

Chapter 2 - Kinematics: Theory of Strain

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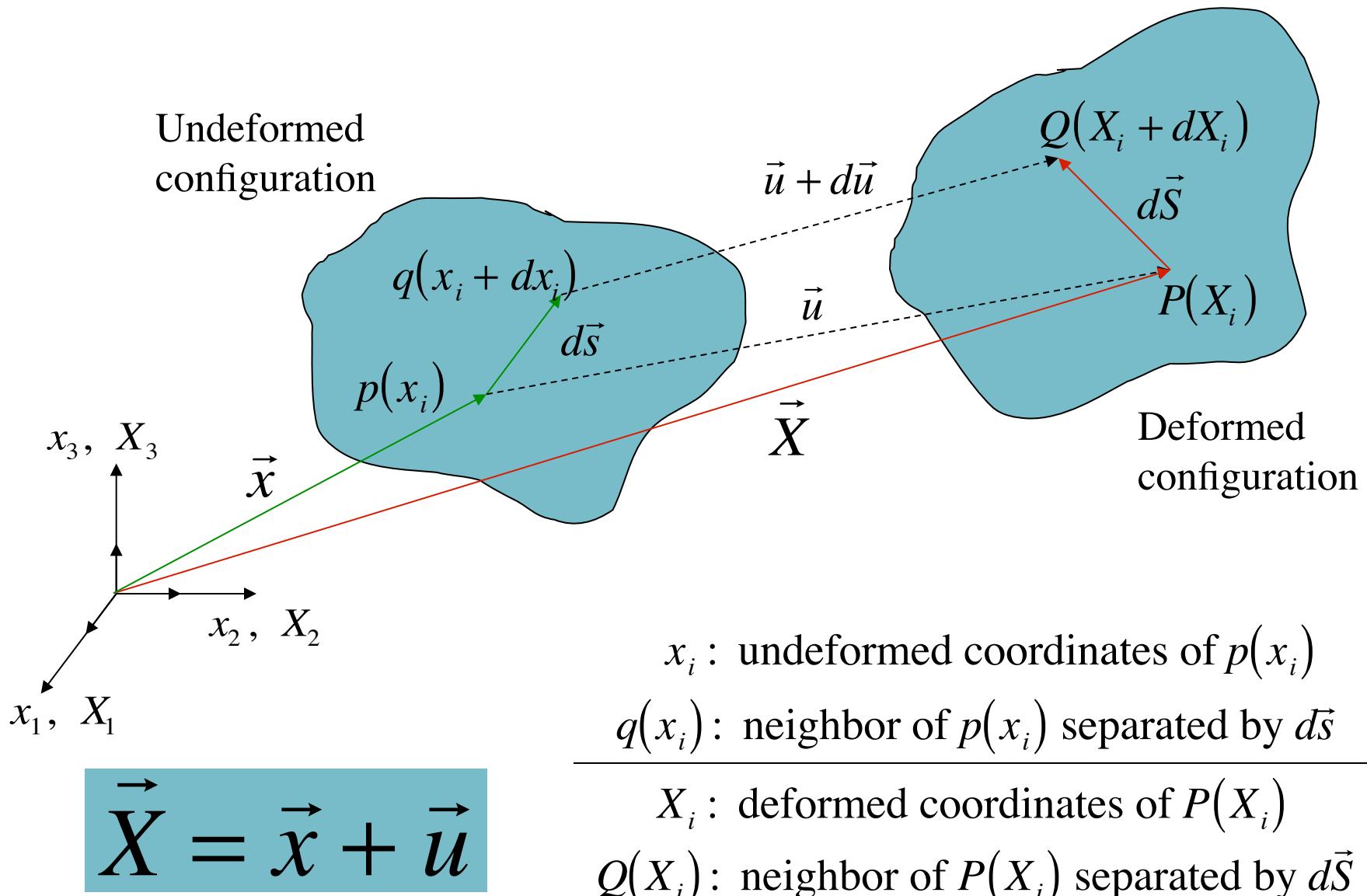
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Measures of Deformation



Measures of Deformation (cont.)

- If p and q move such that $\Rightarrow d\vec{s} = d\vec{S}$
 \Rightarrow rigid motion
 \Rightarrow no deformation

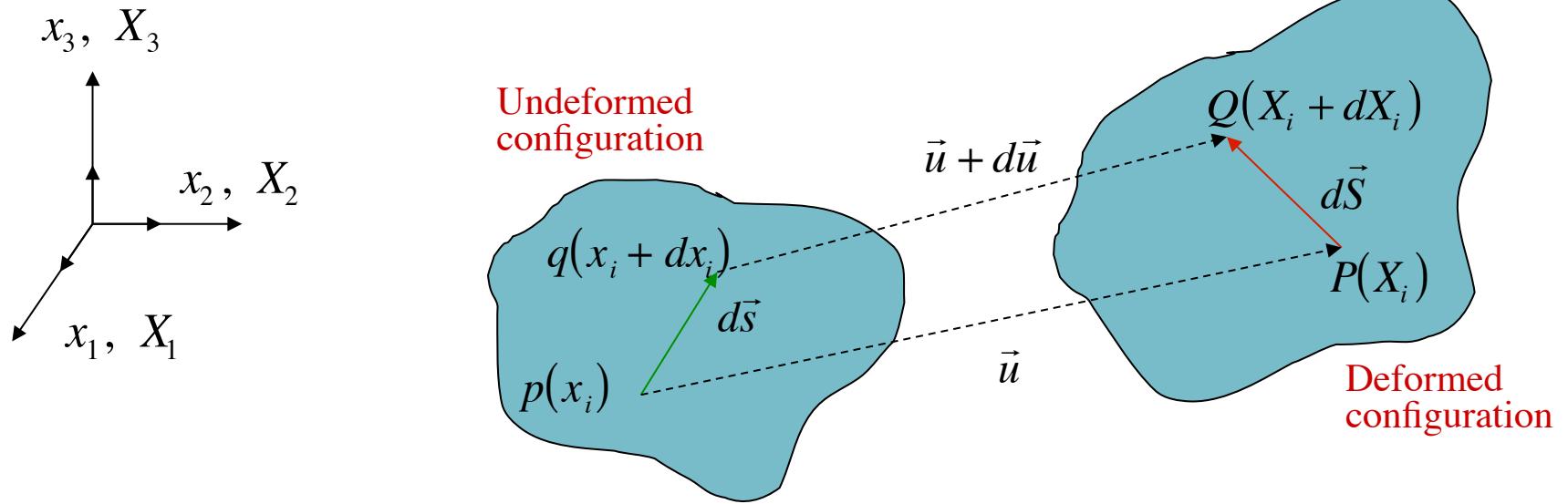
- Possible measures of deformation

$$dS - ds, \sqrt{dS} - \sqrt{ds}, f(dS) - f(ds)$$

with $ds = |d\vec{s}|$, $dS = |d\vec{S}|$

- Choose: $dS^2 - ds^2$

Lagrangian Strain Tensor



$$ds = |d\vec{s}| = \sqrt{dx_1^2 + dx_2^2 + dx_3^2} = \sqrt{dx_i dx_i}$$

$$dS = |d\vec{S}| = \sqrt{dX_1^2 + dX_2^2 + dX_3^2} = \sqrt{dX_i dX_i}$$

$$\Rightarrow dS^2 - ds^2 = dX_i dX_i - dx_i dx_i$$

Lagrangian Strain Tensor (cont.)

$dS^2 - ds^2 = dX_i dX_i - dx_i dx_i$ is a field (i.e. depends on coordinate)

- Choose **undeformed configuration** x_i as independent variable

$$X_i = X_i(x_1, x_2, x_3) = X_i(x_i)$$

$$\Rightarrow dX_i = \frac{\partial X_i}{\partial x_1} dx_1 + \frac{\partial X_i}{\partial x_2} dx_2 + \frac{\partial X_i}{\partial x_3} dx_3$$

$$\Rightarrow dX_i = X_{i,j} dx_j$$

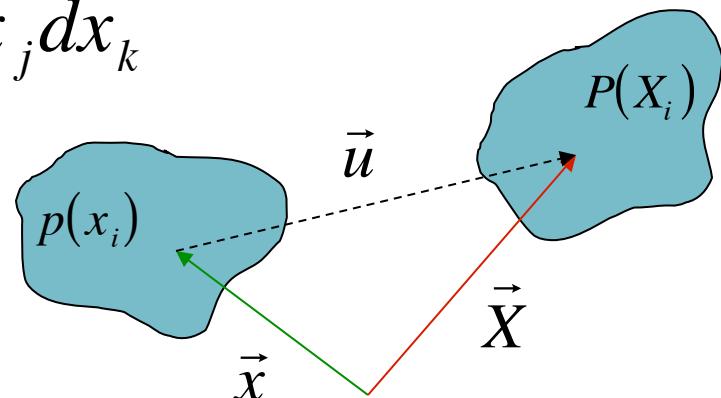
$$\Rightarrow dS^2 - ds^2 = (X_{i,j} dx_j)(X_{i,k} dx_k) - dx_i dx_i$$

$$\therefore dS^2 - ds^2 = (X_{i,j} X_{i,k} - \delta_{ij} \delta_{ik}) dx_j dx_k$$

Lagrangian Strain Tensor (cont.)

$$dS^2 - ds^2 = (X_{i,j}X_{i,k} - \delta_{ij}\delta_{ik})dx_jdx_k$$

$$\vec{X} = \vec{x} + \vec{u} \Rightarrow X_i = x_i + u_i$$



$$dS^2 - ds^2 = [(x_i + u_i)_{,j}(x_i + u_i)_{,k} - \delta_{jk}]dx_jdx_k$$

• Recall:

$$x_{i,j} = \frac{\partial x_i}{\partial x_j} = \delta_{ij}$$

Lagrangian Strain Tensor (cont.)

$$\begin{aligned} dS^2 - ds^2 &= \left[(\delta_{ij} + u_{i,j})(\delta_{ik} + u_{i,k}) - \delta_{jk} \right] dx_j dx_k \\ &= (\delta_{ij}\delta_{ik} + \delta_{ij}u_{i,k} + u_{i,j}\delta_{ik} + u_{i,j}u_{i,k} - \delta_{jk}) dx_j dx_k \\ &= (\delta_{jk} + u_{j,k} + u_{k,j} + u_{i,j}u_{i,k} - \delta_{jk}) dx_j dx_k \\ &= (u_{j,k} + u_{k,j} + u_{i,j}u_{i,k}) dx_j dx_k \end{aligned}$$

- Define:

$$E_{ij} = \varepsilon_{ij}^L = \frac{1}{2} (u_{i,j} + u_{j,i} + u_{k,i}u_{k,j})$$

- So that:

$$dS^2 - ds^2 = 2E_{ij} dx_i dx_j$$

Lagrangian Strain Tensor (cont.)

$$E_{ij} = \epsilon_{ij}^L = \frac{1}{2} \left(u_{i,j} + u_{j,i} + u_{k,i} u_{k,j} \right)$$

- E_{ij} are the components of the **Lagrangian** or **Green's** or **material** strain tensor
- Lagrangian: is expressed in terms of undeformed (or Lagrangian coordinates)
- By definition $E_{ij} = E_{ji}$, i.e. symmetric
- No assumptions on loading, material or amount of deformation

Eulerian Strain Tensor

- Recall $dS^2 - ds^2 = dX_i dX_i - dx_i dx_i$

- Choose deformed X_i as independent variable

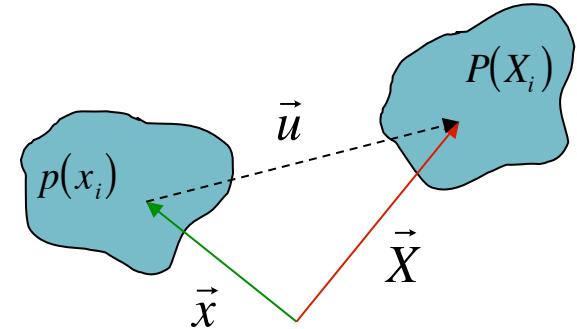
$$x_i = x_i(X_1, X_2, X_3) = x_i(X_i)$$

$$\Rightarrow dx_i = \frac{\partial x_i}{\partial X_1} dX_1 + \frac{\partial x_i}{\partial X_2} dX_2 + \frac{\partial x_i}{\partial X_3} dX_3$$

$$\Rightarrow dx_i = x_{i,J} dX_J \quad \leftarrow \text{Differentiation wrt X}$$

$$\Rightarrow dS^2 - ds^2 = dX_i dX_i - (x_{i,J} dX_J)(x_{i,K} dX_K)$$

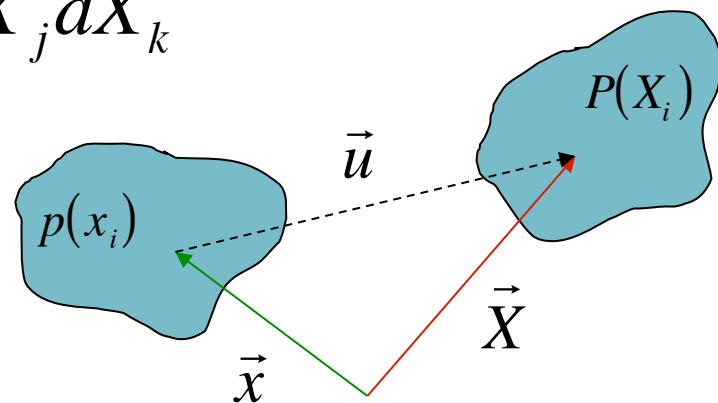
$$\therefore dS^2 - ds^2 = (\delta_{ij} \delta_{ik} - x_{i,J} x_{i,K}) dX_J dX_K$$



Eulerian Strain Tensor (cont.)

$$dS^2 - ds^2 = (\delta_{ij}\delta_{ik} - x_{i,J}x_{i,K})dX_j dX_k$$

$$\vec{X} = \vec{x} + \vec{u} \quad \Rightarrow \quad x_i = X_i - u_i$$



$$dS^2 - ds^2 = [\delta_{jk} - (X_i - u_i)_{,J} (X_i - u_i)_{,K}] dx_j dx_k$$

- Recall: $X_{i,J} = \frac{\partial X_i}{\partial X_j} = \delta_{ij}$

Eulerian Strain Tensor (cont.)

$$\begin{aligned} dS^2 - ds^2 &= [\delta_{jk} - (\delta_{ij} - u_{i,J})(\delta_{ik} - u_{i,K})] dX_j dX_k \\ &= (\delta_{jk} - \delta_{ij}\delta_{ik} + \delta_{ij}u_{i,K} + u_{i,J}\delta_{ik} - u_{i,J}u_{i,K}) dX_j dX_k \\ &= (\delta_{jk} - \delta_{jk} + u_{j,K} + u_{k,J} - u_{i,J}u_{i,K}) dX_j dX_k \\ &= (u_{j,K} + u_{k,J} - u_{i,J}u_{i,K}) dX_j dX_k \end{aligned}$$

- Define:

$$e_{ij} = \epsilon_{ij}^E = \frac{1}{2} (u_{i,J} + u_{j,I} - u_{k,I}u_{k,J})$$

Differentiation
wrt X_i

- So that:

$$dS^2 - ds^2 = 2e_{ij} dX_i dX_j$$

Eulerian Strain Tensor (cont.)

$$e_{ij} = \mathcal{E}_{ij}^E = \frac{1}{2} \left(u_{i,J} + u_{j,I} - u_{k,I} u_{k,J} \right)$$

Differentiation
wrt X_i

- e_{ij} are the components of the **Eulerian** or **Almansi** or **spatial** strain tensor
- Eulerian: is expressed in terms of deformed (or Eulerian coordinates)
- By definition $e_{ij} = e_{ji}$, i.e. symmetric
- No assumptions on loading, material or amount of deformation
- Differentiation is with respect to X_i

Infinitesimal Strain

- Lagrangian strain:

$$E_{ij} = \varepsilon_{ij}^L = \frac{1}{2} \left(u_{i,j} + u_{j,i} + u_{k,i} u_{k,j} \right)$$

- Eulerian strain:

$$e_{ij} = \varepsilon_{ij}^E = \frac{1}{2} \left(u_{i,J} + u_{j,I} - u_{k,I} u_{k,J} \right)$$

- Assumptions:

(a) If $|u_{i,j}| \ll 1$ then $u_{k,i} u_{k,j}$ can be ignored wrt $u_{i,j}$

(b) If $x_i \approx X_i$ then $\frac{\partial(\)}{\partial x_i} \approx \frac{\partial(\)}{\partial X_i}$

$$E_{ij} \approx e_{ij} \approx \frac{1}{2} \left(u_{i,j} + u_{j,i} \right) \equiv \varepsilon_{ij}$$

Infinitesimal Strain (cont.)

$$\varepsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i})$$

- ε_{ij} are the components of the **infinitesimal strain tensor**
- By definition $\varepsilon_{ij} = \varepsilon_{ji}$, i.e. symmetric
- Condition (a) implies small displacement gradients. Condition (b) implies small displacements
- Above equations are the **strain-displacement relations**
- # of equations vs. # of unknowns
- Dimensions?

Infinitesimal Strain (cont.)

$$\varepsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i}) = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

- Expanding:

$$\left. \begin{aligned} \varepsilon_{11} &= \frac{\partial u_1}{\partial x_1} \\ \varepsilon_{22} &= \frac{\partial u_2}{\partial x_2} \\ \varepsilon_{33} &= \frac{\partial u_3}{\partial x_3} \end{aligned} \right\} \text{Normal Strains}$$

$$\left. \begin{aligned} \varepsilon_{12} &= \varepsilon_{21} = \frac{1}{2} \left(\frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) \\ \varepsilon_{13} &= \varepsilon_{31} = \frac{1}{2} \left(\frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1} \right) \\ \varepsilon_{23} &= \varepsilon_{32} = \frac{1}{2} \left(\frac{\partial u_2}{\partial x_3} + \frac{\partial u_3}{\partial x_2} \right) \end{aligned} \right\} \text{Shear Strains}$$

Infinitesimal Strain (cont.)

$$[\underline{\epsilon}_{ij}] = \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ & \epsilon_{22} & \epsilon_{23} \\ sym. & & \epsilon_{33} \end{bmatrix}$$

- Since infinitesimal strain is a two-tensor it transforms (rotates) according to:

$$\underline{\epsilon}'_{ij} = \alpha_{ik} \alpha_{jl} \epsilon_{kl}$$

or

$$[\underline{\epsilon}'] = [R][\underline{\epsilon}][R]^T$$

Engineering Strain

- Sometimes (e.g. Timoshenko and Goodier) infinitesimal shear strain is defined:

$$\gamma_{ij} = (u_{i,j} + u_{j,i}) \quad i \neq j$$

- But then strain is NOT a two-tensor

$$[\epsilon_{ij}] = \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ & \epsilon_{22} & \epsilon_{23} \\ sym. & & \epsilon_{33} \end{bmatrix} \quad \text{A tensor}$$

$$[\epsilon_{ij}] = \begin{bmatrix} \epsilon_{11} & \gamma_{12} & \gamma_{13} \\ & \epsilon_{22} & \gamma_{23} \\ sym. & & \epsilon_{33} \end{bmatrix} \quad \text{Not a tensor - \textcolor{red}{Engineering strain}}$$

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- Measures of deformation
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★ Physical interpretation

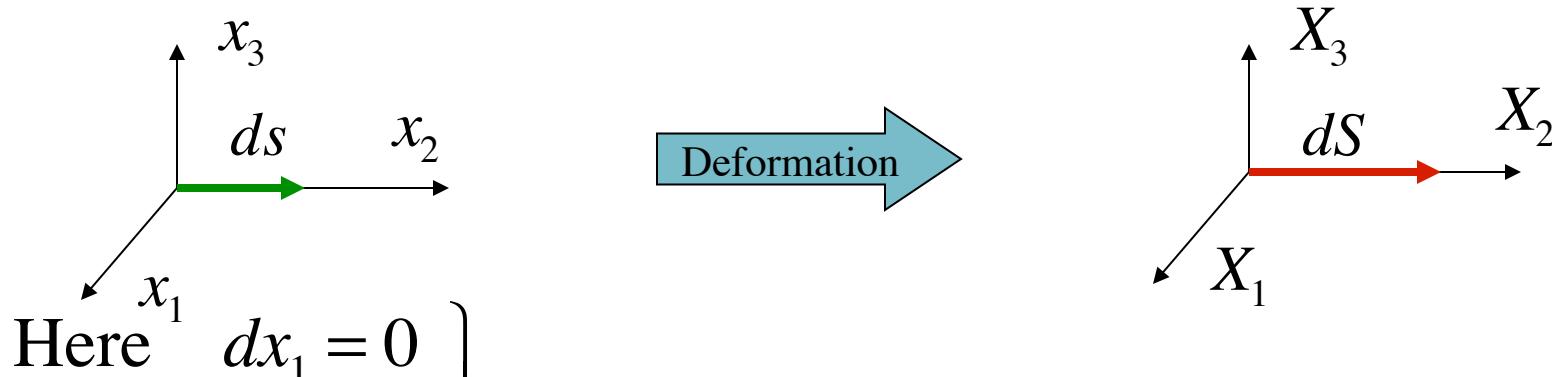
- Normal components
- Shear components
- Volume and shape changes

Compatibility

Summary of stress and strain

Normal components

Take $d\vec{s} \parallel dx_2$



$$\left. \begin{aligned} \text{Here } dx_1 &= 0 \\ dx_2 &= ds \\ dx_3 &= 0 \end{aligned} \right\} \Rightarrow dS^2 - ds^2 = 2E_{ij}dx_i dx_j = 2E_{22}ds^2$$

$$\Rightarrow dS^2 = (1 + 2E_{22})ds^2$$

$$\Rightarrow dS = \sqrt{(1 + 2E_{22})}ds$$

$$\therefore \frac{dS}{ds} = \frac{\text{final length}}{\text{original length}} = \sqrt{1 + 2E_{22}}$$

Stretch ratio along x_2

Normal components (cont.)

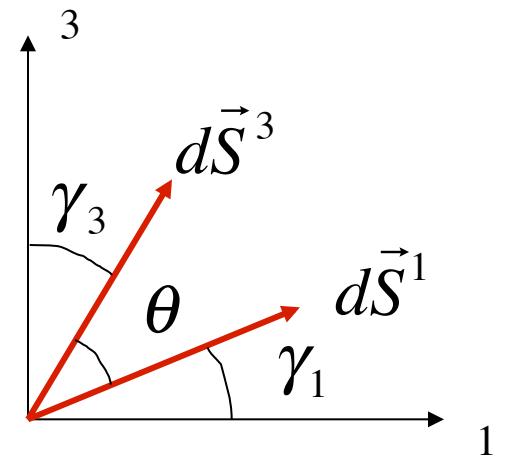
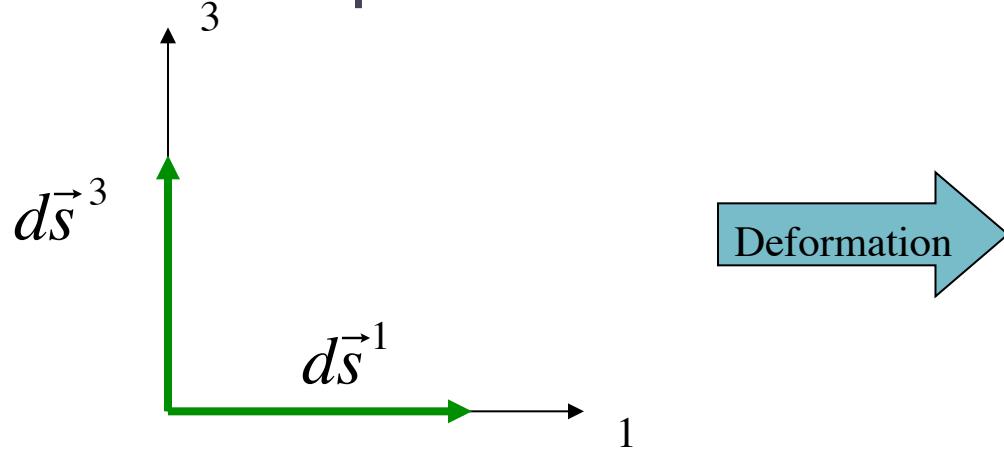
- **Extension ratio:**
$$\frac{dS - ds}{ds} = \frac{\text{change of length}}{\text{initial length}} = \sqrt{1 + 2E_{22}} - 1$$

$$\therefore E_{22} \propto \Delta L / L \text{ along } x_2$$

- For small strains $E_{22} \ll 1$:
$$\begin{aligned} \frac{dS - ds}{ds} &= \sqrt{1 + 2E_{22}} - 1 \\ &= (1 + 2E_{22})^{1/2} - 1 \\ &= 1 + \frac{1}{2}(2E_{22}) + \dots - 1 \\ &\approx E_{22} = \varepsilon_{22} \end{aligned}$$

\therefore normal strains ε_{ii} (*nosum*) represent $\frac{\text{change of length}}{\text{initial length}}$ along x_i

Shear Components



$$\begin{aligned} d\vec{S}^1 \cdot d\vec{S}^3 &= dS_i^1 dS_i^3 \\ &= dS^1 dS^3 \cos \theta \end{aligned}$$

$$= dS^1 dS^3 \cos\left(\frac{\pi}{2} - \gamma_1 - \gamma_3\right)$$

$$= dS^1 dS^3 \cos\left(\frac{\pi}{2} - \gamma\right)$$

with $\gamma = \gamma_1 + \gamma_3$

$$\Rightarrow dS_i^1 dS_i^3 = dS^1 dS^3 \cos\left(\frac{\pi}{2} - \gamma\right) \quad \star$$

Shear Components (cont.)

- Recall: $X_i = x_i + u_i$

$$\therefore dS_i = dX_i = X_{i,j} dx_j = (\delta_{ij} + u_{i,j}) dx_j$$

$$\begin{aligned} \Rightarrow dS_i^1 dS_i^3 &= (\delta_{ij} + u_{i,j}) dx_j^1 (\delta_{ik} + u_{i,k}) dx_k^3 \\ &= (\delta_{jk} + u_{j,k} + u_{k,j} + u_{i,j} u_{i,k}) dx_j^1 dx_k^3 \end{aligned}$$

- For this choice of geometry:

$$dx_1^1 = ds^1 \quad dx_2^1 = dx_3^1 = 0$$

$$dx_2^3 = dx_3^3 = 0 \quad dx_3^3 = ds^3$$

Shear Components (cont.)

- Then $j=1$ and $k=3$ survive:

$$\begin{aligned} dS_i^1 dS_i^3 &= (\delta_{13} + u_{1,3} + u_{3,1} + u_{i,1}u_{i,3}) ds^1 ds^3 \\ &= 2E_{13} ds^1 ds^3 \end{aligned}$$

E_{13} Lagrangian strain component

- Substitute into 

$$\begin{aligned} dS_i^1 dS_i^3 &= dS^1 dS^3 \cos\left(\frac{\pi}{2} - \gamma\right) \\ &= 2E_{13} ds^1 ds^3 \end{aligned} \quad \left. \right\} \Rightarrow$$

- Using the normal result:

$$\frac{dS^1}{ds^1} = \sqrt{1 + 2E_{11}}, \quad \frac{dS^3}{ds^3} = \sqrt{1 + 2E_{33}}$$

Shear Components (cont.)

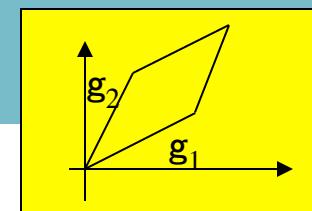
$$\Rightarrow \cos\left(\frac{\pi}{2} - \gamma\right) = \sin \gamma = \frac{2E_{13}}{\sqrt{1+2E_{11}}\sqrt{1+2E_{33}}}$$

- Under small strain conditions: $E_{11} \ll 1$, $E_{33} \ll 1$ and γ small

$$\sin \gamma \approx \gamma, \quad \sqrt{1+2E_{11}} \approx 1, \quad \sqrt{1+2E_{33}} \approx 1, \quad E_{13} \approx \varepsilon_{13}$$

$$\therefore \gamma \approx 2\varepsilon_{13} \Rightarrow \varepsilon_{13} \approx \frac{1}{2}\gamma = \frac{1}{2}(\gamma_1 + \gamma_3)$$

\therefore shear strains ε_{ij} ($i \neq j$) represent $\frac{1}{2}\left(\text{change of angle from } \frac{\pi}{2}\right)$
between x_i and x_j



Remarks

- Under small displacement gradients ε_{ij} attain physical meaning

$$\varepsilon_{ii}(\text{nosum}) = \frac{\text{change of length}}{\text{initial length}}$$

$$\varepsilon_{ij} (i \neq j) = \frac{1}{2} \left(\text{change of angle from } \frac{\pi}{2} \right)$$

- Sometimes shear strain is taken as “total change from $\pi/2$ ”. This is the **Engineering Strain**, but is NOT a tensor.

- Note that: $u_{i,j} = \varepsilon_{ij} + \omega_{ij}$ where $\omega_{ij} = \frac{1}{2} (u_{i,j} - u_{j,i})$

Clearly: $\omega_{ij} = 0$ ($i = j$) and $\omega_{ij} = -\omega_{ji}$ ($i \neq j$)

ω is a skew-symmetric tensor called the **rotation tensor**, e.g.

$$\omega_{13} = \frac{\gamma_1 - \gamma_3}{2}, \quad \text{average rotation about } x_2$$

Volume and Shape Changes

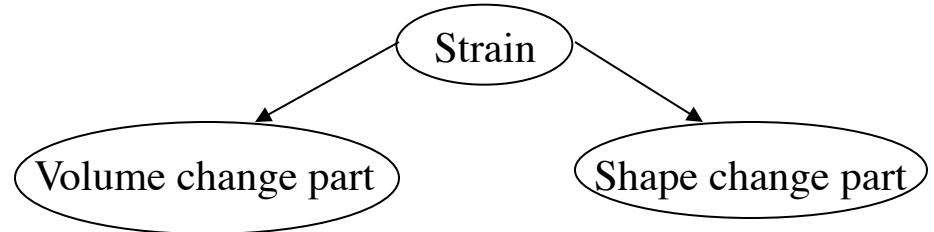
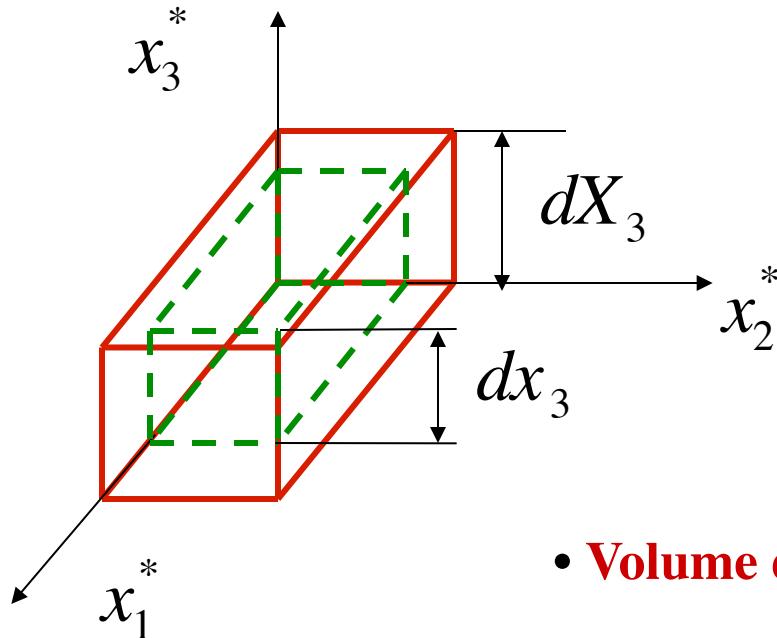
- Principal strains: $(\varepsilon_{ij} - \lambda \delta_{ij})n_j = 0 \Rightarrow \det[\varepsilon_{ij} - \lambda \delta_{ij}] = 0$
$$\Rightarrow \begin{cases} \lambda_1 = \varepsilon_1 \\ \lambda_2 = \varepsilon_2 \\ \lambda_3 = \varepsilon_3 \end{cases} \text{ principal strains}$$

- In a principal frame for strain:

$$[\varepsilon_{ij}] = \begin{bmatrix} \varepsilon_1 & 0 & 0 \\ 0 & \varepsilon_2 & 0 \\ 0 & 0 & \varepsilon_3 \end{bmatrix}$$

- Principal frames for strain and stress coincide only for an **isotropic** material

Volume and Shape Changes (cont.)



• **Volume dilatation:** $\Delta = \frac{dV - dv}{dv} = \frac{\text{volume change}}{\text{original volume}}$

$$dv = dx_1 dx_2 dx_3 \quad \text{and} \quad dV = dX_1 dX_2 dX_3 \quad \left. \right\}$$

But $dX_k = (1 + \varepsilon_k) dx_k$ (no sum)

principal strains

Volume and Shape Changes (cont.)

$$dv = dx_1 dx_2 dx_3$$

$$dV = (1 + \varepsilon_1) dx_1 (1 + \varepsilon_2) dx_2 (1 + \varepsilon_3) dx_3$$

$$\begin{aligned}\Delta &= \frac{dv[(1 + \varepsilon_1)(1 + \varepsilon_2)(1 + \varepsilon_3) - 1]}{dv} \\ &= 1 + (\varepsilon_1 + \varepsilon_2 + \varepsilon_3) + H.O.T + \dots - 1 \\ &\approx \varepsilon_1 + \varepsilon_2 + \varepsilon_3 \\ &= \varepsilon_{ii}\end{aligned}$$

Volume dilatation:

$$\Delta = \varepsilon_{ii} = \text{tr}(\underline{\varepsilon})$$

Volume and Shape Changes (cont.)

- Deviatoric and volumetric strain

$$[\varepsilon_{ij}] = \begin{bmatrix} \varepsilon_m & 0 & 0 \\ 0 & \varepsilon_m & 0 \\ 0 & 0 & \varepsilon_m \end{bmatrix} + \begin{bmatrix} \varepsilon_{11} - \varepsilon_m & \varepsilon_{12} & \varepsilon_{13} \\ \varepsilon_{12} & \varepsilon_{22} - \varepsilon_m & \varepsilon_{23} \\ \varepsilon_{13} & \varepsilon_{23} & \varepsilon_{33} - \varepsilon_m \end{bmatrix}$$

Mean strain
(volumetric)

$$\varepsilon_m = \frac{1}{3} \varepsilon_{ii} \Rightarrow \Delta = 3\varepsilon_m = \varepsilon_{ii}$$

Volume change with no shape change

Deviatoric strain

$$\Delta = 0$$

Shape change at constant volume

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★ Compatibility

Summary of stress and strain

Compatibility

$$\varepsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i})$$

- Given u_i can find ε_{ij}
 - Given ε_{ij} , $u_i = ?$
 - System is overdetermined
 - Conditions of **Compatibility** are restrictions on ε_{ij} so that **single valued** u_i are produced

$\varepsilon_{ij} \rightarrow u_i$

6 components 3 unknowns

Compatibility (cont.)

$$\varepsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i})$$

$$\varepsilon_{ij,kl} = \frac{\partial^2}{\partial x_k \partial x_l} \left[\frac{1}{2} (u_{i,j} + u_{j,i}) \right] = \frac{1}{2} (u_{i,jkl} + u_{j,ikl})$$

$$\Rightarrow \varepsilon_{ij,kl} + \varepsilon_{kl,ij} = \frac{1}{2} (u_{i,jkl} + u_{j,ikl}) + \frac{1}{2} (u_{k,lij} + u_{l,kij})$$

Also:

$$\varepsilon_{ik,jl} + \varepsilon_{jl,ik} = \frac{1}{2} (u_{i,kjl} + u_{k,ijl}) + \frac{1}{2} (u_{j,lik} + u_{l,jik})$$

Differentiation
order immaterial

$$\therefore \varepsilon_{ij,kl} + \varepsilon_{kl,ij} = \varepsilon_{ik,jl} + \varepsilon_{jl,ik}$$

Compatibility equations
of eqns. = ?

Compatibility (cont.)

- Only 6 are independent (why?):

$$\varepsilon_{11,22} + \varepsilon_{22,11} - 2\varepsilon_{12,12} = 0$$

$$\varepsilon_{22,33} + \varepsilon_{33,22} - 2\varepsilon_{23,23} = 0$$

$$\varepsilon_{33,11} + \varepsilon_{11,33} - 2\varepsilon_{13,13} = 0$$

$$\varepsilon_{12,13} + \varepsilon_{13,12} - \varepsilon_{23,11} - \varepsilon_{11,23} = 0$$

$$\varepsilon_{23,21} + \varepsilon_{21,23} - \varepsilon_{31,22} - \varepsilon_{22,31} = 0$$

$$\varepsilon_{31,32} + \varepsilon_{32,31} - \varepsilon_{12,33} - \varepsilon_{33,12} = 0$$

- These are **necessary and sufficient** to ensure a unique u_i

- Examples...

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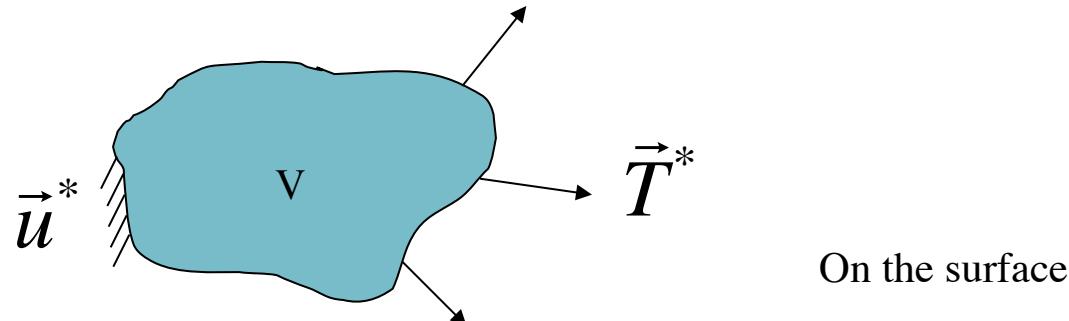
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Compatibility

★ **Summary of stress and strain**

3.4 Summary of Stress and Strain

In the volume



On the surface

- Equilibrium:

$$\sigma_{ij,j} + f_i = 0$$

- On traction boundary:

$$T_i^* = \sigma_{ij} n_j$$

- Strain-displacement:

$$\varepsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i})$$

- On displacement boundary:

$$u_i = u_i^*$$

of unknowns = ?

of equations = ?

- Need 6 additional equations: Must relate σ and ε

END OF CHAPTER 3