Question 4

Part 1

$$\sigma_{\theta\theta}(r=a) = 0$$

$$\sigma_{r\theta}(r=a) = 0$$

Part 2

$$\sigma_{rr}(\frac{r}{a} = \infty) = \sigma_0 \sin^2(\theta)$$

$$\sigma_{r\theta}(\frac{r}{a} = \infty) = -\frac{\sigma_0 \sin(2\theta)}{2}$$

$$\sigma_{\theta\theta}(\frac{r}{a} = \infty) = \sigma_0 \sin^2(\theta)$$

Part 3

$$\sigma_{xx}(\frac{r}{a} = \infty) = -\sigma_0 \sin^2(\theta) \cos(2\theta)$$

$$\sigma_{yy}(\frac{r}{a} = \infty) = -\frac{\sigma_0 \sin(4\theta)}{4}$$

$$\sigma_{xy}(\frac{r}{a} = \infty) = -\sigma_0 \sin^2(\theta) \cos(2\theta)$$

Part 4

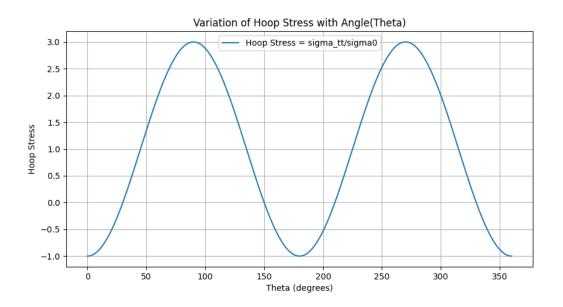


Figure 1: Hoop Stress vs. $Angle(\theta)$

The maximum value of hs is 3, which occurs at $\theta = 90.0$ degrees.

Python Code for Q4

```
from sympy import symbols,log,
   diff,cos,sin,simplify,oo,limit,solveset,pi,Interval,latex
import matplotlib.pyplot as plt
import numpy as np
sigma0, a, r, theta = symbols('sigma0 a r theta')
phi = -0.5*sigma0*a**2*log(r) + 0.25*sigma0*r**2 + 0.25*sigma0*(2*a**2 - r**2)
   - a^{**4}/(r^{**2}) *cos(2*theta)
dphi_dr = diff(phi,r)
dphi_dtheta = diff(phi,theta)
d2phi_dr2 = diff(dphi_dr,r)
d2phi_dtheta2 =diff(dphi_dtheta,r)
d2phi_drt = diff(dphi_dr,theta)
LHS = (d2phi_dr2 + 1/r*dphi_dr + 1/(r**2)*d2phi_dtheta2)**2
sigma_rr = 1/r*dphi_dr + 1/(r**2)*d2phi_dtheta2
sigma_rtheta = (1/r**2)*dphi_dtheta-1/r*d2phi_drt #- diff((1/r)*dphi_dtheta,r)
sigma_tt = d2phi_dr2
# Part 1 r = a
```

```
s_rr_eva = sigma_rr.subs({r:a})
s_rt_eva = sigma_rtheta.subs({r:a})
print('Part 1')
print('sigma_rr (r=a) =', latex(simplify(s_rr_eva)))
print('sigma_rt (r=a) =', latex(simplify(s_rt_eva)))
# Part 2 r/a-->inf
s_rr_eva2 = limit(sigma_rr,r,oo)
s_rt_eva2 = limit(sigma_rtheta,r,oo)
s_tt_eva2 = limit(sigma_tt,r,oo)
print('Part 2')
print('sigma_rr (r/a=inf) =', latex(simplify(s_rr_eva2)))
print('sigma_rt (r/a=inf) =', latex(simplify(s_rt_eva2)))
print('sigma_tt (r/a=inf) =', latex(simplify(s_tt_eva2)))
# Part 3 x&y-->inf
# for the convertion from sigma_rr to sigma_xx, see the reference
sigma_xx = (sigma_rr * sin(theta) + sigma_rtheta * cos(theta))*sin(theta) +
   (sigma_rtheta * sin(theta) + sigma_tt * cos(theta))*cos(theta)
sigma_yy = (sigma_rr * cos(theta) + sigma_rtheta * sin(theta))*sin(theta) +
   (sigma_rtheta * cos(theta) + sigma_tt * sin(theta))*cos(theta)
sigma_xy = (sigma_rr * sin(theta) + sigma_rtheta * cos(theta))*sin(theta) +
   (sigma_rtheta * sin(theta) + sigma_tt * cos(theta))*cos(theta)
s_xx_eva = limit(sigma_xx,r,oo)
s_yy_eva = limit(sigma_yy,r,oo)
s_xy_eva = limit(sigma_xy,r,oo)
print('Part 3')
print('sigma_xx (x&y=inf) =', latex(simplify(s_xx_eva)))
print('sigma_yy (x&y=inf) =', latex(simplify(s_yy_eva)))
print('sigma_xy (x&y=inf) =', latex(simplify(s_xy_eva)))
print('Part 4')
import sympy as sp
# Part 4
print('Part 4')
hs = sigma_tt / sigma0
hs_{eva} = hs.subs(r, a)
theta_radians = np.radians(np.linspace(0, 360, 400))
hs_values = [hs_eva.subs(theta, rad).evalf() for rad in theta_radians]
# Differentiating and solving for critical points
hs_prime = sp.diff(hs_eva, theta)
critical_points = sp.solveset(hs_prime, theta, domain=sp.Interval(0, 2 *
   sp.pi))
# Checking and evaluating at critical points
max_value = -float('inf')
theta_max = 0
if isinstance(critical_points, sp.FiniteSet):
   for point in critical_points:
      # Ensure that point is real
```

```
if point.is_real:
          value = hs_eva.subs(theta, point).evalf()
          if value > max_value:
              max value = value
              theta_max = point
# Convert theta_max from radians to degrees
if theta_max != 0: # Check to ensure theta_max was set
   theta_max_degrees = np.degrees(float(theta_max))
   print(f"The maximum value of hs is {max_value}, which occurs at theta =
       {theta_max_degrees} degrees.")
else:
   print("No real critical points found within the domain.")
plt.figure(figsize=(10, 5))
plt.plot(np.linspace(0, 360, 400), hs_values, label=f'Hoop Stress =
   sigma_tt/sigma0')
plt.title('Variation of Hoop Stress with Angle(Theta)')
plt.xlabel('Theta (degrees)')
plt.ylabel('Hoop Stress')
plt.grid(True)
plt.legend()
plt.show()
```

Reference for Q4

Reference Link

Question 5

```
r_2 = 0.05

r_1 = 0.025

p_1 = 0forccase 2 = 5e6forccase 1

p_2 = -5e6forccase 2 = 0forccase 1
```

Lastly, though in the statement of all the problems, validation with FEM solvers are required, I actually do not have access to Abaqus or Ansys at this moment, so the validation part remains for future work.

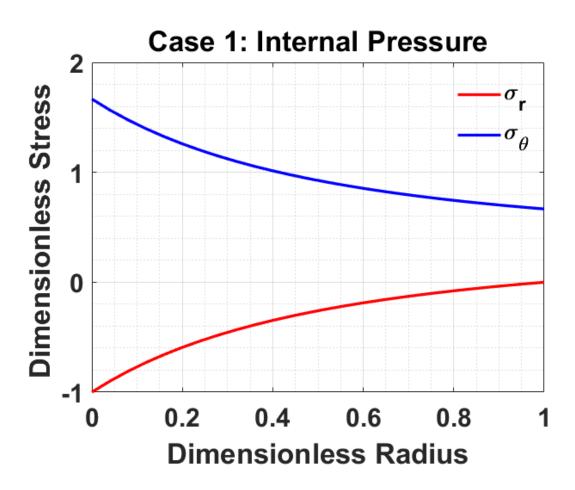


Figure 2: Case 1

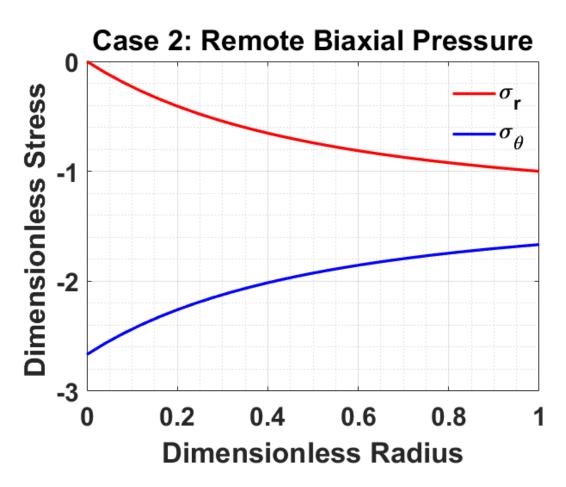


Figure 3: Case 2