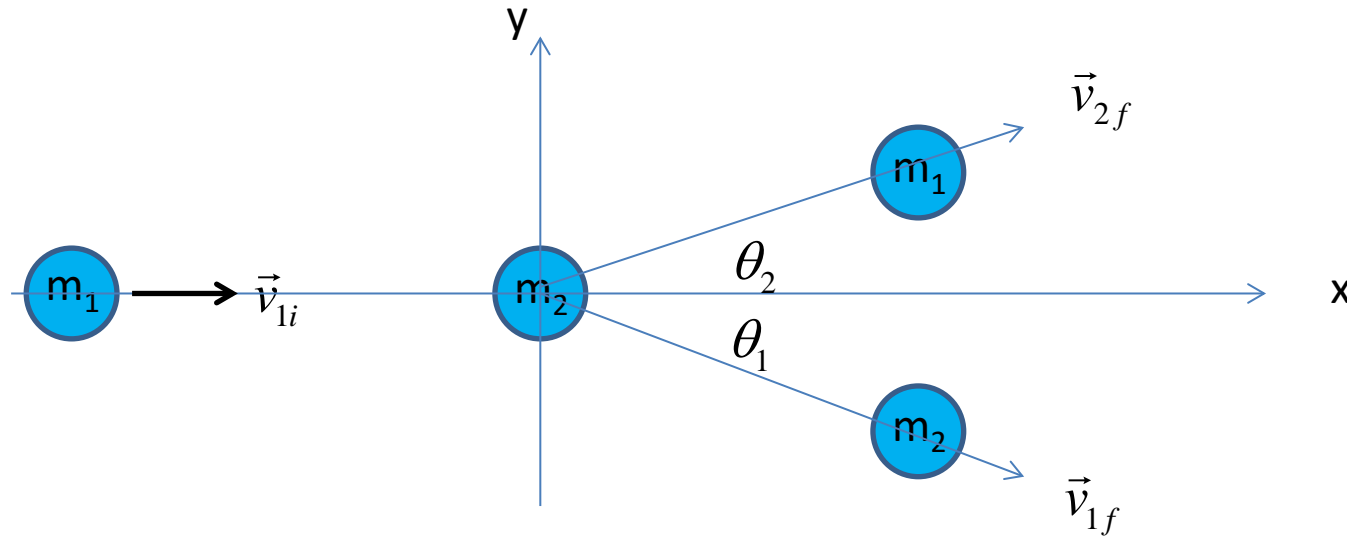
Some cases of special interest for a stationary target (i.e. $v_{2i} = 0$)

(i) Equal Mass $m_1 = m_2 \Rightarrow v_{1f} = 0$; $v_{2f} = v_{1i}$

(ii) A massive target $m_2 \gg m_1 \Rightarrow v_{1f} \simeq -v_{1i}$; $v_{2f} \simeq \frac{2m_1}{m_2} v_{1i}$

(iii) A massive projectile $m_1 \gg m_2 \Rightarrow v_{1f} \simeq v_{1i}$; $v_{2f} \simeq 2v_{1i}$

Collision in two dimensions



Glancing Collision

Momentum conservation

$$\text{x-component } m_1 v_{1i} = m_1 v_{1f} \cos \theta_1 + m_2 v_{2f} \cos \theta_2$$

$$\text{y-component } 0 = -m_1 v_{1f} \sin \theta_1 + m_2 v_{2f} \sin \theta_2$$

Kinetic energy conservation

$$\frac{1}{2} m_1 v_{1i}^2 = \frac{1}{2} m_1 v_{1f}^2 + \frac{1}{2} m_2 v_{2f}^2$$

Given one of the following 4 variables $v_{1f}, v_{2f}, \theta_1, \theta_2$

\Rightarrow *problem can be solved*

Impulse

Definition: Impulse $\vec{J} = \int_{t_1}^{t_2} \vec{F}(t) dt$

I. For a particle of mass m ,

Newton's 2nd Law $\vec{F}_{net} = m\vec{a} = m \frac{d\vec{v}}{dt} = \frac{d(m\vec{v})}{dt} = \frac{d\vec{p}}{dt}$

Therefore, **impulse by the net force** $\vec{J} = \int_{t_1}^{t_2} \vec{F}_{net} dt = \int_{\vec{p}(t_1)}^{\vec{p}(t_2)} d\vec{p} = \vec{p}(t_2) - \vec{p}(t_1) = \Delta\vec{p}$

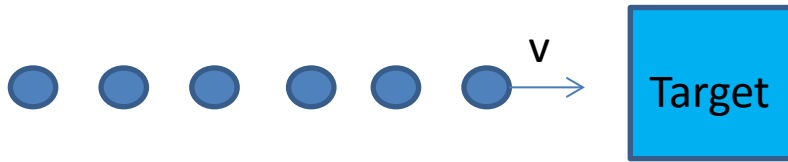
$\vec{J} = \Delta\vec{p}$ Linear momentum-impulse theorem

II. For a system of particles,

$$\sum_i \vec{F}_{ext,i} = \frac{d}{dt} \left(\sum_{i=1}^n \vec{p}_i \right)$$

$$\vec{J} = \int_{t_1}^{t_2} \left(\sum_i \vec{F}_{ext,i} \right) dt = \left[\sum_{i=1}^n \vec{p}_i(t_2) \right] - \left[\sum_{i=1}^n \vec{p}_i(t_1) \right] = \vec{P}(t_2) - \vec{P}(t_1) = \Delta\vec{P}$$

Example I. As shown in the figure, n identical projectiles collide with a target during time interval Δt . What is the average force exerted on the target?



$\Delta \vec{p}_{\text{target}} = -\Delta \vec{p}_{\text{projectile}}$ for each collision. (momentum conservation)

Let $\Delta \vec{p} = \Delta \vec{p}_{\text{projectile}}$

The impulse \vec{J} on the target during Δt is $\vec{J} = -n\Delta \vec{p}$

The average force \vec{F}_{avg} on the target is

$$\vec{F}_{\text{avg}} = \frac{\vec{J}}{\Delta t} = \frac{-n\Delta \vec{p}}{\Delta t} = \frac{-n(m\Delta \vec{v})}{\Delta t} = -\frac{(nm)\Delta \vec{v}}{\Delta t} = -\frac{\Delta m}{\Delta t} \Delta \vec{v}$$

where $\Delta \vec{v} = \vec{v}_f - \vec{v}_i$ for the projectile; $\Delta m = nm$

Example II. A rocket of mass $M(t)$ traveling in a straight course is ejecting exhaust products at velocity v_{rel} relative to the rocket. (a) What is the acceleration a of the rocket? (b) If the velocity is v_i at time t_i , what is the velocity v_f at time t_f ?

At time t , the rocket has mass $M(t)$ traveling with velocity $v(t)$.

Let the rocket at time t be the system of interest.



The momentum of the system at time t is therefore Mv .

At time $t + dt$, the rocket has mass $M(t + dt)$ traveling with velocity $v(t + dt)$.

Note that exhaust products of mass $M(t) - M(t + dt) = -dM$



(recall $\frac{dM}{dt} = \frac{M(t + dt) - M(t)}{dt}$) were ejected during the time interval dt .

So, the system at time $t + dt$ includes the rocket and the exhaust products of mass $-dM$ ejected from t to $t + dt$. We also note $v(t + dt) = v(t) + dv$ and the velocity of the exhaust products is $v(t) + dv - v_{rel}$.

The momentum of the system at time $t + dt$ is

$$\begin{aligned} M(t + dt)v(t + dt) + (-dM)[v(t + dt) - v_{rel}] &= (M + dM)(v + dv) + (-dM)(v + dv - v_{rel}) \\ &= Mv + Mdv + v_{rel}dM \end{aligned}$$

$$dP = (Mv + Mdv + v_{rel}dM) - Mv = Mdv + v_{rel}dM$$

(A) In the outer space

No external force exerting on the system, $dP = 0$

$$\Rightarrow Mdv + v_{rel}dM = 0 \Rightarrow -v_{rel} \frac{dM}{dt} = M \frac{dv}{dt} = Ma$$

Let $R = -\frac{dM}{dt}$ (fuel consumption rate)

We have $Rv_{rel} = Ma$ (1st Rocket Equation)

Define thrust $T = Rv_{rel} \Rightarrow T = Ma$

$$Mdv + v_{rel}dM = 0 \Rightarrow dv = -v_{rel} \frac{1}{M} dM \Rightarrow \int_{v_i}^{v_f} dv = -v_{rel} \int_{M_i}^{M_f} \frac{1}{M} dM$$

$$\Rightarrow v_f - v_i = -v_{rel} (\ln M_f - \ln M_i) = v_{rel} (\ln M_i - \ln M_f) = v_{rel} \ln \frac{M_i}{M_f}$$

$$\Rightarrow v_f - v_i = v_{rel} \ln \frac{M_i}{M_f} \text{ (2nd Rocket Equation)}$$



$$dP = (Mv + Mdv + v_{rel}dM) - Mv = Mdv + v_{rel}dM$$

(B) Firing vertically on the ground

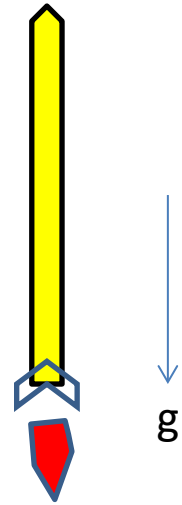
$$dP = dJ = F_g dt = -Mgdt \text{ (linear momentum-impulse theorem)}$$

$$\Rightarrow Mdv + v_{rel}dM = -Mgdt \Rightarrow -v_{rel} \frac{dM}{dt} = M \frac{dv}{dt} + Mg \frac{dt}{dt} = Ma + Mg$$

$$R = -\frac{dM}{dt} \text{ (fuel consumption rate)}$$

$$\text{We have } Rv_{rel} = M(a + g)$$

$$\text{Thrust } T = Rv_{rel} \Rightarrow T = M(a + g)$$



$$Mdv + v_{rel}dM = -Mgdt \Rightarrow dv = -v_{rel} \frac{1}{M} dM - gdt \Rightarrow \int_{v_i}^{v_f} dv = -v_{rel} \int_{M_i}^{M_f} \frac{1}{M} dM - g \int_{t_i}^{t_f} dt$$

$$\Rightarrow v_f - v_i = -v_{rel} (\ln M_f - \ln M_i) - g(t_f - t_i) = v_{rel} (\ln M_i - \ln M_f) - g(t_f - t_i)$$

$$= v_{rel} \ln \frac{M_i}{M_f} - g(t_f - t_i)$$

$$\Rightarrow v_f - v_i = v_{rel} \ln \frac{M_i}{M_f} - g(t_f - t_i)$$

Chapter 10 Rotation

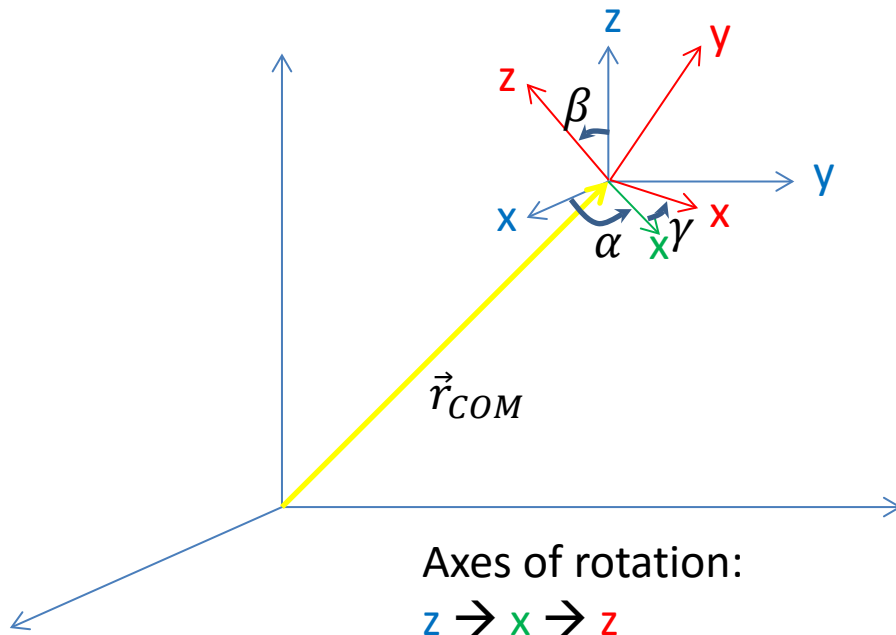
(Rotation of a rigid body about a fixed axis)

Rigid body : relative positions between particles are fixed (independent of time).

Degrees of freedom :

$3 \times$ number of particles $\rightarrow 6$ (3 for position, 3 for orientation)

Position $\vec{r}_{COM} = (x_{COM}, y_{COM}, z_{COM})$; Orientation α, β, γ (Euler angles)



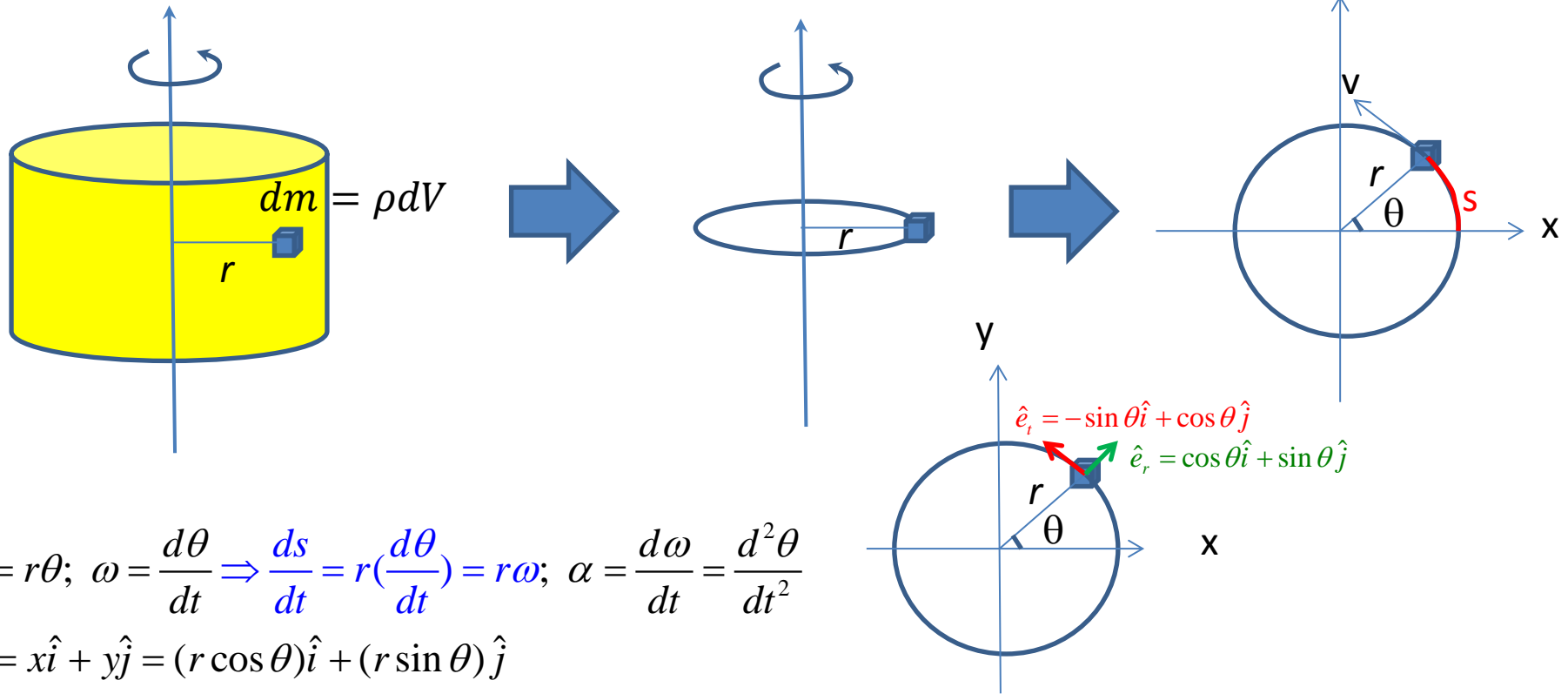
Motion in 6-dimensional space

Special cases:

(i) Rotation about a fixed axis
Variable θ only \rightarrow one dimensional motion.

(ii) Rolling: variables x_{COM}, θ
Constraint: $x_{COM} = R\theta \rightarrow$ one dimensional motion.

Rotation of a rigid body about a fixed axis



$$s = r\theta; \quad \omega = \frac{d\theta}{dt} \Rightarrow \frac{ds}{dt} = r\left(\frac{d\theta}{dt}\right) = r\omega; \quad \alpha = \frac{d\omega}{dt} = \frac{d^2\theta}{dt^2}$$

$$\vec{r} = x\hat{i} + y\hat{j} = (r \cos \theta)\hat{i} + (r \sin \theta)\hat{j}$$

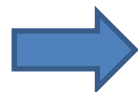
$$\vec{v} = \frac{d\vec{r}}{dt} = \frac{d(r \cos \theta)}{dt}\hat{i} + \frac{d(r \sin \theta)}{dt}\hat{j} = -r \sin \theta \frac{d\theta}{dt}\hat{i} + r \cos \theta \frac{d\theta}{dt}\hat{j} = -r\omega \sin \theta \hat{i} + r\omega \cos \theta \hat{j}$$

$$\Rightarrow v = \sqrt{(-r\omega \sin \theta)^2 + (r\omega \cos \theta)^2} = r\omega \quad \left(= \frac{ds}{dt}\right)$$

$$\begin{aligned} \vec{a} &= \frac{d\vec{v}}{dt} = -\frac{d(r\omega \sin \theta)}{dt}\hat{i} + \frac{d(r\omega \cos \theta)}{dt}\hat{j} = -\left(r \frac{d\omega}{dt} \sin \theta + r\omega \cos \theta \frac{d\theta}{dt}\right)\hat{i} + \left(r \frac{d\omega}{dt} \cos \theta - r\omega \sin \theta \frac{d\theta}{dt}\right)\hat{j} \\ &= r\alpha(-\sin \theta \hat{i} + \cos \theta \hat{j}) + r\omega^2(-\cos \theta \hat{i} - \sin \theta \hat{j}) = r\alpha \hat{e}_t - r\omega^2 \hat{e}_r = \vec{a}_t + \vec{a}_r \end{aligned}$$

$$\Rightarrow \vec{a}_t = r\alpha \hat{e}_t \text{ (tangential acceleration); } \vec{a}_r = -r\omega^2 \hat{e}_r \text{ (centripetal acceleration)}$$

Recall Motion with constant acceleration in one dimension



Rotation of a rigid body about a fixed axis with constant angular acceleration
(a one dimension problem)

$$\frac{dv}{dt} = a \text{ (a constant)} \Rightarrow dv = a dt$$

$$\Rightarrow \int dv = \int a dt \Rightarrow v = at + c$$

$$\text{Let } v(0) = v_0 \Rightarrow c = v_0 \Rightarrow v(t) = at + v_0$$

$$\frac{dx}{dt} = v(t) = at + v_0 \Rightarrow dx = (at + v_0) dt$$

$$\int dx = \int (at + v_0) dt \Rightarrow x = \frac{1}{2} at^2 + v_0 t + c$$

$$\text{Let } x(0) = x_0 \Rightarrow c = x_0$$

$$\Rightarrow x(t) = \frac{1}{2} at^2 + v_0 t + x_0$$

$$\frac{d\omega}{dt} = \alpha \text{ (a constant)}$$

$$\frac{d\theta(t)}{dt} = \omega(t)$$

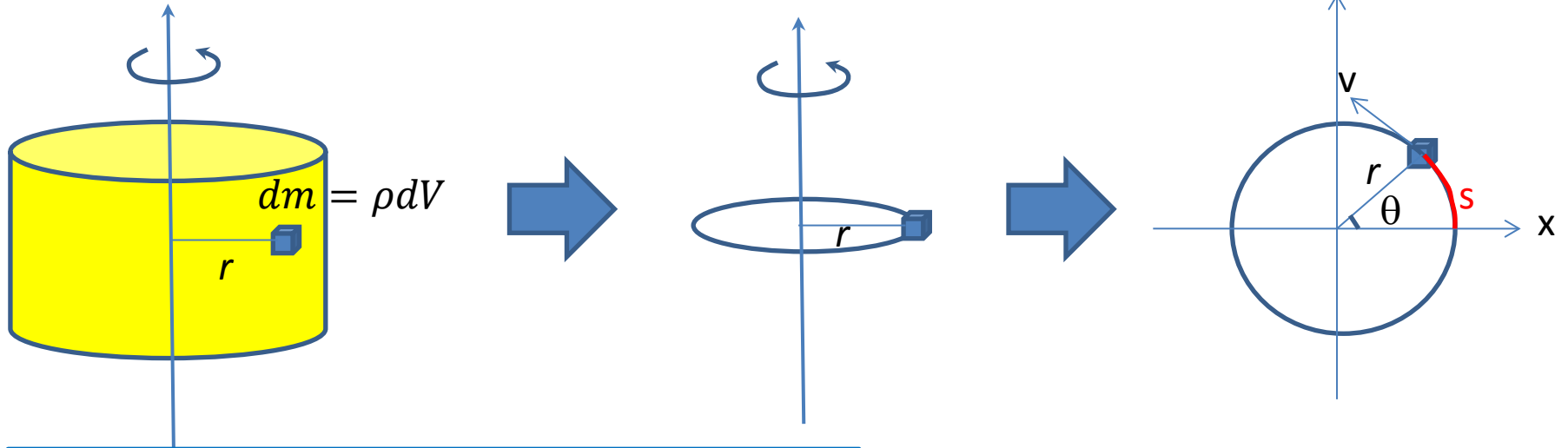
$$\text{Let } \omega(0) = \omega_0; \theta(0) = \theta_0$$



$$\omega(t) = \alpha t + \omega_0$$

$$\theta(t) = \frac{1}{2} \alpha t^2 + \omega_0 t + \theta_0$$

Kinetic energy of a rigid body rotating about a fixed axis.



$$v = \sqrt{(-r\omega \sin \theta)^2 + (r\omega \cos \theta)^2} = r\omega$$

$$dK = \frac{1}{2} dm \times v^2 = \frac{1}{2} \rho dV \times r^2 \omega^2 = \frac{1}{2} (r^2 \rho dV) \omega^2$$

$$K = \int_V dK = \frac{1}{2} \left(\int_V r^2 \rho dV \right) \omega^2$$

Define rotational inertia $I = \int_V r^2 \rho dV$
(moment of inertia)

We have $K = \frac{1}{2} I \omega^2$.

For discrete distribution of mass

$$I = \sum_i m_i r_i^2$$

$$K = \frac{1}{2} I \omega^2.$$

Note: r is the distance to
the axis of rotation
(not the origin)!

Examples

- I. A uniform solid sphere of radius R and mass M rotating about any diameter.
(Select a spherical coordinate system with origin at the center of the sphere.)

$$I = \int_V (r \sin \theta)^2 \rho dV$$

$$dV = r^2 \sin \theta dr d\theta d\phi$$

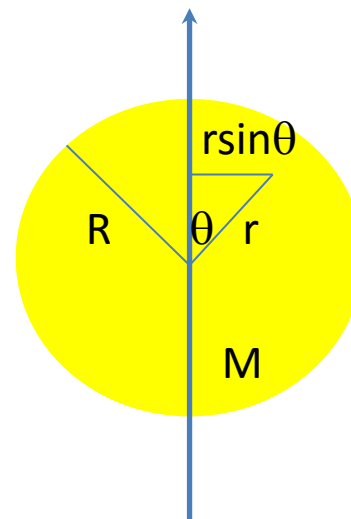
$$\Rightarrow I = \rho \int_0^{2\pi} \int_0^\pi \int_0^R r^4 \sin^3 \theta dr d\theta d\phi = \rho \int_0^{2\pi} \int_0^\pi \sin^3 \theta \left(\int_0^R r^4 dr \right) d\theta d\phi$$

$$= \frac{R^5}{5} \rho \int_0^{2\pi} \left(\int_0^\pi \sin^3 \theta d\theta \right) d\phi = \frac{R^5}{5} \rho \int_0^{2\pi} \left[\frac{1}{12} (\cos 3\theta - 9 \cos \theta) \right]_0^\pi d\phi$$

$$= \frac{16}{12} \frac{R^5}{5} \rho \int_0^{2\pi} d\phi = 2\pi \frac{16}{12} \frac{R^5}{5} \rho = \frac{2R^5}{5} \frac{4\pi}{3} \rho$$

For a uniform sphere $\rho = \frac{M}{\frac{4}{3}\pi R^3}$

$$\Rightarrow I = \frac{2}{5} MR^2$$



II. A uniform solid cylinder of radius R and mass M rotating about the central axis. (Select a cylindrical coordinate system .)

$$I = \int_V r^2 \rho dV$$

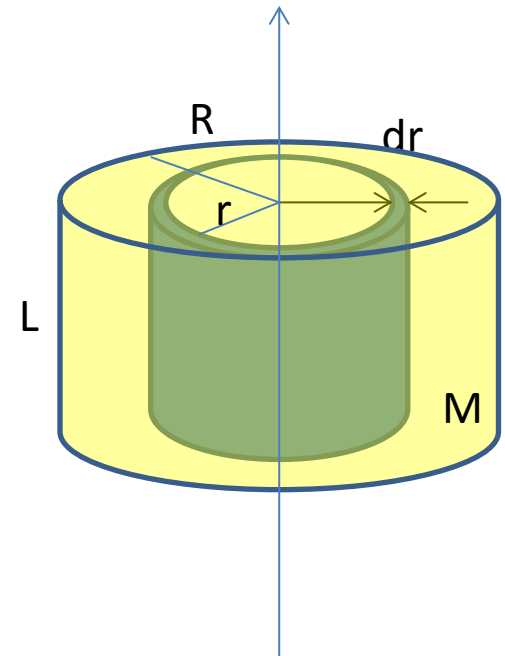
$$dV = 2\pi r L dr$$

$$\Rightarrow I = \int_0^R r^2 \rho \times 2\pi r L dr = 2\pi L \rho \int_0^R r^3 dr$$

$$= \frac{R^4}{4} 2\pi L \rho = \frac{R^4}{2} \pi L \rho$$

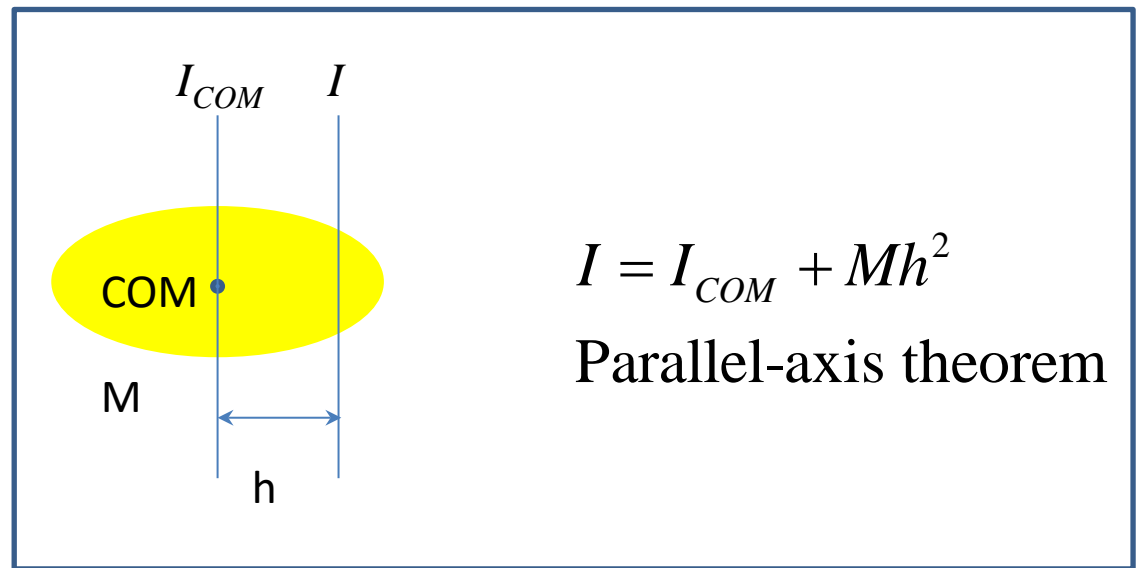
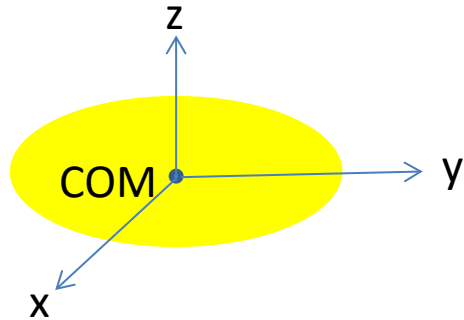
For a uniform cylinder $\rho = \frac{M}{\pi R^2 L}$

$$\Rightarrow I = \frac{1}{2} M R^2$$



Parallel-Axis Theorem

Proof: Let the origin be the center of mass.



Proof : Let the origin be the center of mass.

$$\Rightarrow \vec{r}_{COM} = \frac{1}{M} \int_V \vec{r} \rho dV = 0 \Rightarrow \int_V x \rho dV = \int_V y \rho dV = \int_V z \rho dV = 0$$

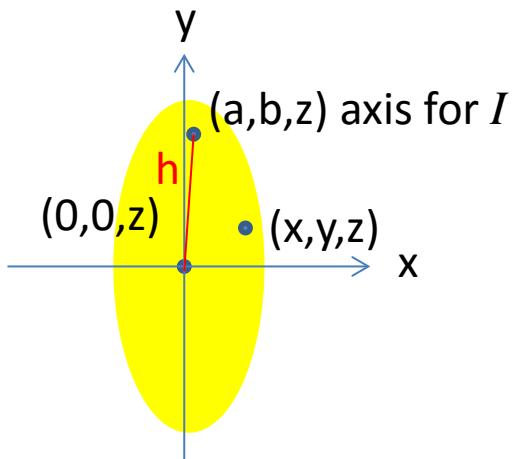
Let the z-axis be the axis for I_{COM} . $\Rightarrow I_{COM} = \int_V (x^2 + y^2) \rho dV$

$$I = \int_V [(x-a)^2 + (y-b)^2] \rho dV$$

$$= \int_V (x^2 + y^2) \rho dV - 2a \int_V x \rho dV - 2b \int_V y \rho dV + \int_V (a^2 + b^2) \rho dV$$

$$= I_{COM} + (a^2 + b^2) \int_V \rho dV \quad (\text{Note: } \sqrt{a^2 + b^2} = h ; \int_V \rho dV = M)$$

$$\Rightarrow I = I_{COM} + Mh^2$$



Note: For a system of particles,

$$\vec{r}_{COM} = \frac{1}{M} \sum_i m_i \vec{r}_i \Rightarrow \vec{v}_{COM} = \frac{d\vec{r}_{COM}}{dt} = \frac{1}{M} \sum_i m_i \frac{d\vec{r}_i}{dt} = \frac{1}{M} \sum_i m_i \vec{v}_i$$

$$\vec{v}_i = (\vec{v}_i - \vec{v}_{COM}) + \vec{v}_{COM}$$

$$K = \sum_i \frac{1}{2} m_i v_i^2 = \sum_i \frac{1}{2} m_i [(\vec{v}_i - \vec{v}_{COM}) + \vec{v}_{COM}] \cdot [(\vec{v}_i - \vec{v}_{COM}) + \vec{v}_{COM}]$$

$$= \sum_i \frac{1}{2} m_i |\vec{v}_i - \vec{v}_{COM}|^2 + \sum_i \frac{1}{2} m_i v_{COM}^2 + \sum_i \frac{1}{2} m_i \times 2(\vec{v}_i - \vec{v}_{COM}) \cdot \vec{v}_{COM}$$

$$= \sum_i \frac{1}{2} m_i |\vec{v}_i - \vec{v}_{COM}|^2 + \frac{1}{2} (\sum_i m_i) v_{COM}^2 + \sum_i m_i (\vec{v}_i - \vec{v}_{COM}) \cdot \vec{v}_{COM}$$

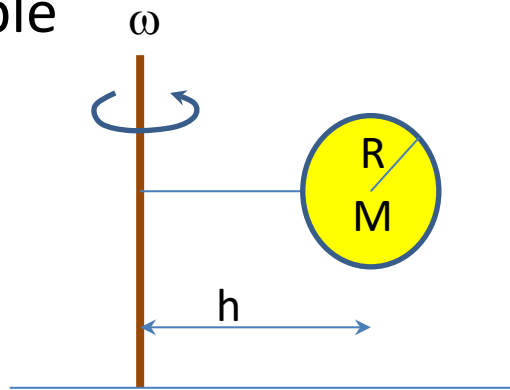
$$= \sum_i \frac{1}{2} m_i |\vec{v}_i - \vec{v}_{COM}|^2 + \frac{1}{2} M v_{COM}^2 + \left(\sum_i m_i \vec{v}_i \cdot \vec{v}_{COM} - \sum_i m_i v_{COM}^2 \right)$$

$$= \sum_i \frac{1}{2} m_i |\vec{v}_i - \vec{v}_{COM}|^2 + \frac{1}{2} M v_{COM}^2 + M \left(\frac{1}{M} \sum_i m_i \vec{v}_i \right) \cdot \vec{v}_{COM} - (\sum_i m_i) v_{COM}^2$$

$$= \sum_i \frac{1}{2} m_i |\vec{v}_i - \vec{v}_{COM}|^2 + \frac{1}{2} M v_{COM}^2 + M \vec{v}_{COM} \cdot \vec{v}_{COM} - M v_{COM}^2 = \sum_i \frac{1}{2} m_i |\vec{v}_i - \vec{v}_{COM}|^2 + \frac{1}{2} M v_{COM}^2$$

$$= \sum_i \frac{1}{2} m_i v_{rel,i}^2 + \frac{1}{2} M v_{COM}^2, \text{ where } \vec{v}_{rel,i} = \vec{v}_i - \vec{v}_{COM}$$

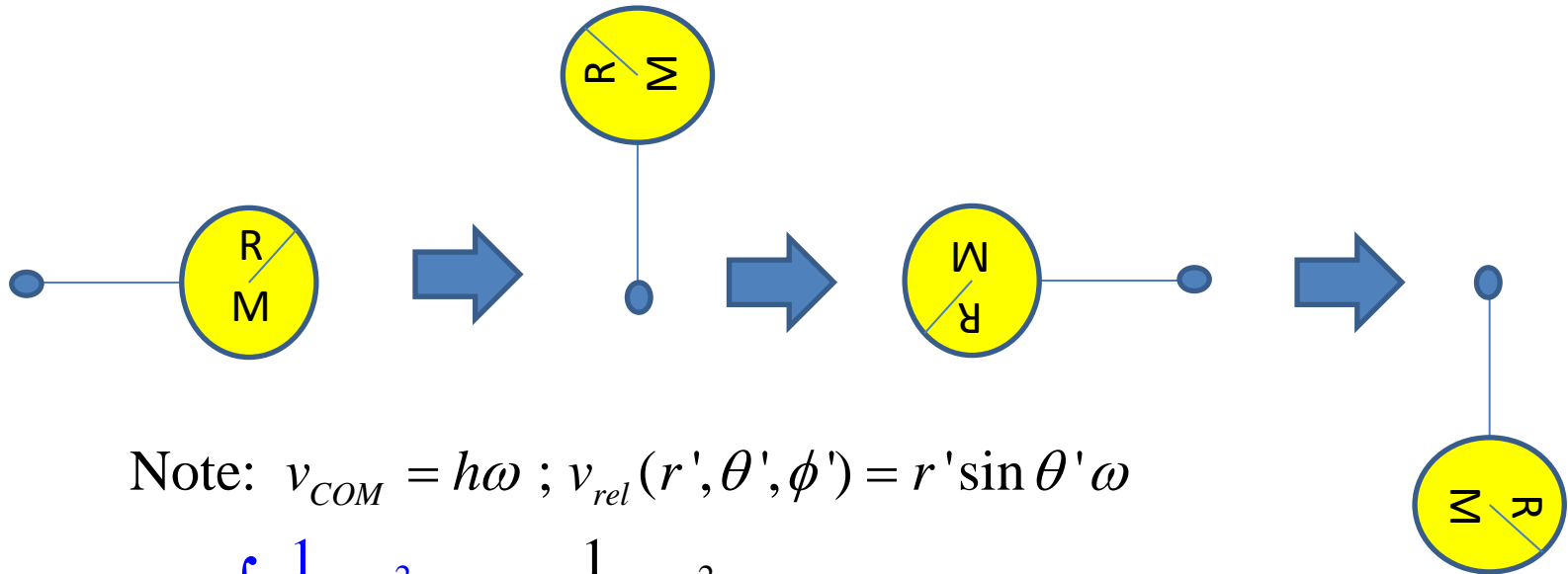
Example



$$I = I_{COM} + Mh^2$$

$$= \frac{2}{5}MR^2 + Mh^2$$

$$\Rightarrow K = \frac{1}{2}I\omega^2 = \frac{1}{5}MR^2\omega^2 + \frac{1}{2}Mh^2\omega^2$$



Note: $v_{COM} = h\omega$; $v_{rel}(r', \theta', \phi') = r' \sin \theta' \omega$

$$K = \int_V \frac{1}{2} v_{rel}^2 \rho dV + \frac{1}{2} M v_{COM}^2$$

$$= \frac{1}{2} \left(\frac{2}{5} MR^2 \right) \omega^2 + \frac{1}{2} M (h\omega)^2 = \frac{1}{5} MR^2 \omega^2 + \frac{1}{2} Mh^2 \omega^2$$

Angular Momentum and Torque

I. Definition of angular momentum for a particle: $\vec{l} = \vec{r} \times \vec{p}$

$$\frac{d\vec{l}}{dt} = \frac{d\vec{r}}{dt} \times \vec{p} + \vec{r} \times \frac{d\vec{p}}{dt} = \vec{v} \times (m\vec{v}) + \vec{r} \times \frac{d\vec{p}}{dt} = \vec{r} \times \frac{d\vec{p}}{dt}$$

Definitin of Torque: $\vec{\tau} = \vec{r} \times \vec{F}$

By Newton's 2nd Law $\vec{F}_{net} = \frac{d\vec{p}}{dt} \Rightarrow \vec{\tau}_{net} = \vec{r} \times \vec{F}_{net} = \vec{r} \times \frac{d\vec{p}}{dt}$

$$\Rightarrow \vec{\tau}_{net} = \frac{d\vec{l}}{dt}$$

II. A system of n particles

Total angular momentum for the system:

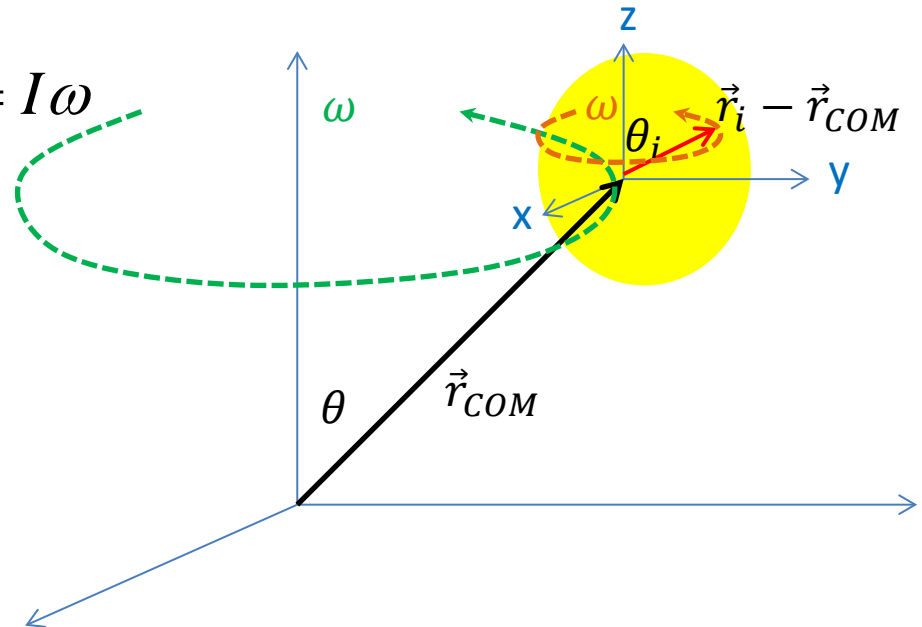
$$\begin{aligned}\vec{L} &= \sum_{i=1}^n \vec{l}_i = \sum_{i=1}^n \vec{r}_i \times \vec{p}_i = \sum_{i=1}^n \{ [\vec{r}_{COM} + (\vec{r}_i - \vec{r}_{COM})] \times m_i [\vec{v}_{COM} + (\vec{v}_i - \vec{v}_{COM})] \} \\&= \sum_{i=1}^n (\vec{r}_{COM} \times m_i \vec{v}_{COM}) + \sum_{i=1}^n [\vec{r}_{COM} \times m_i (\vec{v}_i - \vec{v}_{COM})] \\&\quad + \sum_{i=1}^n [(\vec{r}_i - \vec{r}_{COM}) \times m_i \vec{v}_{COM}] + \sum_{i=1}^n [(\vec{r}_i - \vec{r}_{COM})] \times m_i [(\vec{v}_i - \vec{v}_{COM})] \\&= \vec{r}_{COM} \times \left(\sum_{i=1}^n m_i \right) \vec{v}_{COM} + \vec{r}_{COM} \times \left(\sum_{i=1}^n m_i \vec{v}_i \right) - \sum_{i=1}^n [\vec{r}_{COM} \times m_i \vec{v}_{COM}] \\&\quad + \sum_{i=1}^n (\vec{r}_i \times m_i \vec{v}_{COM}) - \sum_{i=1}^n (\vec{r}_{COM} \times m_i \vec{v}_{COM}) + \sum_{i=1}^n [(\vec{r}_i - \vec{r}_{COM})] \times m_i [(\vec{v}_i - \vec{v}_{COM})] \\&= \vec{r}_{COM} \times M \vec{v}_{COM} + \vec{r}_{COM} \times M \left(\frac{1}{M} \sum_{i=1}^n m_i \vec{v}_i \right) - \vec{r}_{COM} \times \left(\sum_{i=1}^n m_i \right) \vec{v}_{COM} \\&\quad + M \left(\frac{1}{M} \sum_{i=1}^n m_i \vec{r}_i \right) \times \vec{v}_{COM} - \vec{r}_{COM} \times \left(\sum_{i=1}^n m_i \right) \vec{v}_{COM} + \sum_{i=1}^n [(\vec{r}_i - \vec{r}_{COM})] \times m_i [(\vec{v}_i - \vec{v}_{COM})] \\&= \vec{r}_{COM} \times M \vec{v}_{COM} + \vec{r}_{COM} \times M \vec{v}_{COM} - \vec{r}_{COM} \times M \vec{v}_{COM} \\&\quad + M \vec{r}_{COM} \times \vec{v}_{COM} - M \vec{r}_{COM} \times \vec{v}_{COM} + \sum_{i=1}^n [(\vec{r}_i - \vec{r}_{COM})] \times m_i [(\vec{v}_i - \vec{v}_{COM})] \\&= \vec{r}_{COM} \times M \vec{v}_{COM} + \sum_{i=1}^n [(\vec{r}_i - \vec{r}_{COM})] \times m_i [(\vec{v}_i - \vec{v}_{COM})]\end{aligned}$$

$$\vec{L} = \sum_{i=1}^n \vec{r}_i \times \vec{p}_i = \vec{r}_{COM} \times M\vec{v}_{COM} + \sum_{i=1}^n [(\vec{r}_i - \vec{r}_{COM})] \times m_i [(\vec{v}_i - \vec{v}_{COM})]$$

For a rigid body rotating about a fixed axis (e.g. the z-axis)

$$\begin{aligned} L_z &= r_{COM} \sin \theta \times M v_{COM} + \sum_{i=1}^n (|\vec{r}_i - \vec{r}_{COM}| \sin \theta_i \times m_i |\vec{v}_i - \vec{v}_{COM}|) \\ &= r_{COM} \sin \theta \times M r_{COM} \sin \theta \cdot \omega + \sum_{i=1}^n (|\vec{r}_i - \vec{r}_{COM}| \sin \theta_i \times m_i |\vec{r}_i - \vec{r}_{COM}| \sin \theta_i \cdot \omega) \\ &= M (r_{COM} \sin \theta)^2 \omega + \left[\sum_{i=1}^n m_i (|\vec{r}_i - \vec{r}_{COM}| \sin \theta_i)^2 \right] \omega \\ &= M h^2 \omega + I_{COM} \omega = (I_{COM} + M h^2) \omega = I \omega \end{aligned}$$

$$\Rightarrow \frac{dL_z}{dt} = I \frac{d\omega}{dt} = I \alpha$$



$$\begin{aligned}\frac{d\vec{L}}{dt} &= \frac{d}{dt} \left(\sum_{i=1}^n \vec{r}_i \times \vec{p}_i \right) = \sum_{i=1}^n \left(\frac{d\vec{r}_i}{dt} \times \vec{p}_i + \vec{r}_i \times \frac{d\vec{p}_i}{dt} \right) = \sum_{i=1}^n (\vec{v}_i \times m\vec{v}_i + \vec{r}_i \times \vec{F}_{net,i}) \\ &= \sum_{i=1}^n \vec{r}_i \times \vec{F}_{net,i} = \sum_{i=1}^n \vec{r}_i \times (\vec{F}_{ext,i} + \sum_{j \neq i} \vec{F}_{ji}) = \sum_{i=1}^n \vec{r}_i \times \vec{F}_{ext,i} + \sum_{i=1}^n \sum_{j \neq i} \vec{r}_i \times \vec{F}_{ji}\end{aligned}$$

Note: $|\vec{r}_i \times \vec{F}_{ji}| = r_i F_{ji} \sin \Theta_i = (r_i \sin \Theta_i) F_{ji} = r_0 F_{ji}$

$|\vec{r}_j \times \vec{F}_{ij}| = r_j F_{ij} \sin \Theta_j = (r_j \sin \Theta_j) F_{ij} = r_0 F_{ij}$

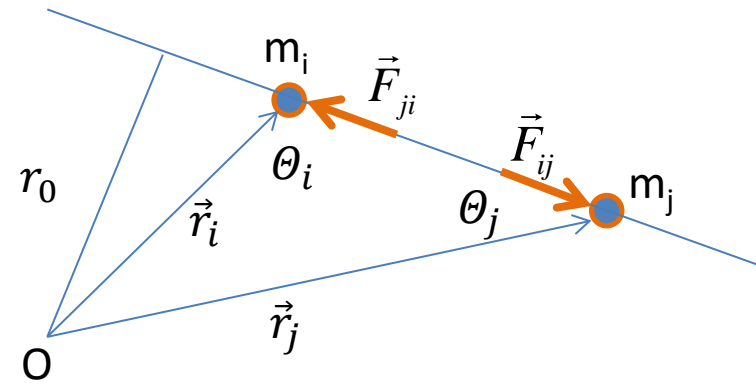
$\vec{F}_{ji} = -\vec{F}_{ij}$ (Newton's 3rd Law)

$$\Rightarrow \vec{r}_i \times \vec{F}_{ji} = -\vec{r}_j \times \vec{F}_{ij} \Rightarrow \sum_{i=1}^n \sum_{j \neq i} \vec{r}_i \times \vec{F}_{ji} = 0$$

$$\Rightarrow \frac{d\vec{L}}{dt} = \sum_{i=1}^n \vec{r}_i \times \vec{F}_{ext,i} = \sum_{i=1}^n \vec{\tau}_{ext,i} = \vec{\tau}_{net} \text{ (net external torque)}$$

$$\Rightarrow \frac{dL_z}{dt} = \sum_{i=1}^n r_i \sin \theta_i F_{ext,i,t} = \tau_z$$

($F_{ext,i,t}$: tangential component of the net external force on the i th particle)



For a rigid body rotating about the z-axis (a one dimensional motion), only the z-component of \vec{L} and the z-component of $\vec{\tau}$, that comes from the tangential component of external force $\vec{F}_{ext,i}$, are of interest.

$$L_z = I\omega \Rightarrow \frac{dL_z}{dt} = I\alpha$$

$$\frac{dL_z}{dt} = \sum_{i=1}^n r_i \sin \theta_i F_{ext,i,t} = \tau_z$$

For such one dimensional motions, we drop the subscript z:

$$L = I\omega$$

$$\tau_{net} = I\alpha \quad (\text{Newton's 2nd Law for rotation})$$

Analogy

$$p = mv$$

$$F_{net} = ma$$

Note: $\sum_{i=1}^n r_i \sin \theta_i F_{ext,i,t} = \tau \Rightarrow dW = \sum_{i=1}^n dW_i = \sum_{i=1}^n F_{ext,i,t} ds_i$

$$= \sum_{i=1}^n F_{ext,i,t} [(r_i \sin \theta_i) d\theta] = \left[\sum_{i=1}^n r_i \sin \theta_i F_{ext,i,t} \right] d\theta = \tau d\theta \Rightarrow W_{12} = \int_{\theta_1}^{\theta_2} \tau d\theta$$

Translation in 1-D

position x

velocity $v = \frac{dx}{dt}$

acceleration $a = \frac{dv}{dt}$

mass m

Newton's 2nd Law $F_{net} = ma$

work $W = \int_{x_i}^{x_f} F dx$

kinetic energy $K = \frac{1}{2}mv^2$

Power $P = \frac{dW}{dt} = Fv$

work-kinetic energy theorem

$$\Delta K = \frac{1}{2}mv_f^2 - \frac{1}{2}mv_i^2 = W_{net}$$



Rotation with a fixed axis

angular position θ

angular velocity $\omega = \frac{d\theta}{dt}$

angular acceleration $\alpha = \frac{d\omega}{dt}$

rotational inertia I

Newton's 2nd Law $\tau_{net} = I\alpha$

work $W = \int_{\theta_i}^{\theta_f} \tau d\theta$

kinetic energy $K = \frac{1}{2}I\omega^2$

Power $P = \frac{dW}{dt} = \tau\omega$

work-kinetic energy theorem

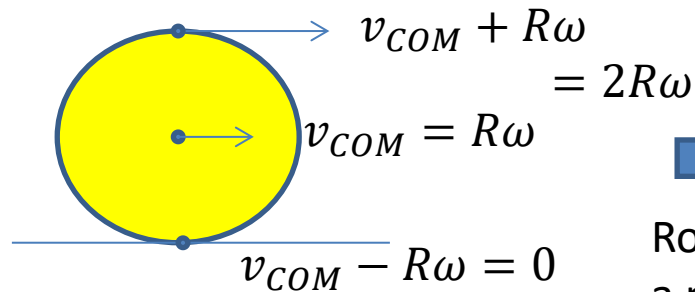
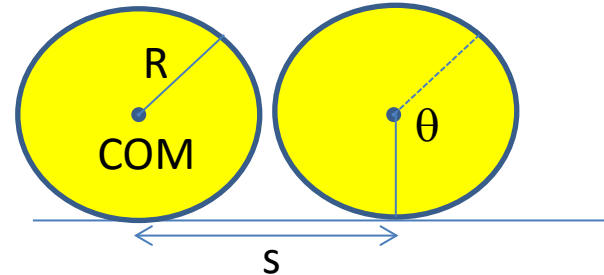
$$\Delta K = \frac{1}{2}I\omega_f^2 - \frac{1}{2}I\omega_i^2 = W_{net}$$

Chapter 11 Rolling Torque and Angular Momentum

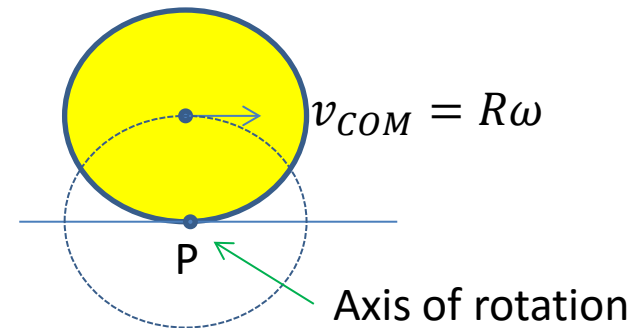
Smooth rolling (no slipping or bouncing on the surface)

$$s = R\theta$$

$$v_{COM} = \frac{ds}{dt} = R \frac{d\theta}{dt} = R\omega$$



Rolling viewed as
a pure rotation



$$I_P = I_{COM} + MR^2 \quad (\text{parallel axis theorem})$$

$$K = \frac{1}{2} I_P \omega_P^2 \quad (v_{COM} = R\omega = R\omega_P \Rightarrow \omega_P = \omega)$$

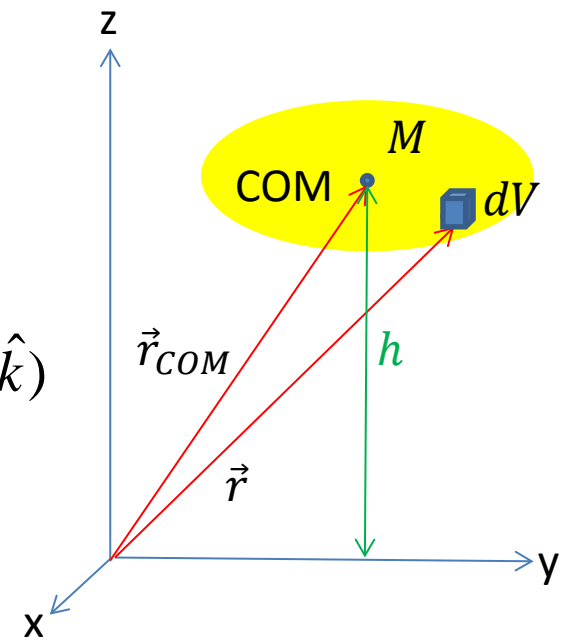
$$\Rightarrow K = \frac{1}{2} (I_{COM} + MR^2) \omega^2 = \frac{1}{2} I_{COM} \omega^2 + \frac{1}{2} MR^2 \omega^2 = \frac{1}{2} I_{COM} \omega^2 + \frac{1}{2} M v_{COM}^2$$

Center of Mass \rightarrow Center of Gravity

I. Torque of dV about the origin: $d\vec{\tau} = \vec{r} \times \rho dV (-g\hat{k})$

\Rightarrow The total torque of the system about the origin:

$$\begin{aligned}\vec{\tau}_{net} &= \int_V d\vec{\tau} = \left[\int_V \vec{r} \rho dV \right] \times (-g\hat{k}) = \left[\frac{1}{M} \int_V \vec{r} \rho dV \right] \times (-Mg\hat{k}) \\ &= \vec{r}_{COM} \times (-Mg\hat{k})\end{aligned}$$



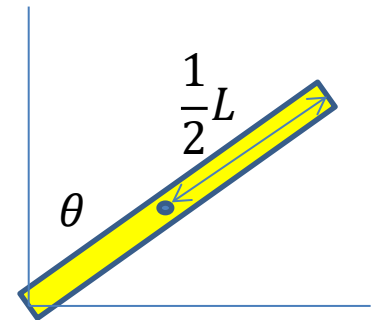
II. Gravitational Potential

$$U = \int_V dU = \int_V z g \rho dV = g \int_V z \rho dV = Mg \left[\frac{1}{M} \int_V z \rho dV \right] = Mg z_{COM} = Mgh$$

Example

The total torque of the system about the origin: $\frac{1}{2} LMg \sin \theta$

The total gravitational potential: $Mg \left(\frac{1}{2} L \cos \theta \right) = \frac{1}{2} LMg \cos \theta$

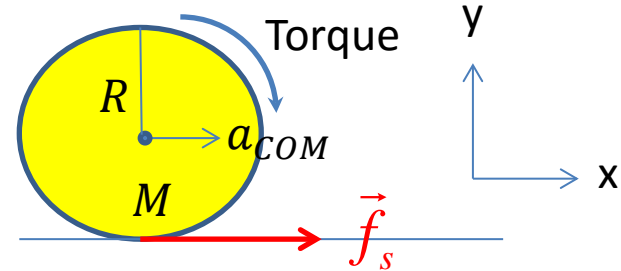


The Force of Rolling

1) Apply a torque to a round object.

e.g. wheels of a car

$$a_{COM} = R\alpha$$



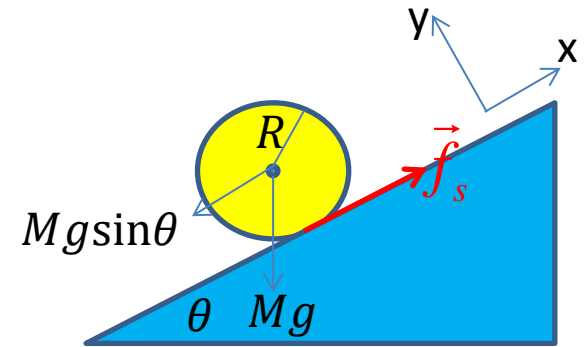
2) Rolling down a ramp

$$f_s - Mg \sin \theta = Ma_{COM,x}$$

$$Rf_s = I_{COM} \alpha$$

$$a_{COM,x} = -R\alpha$$

$$\Rightarrow a_{COM,x} = -\frac{g \sin \theta}{1 + \frac{I_{COM}}{MR^2}}$$

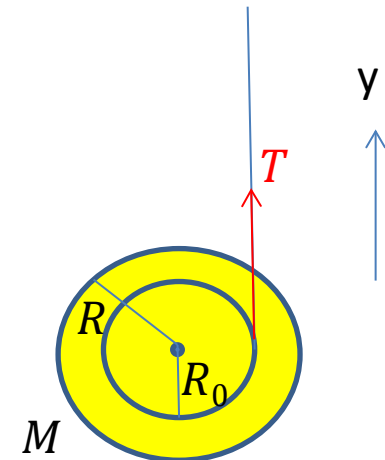


3) Yo-Yo

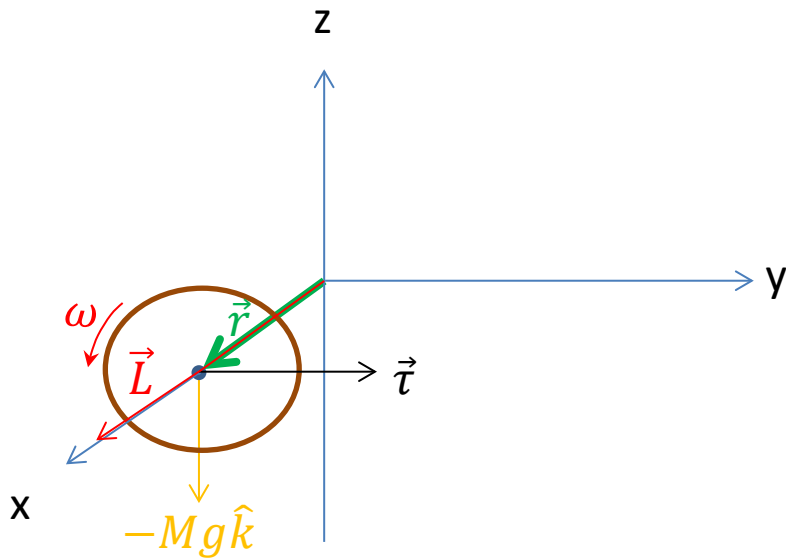
$$T - Mg = Ma_{COM,x}$$

$$R_0 T = I_{COM} \alpha \Rightarrow a_{COM,x} = -\frac{g}{1 + \frac{I_{COM}}{MR_0^2}}$$

$$a_{COM,x} = -R_0 \alpha$$



Precession of a Gyroscope



$$\vec{\tau} = \frac{d\vec{L}}{dt}$$

$$\vec{\tau} = \vec{r} \times (-Mg\hat{k}) = Mgr\hat{j}$$

$$\tau = Mgr ,$$

also $L = I\omega$ (about the axis of rotation)

For a rapid spinning gyroscope,

the magnitude of \vec{L} is fixed.

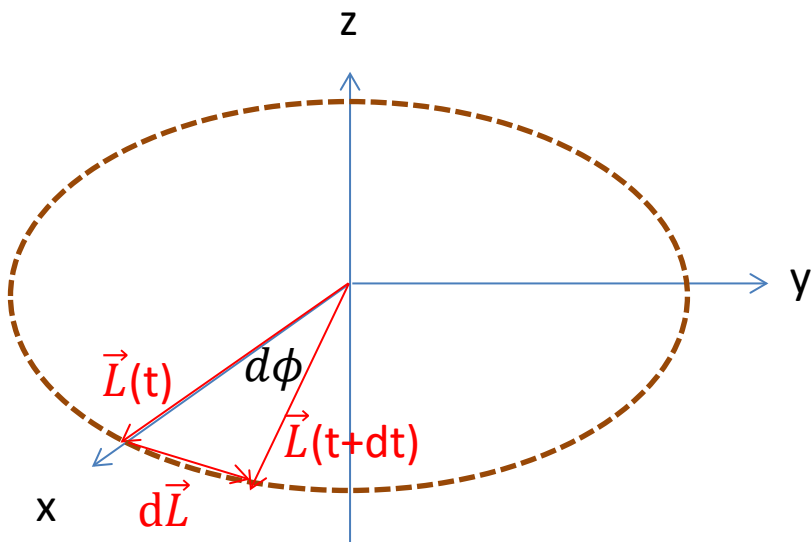
$$L(t) = L(t + dt)$$

$$dL = Ld\phi \Rightarrow d\phi = \frac{dL}{L}$$

The precession rate Ω :

$$\Omega = \frac{d\phi}{dt} = \frac{1}{L} \frac{dL}{dt} = \frac{1}{L} \frac{\tau dt}{dt}$$

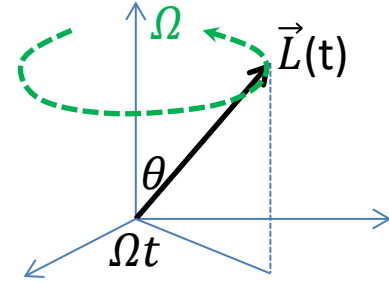
$$= \frac{\tau}{L} = \frac{Mgr}{L} = \frac{Mgr}{I\omega}$$



Advanced derivation:

$$\frac{d\vec{L}}{dt} = \vec{r} \times M\vec{g}, \text{ Let } L = \sqrt{L_x^2 + L_y^2 + L_z^2} \text{ be a constant and } \vec{r} = \gamma\vec{L}$$

$$\Rightarrow \frac{d\vec{L}}{dt} = \gamma M \vec{L} \times \vec{g} = \gamma M \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ L_x & L_y & L_z \\ 0 & 0 & -g \end{vmatrix} = -\gamma M g (L_y \hat{i} - L_x \hat{j})$$



$$\Rightarrow \begin{cases} \frac{dL_x}{dt} = -\gamma M g L_y \\ \frac{dL_y}{dt} = \gamma M g L_x \\ \frac{dL_z}{dt} = 0 \end{cases} \Rightarrow \begin{cases} \frac{d^2 L_x}{dt^2} = -\gamma M g \frac{dL_y}{dt} \\ \frac{d^2 L_y}{dt^2} = \gamma M g \frac{dL_x}{dt} \\ L_z(t) = L_z(0) \end{cases} \Rightarrow \begin{cases} \frac{d^2 L_x}{dt^2} = -\gamma^2 M^2 g^2 L_x \\ \frac{d^2 L_y}{dt^2} = -\gamma^2 M^2 g^2 L_y \\ L_z(t) = L_z(0) \end{cases}$$

$$\text{Let } L_x(0) = L \sin \theta, L_y(0) = 0, L_z(0) = L \cos \theta \Rightarrow L_z(t) = L \cos \theta$$

$$\frac{d^2 L_y}{dt^2} + \gamma^2 M^2 g^2 L_y = 0$$

(a 2nd-order homogeneous differentiation equation)

To find two independent solutions for the basis, try $L_y(t) = e^{\alpha t}$

$$\Rightarrow \alpha^2 + \gamma^2 M^2 g^2 = 0 \Rightarrow \alpha = \pm i \gamma M g$$

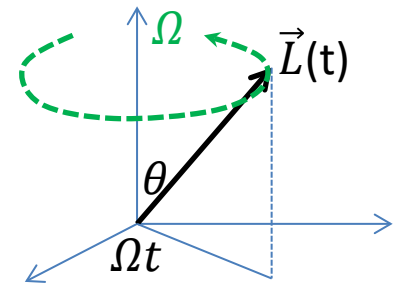
$$L_y(t) = A e^{i \gamma M g t} + B e^{-i \gamma M g t}$$

$$L_y(0) = 0 \Rightarrow B = -A \Rightarrow L_y(t) = A(e^{i \gamma M g t} - e^{-i \gamma M g t}) = 2A i \sin(\gamma M g t)$$

$$\text{recall } \frac{dL_y}{dt} = \gamma M g L_x \Rightarrow L_x(t) = \frac{1}{\gamma M g} \frac{dL_y}{dt} = 2A i \cos(\gamma M g t)$$

$$L_x(0) = L \sin \theta \Rightarrow A = \frac{L \sin \theta}{2i}, \text{ Let } \Omega = \gamma M g$$

$$\Rightarrow \begin{cases} L_x(t) = [L \sin \theta] \cos(\Omega t) \\ L_y(t) = [L \sin \theta] \sin(\Omega t) \\ L_z(t) = L \cos \theta \end{cases} \quad \text{Note: } r = \gamma L \Rightarrow \Omega = \gamma M g = \frac{M g r}{L} = \frac{M g r}{I \omega}$$



Chapter 12 Equilibrium and Elasticity

For an object in

1) equilibrium $\Rightarrow \vec{P} = \text{a constant}$ and $\vec{L} = \text{a constant}$,

2) static equilibrium $\Rightarrow \vec{P} = 0$ and $\vec{L} = 0$.

\Rightarrow Requirements of Equilibrium:

$$\vec{F}_{net} = \sum_i \vec{F}_{ext,i} = \frac{d\vec{P}}{dt} = 0$$

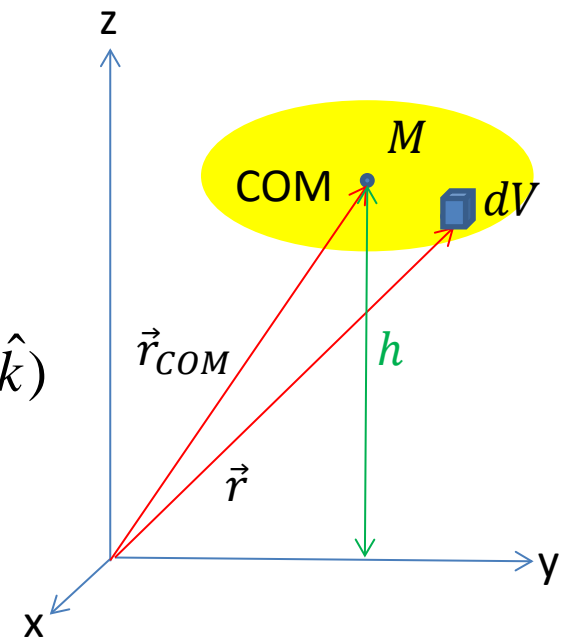
$$\vec{\tau}_{net} = \sum_i \vec{\tau}_{ext,i} = \frac{d\vec{L}}{dt} = 0 \text{ (about any possible point)}$$

Center of Mass \rightarrow Center of Gravity

I. Torque of dV about the origin: $d\vec{\tau} = \vec{r} \times \rho dV (-g\hat{k})$

\Rightarrow The total torque of the system about the origin:

$$\begin{aligned}\vec{\tau}_{net} &= \int_V d\vec{\tau} = \left[\int_V \vec{r} \rho dV \right] \times (-g\hat{k}) = \left[\frac{1}{M} \int_V \vec{r} \rho dV \right] \times (-Mg\hat{k}) \\ &= \vec{r}_{COM} \times (-Mg\hat{k})\end{aligned}$$



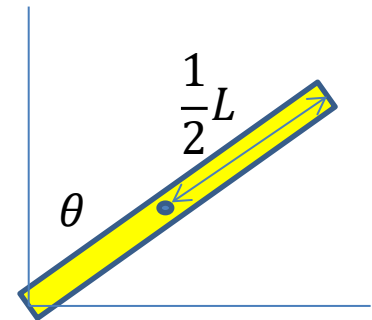
II. Gravitational Potential

$$U = \int_V dU = \int_V z g \rho dV = g \int_V z \rho dV = Mg \left[\frac{1}{M} \int_V z \rho dV \right] = Mg z_{COM} = Mgh$$

Example

The total torque of the system about the origin: $\frac{1}{2} LMg \sin \theta$

The total gravitational potential: $Mg \left(\frac{1}{2} L \cos \theta \right) = \frac{1}{2} LMg \cos \theta$



EXAMPLE

$$\text{i) } \sum_i \vec{F}_{ext,i} = 0$$

The gravitational force *effectively* acts at the center of mass.

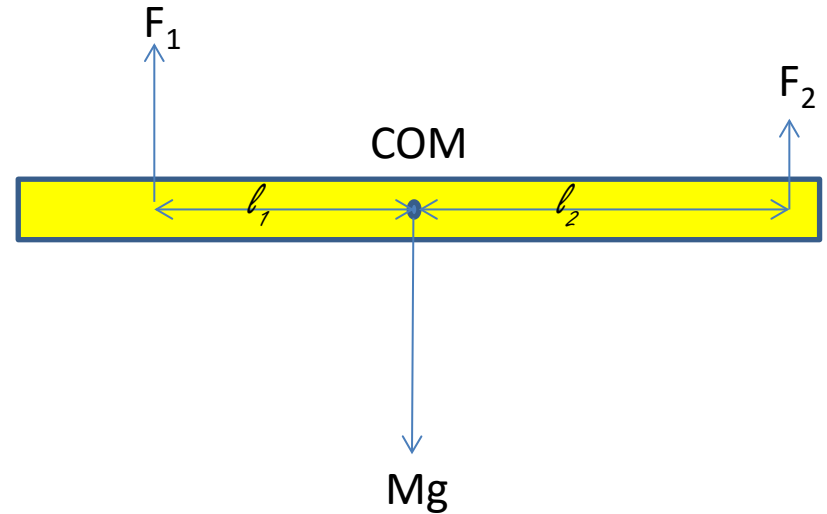
$$F_1 + F_2 - Mg = 0$$

$$\text{ii) } \sum_i \vec{\tau}_{ext,i} = 0$$

To simplify the calculation, select a point, that one of the forces acts at, to be the point about which we calculate the torques.

$$Mgl_1 - F_2(l_1 + l_2) = 0$$

$$F_1 = \frac{Mgl_2}{l_1 + l_2}; \quad F_2 = \frac{Mgl_1}{l_1 + l_2}$$



Elasticity

Rigid body (ideal) → Elastic (reality)

Tensile/Compressive Stress and Strain

$$\text{Stress} = \frac{F}{A}; \text{Strain} = \frac{\Delta L}{L}$$

$$\frac{F}{A} = E \frac{\Delta L}{L}; E : \text{Young's modulus}$$

Shearing Stress and Strain

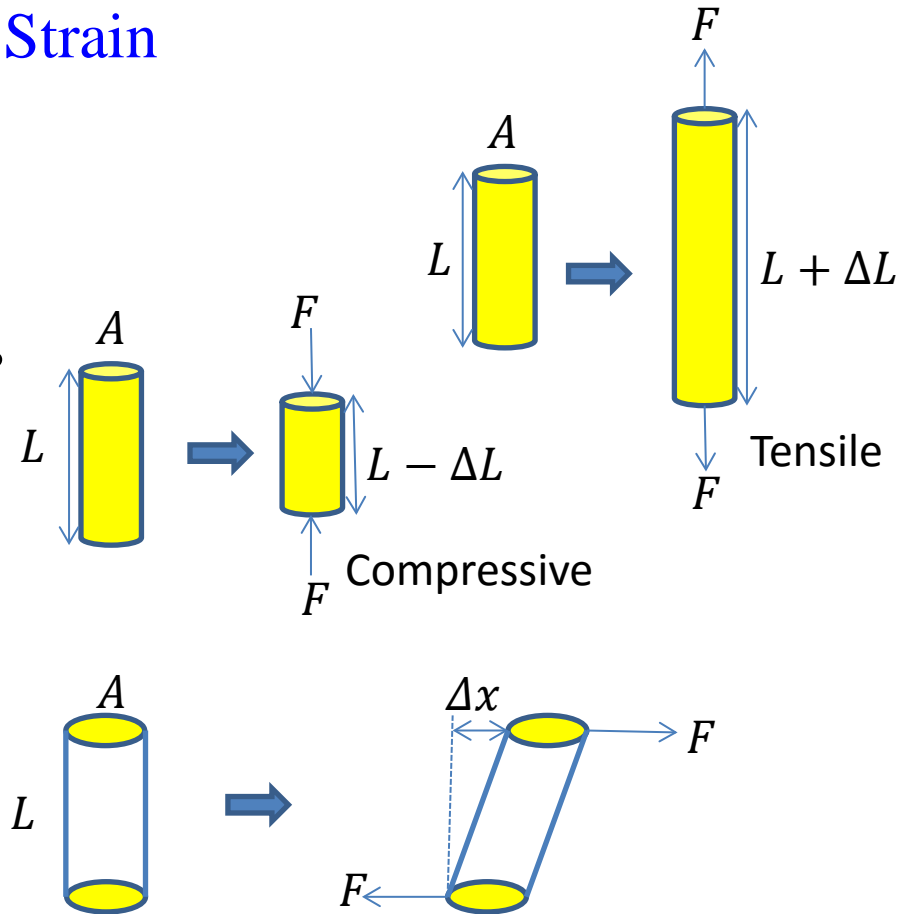
$$\text{Stress} = \frac{F}{A}; \text{Strain} = \frac{\Delta x}{L}$$

$$\frac{F}{A} = G \frac{\Delta x}{L}; G : \text{Shear modulus}$$

Hydraulic pressure (stress)

$$P = B \frac{\Delta V}{V}; P : \text{pressure}; V : \text{volume}; B : \text{Bulk modulus}$$

$$\text{Stress} = \text{Modulus} \times \text{Strain}$$



Stress

Consider a volume element $dV = dxdydz$

$$\text{Stress} = \frac{dF}{dA}; \text{ where } dA = dxdy, dydz, \text{ or } dzdx$$

Notation: X_y is the force per unit area applied in the x-direction to a plane with normal in the y-direction.

\Rightarrow tensile/compressive stresses X_x, Y_y, Z_z

shearing stresses $X_y, X_z, Y_x, Y_z, Z_x, Z_y$

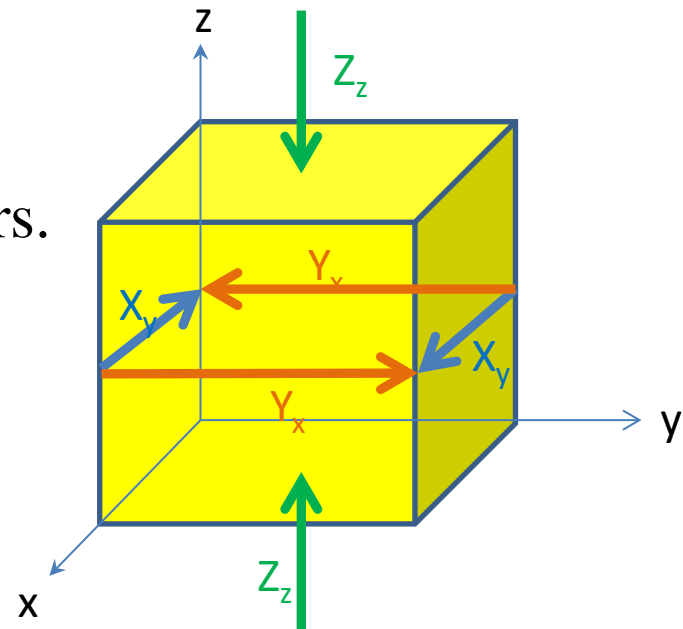
Static Equilibrium

$\vec{F}_{net} = 0 \Rightarrow$ Stress components appear in pairs.

$\vec{\tau}_{net} = 0 \Rightarrow X_y = Y_x; Z_x = X_z; Y_z = Z_y$

Six independent stress components:

$X_x, Y_y, Z_z, Y_z, Z_x, X_y$



Strain

Deformation:

$$\vec{r} = x\hat{i} + y\hat{j} + z\hat{k} \rightarrow \vec{r}' = [x + \mu(\vec{r})]\hat{i} + [y + \nu(\vec{r})]\hat{j} + [z + \omega(\vec{r})]\hat{k}$$

Consider two points $P_1 = x\hat{i} + y\hat{j} + z\hat{k}$

$$P_2 = (x + dx)\hat{i} + (y + dy)\hat{j} + (z + dz)\hat{k}$$

The vector from P_1 to P_2 is $dx\hat{i} + dy\hat{j} + dz\hat{k}$

After deformation

$$P_1 = [x + \mu(x, y, z)]\hat{i} + [y + \nu(x, y, z)]\hat{j} + [z + \omega(x, y, z)]\hat{k}$$

$$\begin{aligned} P_2 = & [x + dx + \mu(x + dx, y + dy, z + dz)]\hat{i} \\ & + [y + dy + \nu(x + dx, y + dy, z + dz)]\hat{j} \\ & + [z + dz + \omega(x + dx, y + dy, z + dz)]\hat{k} \end{aligned}$$

The vector from P_1 to P_2 becomes

$$\begin{aligned} & [dx + \mu(x + dx, y + dy, z + dz) - \mu(x, y, z)]\hat{i} \\ & + [dy + \nu(x + dx, y + dy, z + dz) - \nu(x, y, z)]\hat{j} \\ & + [dz + \omega(x + dx, y + dy, z + dz) - \omega(x, y, z)]\hat{k} \end{aligned}$$

$$\begin{aligned}
&= [dx + d\mu(x, y, z)]\hat{i} + [dy + d\nu(x, y, z)]\hat{j} + [dz + d\omega(x, y, z)]\hat{k} \\
&= [dx + \frac{\partial\mu}{\partial x}dx + \frac{\partial\mu}{\partial y}dy + \frac{\partial\mu}{\partial z}dz]\hat{i} + [dy + \frac{\partial\nu}{\partial x}dx + \frac{\partial\nu}{\partial y}dy + \frac{\partial\nu}{\partial z}dz]\hat{j} \\
&\quad + [dz + \frac{\partial\omega}{\partial x}dx + \frac{\partial\omega}{\partial y}dy + \frac{\partial\omega}{\partial z}dz]\hat{k}
\end{aligned}$$

The change of vector P_1P_2 due to deformation is

$$\begin{aligned}
&[\frac{\partial\mu}{\partial x}dx + \frac{\partial\mu}{\partial y}dy + \frac{\partial\mu}{\partial z}dz]\hat{i} + [\frac{\partial\nu}{\partial x}dx + \frac{\partial\nu}{\partial y}dy + \frac{\partial\nu}{\partial z}dz]\hat{j} \\
&+ [\frac{\partial\omega}{\partial x}dx + \frac{\partial\omega}{\partial y}dy + \frac{\partial\omega}{\partial z}dz]\hat{k}
\end{aligned}$$

Compared to the original vector P_1P_2 : $dx\hat{i} + dy\hat{j} + dz\hat{k}$



The tensile/compressive strain in the x -direction: $e_{xx} = \frac{\frac{\partial \mu}{\partial x} dx}{dx} = \frac{\partial \mu}{\partial x}$

The tensile/compressive strain in the y -direction: $e_{yy} = \frac{\frac{\partial \nu}{\partial y} dy}{dy} = \frac{\partial \nu}{\partial y}$

The tensile/compressive strain in the z -direction: $e_{zz} = \frac{\frac{\partial \omega}{\partial z} dz}{dz} = \frac{\partial \omega}{\partial z}$

The total shearing strain in the y - z plane: $e_{yz} = \frac{\frac{\partial \nu}{\partial z} dz}{dz} + \frac{\frac{\partial \omega}{\partial y} dy}{dy} = \frac{\partial \nu}{\partial z} + \frac{\partial \omega}{\partial y}$

The total shearing strain in the z - x plane: $e_{zx} = \frac{\frac{\partial \omega}{\partial x} dx}{dx} + \frac{\frac{\partial \mu}{\partial z} dz}{dz} = \frac{\partial \omega}{\partial x} + \frac{\partial \mu}{\partial z}$

The total shearing strain in the x - y plane: $e_{xy} = \frac{\frac{\partial \mu}{\partial y} dy}{dy} + \frac{\frac{\partial \nu}{\partial x} dx}{dx} = \frac{\partial \mu}{\partial y} + \frac{\partial \nu}{\partial x}$

Six strain components

$$e_{xx} = \frac{\partial \mu}{\partial x}; \quad e_{yy} = \frac{\partial \nu}{\partial y}; \quad e_{zz} = \frac{\partial \omega}{\partial z}$$

$$e_{yz} = \frac{\partial \nu}{\partial z} + \frac{\partial \omega}{\partial y}; \quad e_{zx} = \frac{\partial \omega}{\partial x} + \frac{\partial \mu}{\partial z}; \quad e_{xy} = \frac{\partial \mu}{\partial y} + \frac{\partial \nu}{\partial x}$$

Hooke's Law

$$\begin{bmatrix} X_x \\ Y_y \\ Z_z \\ Y_z \\ Z_x \\ X_y \end{bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} & C_{14} & C_{15} & C_{16} \\ C_{21} & C_{22} & C_{23} & C_{24} & C_{25} & C_{26} \\ C_{31} & C_{32} & C_{33} & C_{34} & C_{35} & C_{36} \\ C_{41} & C_{42} & C_{43} & C_{44} & C_{45} & C_{46} \\ C_{51} & C_{52} & C_{53} & C_{54} & C_{55} & C_{56} \\ C_{61} & C_{62} & C_{63} & C_{64} & C_{65} & C_{66} \end{bmatrix} \begin{bmatrix} e_{xx} \\ e_{yy} \\ e_{zz} \\ e_{yz} \\ e_{zx} \\ e_{xy} \end{bmatrix} \quad \text{Symmetrical Matrix } C_{\alpha\beta} = C_{\beta\alpha}$$

Stresses = Moduli \times strains

Chapter 13 Gravitation

Newton's law of gravitation

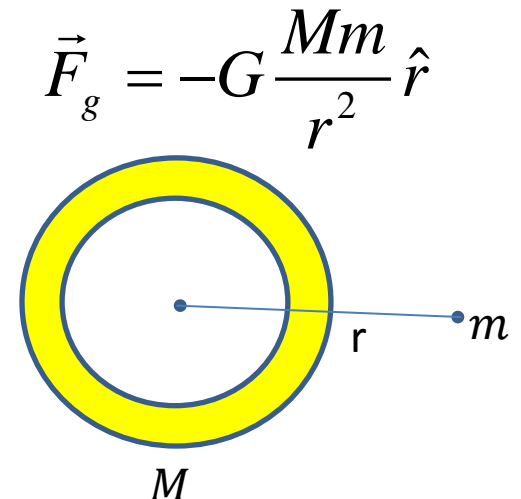
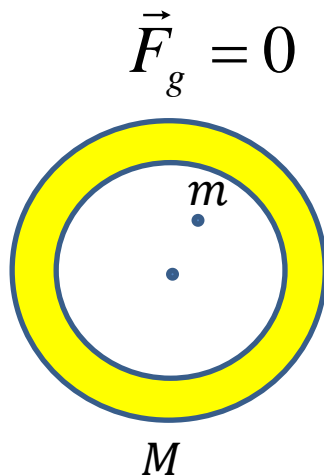
$$F = G \frac{m_1 m_2}{r^2} \quad G = 6.67 \times 10^{-11} \text{ m}^3 / \text{kg} \cdot \text{s}^2$$

Principle of superposition:

Individual gravitational forces are not altered by each other.

$$\vec{F}_{net} = \sum_{i=1}^n G \frac{m_0 m_i}{r_i^2} \hat{r}_i$$

Shell theorem



Proof:

1) At an external point

Let the z-axis be in the direction of \vec{r} .

$$dV' = r'^2 \sin \theta' dr' d\theta' d\phi'$$

$$d\vec{F}_g = -G \frac{m \rho dV'}{|\vec{r} - \vec{r}'|^2} \frac{\vec{r} - \vec{r}'}{|\vec{r} - \vec{r}'|} \quad (\text{Newton's gravitational force law})$$

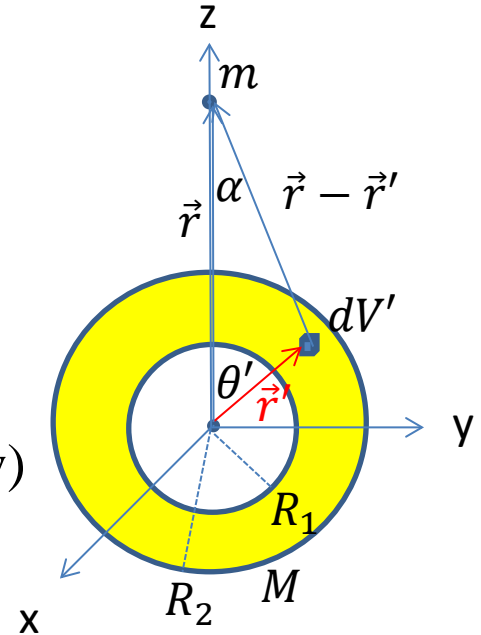
By symmetry, $\vec{F}_g = F_{g,z} \hat{k}$ and therefore

only the z-component of $d\vec{F}_g$ is of interest.

$$dF_{g,z} = d\vec{F}_g \cdot \hat{k} = -G \frac{m \rho dV'}{|\vec{r} - \vec{r}'|^2} \left(\frac{\vec{r} - \vec{r}'}{|\vec{r} - \vec{r}'|} \cdot \hat{k} \right) = -G \frac{m \rho dV'}{|\vec{r} - \vec{r}'|^2} \cos \alpha$$

Let $|\vec{r} - \vec{r}'| = s$

By cosine law, we have $\cos \alpha = \frac{s^2 + r^2 - r'^2}{2sr}$; $\cos \theta' = \frac{r'^2 + r^2 - s^2}{2r'r}$



$$\vec{F}_g = F_{g,z} \hat{k} = \hat{k} \int dF_{g,z} = -Gm\rho\hat{k} \int_V \frac{\cos \alpha}{|\vec{r} - \vec{r}'|^2} dV'$$

$$= -Gm\rho\hat{k} \int_{R_1}^{R_2} \int_0^\pi \int_0^{2\pi} \frac{r'^2 \cos \alpha \sin \theta'}{|\vec{r} - \vec{r}'|^2} d\phi' d\theta' dr'$$

$$= -Gm\rho\hat{k} \int_{R_1}^{R_2} \int_0^{2\pi} \left[\int_0^\pi \frac{r'^2}{s^2} \frac{s^2 + r^2 - r'^2}{2sr} (\sin \theta' d\theta') \right] d\phi' dr'$$

Noting that $\cos \theta' = \frac{r'^2 + r^2 - s^2}{2r'r} \Rightarrow d(\cos \theta') = d\left(\frac{r'^2 + r^2 - s^2}{2r'r}\right)$

$$\Rightarrow \sin \theta' d\theta' = \frac{sds}{r'r} \Leftarrow r' \text{ is considered constant}$$

when calculating the integral in the bracket.

s is a function of θ' .

$$\Rightarrow \vec{F}_g = -Gm\rho\hat{k} \int_{R_1}^{R_2} \int_0^{2\pi} \left[\int_{r-r'}^{r+r'} \frac{r'^2}{s^2} \frac{s^2 + r^2 - r'^2}{2sr} \frac{sds}{r'r} \right] d\phi' dr'$$

$$\begin{aligned}
\vec{F}_g &= -Gm\rho\hat{k}\int_{R_1}^{R_2}\int_0^{2\pi}\left[\int_{r-r'}^{r+r'}\frac{r'^2}{s^2}\frac{s^2+r^2-r'^2}{2sr}\frac{sds}{r'r}\right]d\phi'dr' \\
&= -Gm\rho\hat{k}\int_{R_1}^{R_2}\int_0^{2\pi}\left[\frac{r'}{2r^2}\int_{r-r'}^{r+r'}\left(1+\frac{r^2-r'^2}{s^2}\right)ds\right]d\phi'dr' \\
&= -Gm\rho\hat{k}\int_{R_1}^{R_2}\int_0^{2\pi}\left[\frac{r'}{2r^2}\int_{r-r'}^{r+r'}ds+\frac{r'}{2r^2}(r^2-r'^2)\int_{r-r'}^{r+r'}\frac{1}{s^2}ds\right]d\phi'dr' \\
&= -Gm\rho\hat{k}\int_{R_1}^{R_2}\int_0^{2\pi}\left[\frac{r'}{2r^2}(2r')+ \frac{r'}{2r^2}(r^2-r'^2)\left(\frac{1}{r-r'}-\frac{1}{r+r'}\right)\right]d\phi'dr' \\
&= -Gm\rho\hat{k}\int_{R_1}^{R_2}\int_0^{2\pi}\left[\frac{r'}{2r^2}(2r'+2r')\right]d\phi'dr' \\
&= -\frac{4\pi Gm\rho}{r^2}\hat{k}\int_{R_1}^{R_2}r'^2dr' = -\frac{4\pi Gm\rho(R_2^3-R_1^3)}{3r^2}\hat{k} \\
&= -G\frac{m}{r^2}\left[\rho\left(\frac{4\pi}{3}R_2^3-\frac{4\pi}{3}R_1^3\right)\right]\hat{k} = -G\frac{mM}{r^2}\hat{k} \\
&\Rightarrow \vec{F}_g = -G\frac{mM}{r^2}\hat{k} \text{ at an exterior point}
\end{aligned}$$

2) At an internal point

Consider an infinitesimal shell

$$d\vec{F}_{g,1} = -G \frac{m\rho dV_1'}{s_1^2} \hat{e}_{s_1} = -G \frac{m\rho dr' da_1}{s_1^2} \hat{e}_{s_1} = -G \frac{m\rho dr' s_1^2 d\Omega}{s_1^2 \cos \alpha} \hat{e}_{s_1} = -G \frac{m\rho dr' d\Omega}{\cos \alpha} \hat{e}_{s_1}$$

$$d\vec{F}_{g,2} = -G \frac{m\rho dV_2'}{s_2^2} \hat{e}_{s_2} = -G \frac{m\rho dr' da_2}{s_2^2} \hat{e}_{s_2} = -G \frac{m\rho dr' s_2^2 d\Omega}{s_2^2 \cos \alpha} \hat{e}_{s_2} = -G \frac{m\rho dr' d\Omega}{\cos \alpha} \hat{e}_{s_2}$$

Since $\hat{e}_{s_1} = -\hat{e}_{s_2} \Rightarrow d\vec{F}_g = d\vec{F}_{g,1} + d\vec{F}_{g,2} = 0$

$\Rightarrow \vec{F}_g = \int d\vec{F}_g = 0 \Rightarrow \vec{F}_g = 0$ at an internal point

Note:

Angle $\theta = \frac{s}{r}$;

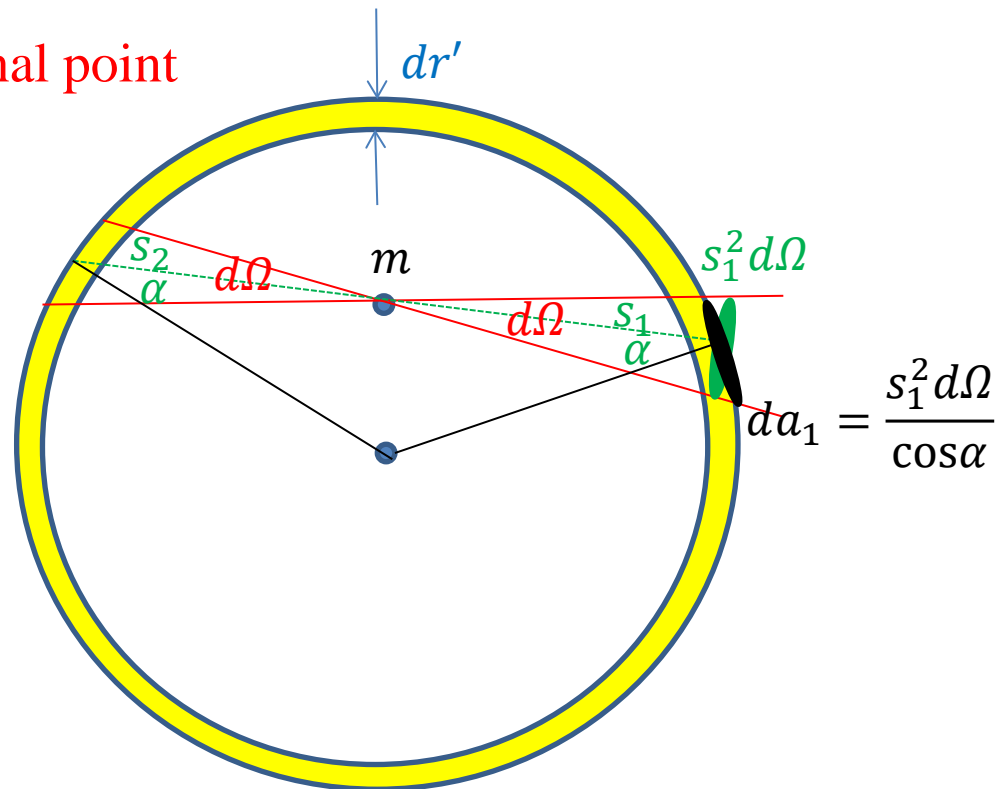
$r \rightarrow$ radius of the circle,

$s \rightarrow$ arc length

Solid Angle $\Omega = \frac{S}{r^2}$;

$r \rightarrow$ radius of the sphere,

$S \rightarrow$ sphere segment area



Examples

I. Gravitation near Earth's surface

$$F_g = G \frac{Mm}{r^2}; \quad r = R \Rightarrow F_g = G \frac{Mm}{R^2} = ma_g \Rightarrow a_g = \frac{GM}{R^2}$$

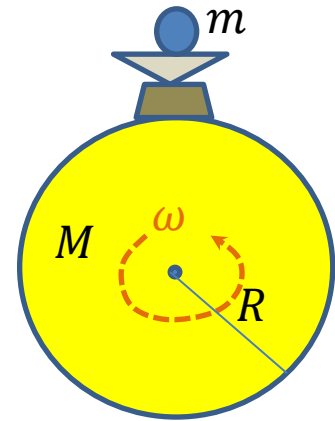
However, a scale that weights an object actually shows the magnitude of the normal force applied by the scale on the object.

At the Equator, for example

$$F_N = mg$$

$$F_g - F_N = m \frac{v^2}{R} = mR\omega^2$$

$$\Rightarrow ma_g - mg = mR\omega^2 \Rightarrow g = a_g - R\omega^2 \text{ at the Equator.}$$



$$R = 6.37 \times 10^6 \text{ m}; \quad \omega = \frac{2\pi}{24 \times 60 \times 60} \text{ s}^{-1} \Rightarrow R\omega^2 = 0.034 \text{ m/s}^2 \ll 9.8 \text{ m/s}^2$$

II. Gravitation inside the Earth

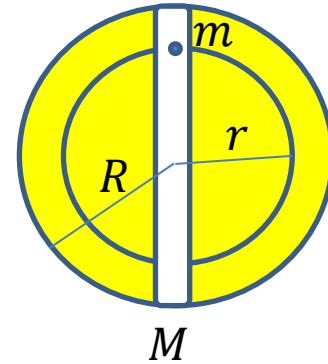
$$M = \rho \frac{4}{3} \pi R^3$$

$$M' = \rho \frac{4}{3} \pi r^3 = M \frac{r^3}{R^3}$$

$$\vec{F}_g = -G \frac{M' m}{r^2} \hat{r} = -G \frac{M m}{R^3} r \hat{r} = -k r \hat{r}$$

$$\Rightarrow \text{Simple Harmonic Motion } k = \frac{GMm}{R^3}$$

$$\Rightarrow \omega = \sqrt{\frac{k}{m}} = \sqrt{\frac{GM}{R^3}} \Rightarrow T = \frac{2\pi}{\omega} = 2\pi \sqrt{\frac{R^3}{GM}}$$



Gravitation potential

Conservative force: $\vec{F} = -\nabla U$

$$\begin{aligned}\Rightarrow \int_1^2 \vec{F} \cdot d\vec{r} &= \int_1^2 (-\nabla U) \cdot d\vec{r} = -\int_1^2 \left(\hat{i} \frac{\partial U}{\partial x} + \hat{j} \frac{\partial U}{\partial y} + \hat{k} \frac{\partial U}{\partial z} \right) \cdot (\hat{i} dx + \hat{j} dy + \hat{k} dz) \\ &= -\int_1^2 \left(\frac{\partial U}{\partial x} dx + \frac{\partial U}{\partial y} dy + \frac{\partial U}{\partial z} dz \right) = -\int_1^2 dU = -[U(\vec{r}_2) - U(\vec{r}_1)] = U(\vec{r}_1) - U(\vec{r}_2)\end{aligned}$$

$$\text{Let } U(\vec{r}_0) = 0 \Rightarrow U(\vec{r}) = \int_{\vec{r}}^{\vec{r}_0} \vec{F} \cdot d\vec{r}$$

For the Earth's gravitational force on a particle of mass m located at a distance r from the center of the Earth:

$$\begin{aligned}\text{Let } U(\infty) = 0 \Rightarrow U(r) &= \int_r^\infty \left(-\frac{GMm}{r'^2} \right) \hat{r}' \cdot d\vec{r}' = -\int_r^\infty \frac{GMm}{r'^2} dr' \\ &= -GMm \left[-\frac{1}{r'} \right]_r^\infty = GMm \left(\frac{1}{\infty} - \frac{1}{r} \right) = -\frac{GMm}{r}\end{aligned}$$

Note: $K + U = \frac{1}{2}mv^2 - \frac{GMm}{R} = 0$ The object escapes from the Earth's gravitation.

$$v = \sqrt{\frac{2GM}{R}} \quad \text{Escape Speed}$$

Kepler's Laws

$$\left\{ \begin{array}{l} \text{Newton's 2nd law } \vec{F} = m\vec{a} = m \frac{d^2 \vec{r}}{dt^2} \\ \text{Newton's law of gravitation } \vec{F} = -G \frac{Mm}{r^2} \hat{r} \end{array} \right. \Rightarrow \frac{d^2 \vec{r}}{dt^2} = -\frac{GM}{r^2} \hat{r}$$

In polar coordinates

$$\vec{r} = r\hat{r} = r(\hat{i} \cos \theta + \hat{j} \sin \theta)$$

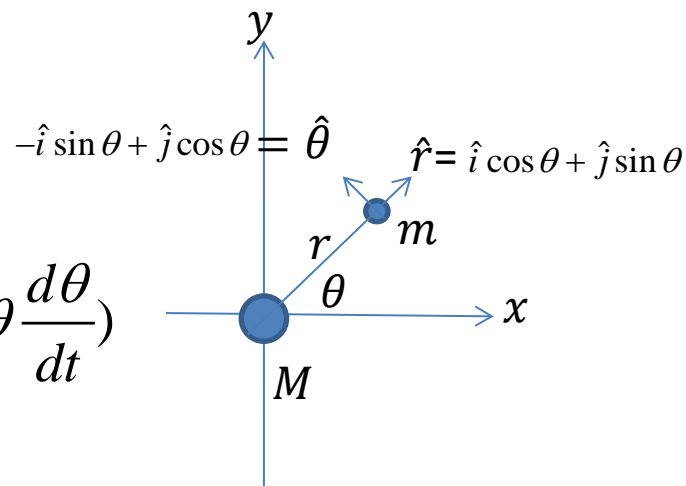
$$\Rightarrow \frac{d\vec{r}}{dt} = \frac{dr}{dt}(\hat{i} \cos \theta + \hat{j} \sin \theta) + r(-\hat{i} \sin \theta \frac{d\theta}{dt} + \hat{j} \cos \theta \frac{d\theta}{dt})$$

$$= \dot{r}\hat{r} + r\omega\hat{\theta}$$

$$\Rightarrow \frac{d^2 \vec{r}}{dt^2} = \frac{d^2 r}{dt^2}(\hat{i} \cos \theta + \hat{j} \sin \theta) + \frac{dr}{dt}(-\hat{i} \sin \theta \frac{d\theta}{dt} + \hat{j} \cos \theta \frac{d\theta}{dt})$$

$$+ \frac{dr}{dt}(-\hat{i} \sin \theta \frac{d\theta}{dt} + \hat{j} \cos \theta \frac{d\theta}{dt})$$

$$+ r[-\hat{i} \cos \theta (\frac{d\theta}{dt})^2 - \hat{i} \sin \theta \frac{d^2 \theta}{dt^2} - \hat{j} \sin \theta (\frac{d\theta}{dt})^2 + \hat{j} \cos \theta \frac{d^2 \theta}{dt^2}]$$



$$= \ddot{r}\hat{r} + 2\dot{r}\omega\hat{\theta} - r\omega^2\hat{r} + r\alpha\hat{\theta} = (\ddot{r} - r\omega^2)\hat{r} + (2\dot{r}\omega + r\alpha)\hat{\theta}$$

$$\Rightarrow (\ddot{r} - r\omega^2)\hat{r} + (2\dot{r}\omega + r\alpha)\hat{\theta} = -\frac{GM}{r^2}\hat{r}$$

$$\Rightarrow \begin{cases} \ddot{r} - r\omega^2 = -\frac{GM}{r^2} \\ 2\dot{r}\omega + r\alpha = 0 \end{cases}$$

$$[\text{Note: } \vec{l} = \vec{r} \times \vec{p} = \vec{r} \times m \frac{d\vec{r}}{dt} = r\hat{r} \times m(\dot{r}\hat{r} + r\omega\hat{\theta}) = (r^2 m \omega)\hat{r} \times \hat{\theta};$$

$$\vec{\tau}_{net} = \vec{r} \times \vec{F} = r\hat{r} \times (-G \frac{Mm}{r^2} \hat{r}) = 0 \Rightarrow \frac{d\vec{l}}{dt} = \vec{\tau}_{net} = 0 \Rightarrow l = r^2 m \omega \text{ is a constant.}]$$

$$\ddot{r} - r\omega^2 = -\frac{GM}{r^2} \Rightarrow \ddot{r} - r\left(\frac{l}{mr^2}\right)^2 = -\frac{GM}{r^2} \Rightarrow \ddot{r} = \frac{l^2}{m^2 r^3} - \frac{GM}{r^2}$$

$$\text{Let } r = \frac{1}{u} \Rightarrow dr = -u^{-2} du = -\frac{du}{u^2}$$

$$\text{Also } l = mr^2 \omega = mr^2 \frac{d\theta}{dt} \Rightarrow dt = \frac{m}{u^2 l} d\theta$$

$$\Rightarrow \frac{dr}{dt} = -\frac{l}{m} \frac{du}{d\theta} \Rightarrow \frac{d^2 r}{dt^2} = -\frac{l}{m} \frac{d(\frac{du}{d\theta})}{dt} = -\frac{l}{m} \frac{d(\frac{du}{d\theta})}{\frac{m}{u^2 l} d\theta} = -\frac{l^2}{m^2} u^2 \frac{d^2 u}{d\theta^2}$$

Therefore $\ddot{r} = \frac{l^2}{m^2 r^3} - \frac{GM}{r^2}$ can be written as

$$-\frac{l^2}{m^2} u^2 \frac{d^2 u}{d\theta^2} = \frac{l^2}{m^2} u^3 - GMu^2 \Rightarrow \frac{d^2 u}{d\theta^2} + u = \frac{GMm^2}{l^2}$$

$$\Rightarrow u = u_c + u_p$$

where u_p is any function that satisfies the equation

and u_c is the general solution of $\frac{d^2 u}{d\theta^2} + u = 0$.

$u_p = \frac{GMm^2}{l^2}$ is apparently a solution.

To solve $\frac{d^2 u_c}{d\theta^2} + u_c = 0$, try $u_c = e^{\alpha\theta} \Rightarrow \alpha = \pm i$

$$\Rightarrow u_c = C_1 e^{i\theta} + C_2 e^{-i\theta} \Rightarrow u = C_1 e^{i\theta} + C_2 e^{-i\theta} + \frac{GMm^2}{l^2}$$

$$\Rightarrow \frac{1}{r} = C_1 e^{i\theta} + C_2 e^{-i\theta} + \frac{GMm^2}{l^2}$$

Note: $\frac{1}{r}$ is real for all θ .

$$1) \text{ When } \theta = 0, \frac{1}{r} = (C_1 + C_2) + \frac{GMm^2}{l^2}. \Rightarrow C_1 + C_2 \text{ is real}$$

$$\Rightarrow \text{Im}[C_1] + \text{Im}[C_2] = 0 \Rightarrow \text{Im}[C_2] = -\text{Im}[C_1]$$

$$2) \text{ When } \theta = \frac{\pi}{2}, \frac{1}{r} = i(C_1 - C_2) + \frac{GMm^2}{l^2}. \Rightarrow C_1 - C_2 \text{ is imaginary}$$

$$\Rightarrow \text{Re}[C_1] - \text{Re}[C_2] = 0 \Rightarrow \text{Re}[C_2] = \text{Re}[C_1]$$

$$\text{Let } C_1 = Ce^{i\phi} \Rightarrow C_2 = Ce^{-i\phi}$$

$$\Rightarrow \frac{1}{r} = C_1 e^{i\theta} + C_2 e^{-i\theta} + \frac{GMm^2}{l^2} = Ce^{i(\theta+\phi)} + Ce^{-i(\theta+\phi)} + \frac{GMm^2}{l^2}$$

$$= 2C \cos(\theta + \phi) + \frac{GMm^2}{l^2} = \frac{GMm^2}{l^2} \left[1 + \frac{2Cl^2}{GMm^2} \cos(\theta + \phi) \right]$$

We can select an x-axis such that $\phi = 0$

$$\text{let } e = \frac{2Cl^2}{GMm^2} \Rightarrow \frac{1}{r} = \frac{GMm^2}{l^2} \left[1 + \frac{2Cl^2}{GMm^2} \cos(\theta + \phi) \right] = \frac{GMm^2}{l^2} (1 + e \cos \theta)$$

$$\text{and } r_p = r(\theta = 0) \Rightarrow \frac{1}{r_p} = \frac{GMm^2}{l^2} (1 + e)$$

$$\Rightarrow \frac{r_p}{r} = \frac{1 + e \cos \theta}{(1 + e)} \Rightarrow \frac{1}{r} = \frac{1 + e \cos \theta}{r_p (1 + e)}$$

Recall that $x = r \cos \theta$ and $y = r \sin \theta$, the above equation can be written as

$$\frac{1}{\sqrt{x^2 + y^2}} = \frac{1 + e \frac{x}{\sqrt{x^2 + y^2}}}{r_p (1 + e)} \Rightarrow \sqrt{x^2 + y^2} = r_p (1 + e) - ex$$

$$\Rightarrow (1 - e^2)x^2 + 2er_p(1 + e)x + y^2 = r_p^2(1 + e)^2$$

$$\Rightarrow x^2 + \frac{2er_p(1 + e)}{(1 - e^2)}x + \frac{y^2}{(1 - e^2)} = \frac{r_p^2(1 + e)^2}{(1 - e^2)}$$

$$\Rightarrow x^2 + \frac{2er_p}{(1 - e)}x + \left[\frac{er_p}{(1 - e)} \right]^2 + \frac{y^2}{(1 - e^2)} = \frac{r_p^2(1 + e)}{(1 - e)} + \left[\frac{er_p}{(1 - e)} \right]^2$$

$$\Rightarrow \left[x + \frac{er_p}{(1-e)} \right]^2 + \frac{y^2}{(1-e^2)} = \frac{r_p^2 - r_p^2 e^2}{(1-e)^2} + \frac{e^2 r_p^2}{(1-e)^2} = \frac{r_p^2}{(1-e)^2}$$

$$\Rightarrow \frac{\left[x + \frac{er_p}{(1-e)} \right]^2}{\left(\frac{r_p}{1-e} \right)^2} + \frac{y^2}{(1-e^2) \frac{r_p^2}{(1-e)^2}} = 1$$

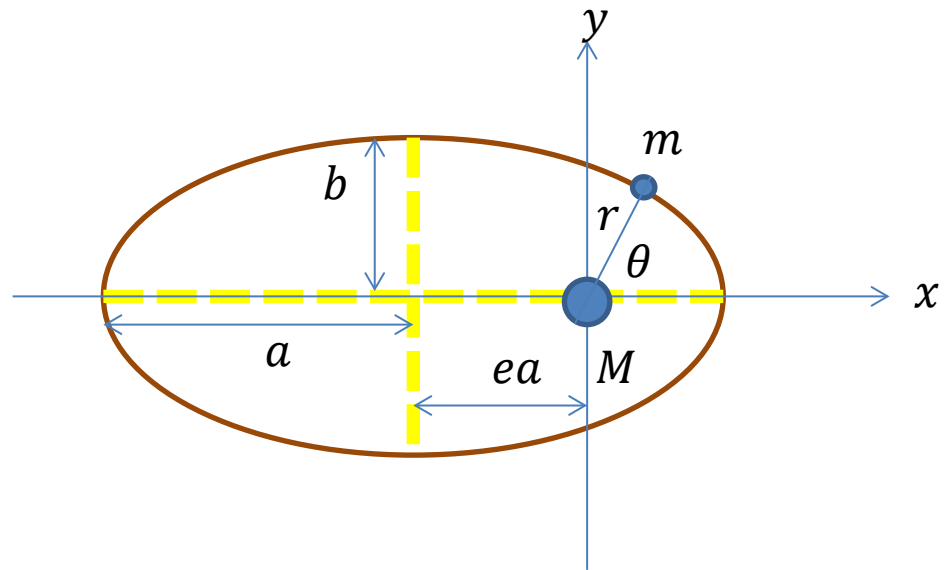
Let $a = \frac{r_p}{1-e}$ and $b = a\sqrt{1-e^2}$

$$\Rightarrow \frac{(x+ea)^2}{a^2} + \frac{y^2}{b^2} = 1$$

(Elliptical Orbit)

Kepler's first law: The law of orbits

Note: If $e = 0 \Rightarrow \frac{x^2}{a^2} + \frac{y^2}{a^2} = 1$ (Circular Orbit)



Recall $\frac{d\vec{r}}{dt} = \dot{r}\hat{r} + r\omega\hat{\theta}$.

The area swept by \vec{r} during time interval dt is

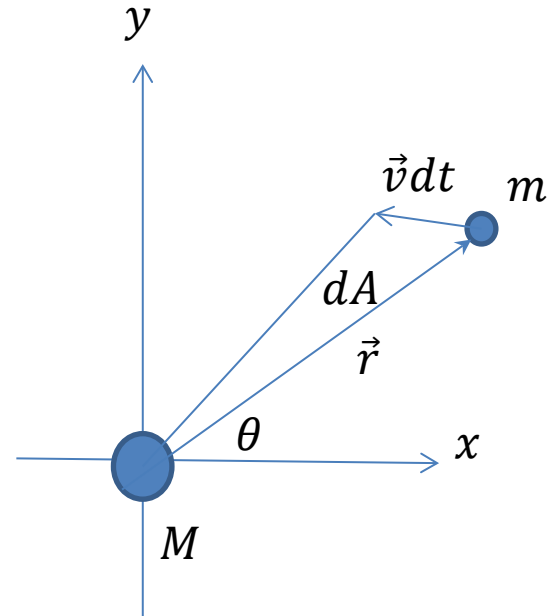
$$dA = \frac{1}{2} |\vec{r} \times \vec{v} dt| = \frac{1}{2} |r\hat{r} \times (\dot{r}\hat{r} + r\omega\hat{\theta})| dt = \frac{1}{2} r^2 \omega dt$$

$$\Rightarrow \frac{dA}{dt} = \frac{1}{2} r^2 \omega$$

Recall that $l = r^2 m \omega$ is a constant. Therefore,

$$\frac{dA}{dt} = \frac{l}{2m} \text{ is a constant.}$$

Kepler's second law: The law of area



The area of an ellipse is πab and recall that $b = a\sqrt{1-e^2}$

$$\Rightarrow T = \frac{\pi ab}{\frac{dA}{dt}} = \frac{\pi ab}{\frac{l}{2m}} = \frac{2m\pi ab}{l} = \frac{2m\pi a^2 \sqrt{1-e^2}}{l}$$

Recall that $\frac{1}{r} = \frac{GMm^2}{l^2} (1 + e \cos \theta) \Rightarrow r = \frac{l^2}{GMm^2 (1 + e \cos \theta)}$

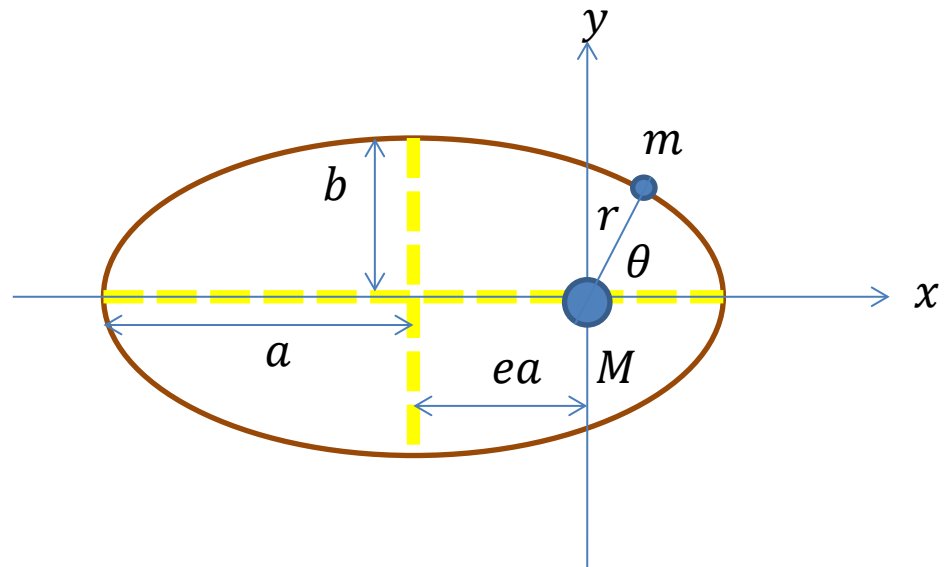
$$2a = r|_{\theta=0} + r|_{\theta=\pi} = \frac{l^2}{GMm^2} \left(\frac{1}{1+e} + \frac{1}{1-e} \right) = \frac{l^2}{GMm^2} \left(\frac{2}{1-e^2} \right)$$

$$\Rightarrow 1-e^2 = \frac{l^2}{aGMm^2}$$

$$\Rightarrow T^2 = \frac{4m^2 \pi^2 a^4 \frac{l^2}{aGMm^2}}{l^2} = \frac{4\pi^2}{GM} a^3$$

$$\Rightarrow \frac{T^2}{a^3} = \frac{4\pi^2}{GM}$$

Kepler's third law: The law of period



Satellites: Orbits and Energy

Potential energy for gravitation: $U = -\frac{GMm}{r}$; Kinetic energy: $\frac{1}{2}mv^2$

Let $r = r_p$ at $\theta = 0$ and $r = r_a$ at $\theta = \pi$.

Note $r_p + r_a = 2a$; $l = r_p m v_p = r_a m v_a \Rightarrow v_p = \frac{r_a}{r_p} v_a$;

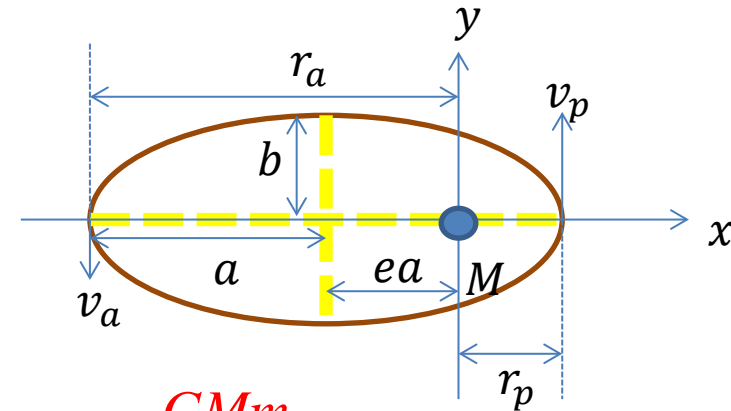
$$E = \frac{1}{2} m v_p^2 - \frac{GMm}{r_p} = \frac{1}{2} m v_a^2 - \frac{GMm}{r_a} \Rightarrow \frac{1}{2} m \frac{r_a^2}{r_p^2} v_a^2 - \frac{GMm}{r_p} = \frac{1}{2} m v_a^2 - \frac{GMm}{r_a}$$

$$\Rightarrow v_a^2 = \left[\frac{GM}{r_p} - \frac{GM}{r_a} \right] \frac{2r_p^2}{r_a^2 - r_p^2} = 2GM \frac{r_p}{r_a(r_a + r_p)} = \frac{GM}{a} \frac{r_p}{r_a}$$

$$\Rightarrow v_p^2 = \frac{r_a^2}{r_p^2} v_a^2 = \frac{GM}{a} \frac{r_a}{r_p}$$

$$E = \frac{1}{2} m v_a^2 - \frac{GMm}{r_a} = \frac{1}{2} \frac{GMm}{a} \frac{r_p}{r_a} - \frac{GMm}{r_a}$$

$$= GMm \frac{r_p - 2a}{2ar_a} = GMm \frac{r_p - (r_p + r_a)}{2ar_a} = -\frac{GMm}{2a} \Rightarrow \mathbf{E = -\frac{GMm}{2a}}$$



Einstein and Gravitation

Principle of Equivalence: Gravitation and acceleration are equivalent.

Newton: Masses \rightarrow Gravitational force

Einstein: Masses \rightarrow Curvature of spacetime \rightarrow Gravitation

Chapter 14 Fluids

Fluid: A substance that flow (liquids, gases)

- Cannot withstand a shearing stress (no force tangential to its surface)
- Can exert a force perpendicular to its surface

Pressure $P = \frac{dF}{dA}$ Density $\rho = \frac{dm}{dV}$

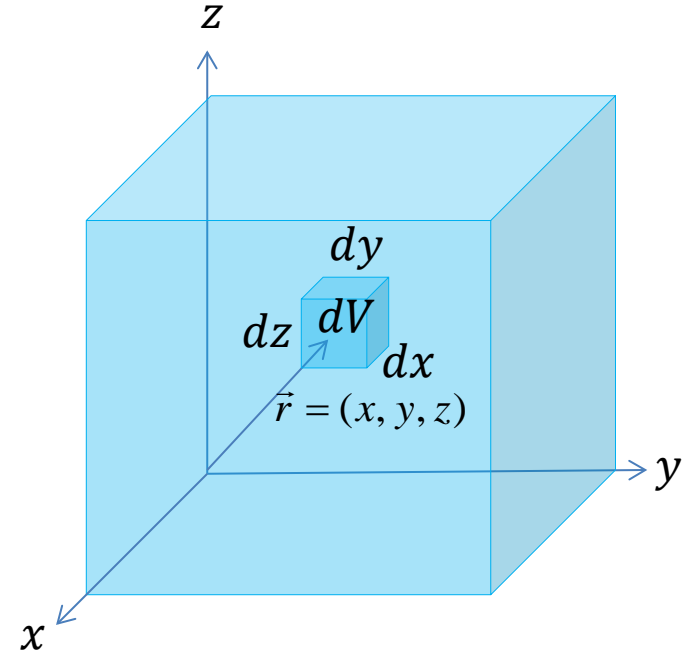
Ideal fluid in motion

- Steady flow (laminar flow): time-independent velocity everywhere
- Incompressible flow: constant and uniform density
- Non-viscous flow: no drag force
- Irrotational flow: no whirlpool

Consider a volume element $dV = dxdydz$ located at a point (x, y, z) inside a fluid.

By Newton's 2nd law, we have

$$\left\{ \begin{array}{l} -[P(x+dx, y, z) - P(x, y, z)]dydz = \rho dxdydz \frac{dv_x}{dt} \\ -[P(x, y+dy, z) - P(x, y, z)]dzdx = \rho dxdydz \frac{dv_y}{dt} \\ -[P(x, y, z+dz) - P(x, y, z)]dxdy - \rho dxdydzg \\ = \rho dxdydz \frac{dv_z}{dt} \end{array} \right.$$



$$\Rightarrow \left\{ \begin{array}{l} -\frac{\partial P}{\partial x} dydz = \rho dydz \frac{dv_x}{dt} \\ -\frac{\partial P}{\partial y} dzdx = \rho dxdz \frac{dv_y}{dt} \\ -\frac{\partial P}{\partial z} dxdy - \rho dxdy g = \rho dxdy \frac{dv_z}{dt} \end{array} \right.$$

$$\Rightarrow \left\{ \begin{array}{l} -\frac{\partial P}{\partial x} = \rho \frac{dv_x}{dt} \\ -\frac{\partial P}{\partial y} = \rho \frac{dv_y}{dt} \\ -\frac{\partial P}{\partial z} - \rho g = \rho \frac{dv_z}{dt} \end{array} \right.$$

$$\Rightarrow \begin{cases} -\frac{\partial P}{\partial x} dx = \rho \frac{dv_x}{dt} dx = \rho \frac{dx}{dt} dv_x = \rho v_x dv_x \\ -\frac{\partial P}{\partial y} dy = \rho \frac{dv_y}{dt} dy = \rho \frac{dy}{dt} dv_y = \rho v_y dv_y \\ -\frac{\partial P}{\partial z} dz = \rho \frac{dz}{dt} dv_z + \rho g dz = \rho v_z dv_z + \rho g dz \end{cases}$$

$$\Rightarrow -dP = -\left(\frac{\partial P}{\partial x} dx + \frac{\partial P}{\partial y} dy + \frac{\partial P}{\partial z} dz\right) = \rho v_x dv_x + \rho v_y dv_y + \rho v_z dv_z + \rho g dz$$

$$\Rightarrow -\int_1^2 dP = \int_1^2 \rho v_x dv_x + \int_1^2 \rho v_y dv_y + \int_1^2 \rho v_z dv_z + \int_1^2 \rho g dz$$

$$\Rightarrow P_1 - P_2 = \frac{1}{2} \rho (v_{x,2}^2 + v_{y,2}^2 + v_{z,2}^2) - \frac{1}{2} \rho (v_{x,1}^2 + v_{y,1}^2 + v_{z,1}^2) + \rho g (z_2 - z_1)$$

$$\Rightarrow P_1 - P_2 = \frac{1}{2} \rho v_2^2 - \frac{1}{2} \rho v_1^2 + \rho g (z_2 - z_1)$$

$$\Rightarrow P_1 + \frac{1}{2} \rho v_1^2 + \rho g z_1 = P_2 + \frac{1}{2} \rho v_2^2 + \rho g z_2$$

$$\Rightarrow \textcolor{red}{P} + \textcolor{red}{\frac{1}{2} \rho v^2} + \textcolor{red}{\rho g z} = \textcolor{red}{\text{a constant (Bernoulli's Equation)}}$$

$$P + \frac{1}{2}\rho v^2 + \rho gz = \text{a constant (Bernoulli's Equation)}$$

For a fluid at rest: $v = 0 \Rightarrow P + \rho gz = \text{a constant}$

Examples:

1. A tank of water open to the atmosphere

$P + \rho gz = P_0 + \rho gz_0$ where P_0 is the atmospheric pressure and z_0 is the water level.

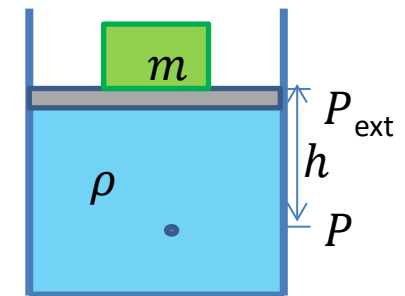
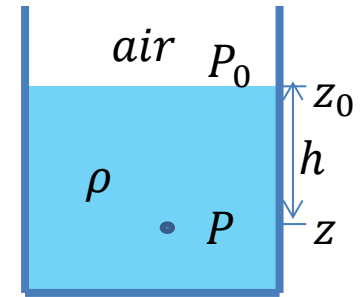
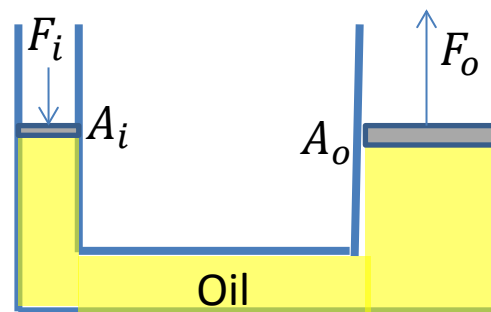
$\Rightarrow P = P_0 + \rho g(z_0 - z) = P_0 + \rho gh$, h is the depth underwater.

2. Pascal's principle: A change in the pressure applied to an enclosed incompressible fluid is transmitted undiminished to every portion of the fluid and to the wall of its container.

$$P = P_{ext} + \rho gh$$

Application: Hydraulic Lever

$$\Delta P = \frac{F_i}{A_i} = \frac{F_o}{A_o} \Rightarrow F_o = F_i \frac{A_o}{A_i} > F_i;$$



$$\text{Incompressibility: } V = A_i d_i = A_o d_o \Rightarrow d_o = d_i \frac{A_i}{A_o} \Rightarrow W = F_o d_o = \left(F_i \frac{A_o}{A_i}\right) \left(d_i \frac{A_i}{A_o}\right) = F_i d_i$$

3. Archimedes' principle

$$P_1 + \rho g z_1 = P_2 + \rho g z_2 \Rightarrow P_2 - P_1 = \rho g(z_1 - z_2)$$

$$dA_1 \cdot \cos \theta_1 = dA_2 \cdot \cos \theta_2 = dA'$$

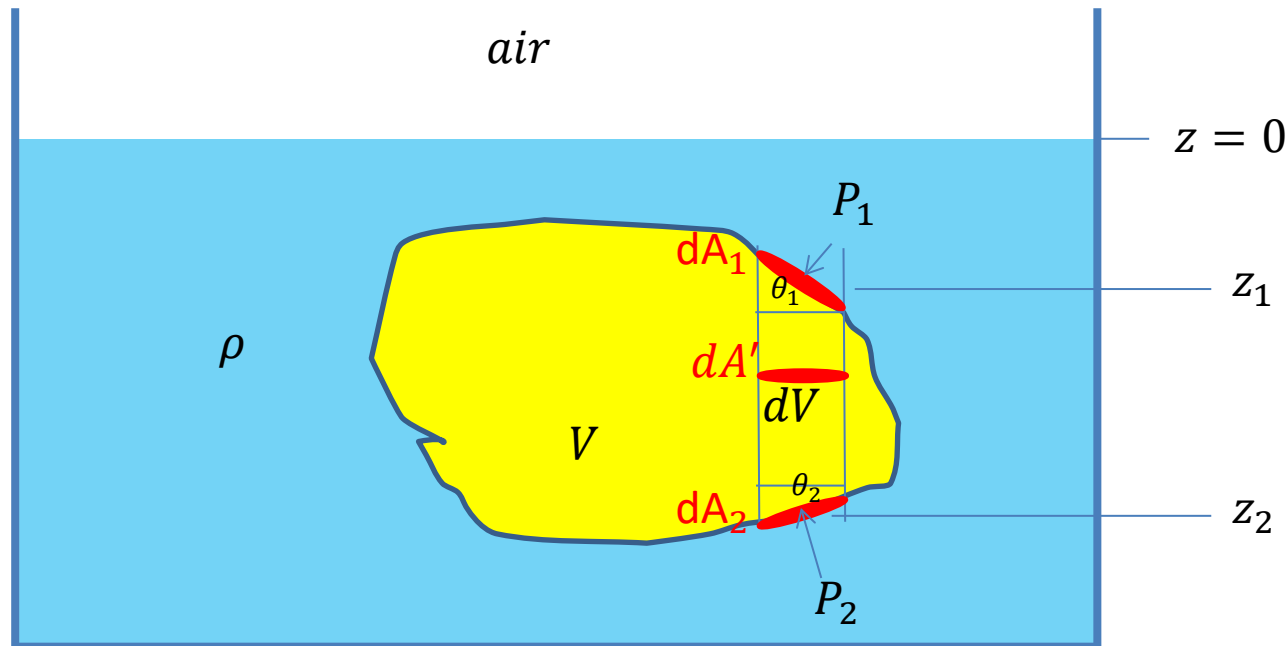
$$dF_b = (P_2 dA_2) \cos \theta_2 - (P_1 dA_1) \cos \theta_1 = (P_2 - P_1) dA' = \rho g(z_1 - z_2) dA' = \rho g dV$$

$$\mathbf{F}_b = \int_V dF_b = \int_V \rho g dV = \rho g \int_V dV = \rho V g$$

The buoyant force is equal to the weight of the fluid displaced by the body.

Note: Floating $\Rightarrow F_b = F_g$

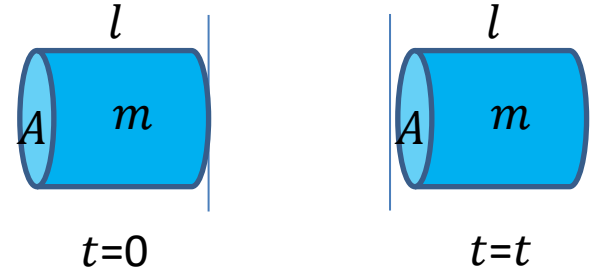
Apparent weight in a fluid $\Rightarrow \text{weight}_{app} = \text{weight} - F_b$



Equation of Continuity

Current density $\vec{J} = \rho \vec{v}$

(Note: for a uniform flow, $\rho v = \frac{m}{Al} \frac{l}{t} = \frac{m}{At}$)



Consider a volume V enclosed by a closed surface S

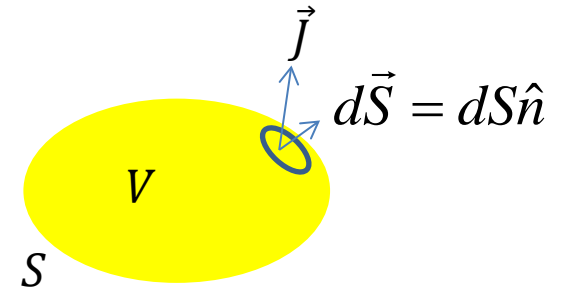
The outgoing mass from V through S per unit time is $\oint_S \vec{J} \cdot d\vec{S}$.

The decrease of mass in V per unit time is $-\frac{d}{dt} \int_V \rho dV = -\int_V \frac{\partial \rho}{\partial t} dV$

Conservation of mass $\Rightarrow \oint_S \vec{J} \cdot d\vec{S} = -\int_V \frac{\partial \rho}{\partial t} dV$

By divergence theorem $\oint_S \vec{J} \cdot d\vec{S} = \int_V \nabla \cdot \vec{J} dV$,

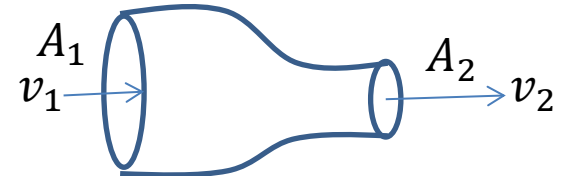
we have $\int_V \nabla \cdot \vec{J} dV = -\int_V \frac{\partial \rho}{\partial t} dV \Rightarrow \nabla \cdot \vec{J} + \frac{\partial \rho}{\partial t} = 0$ (Equation of Continuity)



Example:

$$\frac{\partial \rho}{\partial t} = 0 \Rightarrow \oint_S \vec{J} \cdot d\vec{S} = -\rho v_1 A_1 + \rho v_2 A_2 = 0$$

$$\Rightarrow A_1 v_1 = A_2 v_2$$



Note:

$$\nabla = \hat{i} \frac{\partial}{\partial x} + \hat{j} \frac{\partial}{\partial y} + \hat{k} \frac{\partial}{\partial z}$$

$$\Rightarrow \nabla \cdot \vec{J} = \left(\hat{i} \frac{\partial}{\partial x} + \hat{j} \frac{\partial}{\partial y} + \hat{k} \frac{\partial}{\partial z} \right) \cdot (\hat{i} J_x + \hat{j} J_y + \hat{k} J_z) = \frac{\partial J_x}{\partial x} + \frac{\partial J_y}{\partial y} + \frac{\partial J_z}{\partial z}$$

$$= \frac{J_x(x+dx, y, z) - J_x(x, y, z)}{dx} + \frac{J_y(x, y+dy, z) - J_y(x, y, z)}{dy}$$

$$+ \frac{J_z(x, y, z+dz) - J_z(x, y, z)}{dz}$$

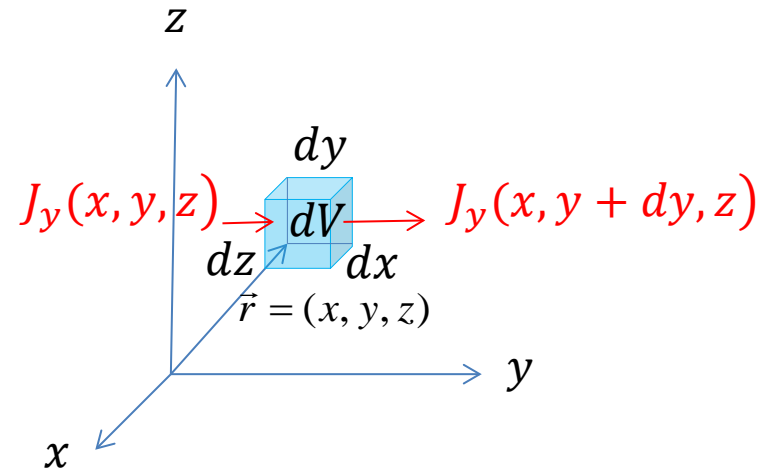
$$= \underbrace{[J_x(x+dx, y, z) - J_x(x, y, z)]dydz}_{\text{(outgoing flux in the x-diection)}} + \underbrace{[J_y(x, y+dy, z) - J_y(x, y, z)]dzdx}_{\text{(outgoing flux in the y-diection)}}$$

$$+ [J_z(x, y, z+dz) - J_z(x, y, z)]dxdy \} \frac{1}{dxdydz}$$

(outgoing flux in the y-diection)

$$\Rightarrow \nabla \cdot \vec{J} \text{ (divergence of } \vec{J} \text{)}$$

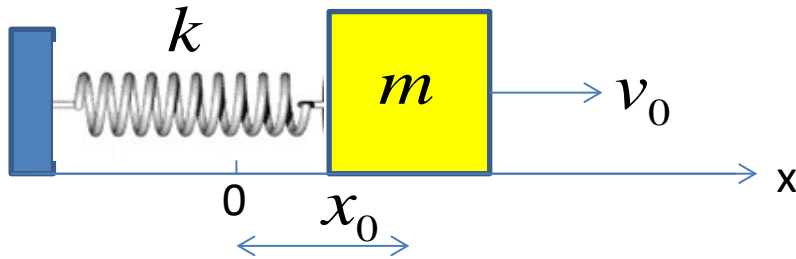
→ outgoing flux of \vec{J} per unit volume



Chapter 15 Oscillations

1. Simple Harmonic Motion
 - Pendulum
2. Damped Simple Harmonic Motion
3. Forced Oscillations

Simple Harmonic Motion



Initial State $(x(0), p(0)) = (x_0, mv_0)$

Force $F_{net} = -kx$

$$\text{Newton's 2nd Law: } F_{ext} = ma = m \frac{d}{dt} \left(\frac{dx}{dt} \right) = m \frac{d^2 x}{dt^2} \Rightarrow m \frac{d^2 x}{dt^2} = -kx \Rightarrow \frac{d^2 x}{dt^2} + \frac{k}{m} x = 0$$

$$\text{Let } \omega^2 = \frac{k}{m}, \text{ we have } \boxed{\frac{d^2 x}{dt^2} + \omega^2 x = 0} \text{ (a second-order linear differential equation)}$$

Note:

The solutions of a second-order linear homogeneous differential equation

$$a \frac{d^2 f(x)}{dx^2} + b \frac{df(x)}{dx} + cf(x) = 0$$

form a 2 dimensional linear space (set of functions).

Any linear combination $a_1 f_1(x) + a_2 f_2(x)$ of solutions $f_1(x)$ and $f_2(x)$ is also a solution.

If $f_1(x)$ and $f_2(x)$ are linearly independent solutions, then the general solution is given by

$f(x) = a_1 f_1(x) + a_2 f_2(x)$, where a_1 and a_2 are arbitrary constants.

To find two independent solutions, try $x(t) = e^{\alpha t}$.

$$\frac{d^2}{dt^2} e^{\alpha t} + \omega^2 e^{\alpha t} = 0 \Rightarrow \alpha \frac{d}{dt} e^{\alpha t} + \omega^2 e^{\alpha t} = 0 \Rightarrow \alpha^2 e^{\alpha t} + \omega^2 e^{\alpha t} = 0$$

$$\Rightarrow \alpha^2 + \omega^2 = 0 \Rightarrow \alpha = \pm i\omega$$

\Rightarrow We have two independent solutions $x_1(t) = e^{i\omega t}$, $x_2(t) = e^{-i\omega t}$

And the general solution is $x(t) = c_1 e^{i\omega t} + c_2 e^{-i\omega t}$

$$\Rightarrow v(t) = \frac{dx}{dt} = i\omega c_1 e^{i\omega t} - i\omega c_2 e^{-i\omega t}$$

Initial conditions $x(0) = x_0$, $v(0) = v_0$

$$\Rightarrow c_1 + c_2 = x_0$$

$$c_1 - c_2 = \frac{v_0}{i\omega} = -i \frac{v_0}{\omega}$$

$$\Rightarrow c_1 = \frac{x_0}{2} - i \frac{v_0}{2\omega} = \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp\left[-i \tan^{-1}\left(\frac{v_0}{x_0 \omega}\right)\right]$$

$$c_2 = \frac{x_0}{2} + i \frac{v_0}{2\omega} = \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp\left[i \tan^{-1}\left(\frac{v_0}{x_0 \omega}\right)\right]$$

Note:

$$e^{i\theta} = \cos \theta + i \sin \theta$$

$$\text{let } \cos \theta = \frac{x_0 / 2}{\sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}}},$$

$$\sin \theta = \frac{v_0 / 2\omega}{\sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}}}$$

$$\begin{aligned}
x(t) &= \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp[-i \tan^{-1}(\frac{v_0}{x_0\omega})] e^{i\omega t} + \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp[i \tan^{-1}(\frac{v_0}{x_0\omega})] e^{-i\omega t} \\
&= \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \left\{ \exp[i(\omega t - \tan^{-1}(\frac{v_0}{x_0\omega}))] + \exp[-i(\omega t - \tan^{-1}(\frac{v_0}{x_0\omega}))] \right\} \\
&= \sqrt{x_0^2 + \frac{v_0^2}{\omega^2}} \cos[\omega t - \tan^{-1}(\frac{v_0}{x_0\omega})] \\
v(t) &= i\omega \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp[-i \tan^{-1}(\frac{v_0}{x_0\omega})] e^{i\omega t} - i\omega \sqrt{\frac{x_0^2}{4} + \frac{v_0^2}{4\omega^2}} \exp[i \tan^{-1}(\frac{v_0}{x_0\omega})] e^{-i\omega t} \\
&= -\omega \sqrt{x_0^2 + \frac{v_0^2}{\omega^2}} \sin[\omega t - \tan^{-1}(\frac{v_0}{x_0\omega})]
\end{aligned}$$

$$\left. \begin{aligned} F_{net} &= m \frac{d^2 x}{dt^2} = -kx \\ x(0) &= x_0; \quad v(0) = v_0 \end{aligned} \right\} \Rightarrow \left\{ \begin{aligned} x(t) &= x_m \cos(\omega t + \phi) \\ v(t) &= -\omega x_m \sin(\omega t + \phi) \end{aligned} \right.$$

where $\omega = \sqrt{\frac{k}{m}}$, $x_m = \sqrt{x_0^2 + \frac{v_0^2}{\omega^2}}$, $\phi = -\tan^{-1}(\frac{v_0}{x_0\omega})$

Note:

$$1. \omega = \sqrt{\frac{k}{m}} \Rightarrow T = \frac{2\pi}{\omega} = 2\pi \sqrt{\frac{m}{k}}$$

$$2. F = -kx \Rightarrow \text{If } F \text{ is a conservative force then } F = -\frac{dU}{dx}$$

$$\Rightarrow kx = \frac{dU}{dx} \Rightarrow dU = kx dx \Rightarrow \int dU = \int kx dx \Rightarrow U(x) = \frac{1}{2} kx^2 + C$$

$$\text{Let } U(0) = 0 \Rightarrow U(x) = \frac{1}{2} kx^2 = \frac{1}{2} kx_m^2 \cos^2(\omega t + \phi)$$

$$K = \frac{1}{2} mv^2 = \frac{1}{2} m\omega^2 x_m^2 \sin^2(\omega t + \phi) = \frac{1}{2} m \frac{k}{m} x_m^2 \sin^2(\omega t + \phi) = \frac{1}{2} kx_m^2 \sin^2(\omega t + \phi)$$

$$\Rightarrow E = K + U = \frac{1}{2} kx_m^2 \cos^2(\omega t + \phi) + \frac{1}{2} kx_m^2 \sin^2(\omega t + \phi) = \frac{1}{2} kx_m^2$$

Pendulum

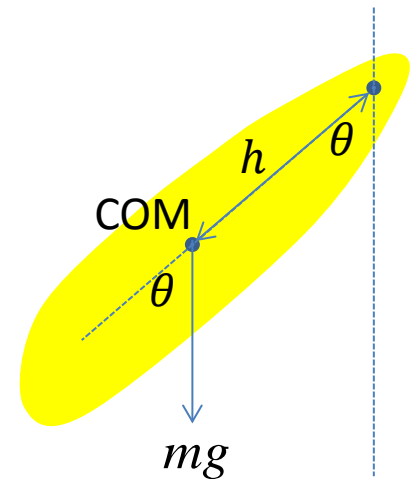
Newton's 2nd Law $\tau_{net} = I\alpha$

Torque $\vec{\tau} = \vec{r} \times \vec{F}$

$$\tau_{net} = -hmg \sin \theta = I\alpha$$

(Note: $\vec{\tau}$ is in the direction pertaining to the decreasing θ)

$$\alpha = -\frac{hmg}{I} \sin \theta = -\frac{hmg}{I} \left(\theta - \frac{\theta^3}{3!} + \frac{\theta^5}{5!} + \dots \right) \cong -\frac{hmg}{I} \theta \text{ (for small } \theta \text{)}$$



Note:

1. Taylor's series $f(x) = \sum_{n=0}^{\infty} \frac{1}{n!} \frac{d^n f(x)}{dx^n} \bigg|_{x=x_0} (x-x_0)^n$

$$f(x) \rightarrow \sin \theta; x \rightarrow \theta; x_0 \rightarrow \theta_0 = 0 \Rightarrow \sin \theta = \theta - \frac{\theta^3}{3!} + \frac{\theta^5}{5!} + \dots$$

2. How small is small enough for θ such that $\sin \theta \cong \theta$?

$$\frac{\theta^3}{3!} = \frac{\theta^2}{3!} \cong 0.005 (\theta = 10^\circ), 0.01 (\theta = 15^\circ), 0.02 (\theta = 20^\circ)$$

$$\alpha = -\frac{hmg}{I}\theta \Rightarrow \frac{d^2\theta}{dt^2} = -\frac{hmg}{I}\theta$$

analogous to

$$\frac{d^2x}{dt^2} = -\frac{k}{m}x \Rightarrow \begin{cases} x(t) = x_m \cos(\omega t + \phi) \\ v(t) = -\omega x_m \sin(\omega t + \phi) \end{cases}$$

where $\omega = \sqrt{\frac{k}{m}}$, $x_m = \sqrt{x_0^2 + \frac{v_0^2}{\omega^2}}$, $\phi = -\tan^{-1}\left(\frac{v_0}{x_0\omega}\right)$

$$\Rightarrow \frac{d^2\theta}{dt^2} = -\frac{hmg}{I}\theta \Rightarrow \begin{cases} \theta(t) = \theta_m \cos(\Omega t + \phi) \\ \omega(t) = -\Omega \theta_m \sin(\Omega t + \phi) \end{cases}$$

where $\Omega = \sqrt{\frac{hmg}{I}}$, $\theta_m = \sqrt{\theta_0^2 + \frac{\omega_0^2}{\Omega^2}}$, $\phi = -\tan^{-1}\left(\frac{\omega_0}{\theta_0\Omega}\right)$

Note: $T = \frac{2\pi}{\Omega} = 2\pi\sqrt{\frac{I}{hmg}}$ (Physical pendulum, small amplitude)

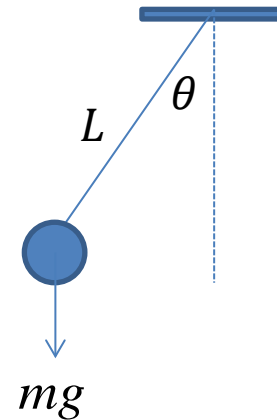
Simple pendulum

$$I = mL^2$$

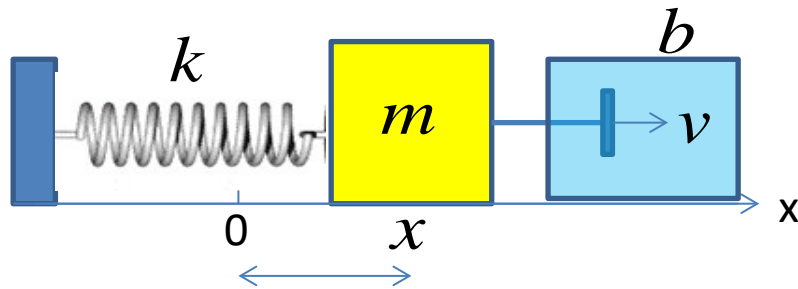
$$h = L$$

$$\Rightarrow T = 2\pi \sqrt{\frac{mL^2}{Lmg}} = 2\pi \sqrt{\frac{L}{g}}$$

(Simple pendulum, small amplitude)



Damped Simple Harmonic Motion



Initial State $(x(0), p(0)) = (x_0, mv_0)$

Force $F_{net} = -kx - bv$

Newton's 2nd Law: $m \frac{d^2 x}{dt^2} = -kx - b \frac{dx}{dt} \Rightarrow \frac{d^2 x}{dt^2} + \frac{b}{m} \frac{dx}{dt} + \frac{k}{m} x = 0$

Let $\omega^2 = \frac{k}{m}$ and $2\beta = \frac{b}{m}$, we have $\frac{d^2 x}{dt^2} + 2\beta \frac{dx}{dt} + \omega^2 x = 0$

To find two independent solutions, try $x(t) = e^{\alpha t}$.

$$\frac{d^2}{dt^2} e^{\alpha t} + 2\beta \frac{d}{dt} e^{\alpha t} + \omega^2 e^{\alpha t} = 0 \Rightarrow \alpha^2 e^{\alpha t} + 2\beta \alpha e^{\alpha t} + \omega^2 e^{\alpha t} = 0$$

$$\Rightarrow \alpha^2 + 2\beta \alpha + \omega^2 = 0 \Rightarrow \alpha = \frac{-2\beta \pm \sqrt{4\beta^2 - 4\omega^2}}{2} = -\beta \pm i\sqrt{\omega^2 - \beta^2} \quad (\text{if } \omega > \beta)$$

$$\Rightarrow \text{We have two independent solutions } x_1(t) = e^{-\beta t + i\sqrt{\omega^2 - \beta^2} t}, x_2(t) = e^{-\beta t - i\sqrt{\omega^2 - \beta^2} t}$$

Let $\omega' = \sqrt{\omega^2 - \beta^2}$. The general solution is $x(t) = c_1 e^{-\beta t + i\omega' t} + c_2 e^{-\beta t - i\omega' t}$

$$\Rightarrow v(t) = \frac{dx}{dt} = (-\beta + i\omega') c_1 e^{-\beta t + i\omega' t} + (-\beta - i\omega') c_2 e^{-\beta t - i\omega' t}$$

Initial conditions $x(0) = x_0$, $v(0) = v_0$

$$\Rightarrow c_1 + c_2 = x_0;$$

$$(-\beta + i\omega')c_1 + (-\beta - i\omega')c_2 = v_0$$

$$\Rightarrow c_1 = \frac{x_0}{2} - i \frac{v_0 + \beta x_0}{2\omega'}$$

$$= \sqrt{\frac{x_0^2}{4} + \frac{(v_0 + \beta x_0)^2}{4\omega'^2}} \exp[-i \tan^{-1}(\frac{v_0 + \beta x_0}{x_0 \omega'})]$$

$$c_2 = \frac{x_0}{2} + i \frac{v_0 + \beta x_0}{2\omega'}$$

$$= \sqrt{\frac{x_0^2}{4} + \frac{(v_0 + \beta x_0)^2}{4\omega'^2}} \exp[i \tan^{-1}(\frac{v_0 + \beta x_0}{x_0 \omega'})]$$

Note:

$$e^{i\theta} = \cos \theta + i \sin \theta$$

$$\text{let } \cos \theta = \frac{x_0 / 2}{\sqrt{\frac{x_0^2}{4} + \frac{(v_0 + \beta x_0)^2}{4\omega'^2}}},$$

$$\sin \theta = \frac{(v_0 + \beta x_0) / 2\omega'}{\sqrt{\frac{x_0^2}{4} + \frac{(v_0 + \beta x_0)^2}{4\omega'^2}}}$$

$$\begin{aligned}
 x(t) &= \sqrt{\frac{x_0^2}{4} + \frac{(v_0 + \beta x_0)^2}{4\omega'^2}} \exp[-i \tan^{-1}(\frac{v_0 + \beta x_0}{x_0 \omega'})] e^{-\beta t + i\omega' t} \\
 &+ \sqrt{\frac{x_0^2}{4} + \frac{(v_0 + \beta x_0)^2}{4\omega'^2}} \exp[i \tan^{-1}(\frac{v_0 + \beta x_0}{x_0 \omega'})] e^{-\beta t - i\omega' t} \\
 &= \sqrt{x_0^2 + \frac{(v_0 + \beta x_0)^2}{\omega'^2}} e^{-\beta t} \cos[\omega' t - \tan^{-1}(\frac{v_0 + \beta x_0}{x_0 \omega'})]
 \end{aligned}$$

$$x(t) = x'_m e^{-\beta t} \cos(\omega' t + \phi'),$$

$$\text{where } \omega' = \sqrt{\omega^2 - \beta^2}; \omega = \sqrt{\frac{k}{m}}; x'_m = \sqrt{x_0^2 + \frac{(v_0 + \beta x_0)^2}{\omega'^2}}; \phi' = -\tan^{-1}(\frac{v_0 + \beta x_0}{x_0 \omega'})$$

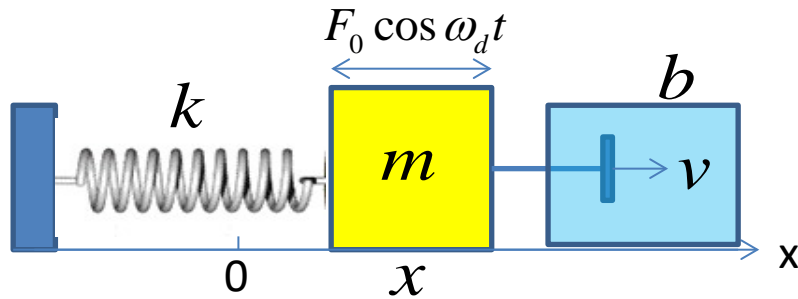
Note: I. Compare to undamped SHM

$$x(t) = x_m \cos(\omega t + \phi), \text{ where } \omega = \sqrt{\frac{k}{m}}, x_m = \sqrt{x_0^2 + \frac{v_0^2}{\omega^2}}, \phi = -\tan^{-1}(\frac{v_0}{x_0 \omega})$$

II. 1. Amplitude $x'_m e^{-\beta t}$ decreases exponentially with time.

$$2. \omega' = \sqrt{\omega^2 - \beta^2} < \omega \Rightarrow T' = \frac{2\pi}{\omega'} > T = \frac{2\pi}{\omega}$$

Forced Oscillations



Initial State $(x(0), p(0)) = (x_0, mv_0)$

Force $F_{net} = -kx - bv + F_0 \cos \omega_d t$

Newton's 2nd Law: $m \frac{d^2 x}{dt^2} = -kx - b \frac{dx}{dt} + F_0 \cos \omega_d t \Rightarrow \frac{d^2 x}{dt^2} + \frac{b}{m} \frac{dx}{dt} + \frac{k}{m} x = \frac{F_0}{m} \cos \omega_d t$

Let $\omega^2 = \frac{k}{m}$, $2\beta = \frac{b}{m}$ and $A = \frac{F_0}{m}$, we have $\frac{d^2 x}{dt^2} + 2\beta \frac{dx}{dt} + \omega^2 x = A \cos \omega_d t$

The general solution $x(t) = x_c(t) + x_p(t)$

$x_p(t)$ (particular solution) is any solution of the nonhomogeneous equation.

$x_c(t)$ (complementary function) is the general solution of $\frac{d^2 x}{dt^2} + 2\beta \frac{dx}{dt} + \omega^2 x = 0$

$\Rightarrow x_c(t) = c_1 e^{-\beta t + i\omega' t} + c_2 e^{-\beta t - i\omega' t}$ where $\omega' = \sqrt{\omega^2 - \beta^2}$.

To find a particular solution $x_p(t)$, try $x_p(t) = D \cos(\omega_d t - \delta) = \text{Re}[D e^{i(\omega_d t - \delta)}]$

$\frac{d^2}{dt^2} \text{Re}[D e^{i(\omega_d t - \delta)}] + 2\beta \frac{d}{dt} \text{Re}[D e^{i(\omega_d t - \delta)}] + \omega^2 \text{Re}[D e^{i(\omega_d t - \delta)}] = A \cos \omega_d t = \text{Re}[A e^{i\omega_d t}]$

$$\Rightarrow \operatorname{Re}\left[\frac{d^2}{dt^2}De^{i(\omega_d t - \delta)} + 2\beta\frac{d}{dt}De^{i(\omega_d t - \delta)} + \omega^2 De^{i(\omega_d t - \delta)}\right] = \operatorname{Re}[Ae^{i\omega_d t}]$$

Apparently, the above equation can be automatically satisfied if

$$\frac{d^2}{dt^2}De^{i(\omega_d t - \delta)} + 2\beta\frac{d}{dt}De^{i(\omega_d t - \delta)} + \omega^2 De^{i(\omega_d t - \delta)} = Ae^{i\omega_d t}.$$

$$\Rightarrow -\omega_d^2 De^{i(\omega_d t - \delta)} + i2\omega_d\beta De^{i(\omega_d t - \delta)} + \omega^2 De^{i(\omega_d t - \delta)} = Ae^{i\omega_d t}$$

$$\Rightarrow -\omega_d^2 De^{-i\delta} + i2\omega_d\beta De^{-i\delta} + \omega^2 De^{-i\delta} = A$$

$$\Rightarrow -\omega_d^2 D(\cos \delta - i \sin \delta) + i2\omega_d\beta D(\cos \delta - i \sin \delta) + \omega^2 D(\cos \delta - i \sin \delta) = A$$

$$\Rightarrow \begin{cases} -\omega_d^2 D \cos \delta + 2\omega_d\beta D \sin \delta + \omega^2 D \cos \delta = A \\ \omega_d^2 D \sin \delta + 2\omega_d\beta D \cos \delta - \omega^2 D \sin \delta = 0 \end{cases} \Rightarrow \begin{cases} (\omega^2 - \omega_d^2) D \cos \delta + 2\omega_d\beta D \sin \delta = A \\ -(\omega^2 - \omega_d^2) D \sin \delta + 2\omega_d\beta D \cos \delta = 0 \end{cases}$$

$$\Rightarrow \delta = \tan^{-1}\left(\frac{2\omega_d\beta}{\omega^2 - \omega_d^2}\right) \Rightarrow \cos \delta = \frac{\omega^2 - \omega_d^2}{\sqrt{(\omega^2 - \omega_d^2)^2 + (2\omega_d\beta)^2}}; \sin \delta = \frac{2\omega_d\beta}{\sqrt{(\omega^2 - \omega_d^2)^2 + (2\omega_d\beta)^2}}$$

$$\Rightarrow D = \frac{A}{(\omega^2 - \omega_d^2) \cos \delta + 2\omega_d\beta \sin \delta} = \frac{A}{\frac{(\omega^2 - \omega_d^2)^2}{\sqrt{(\omega^2 - \omega_d^2)^2 + (2\omega_d\beta)^2}} + \frac{(2\omega_d\beta)^2}{\sqrt{(\omega^2 - \omega_d^2)^2 + (2\omega_d\beta)^2}}}$$

$$= \frac{A}{\sqrt{(\omega^2 - \omega_d^2)^2 + (2\omega_d\beta)^2}} = \frac{A}{\sqrt{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2\beta^2}}$$

We have $x_p(t) = \frac{A}{\sqrt{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2\beta^2}} \cos(\omega_d t - \delta)$

and $x(t) = x_c(t) + x_p(t) = c_1 e^{-\beta t + i\omega' t} + c_2 e^{-\beta t - i\omega' t} + \frac{A}{\sqrt{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2\beta^2}} \cos(\omega_d t - \delta)$

$$v(t) = \frac{dx}{dt} = (-\beta + i\omega')c_1 e^{-\beta t + i\omega' t} + (-\beta - i\omega')c_2 e^{-\beta t - i\omega' t} - \frac{A\omega_d}{\sqrt{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2\beta^2}} \sin(\omega_d t - \delta)$$

initial conditions $x(0) = x_0 \Rightarrow c_1 + c_2 + \frac{(\omega^2 - \omega_d^2)A}{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2\beta^2} = x_0$

$$v(0) = v_0 \Rightarrow (-\beta + i\omega')c_1 + (-\beta - i\omega')c_2 + \frac{2\omega_d^2\beta A}{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2\beta^2} = v_0$$

Let $x'_0 = x_0 - \frac{(\omega^2 - \omega_d^2)A}{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2\beta^2}$ and $v'_0 = v_0 - \frac{2\omega_d^2\beta A}{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2\beta^2}$

$$\Rightarrow c_1 + c_2 = x'_0 ; (-\beta + i\omega')c_1 + (-\beta - i\omega')c_2 = v'_0$$

$$\Rightarrow x(t) = x''_m e^{-\beta t} \cos(\omega' t + \phi'') + \frac{A}{\sqrt{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2\beta^2}} \cos(\omega_d t - \delta), \text{ where } \omega' = \sqrt{\omega^2 - \beta^2};$$

$$x''_m = \sqrt{x'^2_0 + \frac{(v'_0 + \beta x'_0)^2}{\omega'^2}}; \phi'' = -\tan^{-1}\left(\frac{v'_0 + \beta x'_0}{x'_0 \omega'}\right); \delta = \tan^{-1}\left(\frac{2\omega_d \beta}{\omega^2 - \omega_d^2}\right); \text{ and } A = \frac{F_0}{m}$$

$$x(t) = x_m'' e^{-\beta t} \cos(\omega' t + \phi'') + \frac{A}{\sqrt{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2 \beta^2}} \cos(\omega_d t - \delta)$$

when $t \gg \frac{1}{\beta}$, the transient term $x_m'' e^{-\beta t} \cos(\omega' t + \phi'')$ is negligible. (damped out with time)

$$\Rightarrow x(t) \cong \frac{A}{\sqrt{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2 \beta^2}} \cos(\omega_d t - \delta) \text{ at large } t.$$

$$\Rightarrow D(\omega_d) = \frac{A}{\sqrt{(\omega^2 - \omega_d^2)^2 + 4\omega_d^2 \beta^2}}$$

To find the maximum amplitude, $\frac{dD(\omega_d)}{d\omega_d} = 0$

$$\Rightarrow -\frac{1}{2} \frac{A}{[(\omega^2 - \omega_d^2)^2 + 4\omega_d^2 \beta^2]^{\frac{3}{2}}} [-4\omega_d(\omega^2 - \omega_d^2) + 8\beta^2 \omega_d] = 0$$

$$\Rightarrow \frac{A\{-2\omega_d[\omega_d^2 - (\omega^2 - 2\beta^2)]\}}{[(\omega^2 - \omega_d^2)^2 + 4\omega_d^2 \beta^2]^{\frac{3}{2}}} = 0 \Rightarrow \omega_d = \sqrt{\omega^2 - 2\beta^2} = \omega_R \text{ (resonance frequency)}$$

Note: 1. When the frequency ω_d of the applied periodical force is $\sqrt{\omega^2 - 2\beta^2}$, the oscillations have maximum amplitude.

2. If there is no damping, $b = 0 \Rightarrow \beta = 0 \Rightarrow \omega_R = \omega$

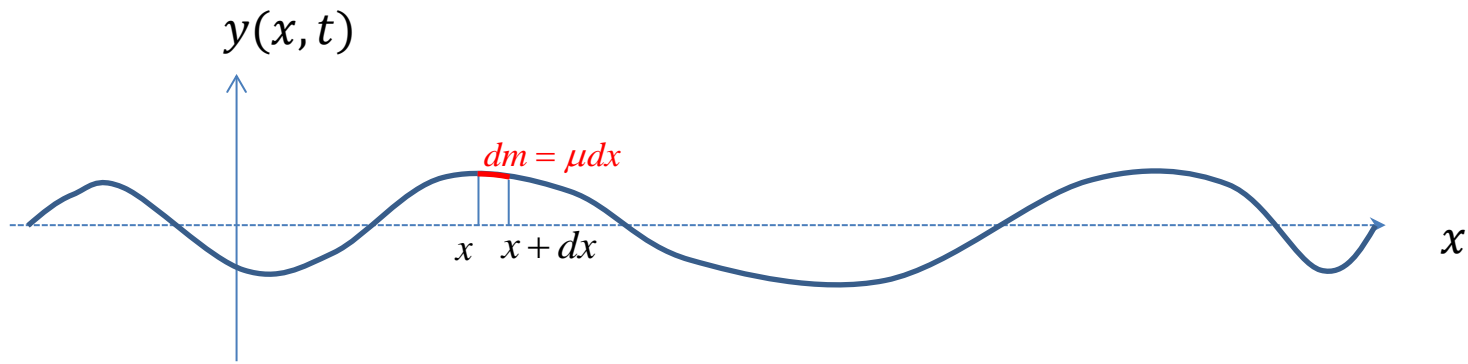
Chapter 16 Waves I.

1. Mechanical waves (transverse, longitudinal)

- governed by Newton's laws
- exist only within a material medium

2. Electromagnetic waves

3. Matter waves



For a taut and transversely waved string of linear density μ and tension τ in the x -direction, the x -component of the tension is τ , and the slope of the string is $\frac{\partial y(x, t)}{\partial x}$. $\left. \begin{array}{l} \text{the } x\text{-component of the tension is } \tau, \\ \text{and the slope of the string is } \frac{\partial y(x, t)}{\partial x}. \end{array} \right\} \Rightarrow \text{The } y\text{-component of the tension is } \tau \frac{\partial y(x, t)}{\partial x}$

Consider an infinitesimal section dx of mass $dm = \mu dx$.

The net force on dm has an x -component equal to zero

and y -component $\tau \frac{\partial y(x, t)}{\partial x} \Big|_{x=x+dx} - \tau \frac{\partial y(x, t)}{\partial x} \Big|_{x=x}$.

By Newton's 2nd law, $\tau \frac{\partial y(x, t)}{\partial x} \Big|_{x=x+dx} - \tau \frac{\partial y(x, t)}{\partial x} \Big|_{x=x} = dm \frac{\partial^2 y(x, t)}{\partial t^2} = \mu dx \frac{\partial^2 y(x, t)}{\partial t^2}$

$$\Rightarrow \frac{\tau \frac{\partial y(x, t)}{\partial x} \Big|_{x=x+dx} - \tau \frac{\partial y(x, t)}{\partial x} \Big|_{x=x}}{dx} = \mu \frac{\partial^2 y(x, t)}{\partial t^2} \Rightarrow \boxed{\frac{\partial^2 y(x, t)}{\partial x^2} - \frac{\mu}{\tau} \frac{\partial^2 y(x, t)}{\partial t^2} = 0} \quad (\text{wave equation})$$

On the other hand, consider a sinusoidal wave function $y(x, t) = y_m \sin(kx - \omega t)$

y_m : Amplitude; $(kx - \omega t)$: Phase; k : wave number; ω : angular frequency

Note:

1. $y(x + n \frac{2\pi}{k}, t) = y(x, t), n = 0, 1, 2, \dots \Rightarrow \frac{2\pi}{k} = \lambda$ (wavelength)

2. $y(x, t + n \frac{2\pi}{\omega}) = y(x, t), n = 0, 1, 2, \dots \Rightarrow \frac{2\pi}{\omega} = T$ (period)

3. For any phase $\Phi = kx_0 - \omega t_0$, if $k(x_0 + dx) - \omega(t_0 + dt) = \Phi \Rightarrow \frac{dx}{dt} = \frac{\omega}{k} = v$ (phase velocity)

4. $\frac{\partial^2 y(x, t)}{\partial x^2} = -k^2 y(x, t)$ and $\frac{\partial^2 y(x, t)}{\partial t^2} = -\omega^2 y(x, t) \Rightarrow \frac{\partial^2 y(x, t)}{\partial x^2} - \frac{1}{v^2} \frac{\partial^2 y(x, t)}{\partial t^2} = 0$

5. Apparently, $y(x, t) = y_m \sin(kx - \omega t)$ is a solution for the wave equation

$$\frac{\partial^2 y(x, t)}{\partial x^2} - \frac{\mu}{\tau} \frac{\partial^2 y(x, t)}{\partial t^2} = 0 \text{ if } \frac{\omega}{k} = \sqrt{\frac{\tau}{\mu}}.$$

\Rightarrow The phase velocity for a sinusoidal traveling wave on a string is $v = \sqrt{\frac{\tau}{\mu}}$.

6. Let $y_1(x, t) = y_{m,1} \sin(k_1 x - \omega_1 t)$; $y_2(x, t) = y_{m,2} \sin(k_2 x - \omega_2 t)$.

If $\frac{\omega_1}{k_1} = \frac{\omega_2}{k_2} = \sqrt{\frac{\tau}{\mu}}$, then y_1, y_2 , and $c_1 y_1 + c_2 y_2$ all satisfy $\frac{\partial^2 y(x, t)}{\partial x^2} - \frac{\mu}{\tau} \frac{\partial^2 y(x, t)}{\partial t^2} = 0$.

Energy Transport

Consider a sinusoidal traveling wave function $y(x, t) = y_m \sin(kx - \omega t)$

At any fixed point x_0 , $y(x_0, t) = y_m \sin(kx_0 - \omega t) = -y_m \sin(\omega t - kx_0) = y_m \cos(\omega t - kx_0 + \frac{\pi}{2})$

\Rightarrow a string element at x_0 of mass $dm = \mu dx$ undergoes a simple harmonic motion.

Note: In a sinusoidal traveling wave, the string is arranged to exert a spring-like force on the

string element dm with a spring constant $\omega^2 dm$ (recall $\omega = \sqrt{\frac{k}{m}}$ in the spring force motion.)

\Rightarrow The mechanical energy of the string element dm is $dE = \frac{1}{2}(dm)\omega^2 y_m^2 = \frac{1}{2}(\mu dx)\omega^2 y_m^2$

(recall $E = K + U = \frac{1}{2}kx_m^2$ in the spring force motion.)

As the wave propagates, such energy is transmitted at a velocity $v = \frac{dx}{dt}$ to the positive x -direction.

\Rightarrow Power $P = \frac{dE}{dt} = \frac{1}{2}\mu \frac{dx}{dt} \omega^2 y_m^2 = \frac{1}{2}\mu v \omega^2 y_m^2$

Principle of Superposition for Waves; Consider $\frac{\partial^2 y(x,t)}{\partial x^2} - \frac{1}{v^2} \frac{\partial^2 y(x,t)}{\partial t^2} = 0$

(For mechanical waves, the above equation implies that $y(x,t)$ satisfies Newton's 2nd Law.)

$$\text{If } \frac{\partial^2 y_1(x,t)}{\partial x^2} - \frac{1}{v^2} \frac{\partial^2 y_1(x,t)}{\partial t^2} = 0 \text{ and } \frac{\partial^2 y_2(x,t)}{\partial x^2} - \frac{1}{v^2} \frac{\partial^2 y_2(x,t)}{\partial t^2} = 0$$

$$\text{then } \frac{\partial^2}{\partial x^2} [y_1(x,t) + y_2(x,t)] - \frac{1}{v^2} \frac{\partial^2}{\partial t^2} [y_1(x,t) + y_2(x,t)] = 0.$$

(i.e. The resultant wave $y_1(x,t) + y_2(x,t)$ is also allowed by Newton's 2nd Law.)

Example I. Interference of waves

$$y_1(x,t) = y_m \sin(kx - \omega t); y_2(x,t) = y_m \sin(kx - \omega t + \phi); \phi : \text{phase shift}$$

$$y(x,t) = y_1(x,t) + y_2(x,t) = y_m \sin(kx - \omega t) + y_m \sin(kx - \omega t + \phi)$$

$$= y_m \sin[(kx - \omega t + \frac{\phi}{2}) - \frac{\phi}{2}] + y_m \sin[(kx - \omega t + \frac{\phi}{2}) + \frac{\phi}{2}]$$

$$= y_m [\sin(kx - \omega t + \frac{\phi}{2}) \cos \frac{\phi}{2} - \sin \frac{\phi}{2} \cos(kx - \omega t + \frac{\phi}{2})$$

$$+ \sin(kx - \omega t + \frac{\phi}{2}) \cos \frac{\phi}{2} + \sin \frac{\phi}{2} \cos(kx - \omega t + \frac{\phi}{2})] = (2y_m \cos \frac{\phi}{2}) \sin(kx - \omega t + \frac{\phi}{2})$$

$$(i) \phi = 0 \Rightarrow y(x,t) = 2y_m \sin(kx - \omega t + \frac{\phi}{2}) \text{ fully constructive.}$$

$$(ii) \phi = \pi \Rightarrow y(x,t) = 0 \text{ fully destructive. (iii) } 0 < \phi < \pi \Rightarrow \text{intermediate interference.}$$

Example II. Standing waves

$$y_1(x, t) = y_m \sin(kx - \omega t); \quad y_2(x, t) = y_m \sin(kx + \omega t); \quad v_1 = \frac{\omega}{k}, v_2 = -\frac{\omega}{k}, \quad v_2 = -v_1$$

$$y(x, t) = y_1(x, t) + y_2(x, t) = y_m \sin(kx - \omega t) + y_m \sin(kx + \omega t)$$

$$= y_m [\sin kx \cos \omega t - \sin \omega t \cos kx + \sin kx \cos \omega t + \sin \omega t \cos kx] = (2y_m \sin kx) \cos \omega t$$

(i) $kx = n\pi, n = 0, 1, 2, \dots \Rightarrow y(x, t) = 0 \Rightarrow x = \frac{n\pi}{k} = n \frac{\lambda}{2}, n = 0, 1, 2, \dots$ (the nodes)

(ii) $kx = (n + \frac{1}{2})\pi, n = 0, 1, 2, \dots \Rightarrow y(x, t) = 2y_m \cos \omega t$

$$\Rightarrow x = \frac{(n + \frac{1}{2})\pi}{k} = (n + \frac{1}{2}) \frac{\lambda}{2}, n = 0, 1, 2, \dots \text{ (the antinodes).}$$

Note: Standing waves by reflection

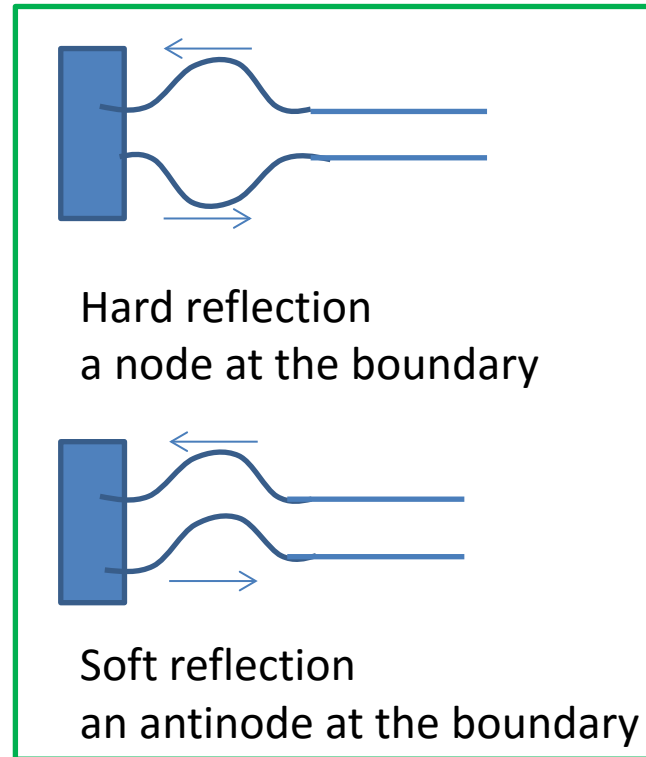
For a string, there is a node on each fixed point of the string.

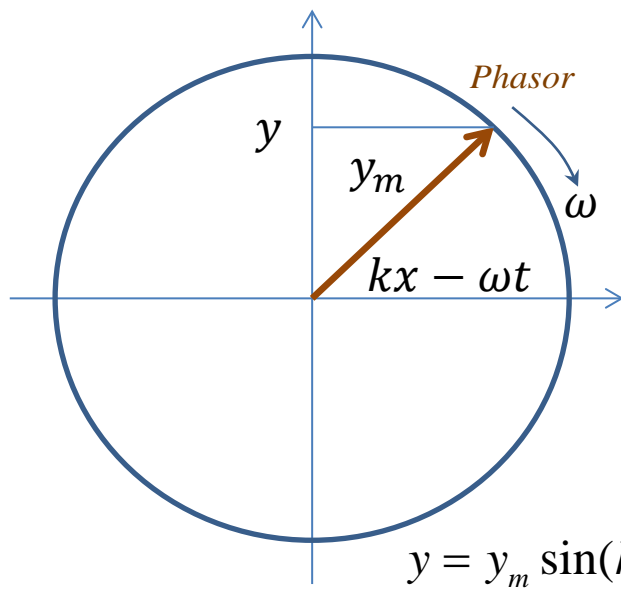
$$L = x_2 - x_1 = n_2 \frac{\lambda}{2} - n_1 \frac{\lambda}{2} = (n_2 - n_1) \frac{\lambda}{2} = n \frac{\lambda}{2}$$

$$\Rightarrow \text{allowed wavelengths are } \lambda_n = \frac{2L}{n}, \quad n = 1, 2, 3, \dots$$

$$\Rightarrow \text{allowed frequencies are } f_n = \frac{v}{\lambda} = n \frac{v}{2L}, \quad n = 1, 2, 3, \dots \text{ (resonant frequencies)}$$

n : harmonic number \rightarrow n th harmonic. The string resonates at these frequencies.





Phasor: A vector with a magnitude equal to the amplitude y_m of the wave and rotates around an origin with angular speed equal to the angular frequency ω of the wave.

Application: Summation of two waves of the same ω (and $k = \frac{\omega}{v}$)

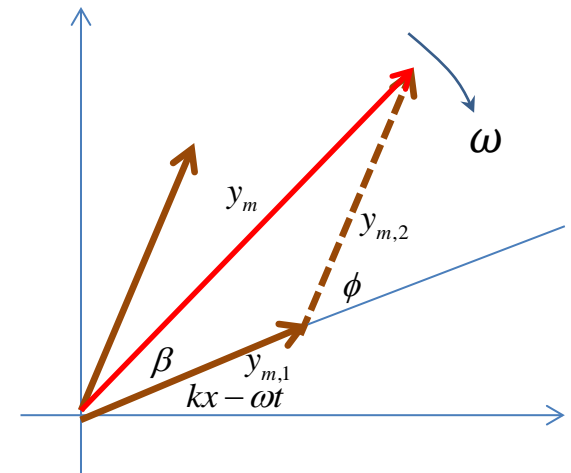
$$y_1(x, t) = y_{m,1} \sin(kx - \omega t); \quad y_2(x, t) = y_{m,2} \sin(kx - \omega t + \phi)$$

$$\Rightarrow y(x, t) = y_1(x, t) + y_2(x, t) = y_m \sin(kx - \omega t + \beta)$$

Given $y_{m,1}$, $y_{m,2}$ and ϕ , use the phasor method to calculate y_m and β .

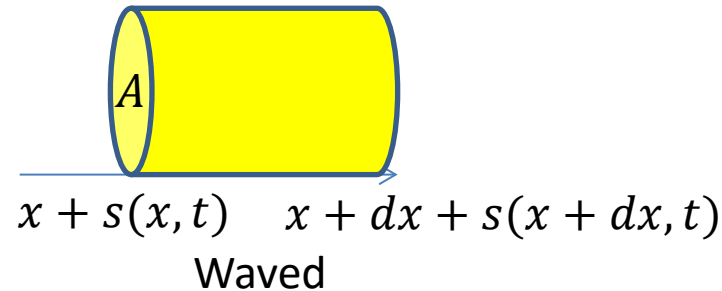
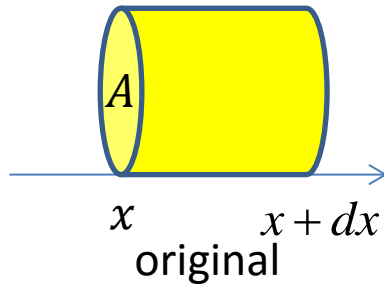
Cosine law

$$\cos(\pi - \phi) = \frac{y_{m,1}^2 + y_{m,2}^2 - y_m^2}{2y_{m,1}y_{m,2}}; \quad \cos \beta = \frac{y_m^2 + y_{m,1}^2 - y_{m,2}^2}{2y_m y_{m,1}}$$



Chapter 17 Waves II.

Sound waves: longitudinal mechanical waves



A body of air is waved in the x -direction with a longitudinal displacement function $s(x, t)$ such that the original position x of any point of the air is moved to $x + s(x, t)$

Consider an air element of cross sectional area A originally located between x and $x + dx$.

The original volume of the air element is $A dx$.

The waved volume of the air element is

$$A\{[(x + dx) + s(x + dx, t)] - [x + s(x, t)]\} = A dx + A[s(x + dx, t) - s(x, t)] = A dx + A \frac{\partial s(x, t)}{\partial x} dx.$$

The increase of volume $A \frac{\partial s(x, t)}{\partial x} dx$ leads to a decrease of pressure $B \frac{A \frac{\partial s(x, t)}{\partial x} dx}{A dx} = B \frac{\partial s(x, t)}{\partial x}$.

(Recall $\Delta P = -B \frac{\Delta V}{V}$, where P is the pressure and B is the bulk modulus.)

The difference between pressure on the left and that on the right results in a net external force

on the air element.
$$F_{net} = A \left(B \frac{\partial s(x,t)}{\partial x} \Big|_{x=x+dx} - B \frac{\partial s(x,t)}{\partial x} \Big|_{x=x} \right) = AB \frac{\partial^2 s(x,t)}{\partial x^2} dx$$

Noting that the acceleration of the air element is $\frac{\partial^2 s(x,t)}{\partial t^2}$ in the x -direction.

\Rightarrow By Newton's 2nd Law
$$AB \frac{\partial^2 s(x,t)}{\partial x^2} dx = \rho A dx \frac{\partial^2 s(x,t)}{\partial t^2}$$

$$\frac{\partial^2 s(x,t)}{\partial x^2} - \frac{\rho}{B} \frac{\partial^2 s(x,t)}{\partial t^2} = 0 \quad (\text{wave equation})$$

Consider the traveling sinusoidal function $s(x,t) = s_m \cos(kx - \omega t)$.

$$\frac{\partial^2 s(x,t)}{\partial x^2} = -s_m k^2 \cos(kx - \omega t); \quad \frac{\partial^2 s(x,t)}{\partial t^2} = -s_m \omega^2 \cos(kx - \omega t)$$

$$\Rightarrow \frac{\partial^2 s(x,t)}{\partial x^2} - \frac{1}{v^2} \frac{\partial^2 s(x,t)}{\partial t^2} = 0, \quad \text{where } v = \frac{\omega}{k}.$$

$$\Rightarrow s(x,t) = s_m \cos(kx - \omega t) \text{ is a solution to the sound wave equation if } \frac{\omega}{k} = v = \sqrt{\frac{B}{\rho}}.$$

Note:
$$\Delta P = -B \frac{\partial s(x,t)}{\partial x} = B s_m k \sin(kx - \omega t) = \rho v^2 s_m \frac{\omega}{v} \sin(kx - \omega t) = (\rho v \omega) s_m \sin(kx - \omega t)$$

Interference

If $L_1 \gg d$ and $L_2 \gg d$

\Rightarrow Two waves travel in the same direction at P.

Suppose S_1 and S_2 are in phase.

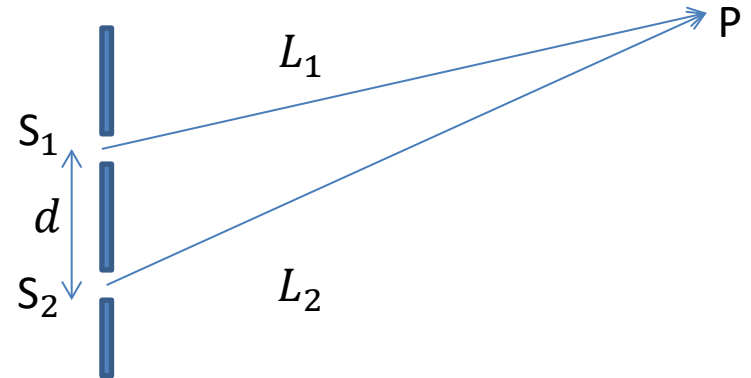
At point P, path length difference $\Delta L = |L_2 - L_1|$.

\Rightarrow phase difference $\phi = k\Delta L = \frac{\Delta L}{\lambda} 2\pi$

(i) $\frac{\Delta L}{\lambda} = m = 0, 1, 2, \dots \Rightarrow \phi = 2m\pi$ fully constructive interference

(ii) $\frac{\Delta L}{\lambda} = m + \frac{1}{2} = \frac{1}{2}, \frac{3}{2}, \frac{5}{2}, \dots \Rightarrow \phi = (2m+1)\pi$ fully destructive interference

(iii) everything else \Rightarrow intermediate interference.



Similar to the sinusoidal string wave $y(x, t) = y_m \sin(kx - \omega t)$

in which $dm = \mu dx$ and power $P = \frac{dE}{dt} = \frac{1}{2} \mu v \omega^2 y_m^2$,

for sinusoidal sound wave $s(x, t) = s_m \cos(kx - \omega t)$

in which $dm = \rho A dx$ the energy transport rate is $P = \frac{dE}{dt} = \frac{1}{2} \rho A v \omega^2 s_m^2$.

The intensity of the wave is defined as $I = \frac{P}{A} = \frac{1}{2} \rho v \omega^2 s_m^2$.

For a point source, $A = 4\pi r^2 \Rightarrow I = \frac{P_s}{4\pi r^2}$.

The Decibel Scale

Deci-: $\frac{1}{10}$; -bel: Bell (Alexander Graham Bell)

Sound Level $\beta = (10dB) \log \frac{I}{I_0}$ where $I_0 = 10^{-12} W / m^2$

Note: If $I = I_0 \Rightarrow \beta = 0$

Examples: Hearing threshold $0dB$, Rustle of leaves $10dB$, Conversation $60dB$

Rock concert $110dB$, Pain threshold $120dB$, Jet engine $130dB$

Sound of Music: Standing waves

Recall the standing waves on a string

$$y(x, t) = y_1(x, t) + y_2(x, t) = y_m \sin(kx - \omega t) + y_m \sin(kx + \omega t) = (2y_m \sin kx) \cos \omega t$$

$$(i) \ x = \frac{n\pi}{k} = n \frac{\lambda}{2}, n = 0, 1, 2, \dots \text{(the nodes)}; (ii) \ x = (n + \frac{1}{2}) \frac{\lambda}{2}, n = 0, 1, 2, \dots \text{(the antinodes)}.$$

Standing waves by reflection

For a string, there is a node on each fixed point of the string.

$$L = x_2 - x_1 = n_2 \frac{\lambda}{2} - n_1 \frac{\lambda}{2} = (n_2 - n_1) \frac{\lambda}{2} = n \frac{\lambda}{2}$$

$$\Rightarrow \lambda_n = \frac{2L}{n}, n = 1, 2, 3, \dots; f_n = \frac{v}{\lambda} = n \frac{v}{2L}, n = 1, 2, 3, \dots \text{ Note: } v = \sqrt{\frac{\tau}{\mu}} \text{ (tunable)}$$

For pipes: open ends \Rightarrow antinodes; closed ends \Rightarrow nodes

(a) Two open ends

$$L = x_2 - x_1 = (n_2 + \frac{1}{2}) \frac{\lambda}{2} - (n_1 + \frac{1}{2}) \frac{\lambda}{2} = (n_2 - n_1) \frac{\lambda}{2} = n \frac{\lambda}{2} \Rightarrow \lambda_n = \frac{2L}{n}; f_n = \frac{v}{\lambda} = n \frac{v}{2L}, n = 1, 2, 3, \dots$$

(b) One open end and one closed end

$$L = x_2 - x_1 = (n_2 + \frac{1}{2}) \frac{\lambda}{2} - n_1 \frac{\lambda}{2} = (n_2 - n_1) \frac{\lambda}{2} + \frac{\lambda}{4} = \frac{(2n+1)\lambda}{4} \Rightarrow \lambda_n = \frac{4L}{2n+1}; f_n = \frac{v}{\lambda} = \frac{(2n+1)v}{4L},$$

$$\text{Note: } v = \sqrt{\frac{B}{\rho}} \text{ (not tunable)}$$

$$n = 1, 2, 3, \dots$$

Beats

Consider two waves $s_1(x, t) = s_m \cos(k_1 x - \omega_1 t + \phi_1)$; $s_2(x, t) = s_m \cos(k_2 x - \omega_2 t + \phi_2)$

Note: $\frac{\omega_1}{k_1} = \frac{\omega_2}{k_2} = v$, $f_1 = \frac{\omega_1}{2\pi}$, $f_2 = \frac{\omega_2}{2\pi}$

At point P where $x = x_0$,

$$s(x_0, t) = s_1(x_0, t) + s_2(x_0, t) = s_m [\cos(k_1 x_0 + \phi_1 - \omega_1 t) + \cos(k_2 x_0 + \phi_2 - \omega_2 t)]$$

Noting that $\cos \alpha + \cos \beta = \cos\left(\frac{\alpha + \beta}{2} + \frac{\alpha - \beta}{2}\right) + \cos\left(\frac{\alpha + \beta}{2} - \frac{\alpha - \beta}{2}\right) = \cos \frac{\alpha + \beta}{2} \cos \frac{\alpha - \beta}{2}$

Let $\alpha = k_1 x_0 + \phi_1 - \omega_1 t$ and $\beta = k_2 x_0 + \phi_2 - \omega_2 t$

We have $s(x_0, t) = s_m [\cos \alpha + \cos \beta] = 2s_m \cos\left(\phi'_1 - \frac{\omega_1 + \omega_2}{2} t\right) \cos\left(\phi'_2 - \frac{\omega_1 - \omega_2}{2} t\right)$,

where $\phi'_1 = \frac{1}{2}(k_1 x_0 + \phi_1 + k_2 x_0 + \phi_2)$ and $\phi'_2 = \frac{1}{2}(k_1 x_0 + \phi_1 - k_2 x_0 - \phi_2)$

\Rightarrow Oscillating function $2s_m \cos\left(\phi'_1 - \frac{\omega_1 + \omega_2}{2} t\right)$; $f = \frac{\frac{\omega_1 + \omega_2}{2}}{2\pi} = \frac{f_1 + f_2}{2}$

Upper envelope $2s_m \left| \cos\left(\phi'_2 - \frac{\omega_1 - \omega_2}{2} t\right) \right|$ and lower envelope $-2s_m \left| \cos\left(\phi'_2 - \frac{\omega_1 - \omega_2}{2} t\right) \right|$

beat frequency $f_{beat} = 2 \times \frac{\frac{\omega_1 - \omega_2}{2}}{2\pi} = f_1 - f_2$

Doppler Effect

A source traveling with a velocity v_s emits a wave of frequency f and wave velocity v traveling on a stationery medium towards a detector which is traveling with a velocity v_D .

If n wavefronts are detected by the detector during a time interval Δt , the frequency seen by the detector is $f' = \frac{n}{\Delta t}$.

Let d be the distance between consecutive wavefronts on the medium.

$$\Rightarrow d = (v - v_s)T = \frac{v - v_s}{f}.$$

The speed of the wavefronts with respect to the detector is $v_{rel} = v - v_D$.

Therefore, the number of wavefronts detected by the detector during Δt is

$$n = \frac{(v - v_D)\Delta t}{d} = \frac{(v - v_D)\Delta t}{\frac{v - v_s}{f}} = \frac{v - v_D}{v - v_s} f \Delta t$$

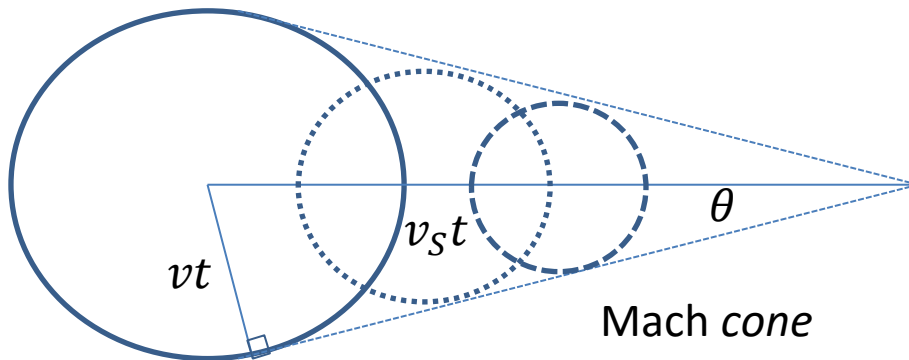
$$\Rightarrow f' = \frac{n}{\Delta t} = \frac{v - v_D}{v - v_s} f$$

Supersonic Speeds, Shock waves.

$$\text{If } v_s = v \Rightarrow f' = \frac{v - v_D}{v - v_s} f \rightarrow \infty$$

When $v_s > v \Rightarrow$ shock wave.

$$\sin \theta = \frac{vt}{v_s t} = \frac{v}{v_s} \quad (\text{Mach cone angle})$$



$$\frac{\partial^2 s(x,t)}{\partial x^2} - \frac{1}{v^2} \frac{\partial^2 s(x,t)}{\partial t^2} = 0$$

Separation of variables: $s(x,t) = X(x)T(t)$

$$\text{substituted into the equation} \Rightarrow T(t) \frac{d^2 X(x)}{dx^2} = \frac{1}{v^2} X(x) \frac{d^2 T(t)}{dt^2}$$

$$\Rightarrow \frac{T(t) \frac{d^2 X(x)}{dx^2}}{X(x)T(t)} = \frac{\frac{1}{v^2} X(x) \frac{d^2 T(t)}{dt^2}}{X(x)T(t)} \Rightarrow \frac{\frac{d^2 X(x)}{dx^2}}{X(x)} = \frac{\frac{1}{v^2} \frac{d^2 T(t)}{dt^2}}{T(t)}$$

$$\text{Note: } \frac{\frac{d^2 X(x)}{dx^2}}{X(x)} \text{ is a function of } x \text{ and } \frac{\frac{1}{v^2} \frac{d^2 T(t)}{dt^2}}{T(t)} \text{ is a function of } t.$$

For the equality to hold, both of them have to be the same constant (say $-k^2$).

$$\Rightarrow \begin{cases} \frac{\frac{d^2 X(x)}{dx^2}}{X(x)} = -k^2 \\ \frac{\frac{1}{v^2} \frac{d^2 T(t)}{dt^2}}{T(t)} = -k^2 \end{cases} \Rightarrow \begin{cases} \frac{d^2 X(x)}{dx^2} + k^2 X(x) = 0 \\ \frac{d^2 T(t)}{dt^2} + k^2 v^2 T(t) = 0 \end{cases} \Rightarrow \begin{cases} \frac{d^2 X(x)}{dx^2} + k^2 X(x) = 0 \\ \frac{d^2 T(t)}{dt^2} + \omega^2 T(t) = 0 \text{ (Let } k^2 v^2 = \omega^2) \end{cases}$$

$$\Rightarrow \begin{cases} X(x) = c_1 e^{ikx} + c_2 e^{-ikx} \\ T(t) = c_3 e^{i\omega t} + c_4 e^{-i\omega t} \end{cases}$$

$$\Rightarrow s(x, t) = X(x)T(t) = A_1 e^{i(kx - \omega t)} + A_2 e^{i(kx + \omega t)} + A_3 e^{-i(kx - \omega t)} + A_4 e^{-i(kx + \omega t)}, \text{ where } \frac{\omega^2}{k^2} = v^2$$

Of special interest, if $A_1 = A_3 = \frac{1}{2} s_m$ and $A_2 = A_4 = 0$, we have $s(x, t) = s_m \cos(kx - \omega t)$

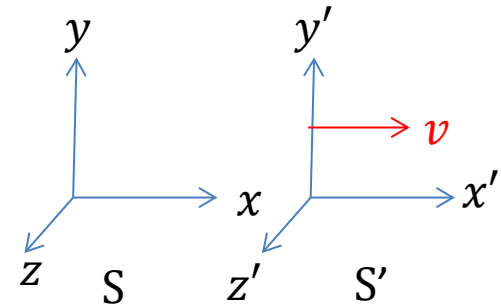
Chapter 37 Relativity

- **Relativity**: transforming measurements between reference frames that move relative to each other.
 - **Special Relativity** → Inertial reference frames, where Newton's laws are valid.
 - General Relativity → Reference frames can undergo gravitational acceleration

Galilean Transformation

$$\begin{cases} x' = x - vt \\ y' = y \\ z' = z \\ t' = t \end{cases} \Rightarrow \begin{cases} dx' = dx - vdt \\ dy' = dy \\ dz' = dz \\ dt' = dt \end{cases} \Rightarrow \begin{cases} u'_x = \frac{dx'}{dt'} = \frac{dx - vdt}{dt} = \frac{dx}{dt} - v = u_x - v \\ u'_y = \frac{dy'}{dt'} = \frac{dy}{dt} = u_y \\ u'_z = \frac{dz'}{dt'} = \frac{dz}{dt} = u_z \end{cases}$$

$$\Rightarrow \begin{cases} du'_x = du_x \\ du'_y = du_y \\ du'_z = du_z \end{cases} \Rightarrow \begin{cases} a'_x = \frac{du'_x}{dt'} = \frac{du_x}{dt} = a_x \\ a'_y = \frac{du'_y}{dt'} = \frac{du_y}{dt} = a_y \\ a'_z = \frac{du'_z}{dt'} = \frac{du_z}{dt} = a_z \end{cases}$$



Newton's laws are the same in both reference frames. \Rightarrow Galilean Invariance.

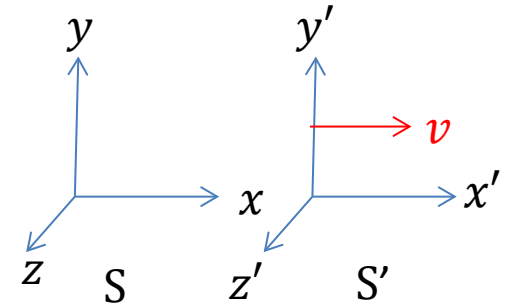
However, the Maxwell's equations do not have Galilean invariance.

($c' \neq c$ under Galilean transformation.)

To fix this problem \rightarrow Lorentz Transformation (H. A. Lorentz)

Lorentz Transformation :

$$\begin{cases} x' = \gamma(x - vt) \\ y' = y \\ z' = z \\ t' = \gamma\left(t - \frac{vx}{c^2}\right) \end{cases}, \text{ where } \begin{cases} \gamma = \frac{1}{\sqrt{1 - \beta^2}} & \text{(Lorentz factor)} \\ \beta = \frac{v}{c} & \text{(Speed parameter)} \end{cases}$$



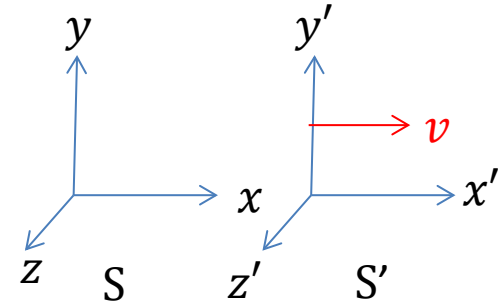
Maxwell's equations are invariant under Lorentz transformation.

Note: If $v \ll c$, then $\beta \simeq 0$, $\gamma \simeq 1$, and $\frac{v}{c^2} \simeq 0$.

$$\Rightarrow \begin{cases} x' = \gamma(x - vt) \simeq x - vt \\ y' = y \\ z' = z \\ t' = \gamma\left(t - \frac{vx}{c^2}\right) \simeq t \end{cases} \rightarrow \text{Galilean Transformation}$$

$$\begin{cases} x' = \gamma(x - vt) \\ y' = y \\ z' = z \\ t' = \gamma(t - \frac{vx}{c^2}) \end{cases} \Rightarrow \begin{cases} dx' = \gamma(dx - vdt) \\ dy' = dy \\ dz' = dz \\ dt' = \gamma(dt - \frac{vdx}{c^2}) \end{cases}$$

$$\Rightarrow \begin{cases} u'_x = \frac{dx'}{dt'} = \frac{\gamma(dx - vdt)}{\gamma(dt - \frac{vdx}{c^2})} = \frac{(\frac{dx}{dt} - v)}{(1 - \frac{v}{c^2} \frac{dx}{dt})} = \frac{(u_x - v)}{(1 - \frac{vu_x}{c^2})} \\ u'_y = \frac{dy'}{dt'} = \frac{dy}{\gamma(dt - \frac{vdx}{c^2})} = \frac{1}{\gamma} \frac{\frac{dy}{dt}}{(1 - \frac{v}{c^2} \frac{dx}{dt})} = \frac{1}{\gamma} \frac{u_y}{(1 - \frac{vu_x}{c^2})} \\ u'_z = \frac{dz'}{dt'} = \frac{dz}{\gamma(dt - \frac{vdx}{c^2})} = \frac{1}{\gamma} \frac{\frac{dz}{dt}}{(1 - \frac{v}{c^2} \frac{dx}{dt})} = \frac{1}{\gamma} \frac{u_z}{(1 - \frac{vu_x}{c^2})} \end{cases}$$



Note: If $u_x = c \Rightarrow$

$$u'_x = \frac{(c - v)}{(1 - \frac{vc}{c^2})} = \frac{(c - v)}{(1 - \frac{v}{c})} = c$$

The speed of the light is
the same in all reference frames!

For events (x_1, y_1, z_1, t_1) and (x_2, y_2, z_2, t_2) ,

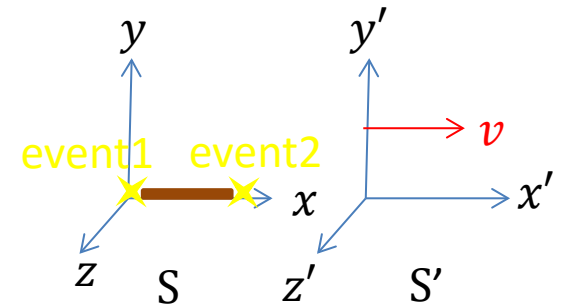
$$\left\{ \begin{array}{l} dx' = \gamma(dx - vdt) \\ dy' = dy \\ dz' = dz \\ dt' = \gamma(dt - \frac{vdx}{c^2}) \end{array} \right. \Rightarrow \left\{ \begin{array}{l} \Delta x' = x'_2 - x'_1 = \int_1^2 dx' = \gamma(\int_1^2 dx - v \int_1^2 dt) = \gamma(\Delta x - v\Delta t) \\ \Delta y' = y'_2 - y'_1 = \int_1^2 dy' = \int_1^2 dy = \Delta y \\ \Delta z' = z'_2 - z'_1 = \int_1^2 dz' = \int_1^2 dz = \Delta z \\ \Delta t' = t'_2 - t'_1 = \int_1^2 dt' = \gamma(\int_1^2 dt - \frac{v}{c^2} \int_1^2 dx) = \gamma(\Delta t - \frac{v}{c^2} \Delta x) \end{array} \right.$$

Simultaneity: $\Delta t = 0$ but $\Delta x \neq 0 \Rightarrow \Delta t' = \gamma(-\frac{v}{c^2} \Delta x) \neq 0$

Time Dilation: If $\Delta x = 0$ and $v \neq 0$, $\gamma = \frac{1}{\sqrt{1 - \frac{v^2}{c^2}}} > 1 \Rightarrow \Delta t' = \gamma \Delta t > \Delta t$

Length Contraction: $\Delta t' = 0 \Rightarrow \Delta t = \frac{v}{c^2} \Delta x$

$$\Rightarrow \Delta x' = \gamma(\Delta x - v\Delta t) = \gamma(\Delta x - \frac{v^2}{c^2} \Delta x) = \gamma(1 - \frac{v^2}{c^2}) \Delta x = \frac{1}{\gamma} \Delta x < \Delta x$$

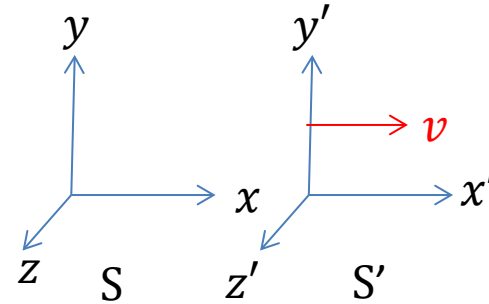


e.g. The length measured by two events is stationary in S . Since it is moving in S' , for $\Delta x'$ to be the length the two events have to be simultaneous. $\Rightarrow \Delta t'$ has to be zero.

Einstein's Postulates

1. The relativity postulate: The law of physics are the same in all inertial reference frames.
2. The speed of light postulate: The speed of light in vacuum has the same value c in all directions and in all inertial reference frames.

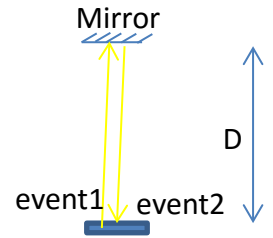
A light source and a mirror is stationary in S' .
 Event 1: a pulse of light leaves the light source.
 Event 2: the pulse is detected at the source.



In S'

The time interval between event 1 and event 2 is $\Delta t' = \frac{2D}{c}$.

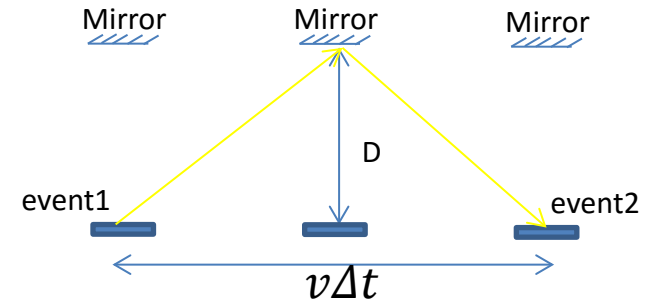
The two events occur at the same location, $\Delta t_0 = \Delta t'$ is called proper time.



In S

The time interval between event 1

and event 2 is $\Delta t = \frac{2[D^2 + (\frac{1}{2}v\Delta t)^2]^{\frac{1}{2}}}{c}$

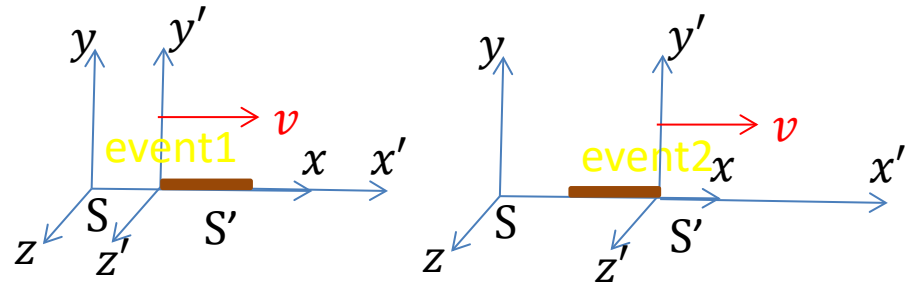


$$\Rightarrow \left(\frac{c^2}{4} - \frac{v^2}{4}\right)(\Delta t)^2 = D^2 = \frac{c^2}{4}(\Delta t')^2 \Rightarrow \Delta t = \frac{\Delta t'}{\sqrt{1 - \frac{v^2}{c^2}}} = \gamma \Delta t' = \gamma \Delta t_0 \text{ (time dilation)}$$

A rod is stationary in S .

Event 1: The origin of S' reaches the left end of the rod.

Event 2: The origin of S' reaches the right end of the rod.



In S

The length of the rod is the distance between the two events $\Delta x = v\Delta t$.

The rod is stationary in S , $\Delta x = L_0$ is called proper length. $\Rightarrow v\Delta t = L_0$

In S'

The length of the rod $L = v\Delta t'$.

Event 1 and event 2 occur at the same location. $\Rightarrow \Delta t'$ is the proper time.

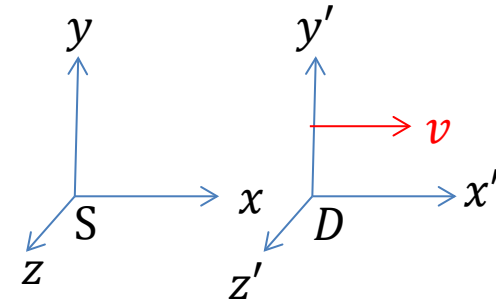
From time dilation, we have $\Delta t = \gamma\Delta t'$. $\Rightarrow L = v\Delta t' = \frac{v\Delta t}{\gamma} = \frac{L_0}{\gamma}$ (length contraction)

Doppler Effect for Light

S: source, D: detector

Event 1: The source S emits the first wavefront.

Event 2: The source S emits the second wavefront..



In S

The period is Δt . \Rightarrow the frequency is $f = \frac{1}{\Delta t}$

Event 1 and event 2 occur at the same location. $\Rightarrow \Delta t$ is the proper time.

In D

The time interval between event 1 and event 2 is $\Delta t' = \gamma \Delta t$. (time dilation)

During the time interval $\Delta t'$, the first wavefront travels a distance $c\Delta t' = \gamma c\Delta t$ towards the detector while the source travels a distance $v\Delta t' = \gamma v\Delta t$ away from the detector. \Rightarrow The wavelength $\lambda' = \gamma(c + v)\Delta t$.

$$\Rightarrow f' = \frac{c}{\lambda'} = \frac{c}{\gamma(c + v)\Delta t} = \frac{1}{\frac{1}{\sqrt{1 - \beta^2}}(1 + \beta)} \frac{1}{\Delta t} = f \sqrt{\frac{1 - \beta}{1 + \beta}}$$

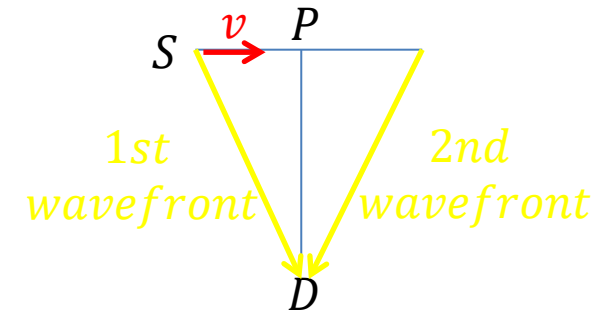
Transverse Doppler Effect (A relativistic effect)

S: source, D: detector

Event 1: The source S emits the first wavefront.

Event 2: The source S emits the second wavefront..

The source S travels in a trajectory perpendicular to \overline{PD} .



In S

The period is Δt . \Rightarrow the frequency is $f = \frac{1}{\Delta t}$

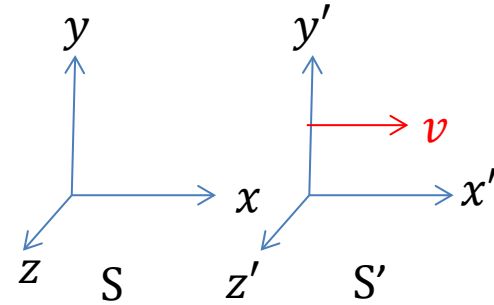
Event 1 and event 2 occur at the same location. $\Rightarrow \Delta t$ is the proper time.

In D

The time interval between event 1 and event 2 is $\Delta t' = \gamma \Delta t$. (time dilation)

$$\Rightarrow f' = \frac{1}{\gamma \Delta t} = \sqrt{1 - \beta^2} \frac{1}{\Delta t} = f \sqrt{1 - \beta^2}$$

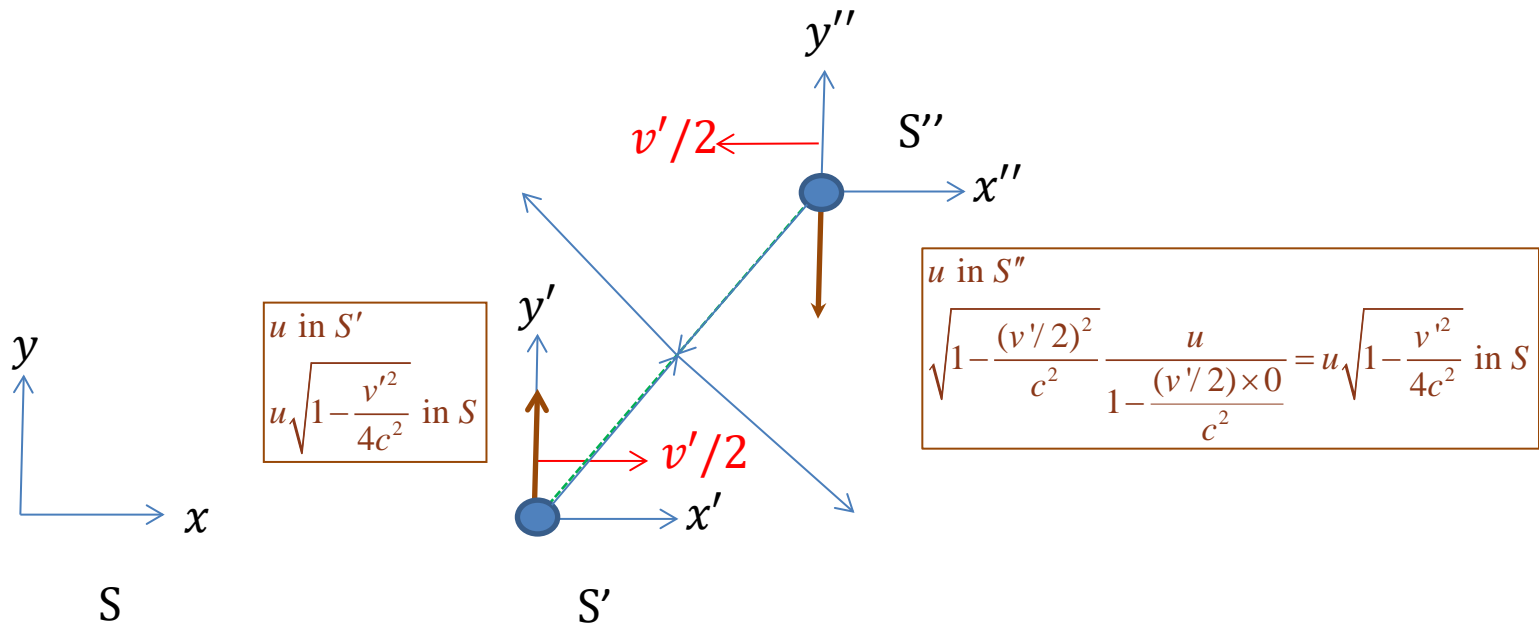
$$u'_x = \frac{(u_x - v)}{(1 - \frac{vu_x}{c^2})} ; u'_y = \sqrt{1 - \frac{v^2}{c^2}} \frac{u_y}{(1 - \frac{vu_x}{c^2})}$$

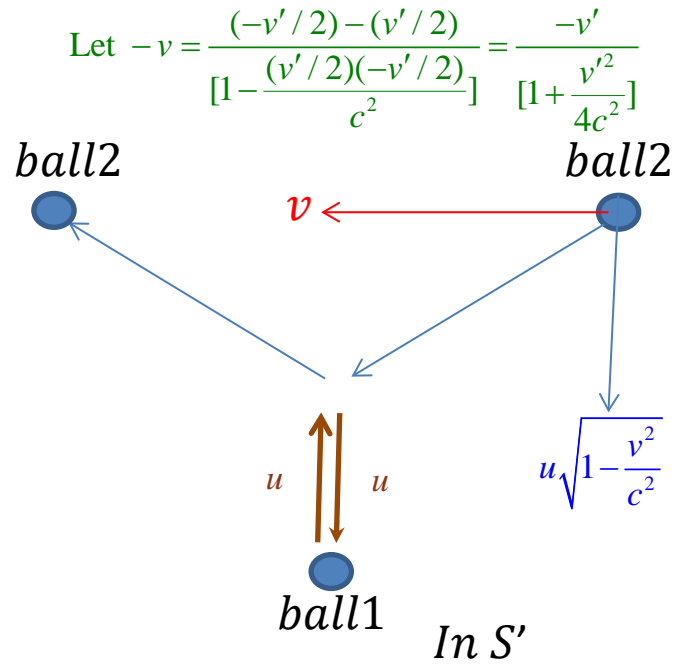
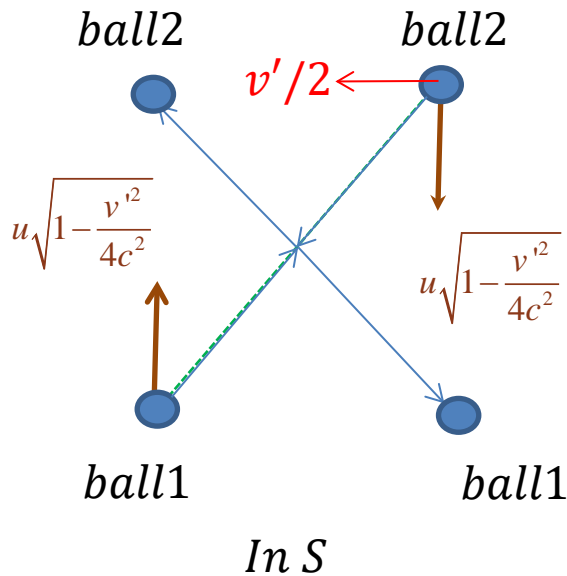


Consider an elastic collision between two balls traveling with a speed

$\frac{1}{2}v'$ towards each other in the x – direction in a stationary reference frame S

and u in the y -direction in their own reference frames S' and S'' , respectively.





$$\text{Let } -v = \frac{(-v'/2) - (v'/2)}{[1 - \frac{(v'/2)(-v'/2)}{c^2}]} = \frac{-v'}{[1 + \frac{v'^2}{4c^2}]}$$

Classical definition of momentum $\vec{p} = m\vec{v}$

1. In S

$$\text{For ball 1: } \Delta p_{1,y} = m(-u\sqrt{1 - \frac{v'^2}{4c^2}}) - mu\sqrt{1 - \frac{v'^2}{4c^2}} = -2mu\sqrt{1 - \frac{v'^2}{4c^2}}$$

$$\text{For ball 2: } \Delta p_{2,y} = mu\sqrt{1 - \frac{v'^2}{4c^2}} - m(-u\sqrt{1 - \frac{v'^2}{4c^2}}) = 2mu\sqrt{1 - \frac{v'^2}{4c^2}}$$

$$\Rightarrow \Delta p_y = \Delta p_{1,y} + \Delta p_{2,y} = 0 \Rightarrow p_y \text{ is conserved.}$$

However

1. In S'

For ball 1: $\Delta p_{1,y} = m(-u) - mu = -2mu$

For ball 2: $\Delta p_{2,y} = mu\sqrt{1 - \frac{v^2}{c^2}} - m(-u\sqrt{1 - \frac{v^2}{c^2}}) = 2mu\sqrt{1 - \frac{v^2}{c^2}}$

$\Rightarrow \Delta p_y = \Delta p_{1,y} + \Delta p_{2,y} = -2mu + 2mu\sqrt{1 - \frac{v^2}{c^2}} \neq 0 \Rightarrow p_y$ is not conserved. (problematic!)

To conserve p_y in S' , re-define the momentum as $\vec{p} = f(v)m\vec{v}$.

We have $\Delta p_{1,y} = -2f(u)mu$ and $\Delta p_{2,y} = 2f(\sqrt{v^2 + (1 - \frac{v^2}{c^2})u^2})mu\sqrt{1 - \frac{v^2}{c^2}}$.

Let $\Delta p_{1,y} + \Delta p_{2,y} = 0 \Rightarrow -2f(u)mu + 2f(\sqrt{v^2 + (1 - \frac{v^2}{c^2})u^2})mu\sqrt{1 - \frac{v^2}{c^2}} = 0$

$\Rightarrow f(u) = f(\sqrt{v^2 + (1 - \frac{v^2}{c^2})u^2})\sqrt{1 - \frac{v^2}{c^2}} \Rightarrow f(0) = f(v)\sqrt{1 - \frac{v^2}{c^2}}$

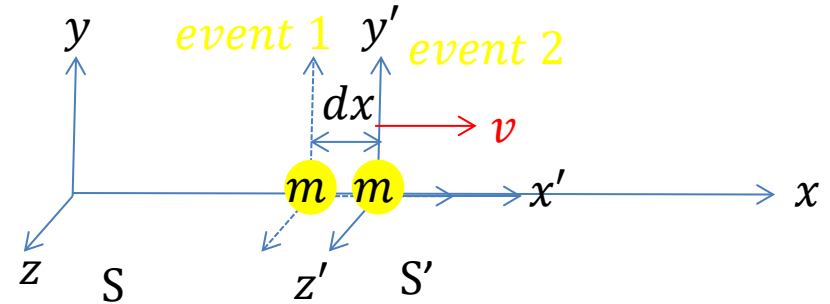
Let $f(0) = 1 \Rightarrow f(v) = \frac{1}{\sqrt{1 - \frac{v^2}{c^2}}} = \gamma \Rightarrow \vec{p} = \gamma m\vec{v}$ (relativistic momentum)

Note: $dx \rightarrow$ proper length; $dt' \rightarrow$ proper time

$$v = \frac{dx}{dt} \quad (\text{ordinary velocity})$$

$$\eta = \frac{dx}{dt'} = \frac{dx}{dt / \gamma} = \gamma \frac{dx}{dt} = \gamma v \quad (\text{proper velocity})$$

Relativistic Momentum $p = \gamma m v = m \gamma v = m \eta$ (mass \times proper velocity)



Relativistic Energy

$$p = \gamma mv$$

$$F = \frac{dp}{dt} \quad [\text{Note: } F = \frac{dp}{dt} = \frac{d}{dt} \left(\frac{1}{\sqrt{1 - \frac{v^2}{c^2}}} mv \right) \neq ma]$$

$$\begin{aligned} \Rightarrow W &= \int_{x_i}^{x_f} F dx = \int_{x_i}^{x_f} \frac{dp}{dt} dx = \int_{p_i}^{p_f} \frac{dx}{dt} dp = \int_{p_i}^{p_f} v dp = \int_{v_i p_i}^{v_f p_f} d(vp) - \int_{v_i}^{v_f} p dv \\ &= [v_f p_f - v_i p_i] - \int_{v_i}^{v_f} \gamma m v dv = [v_f p_f - v_i p_i] - \int_{v_i}^{v_f} \frac{1}{\sqrt{1 - \frac{v^2}{c^2}}} m v dv \\ &= [v_f p_f - v_i p_i] - m \int_{v_i}^{v_f} \frac{2v}{2\sqrt{1 - \frac{v^2}{c^2}}} dv = [v_f p_f - v_i p_i] - m \int_{v_i^2}^{v_f^2} \frac{1}{2\sqrt{1 - \frac{v^2}{c^2}}} d(v^2) \\ &= [v_f p_f - v_i p_i] + mc^2 \left[\sqrt{1 - \frac{v^2}{c^2}} \right]_{v^2=v_i^2}^{v^2=v_f^2} = [v_f p_f - v_i p_i] + mc^2 \left[\sqrt{1 - \frac{v_f^2}{c^2}} - \sqrt{1 - \frac{v_i^2}{c^2}} \right] \end{aligned}$$

Let $v_i = 0$ and $v_f = v \Rightarrow p_i = \gamma_i m v_i = 0$ and $p_f = p = \gamma m v$,

$$\begin{aligned} \text{we have } W &= \gamma m v^2 + mc^2 \left(\frac{1}{\gamma} - 1 \right) = \gamma m c^2 \left(\frac{v^2}{c^2} + \frac{1}{\gamma^2} \right) - mc^2 = \gamma m c^2 \left(\frac{v^2}{c^2} + 1 - \frac{v^2}{c^2} \right) - mc^2 \\ &= \gamma m c^2 - mc^2 \end{aligned}$$

$$W = \gamma mc^2 - mc^2$$

Work-kinetic energy theorem $W = \Delta K$

Since $v_i = 0$, $\Delta K = K \Rightarrow K = \gamma mc^2 - mc^2$

Define mass energy $E_0 = mc^2$

If the potential energy $U = 0$, the total energy $E = K + E_0 = \gamma mc^2 \Rightarrow E = \gamma mc^2$

$$E^2 = \gamma^2 m^2 c^4 = m^2 c^4 \left(\frac{1}{1 - \frac{v^2}{c^2}} \right) = m^2 c^4 \left(\frac{c^2}{c^2 - v^2} \right) = m^2 c^4 \left(\frac{c^2 - v^2}{c^2 - v^2} + \frac{v^2}{c^2 - v^2} \right) = m^2 c^4 \left(1 + \frac{v^2}{c^2 - v^2} \right)$$

$$= m^2 c^4 + c^2 m^2 \left(\frac{v^2}{1 - \frac{v^2}{c^2}} \right) = m^2 c^4 + c^2 m^2 v^2 \left(\frac{1}{1 - \frac{v^2}{c^2}} \right) = m^2 c^4 + c^2 (\gamma^2 m^2 v^2) = m^2 c^4 + c^2 p^2$$

$$\Rightarrow E^2 = c^2 p^2 + m^2 c^4$$

Note:

Recall Taylor's series $f(x) = \sum_{n=0}^{\infty} \frac{1}{n!} \frac{d^n f(x)}{dx^n} \Big|_{x=x_0} (x-x_0)^n$

$$\gamma = \frac{1}{\sqrt{1 - \frac{v^2}{c^2}}}; \text{ Let } x \rightarrow \frac{v^2}{c^2}, x_0 \rightarrow 0, \text{ we have}$$

$$\gamma = \frac{1}{\sqrt{1 - \frac{v^2}{c^2}}} = \left(1 - \frac{v^2}{c^2}\right)^{-\frac{1}{2}} = (1-x)^{-\frac{1}{2}}$$

$$= \frac{1}{0!} [(1-x)^{-\frac{1}{2}}]_{x=0} (x-0)^0 + \frac{1}{1!} \left[\left(-\frac{1}{2}\right) (1-x)^{-\frac{3}{2}} (-1) \right]_{x=0} (x-0)^1$$

$$+ \frac{1}{2!} \left[\left(-\frac{1}{2}\right) \left(-\frac{3}{2}\right) (1-x)^{-\frac{5}{2}} (-1)(-1) \right]_{x=0} (x-0)^2 + \dots = 1 + \frac{1}{2}x + \frac{3}{8}x^2 + \dots = 1 + \frac{1}{2} \frac{v^2}{c^2} + \frac{3}{8} \left(\frac{v^2}{c^2}\right)^2 + \dots$$

$$\gamma = \frac{1}{\sqrt{1 - \frac{v^2}{c^2}}} = 1 + \frac{1}{2} \frac{v^2}{c^2} + \frac{3}{8} \left(\frac{v^2}{c^2}\right)^2 + \dots$$

If $v \ll c$, then

$$\left\{ \begin{array}{l} x' = \gamma(x - vt) = \left[1 + \frac{1}{2} \frac{v^2}{c^2} + \frac{3}{8} \left(\frac{v^2}{c^2}\right)^2 + \dots\right](x - vt) \simeq x - vt \\ y' = y \\ z' = z \\ t' = \gamma\left(t - \frac{vx}{c^2}\right) = \left[1 + \frac{1}{2} \frac{v^2}{c^2} + \frac{3}{8} \left(\frac{v^2}{c^2}\right)^2 + \dots\right]\left(t - \frac{vx}{c^2}\right) \simeq t \end{array} \right.$$

$$\vec{p} = \gamma m \vec{v} = \left[1 + \frac{1}{2} \frac{v^2}{c^2} + \frac{3}{8} \left(\frac{v^2}{c^2}\right)^2 + \dots\right] m \vec{v} \simeq m \vec{v}$$

$$K = \gamma mc^2 - mc^2 = \left(1 + \frac{1}{2} \frac{v^2}{c^2} + \frac{3}{8} \left(\frac{v^2}{c^2}\right)^2 + \dots - 1\right) mc^2 \simeq \frac{1}{2} \frac{v^2}{c^2} mc^2 = \frac{1}{2} mv^2 .$$

Chapters 18-20 Thermodynamics and Statistical Mechanics

Consider a system of a large number of particles:

Microscopic states or **Microstates** $\rightarrow (\vec{r}_1, \vec{v}_1, \vec{r}_2, \vec{v}_2 \cdots \vec{r}_N, \vec{v}_N)$

Calculating the dynamics of such systems is a formidable task.



an alternative approach

Thermodynamics: a phenomenological theory directly drawn from experiments.

Macroscopic states or **Macrostates** \rightarrow specified by a set of state variables or state parameters.

Note:

Equations of states and **thermodynamic laws** reduce the number of independent state variables.

A set of independent **state variables** can be selected to uniquely specify the macrostate.

Other state variables are treated as **state functions** of the selected independent state variables.

Example: Consider an ideal gas.

State variables $\rightarrow P, V, E_{\text{int}}, T, S(\text{entropy}), H(\text{enthalpy}), A(\text{free energy}), G(\text{Gibbs Potential}) \cdots$

Equations of states: $PV = nRT, H = E_{\text{int}} + PV, A = E_{\text{int}} - TS, G = A + PV \cdots$

Thermodynamic laws: $dE_{\text{int}} = TdS - PdV$ (in reversible processes); $S \rightarrow 0$ as $T \rightarrow 0$

State functions: $E_{\text{int}}(P, V), T(P, V), S(P, V), H(P, V), A(P, V), G(P, V) \cdots$

Statistical Mechanics:

Using probability to connect the macroscopic theory of thermodynamics with microscopic mechanical theory.

In thermodynamics, a macroscopic physical quantity, temperature T , and a temperature-related energy term, heat Q are introduced.

Other temperature-related state functions are also defined.

Constant-volume gas thermometer

$$T = (273.16K) \lim_{gas \rightarrow 0} \frac{P}{P_3}$$

.

Note: Temperature and Heat

Temperature scale: 1. Kelvin (K) $T = (273.16K) \lim_{gas \rightarrow 0} \frac{P}{P_3}$;

2. Celsius ($^{\circ}C$) $T_C = T - 273.15$;

3. Fahrenheit ($^{\circ}F$) $T_F = \frac{9}{5}T_C + 32^{\circ}$

Thermal Expansion

$\frac{dL}{L} = \alpha dT$; L : length, dL : length increase due to temperature increase dT .

$\Rightarrow \alpha$: linear expansion coefficient.

$\frac{dV}{V} = \beta dT$; V : volume, dV : volume increase due to temperature increase dT .

$\Rightarrow \beta$: volume expansion coefficient.

$$\frac{dV}{V} = \frac{(L + dL)^3 - L^3}{L^3} = \frac{L^3 + 3L^2 dL + 3L(dL)^2 + (dL)^3 - L^3}{L^3} = \frac{3L^2 dL + O((dL)^2)}{L^3}$$

$$= 3 \frac{dL}{L} \Rightarrow \beta dT = 3\alpha dT \Rightarrow \beta = 3\alpha$$

Heat Q : energy transferred because of temperature difference

1 cal (calories) = $3.969 \times 10^{-3} \text{ Btu}$ (British thermal unit) = 4.1868 J

Heat Capacity C : $Q = C\Delta T = C(T_f - T_i)$

Specific heat c : $Q = cm(T_f - T_i)$, where m is the mass $\Rightarrow c = \frac{C}{m}$

Molar specific heat: heat capacity per mole (6.02×10^{23})

c_v molar specific heat at constant volume (no work done)

c_p molar specific heat at constant pressure (larger than c_v)



to compensate energy outflow through work)

Heat of Transformation L : $Q = Lm$, $Q \rightarrow$ the heat required to transform the material between physical states.

L_v : heat of vaporization, L_f : heat of fusion

Heat Transfer Mechanism

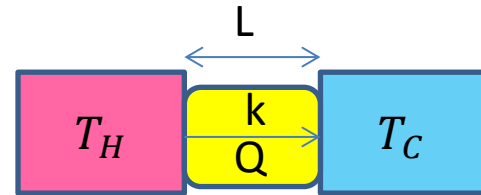
1. Conduction

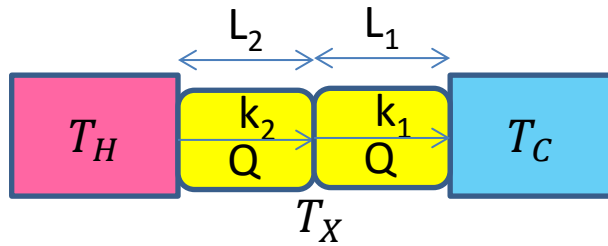
Consider a slab of solid material between a hot reservoir T_H and a cold reservoir T_C .

The conduction rate $P_{cond} = \frac{Q}{t} = kA \frac{T_H - T_C}{L}$;

Q : heat transferred through the slab in time t ; A : area of the slab; L : thickness of the slab

k : thermal conductivity \Rightarrow thermal resistance $R = \frac{L}{k}$





$$P_{cond,1} = P_{cond,2} = P_{cond} \Rightarrow k_2 A \frac{T_H - T_X}{L_2} = k_1 A \frac{T_X - T_C}{L_1} \Rightarrow T_X = \frac{k_1 L_2 T_C + k_2 L_1 T_H}{k_1 L_2 + k_2 L_1}$$

$$P_{cond} = \frac{A(T_H - T_C)}{\frac{L_1}{k_1} + \frac{L_2}{k_2}} \text{ can be generalized to } \frac{A(T_H - T_C)}{\sum_i \frac{L_i}{k_i}}$$

2. Convection: Diffusion of hot particle into cold region of a fluid.

3. Radiation

$P_{rad} = \sigma \varepsilon A T^4$; σ : Stefan-Boltzmann constant, ε : emissivity (0~1, 1 for black body)

$$P_{abs} = \sigma \varepsilon A T_{env}^4 \Rightarrow P_{net} = P_{abs} - P_{rad} = \sigma \varepsilon A (T_{env}^4 - T^4)$$

Laws of Thermodynamics

1. The zeroth law of thermodynamics:

If bodies A and T are in thermal equilibrium and B and T are in thermal equilibrium, then A and B are in thermal equilibrium.

2. The first law of thermodynamics:

$$dE_{\text{int}} = dQ - dW \text{ (conservation of energy including heat)}$$

dE_{int} : internal energy increase of a system

dQ : heat supplied to the system

dW : work done by the system

Q : heat \rightarrow energy transferred between a system and its environment
because of temperature difference between them.

$$\Delta E_{\text{int}} = Q - W$$

3. The second law of thermodynamics:

The increase of entropy of a closed system $\Delta S \geq 0$ for all thermodynamic processes.

$$\text{Entropy increase } \Delta S = S_f - S_i = \int_i^f dS = \int_i^f \frac{dQ}{T} \text{ (for reversible processes } dQ = TdS)$$

4. The third law of thermodynamics:

$$S \rightarrow 0 \text{ as } T \rightarrow 0.$$

Thermodynamic transformations: changes of thermodynamic states (Macrostates).

Processes of thermodynamical transformations of special interest for a system of gas:

Note: $dE_{\text{int}} = dQ - dW$

$dW = Fdl = PAdl = PdV (\Leftarrow F = PA, dV = Adl)$ for a system of gas.

$dQ = TdS$ for a reversible process.

1. Adiabatic processes: $dQ = 0 \Rightarrow dE_{\text{int}} = -dW = -PdV$
2. Isothermal processes: T is a constant $\Rightarrow Q = \int_i^f dQ = T \int_i^f dS = T(S_f - S_i) = T\Delta S$
3. Constant-volume processes: $dV = 0 \Rightarrow dW = PdV = 0 \Rightarrow dE_{\text{int}} = dQ$
 $\Rightarrow dE_{\text{int}} = dQ = TdS$ (reversible processes)
4. Constant-pressure processes: P is a constant $\Rightarrow W = \int_i^f dW = P \int_i^f dV = P(V_f - V_i) = P\Delta V$
5. Cyclical processes: $\Delta E_{\text{int}} = 0 \Rightarrow Q = W$ (e.g. engine or refrigerator cycles)
6. Free expansions: $dQ = dW = dE_{\text{int}} = 0$

Ideal Gas: equation of states $PV = nRT$

Work done by an ideal gas

1. in an isothermal process (T is a constant)

$$P(V) = nRT \frac{1}{V} \Rightarrow dW = PdV = nRT \frac{1}{V} dV \Rightarrow W = \int_{V_i}^{V_f} nRT \frac{1}{V} dV = nRT [\ln V]_{V_i}^{V_f} = nRT \ln \frac{V_f}{V_i}$$

$$W = nRT \ln \frac{V_f}{V_i}$$

2. in a constant-volume process (V is a constant $\Rightarrow dV = 0$)

$$dW = PdV = 0 \Rightarrow W = 0$$

3. in a constant pressure process (P is a constant)

$$W = \int_{V_i}^{V_f} PdV = P \int_{V_i}^{V_f} dV = P(V_f - V_i)$$

$$W = P(V_f - V_i)$$

Ideal Gas: equation of states $PV = nRT \Rightarrow VdP + PdV = nRdT$

Note: It can be proven later from the kinetic theory of gases that E_{int} is a function of T only ($E_{\text{int}} = \frac{3}{2}nRT \Rightarrow dE_{\text{int}} = \frac{3}{2}nRdT$). The increase of internal energy dE_{int} depends only on the increase of temperature dT .

For the molar specific heat at constant volume c_V ,

$$\begin{cases} (dQ)_V = nc_V dT \\ \text{constant volume} \Rightarrow dV = 0 \Rightarrow dE_{\text{int}} = (dQ)_V - PdV = (dQ)_V \end{cases} \Rightarrow dE_{\text{int}} = nc_V dT$$

For the molar specific heat at constant pressure c_P ,

$$\begin{cases} (dQ)_P = nc_P dT \\ \text{constant pressure} \Rightarrow dP = 0 \Rightarrow nRdT = VdP + PdV = PdV \\ dE_{\text{int}} = (dQ)_P - PdV = nc_P dT - nRdT = n(c_P - R)dT \end{cases} \Rightarrow dE_{\text{int}} = n(c_P - R)dT$$

$$\Rightarrow c_P = c_V + R$$

$$dE_{\text{int}} = nc_V dT = \frac{3}{2}nRdT \Rightarrow c_V = \frac{3}{2}R; \quad c_P = c_V + R = \frac{5}{2}R \quad (\text{for monoatomic gases})$$

Adiabatic expansion of an ideal gas

$$\begin{cases} \text{adiabatic} \Rightarrow dQ = 0 \Rightarrow dE_{\text{int}} = dQ - PdV = -PdV \\ dE_{\text{int}} = nc_V dT \end{cases} \Rightarrow ndT = -\left(\frac{P}{c_V}\right)dV$$

$$\text{recall } \begin{cases} c_P = c_V + R \\ VdP + PdV = nRdT \end{cases} \Rightarrow VdP + PdV = n(c_P - c_V)dT \Rightarrow ndT = \frac{VdP + PdV}{c_P - c_V}$$

$$\Rightarrow \frac{VdP + PdV}{c_P - c_V} + \frac{PdV}{c_V} = 0 \Rightarrow c_V VdP + (c_V + c_P - c_V)PdV = 0 \Rightarrow \frac{dP}{P} + \left(\frac{c_P}{c_V}\right)\frac{dV}{V} = 0$$

$$\text{Let } \gamma = \frac{c_P}{c_V} \Rightarrow \frac{dP}{P} = -\gamma \frac{dV}{V} \Rightarrow \ln P = -\gamma \ln V + C' \Rightarrow P = e^{C'} \exp(-\gamma \ln V) = C'' V^{-\gamma}$$

$$\Rightarrow PV^\gamma = \text{a constant.}$$

$$PV = nRT \Rightarrow PV^\gamma = PVV^{\gamma-1} = nRTV^{\gamma-1} = \text{a constant} \Rightarrow TV^{\gamma-1} = \text{a constant}$$

$$\Rightarrow \text{For an adiabatic process } \begin{cases} PV^\gamma = \text{a constant} \\ TV^{\gamma-1} = \text{a constant} \end{cases}$$

Entropy change from state (V_i, T_i) to state (V_f, T_f) for an ideal gas

Note: The entropy S is a state function $S(V, T)$.

Therefore $\Delta S = S_f - S_i$ is independent of the process the transformation takes.

We can always select a reversible process where $dQ = TdS$ for calculating ΔS .

$$\left\{ \begin{array}{l} \text{For an ideal gas, } PV = nRT \Rightarrow P = \frac{nRT}{V} \\ \text{The 1st law of thermodynamics } dE_{\text{int}} = TdS - PdV = TdS - \frac{nRT}{V}dV \\ \text{Recall } dE_{\text{int}} = nc_V dT \end{array} \right.$$

$$\Rightarrow nc_V dT = TdS - \frac{nRT}{V}dV \Rightarrow dS = nc_V \frac{dT}{T} + nR \frac{dV}{V} \Rightarrow \int_i^f dS = \int_i^f nc_V \frac{dT}{T} + \int_i^f nR \frac{dV}{V}$$

$$\Rightarrow \Delta S = S_f - S_i = nc_V \ln \frac{T_f}{T_i} + nR \ln \frac{V_f}{V_i}$$

Example: A free expansion of an ideal gas from volume V to $3V$.

For a free expansion, we have $T_f = T_i$. We first calculate ΔS for a reversible

isothermal expansion from (V, T) to $(3V, T)$. $\Delta S = nc_V \ln \frac{T}{T} + nR \ln \frac{3V}{V} = nR \ln 3$

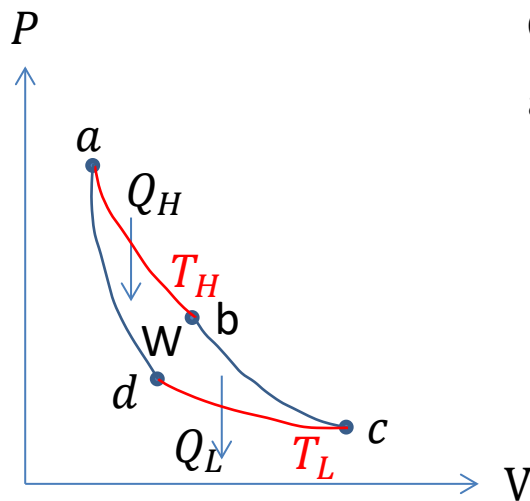
S is a state function $\Rightarrow \Delta S = nR \ln 3$ is also valid for the irreversible free expansion process.

Engines and Refrigerators: Cyclical processes

A heat engine (engine): a device that extracts energy from its environment
in the form of heat and does useful work.

An ideal engine: all processes are reversible; no waste of energy due to friction,
and turbulence etc.

Carnot engine: an ideal engine with a cycle composed of two isothermal processes
and two adiabatic processes.



Carnot cycle

$a \rightarrow b \rightarrow c \rightarrow d \rightarrow a$

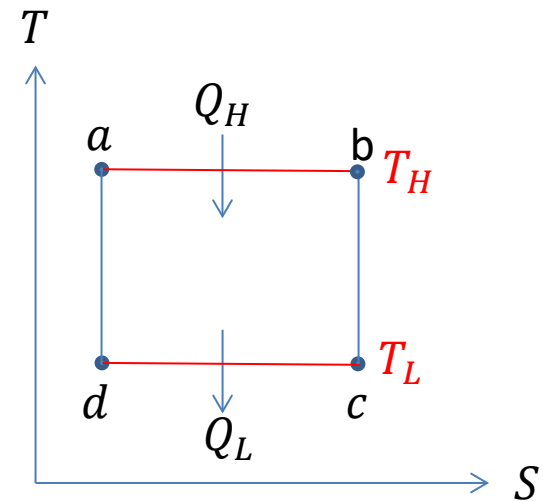
ab, cd : isothermal

$dT = 0 \Rightarrow \text{constant } T$

bc, da : adiabatic

$dQ = TdS = 0 \Rightarrow dS = 0$

$\Rightarrow \text{constant } S$



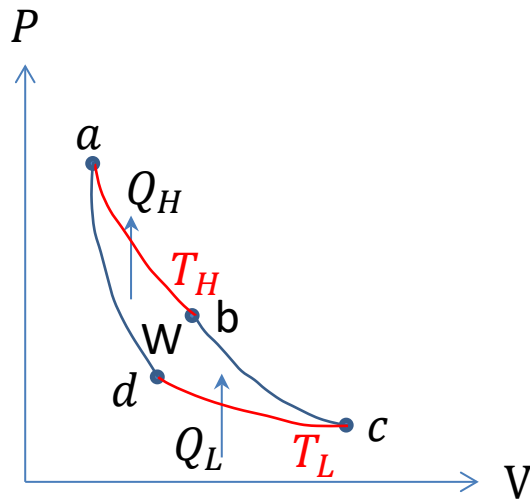
$$W = |Q_H| - |Q_L|; \Delta S = \Delta S_H + \Delta S_L = \frac{|Q_H|}{T_H} - \frac{|Q_L|}{T_L} = 0 \text{ (for a cycle } \Delta S = 0) \Rightarrow |Q_H| = \frac{T_H}{T_L} |Q_L| > |Q_L|$$

$$\text{Efficiency } \varepsilon = \frac{|W|}{|Q_H|} = \frac{|Q_H| - |Q_L|}{|Q_H|} = 1 - \frac{|Q_L|}{|Q_H|} = 1 - \frac{T_L}{T_H} < 1 \text{ (there is no perfect engine } \rightarrow \varepsilon = 1)$$

A refrigerator: a device that use work to transfer heat from a low-temperature reservoir to a high-temperature reservoir.

An ideal refrigerator: all processes are reversible; no waste of energy due to friction, and turbulence etc.

Carnot refrigerator: an ideal refrigerator with a cycle composed of two isothermal processes and two adiabatic processes.



Carnot cycle

$a \rightarrow d \rightarrow c \rightarrow b \rightarrow a$

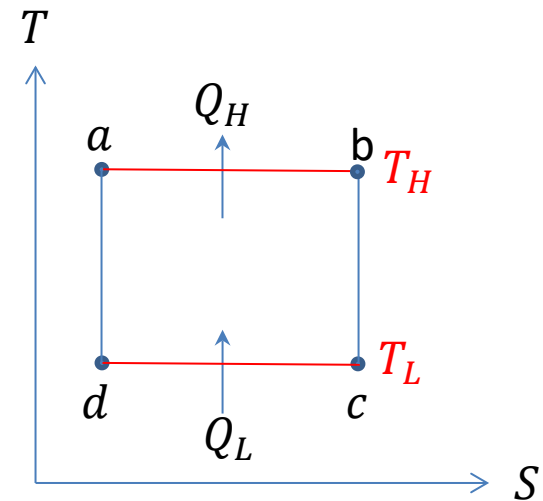
ba, dc : isothermal

$dT = 0 \Rightarrow \text{constant } T$

cb, ad : adiabatic

$dQ = TdS = 0 \Rightarrow dS = 0$

$\Rightarrow \text{constant } S$



$$|W| = |Q_H| - |Q_L|; \Delta S = \Delta S_L + \Delta S_H = \frac{|Q_L|}{T_L} - \frac{|Q_H|}{T_H} = 0 \text{ (for a cycle } \Delta S = 0) \Rightarrow \frac{|Q_L|}{T_L} = \frac{|Q_H|}{T_H}$$

$$\text{Coefficient of performance } K = \frac{|Q_L|}{|W|} = \frac{|Q_L|}{|Q_H| - |Q_L|} = \frac{|Q_L|}{(T_H / T_L)|Q_L| - |Q_L|} = \frac{T_L}{T_H - T_L}$$

Is there a perfect refrigerator that $W = 0$?

Consider an ideal refrigerator. $\Rightarrow |W| = |Q_H| - |Q_L|$

$$W = 0 \Rightarrow |Q_H| = |Q_L|$$

ΔS of the closed system (Hi-T reservoir+Lo-T reservoir+Working substance):

$$\text{Hi-T reservoir} \rightarrow \frac{|Q_H|}{T_H}$$

$$\text{Lo-T reservoir} \rightarrow -\frac{|Q_L|}{T_L}$$

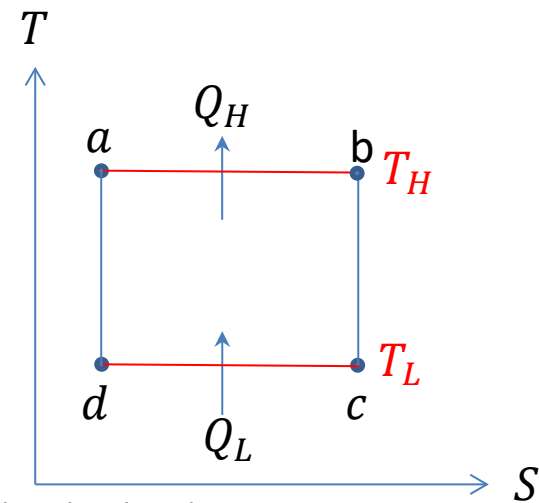
Working substance $\rightarrow 0$ (cyclical)

$$\Rightarrow \Delta S = \frac{|Q_H|}{T_H} - \frac{|Q_L|}{T_L} + 0 = \frac{|Q_H|}{T_H} - \frac{|Q_L|}{T_L}$$

$$\text{Since } |Q_H| = |Q_L| \text{ and } T_H > T_L, \text{ we have } \Delta S = \frac{|Q_L|}{T_H} - \frac{|Q_L|}{T_L} < 0$$

$\Delta S < 0$ violates the 2nd law of thermodynamics

\Rightarrow **No perfect refrigerators.**



Can an engine X with efficiency ε_X greater than that of the Carnot engine ε_C exist?

Consider that engine X operates between high-temperature reservoir T_H and low-temperature reservoir T_L .

Presumably, engine X has to be an ideal engine. $\Rightarrow |W_X| = |Q_{H,X}| - |Q_{L,X}|$ and $\varepsilon_X = \frac{|W_X|}{|Q_{H,X}|}$

Consider a Carnot refrigerator working between the same reservoirs T_H and T_L .

$\Rightarrow |W_C| = |Q_{H,C}| - |Q_{L,C}|$. We have $\varepsilon_C = \frac{|W_C|}{|Q_{H,C}|}$ for its corresponding Carnot engine.

Now, couple engine X to the Carnot refrigerator such that W_X is used to drive

the Carnot refrigerator. $\Rightarrow W_C = W_X \Rightarrow |Q_{H,C}| - |Q_{L,C}| = |Q_{H,X}| - |Q_{L,X}|$

\Rightarrow The net heat extracted by the combined device from T_L , $|Q_{L,C}| - |Q_{L,X}|$, is equal to the net heat flowing into T_H from the combined device, $|Q_{H,C}| - |Q_{H,X}|$.

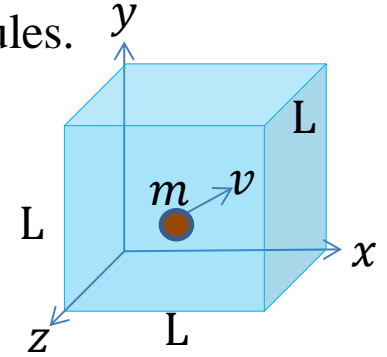
If $\varepsilon_X > \varepsilon_C$ then $\frac{|W_X|}{|Q_{H,X}|} = \frac{|W_C|}{|Q_{H,X}|} > \frac{|W_C|}{|Q_{H,C}|} \Rightarrow |Q_{H,C}| > |Q_{H,X}| \Rightarrow |Q_{L,C}| - |Q_{L,X}| = |Q_{H,C}| - |Q_{H,X}| > 0$

\Rightarrow The combined device is a perfect refrigerator that violate the 2nd law of thermodynamics.

\Rightarrow **Engine X cannot exist.** No real engine can have efficiency greater than that of a Carnot engine working between the same T_H and T_L .

The kinetic theory of gases: To express macroscopic thermodynamic quantities in terms of microscopic quantities of motion of molecules.

Consider a cubic container of side length L filled with an ideal gas.



For a molecule of mass m moving with a velocity of $\vec{v} = v_x \hat{i} + v_y \hat{j} + v_z \hat{k}$, the time interval between two consecutive collisions on a wall perpendicular to the x – axis is $\Delta t = \frac{2L}{|v_x|}$.

The momentum transferred to the wall in one collision is $\Delta p_x = 2m|v_x|$.

On average, the force exerted on the wall by that molecule is $F_x = \frac{\Delta p_x}{\Delta t} = \frac{mv_x^2}{L}$

On average, the pressure exerted on the wall by that molecule is $\frac{F_x}{L^2} = \frac{mv_x^2}{L^3} = \frac{mv_x^2}{V}$

For a system of N ideal gas particles the pressure $P = N \frac{m \langle v_x^2 \rangle}{V}$

Noting that $\langle v^2 \rangle = \langle v_x^2 + v_y^2 + v_z^2 \rangle = \langle v_x^2 \rangle + \langle v_y^2 \rangle + \langle v_z^2 \rangle$ and, on average $\langle v_x^2 \rangle = \langle v_y^2 \rangle = \langle v_z^2 \rangle$

$$\Rightarrow \langle v_x^2 \rangle = \frac{1}{3} \langle v^2 \rangle = \frac{1}{3} v_{rms}^2, \text{ where rms stands for root-mean-square.}$$

$$\Rightarrow P = \frac{Nmv_{rms}^2}{3V} = \frac{nMv_{rms}^2}{3V}, \text{ } n: \text{ number of moles, } M: \text{ molecular mass}$$

$$P = \frac{nMv_{rms}^2}{3V} \Rightarrow PV = \frac{nMv_{rms}^2}{3} = nRT \Rightarrow T = \frac{Mv_{rms}^2}{3R} \Rightarrow v_{rms} = \sqrt{\frac{3RT}{M}}$$

$$\begin{aligned} \text{The average translational kinetic energy } K_{avg.} &= \frac{1}{2}mv_{rms}^2 = \frac{1}{2}m\frac{3RT}{M} \\ &= \frac{3}{2}\frac{mRT}{M} = \frac{3}{2}\frac{mN_A kT}{M} = \frac{3}{2}kT \end{aligned}$$

$$E_{int} = NK_{avg.} = \frac{3}{2}NkT = \frac{3}{2}nN_A kT = \frac{3}{2}nRT$$

In summary:

$$P = \frac{nMv_{rms}^2}{3V}; \quad T = \frac{Mv_{rms}^2}{3R}; \quad E_{int} = \frac{3}{2}nRT$$

$$\text{Also note } v_{rms} = \sqrt{\frac{3RT}{M}}$$

The mean free path: The average distance traversed by a molecule between collisions.

$$\text{mean free path } \lambda = \frac{\text{average length of path traversed during } \Delta t}{\text{average number of collisions in } \Delta t}$$

$$\text{averaged length of path traversed during } \Delta t \Rightarrow v_{rms} \Delta t$$

$$\text{number of collisions in } \Delta t = \text{number of molecules within the path of cross section } \pi d^2$$

However, the target molecules are also moving. Therefore, the average relative velocity $v_{rel,rms}$ should be used to calculate the number of collisions in Δt .

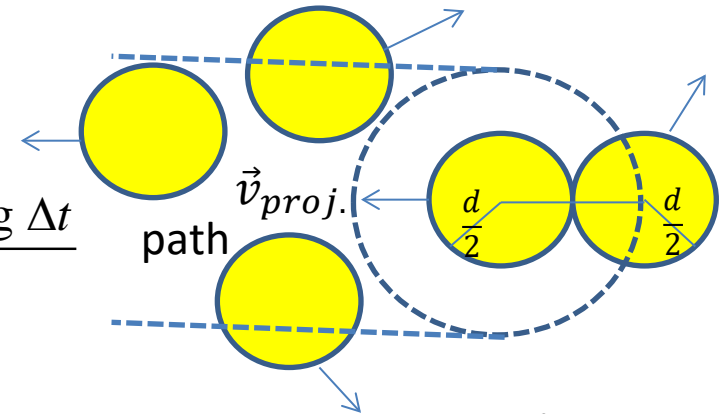
$$\text{number of collisions in } \Delta t \text{ is } [\pi d^2 \times (v_{rel,rms} \Delta t)] \times \frac{N}{V}$$

$$\text{Noting that } v_{rel,rms} = \sqrt{\langle v_{rel}^2 \rangle} = \sqrt{\langle (\vec{v}_{proj.} - \vec{v}_{target})^2 \rangle} = \sqrt{\langle v_{proj.}^2 \rangle + \langle v_{target}^2 \rangle + 2 \langle \vec{v}_{proj.} \cdot \vec{v}_{target} \rangle}$$

$$\langle v_{proj.}^2 \rangle = \langle v_{target}^2 \rangle = \langle v^2 \rangle \text{ and } \langle \vec{v}_{proj.} \cdot \vec{v}_{target} \rangle = 0 \text{ for large } N$$

$$v_{rel,rms} = \sqrt{2 \langle v^2 \rangle} = \sqrt{2} v_{rms}$$

$$\lambda = \frac{v_{rms} \Delta t}{[\pi d^2 \times (\sqrt{2} v_{rms} \Delta t)] \times \frac{N}{V}} = \frac{V}{\sqrt{2} \pi d^2 N}$$



Statistical Mechanics

The equal a priori probability postulate : If a system is in a specific macrostate, the system can be found with equal probability in any microstate consistent with the given macroscopic conditions.

Consider a thermodynamic system of a total energy E and N particles distributed in a finite region in the 6-dimensional (\vec{r}, \vec{p}) space. Divide such a finite region into K equal cells and distribute the N particles among these K cells.

The number of ways to assign n_i particles in the i th cell for $i = 1, 2, \dots, K$ is $\frac{N!}{n_1! n_2! \dots n_K!}$.

By the equal a priori probability postulate, to find the most probable distribution is

to maximize $\frac{N!}{n_1! n_2! \dots n_K!}$ as a function of n_1, n_2, \dots , and n_K .

On the other hand, since \ln is a monotonically increasing function, maximizing $\frac{N!}{n_1! n_2! \dots n_K!}$

is equivalent to maximizing $\ln\left(\frac{N!}{n_1! n_2! \dots n_K!}\right)$, which can be largely simplified by using the

Stirling's approximation $\ln n_i! \approx n_i \ln n_i - n_i$ for large n_i .

$$\Rightarrow \ln\left(\frac{N!}{n_1! n_2! \dots n_K!}\right) = \ln N! - \sum_{i=1}^K \ln n_i! \approx N \ln N - N - \sum_{i=1}^K (n_i \ln n_i - n_i) = N \ln N - \sum_{i=1}^K n_i \ln n_i$$

To maximize $\ln\left(\frac{N!}{n_1!n_2!\cdots n_K!}\right)$ under the constraints $\sum_{i=1}^K n_i = N$ and $\sum_{i=1}^K n_i E_i = E$

we use the variational method with Lagrange multipliers α, β

$$\delta\left[\ln\left(\frac{N!}{n_1!n_2!\cdots n_K!}\right)\right] - \delta\left[\alpha\left(\sum_{i=1}^K n_i - N\right) + \beta\left(\sum_{i=1}^K n_i E_i - E\right)\right] = 0$$

$$\Rightarrow \delta\left(N \ln N - \sum_{i=1}^K n_i \ln n_i\right) - \delta\left[\alpha\left(\sum_{i=1}^K n_i - N\right) + \beta\left(\sum_{i=1}^K n_i E_i - E\right)\right] = 0$$

$$\Rightarrow -\sum_{i=1}^K \left(\ln n_i + n_i \frac{1}{n_i}\right) \delta n_i - \left(\alpha \sum_{i=1}^K \delta n_i + \beta \sum_{i=1}^K E_i \delta n_i\right) = 0$$

$$\Rightarrow -\sum_{i=1}^K (\ln n_i + 1 + \alpha + \beta E_i) \delta n_i = 0$$

$$\Rightarrow \ln n_i + 1 + \alpha + \beta E_i = 0 \Rightarrow \ln n_i = -(1 + \alpha) - \beta E_i \Rightarrow n_i = e^{-(1+\alpha)} e^{-\beta E_i}$$

$$\text{Let } A = e^{-(1+\alpha)} \Rightarrow n_i = A e^{-\beta E_i}$$

The most probable distribution function $f(\vec{r}_i, \vec{p}_i) \propto e^{-\beta E_i}$

For a system of non-interacting particles of the same mass m in thermal equilibrium, the $f(\vec{r}_i, \vec{p}_i)$ is homogeneous and therefore can be replaced by $f(\vec{v}_i)$.

$$\text{and } E_i = K_i = \frac{1}{2} m v_i^2 \Rightarrow f(\vec{v}_i) \propto \exp\left(-\frac{\beta}{2} m v_i^2\right)$$

Note that the distribution function depends on v_i^2 and is therefore isotropic.

Let the speed be continuous $v_i \rightarrow v$ and include all possible speed from 0 to ∞ .

$$\text{To normalize } f(\vec{v}) = C \exp\left(-\frac{\beta}{2}mv^2\right) \Rightarrow \int_0^\infty f(\vec{v})(4\pi v^2 dv) = \int_0^\infty f(v)dv = 1$$

$$\Rightarrow 4\pi C \int_0^\infty v^2 \exp\left(-\frac{\beta}{2}mv^2\right) dv = 1 ; \text{ Note } f(v) = 4\pi v^2 f(\vec{v})$$

$$\Rightarrow 4\pi C \left(\frac{\sqrt{\pi} \operatorname{erf}\left(\sqrt{\frac{\beta m}{2}}v\right)}{4\left(\sqrt{\frac{\beta m}{2}}\right)^3} - \frac{v \exp\left(-\frac{\beta m}{2}v^2\right)}{\beta m} \right) \Bigg|_{v=0}^{v=\infty} = 1 \Rightarrow 4\pi C \frac{\sqrt{\pi}}{4\left(\sqrt{\frac{\beta m}{2}}\right)^3} = 1 \Rightarrow C = \left(\sqrt{\frac{\beta m}{2\pi}}\right)^3$$

$$\Rightarrow f(\vec{v}) = \left(\sqrt{\frac{\beta m}{2\pi}}\right)^3 \exp\left(-\frac{\beta}{2}mv^2\right); f(v) = 4\pi v^2 \left(\sqrt{\frac{\beta m}{2\pi}}\right)^3 \exp\left(-\frac{\beta}{2}mv^2\right)$$

$$\text{Recall } K_{avg.} = \frac{3}{2}kT \Rightarrow K_{avg.} = \int_0^\infty \frac{1}{2}mv^2 f(v)dv = \int_0^\infty \frac{1}{2}mv^2 \left(\sqrt{\frac{\beta m}{2\pi}}\right)^3 \exp\left(-\frac{\beta}{2}mv^2\right) 4\pi v^2 dv$$

$$= 2\pi m \left(\sqrt{\frac{\beta m}{2\pi}}\right)^3 \int_0^\infty v^4 \exp\left(-\frac{\beta}{2}mv^2\right) dv$$

$$= 2\pi m \left(\sqrt{\frac{\beta m}{2\pi}} \right)^3 \left(\frac{3\sqrt{\pi} \operatorname{erf}\left(\sqrt{\frac{\beta m}{2}} v\right)}{8 \left(\sqrt{\frac{\beta m}{2}} \right)^5} - \frac{v \exp\left(-\frac{\beta}{2} m v^2\right) (\beta m v^2 + 3)}{\beta^2 m^2} \right) \Bigg|_{v=0}^{v=\infty}$$

$$= 2\pi m \left(\sqrt{\frac{\beta m}{2\pi}} \right)^3 \frac{3\sqrt{\pi}}{8 \left(\sqrt{\frac{\beta m}{2}} \right)^5} = \frac{3}{2\beta} = \frac{3}{2} kT \Rightarrow \beta = \frac{1}{kT}$$

$$\Rightarrow f(v) = 4\pi v^2 \left(\sqrt{\frac{m}{2\pi kT}} \right)^3 \exp\left(-\frac{1}{2} m v^2 \frac{1}{kT}\right) = 4\pi \left(\sqrt{\frac{N_A m}{2\pi N_A kT}} \right)^3 v^2 \exp\left(-\frac{1}{2} N_A m v^2 \frac{1}{N_A kT}\right)$$

$$\Rightarrow f(v) = 4\pi \left(\frac{M}{2\pi RT} \right)^{\frac{3}{2}} v^2 \exp\left(-\frac{M v^2}{2RT}\right) \text{ Maxwell-Boltzmann distribution}$$

$$\text{Note: } f(\vec{v}) = \left(\sqrt{\frac{\beta m}{2\pi}} \right)^3 \exp\left(-\frac{\beta}{2} m v^2\right) = \left(\sqrt{\frac{m}{2\pi kT}} \right)^3 \exp\left(-\frac{1}{2kT} m v^2\right) = \left(\sqrt{\frac{m}{2\pi kT}} \right)^3 \exp\left(-\frac{E}{kT}\right)$$

$\exp\left(-\frac{E}{kT}\right)$ is known as the Boltzmann factor

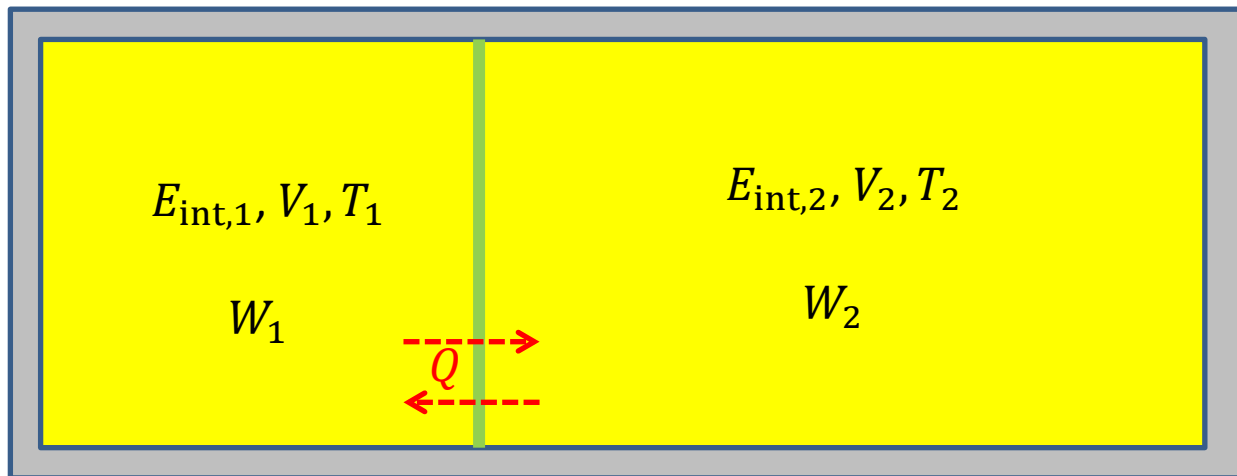
A statistical view of entropy

Consider a closed system divided into two subsystems. Let the two subsystems have fixed volumes and are in thermal contact with each other but both of them are isolated from the environment.

$$\Rightarrow E_{\text{int},1} + E_{\text{int},2} = E_{\text{int}} = \text{a constant} \Rightarrow E_{\text{int},2} = E_{\text{int}} - E_{\text{int},1}$$

Since the two subsystems are independent of each other, the number of microstates (multiplicity) of the closed system is the product of those of the two subsystems.

$$W_1(E_{\text{int},1}) \times W_2(E_{\text{int},2})$$



By the equal a priori probability postulate, $W_1 \times W_2$ is maximized at equilibrium.

$$\frac{d(W_1 \times W_2)}{dE_{\text{int},1}} = W_2 \times \frac{dW_1}{dE_{\text{int},1}} + W_1 \times \frac{dW_2}{dE_{\text{int},2}} \frac{dE_{\text{int},2}}{dE_{\text{int},1}} = W_2 \times \frac{dW_1}{dE_{\text{int},1}} - W_1 \times \frac{dW_2}{dE_{\text{int},2}} = 0$$

$$\Rightarrow W_2 \times \frac{dW_1}{dE_{\text{int},1}} = W_1 \times \frac{dW_2}{dE_{\text{int},2}} \Rightarrow \frac{\frac{1}{W_1} dW_1}{dE_{\text{int},1}} = \frac{\frac{1}{W_2} dW_2}{dE_{\text{int},2}} \Rightarrow \frac{d[\ln W_1]}{dE_{\text{int},1}} = \frac{d[\ln W_2]}{dE_{\text{int},2}}$$

$$\Rightarrow \frac{dE_{\text{int},1}}{d[k \ln W_1]} = \frac{dE_{\text{int},2}}{d[k \ln W_2]}$$

Noting that $dV_1 = dV_2 = 0 \Rightarrow \frac{dE_{\text{int},1}}{dS_1} = \frac{T_1 dS_1 - P_1 dV_1}{dS_1} = T_1; \frac{dE_{\text{int},2}}{dS_2} = \frac{T_2 dS_2 - P_2 dV_2}{dS_2} = T_2$

\Rightarrow If $k \ln W_1 = S_1$ and $k \ln W_2 = S_2$ then $T_1 = T_2$ (equilibrium in thermodynamics)

$S = k \ln W$ automatically equate equilibrium in statistical mechanics
with that in thermodynamics.

A general derivation of $S = k \ln W$ can be performed using Helmholtz theorem.