

Chapter 2 - Kinematics: Theory of Strain

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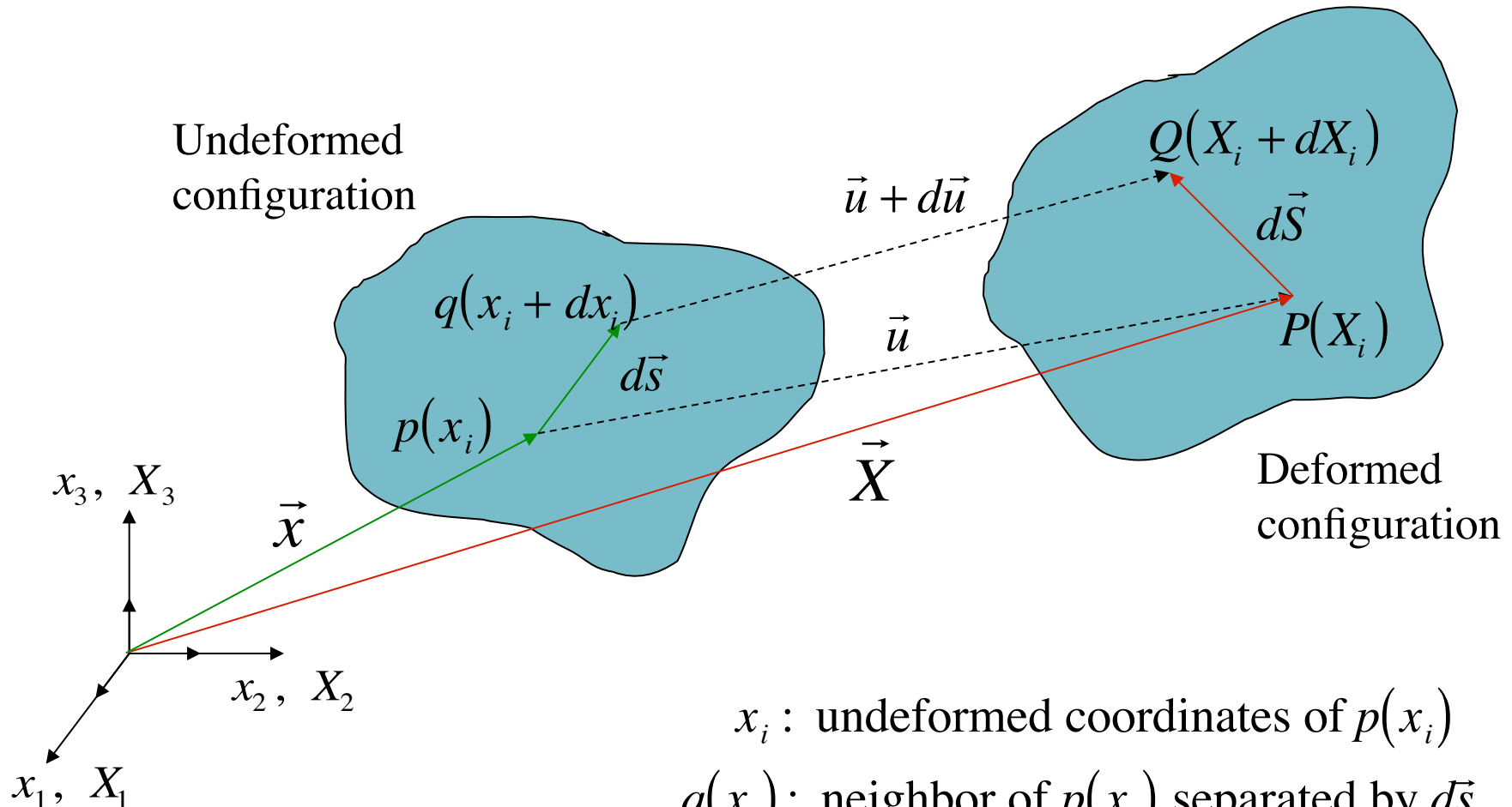
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Measures of Deformation



$$\vec{X} = \vec{x} + \vec{u}$$

x_i : undeformed coordinates of $p(x_i)$

$q(x_i)$: neighbor of $p(x_i)$ separated by $d\vec{s}$

X_i : deformed coordinates of $P(X_i)$

$Q(X_i)$: neighbor of $P(X_i)$ separated by $d\vec{S}$

Measures of Deformation (cont.)

- If p and q move such that $\Rightarrow d\vec{s} = d\vec{S}$
 \Rightarrow rigid motion
 \Rightarrow no deformation

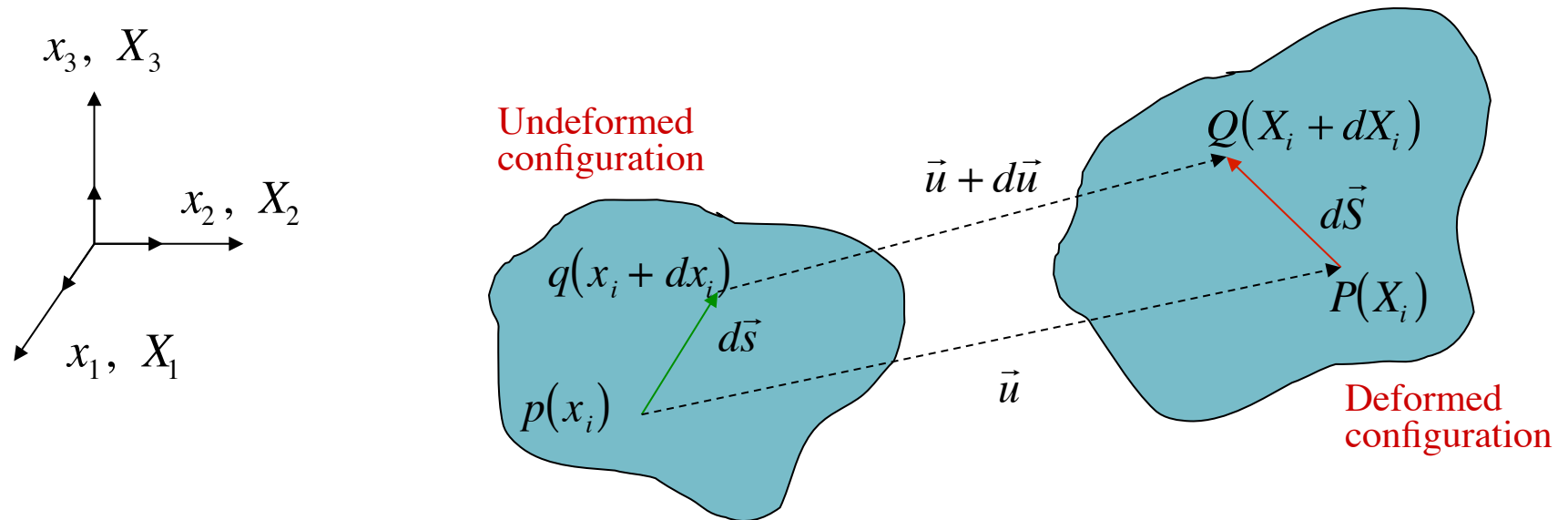
- Possible measures of deformation

$$dS - ds, \sqrt{dS} - \sqrt{ds}, f(dS) - f(ds)$$

with $ds = |d\vec{s}|$, $dS = |d\vec{S}|$

- Choose: $dS^2 - ds^2$

Lagrangian Strain Tensor



$$ds = |d\vec{S}| = \sqrt{dx_1^2 + dx_2^2 + dx_3^2} = \sqrt{dx_i dx_i}$$

$$dS = |d\vec{S}| = \sqrt{dX_1^2 + dX_2^2 + dX_3^2} = \sqrt{dX_i dX_i}$$

$$\Rightarrow dS^2 - ds^2 = dX_i dX_i - dx_i dx_i$$

Lagrangian Strain Tensor (cont.)

$dS^2 - ds^2 = dX_i dX_i - dx_i dx_i$ is a field (i.e. depends on coordinate)

- Choose **undeformed configuration** x_i as independent variable

$$X_i = X_i(x_1, x_2, x_3) = X_i(x_i)$$

$$\Rightarrow dX_i = \frac{\partial X_i}{\partial x_1} dx_1 + \frac{\partial X_i}{\partial x_2} dx_2 + \frac{\partial X_i}{\partial x_3} dx_3$$

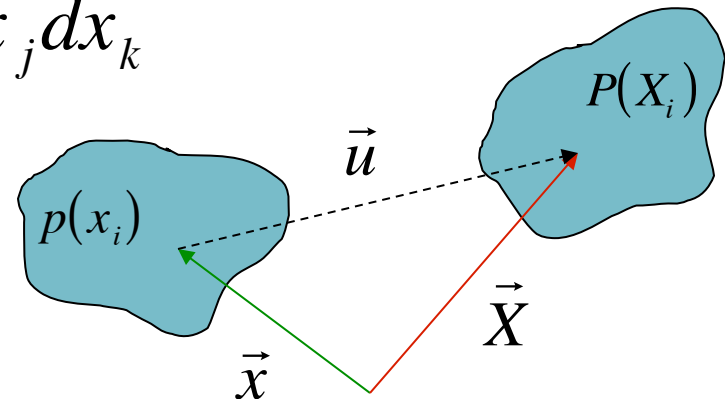
$$\Rightarrow dX_i = X_{i,j} dx_j$$

$$\Rightarrow dS^2 - ds^2 = (X_{i,j} dx_j)(X_{i,k} dx_k) - dx_i dx_i$$

$$\therefore dS^2 - ds^2 = (X_{i,j} X_{i,k} - \delta_{ij} \delta_{ik}) dx_j dx_k$$

Lagrangian Strain Tensor (cont.)

$$dS^2 - ds^2 = (X_{i,j}X_{i,k} - \delta_{ij}\delta_{ik})dx_jdx_k$$



$$\vec{X} = \vec{x} + \vec{u} \quad \Rightarrow \quad X_i = x_i + u_i$$

$$dS^2 - ds^2 = \left[(x_i + u_i)_{,j} (x_i + u_i)_{,k} - \delta_{jk} \right] dx_j dx_k$$

• Recall:

$$x_{i,j} = \frac{\partial x_i}{\partial x_j} = \delta_{ij}$$

Lagrangian Strain Tensor (cont.)

$$\begin{aligned}dS^2 - ds^2 &= \left[(\delta_{ij} + u_{i,j})(\delta_{ik} + u_{i,k}) - \delta_{jk} \right] dx_j dx_k \\&= (\delta_{ij}\delta_{ik} + \delta_{ij}u_{i,k} + u_{i,j}\delta_{ik} + u_{i,j}u_{i,k} - \delta_{jk}) dx_j dx_k \\&= (\delta_{jk} + u_{j,k} + u_{k,j} + u_{i,j}u_{i,k} - \delta_{jk}) dx_j dx_k \\&= (u_{j,k} + u_{k,j} + u_{i,j}u_{i,k}) dx_j dx_k\end{aligned}$$

- Define:

$$E_{ij} = \varepsilon_{ij}^L = \frac{1}{2} (u_{i,j} + u_{j,i} + u_{k,i}u_{k,j})$$

- So that:

$$dS^2 - ds^2 = 2E_{ij} dx_i dx_j$$

Lagrangian Strain Tensor (cont.)

$$E_{ij} = \varepsilon_{ij}^L = \frac{1}{2} \left(u_{i,j} + u_{j,i} + u_{k,i} u_{k,j} \right)$$

- E_{ij} are the components of the **Lagrangian** or **Green's** or **material** strain tensor
- Lagrangian: is expressed in terms of undeformed (or Lagrangian coordinates)
- By definition $E_{ij} = E_{ji}$, i.e. symmetric
- No assumptions on loading, material or amount of deformation

Eulerian Strain Tensor

- Recall

$$dS^2 - ds^2 = dX_i dX_i - dx_i dx_i$$

- Choose deformed X_i as independent variable

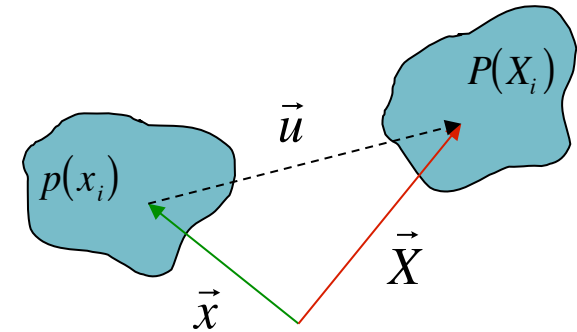
$$x_i = x_i(X_1, X_2, X_3) = x_i(X_i)$$

$$\Rightarrow dx_i = \frac{\partial x_i}{\partial X_1} dX_1 + \frac{\partial x_i}{\partial X_2} dX_2 + \frac{\partial x_i}{\partial X_3} dX_3$$

$$\Rightarrow dx_i = x_{i,J} dX_J \quad \leftarrow \text{Differentiation wrt } X$$

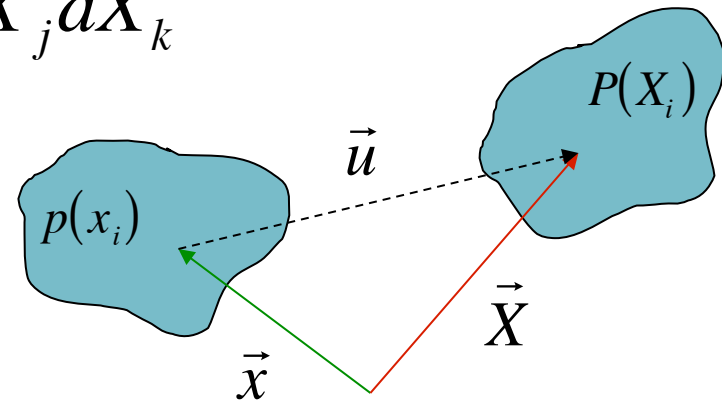
$$\Rightarrow dS^2 - ds^2 = dX_i dX_i - (x_{i,J} dX_J)(x_{i,K} dX_K)$$

$$\therefore dS^2 - ds^2 = (\delta_{ij} \delta_{ik} - x_{i,J} x_{i,K}) dX_J dX_K$$



Eulerian Strain Tensor (cont.)

$$dS^2 - ds^2 = \left(\delta_{ij} \delta_{ik} - x_{i,J} x_{i,K} \right) dX_j dX_k$$



$$\vec{X} = \vec{x} + \vec{u} \quad \Rightarrow \quad x_i = X_i - u_i$$

$$dS^2 - ds^2 = \left[\delta_{jk} - (X_i - u_i)_{,J} (X_i - u_i)_{,K} \right] dx_j dx_k$$

• Recall: $X_{i,J} = \frac{\partial X_i}{\partial X_j} = \delta_{ij}$

Eulerian Strain Tensor (cont.)

$$\begin{aligned}
 dS^2 - ds^2 &= \left[\delta_{jk} - (\delta_{ij} - u_{i,J})(\delta_{ik} - u_{i,K}) \right] dX_j dX_k \\
 &= \left(\delta_{jk} - \delta_{ij}\delta_{ik} + \delta_{ij}u_{i,K} + u_{i,J}\delta_{ik} - u_{i,J}u_{i,K} \right) dX_j dX_k \\
 &= \left(\delta_{jk} - \delta_{jk} + u_{j,K} + u_{k,J} - u_{i,J}u_{i,K} \right) dX_j dX_k \\
 &= \left(u_{j,K} + u_{k,J} - u_{i,J}u_{i,K} \right) dX_j dX_k
 \end{aligned}$$

- Define:

$$e_{ij} = \varepsilon_{ij}^E = \frac{1}{2} \left(u_{i,J} + u_{j,I} - u_{k,I}u_{k,J} \right)$$

Differentiation
wrt X_i

- So that:

$$dS^2 - ds^2 = 2e_{ij} dX_i dX_j$$

Eulerian Strain Tensor (cont.)

$$e_{ij} = \varepsilon_{ij}^E = \frac{1}{2} \left(u_{i,J} + u_{j,I} - u_{k,I} u_{k,J} \right)$$

Differentiation
wrt X_i

- e_{ij} are the components of the **Eulerian** or **Almansi** or **spatial** strain tensor
- Eulerian: is expressed in terms of deformed (or Eulerian coordinates)
- By definition $e_{ij} = e_{ji}$, i.e. symmetric
- No assumptions on loading, material or amount of deformation
- Differentiation is with respect to X_i

Infinitesimal Strain

- Lagrangian strain: $E_{ij} = \epsilon_{ij}^L = \frac{1}{2} (u_{i,j} + u_{j,i} + u_{k,i} u_{k,j})$

- Eulerian strain: $e_{ij} = \epsilon_{ij}^E = \frac{1}{2} (u_{i,J} + u_{j,I} - u_{k,I} u_{k,J})$

- Assumptions:

(a) If $|u_{i,j}| \ll 1$ then $u_{k,i} u_{k,j}$ can be ignored wrt $u_{i,j}$

(b) If $x_i \approx X_i$ then $\frac{\partial(\quad)}{\partial x_i} \approx \frac{\partial(\quad)}{\partial X_i}$

$$E_{ij} \approx e_{ij} \approx \frac{1}{2} (u_{i,j} + u_{j,i}) \equiv \epsilon_{ij}$$

Infinitesimal Strain (cont.)

$$\varepsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i})$$

- ε_{ij} are the components of the **infinitesimal strain tensor**
- By definition $\varepsilon_{ij} = \varepsilon_{ji}$, i.e. symmetric
- Condition (a) implies small displacement gradients. Condition (b) implies small displacements
- Above equations are the **strain-displacement relations**
- # of equations vs. # of unknowns
- Dimensions?

Infinitesimal Strain (cont.)

$$\epsilon_{ij} = \frac{1}{2}(u_{i,j} + u_{j,i}) = \frac{1}{2}\left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i}\right)$$

- Expanding:

$$\left. \begin{aligned} \epsilon_{11} &= \frac{\partial u_1}{\partial x_1} \\ \epsilon_{22} &= \frac{\partial u_2}{\partial x_2} \\ \epsilon_{33} &= \frac{\partial u_3}{\partial x_3} \end{aligned} \right\} \text{Normal Strains}$$

$$\left. \begin{aligned} \epsilon_{12} = \epsilon_{21} &= \frac{1}{2}\left(\frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1}\right) \\ \epsilon_{13} = \epsilon_{31} &= \frac{1}{2}\left(\frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1}\right) \\ \epsilon_{23} = \epsilon_{32} &= \frac{1}{2}\left(\frac{\partial u_2}{\partial x_3} + \frac{\partial u_3}{\partial x_2}\right) \end{aligned} \right\} \text{Shear Strains}$$

Infinitesimal Strain (cont.)

$$[\underline{\epsilon}] = \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ & \epsilon_{22} & \epsilon_{23} \\ \text{sym.} & & \epsilon_{33} \end{bmatrix}$$

- Since infinitesimal strain is a two-tensor it transforms (rotates) according to:

$$\epsilon'_{ij} = \alpha_{ik} \alpha_{jl} \epsilon_{kl}$$

or

$$[\underline{\epsilon}'] = [\underline{R}][\underline{\epsilon}][\underline{R}]^T$$

Engineering Strain

- Sometimes (e.g. Timoshenko and Goodier) infinitesimal shear strain is defined:

$$\gamma_{ij} = (u_{i,j} + u_{j,i}) \quad i \neq j$$

- But then strain is NOT a two-tensor

$$[\epsilon_{ij}] = \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ & \epsilon_{22} & \epsilon_{23} \\ \text{sym.} & & \epsilon_{33} \end{bmatrix} \quad \text{A tensor}$$

$$[\epsilon_{ij}] = \begin{bmatrix} \epsilon_{11} & \gamma_{12} & \gamma_{13} \\ & \epsilon_{22} & \gamma_{23} \\ \text{sym.} & & \epsilon_{33} \end{bmatrix} \quad \text{Not a tensor - Engineering strain}$$

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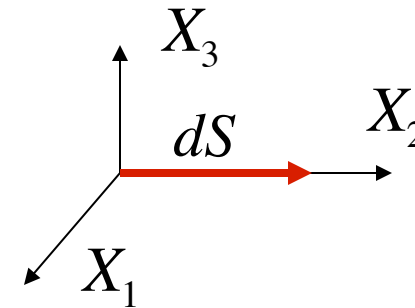
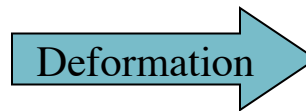
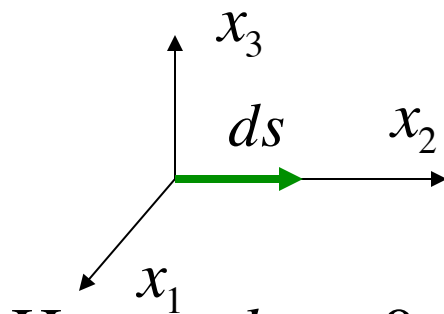
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Normal components

Take $d\vec{s} \parallel dx_2$



Here $\left. \begin{aligned} dx_1 &= 0 \\ dx_2 &= ds \\ dx_3 &= 0 \end{aligned} \right\} \Rightarrow$

$$\begin{aligned} dS^2 - ds^2 &= 2E_{ij}dx_i dx_j \\ &= 2E_{22}ds^2 \end{aligned}$$

$$\Rightarrow dS^2 = (1 + 2E_{22})ds^2$$

$$\Rightarrow dS = \sqrt{(1 + 2E_{22})}ds$$

$$\therefore \frac{dS}{ds} = \frac{\text{final length}}{\text{original length}} = \sqrt{1 + 2E_{22}}$$

Stretch ratio along x_2

Normal components (cont.)

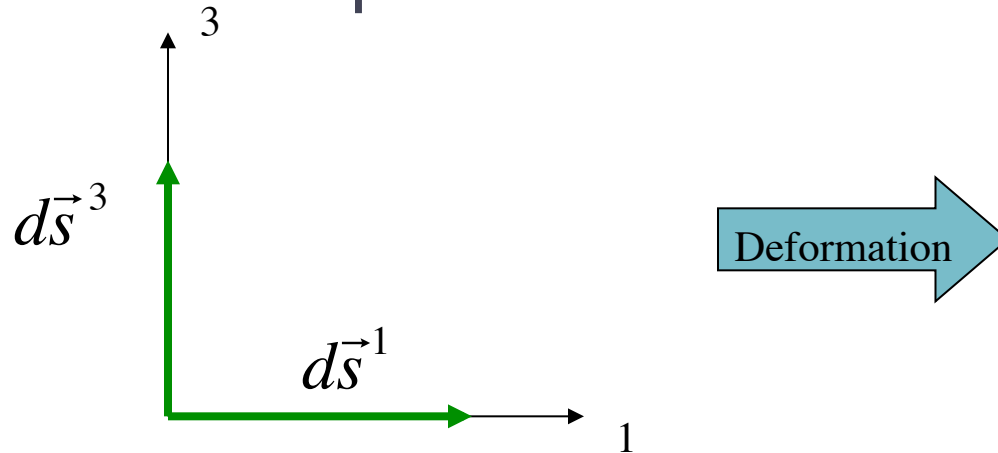
- **Extension ratio:** $\frac{dS - ds}{ds} = \frac{\text{change of length}}{\text{initial length}} = \sqrt{1 + 2E_{22}} - 1$

$$\therefore E_{22} \propto \Delta L / L \text{ along } x_2$$

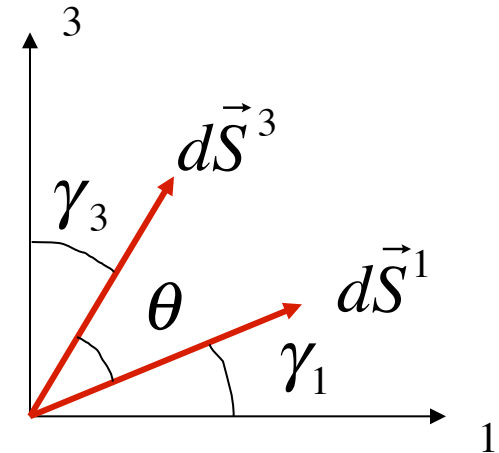
- For small strains $E_{22} \ll 1$:
$$\begin{aligned}\frac{dS - ds}{ds} &= \sqrt{1 + 2E_{22}} - 1 \\ &= (1 + 2E_{22})^{1/2} - 1 \\ &= 1 + \frac{1}{2}(2E_{22}) + \dots - 1 \\ &\approx E_{22} = \varepsilon_{22}\end{aligned}$$

\therefore normal strains ε_{ii} (*no sum*) represent $\frac{\text{change of length}}{\text{initial length}}$ along x_i

Shear Components



Deformation



$$d\vec{S}^1 \cdot d\vec{S}^3 = dS_i^1 dS_i^3$$

$$= dS^1 dS^3 \cos \theta$$

$$= dS^1 dS^3 \cos \left(\frac{\pi}{2} - \gamma_1 - \gamma_3 \right)$$

$$= dS^1 dS^3 \cos \left(\frac{\pi}{2} - \gamma \right)$$

with $\gamma = \gamma_1 + \gamma_3$

$$\Rightarrow dS_i^1 dS_i^3 = dS^1 dS^3 \cos \left(\frac{\pi}{2} - \gamma \right) \quad \star$$

Shear Components (cont.)

- Recall: $X_i = x_i + u_i$

$$\therefore dS_i = dX_i = X_{i,j} dx_j = (\delta_{ij} + u_{i,j}) dx_j$$

$$\begin{aligned}\Rightarrow dS_i^1 dS_i^3 &= (\delta_{ij} + u_{i,j}) dx_j^1 (\delta_{ik} + u_{i,k}) dx_k^3 \\ &= (\delta_{jk} + u_{j,k} + u_{k,j} + u_{i,j} u_{i,k}) dx_j^1 dx_k^3\end{aligned}$$

- For this choice of geometry:


$$\begin{aligned}dx_1^1 &= ds^1 & dx_2^1 &= dx_3^1 = 0 \\ dx_2^3 &= dx_3^3 = 0 & dx_3^3 &= ds^3\end{aligned}$$

Shear Components (cont.)

- Then $j=1$ and $k=3$ survive:

$$\begin{aligned} dS_i^1 dS_i^3 &= (\delta_{13} + u_{1,3} + u_{3,1} + u_{i,1} u_{i,3}) ds^1 ds^3 \\ &= 2E_{13} ds^1 ds^3 \end{aligned}$$

E_{13} Lagrangian strain component

- Substitute into  $dS_i^1 dS_i^3 = dS^1 dS^3 \cos\left(\frac{\pi}{2} - \gamma\right)$
 $= 2E_{13} ds^1 ds^3$
 - Using the normal result: $\frac{dS^1}{ds^1} = \sqrt{1 + 2E_{11}}, \quad \frac{dS^3}{ds^3} = \sqrt{1 + 2E_{33}}$
- } \Rightarrow

Shear Components (cont.)

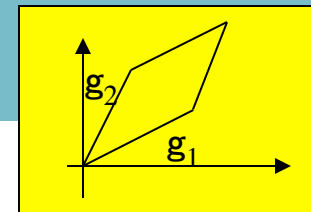
$$\Rightarrow \cos\left(\frac{\pi}{2} - \gamma\right) = \sin\gamma = \frac{2E_{13}}{\sqrt{1+2E_{11}}\sqrt{1+2E_{33}}}$$

- Under small strain conditions: $E_{11} \ll 1$, $E_{33} \ll 1$ and γ small

$$\sin\gamma \approx \gamma, \quad \sqrt{1+2E_{11}} \approx 1, \quad \sqrt{1+2E_{33}} \approx 1, \quad E_{13} \approx \epsilon_{13}$$

$$\therefore \gamma \approx 2\epsilon_{13} \quad \Rightarrow \quad \epsilon_{13} \approx \frac{1}{2}\gamma = \frac{1}{2}(\gamma_1 + \gamma_3)$$

\therefore shear strains ϵ_{ij} ($i \neq j$) represent $\frac{1}{2}$ (change of angle from $\frac{\pi}{2}$)
between x_i and x_j



Remarks

- Under small displacement gradients ε_{ij} attain physical meaning

$$\varepsilon_{ii}(\text{no sum}) = \frac{\text{change of length}}{\text{initial length}}$$
$$\varepsilon_{ij}(i \neq j) = \frac{1}{2} \left(\text{change of angle from } \frac{\pi}{2} \right)$$

- Sometimes shear strain is taken as “total change from $\pi/2$ ”. This is the **Engineering Strain**, but is NOT a tensor.

- Note that: $u_{i,j} = \varepsilon_{ij} + \omega_{ij}$ where $\omega_{ij} = \frac{1}{2}(u_{i,j} - u_{j,i})$

Clearly: $\omega_{ij} = 0$ ($i = j$) and $\omega_{ij} = -\omega_{ji}$ ($i \neq j$)

ω is a skew-symmetric tensor called the **rotation tensor**, e.g.

$$\omega_{13} = \frac{\gamma_1 - \gamma_3}{2}, \quad \text{average rotation about } x_2$$

Volume and Shape Changes

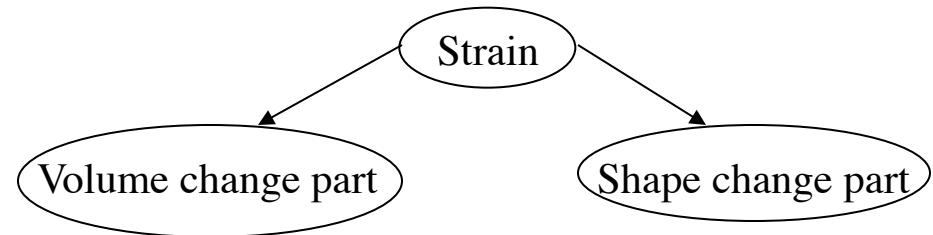
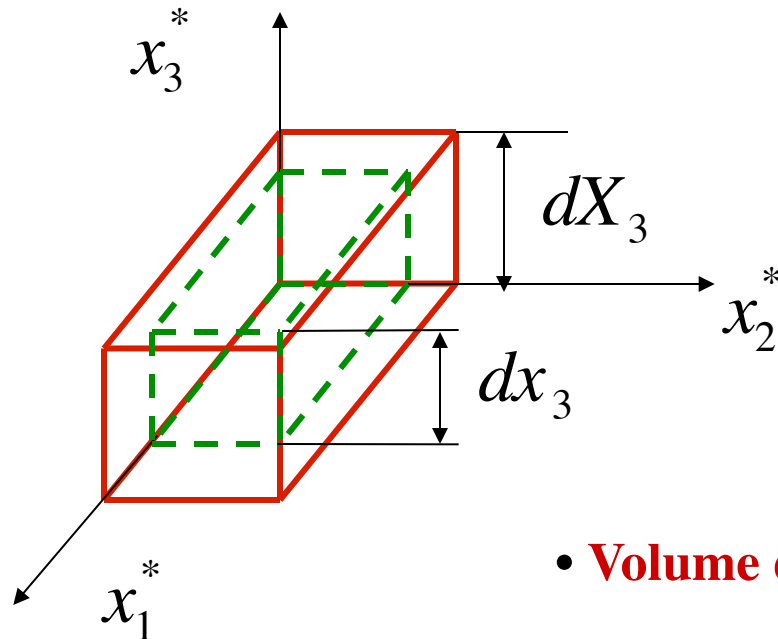
- Principal strains: $(\varepsilon_{ij} - \lambda \delta_{ij})n_j = 0 \Rightarrow \det[\varepsilon_{ij} - \lambda \delta_{ij}] = 0$
$$\Rightarrow \left. \begin{array}{l} \lambda_1 = \varepsilon_1 \\ \lambda_2 = \varepsilon_2 \\ \lambda_3 = \varepsilon_3 \end{array} \right\} \text{principal strains}$$

- In a principal frame for strain:

$$[\varepsilon_{ij}] = \begin{bmatrix} \varepsilon_1 & 0 & 0 \\ 0 & \varepsilon_2 & 0 \\ 0 & 0 & \varepsilon_3 \end{bmatrix}$$

- Principal frames for strain and stress coincide only for an **isotropic** material

Volume and Shape Changes (cont.)



• **Volume dilatation:** $\Delta = \frac{dV - dv}{dv} = \frac{\text{volume change}}{\text{original volume}}$

$$\left. \begin{aligned} dv &= dx_1 dx_2 dx_3 \quad \text{and} \quad dV = dX_1 dX_2 dX_3 \\ \text{But } dX_k &= (1 + \varepsilon_k) dx_k \quad (\text{no sum}) \end{aligned} \right\}$$

↑
principal strains

Volume and Shape Changes (cont.)

$$dv = dx_1 dx_2 dx_3$$

$$dV = (1 + \varepsilon_1) dx_1 (1 + \varepsilon_2) dx_2 (1 + \varepsilon_3) dx_3$$

$$\begin{aligned}\Delta &= \frac{dv[(1 + \varepsilon_1)(1 + \varepsilon_2)(1 + \varepsilon_3) - 1]}{dv} \\ &= 1 + (\varepsilon_1 + \varepsilon_2 + \varepsilon_3) + H.O.T + \dots - 1 \\ &\approx \varepsilon_1 + \varepsilon_2 + \varepsilon_3 \\ &= \varepsilon_{ii}\end{aligned}$$

Volume dilatation:

$$\Delta = \varepsilon_{ii} = tr(\underline{\varepsilon})$$

Volume and Shape Changes (cont.)

- Deviatoric and volumetric strain

$$[\epsilon_{ij}] = \begin{bmatrix} \epsilon_m & 0 & 0 \\ 0 & \epsilon_m & 0 \\ 0 & 0 & \epsilon_m \end{bmatrix} + \begin{bmatrix} \epsilon_{11} - \epsilon_m & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{12} & \epsilon_{22} - \epsilon_m & \epsilon_{23} \\ \epsilon_{13} & \epsilon_{23} & \epsilon_{33} - \epsilon_m \end{bmatrix}$$

Mean strain
(volumetric)

$$\epsilon_m = \frac{1}{3} \epsilon_{ii} \Rightarrow \Delta = 3\epsilon_m = \epsilon_{ii}$$

Volume change with no shape change

Deviatoric strain

$$\Delta = 0$$

Shape change at constant volume

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Compatibility

$$\epsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i})$$

- Given u_i can find ϵ_{ij}

- Given ϵ_{ij} , $u_i = ?$

ϵ_{ij}	\rightarrow	u_i
6 components		3 unknowns

- System is overdetermined

- Conditions of **Compatibility** are restrictions on ϵ_{ij} so that **single valued** u_i are produced

Compatibility (cont.)

$$\epsilon_{ij} = \frac{1}{2}(u_{i,j} + u_{j,i})$$

$$\epsilon_{ij,kl} = \frac{\partial^2}{\partial x_k \partial x_l} \left[\frac{1}{2}(u_{i,j} + u_{j,i}) \right] = \frac{1}{2}(u_{i,jkl} + u_{j,ikl})$$

$$\Rightarrow \epsilon_{ij,kl} + \epsilon_{kl,ij} = \frac{1}{2}(u_{i,jkl} + u_{j,ikl}) + \frac{1}{2}(u_{k,lij} + u_{l,kij})$$

Also:

$$\epsilon_{ik,jl} + \epsilon_{jl,ik} = \frac{1}{2}(u_{i,kjl} + u_{k,ijl}) + \frac{1}{2}(u_{j,lik} + u_{l,jik})$$

} Differentiation
order immaterial

$$\therefore \epsilon_{ij,kl} + \epsilon_{kl,ij} = \epsilon_{ik,jl} + \epsilon_{jl,ik}$$

Compatibility equations

of eqns. = ?

Compatibility (cont.)

- Only 6 are independent (why?):

$$\epsilon_{11,22} + \epsilon_{22,11} - 2\epsilon_{12,12} = 0$$

$$\epsilon_{22,33} + \epsilon_{33,22} - 2\epsilon_{23,23} = 0$$

$$\epsilon_{33,11} + \epsilon_{11,33} - 2\epsilon_{13,13} = 0$$

$$\epsilon_{12,13} + \epsilon_{13,12} - \epsilon_{23,11} - \epsilon_{11,23} = 0$$

$$\epsilon_{23,21} + \epsilon_{21,23} - \epsilon_{31,22} - \epsilon_{22,31} = 0$$

$$\epsilon_{31,32} + \epsilon_{32,31} - \epsilon_{12,33} - \epsilon_{33,12} = 0$$

- These are **necessary and sufficient** to ensure a unique u_i
- Examples...

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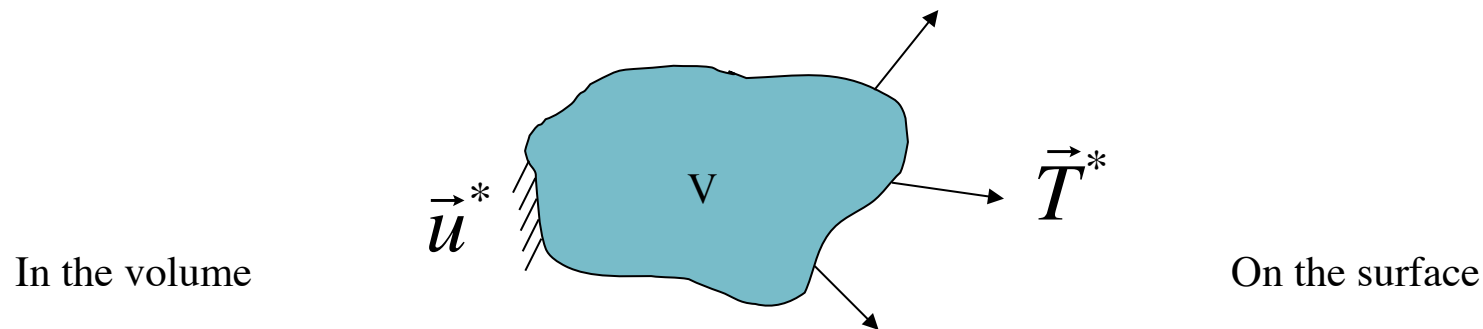
- Volume and shape changes

Compatibility



Summary of stress and strain

3.4 Summary of Stress and Strain



- Equilibrium:

$$\sigma_{ij,j} + f_i = 0$$

- On traction boundary:

$$T_i^* = \sigma_{ij} n_j$$

- Strain-displacement:

$$\epsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i})$$

- On displacement boundary:

$$u_i = u_i^*$$

of unknowns= ?

of equations= ?

- Need 6 additional equations: Must relate σ and ϵ

END OF CHAPTER 3