#### 72320 Roversystemtechnik Summer Semester 2021

#### **INSPIRE**

#### IN-situ Sampling and Primal Investigation Rover on Europa

Phase 0/A-Study of a Rover Mission on the Surface of the Jupiter moon Europa

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# Symbols

Symbol	Definition	Unit
a	Constant for the Geometry of a Porous Media	nm
A	Wheel Ground Contact Area	$\mathbf{m}$
b	Wheel Width	$\mathbf{m}$
c	Coefficient of Soil Cohesion	Pa
$C_{\text{Batt,req}}$	Total Required Battery Capacity	Wh
$C_{\text{Batt,req}}$	Battery Nominal Capacity	$\operatorname{Wh}$
$C_{ m cell}$	Cell Voltage	V
$C_{ m rr}$	Rolling Resistance Coefficient	-
d	Wheel Diameter	$\mathbf{m}$
DoD	Depth of Discharge	%
DP	Drawbar Pull	N
H	Soil Thrust	N
$h_{ m Ice}$	Ice Crust Surface Thickness on Europa	$\mathbf{m}$
$k_{ m c}$	Sinkage Modulus	$\frac{\mathrm{kN}}{\mathrm{m}^{n+2}}$
$k_{\phi}$	Soil Friction Angle Sinkage Modulus	$\frac{\mathrm{kN}}{\mathrm{m}^{n+3}}$
l	Ground Contact Length	m
$T_{ m Surface}$	Surface Temperature on Europa	K
M	Number of Cells in Parallel	-
$m_{ m RTG}$	RTG Mass	kg
n	Soil Deformation Exponent	-
N	Number of Cells in Series	-
$P_{ m el}$	RTG Electrical Power	$W_{ m el}$
$P_{\rm el,req}$	Required Electrical Power	$W_{ m el}$
$P_{\mathrm{mode}}$	Demanded Electrical Power per Mode	$W_{ m el}$
$R_{\rm b}$	Bulldozing Resistance	N
$R_{\rm c}$	Compaction Resistance	N
$R_{\rm g}$	Gravitational Resistance	N
$R_{ m r}$	Rolling Resistance	N
$t_{ m e}$	Mode Duration	S
$W_{\rm wheel}$	Normal Force per Wheel	N
z	Sinkage	$\mathbf{m}$
$\alpha_{ m BOL}$	BOL Specific Power	$\frac{W_{el}}{kg}$
$\epsilon$	Emissivity	-
$\eta_{ m LiIon}$	Efficiency of LiIon Cells	-
$\phi$	Friciton Angle	0
$ ho_{ m Ice}$	Inner Encoder Ring Diameter	$\frac{\text{kg}}{m^3}$
$\theta$	Slope Angle	0

### Abbreviations

BOL Begin of Life
BOM Begin of Mission
COMM Communications
DoD Depth of Discharge
EOM End of Mission

EPS Electrical Power System

2D Two Dimensional3D Three Dimensional

PCDU Power Control and Distribution Unit

PCU Power Control Unit

PDU Power Distribution and Control Unit

IMU Inertial Measurement Unit

IRS Institute of space Systems at the University of Stuttgart
INSPIRE IN-situ Sampling and Primal Investigation Rover on Europa

ESA European Space Agency MMP Mean Maximum Pressure

NASA National Aeronautics and Space Administration

SPENVIS SPace ENVironment Information System

HPC High Priority Commands

RTG Radioisotope Thermoelectric Generator

eMMRTG Enhanced Multi Mission Radioisotope Thermoelectric Generator

eSMMRTG Enhanced and Scaled Multi Mission Radioisotope Thermoelectric Generator (3kg)

TID Total Ionizing Dose
OBC On-Board Computer

S/C

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### The Mission

During the observation of Jupiter, the Glileo spacecraft did also some flybys of the Jupiter moons. The scientist gahtered data from Europa, which suported the evidence of a thick icy surface. The possibility of liquid water underneath lead astrobiologists to the assumption that extraterrestrial life could exist on Europa. That is why Europo is - beside Mars - an interesting object of research.

Therefore, the ESA will launch the *JUICE* oriber in 2022 to investigate Europa in more detail. But also the NASA is developing *Europa Clipper* to get detailed information. Additionally, they plan a lander for Europer to bring scientific instruments onto the surface. The DLR will perform the side mission TRIPLE, with project coordinator Dr. Waldmann, which take samples of the water by melting through the ice with an special testing probe.

Under the leadership of Prof. Dr.-Ing. Klinkner, the Institute of Aero Space Systems started within a seminar a feasibility study about a rover system to explore Europa surface, which shall be part of the *TRIPLE* mission. This challenge was given to five student teams in order to develop concepts, construct preliminary designs, perform analysis and make evaluations to meet the mission objectives and fit the mandatory requirements cite.

This report contain the results of the Phase A study of the rover system IN-SITU SAMPLING AND PRIMAL INVESTIGATION ROVER ON EUROPA (INSPIRE).

## Payload

#### 2.1 Sterovision Camera / Observation / Perception

The INSPIRE rover is equipped with five individual cameras. Two are used as stereo vision cameras on an hight adjustable and rotatable telescope arm on the front side of the rover. This ist used to capture a detailed 3D model of the environment with which sizes and distances can be estimated. The remaining three cameras are used as has-cameras which are necessary to obtain data regarding the nearby environment. All cameras are equipped with radiation hardened lenses to prevent browning of the lenses. The main tasks of the camera system is the provision of scientific data and navigation related data. More details regarding the navigation and autonomy are provided in ??.

#### 2.2 Ground RADAR

#### 2.3 Ice Core Drill

#### 2.4 RadHard Solar Arrays

As a secondary mission goal for INSPIRE a cooperation with the european project RadHard which is led by the german solar array manufacturer Azure Space is intended. They are currently developing a new generation of 4 Juniction solar cells with an efficiency of up to 35%. But the main feature of the new solar arrays is their radiation hardness which will be the highest radiation hardness ever designed with an efficiency of > 3% after  $1E15~cm^{-2}~1MeV$  electron irradiation. So the Jupiter environment with its extreme radiation would be the best suitable destination for a test and evaulation mission of this new technology. Therefore INSPIRE will be equipped with 8 RadHard solar cells with a total surface area of  $0.0248m^2$  for a technology demonstration[1].

### Operation

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#### 3.1 Mission Phases

For the INSPIRE Mission Phase 0 study five basic mission phases have been defined. Furthermore a sixed optional mission phase after the nominal mission lifetime has been established which will be conducted if the rover is still operational after its nominal lifetime.

- Phase 0: Launch and Flight Phase
- Phase 1: Entry, Descent and Landing Phase
- Phase 2: Depolyment Phase
- Phase 3: Egress, Comissioning and Early Operation Phase
- Phase 4: Mission Operation Phase
- (Phase 5: Exceeding Mission Operation Phase)



NErD: Nominal Europa reference Day (85h or roughly 3,6 Earth Days)

Figure 3.1: Preliminary Mission Timeline for INSPIRE.

Based on these missions phases some preliminary rover system modes as well as a basic mission timeline were concluded.

#### 3.1.1 Rover System Modes

For this case study several rover system modes were defined. All ten modes are listed in Table 3.1. They are seperated into two groups. The design critical modes are displayed in white and are defined as system modes, which significantly influence the preliminary design of the rover subsystem like the thermal or power subsystem. None design critical modes (grey) also have a major influence on multiple subsystems of the rover but play a secondary role in the thermal and power budget of the rover for this Phase 0 study. These non design critical modes extend from the rover storage and launch until the finale deployment of the rover is completed. These modes and their design options depend heavily on the final design of the lander with which INSPIRE flies to Europe. Therefore, a clear definition of such modes is not possible at this time in the course of this phase 0 study. However, the respective considerations, preferences and options have been briefly described in the mode descriptions. It is important to note that INSPIRE's

goal is to provide a flexible rover design with as few hard requirements as possible for the parent lander. Therefore, many aspects of the rover, as well as the none design critical modes, will need to be further defined and elaborated in later phases of the project in close consultation with the customer.

For example, the exact interfaces between rover and lander should be defined in more detail. Depending on the subsequently chosen interfaces, many possibilities may arise in the corresponding rover system modes. With an appropriate interface, for example, the excess electrical and thermal energy of the RTG, which is already active during the flight, could be used to supply the lander system with heat and power. A corresponding interface could also enable the transmission of health checks from INSPIRE.

The deployment phase will strongly depend on the final design of the lander, INSPIRE's position within the lander and also the possibilities that the lander provides to INSPIRE. Possible deployment strategies would be as follows:

• Option 1: If INSPIRE on ground level: Release from storage box through spring mechanism or actuators. Rover storage configuration allows rolling and possible motorized actuation

• Option 2: If INSPIRE is above ground level: Similar as Option 1 but an additional ramp and ramp deployment would be required.

• Option 3: INSPIRE will be deployed through the landers robotic arm if it is capable of lifting its mass.

Number	Rover System Modes	Abbrevation	Definition
	-	1	From Launch until EDL Phase
			Rover System is OFF
			Exact mode description t.b.d. and can be adapted to meet the lander demands
0	Launch/Off Mode	OFF	Health tests on Occasion during flight time are foreseen (PCDU could be active)  Batteries on Storage Capacity at launch and may be recharged on occasion (like Rosetta Mission)
			Telemetry data shall be sent by the Lander (optional if possible)
			RTG on =>Electrical and Thermal Power may be used
			(for Lander Power and Thermal Systems) or is disposed of by shunts
			From Entry until next morning after secure landing of Lander on Europa
			See Mode OFF
1	Entry, Descent and Landing	EDL	PCDU ON after secure landing (Powered by RTG)
			Heaters ON (powered by remaining RTG Power)
			Battery charging if no Kill Switch is used
			First Morning after EDL
			Exact mode description t.b.d. and can be customized to lander =>Dependant on final Lander Design
			Critical Deployments (Egress System) and leaving the lander
			Optional whether Kill Switch ejected =>Battery charging can start
2	Deployment and Farly Operation Mode	EOP	Rover System Activation possibilities: Kill Switch, Lander Interface, HPC from Earth
2	Deployment and Early Operation Mode	EOF	PCDU ON
			OBC ON
			Heaters ON
			After sufficient Battery Capacity is reached (50%): Deployment of Rover Boogie
			and checkout/health check of all Rover Systems Afterward switching to Charging Mode
			During Idle Operation Time
			Rover powered by RTG or Batteries (Excess Power charges Batteries)
			PCDU ON
3	Idle/ Perception	ID	All Components in Standby or Power Saving Mode if possible
			Stereovision Camera ON for Orientation and Observation (Science Data)
			Hazcams and OBC ON for Orientation and Path Analysation
			COMM ON for larger time intervals (Listening Mode)
			Entered in case of emergency or contingency Rover
			Survival Mode =>Minimum Power PCDU ON
			COMM sends Emergency Signal then switches to
			COMM ON for small time intervals (Listening Mode)
4	Safe Mode/ Hibernation (SAFE)	SAFE	OBC OFF until Command received =>High Power Commands (HPC)
	,		Heaters ON
			Science data shall be stored without data loss
			Applicable during Day and Nighttime
			Exit after receiving the corresponding command
			(Optional: Timer ON and Restart of Rover System after time period has passed)
			During Transmission of major Telemetry or Science Data
			Rover powered by RTG or Batteries (Excess Power charges Batteries) PCDU ON
6	Communication	COMM	All Components in Standby or Power Saving Mode if possible
			OBC SB
			COMM ON (Transmission Mode)
			For Battery charging
			Rover batteries charged by RTG
7	Charging	BAT	PCDU ON
	~ ~		All Components in Standby or Power Saving Mode if possible
			OBC SB  Quit after sufficient charge is reached
			For Rover Movement and Observation
			Locomotion and Navigation ON
			Hazcams and Traversing Path Analysis ON
8	Locomotion	LOC	OBC ON
0	Locomotion	100	PCDU ON
			COMM OFF
			Stereovision Camera ON for Orientation and Observation (Science Data)
			Only during Daytime  Paulord Mode for Science Date Collection during Daytime
			Payload Mode for Science Data Collection during Daytime OBC ON
			PCDU ON
9	Payload Observation Mode	OBS	COMM OFF
	•		Stereovision Camera ON for Orientation and Observation (Science Data)
			RADAR ON for Ground Investigation =>Drill Location
			Only during Daytime
			Payload Mode for Science Data Collection during Daytime or Nighttime
			OBC ON
10	Payload: Ice Core Mode	ICE	PCDU ON
10			COMM OFF Lee Core Drill ON during Lee Core Sample Collection
			Ice Core Drill ON during Ice Core Sample Collection  Afterwards Sample will be analysed — APXS ON
			Afterwards Sample will be analysed =>APXS ON

Table 3.1: Collection of Rover System Modes. [Kommt noch in Anhang]

# Subsystems

4.1 Rover

4.2 Structure and Mechanics

4.3 Communications and Command and Data-Handling

4.4 Payload

...

#### 4.5 Thermal Control System

The main object of the Thermal Control System (TSC) is to keep the eletric components within their temperature limits, listed in Table B.3. As a result of Europas low ground temperature, a small solar constant and the thin atmosphere the heat loss of the rover has to bin minimised. This shall be reached by a smart heat distribution as well as by an adequate insulation and surface finishing.

#### 4.5.1 Thermal concepts

The main heat source of the rover is the waste heat of the RTG (see Section 4.6), which will be lead by thermal straps to the thermal critical components. The first attempt to use straps made out of copper was rejected du to the high resulting mass. Therefore, carbon-based straps (Thermal LyNX) with a high thermal conductivity to density ratio will be used, cite. However, the thermal conductivity is highly depend on the materials temperature, see ??. The curve was approximated by an qubic interpolation, autoref appendix. To consider heat loss as a result of contact and radiation, the thermal conductivity was reduced about 20%.

[Temperature dependent thermal conductivity/density of  $LyNX^{\textcircled{8}}$  (yellow curve), copper, and

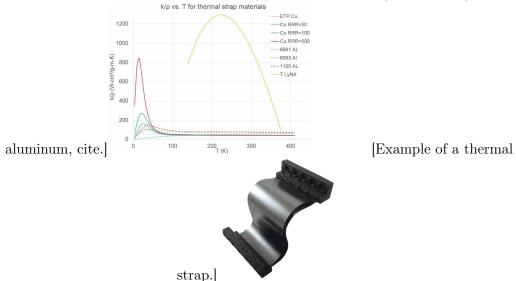


Figure 4.1: Carbon-based thermal strap  $LyNX^{\mathbb{R}}$ .

The camera, exposed on a mast, will be heated by a seperate, light weight Radioisotope Heater Unit (RHU), citeRHU, which has been used during several NASA missions. For the insulation, the material *aerogel* will be applied, which has a very low heat conductivity (cite aerogel) as well as a low density and has also been used in space applications (citeaerogel).

A surface finishing with an low emisivity is necessary. For the most components, a cost-efficient surface polishing is applicable. The camera will get a special white paint with a high absorptivity to gather the sun light.

But ther is also a risk of overheating, because of the lack of heat convection. This circumstance concern the engines and also the camera. To get rid of the extensive heat and prevent the damage of the components special heat switches will be placed (see Figure 4.2). These switches change their heat conductivity beyond a certain temperatur due to the expansion of the disk (see Figure 4.3). It was assumed, that the toggle temperatur can be adapted by increasing the disk height. The influence of the changed disk stiffens on the contact pressure and the heat conductivity was neglected for this study. The measured heat conductivity characteristic was

divided in three linear sections (Figure ??), to enable a simple modelling in the upcoming thermal calculation.

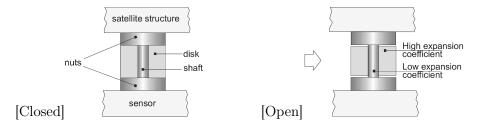


Figure 4.2: Heat switch.

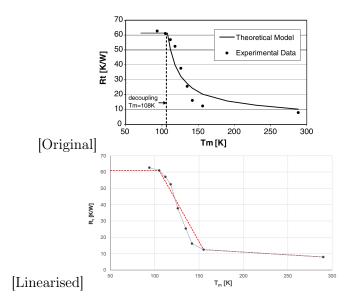


Figure 4.3: Change of the heat switch conductivity  $R_t$  over the mean temperature  $T_M$ .

#### 4.5.2 Thermal Network

A thermal analysis was performed in order to get

- the dimension of the isnulation and heat straps,
- the necessary amount of RHUs and heat switches,
- the required surface finishing.

For that, a thermal network with ten nodes was derived from the rover, shown in Figure 4.4. At the intersection of the steering and drive engien, two additional nodes were defined to calculate the heat flow.

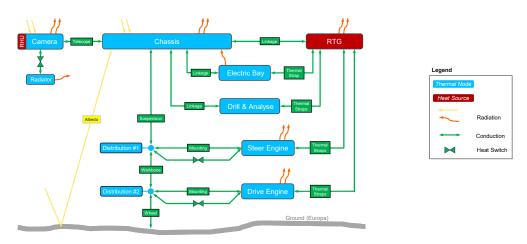


Figure 4.4: Thermal network of the rover.

On the basis of the thermal network, the heat energy equlibrium for each node was defined (see Section B). The calculation were considered as a quasi-static analysis, were the component temperatures stay constant,  $\frac{dT}{dt} = 0$ . Due to the early state of the rover, simplifications and asumptions were made.

- The convection was neglected due to the thin atmosphere.
- The whole electrical power of the components will be dissipated into heat.
- A variation of  $\pm 20\%$  for the emisivity and absorpsivity values was considered, if applicable (seet Table B.4).
- The heat as a result of retardation radiation inside the shielding was neglected.
- The E-Bay emitts heat energy only in one direction to the chassis.
- Only the chassis and the camera absorb sun radiation, detailed description see Subsection B.4.
- The ice core drill and the APXS analyser were summerised as one single node.
- No dicrete nodes for the radar, hazcams and deployment engines were considered. Their heat will be lead into the chassis.
- The thermal resistance between two surfaces and ther radiation of the thermal strap was neglected. To compensate this assumption, the heat conductivity was reduced about 20%.

#### 4.5.3 Thermal analysis

The thermal analysis was performed as a Excel calculation, which can be found in the corresponding folder of Team 3. The calculation sheet offers the possibility to adapt and customise input values and dimension.

#### 4.5.4 Results

The results of the temperatures for each node at each for the hot and cold cases are listed in autoreftab:. The temperature margins for uncertainties, acceptance tests and qulification tests were considered with 5K each,  $\pm 15K$  in total. The corresponding temperatures are listed in autoreftab:. All temperature lay between their limits without using a heater. Nevertheless, heaters were considered for the power calculation, see Section 4.6.

Components	Load Case				
	Hot case		Col	d case	
	0K	+15K	0K	-15K	
RTG	380	395	350	335	
Electric Bay	310	325	261	246	
Drill & Analyser					
Camera					
Steering Engine					
Drive Engine					

Table 4.1: Temperature results in K of thermal analysis, including a margin of  $\pm 15K$ .

In a further step, a more detailed analysis has to be carried out with the Finite-Element-Method to identify the correct heat and temperature distribution. The FE results could be used to verify and adjust the analytical analysis to get a helpful tool for fast thermal calculation to evaluate different materials or desings in the further development phases.

#### 4.6 Electrical Power System

The EPS (Electrical Power System) is the subsystem responsible for the electrical power supply of INSPIRE. It consists of four funadmental parts, which are the energy source, the PCDU unit (Power Control and Distribution) and the Energy Storage as well as the rover subsystems as the consumers. The EPS is visualized in Figure 4.5.

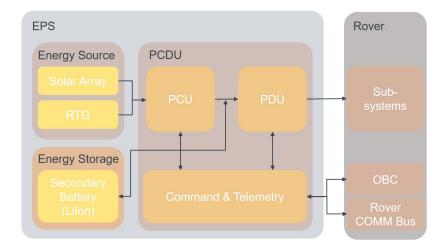


Figure 4.5: Functional Flow Chart Diagram for the EPS Subsystem.

#### 4.6.1 EPS Budget and Overview

Table 4.2 summarizes the power budget of INSPIRE based on the rover system modes defined in Subsection 3.1.1. The complete power budget can be found in Figure A.1. As can be seen, the Locomotion mode has the highest demands on the EPS. Communication mode also has a high consumption. However, since this is primarily used at night and the rover can be charged again afterwards without any problems, it does not place any major restrictions on the power budget. Idle/Perception mode has a low consumption, but is usually used for a long time at a stretch and therefore also places high demands on the EPS. In Charging mode, the EPS is able to charge  $7.04~W_{el}$ .

Rover System Mode	Total Rover Power Demand		
	including battery charge $P_{\mathbf{mode}}$ [ $W_{el}$ ]		
Idle/Perception	15.25		
Safe/Hibernation	-5.84		
Communication	36.92		
Charging	-7.04		
Locomotion	232.36		
Payload: Observation	15.41		
Payload: Ice Core Mode	9.77		

Table 4.2: Overview of the Power Budget of INSPIRE.

#### 4.6.2 Energy Source

For the energy generation of INSPIRE many possible sources were taken into consideration for a trade-off. As a conclusion of this trad-off the decision was made to utilize a Radioisotope Thermoelectric Generator (RTG) as the main energy source for INSPIRE.

As the research couldn't find an RTG with a mass suitable for INSPIRE, the solution was to scale down a bigger RTG as an approximation. As a baseline of the scaling the eMMRTG (Enhanced Multi Mission Radioisotope Thermoelectric Generator) was utilized, which is currently under development at NASA and is especially designed for deep space missions like Europa. For the scaling a goal RTG mass of  $m_{\rm RTG}=3$  kg was defined and the eMMRTG was scaled down using the given data.

In Table 4.3 the scaling results for the eSMMRTG (Enhanced and Scaled Multi Mission Radioisotope Thermoelectric Generator) are listed. The eSMMRTG has a BOL specific power of  $\alpha_{\text{BOL}} = 4.0 \frac{W_{el}}{kg}$  and provides an electrical power of  $P_{el} = 12.08 W_{el}$  during the mission duration[2][3][4][5][6].

Scaled eSMMRTG Parameter			
System Mass $m_{\rm RTG}$ [kg]	3.5		
BOL Specific Power $\alpha_{\text{BOL}} \frac{W_{el}}{kg}$	4.0		
BOL Power $P_{el, \mathrm{BOL}}$ $W_{el}$	14		
Isotrop	Pu-238		
Isotrop Half-Life $[a]$	87.7		
Flight time and Storage (incl. Margins) [a]	7		
Power Loss Degradation until BOM $W_{el}$	0.56		
BOM Power $P_{el, \text{BOM}}$ $W_{el}$	13.44		
Europa Day Duration [h]	85		
Mission Duration $[d]$	106.25		
End of Mission Power $P_{el,EOM}$ [ $W_{el}$ ]	13.42		
Final Power for Study $P_{el}$ [ $W_{el}$ ] (incl. 10% scaling Margin)			

Table 4.3: Parameters for the scaled eSMMRTG based on the eMMRTG.

Furthermore INSPIRE will also be equipped with some radiation hardend solar arrays as already explained in Section 2.4[1]. Since these solar cells are primarily used for technology testing, the mission must also be able to operate completely without this generated energy. For this reason, and because the expected energy generated by the solar cells is minimal, only the energy generated by the RTG is considered for the Phase 0 Study. However, it should be noted that these solar cells will also generate a certain amount of energy, which will benefitial for the EPS.

#### 4.6.3 Energy Storage

For the energy storage of INSPIRE many possible battery types were taken into consideration for a trade-off. As a conclusion of this trad-off the decision was made to utilize LiIon batteries as the secondary batteries of INSPIRE. This decision is primarly based on LiIon batteries high energy density, temperature range, robust performance and long operating and cycle life in extreme environments[7].

As the RTG only generates a small constant power the main energy source during the mission will be the accumulated energy of the batteries. The rover will charge the batteries at night,

so the next exploration day can start with full capacity. Furthermore the batteries have to be charged during day time to maintain operations.

For the sizing of the batteries, the rover motion was chosen as the design driver, since this is the highest energy consuming state of the rover and additionally mission critical for INSPIRE. The rover motion consists of an interaction of the Locomotion and Perception mode as already mentioned in Chapter 3. Therefore it was defined that INSPIRE shall be able to drive 50 m (including alternating Locomotion and Perception Mode) with a fully charged Battery. The required battery capacity  $C_{\text{Batt,req}}$  can be caculated using Equation 4.1. The results are listed in Table 4.4 [8].

$$C_{\text{Batt,req}} = \frac{P_{\text{el,req}} \cdot t_e}{DoD \cdot \eta_{\text{LiIon}}} \tag{4.1}$$

Power Consumption Mode:	Locomotion	Perception
Required Electrical Power $P_{\rm el,req}$ [ $W_{el}$ ]	283.43	14.01
Duration of the mode $t_e$ [s]	500	15000
DOD for Dimensioning [-]	0.90	0.90
Efficiency of LiIon Cells $\eta_{\text{LiIon}}$ [-]	0.95	0.95
Required Battery Capacity per mode $C_{\text{mode}}$ [Wh]	46.04	68.27
Total Required Battery Capacity $C_{\text{Batt,req}}$ [Wh]	114	.32

Table 4.4: Power consumption mode used as design case for the battery sizing.

Using these values a suitable battery cell and battery design configuration were conducted. Under consideration of these parameters the battery capacity  $C_{\text{Batt}}$  can be calcuated:

$$C_{\text{Batt}} = C_{\text{cell}} \cdot V_{\text{cell}} \cdot N \cdot M. \tag{4.2}$$

According to the ECSS reliability restrictions 1 battery string must be substracted for dimensionsing. Furthermore a 30% margin on the energy content was applied. This leads to a final battery configuration with a capacity of  $C_{\text{Batt}} = 138,88 \ Wh$  and a mass of  $m_{\text{Batt}} = 1980 \ g$ . The final battery values are listed in Table A.1 [9].

#### 4.6.4 EPS Power Control and Distribution

In order to ensure the full functionality of the EPS, the last main component to be selected is a suitable PCDU. As described in Figure 4.5, the PCDU forms the heart of the EPS and is an important interface to the OBC and COMM. Furthermore the PCDU shall be able to monitor and control the rover system if necessary through watchdogs, HPC (High Priority Commands) and direct connections to the OBC and COMM.

The PCDU has the challenging task not only to process the RTG as the main energy source, but also to process solar cells as secondary energy sources. Therefore, a PCDU was sought which has the required size, dimensions and range of functions. The research resulted in the Nova PCDU from Bradford DSI. In addition, margins were added to the PCDU to ensure feasibility[10].

#### 4.7 Radiation

Compared to the radiation environment near Earth the radiation environment near Jupiter is multiple times stronger. It has the highest radiation levels of any planet in our solar systems [Platzhalter]. In order to survive these harsh environmental conditions, special emphasis must be placed on the radiation protection. In Figure C.3, the average trapped proton and electron fluxes on Europa's orbit around Jupiter are shown in comparison to the outer Van Allen radiation belt around Earth. However, in contrast to the Van Allen radiation belt, the duration within the radiation environment on Europa cannot be minimised and the rover has to be designed to withstand the entire mission duration of 30 days.

In oder to design and evaluate different radiation protection approaches, different calculations have to be performed. For this purpose the ESA SPace ENVironment Information System (SPENVIS) is used [**Platzhalter**]. All calculations and figures in Section 4.7 are performed with SPENVIS unless otherwise stated.

#### 4.7.1 Radiation Protection

Various options are available to protect the rover against the radiation. A common approach is the use of aluminium or titanium as these materials can also act as structural elements. However, due to the mass constraints of 30 kg other materials or material compositions are taken in consideration which are more mass effective. In Table 4.5, an optimised shield structure is presented for different weight thresholds designed for the radiation environment around Jupiter.

The difference between an aluminium or titanium shielding and an optimised structure listed in Table 4.5 for the total ionizing dose (TID) is shown in Figure C.4.

Due to the mass savings of the optimised structure it will be used where the radiation protection of the aluminium structure is not sufficient. In order to reduce the mass further, a radiation vault is utilised that highly sensible components do not have to be shielded separately.

Table 4.5: Optimal shield structure for an Jupiter mission. [Platzhalter]

Areal Density / g/cm <sup>2</sup>	0.5	1	2	3
Layer No. 1	Pb	Pb	W	Ta
/ mm	0.415	0.829	0.984	1.563
Layer No. 2	Fe	Mg	Mg	Al
/  mm	0.033	0.158	0.540	0.399
Layer No. 3	-	-	-	Mg
/ mm	-	-	-	0.150

#### 4.7.2 Components

Every component on the rover has a different

radiation tolerance and therefore have to be placed at different compartments within the rover. The radiation tolerances are listed in Table C.7. None sensitive components like the electric motors and harness are only shielded by an aluminium structure where components like the metal within the wire are resistant against the radiation. However, isolators around the cables have to be selected to be resistant in order to prevent short circuits. Highly sensitive components like cameras have an additional protective layer in order to reduce the TID to under 30 krad. Components which are within the rover like the on-board computer (OBC) are placed within the radiation vault which reduces the TID to under 20 krad. For this purpose the optimised shield structure with a weight target of  $0.5~\rm g/cm^2$  is used. Detailed TIDs for all components are shown in Figure C.5.

#### 4.7.3 Improvements

Even though the radiation protection is sufficient for the rover to survive at least the nominal mission of 30 days, further improvements can be performed in order to extend the secondary mission.

Local shielding can be applied on less resistant components in order to reduce the wall thickness of the whole radiation wall. If components with a radiation tolerance under 43.27 krad are individually shielded a mass saving of 736.2 g can be achieved. Additionally, water ice extracted from the surface of Europa can be used to improve the radiation protection. With a layer of one centimetre of water, the TID within the radiation vault can be reduced to 16.03 krad without the additional radiation protection beside the 4 mm of aluminium structure. The start mass of the rover can therefore be reduced by 897.2 g by removing the additional shielding.

Detailed calculations for local shielding and water improvements can be found in Subsection C.3 and may be analysed further in Phase B.

#### 4.7.4 Conclusion

In order to protect the rover against the high radiation levels at the surface of Europa, the rover has different compartments. High sensible components are placed within a radiation vault which has a mass optimised structure. Components which has to be outside the radiation vault but are highly sensible are shielded individually. Low sensible Components are protected by the Aluminium structure. Figure 4.6 illustrates the different compartments within the rover and the accorded TIDs.

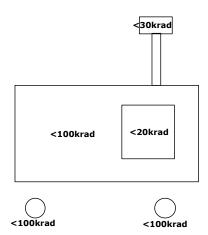


Figure 4.6: Overview of TIDs within different compartments within the rover.

#### 4.8 Locomotion

The locomotion subsystem deals with the aspect of how the rover moves and the technical design, including the selection of components such as motors or gearboxes. Before the components can be determined, however, it is necessary to consider certain parameters. These parameters will be introduced in the following, and the decisions or estimations concerning the rover will be presented.

#### Design drivers

Regarding techniques of rover movement, it is crucial to consider the local conditions in which the rover will be operating. The surface of Europa can be assumed to be mainly covered by ice. However, since there are also geysers that transport water to the surface, surface areas may be covered by snow. Though Europa's surface temperature does not exceed 130 K, rather icy, hard-packed snow can be assumed. For this study, a conservative design of the rover is considered, whereby material parameters of snow in Sweden were selected, see ?? Furthermore, a 4-wheeled rover is chosen, as evidenced by a trade-off in

# Outlook

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# Appendix

### A Electrical Power System

SAFT 176065 xlr [9]							
Configuration:							
Battery Configuration	4s3p						
Cells in Sereis $s$ N [-]	4						
Cells in Parallel $p$ M [-]	3						
Cell Parameters:							
Typical Cell Capacity [Ah]	6.8						
Nominal Cell Voltage $[V]$	3.65						
Nominal Cell Capacity $[Wh]$	24.8						
Typical Cell Mass $[kg]$	0.15						
Energy Density $[Wh/kg]$	165.33						
Actual Battery Configuration Parameters:							
Battery Voltage $V_{\rm Batt}$ [V]	14.6						
Battery Nominal Capacity $E_{\text{Batt}}$ [Wh]	297.6						
Battery Mass $[kg]$	1.8						
Battery Mass $m_{\text{Batt}}$ (incl. 10% Margin) $[kg]$	1.98						
Configruation according to ECSS reliability restr	ictions and margins included:						
Battery Configuration	4s2p						
Cells in Sereis $s$ N [-]	4						
Cells in Parallel $p$ M [-]	2						
Battery Voltage $V_{\text{Batt}}$ [V]	14.6						
Battery Nominal Capacity $E_{\text{Batt}}$ [Wh]	198.4						
30% Margin on Energy Content	0.3						
Battery Nominal Capacity $E_{\text{Batt}}$ incl. Margin $[Wh]$	138.88						
Useable Energy Density $[Wh/kg]$	70.14						

Table A.1: INSPIRE battery parameters.

				Power	demand p	per Unit		EDL		Depli	oyment N	1ode	Idle	/ Percepti	on	Safe	/ Hibe	rnation	Com	municati	on	Chi	arging (R	TG)	Lo	comotio	n	Payloz	id: Obser	vation	Payload	Ice Con	t Moc
Subsystem	Unit	Amount [-]	Maturity Level	Standb y (SB)	Nominal		Status	Duty Cycle	Power [W]	Status	Duty Cycle	Power [W]	Status	Duty Cycle	Power [W]		Duty	Power [W]	Status	Duty Cycle	Power [W]	Status	Duty Cycle	Power [W]	Status	Duty Cycle	Power [W]	Status	Duty Cycle	Power [W]	Status	Duty Cycle	Pow [W
Payload			Margin (%)		(ON)	(MAX)	-	Ė		¥	_	_				-	-		*	_		¥	_	0			-	-	Ė			Ė	Ċ
	Ice Core Drill	1	20,00% *	0	1	15	OFF *			OFF *		0	OFF *		0	OFF *		0	OFF *		0	OFF *		0	OFF +		0	OFF +		0	ON *	50%	0,
	APXS Analyser	1	5,00% *	0	0,15	1,5	OFF *			OFF *		0	OFF *		0	OFF *		0	OFF *		0	OFF *		0	OFF +		0	OFF *		0	MAX *	50%	0,78
	Stereo Vision Camera Hazrams	2	5,00% *	0	2 2	4	OFF +			ON +	100%	4,2 8.4	ON +	25% 25%	1,05	OFF +	-	0	OFF +		0	OFF +		0	ON +	100%	4,2 8.4	ON +	100%	4,2	OFF +	-	0
	Ground RADAR	1	10,00% ~	0	0,3	3	OFF +			OFF +	100%	0	OFF +	23.9	0	OFF +	_	0	OFF +		0	OFF +		0	OFF +	100%	0	MAX +	25%	0,825	OFF +		0
	LED Lampe	8	*				~			~		0			0	~	- 4	0	~		0	*		0	~		0			0			0
Command & Data							-			~	- 1	0			0	-		0			0	-		0			0			0			d
Handling	Processor	1	5,00% *	1	8	10	OFF +			ON +	100%	8,4	MAX *	90%	9.45	OFF *	0%	0	ON ~	100%	8,4	ON +	5%	0,42	ON +	100%	8,4	ON +	100%	8,4	ON +	100%	8.
	Flucessoi		3,00%	1	l °	10	UII ·			UN .	100%	0	WIAA .	30/6	0	UII -	0.0	0	UN .	100%	0,4	UN .	270	0,42	UN .	100/6	0,4	UIV ·	100%	0,4	UN .	100/6	0,
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			~			_	~			*		0	- +		0	*		0	*		0	*		0	+		0	-		0	-		0
			-	-		-	*			*		0	-		0	*		0	*		0	*		0	*		0	7		0	-	-	0
hermal Control System	Heaters	2	5.00% *	0	1	2	ON Y	100%	2,1	ON +	100%	2,1	ON +	100%	2,1	ON +	100%	2,1	ON +	100%	2,1	ON -	85%	1,785	ON Y	100%	2,1	ON +		2,1	ON +	100%	2,:
		-	-,00/4	-	1	T*	· ·	20076	2,4	7	2007/4	0	-	20074	0	Ψ.	200/4	0	*	20079	0	*	4274	0	-	20079	0	· ·	20076	0		20079	0
			*									0			0	Ψ.	- 1	0			0	٧		0			0	*		0	~		0
										·		0		$\perp$	0			0	*		0	¥		0			0	٧		0			0
Electric Power Supply	PCDU	1	10,00% *	0,6	1	1	SB +	100%	0,66	SB +	100%	0,66	ON +	100%	1,1	SB +	100%	0,66	ON *	100%	1,1	SB *	100%	0,66	ON +	100%	1,1	ON +	100%	1,1	ON +	100%	1,:
	PCDU	1	10,00% *	0,6	1	-	30 *	100%	0,00	30 *	100%	0,00	UN +	100%	0	30 *	100%	0,00	UN +	100%	0	30 *	100%	0,00	UN +	100%	0	UN +	100%	0	UN +	100%	0
							*					0	-		0	*		0			0	*		0	-		0	-		0	-		0
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Communication						_						0	*		0	*	_	0	*		0	٧		0	*		0			0		_	0
	Transmitter	1	10,00% *	0,6	10	15	OFF +			OFF +		0	OFF *		0	OFF +	-	0	MAX *	1000	0 16,5	OFF *		0	OFF +		0	OFF +	_	0	OFF *	-	0
	Receiver	1	10,00% ~	0,35	1	2	OFF +			ON +	100%	1,1		100%	1,1	ON *	100%	1,1	MAX Y		2,2	SB +	100%		ON +		1,1		100%	1,1		100%	1,1
			*	1			¥			*		0	~		0	¥		0	~		0	*		0	~		0	~		0	~		0
Locomotion			٧				٧			,		0	*		0	¥		0	¥		0	¥		0	*		0	¥		0	*		0
	Motor (wheels) Motor (steering)	4	5,00% *	0	30 15	45 15	OFF +			OFF +		0	OFF +		0	OFF +	-	0	OFF *		0	OFF *		0	MAX ~		126 6.3	OFF *		0	OFF *	-	0
	IMU	10	5,00% *		15	12	UFF +			OFF +		0	OFF *	1	0	OFF *		0	UFF *		0	UPP +	_	0	WIRA *	10%	0,3	OFF *	_	0	UFF *	_	0
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Chassi & Structure										À		0	¥		0	*		0			0	٧		0	*		0	*		0	Y		0
	Rover Boogle Depolyment		~				~			~		0	¥		0			0			0			0	¥		0			0			0
	Drill Mechanism					<del>                                     </del>	v			v		0	-		0		-	0			0			0	-		0	·	_	0		-	0
			*				٧					0			0			0	*		0	¥		0	*		0			0			0
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S/C Net P	Power Demand [W]		Margin (%)				Г		2,76			24,86			16,90			3,86			30,30			3,25			157,60			17,73		П	14,0
	Distribution loss		2.00%						0.06			0.50			0.34			0.08			0.61			0.07			3.15			0.35		$\vdash$	0,2
100.000	Discharge (PCDU)		7,00%	-		-			0,19			1.74		1	1,18	-	-	0,27	0		2.12			0,23			11,03			1.24		-	0,9
	rgin (Conversion etc.)		5,00%			<del>                                     </del>	-		0,19			1,74	1	+	0,85			0,27			1,52			0,23	<u> </u>	$\vdash$	11,05			1,24		1	0,9
	arness losses		3,50%						0,10			0,87			0,59			0,14			1,06			0,11			5,52			0,62			0,4
Addi	itional losses		5,00%		_				0,14		-	1,24		$\perp \neg$	0,85			0,19			1,52			0,16			7,88			0,89			0,7
S/C Brutto	Power Demand [W]			1			1		3,38			30,45	1		20,70			4,73	l		37,12			3,82	l		185,18			20,83	1		16,
Sys	stem Margin		20,00%						0,68			6,09			4,14			0,95			7,42			0,76			37,04			4,17			3,3
Required power from E	Battery (20% System Ma	rgin) [W]							4,06			36,54			24,84			5,67			44,54			4,58			222,22			24,99			19,8
	emand excluding battery s 5% + Lilon Efficiency 59		10,00%						4,46			40,20			27,33			6,24			49,00			5,04			244,44			27,49			21,8
Incom	ming Power RTG		-						12,08			12,08			12,08			12,08			12,08			12,08			12,08			12,08			12,0
	nand including battery cl	narge [W]	-						-7,62			28,12			15,25			-5,84			36,92			-7,04			232,36			15,41			9,7

Figure A.1: POWER BUDGET DUMMY!

#### B Thermal Controls System

#### B.1 Heat energy eqilibrium

The follwing equiations describe each node of the thermal network. The input values were tanken from EPS or from Subsection B.2, Subsection B.3, Subsection B.4.

RTG:

$$\dot{Q}_{RTG} + CON_1 \cdot (T_{Bay} - T_{RTG}) + CON_2 \cdot (T_{Chassis} - T_{RTG}) + CON_5 \cdot (T_{Drill} - T_{RTG}) -$$
$$-\epsilon_{RTG} \cdot \sigma_b \cdot S_{RTG} \cdot T_{RTG}^4 = 0$$

#### Electric Bay:

$$\dot{Q}_{Bay} + CON_1 \cdot (T_{RTG} - T_{Bay}) + CON_3 \cdot (T_{Chas} - T_{Bay}) - \epsilon_{Bay} \cdot \sigma_b \cdot S_{Bay} \cdot T_{Bay}^4 = 0$$
 with:

$$\dot{Q}_{B,intern} = \dot{Q}_{C\&DH} + \dot{Q}_{Tranceiver} + \dot{Q}_{Receiver} + \dot{Q}_{PCDU}$$

#### Drill & Analyser:

$$\dot{Q}_{Drill} + CON_4 \cdot (T_{Chas} - T_{Drill}) + CON_5 \cdot (T_{RTG} - T_{Drill}) = 0$$

#### Camera:

$$\dot{Q}_{Cam} + \dot{Q}_{RHU} + CON_6 \cdot (T_{Chas} - T_{Cam}) + (CON_{14} + n_{S1} \cdot CON_{S1}) \cdot (T_{Rad} - T_{Cam}) - \epsilon_{Cam} \cdot \sigma_b \cdot S_{Cam} \cdot T_{Cam}^4 = 0$$

Radiator:

$$(CON_{14} + n_{S1} \cdot CON_{S1}) \cdot (T_{Cam} - T_{Rad}) - \epsilon_{Rad} \cdot \sigma_b \cdot S_{Rad} \cdot T_{Rad}^4 = 0$$

Chassis:

$$\dot{Q}_{Radar} + \dot{Q}_{Hazcam} + CON_2 \cdot (T_{RTG} - T_{Chas}) + CON_3 \cdot (T_{Bay} - T_{Chas}) +$$

$$+CON_4 \cdot (T_{Drill} - T_{Chas}) + CON_6 \cdot (T_{Cam} - T_{Chas}) + CON_7 \cdot (T_{Node_1} - T_{Chas}) +$$

$$+\alpha_{Chas} \cdot [q_{Sun} \cdot (1 + \rho_E) \cdot (S_{Chas_1} \cdot \varphi_1 + S_{Chas_2} \cdot \varphi_2) + \epsilon_E \cdot \sigma_b \cdot S_{Chas_3} \cdot T_{Surface}^4] -$$

$$-\epsilon_{Chas} \cdot \sigma_b \cdot S_{Chas} \cdot T_{Chas}^4 = 0$$

#### Steer Enine:

$$4 \cdot [\dot{Q}_{E,S} + (CON_8 + n_{S2} \cdot CON_{S2}) \cdot (T_{Node1} - T_{E,S}) + CON_9 \cdot (T_{RTG} - T_{E,S}) - \epsilon_{E,S} \cdot \sigma_b \cdot S_{E,S} \cdot T_{E,S}^4] = 0$$

#### Distribution Node 1:

$$4 \cdot [CON_7 \cdot (T_{Chas} - T_{Node_1}) + CON_8 \cdot (T_{E,S} - T_{Node_1}) + CON_10 \cdot (T_{Node_2} - T_{Node_1})] = 0$$

#### Drive Engine:

$$4 \cdot [\dot{Q}_{E,D} + (CON_{11} + n_{S3} \cdot CON_{S3}) \cdot (T_{Node_2} - T_{E,D}) + CON_{12} \cdot (T_{RTG} - T_{E,D}) - \epsilon_{E,D} \cdot \sigma_b \cdot S_{E,D} \cdot T_{E,D}^4] = 0$$

#### <u>Distribution Node 2:</u>

$$4 \cdot [CON_{10} \cdot (T_{Node_1} - T_{Node_2}) + CON_{11} \cdot (T_{E,D} - T_{Node_2}) + CON_{13} \cdot (T_{Ground} - T_{Node_2})] = 0$$

#### B.2 Heat conductance

#### B.3 Heat switch

The charateristic of the switch conductance depends on the mean temperature  $T_M$ , shown by measurements in cite. As this temperature wont be calculated in the analysis, the corresponding component temperature  $T_C$  shall be used. In order to describe the characteristic, it was diveded in three sections, Figure B.2. The temperature where the disk decouples is  $T_{toggle} = 108K$  and not applicable for the current application. By reducing the heigt, the toggle temperature can be increased and the characteristic can be shifted to higher termperatures ("'to the right"'). It was assumed, that the gradients of section 2 and 3 as well as the temperature range of section 2 keep constant. The axis interseption is a function the toggle temperature  $(a_2 = f(\Delta T_{toggle}), \Delta T_{Toggle} = T_{new} - T_{old}|_{toggle})$ .

Section	Temperature range	Hear conductance $C_t = \frac{1}{R_t}$
1	$T_{toggle} > T_C$	$C_{t,1} = 16.4 \cdot 10^{-3} \frac{W}{m^2 K} = const.$
2	$T_{toggle} \le T_C < T_1$	$C_{t,2}(T_C) = a_{1.1} \cdot T_C + a_{1.2}$
		$a_{1.1} = +1.272 \cdot 10^{-3} \frac{W}{m^2 K^2}$
		$a_{1.2} = -1.272 \cdot 10^{-3} \frac{W}{m^2 K^2} \cdot \Delta T_{Toggle} - 0.117 \frac{W}{m^2 K}$
3	$T_C$ > $T_1$	$C_{t,3}(T_C) = a_{2.1} \cdot T_C + a_{2.2}$
		$a_{2.1} = +333 \cdot 10^{-6} \frac{W}{m^2 K^2}$
		$a_{2.2} = -333 \cdot 10^{-6} \frac{W}{m^2 K^2} \cdot \Delta T_{Toggle} - 28.3 \cdot 10^{-3} \frac{W}{m^2 K}$

Table B.2: Sections and range of the switch characteristic.

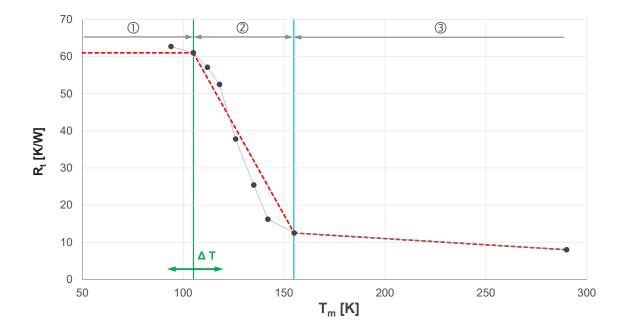


Figure B.2: Switch characteristic diveded in sections.

#### B.4 Rover absorptivity

The amount of solar radiation onto the rover depends on the sun altitude, see autoref(fig:tcsabsop).

View Factor 1, Inclination:  $\varphi_1 = \cos(\lambda) \cdot \cos(\delta)$ 

View Factor 2, Declination:  $\varphi_2 = \cos(90^\circ - \lambda) \cdot \cos(\delta)$ 

#### B.5 Values

	Tempera	ture limits in $[{}^{\circ}C]$
Component	min.	max.
Command & Data Handling		
Transmitter	-10	50
Receiver	-30	70
PCDU	-40	60
Battery	-35	60
Camera	-40	70
Objektive	-40	71
Steering Engine	-30	100
Steering Gear	-30	85
Drive Engine	-40	100
Drive Gear	-40	100

Table B.3: Temperatur limits of the rover components.

The Europas ground temperature varies at the equator between  $T_{e,min} = 80K$  and  $T_{e,max} = 130K$ , depending on the sun inclination. The temperature at the pole is  $T_{Pole} = 50K$ , cite(europa). It was assumed, that the ground temperature depends on the sun altitude. A trigonometrical interpolation was defined as follows.

$$T_{Ground}(\lambda, \delta) = T_{Pole} + \cos(\delta) \cdot \left[ (T_{e,min} + \cos(\lambda) \cdot (T_{e,max} - T_{e,min})) - T_{Pole} \right]$$

For  $\lambda$  and  $\delta$  see Subsection B.4.

	Emisivity [-]		Absorp	Source	
Surface finishing	min.	max.	min.	max.	
Aluminium, polished	0.0	045			
Sand blasted alloy					
White paint					

Table B.4: Minimum and maximum of surface emisivity and absorptivity values.

Material	Nominal	Used	Source
Aerogel	0.002 - 0.05	0.05	
Aluminium	236	236	
$Copper^{2)}$	400	360	
$Steel^{2)}$	21 - 50	21	

 $<sup>^{1)}</sup>$  a reduction of 10% was considered

Table B.5: Heat conductivity in  $\frac{W}{mK}$ 

Part Description Value $m^2$	Part	Description	Value $m^2$
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Table B.6: Radiation surface of components.

 $<sup>^{2)}</sup>$  depends on the alloying component, a conservative value was choosen

#### C Radiation

In this chapter, detailed calculations are performed on which Section 4.7 is based on. All calculations and figures in Section C are performed with SPENVIS unless otherwise stated. In order to simulate the radiation on Europa an orbit around Jupiter is simulated with the orbit parameters of Europa with a total mission duration of 30 days. The chosen parameters were an perijove altitude of 664,862 km, an apojove altitude of 676,938 km, and an inclination of 0.47.

#### C.1 Jupiters Radiation Environment

In order to compare the radiation environment around Jupiter and the radiation environment around Earth the following trapped radiation models were used: for Jupiter the D&G83+Salammbo proton model and D&G83+GIRE+SalammboE electron model was used; for Earth the AP-8 proton model and the AE-8 electron model was used.

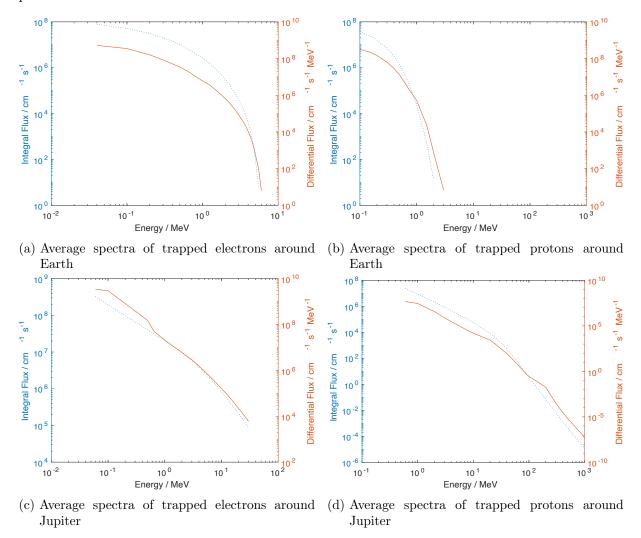


Figure C.3: Average trapped proton and electron fluxes on an orbit around earth at 25,000 km, through the outer Van Allen radiation belt, and on Europa's orbit around Jupiter.

#### C.2 Radiation Exposures

In order to simulate the TID for different radiation protections the Geant4 tool Multi-Layered Shielding Simulation (MULASSIS) is used. As target material silicon is selected with a thickness of 1  $\mu$ m. As shape a planar slap is selected because of the ice ground on one side of the rover.

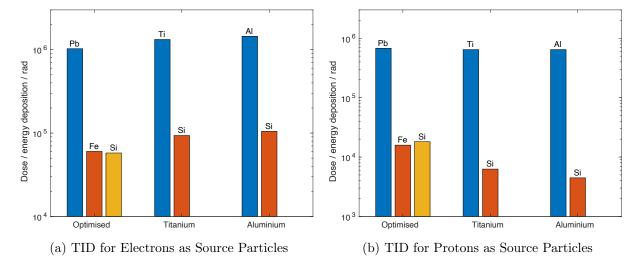


Figure C.4: TID of aluminium, titanium, and the optimised radiation structure shown in Table 4.5 with a weight target of all three structures of  $0.5 \text{ g/cm}^2$  over 30 days of exposure on Europa.

Table C.7: Used components and the respective radiation tolerance and location

Components	Rated TID	Exposed TID	Location
Electric Motors	-	$< 205 \mathrm{\ krad}$	locomotion housing
Harness	-	$<98~\mathrm{krad}$	chassis
Stereo Vision Cams	40	$< 31~\mathrm{krad}$	camera housing
OBC	1000	$< 17 \mathrm{\ krad}$	E-Bay
PCDU	20	$<17~\rm krad$	E-Bay

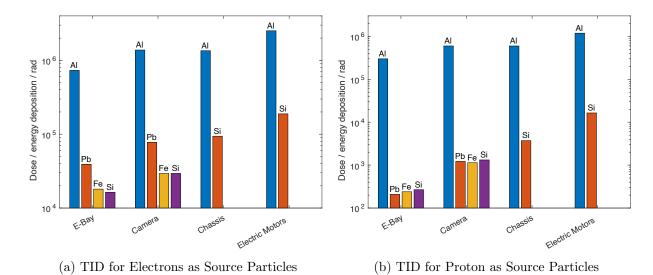


Figure C.5: TID for different compartments as seen in Figure 4.6. The E-Bay is shielded by 4 mm aluminium, 0.415 mm lead, and 0.033 mm iron; the camera compartment by 2 mm aluminium, 0.415 mm lead, and 0.033 mm iron; the chassis by 2 mm aluminium; the electric motors by 1 mm aluminium.

#### C.3 Improvements

All simulations of the improvements introduced in Subsection 4.7.3 are performed in the same way as in Subsection C.2.

In Figure C.6, the TID over 30 days within the E-Bay is shown. If all components with an radiation resistance under 43.27 krad are shielded individually, the additional shielding structure around the E-Bay can be removed and the aluminium structure would be sufficient.

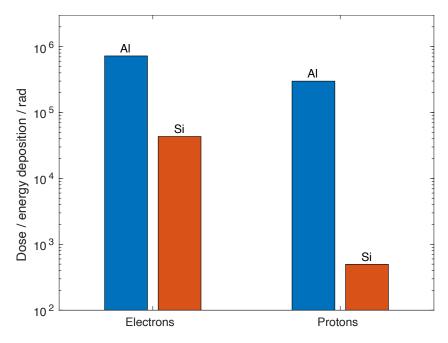


Figure C.6: TID with 4 mm Al shielding over a mission duration of 30 days

The resulting mass savings can be calculated with Equation 5.1 with  $m^*$  as the specific weight of the radiation protection and N as the amount of components within the E-Bay with a radiation

resistance under  $43.27~\rm krad$  as of Table C.7.

$$\Delta m = SA_{\text{E-Bay}} \cdot m_{\text{Shielding}}^* - \sum_{n=0}^{N} SA_{\text{Component, n}} \cdot m_{\text{Shielding}}^*$$
 (5.1)

With inserted values this results in a mass saving of  $\Delta m = 736.2$  g.

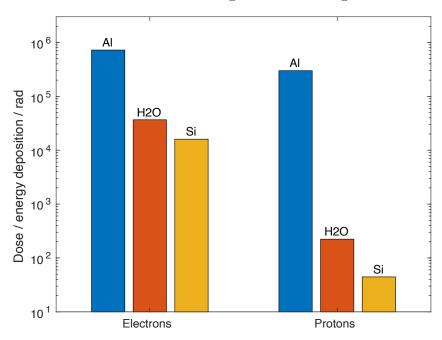


Figure C.7: TID with 4 mm Al shielding and 1 cm of Water over a mission duration of 30 days