

## COORDINATE SYSTEM

### FRAME OF REFERENCE

Right hand  $\diamond$  left hand

$$\begin{cases} \theta_x^L = -\theta_x^R, d_x^L = d_x^R \\ \theta_y^L = -\theta_y^R, d_y^L = d_y^R, \text{ plug in to get } R^L, d^L \\ \theta_z^L = \theta_z^R, d_z^L = -d_z^R \end{cases}$$

### ROTATION

#### Rotation Matrix

Orthonormal, invertible,  $R^T = R^{-1}$

#### Euler Angles

With respect to local frame, post multiplication

Euler Angles  $\diamond$  Rotation Matrix

See appendix 1 for more

#### Six configurations of Euler angles

As for the inverse, take y-x-z for example:

For generally case ( $\cos \theta_x \neq 0$ ), there are 2 solutions:

$$\begin{cases} \theta_x = \text{atan2}(-r_{23}, \sqrt{1 - r_{23}^2}) \\ \theta_y = \text{atan2}(r_{12}, r_{33}) \\ \theta_z = \text{atan2}(r_{21}, r_{22}) \end{cases}, \begin{cases} \theta_x = \text{atan2}(-r_{23}, -\sqrt{1 - r_{23}^2}) \\ \theta_y = \text{atan2}(-r_{12}, -r_{33}) \\ \theta_z = \text{atan2}(-r_{21}, -r_{22}) \end{cases}$$

#### Axis-Angle

Axis  $k$  and angle  $\theta$  are defined in the frame before rotation. Though  $k$  has the same coordinates in both frames.

Axis-Angle  $\diamond$  Rotation Matrix

$$\begin{bmatrix} k_x^2 v_\theta + c_\theta & k_x k_y v_\theta - k_z s_\theta & k_x k_z v_\theta + k_y s_\theta \\ k_x k_y v_\theta + k_z s_\theta & k_y^2 v_\theta + c_\theta & k_y k_z v_\theta - k_x s_\theta \\ k_x k_z v_\theta - k_y s_\theta & k_y k_z v_\theta + k_x s_\theta & k_z^2 v_\theta + c_\theta \end{bmatrix}$$

where  $v_\theta = 1 - \cos \theta$ , and the inverse is:

$$\theta = \cos^{-1}\left(\frac{r_{11} + r_{22} + r_{33} - 1}{2}\right), k = \frac{1}{2\sin \theta} \begin{bmatrix} r_{32} - r_{23} \\ r_{13} - r_{31} \\ r_{21} - r_{12} \end{bmatrix}$$

### Quaternions

Quaternion  $\diamond$  Axis Angle

axis:  $\hat{n} = [n_x, n_y, n_z]^T$ , angle:  $\theta$

$$Q = \left(\cos\left(\frac{\theta}{2}\right), n_x \sin\left(\frac{\theta}{2}\right), n_y \sin\left(\frac{\theta}{2}\right), n_z \sin\left(\frac{\theta}{2}\right)\right)$$

Quaternion  $\diamond$  Rotation Matrix

$1 - 2q_y^2 - 2q_z^2$	$2qxq_y - 2qzq_w$	$2qxq_z + 2qyq_w$
$2qxq_y + 2qzq_w$	$1 - 2q_x^2 - 2q_z^2$	$2qyq_z - 2qxq_w$
$2qxq_z - 2qyq_w$	$2qyq_z + 2qxq_w$	$1 - 2q_x^2 - 2q_y^2$

trace = m00 + m11 + m22

if trace > 0

$S = \sqrt{\text{tr} + 1.0} * 2$ ; //  $S = 4 * q_w$

$q_w = 0.25 * S$ ;

$q_x = (m21 - m12) / S$ ;

$q_y = (m02 - m20) / S$ ;

$q_z = (m10 - m01) / S$ ;

else if m00 > m11 and m00 > m22

$S = \sqrt{1.0 + m00 - m11 - m22} * 2$ ; //  $S = 4 * q_x$

$q_w = (m21 - m12) / S$ ;

$q_x = 0.25 * S$ ;

$q_y = (m01 + m10) / S$ ;

$q_z = (m02 + m20) / S$ ;

else if m11 > m22

$S = \sqrt{1.0 + m11 - m00 - m22} * 2$ ; //  $S = 4 * q_y$

$q_w = (m02 - m20) / S$ ;

$q_x = (m01 + m10) / S$ ;

$q_y = 0.25 * S$ ;

$q_z = (m12 + m21) / S$ ;

else

$S = \sqrt{1.0 + m22 - m00 - m11} * 2$ ; //  $S = 4 * q_z$

$q_w = (m10 - m01) / S$ ;

$q_x = (m02 + m20) / S$ ;

$q_y = (m12 + m21) / S$ ;

$q_z = 0.25 * S$ ;

### More on Quaternions

Identity quaternion:  $Q_I = (1, 0, 0, 0)$

Conjugate quaternion for  $Q$ :  $Q^* = (q_0, -q_1, -q_2, -q_3)$

Multiplying two quaternions:

$X = (x_0, \vec{x}), Y = (y_0, \vec{y})$

$$XY = x_0 y_0 - \vec{x}^T \vec{y} + x_0 \vec{y} + y_0 \vec{x} + \vec{x} \times \vec{y}$$

Apply a unit quaternion's rotation to a vector:

Vector  $\vec{v} = (v_x, v_y, v_z)$  in quaternions:  $Q_v = (0, v_x, v_y, v_z)$ ; Rotated  $Q_{v'} = Q Q_v Q^*$

## INTERPOLATION

### GENERAL PROBLEM

Local fit – splines

Global fit – e.g. regression

Explicit vs implicit vs parametric

Polynomials

#### Represent using monomials

$$f(u) = (x(u), y(u), z(u))^T = a_0 + a_1 u + \dots + a_n u^n$$

degree, coefficients

#### Represent using basis functions

$$f(u) = \sum_{i=0}^n b_i B_i^n(u)$$

### Local fit Interpolation Problem – General Solution

Input: a set of key point  $(p_0, t_0), \dots (p_{m-1}, t_{m-1})$

Output: interpolated spline

Step1: compute control points for each segment

Step2: compute curves using control points

### BEZIER CURVE

#### Using Bernstein polynomials as basis functions

$$B_i^n(u) = \binom{n}{i} u^i (1-u)^{n-i}$$

Derivative of Bezier curves:

$$\frac{df(u)}{du} = n \sum_{i=0}^{n-1} (b_{i+1} - b_i) B_i^{n-1}(u)$$

Cubic Bezier curve:  $n=3$

#### Matrix Form

Rewrite the Bezier curve function as:

$$f(u) = \sum_{i=0}^n b_i B_i^n(u) = b_0 + 3(b_1 - b_0)u + 3(b_2 - 2b_1 + b_0)u^2 + (b_3 - 3b_2 + 3b_1 - b_0)u^3 =$$

$$[b_0 \ b_1 \ b_2 \ b_3] \begin{bmatrix} 1 & -3 & 3 & -1 \\ 0 & 3 & -6 & 3 \\ 0 & 0 & 3 & -3 \\ 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} 1 \\ u \\ u^2 \\ u^3 \end{bmatrix}$$

## DE CASTELJAU ALGORITHM

Take cubic Bezier curve for example:

Given four coefficient vectors/control points:  $b_0, b_1, b_2, b_3$  and  $u$

$$b_0^1 = \text{LERP}(b_0, b_1, u), b_1^1 = \text{LERP}(b_1, b_2, u), b_2^1 =$$

$$\text{LERP}(b_2, b_3, u)$$

$$b_0^2 = \text{LERP}(b_0^1, b_1^1, u), b_1^2 = \text{LERP}(b_1^1, b_2^1, u)$$

$$b_0^3 = \text{LERP}(b_0^2, b_1^2, u)$$

## SPLINE

A spline is a curve comprised of a collection of piece-wise polynomials of arbitrary degree tied together at knot points with certain continuity conditions.

Continuity: 0, 1, 2 ...

## CATMUL-ROM SPLINE

Each segment is a Bezier curve

$$f_j(t) = \sum_{i=0}^n b_{ji} B_i^n(u), \text{ where } u = \frac{t-t_j}{t_{j+1}-t_j}$$

Usually, collect all the control points into a  $n \times m$  length vector, while  $n$  is the degree of the curve,  $m$  is the number of curve segments.

Catmul-Rom Spline is C1 continuous

Generally, we have  $f'(t_i) = (p_{i+1} - p_{i-1}) / (t_{i+1} - t_{i-1})$

To compute control points for Catmul-Rom spline, for segment  $i$ , we have:

$$\begin{cases} b_0 = p_i \\ b_3 = p_{i+1} \\ b_1 = b_0 + s_0/3 \\ b_2 = b_3 - s_1/3 \end{cases} \begin{cases} s_0 = (p_{i+1} - p_{i-1}) / (t_{i+1} - t_{i-1}) \\ s_1 = (p_{i+2} - p_i) / (t_{i+2} - t_i) \end{cases}$$

Special case for endpoints

Option 1:  $s_0 = p_1 - p_0$

Option 2: introduce phantom point  $p_{-1}$  and  $p_{m+1}$

We can use the matrix form to convert between curve types (Bezier curve and monomial curve)

## HERMITE CURVE

$$h(u) = p_0 H_0^3(u) + p_1 H_1^3(u) + p'_0 H_2^3(u) + p'_1 H_3^3(u)$$

Let  $h(u) = f(u)$ , we have:

$$H_0^3 = B_0^3 + B_1^3, H_1^3 = B_2^3 + B_3^3, H_2^3 = B_1^3/3, H_3^3 = B_2^3/3$$

$$h(u) = (1 - 3u^2 + 2u^3)p_0 + (3u^2 - 2u^3)p_1 + (u - 2u^2 + u^3)p'_0 + (-u^2 + u^3)p'_1$$

## Matrix Form

$$h(u) = [p_0 \ p_1 \ p'_0 \ p'_1] \begin{bmatrix} 1 & 0 & -3 & 2 \\ 0 & 0 & 3 & -2 \\ 0 & 1 & -2 & 1 \\ 0 & 0 & -1 & 1 \end{bmatrix} \begin{bmatrix} 1 \\ u \\ u^2 \\ u^3 \end{bmatrix}$$

## HERMITE SPLINE

C2 continuity -  $p'_j + 4p'_{j+1} + p'_{j+2} = -3p_j + 3p_{j+2}$

Special case for endpoints

Option 1: clamped, assign value for endpoint slope

Option 2: natural, let second order derivative at endpoints be 0, so that:

$$2p'_0 + p'_1 = -3p_0 + 3p_1, p'_{n-1} + 2p'_n = 3p_n - 3p_{n-1}$$

## B-SPLINE

B-spline is C2 continuous

See handout 'B-spline Construction Summary' for detail.

## Catmul-Rom Spline vs B-spline

A Catmul-Rom spline is constructed from individual Bezier curves, while a B-spline uses overlapping basis functions. Both provide local function approximation. Catmul-Rom spline is easy to compute and computationally efficient, but has discontinuous second derivatives at the knot points, while B-spline provides an optimal interpolation to the control points and it has continuous second derivatives at the knot points, but is more computationally expensive.

## QUATERNION INTERPOLATION

$$\text{Slerp}(a, b, u) = a(\sin(1-u)\Omega/\sin\Omega) + b(\sin u\Omega/\sin\Omega), \text{ where } \cos\Omega = a \cdot b$$

Why use quaternion?

Euler angles: discontinuous

Rotation matrix: has to be orthonormal

See handout 'Cubic Quaternion Spline Summary' for detail

## 2D SURFACES

### 2D polynomials

$$f(u, v) = (x(u, v), y(u, v), z(u, v))^T =$$

$$\sum_{i=0}^n a_i u^i \sum_{j=0}^n b_j v^j = \sum_{i,j} c_{ij} u^i v^j$$

### Manifold

A lower dimensional surface embedded in a higher dimensional space that is a local deformation of Euclidean space without any tears or intersections

Local coordinate system on surface

$$\frac{\partial f}{\partial v}, \frac{\partial f}{\partial u} \text{ and } \hat{n} = \frac{\frac{\partial f}{\partial u} \times \frac{\partial f}{\partial v}}{\left\| \frac{\partial f}{\partial u} \times \frac{\partial f}{\partial v} \right\|}$$

### 2D Interpolation

Given 4 points  $p_{00}, p_{10}, p_{11}, p_{01}$ , find  $p = f(u, v)$  for  $u, v \in [0, 1]$

2D linear interpolation – start with Lerp in terms of  $v$  (for  $u=0$  and  $u=1$ ), then interpolate the two values in terms of  $u$

$$f(u, v) = p_{00}(1-v)(1-u) + p_{01}v(1-u) + p_{10}(1-v)u + p_{11}vu = \sum_{i=0}^1 \sum_{j=0}^1 p_{ij} B_i^1(u) B_j^1(v)$$

(Cubic) Bezier Surface Interpolation – start with Bezier curves in terms of  $v$ , then then interpolate in terms of  $u$

$$f(u, v) = \sum_{i=0}^3 \sum_{j=0}^3 b_{ij} B_i^3(u) B_j^3(v) \text{ (16 control pts)}$$

Properties:

~Endpoint interpolation  $f(i, j) = b_{ij}$

~Cotangents ~Twist

$$\begin{array}{lll} \frac{\partial f}{\partial u}(0,0) = 3(b_{10} - b_{00}) & \frac{\partial f}{\partial v}(0,0) = 3(b_{01} - b_{00}) & \frac{\partial^2 f}{\partial u \partial v}(0,0) = 9(b_{00} - b_{01} - b_{10} + b_{11}) \\ \frac{\partial f}{\partial u}(1,0) = 3(b_{30} - b_{20}) & \frac{\partial f}{\partial v}(0,1) = 3(b_{03} - b_{02}) & \frac{\partial^2 f}{\partial u \partial v}(0,1) = 9(b_{03} - b_{02} - b_{31} + b_{12}) \\ \frac{\partial f}{\partial u}(0,1) = 3(b_{13} - b_{03}) & \frac{\partial f}{\partial v}(1,0) = 3(b_{31} - b_{30}) & \frac{\partial^2 f}{\partial u \partial v}(1,0) = 9(b_{30} - b_{20} - b_{31} + b_{11}) \\ \frac{\partial f}{\partial u}(1,1) = 3(b_{33} - b_{23}) & \frac{\partial f}{\partial v}(1,1) = 3(b_{33} - b_{32}) & \frac{\partial^2 f}{\partial u \partial v}(1,1) = 9(b_{33} - b_{32} - b_{23} + b_{12}) \end{array}$$

### Spline Cages

~Use 2D parametric surfaces (e.g. Bezier, B-spline)

~Control points used to deform model (surface path subdivision for local control)

~Convert parametric form to polygons

### 3D: FREE FORM DEFORMATION

#### Cubic Bezier Volume Patch

$$f(u, v, w) = \sum_{i=0}^3 \sum_{j=0}^3 \sum_{k=0}^3 b_{ijk} B_i^3(u) B_j^3(v) B_k^3(w)$$

#### FFD

- ~Set up lattice
- ~Polygon embedding: express local coordinates of vertex point  $p$  in a lattice  $p = p_0 + u\hat{u} + v\hat{v} + w\hat{w}$
- ~Initialize control points  $b_{ijk}$  to corners of the cube
- ~Deform lattice by moving the control points of the lattice

## KINEMATICS

### FORWARD KINEMATICS

See 'Forward Kinematics Recap'

### JACOBIAN MATRIX

See Jacobian Matrix Computation

### EULER ANGLE RATE TO ANGULAR VELOCITY

See 'Euler Angle Rate to Angular Velocity Conversion' and Appendix 2

### INVERSE KINEMATIC

Before everything: create IK chain

#### Pseudo Inverse Method

Theory

$$\dot{x} = J\dot{\theta} \Rightarrow \dot{\theta} = J^{\dagger}\dot{x}$$

Left pseudo inverse: more DoF in  $x$  than  $\theta$

$$J^{\dagger} = (J^T J)^{-1} J^T$$

Right pseudo inverse: otherwise

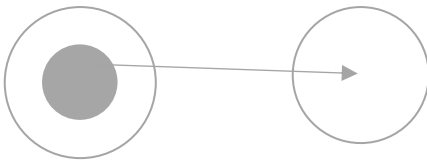
$$J^{\dagger} = J^T (J J^T)^{-1}$$

Integrate  $\Delta\theta$  to get new joint angle

- Null space:  $0 = J\dot{\theta} \Rightarrow \dot{\theta} = (J^{\dagger}J - I)c$

$\dot{\theta}$  space

$\dot{x}$  space



- Spring like behavior:  $\dot{\theta} = k(\theta - \theta_0)$   
 $\dot{\theta} = J^{\dagger}\dot{x} + (J^{\dagger}J - I)(\theta - \theta_0)$
- Damped Pseudo Inverse:  
 To avoid singular case in  $(J J^T)^{-1}$ , let  
 $J^{\dagger} = J^T (\lambda I + J J^T)^{-1}$ ,  $\lambda \ll 1$

#### Implementation Steps

1. Find current and desired position and orientation of the end joint
2. Path planning, design a trajectory and break into small segments (check 'arc length parameterization')
3. Compute current Jacobian matrix, we can use  $J_i = B_i L_i$  (check 'Jacobian Matrix Computation' for detail)
4. For each small  $\Delta x$ , compute  $\Delta\theta$  using  $\Delta\theta = J^{\dagger}\Delta x$

#### Limb IK

Notation

$p_{end}, p_{mid}, p_{base}$  - three joint position

$p_{target}$  - target position

$l_1, l_2$  - length of the upper and lower 'arm'

#### Implementation Steps

$$e \leftarrow p_{target} - p_{end}$$

$$r \leftarrow p_{end} - p_{base}$$

$$r_d \leftarrow p_{target} - p_{base}$$

$$\theta_{mid} \leftarrow \arccos((l_1^2 + l_2^2 - \|r_d\|^2) / (2l_1 l_2)) \text{ //mid joint angle}$$

$$\Delta\theta_{base} \leftarrow \arccos((\|r\|^2 + \|r_d\|^2 - \|e\|^2) / (2\|r\| \|r_d\|))$$

Update the transformations on mid joint and base joint, pay attention to local and global frame

#### CCD

#### Implementation Steps

Repeat:

For each joint  $p_{curr}$  in IK chain (distal to proximal):

//compute axis and angle

$$e \leftarrow p_{end} - p_{base}$$

$$r_{c2e} \leftarrow p_{end} - p_{curr}$$

$$angle \leftarrow \|r_{c2e} \times e\| / (\|r_{c2e}\|^2 + \|r_{c2e}\| \|e\|)$$

(when angle is small, we can omit the 'atan')

$$axis \leftarrow r_{c2e} \times e / \|r_{c2e} \times e\|$$

convert axis to local coordinate

$$\Delta\theta_{curr} = c_{curr} * angle$$

$$\Delta R = AxisAngle2Rot(axis, \Delta\theta_{curr})$$

Update joint transformation

## BODY ANIMATION

### BASICS

- Animation = pose(time)
- Collection of motion curves (6DOF for a single joint)
- Motion curve representation (e.g. cubic splines)

### MAIN APPROACHES

- Key frame
- Motion Capture
- Procedural
- Physically-based

### MOTION CAPTURE

With markers/sensors placed on subject, record motion from real world objects, used recorded motion to animate virtual objects



1. Track motions of actor body and/ or face
2. Convert to skeleton description and joint angle data
3. Use skeleton and joint angle data to animate characters

#### Different types of Motion Capture Technology

- Optical: passive marker, active marker, marker-less (only track position of markers)
- Magnetic
- Inertial Systems
- Exoskeleton

#### File format

BVH: Joint-based skeleton

AMC: bone-based skeleton

### NOTATIONS

- Rigging system (FK and IK)

- Pose space

$$\text{A point in } \sim: \theta = [\theta_1, \dots, \theta_n]^T$$

An animation can be thought of as a point moving through pose space, or alternately as a curve or spline in pose space:  $\theta = \theta(t)$

- Channels: 1-dimensional curves (one for each DOF)

$$\theta_i = \theta_i(t)$$

- Can be a joint angle or arbitrary parameter value

- Represents pre-recorded data
- Channels can be discontinuous in value, but not in time
- Array of channels (flexible, less memory) vs. array of poses (faster) (numDoFs\*numFrames)

### MOTION EDITING

#### **Applications**

Interactive Posing

Adding constraints

Optimizing motion over a sequence of poses

#### **Main Editing Techniques**

- Time warping

$$m'(t) = m(wt)$$

$w > 1$  – speed up

$0 < w < 1$  – slow down

- Blending

Frame level:  $m(t) = (1 - \alpha)m_1(t) + \alpha m_2(t)$

Ctrl pts level:  $c_j = (1 - \alpha)c_{1j} + \alpha c_{2j}$  (when both motion curves have same knot points)

Knot pts level:  $t_j = (1 - \alpha)t_{1j} + \alpha t_{2j}$  (when both motion curves have knot points at different points in time)

- Layering

$$m(t) = (1 - \alpha(t))m_1(t) + \alpha(t)m_2(t)$$

### ARC LENGTH PARAMETERIZATION

#### **Motivation**

Given a 3D path representation  $p = f(u) = [x(u) \ y(u) \ z(u)]^T$ , the x, y and z path coordinates are functions of independent parameter; We want to move along the path in constant speed (or following a given time-velocity curve), but  $s = g(u)$  is usually not linear.

Solution:

Use  $u = g^{-1}(s)$  to choose  $\Delta u$  to get uniform  $\Delta s$

Non-analytical; Need a list of u-s correspondence table for approximation (see slide for detail)

### **BEHAVIOR ANIMATION**

*Contents in this section are covered in 'Lecture Notes on Interactive Animation and Control Architectures'. See notes for details*

### **OPTIMIZATION APPROACHES**

*Contents in this section are covered in slide 'Animation through Optimization'. See notes for details*

### LEAST SQUARES OPTIMIZATION

Example: fit a set of points to a cubic curve

$$\begin{bmatrix} 1 & u_1 & u_1^2 & u_1^3 \\ \vdots & \vdots & \vdots & \vdots \\ 1 & u_m & u_m^2 & u_m^3 \end{bmatrix} \begin{bmatrix} c_0 \\ c_1 \\ c_2 \\ c_3 \end{bmatrix} = \begin{bmatrix} y_1 \\ \vdots \\ y_m \end{bmatrix}$$

$$A \quad x = b$$

Let  $e = Ax - b$

The problem transforms to an optimization problem:

$$\min_c \frac{1}{2} \|e\|_2^2 = \frac{1}{2} (Ax - b)^T (Ax - b)$$

take derivative, we have

$$A^T(Ax - b) = 0 \Rightarrow x = (A^T A)^{-1} A^T b \text{ (MLE solution)}$$

### GRADIENT DESCENT

Sequential quadratic Programming (SQP) / Newton's method or Newton Lagrangian

2 steps (See slide for more)

### PCA

#### **Eigenvector and Eigenvalue**

$$Ae = \lambda e \Rightarrow (A - \lambda I)e = 0$$

calculate eigenvalue  $\det(A - \lambda I) = 0$

#### **PCA Algorithm**

Given a dataset  $X$ , assume  $X$  is centered, let  $A = XX^T$

Let  $E = [e_1 \ e_2 \ \dots \ e_n]$  be the set of all eigenvectors of  $A$ , i.e.  $AE = E\Lambda$ , where  $\Lambda = \text{diag}([\lambda_1 \ \lambda_2 \ \dots \ \lambda_n])$ , therefore:

$$A = E\Lambda E^{-1} = E\Lambda E^T$$

Let  $Y = PX$ , where  $P = E^T$ , the covariance of  $Y$  is

$$\text{Cov}_Y = \frac{1}{m} PXX^T P = \frac{1}{m} P A P = \frac{1}{m} E^T E \Lambda E^T E = \frac{1}{m} \Lambda$$

In many cases, the first  $k$  eigenvectors account for most of the variance, we can use a reduced form

$$\tilde{P}^T = [e_1 \ e_2 \ \dots \ e_k]$$

and  $\tilde{y}_i = \tilde{P} x_i$ , where  $\tilde{y}_i$  is  $k \times 1$ ,  $x_i$  is  $n \times 1$

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#### **PCA Algorithm**

**Input:**  $D\{x^1, \dots, x^n\}$

**Output:** principle components  $z^1, \dots, z^k$

Compute mean and covariance of data  $\bar{x}$ ,  $\Sigma$

Find  $k$  eigenvectors of  $\Sigma$  with largest eigenvalues

$u_1, \dots, u_k$  (loadings)

Get principle components:  $z^i = ((x^i - \bar{x})^T u_1, \dots, (x^i - \bar{x})^T u_k)$

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### **EigenFace**

Implementation steps for calculate Eigenfaces

1. Vectorize and centerize image
2. Calculate covariance matrix
3. Obtain eigenvectors and eigenvalues
4. Choose 16 largest eigenvectors as loadings
5. Re-represent faces using principle components

### SPACETIME CONSTRAINTS

Solve for the object motion by varying the force over the entire time

#### **Problem Statement**

Governing Equation (dynamics)

Boundary Conditions (position constraint)

Objective function (minimize energy consumption)

#### **Implementation process**

- Set up motion equations
- Define objective function
- Evaluate the derivatives
- Pass into SQP solver

**Pros:** 'optimal' motion sequence; don't have to understand control **Cons:** Not real time control (lags); local minima; hard to define appropriate objective functions

**Applications:** optimal transitions, optimal gait and form, synthesis of motion, motion retargeting (see slide for more)

## DYNAMICS

NOTATION					
position	x	velocity	v	acceleration	a
mass	m	momentum	p (mv)	force	f
Moment of Inertia	I	Angular momentum	L	torque	$\tau$

### LINEAR DYNAMICS (TRIVIAL)

#### FORCE TYPE

**Gravity**  $f = mg$

**Springs**  $f_s = -K_s x$

**Dampers**  $f_d = -K_d v$

**Friction**  $f = f_n \mu_d$  (dynamic friction)  $f \leq f_n \mu_s$

**Aerodynamic Drag**  $f_{\text{drag}} = -0.5 \rho \|v\|^2 c_d A \hat{v}$

**Force field**  $f_{\text{field}} = f(x)$

### PARTICLE SYSTEM DYNAMICS

#### Newtonian approach

Simulation Steps:

1. Compute all forces acting on each particle in current configuration
2. Compute the resulting acceleration for each particle
3. Integrate over some small time step to update state
4. Repeat 1~3

#### Particle systems

#### 2<sup>nd</sup> Order ODE

$$1^{\text{st}} \text{ order: } a = \ddot{x} = \frac{f(x, \dot{x}, t)}{m}, 2^{\text{nd}} \text{ order: } \begin{cases} \dot{x} = v \\ \dot{v} = f(x, \dot{x}, t)/m \end{cases}$$

#### Phase Space

$$\text{State } s = \begin{bmatrix} x \\ v \end{bmatrix}, \dot{s} = \begin{bmatrix} \dot{x} \\ \dot{v} \end{bmatrix}$$

$$\text{State dynamics } \begin{bmatrix} \dot{x} \\ \dot{v} \end{bmatrix} = \begin{bmatrix} v \\ f/m \end{bmatrix} \text{ or } \dot{s} = f(s, t)$$

See more in 'Physically Based Modeling: Particle System Dynamics'

#### Solve Particle System Dynamics

Derive Evaluation Loop

- Clear forces
- Calculate forces
- Gather (computer dynamics)
- Update state

#### Rendering

Points, lines, sprites, geometry, ...

#### Rotational dynamics for particles

$$L = r \times p$$

$$\tau = dL/dt = r \times f$$

velocity vs angular velocity:  $v = dr/dt = \omega \times r + v_r$

centripetal acc.:  $a_{cen} = dv/dt = \dot{\omega} \times r + \omega \times (\omega \times r)$

Derivative of Rotation Matrix  $dR/dt = \dot{R} = \omega \times R$

Rotational Inertia (assume constant r and m)

$$L = r \times p = r \times (mv) = mr \times v = mr \times (\omega \times r)$$

$$= -mr \times r \times \omega = -m\Gamma \cdot \Gamma \cdot \omega = I \cdot \omega$$

where  $\Gamma$  is the skew symmetric matrix of  $r$ ,  $I = -m\Gamma \cdot \Gamma$

$$I = -m \begin{bmatrix} -r_y^2 - r_z^2 & r_x r_y & r_x r_z \\ r_x r_y & -r_x^2 - r_z^2 & r_y r_z \\ r_x r_z & r_y r_z & -r_x^2 - r_y^2 \end{bmatrix}$$

#### System of Particles

$$m_{\text{total}} = \sum m_i$$

$$x_{cm} = \sum m_i x_i / \sum m_i$$

$$p_{cm} = \sum p_i = \sum m_i v_i$$

$$v_{cm} = dx_{cm}/dt = (1/\sum m_i) d(\sum m_i x_i)/dt \\ = \sum m_i v_i / \sum m_i = p_{cm} / m_{\text{total}}$$

$$p_{cm} = m_{\text{total}} v_{cm}$$

$$f_{cm} = dp_{cm}/dt = \sum f_i$$

$$L_{cm} = \sum r_i \times p_i = \sum (x_i - x_{cm}) \times p_i$$

$$\tau_{cm} = dL_{cm}/dt = \sum r_i \times f_i$$

#### RIGID BODY DYNAMICS

$$x_{cm} = \int \rho x d\Omega / \int \rho d\Omega$$

$$I = \begin{bmatrix} \int \rho (r_y^2 + r_z^2) d\Omega & -\int \rho r_x r_y d\Omega & -\int \rho r_x r_z d\Omega \\ -\int \rho r_x r_y d\Omega & \int \rho (r_x^2 + r_z^2) d\Omega & -\int \rho r_y r_z d\Omega \\ -\int \rho r_x r_z d\Omega & -\int \rho r_y r_z d\Omega & \int \rho (r_x^2 + r_y^2) d\Omega \end{bmatrix}$$

$$L = I \cdot \omega$$

$$\tau = dL/dt = \omega \times I \cdot \omega + I \cdot \dot{\omega}$$

#### Equation of motions

$$a = f_m / m$$

$$\dot{\omega} = I^{-1}(\tau - \omega \times I \cdot \omega)$$

#### Rigid body simulation

- Translation  
Velocity & position update  $s(t_{k+1}) = s(t_k) + \dot{s}(t_k) \Delta t$
- Orientation  
Angular velocity update  $\omega(t_{k+1}) = \omega(t_k) + \dot{\omega}(t_k) \Delta t$   
Orientation update:  
 $Axis = \frac{\omega(t_k)}{\|\omega(t_k)\|}, Angle = \|\omega(t_k)\| \Delta t$   
 $R(t_{k+1}) = R(t_k) \Delta R$   
where  $\Delta R = AxisAngle2Rot(Axis, Angle)$

### SUMMARY: FORWARD AND INVERSE DYNAMICS

**Forward dynamics** Compute motion resulting from applied forces and torques

$$\ddot{x} = f/m$$

$$\dot{\omega} = I^{-1}(\tau - \omega \times I \cdot \omega)$$

**Inverse dynamics** Compute forces and torques required to generate desired motion

$$f = m\ddot{x}$$

$$\tau = \omega \times I \cdot \omega + I \cdot \dot{\omega}$$

### DIFFERENTIAL EQUATIONS

General form:  $\dot{x} = f(x, t)$

Vector field  $\dot{x} = f(x)$

Taylor expansion

$$x_{t+\Delta t} = x_t + \dot{x}_t \Delta t + \ddot{x}_t \Delta t^2 / 2 + \ddot{\ddot{x}}_t \Delta t^3 / 6 + \dots$$

**Euler's Method (1<sup>st</sup> order)**  $x_{t_{k+1}} = x_{t_k} + \dot{x}_{t_k} \Delta t$

Drawbacks: drift off, error accumulate by time, oscillate at large step

#### Runge Kutta Method (2<sup>nd</sup> and above)

- Use weighted average of slopes across interval
- Order determines approximation error

2<sup>nd</sup> order RK method

$$x_{t_{k+1}}^p = x_{t_k} + \dot{x}_{t_k} \Delta t$$

- $\dot{x}_{t_{k+1}}^p = f(x_{t_{k+1}}^p)$
  - $x_{t_{k+1}} = x_{t_k} + (\dot{x}_{t_{k+1}}^p + \dot{x}_{t_k})\Delta t/2$
- 4<sup>th</sup> order RK method – better fit, expensive  
 $x_{t_{k+1}} = x_{t_k} + (d_1 + 2d_2 + 2d_3 + d_4)\Delta t/6$   
 where

$$\begin{aligned} d_1 &= f(t_k, x(t_k)) \\ d_2 &= f(t_{k+1} + \Delta t/2, x(t_k) + d_1/2) \\ d_3 &= f(t_{k+1} + \Delta t/2, x(t_k) + d_2/2) \\ d_4 &= f(t_{k+1} + \Delta t, x(t_k) + d_3) \end{aligned}$$

### Implicit Method

Explicit Euler method add energy in the form of errors, which is bad for stiff systems (may result in instability)

Backward Euler Method  $x_{t_{k+1}} = x_{t_k} + \dot{x}_{t_{k+1}}\Delta t$

Converges much slower

How to compute  $\dot{x}_{t_{k+1}}$ :

- derive from formula
- predictor-corrector (explicit method + plug in, e.g. RK2)
- linear system

$$\begin{aligned} x_{t_{k+1}} &= x_{t_k} + \dot{x}_{t_{k+1}}\Delta t = x_{t_k} + \Delta x_{t_k} \\ \dot{x}_{t_{k+1}} &= f(x_{t_{k+1}}) = f(x_{t_k} + \Delta x_{t_k}) \\ x_{t_k} + \Delta x_{t_k} &= x_{t_k} + \Delta t f(x_{t_k} + \Delta x_{t_k}) \\ \Delta x_{t_k} &\approx \Delta t (f(x_{t_k} + (\partial f / \partial x_{t_k}) \Delta x_{t_k})) \\ \Delta x_{t_k} &\approx \left( \frac{1}{\Delta t} I - \frac{\partial f}{\partial x_{t_k}} \right)^{-1} f(x_{t_k}) \end{aligned}$$

### Multi-step Methods

Use values from previous time steps to calculate next one

Anchors approximation with more accurate data

Adams Bashforth

$$\begin{aligned} x_{t_{k+1}} &= x_{t_k} + \dot{x}_{t_k}\Delta t + \ddot{x}_{t_k}\Delta t^2/2 \\ \ddot{x}_{t_k} &\approx (\dot{x}_{t_k} - \dot{x}_{t_{k-1}})/\Delta t \\ x_{t_{k+1}} &+ (3\dot{x}_{t_k} - \dot{x}_{t_{k-1}})\Delta t/2 \end{aligned}$$

Verlet Integration

$$\begin{aligned} x_{t_{k+1}} &= x_{t_k} + \dot{x}_{t_k}\Delta t + \ddot{x}_{t_k}\Delta t^2/2 \\ x_{t_{k-1}} &= x_{t_k} - \dot{x}_{t_k}\Delta t + \ddot{x}_{t_k}\Delta t^2/2 \\ x_{t_{k+1}} &= 2x_{t_k} - x_{t_{k-1}} + \ddot{x}_{t_k}\Delta t^2 \end{aligned}$$

### Which to use

In practice, Euler Method or 2nd Order Runge Kutta is usually good enough for real-time apps (60 frames/sec)

If simulation becomes unstable even though system dynamics are stable implies that errors are being introduced by the numerical integration.

- Reduce the step size
- Try using the implicit Euler method
- Try higher order integration scheme

## FEEDBACK CONTROL

### SYSTEM DYNAMICS

Input:  $u = \begin{bmatrix} f \\ \tau \end{bmatrix}$ , State:  $x = \begin{bmatrix} p \\ \theta \\ v \\ \omega \end{bmatrix}$

Dynamics:  $\dot{x} = f(x, u)$

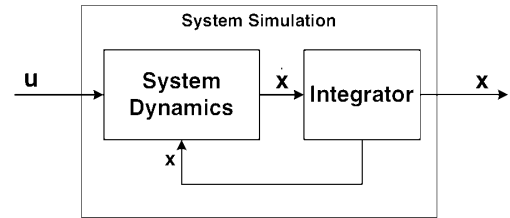
E.g.

2<sup>nd</sup> order system  $\ddot{x} + a_1\dot{x} + a_0x = bu$

3<sup>rd</sup> order system  $\ddot{x} + a_2\ddot{x} + a_1\dot{x} + a_0x = bu$

### SYSTEM SIMULATION

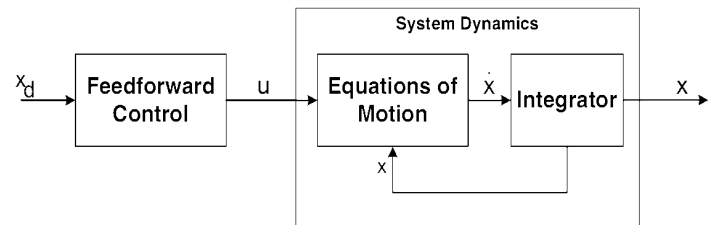
Objective: compute input  $u$  such that state  $x$  moves to desired state  $x_d$  over time



### FEEDFORWARD CONTROL

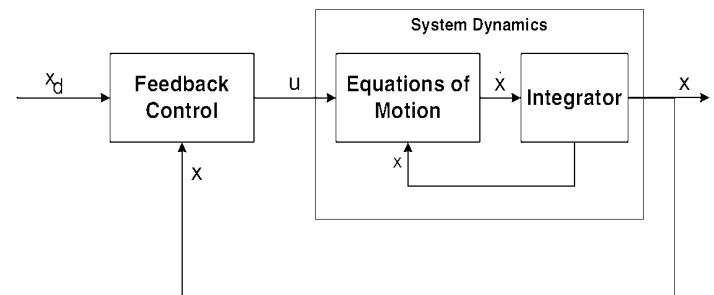
- The control input ( $u$ ) is determined as a function of the desired state ( $x_d$ ) and/or time ( $t$ ) without considering the actual value of the system state ( $x$ ).
- This type of control is only effective if the system can be accurately modeled and there are no disturbances in the environment that can affect the system state.

$$\ddot{x} = \ddot{x}_d \Rightarrow \dot{x} = \dot{x}_d \Rightarrow x = x_d$$



### FEEDBACK CONTROL

- The objective of the feedback controller is to get the system to achieve and maintain the desired state ( $x_d$ ).
- The control input ( $u$ ) is determined based on the error between the actual state ( $x$ ) and desired state ( $x_d$ ) of the dynamic system

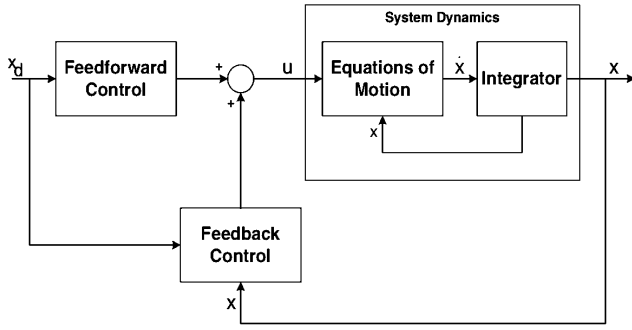


### Types of feedback control

- Proportional (P)  
 $u_{fb} = K_p(x_d - x)$
- Proportional Derivative (PD)  
 $u_{fb} = K_p(x_d - x) + K_d(v_d - v)$
- Proportional Integral Derivative (PID)  
 $u_{fb} = K_p(x_d - x) + K_d(v_d - v) + K_I \int (x_d - x) dt$

### Feedforward + feedback

$$u = u_{ff} + u_{fb}$$



## MASS SPRING DAMPER SYSTEM

### Translation (1D)

Equation of motion:  $\ddot{x} + \left(\frac{c}{m}\right)\dot{x} + \left(\frac{k}{m}\right)x = \frac{f}{m}$

Desired dynamics:  $\ddot{x} + 2\zeta\omega_n\dot{x} + \omega_n^2x = \omega_n^2x_d$

Solution:

$$x(t) = \left(1 - e^{-\zeta\omega_n t} \left(\cos(\omega_n t) + \frac{\zeta}{\sqrt{1-\zeta^2}} \sin(\omega_n t)\right)\right) x_d$$

Where  $\zeta$  is damping ratio and  $\omega_n$  is natural frequency

$\zeta = 0$  - no damping

$0 \leq \zeta \leq 1$  - oscillatory

$\zeta = 1$  - critically damped

$\zeta > 1$  - no oscillation

System time constant  $T_{TC} = 1/\zeta\omega_n$

Settling time  $T_{settle} = 4T_{TC}$  (98% steady)

### Rotation (1D)

Equation of motion:  $\ddot{\theta} + \left(\frac{c}{I_{zz}}\right)\dot{\theta} + \frac{k}{I_{zz}}\theta = \frac{\tau}{I_{zz}}$

## CONTROLLER DESIGN

Compute  $f$  as a function of  $x_d, x, v$  such that the actual dynamics behaves like the desired system dynamics (e.g. settle in certain time, exhibits desired amount of oscillation, etc.)

Take the mass spring damper system for example:

### Position Controller Design

Desired dynamics:  $\ddot{x} + 2\zeta\omega_n\dot{x} + \omega_n^2x = \omega_n^2x_d$

Solution:  $f = (c - 2m\zeta\omega_n)\dot{x} + (k - m\omega_n^2)x + m\omega_n^2x_d = K_Px + K_D\dot{x} + K_0x_d$

### Velocity Controller Design

Desired dynamics:  $\ddot{x} + \alpha\dot{x} = \alpha\dot{x}_d$  or  $(\dot{v} + \alpha v = \alpha v_d)$

Solution:  $f = (c - m\alpha)\dot{x} + kx + m\alpha\dot{x}_d = K_Px + K_Dv + K_1v_d$

### Tracking Controller Design

Desired dynamics:

$$(\ddot{x} - \ddot{x}_d) + 2\zeta\omega_n(\dot{x} - \dot{x}_d) + \omega_n^2(x - x_d) = 0$$

Solution:  $f = (c - 2m\zeta\omega_n)\dot{x} + (k - m\omega_n^2)x + m\omega_n^2x_d + 2m\zeta\omega_n\dot{x}_d + m\ddot{x}_d = K_Px + K_D\dot{x} + K_0x_d + K_1\dot{x}_d + K_2\ddot{x}_d$

## VIHICLE DYNAMICS

### Equations of Motion (world frame)

$f^0 = m\dot{V}^0$  - translation

$\tau^0 = \omega^0 \times I^0 \cdot \omega^0 + I^0 \cdot \dot{\omega}^0$  - rotation

Body frame to world frame  $V^0 = R_B^0 V^B$

### Equations of Motion (body frame)

$f^B = m\dot{V}^B + m\omega^B \times V^B$

$\tau^B = I^B \cdot \dot{\omega}^B + \omega^B \times I^B \cdot \omega^B$

### 2D planar case (world frame)

$$m\dot{V}_x^0 = f_x^0, \quad m\dot{V}_y^0 = f_y^0, \quad I_{zz}\dot{\omega}_z = \tau_z^0$$

### 2D planar case (body frame)

$$V^B = [V_x^B, V_y^B, 0]^T, \quad \omega^B = [0, 0, \dot{\theta}]^T$$

$$m\dot{V}_x^B - m\dot{\theta}V_y^B = f_x^B, \quad m\dot{V}_y^B + m\dot{\theta}V_x^B = f_y^B, \quad I_{zz}\dot{\omega}_z = \tau_z^0$$

### 3D case (world frame)

Desired value:  $V_d, R_d$

Velocity controller:

$$f^0 = m(K_P(V_d - V^0)) \quad (\text{world frame})$$

$$f^B = m(K_P(V_d - V^B) + \omega^B \times V^B) \quad (\text{body frame})$$

Angle controller:

$$\tau = I(K_P\Delta\theta - K_D\omega^B) + \omega^B \times I \cdot \omega^B$$

where  $\Delta R = R^T R_d, \Delta\theta = \text{AxisAngle}(\Delta R)$

Plug in, we got the close loop vehicle dynamics:

$$\dot{V} + K_P V = K_P V_d, \quad \dot{\omega} + K_V \omega = K_P \Delta\theta$$

Integrate to update state  $R$

## FACIAL ANIMATION

### FACE MODELING METHODS

3D modeling

photograph & digitize

sculpt & digitize

scanning

computer vision

### FACE ANIMATION METHODS

#### Blend shapes

Require several key expressions to be modeled ahead of time. The key expressions are then blended on the fly to create a new expression, either locally or globally.

You can use either linear (LERP) or non-linear interpolation (B-spline, Bazier, etc)

$$\text{LERP } v^* = v_{base} + \sum w_i(v_i - v_{base}) = (1 - \sum w_i)v_{base} + \sum w_i v_i$$

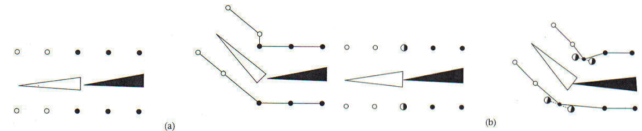
Also for the normal, we have

$$n^* = n_{base} + \sum w_i(v_i - v_{base}), n^* = n^* / \|n^*\|$$

If multiple targets affect the same vertex, their results combine in a reasonable way

Example: Skinning

- Transform blend



center vertices attached to both bones, perform weighted sum to determine actual position of center vertices

$$\mathbf{x}_{VT}^0(\theta) = w\mathbf{H}_{i+1}^0(\theta)\mathbf{x}_V^{i+1}(\theta_0) + (1-w)\mathbf{H}_i^0(\theta)\mathbf{x}_V^i(\theta_0)$$

$$\text{since } \mathbf{x}_V^{i+1}(\theta_0) = \mathbf{H}_i^{i+1}(\theta_0)\mathbf{x}_V^i(\theta_0)$$

- Shape blend

$$\mathbf{x}_{FS}^i(\beta) = \beta \mathbf{x}_{FS}^i(\theta_{\max}) + (1 - \beta) \mathbf{x}_{FS}^i(\theta_0)$$

$$\beta = \frac{(\theta - \theta_0)}{(\theta_{\max} - \theta_0)} \in [0, 1]$$

$$\mathbf{x}_{VST}^0(\theta) = [w \mathbf{H}_{i+1}^0(\theta) \mathbf{H}_i^{i+1}(\theta) + (1 - w) \mathbf{H}_i^0(\theta) [\beta \mathbf{x}_{FS}^i(\theta_{\max}) + (1 - \beta) \mathbf{x}_{FS}^i(\theta_0)]]$$

- smooth skinning
- add intermediate joint-points

*Example: Facial Animation*

As opposed to just two joints, in facial animation, the deformed vertex position is a weighted average over many joints to which the vertex is attached

$$\mathbf{v}'' = \sum w_i \mathbf{H}_i^0 \cdot (\mathbf{B}_i^0)^{-1} \mathbf{v}' \quad \text{where} \quad \sum w_i = 1$$

- $\mathbf{v}'$  is the untransformed vertex position in world coordinates from blend shapes calculation
- $\mathbf{B}_i$  is the binding matrix (global transform of joint  $i$  when the skin was initially attached)
- $\mathbf{v}''$  is the final vertex position in world space
- $w_i$  is the weight of joint  $i$
- $\mathbf{H}_i$  is the current world matrix of joint  $i$  after running the skeleton forward kinematics

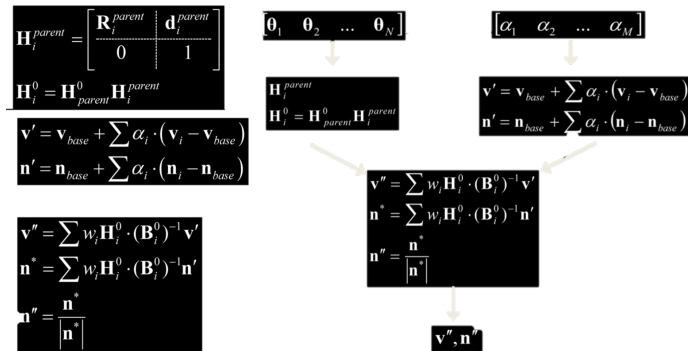
Note:

- $\mathbf{B}$  remains constant, so  $\mathbf{B}^{-1}$  can be computed at load time,  $\mathbf{H} \mathbf{B}^{-1}$  can be computed for each joint before skinning

Blending normals is essentially the same, except we transform them as directions  $(x, y, z, 0)$  and then renormalize the results

$$\mathbf{n}^* = \sum w_i \mathbf{H}_i^0 \cdot (\mathbf{B}_i^0)^{-1} \mathbf{n}' \quad \mathbf{n}'' = \frac{\mathbf{n}^*}{|\mathbf{n}^*|}$$

*Skeleton, Morph, & Skin Data Flow*



### Muscle-based

Each muscle has a zone of influence

### Performance-based

Performance based methods capture the motion of live performers and translate the actions into facial animation control parameters

Performance-driven facial animation system that maps points in images of a face to a 3D Model

Input: Real-time video capture from a single camera (e.g. webcam)

Output:

- Head movement and rotation
- 64 facial points and 100 expressions per frame
- Face texture extraction

### Parametric

FACS (Facial Action Coding System) describes all visually

distinguishable facial movements based on an anatomical analysis of facial actions

### MPEG-4 based

Efficient encoding scheme of the human face intended for multi-media scenes/video conferencing

- Promotes visual speech intelligibility and the recognition of the mood of the speaker with low bandwidth requirements
- Implemented as a collection of nodes and parameters in a scene graph which are animated

Facial Definition Parameters (FDP)

- 84 feature points used to transform a generic face model into a particular face of specific shape and texture

Facial Animation Parameters (FAP)

- 68 parameters used to animate expressions and speech through translation of FDPs and rotation of head, eyes, eyelids and jaw

### LIPSYNCHING

#### Speech Synchronization

First, generate a sequence of (phoneme, duration) pairs from voice recognition or text-to-speech

Next, generate the associated mouth deformation (i.e. Viseme)