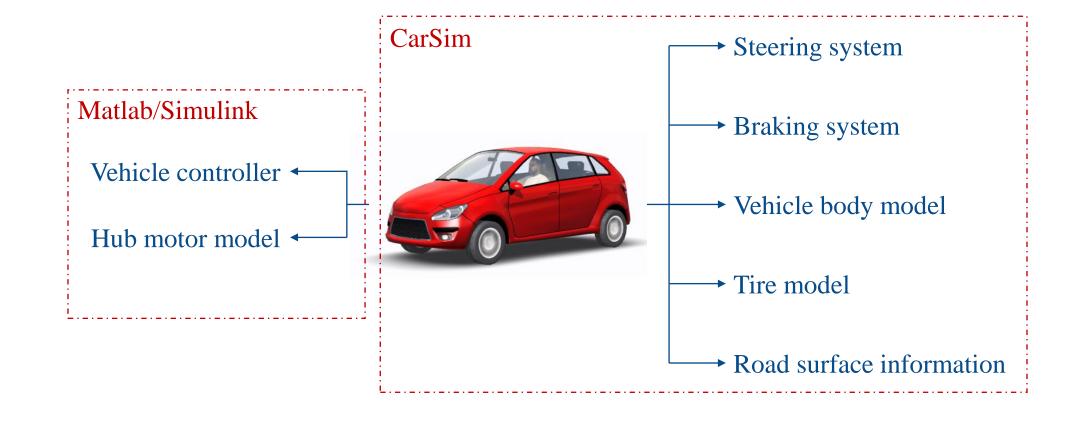


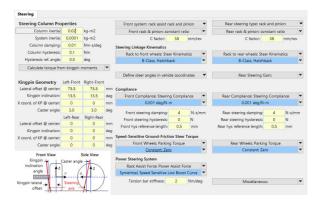
Collaborative Control of Wire-Controlled Chassis Subsystems Based on Game Theory



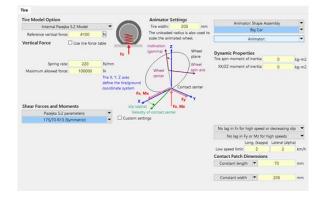
Overall framework - B-Class, Hatchback



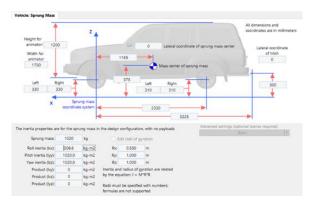
Using built-in components of Carsim



Steering system



Tire model



Vehicle body model



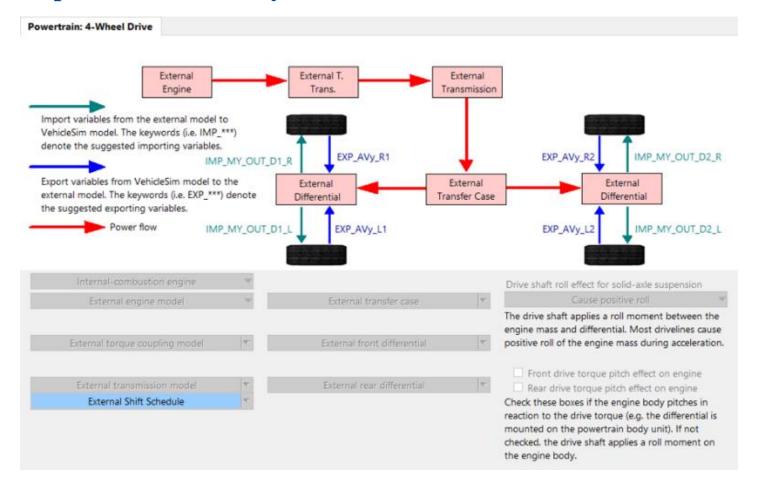
Suspension model

Dynamics modeling

controller design simulation analysis

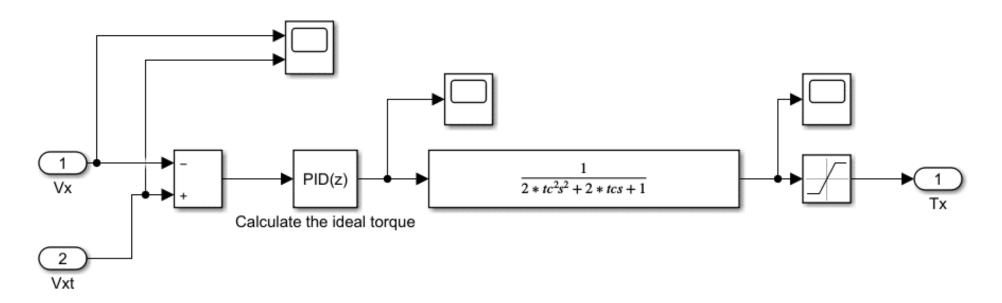


Main modification part - Powertrain system



Electric vehicle (front wheel steering angle and four-wheel torque)

Main Modification Section - Powertrain System



$$K(s) = \frac{T_d}{T_{ld}} = \frac{1}{1 + 2\xi s + 2\xi^2 s^2}$$

$$\xi = 0.001$$



Primary inputs and outputs

Interface	Interface Name	Interface Definitions
Input Interface	IMP_MYUSM_L1	Left front wheel input torque
	IMP_MYUSM_L2	Left rear wheel input torque
	IMP_MYUSM_R1	Right front wheel input torque
	IMP_MYUSM_R2	Right rear wheel input torque
	IMP_STEER_L1	Left front wheel input steering angle
	IMP_STEER_R1	Right front wheel input steering angle
Output Interface	Beta-Vehicle slip angle	Vehicle center of mass lateral deviation angle
	Vx-Longitudinal speed	Vehicle longitudinal velocity
	Vy-Lateral speed	Vehicle lateral velocity
	AVz-Yaw rate	Vehicle yaw rate
	AVy_L1-Wheel L1 spin	Left front wheel speed
	AVy_R1-Wheel R1 spin	Right front wheel speed
	Fz_L1	Left front wheel vertical tire force
	Fz_L2	Left rear wheel vertical tire force
	Fz_R1	Right front wheel vertical tire force
	Fz_R2	Right rear wheel vertical tire force

Non-cooperative game theory

Linear control state equation:

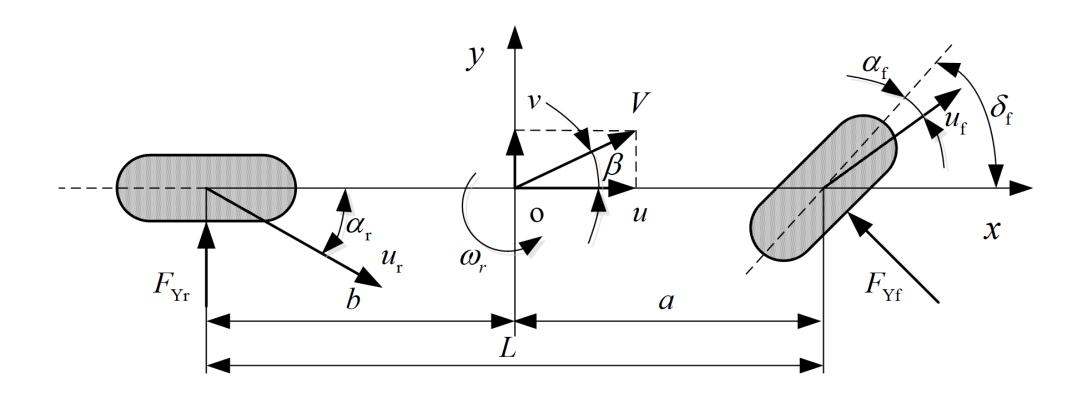
$$\dot{x}(t) = Ax(t) + B_1 u_1(t) + B_2 u_2(t)$$

Cost function:

$$\begin{cases} J_{1} = \int_{0}^{T} \{x^{T}(t)Q_{1}x(t) + u_{1}^{T}(t)R_{11}u_{1}(t) + u_{2}^{T}(t)R_{12}u_{2}(t)\}dt \\ J_{2} = \int_{0}^{T} \{x^{T}(t)Q_{2}x(t) + u_{1}^{T}(t)R_{21}u_{1}(t) + u_{2}^{T}(t)R_{22}u_{2}(t)\}dt \end{cases}$$

Nash equilibrium control strategy:

$$\begin{cases}
J_1(u_{F1}, u_{F2}) \le J_2(u_1, u_{F2}) \\
J_2(u_{F1}, u_{F2}) \le J_2(u_{F1}, u_{2})
\end{cases}$$



• Differential equations for the two-degree-of-freedom model:

$$\begin{cases} (k_f + k_r)\beta + \frac{1}{V_x}(ak_f - bk_r)\omega - k_f\delta_f = m(\dot{V}_y + V_x\omega) \\ (ak_f - bk_r)\beta + \frac{1}{V_x}(a^2k_f - b^2k_r)\omega - ak_f\delta_f = I_z\dot{\omega} \end{cases}$$

• State-space equations for the two-degree-of-freedom model:

$$\begin{bmatrix} \dot{\beta} \\ \dot{\omega} \end{bmatrix} = \begin{bmatrix} -\frac{k_f + k_r}{mV_x} & -\frac{ak_f - bk_r}{mV_x^2} - 1 \\ -\frac{ak_f - bk_r}{I_z} & -\frac{a^2k_f + b^2k_r}{I_zV_x} \end{bmatrix} \begin{bmatrix} \beta \\ \omega \end{bmatrix} + \begin{bmatrix} \frac{k_f}{mV_x} \\ \frac{ak_f}{I_z} \end{bmatrix} \delta_f$$

For the sake of convenience in research, redefine the state-space equations:

$$\begin{bmatrix} \Delta \dot{\beta} \\ \Delta \dot{\omega} \end{bmatrix} = \begin{bmatrix} -\frac{k_f + k_r}{mV_x} & -\frac{ak_f - bk_r}{mV_x^2} - 1 \\ -\frac{ak_f - bk_r}{I_z} & -\frac{a^2k_f + b^2k_r}{I_zV_x} \end{bmatrix} \Delta \beta + \begin{bmatrix} \frac{k_f}{mV_x} \\ \frac{ak_f}{I_z} \end{bmatrix} \Delta \delta_f + \begin{bmatrix} 0 \\ \frac{1}{I_z} \end{bmatrix} \Delta M$$

$$\begin{cases} \Delta \beta = \beta - \beta_{des} \\ \Delta \omega = \omega - \omega_{des} \end{cases}$$

$$\begin{cases} \partial \beta = \beta - \beta_{des} \\ \omega_{des} = \frac{b - \frac{amV_x^2}{k_r(a+b)}}{(a+b)(1 + K_{us}V_x^2)} \delta_f \\ \omega_{des} = \frac{V_x}{(a+b)(1 + K_{us}V_x^2)} \delta_f \\ K_{us} = \frac{m(bk_2 - ak_1)}{(a+b)^2k_fk_r} \end{cases}$$

• For ease of computation, simplify the state-space equations:

$$\begin{bmatrix} \Delta \dot{\beta} \\ \Delta \dot{\omega} \end{bmatrix} = \begin{bmatrix} -\frac{k_f + k_r}{mV_x} & -\frac{ak_f - bk_r}{mV_x^2} - 1 \\ -\frac{ak_f - bk_r}{I_z} & -\frac{a^2k_f + b^2k_r}{I_zV_x} \end{bmatrix} \begin{bmatrix} \Delta \beta \\ \Delta \omega \end{bmatrix} + \begin{bmatrix} \frac{k_f}{mV_x} \\ \frac{ak_f}{I_z} \end{bmatrix} \Delta \delta_f + \begin{bmatrix} 0 \\ \frac{1}{I_z} \end{bmatrix} \Delta M$$

$$\dot{x} = Ax_n + B_1 u_1 + B_2 u_2$$

Upper-level controller design - Cost function

Define the game model based on the cost function:

$$\begin{cases} J_{1}(u_{1}, u_{2}) = \int_{0}^{\infty} \frac{1}{2} \{x_{n}^{T}(t)Q_{1}x_{n}(t) + u_{1}^{T}(t)R_{11}u_{1}(t) + u_{2}^{T}(t)R_{12}u_{2}(t)\}dt \\ J_{2}(u_{1}, u_{2}) = \int_{0}^{\infty} \frac{1}{2} \{x_{n}^{T}(t)Q_{2}x_{n}(t) + u_{1}^{T}(t)R_{21}u_{1}(t) + u_{2}^{T}(t)R_{22}u_{2}(t)\}dt \end{cases}$$

The cost function satisfies the following conditions:

$$\begin{cases} J_1(u_1^*, u_2^*) \le J_1(u_1, u_2^*) \\ J_2(u_1^*, u_2^*) \le J_2(u_1^*, u_2) \end{cases}$$

Upper-level controller design - Hamiltonian equation

Establish the Hamiltonian equation

$$\begin{cases} H_1 = \frac{1}{2} (x_n^T Q_1 x_n + u_1^T R_{11} u_1 + u_2^T R_{12} u_2) + \lambda_1^T (A x_n + B_1 u_1 + B_2 u_2) \\ H_2 = \frac{1}{2} (x_n^T Q_2 x_n + u_1^T R_{21} u_1 + u_2^T R_{22} u_2) + \lambda_2^T (A x_n + B_1 u_1 + B_2 u_2) \end{cases}$$

Construct the extremum condition equation

$$\begin{cases}
\frac{\partial H_1}{\partial x_n} = R_{11}u_1 + B_1^T \lambda_1 = 0 \\
\frac{\partial H_2}{\partial x_n} = R_{22}u_2 + B_2^T \lambda_2 = 0
\end{cases} \qquad \text{solve} \qquad \begin{cases}
u_1 = -R_{11}^{-1} B_1^T \lambda_1 \\
u_2 = -R_{22}^{-1} B_2^T \lambda_2
\end{cases}$$

Upper-level controller design - Hamiltonian equation

Substitute the simplified state-space equations

$$\begin{cases} u_1 = -R_{11}^{-1} B_1^T \lambda_1 \\ u_2 = -R_{22}^{-1} B_2^T \lambda_2 \end{cases} \longrightarrow \dot{x} = Ax_n + B_1 u_1 + B_2 u_2 \longrightarrow \dot{x}_n = Ax_n - B_1 R_{11}^{-1} B_1^T \lambda_1 - B_2 R_{22}^{-1} B_2^T \lambda_2$$

According to the optimization control conditions

$$\begin{cases} \dot{\lambda}_1 = -\frac{\partial H_1}{\partial x_n} = -A^T \lambda_1 - Q_1 x_n \\ \dot{\lambda}_2 = -\frac{\partial H_2}{\partial x_n} = -A^T \lambda_2 - Q_2 x_n \end{cases}$$

Write it in terms of the co-state matrix

$$\begin{bmatrix} \dot{x}_n \\ \dot{\lambda}_1 \\ \dot{\lambda}_2 \end{bmatrix} = \begin{bmatrix} A & -B_1 R_{11}^{-1} B_1^T & -B_2 R_{22}^{-1} B_2^T \\ -Q1 & -A^T & 0 \\ -Q2 & 0 & -A^T \end{bmatrix} \cdot \begin{bmatrix} x_n \\ \lambda_1 \\ \lambda_2 \end{bmatrix}$$

Upper-level controller design - Hamiltonian equation

There is a relationship between state variables

$$\begin{bmatrix} \dot{x}_n \\ \dot{\lambda}_1 \\ \dot{\lambda}_2 \end{bmatrix} = \begin{bmatrix} A & -B_1 R_{11}^{-1} B_1^T & -B_2 R_{22}^{-1} B_2^T \\ -Q1 & -A^T & 0 \\ -Q2 & 0 & -A^T \end{bmatrix} \begin{bmatrix} x_n \\ \lambda_1 \\ \lambda_2 \end{bmatrix} \longrightarrow \begin{cases} \lambda_1 = G_1 x_n \\ \lambda_2 = G_2 x_n \end{cases}$$

Substitute into the extremum condition equation for a solution

$$\begin{cases} u_{1} = -R_{11}^{-1} B_{1}^{T} \lambda_{1} \\ u_{2} = -R_{22}^{-1} B_{2}^{T} \lambda_{2} \end{cases} \qquad \begin{cases} u_{1}^{*} = -R_{11}^{-1} B_{1}^{T} G_{1} x_{n} \\ u_{2}^{*} = -R_{22}^{-1} B_{2}^{T} G_{2} x_{n} \end{cases}$$

 G_1 and G_2 are solutions to the following coupled Riccati equations

$$\begin{cases} (A - T_2 G_2)^T G_1 + G_1 (A - T_2 G_2) - G_1 T_1 G_1 + Q_1 + G_2 T_{12} G_2 = 0 \\ (A - T_1 G_1)^T G_2 + G_2 (A - T_1 G_1) - G_2 T_2 G_2 + Q_2 + G_1 T_{21} G_1 = 0 \end{cases}$$

Expressions for T_{ij} and T_i are as follows

$$\begin{cases}
T_{i} = B_{i} R_{ii}^{-1} B_{i}^{T} & i = 1, 2 \\
T_{ij} = B_{i} R_{ii}^{-1} R_{ji} R_{ii}^{-1} B_{i}^{T} & i, j = 1, 2 \& i \neq j
\end{cases}$$

The game process is specifically reflected in the process of solving the coupled Riccati equations.

Discretize and reconstruct the equations as follows:

Obtain the initial solutions for the decoupled Riccati equations through the following equations:

$$\begin{cases} A^{T}G_{1}^{0} + G_{1}^{0}A + Q_{1} + G_{1}^{0}T_{11}G_{1}^{0} = 0 \\ A^{T}G_{2}^{0} + G_{2}^{0}A + Q_{2} + G_{2}^{0}T_{11}G_{2}^{0} = 0 \end{cases}$$

Treating G^0 as a constant

$$\begin{cases}
(A - T_2 G_2^{c})^T G_1^{c+1} + G_1^{c+1} A - T_2 G_2^{c} \\
(A - T_1 G_1^{c})^T G_2^{c+1} + G_2^{c+1} A - T_1 G_1^{c} \\
(A - T_1 G_1^{c})^T G_2^{c+1} + G_2^{c+1} A - T_1 G_1^{c} \\
A - T_1 G_1^{c}
\end{cases} - G_1^{c+1} T_1 G_1^{c+1} + Q_1 + G_2^{c} T_{12} G_2^{c} \\
+ Q_2 + G_1^{c} T_{21} G_1^{c} = 0$$

$$\begin{cases}
A_1^T = (A - T_2 G_2^{c})^T \\
A_2^T = (A - T_1 G_1^{c})^T
\end{cases} \qquad
\begin{cases}
Q_1' = Q_1 + G_2^{c} T_{12} G_2^{c} \\
Q_2' = Q_2 + G_1^{c} T_{21} G_1^{c}
\end{cases}$$

Transform the equation into the same form as the strong solution

$$\begin{cases} A_1^T G_1^{c+1} + G_1^{c+1} A_1 + Q_1' + G_1^{c+1} T_{11} G_1^{c+1} = 0 \\ A_2^T G_2^{c+1} + G_2^{c+1} A_2 + Q_2' + G_2^{c+1} T_{22} G_2^{c+1} = 0 \end{cases}$$

Iteration termination condition:

Computed results:

$$\begin{cases} u_1^* = -R_{11}^{-1} B_1^T G_1^{c+1} x_n \\ u_2^* = -R_{22}^{-1} B_2^T G_2^{c+1} x_n \end{cases}$$

Corresponding results:

$$\begin{cases} \Delta \delta_f = u_1^* \\ \Delta M = u_2^* \end{cases}$$

$$\begin{cases} \min \varepsilon = \sum_{j} \frac{T_{xj}^{2}}{(\mu F_{zj})^{2} R^{2}} \\ s.t. \quad T_{x} = \sum_{j} T_{xj} \\ \Delta M = \frac{t_{r}}{2} \left(\frac{T_{xfr}}{R} - \frac{T_{xfl}}{R} \right) + \frac{t_{r}}{2} \left(\frac{T_{xrl}}{R} - \frac{T_{xrr}}{R} \right) \\ 0 \le T_{xj} \le \min(T_{m_{max}}, \mu F_{zj} R) \end{cases}$$

$$\min \varepsilon = \sum_{j} \frac{T_{xj}^{2}}{(\mu F_{zj})^{2} R^{2}}$$

$$s.t. \quad T_{x} = \sum_{j} T_{xj}$$

$$\Delta M = \frac{t_{r}}{2} \left(\frac{T_{xfr}}{R} - \frac{T_{xfl}}{R}\right) + \frac{t_{r}}{2} \left(\frac{T_{xrl}}{R} - \frac{T_{xrr}}{R}\right)$$

$$0 \le T_{xj} \le \min(T_{m_{-}\max}, \mu F_{zj}R)$$

With the objective of minimizing tire utilization

$$\begin{cases}
\min \varepsilon = \sum_{j} \frac{T_{xj}^{2}}{(\mu F_{zj})^{2} R^{2}} \\
s.t. \quad T_{x} = \sum_{j} T_{xj}
\end{cases}$$

$$\Delta M = \frac{t_{r}}{2} \left(\frac{T_{xfr}}{R} - \frac{T_{xfl}}{R} \right) + \frac{t_{r}}{2} \left(\frac{T_{xrl}}{R} - \frac{T_{xrr}}{R} \right)$$

$$0 \le T_{xj} \le \min(T_{m_{-}\max}, \mu F_{zj}R)$$

Vehicle propulsion total torque = Sum of torques from all four drive wheels

$$\begin{cases} \min \varepsilon = \sum_{j} \frac{T_{xj}^{2}}{(\mu F_{zj})^{2} R^{2}} \\ s.t. \quad T_{x} = \sum_{j} T_{xj} \\ \Delta M = \frac{t_{r}}{2} \left(\frac{T_{xfr}}{R} - \frac{T_{xfl}}{R} \right) + \frac{t_{r}}{2} \left(\frac{T_{xrl}}{R} - \frac{T_{xrr}}{R} \right) \\ 0 \le T_{xj} \le \min(T_{m-\max}, \mu F_{zj}R) \end{cases}$$
Additional total torque should also satisfy allocation conditions

$$\begin{cases} \min \varepsilon = \sum_{j} \frac{T_{xj}^{2}}{(\mu F_{zj})^{2} R^{2}} \\ s.t. \quad T_{x} = \sum_{j} T_{xj} \\ \Delta M = \frac{t_{r}}{2} \left(\frac{T_{xfr}}{R} - \frac{T_{xfl}}{R} \right) + \frac{t_{r}}{2} \left(\frac{T_{xrl}}{R} - \frac{T_{xrr}}{R} \right) \\ 0 \le T_{xj} \le \min(T_{m_{-}\max}, \mu F_{zj}R) \end{cases}$$
Subject to peak torque limits

$$\begin{cases} \min \varepsilon = \sum_{j} \frac{T_{xj}^{2}}{(\mu F_{zj})^{2} R^{2}} \\ s.t. \quad T_{x} = \sum_{j} T_{xj} \end{cases}$$

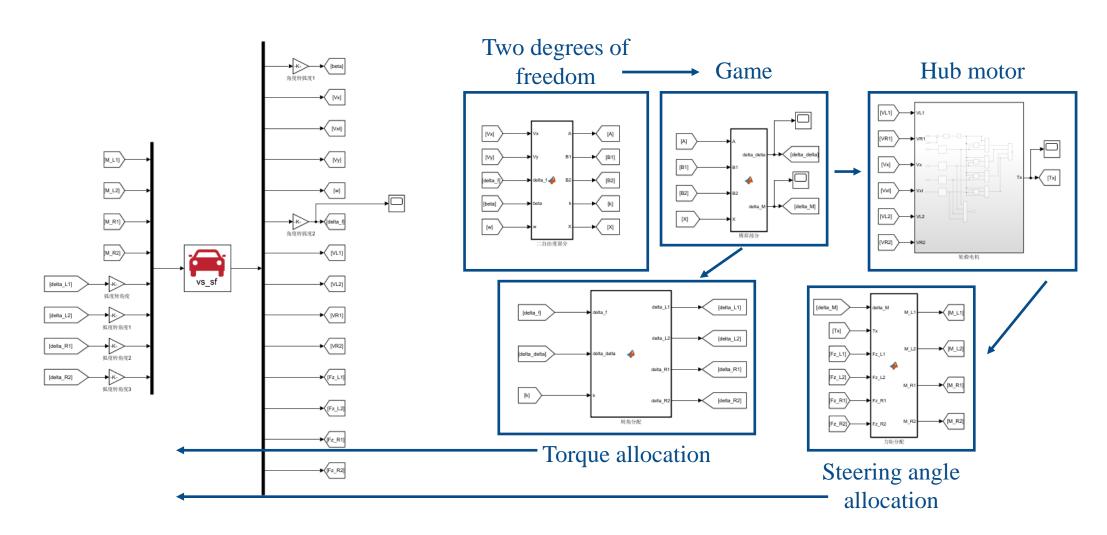
$$\Delta M = \frac{t_{r}}{2} \left(\frac{T_{xfr}}{R} - \frac{T_{xfl}}{R} \right) + \frac{t_{r}}{2} \left(\frac{T_{xrl}}{R} - \frac{T_{xrr}}{R} \right)$$

$$0 \le T_{xj} \le \min(T_{m_{-}\max}, \mu F_{zj}R)$$

- A typical quadratic programming problem.
- Solve it using MATLAB toolbox.



Building a Simulink model

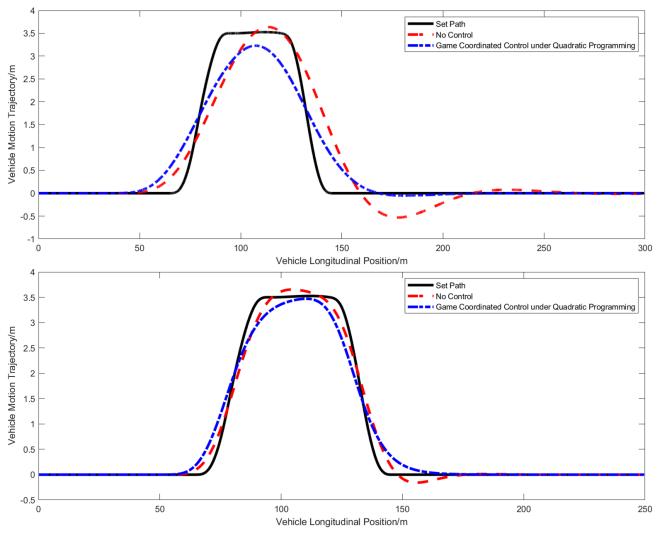




Vehicle trajectory tracking performance

- Vehicle speed: 120 km/h
- Road surface adhesion coefficient: 0.85

- Vehicle speed: 60 km/h
- Road surface adhesion coefficient: 0.3

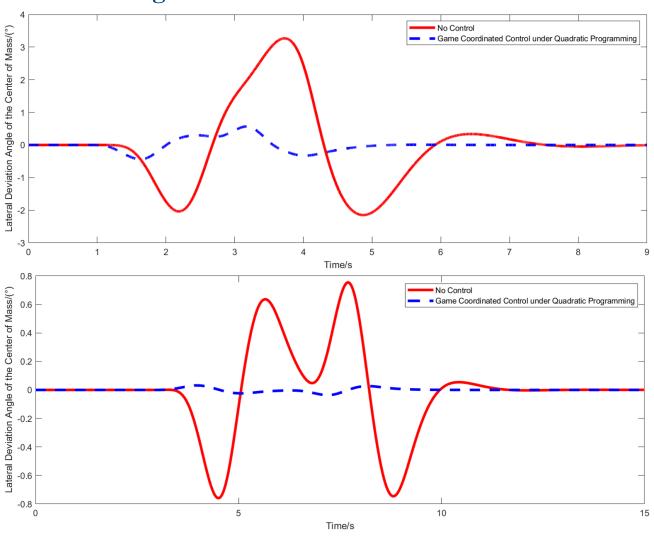




Simulation results for lateral deviation angle of the center of mass

- Vehicle speed: 120 km/h
- Road surface adhesion coefficient: 0.85

- Vehicle speed: 60 km/h
- Road surface adhesion coefficient: 0.3

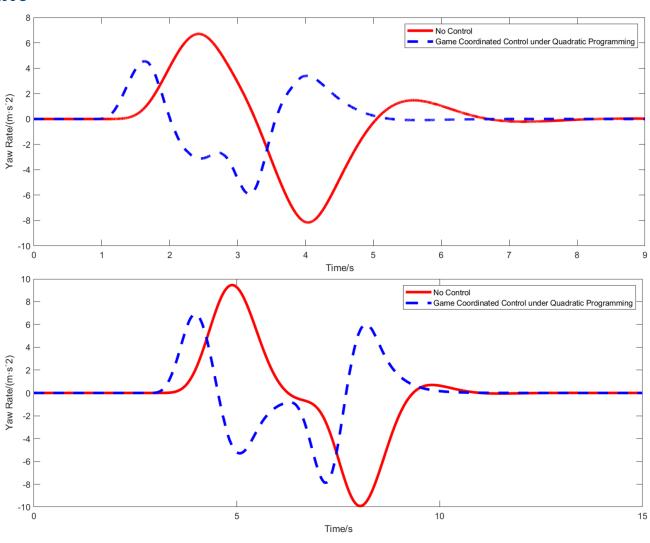




Simulation results for yaw rate

- Vehicle speed: 120 km/h
- Road surface adhesion coefficient: 0.85

- Vehicle speed: 60 km/h
- Road surface adhesion coefficient: 0.3

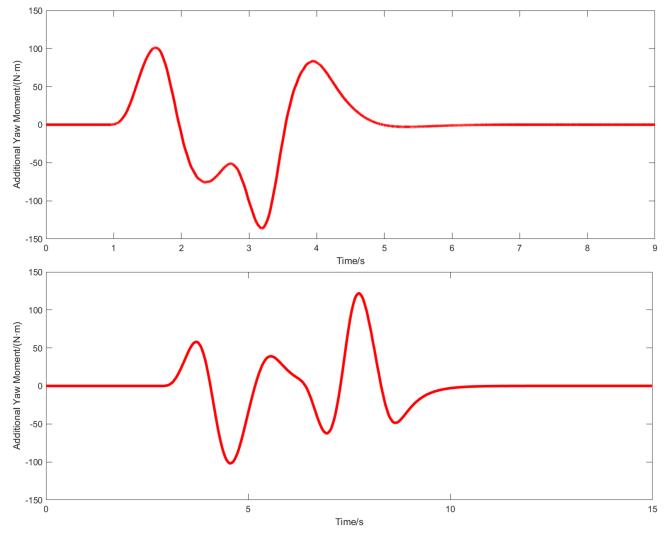




Simulation results for additional yaw moment

- Vehicle speed: 120 km/h
- Road surface adhesion coefficient: 0.85

- Vehicle speed: 60 km/h
- Road surface adhesion coefficient: 0.3

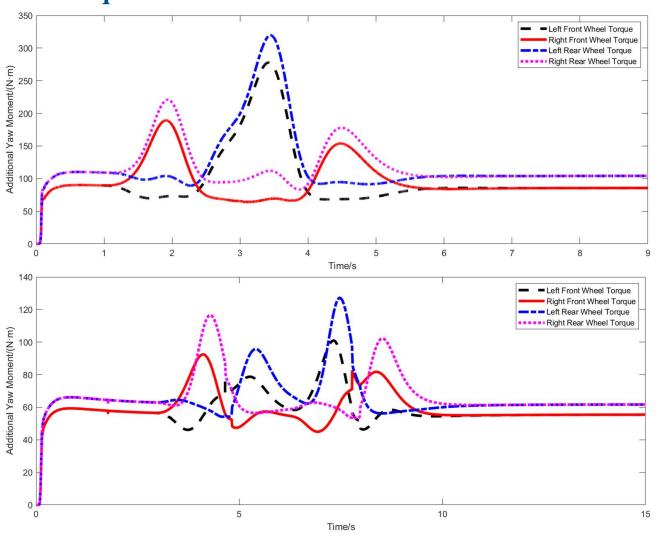




Simulation results for four-wheel torque

- Vehicle speed: 120 km/h
- Road surface adhesion coefficient: 0.85

- Vehicle speed: 60 km/h
- Road surface adhesion coefficient: 0.3

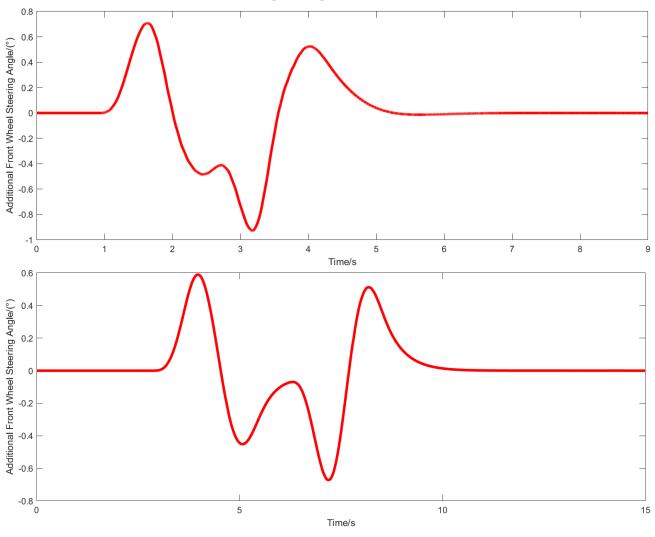




Simulation results for additional front wheel steering angle

- Vehicle speed: 120 km/h
- Road surface adhesion coefficient: 0.85

- Vehicle speed: 60 km/h
- Road surface adhesion coefficient: 0.3





Simulation results for front wheel steering angle

- Vehicle speed: 120 km/h
- Road surface adhesion coefficient: 0.85

- Vehicle speed: 60 km/h
- Road surface adhesion coefficient: 0.3

