

Problem

Create a bar with 3 pin supports and a linear actuator that can lift the maximum possible weight to the highest possible height ($= 50\text{cm}$)

Design Choices

Bar = 1.20m Activator: $a = 1.0\text{m}$ from the left pivot along the bar

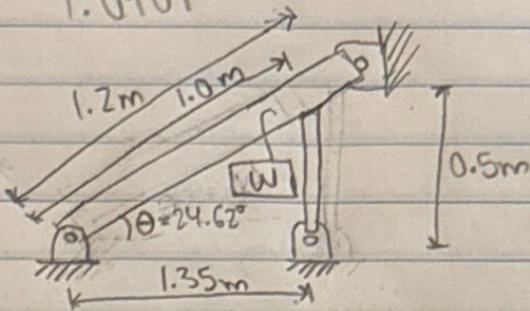
Left pivot: $x = 0$ Right ground anchor for activator at $x = 1.35\text{m}$ $h_{\max} = 0.5\text{m}$

Previous Static Analysis: $\theta = \sin^{-1}\left(\frac{0.5}{1.2}\right) = 24.62^\circ$ $x_a = a \cos \theta = 0.9091\text{m}$

Static load Used = $9000\text{N} = F$ $y_a = a \sin \theta = 0.4167\text{m}$

$$W_{\max} = \frac{Fd}{L_{\text{bar}} \cos \theta} = \frac{(9000)(0.9272)}{1.0909} = 7649.46\text{N} \approx 7.65\text{kN}$$

Final Mechanism:



Now, the bar is no longer rigid

Assume: linear Euler-Bernoulli beam theory applies, $F = 9000\text{N}$, $W = 7650\text{N}$, $E = 200\text{GPa}$

Deflection at free tip W : $\delta_W = \frac{WL^3}{3EI}$ Deflection at free tip P : $\delta_P = \frac{Pa^2(3L-a)}{6EI}$ $\delta_{\text{tot}} = \delta_W + \delta_P$
due to load P

$$A = 6.16\text{in}^2 = 0.003974\text{m}^2 \quad t = 0.00635\text{m} \quad R_m = \frac{A}{2nt} = 0.0996\text{m} \quad D_0 = 0.2056\text{m} \quad I = 1.973 \times 10^5 \text{mm}^4$$

$$\delta_W = \frac{WL^3}{3EI} = 1.12\text{mm} \quad \delta_P = \frac{Pa^2(3L-a)}{6EI} = 0.92\text{mm} \quad \delta_{\text{tot}} = \delta_W + \delta_P = 2.03\text{mm}$$

$$2\% \text{ of } L = 24\text{mm} \quad \delta_{\text{tot}} = 2.03\text{mm} < 24\text{mm}$$

New Mechanism:

