



Air Pump for Paddle Board



Aahil Ali, Christina Ge, Cion Kim,
Langston Johnson



Our Device:

- Pumps air through the hose attached
- Two cylinders and the switch lets you change between high-volume mode (fast, low pressure), and high-pressure mode (easier pumping when the inflatable is almost full).
- There is a pressure gauge embedded in the handle.



Major Dimensions

Mini tube: 15mm

Small tube : 60 mm diameter

Big tube : 70mm diameter

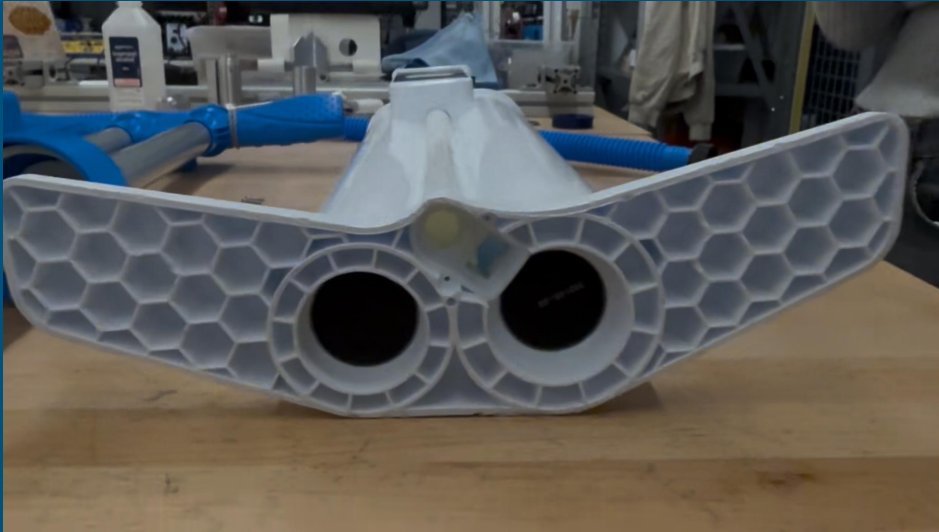
Height: 470 mm

Flexible tube: 25mm diameter, 1500mm length



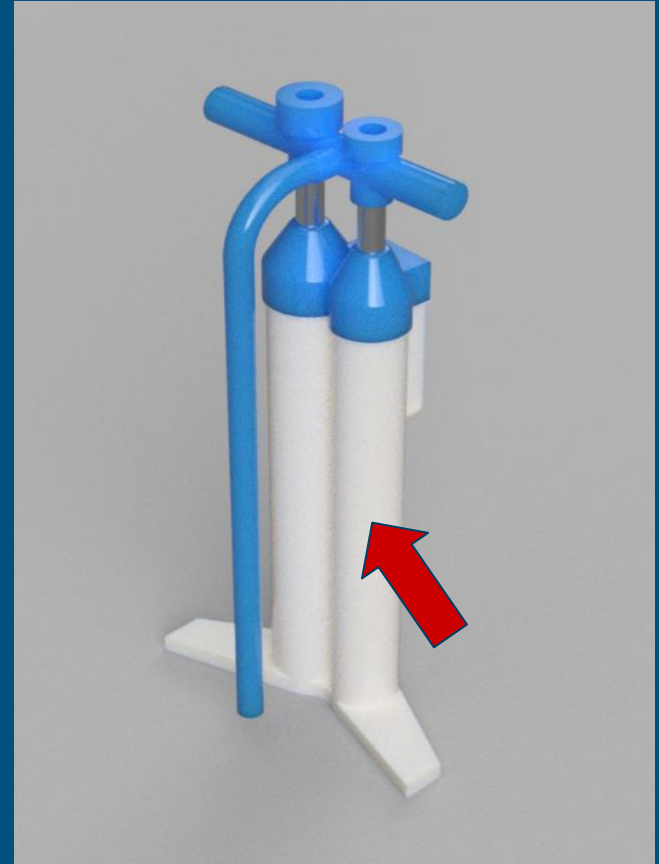
Dissection





Assumptions

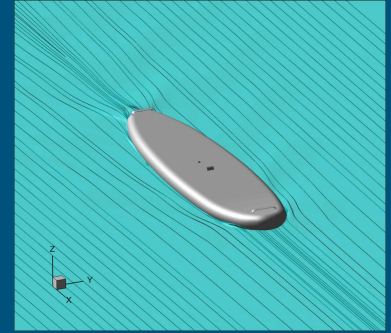
- We are doing calculations assuming we are using the mode 3, when the small tubes is open and big tube is closed to pump the paddle board
- We are also assuming the tubes as perfectly cylindrical tubes for the ease of calculation and analysis



Governing Equations / Parametrization

- The **Mean Flow Approximation** $Q=UA$
- Our flow is incompressible so our mass flow rate is constant and we can apply the **Continuity Equation**: $A_1 V_1 = A_2 V_2$.
- A **Pressure Volume Relationship** $W=P\Delta V$
- The Reynolds Number of the fluid system varies at its flows between different pipes. $Re = \rho UL/\mu$.
- **Viscous dissipation factor** $f_D = 2\Delta B_{loss} d/\rho(U^2)L$
- Bernoulli's Equation $B = p + \rho U^2/2 + \rho gh$

Parametrization



- Governing parameters: Reynolds number (Re) and Mach number (Ma)
- Flow: air in the 25 mm diameter, 1.5 m flexible hose from the barrel
- **Scale: length = hose inner diameter, velocity = average hose speed $U = Q / A$**
- Re using $(\rho, U, D, \mu) > 4000 \rightarrow$ turbulent hose flow
- $Ma = U / \text{speed of sound} \approx 0.02 \rightarrow$ air is effectively incompressible
- Same Re and Ma definitions for Modes 1-3 \rightarrow changing mode only changes U and Re as well

Parametrization

- Re controls the friction factor and pressure loss in the hose
- $Ma \ll 1$ in all modes, so compressibility does not affect performance and can be treated as negligible in our calculations
- Switching to larger barrels and faster strokes increases Q and U , which raises Re , fills the board faster, but also increases hose losses and handle force
- Changing the hose design to be shorter, smoother, or have slightly larger diameter reduces losses and handle force, while a greater viscous fluid would make losses so large the pump would be difficult to use

Calculations

Quantitative calculations and arguments for why the design is what it is. Would the system work if it were bigger? If the holes were bigger or smaller? If the tube were straighter or more curved? If the surfaces were rougher or smoother? If the Reynolds number were bigger or smaller? If the fluid were honey instead of water? Air instead of maple syrup?

Calculations: Reynolds number

Cylinder bore ≈ 60 mm $\rightarrow D_c = 0.06$ m

$D_c = 0.06$ m Cylinder stroke ≈ 470 mm $\rightarrow L_s = 0.47$ m

Hose diameter $D_h = 25$ mm $= 0.025$ m, length $L_h = 1.5$ m

$\rho \approx 1.2$ kg/m³, $\mu \approx 1.8 \times 10^{-5}$ Pas

Area $p = \pi r^2 = 2.83 \times 10^{-3}$

$V_p = \text{Area} \times L = 1.33L$

Assuming one pump per 3 seconds,

$Q = V_p \times f = 3.99 \text{ E}3 \text{ m}^3/\text{s}$

$Re = 2\rho U_r / \mu = 2 \times 1.2 \times 8.126 \times 0.025 / 1.8 \times 10^{-5} = 2.71 \times 10^4$

The air is turbulent as $Re > 4000$

Calculations: Pressure Drop in Hose.

Surface Roughness of Hose: $\varepsilon = 1.5 \cdot 10^{-6} \text{ m}$

Length of hose = 1.5 m

Hose diameter : $d = 25 \cdot 10^{-3} \text{ m}$

Relative Roughness $\varepsilon/d = .00006$

Viscous Drag Coefficient $f_D = 0.02485$

Because the mass flow rate in pump is constant, we treat U in the house as constant such that $\Delta p = \Delta B$.

$\Delta p = 177.8 \text{ Pa}$. Very small losses compared to operating pressure of the Paddle Board.

$$U_{\text{burrel}} A_{\text{burrel}} = U_{\text{hose}} A_{\text{hose}}$$

$$U_{\text{burrel}} \approx 1 \text{ m/s}$$

$$A_{\text{burrel}} = \pi (.03^2 + .035^2)$$

$$A_{\text{hose}} = \pi (.0125^2)$$

$$U_{\text{hose}} = A_{\text{burrel}} / A_{\text{hose}} = \frac{(.03^2 + .035^2)}{(.0125^2)} = 13.6 \text{ m/s}$$

$$Re = \frac{\rho U L}{\mu}$$

$$Re_1 = \frac{(1.29)(1)(.06)}{1.81 \cdot 10^{-5}} = 4276$$

$$Re_2 = \frac{(1.29)(1)(.07)}{1.81 \cdot 10^{-5}} = 4989$$

$$Re_{\text{hose}} = \frac{(1.29)(13.6)(.025)}{1.81 \cdot 10^{-5}} = 24232$$

$$\varepsilon_{\text{hose}} = 1.5 \text{ } \mu\text{m}$$

$$\text{Colebrook Equation } (Re > 2300)$$

$$1/\sqrt{f_D} = -2 \log(\varepsilon/3.7d + 2.51/(Re \sqrt{f_D}))$$

$$\text{Starting Guess } f_D = 0.0246 \text{ (using graph)}$$

$$\text{Final viscous drag coefficient}$$

$$f_D = 0.02485$$

Change in design

If diameter was greater

Handle force at Board pressure

$F = PA = P * \pi * r^2 \rightarrow$ dependent on r^2

$V = \text{Area} * L$

If the diameter was x2 doubled,
the stroke volume would be x4 greater and the
handle force would also be x4 greater. It would be
more difficult to push air into the paddle board for
users. But we could pump more air in.



Change in design

If the Re number was higher

This can be simulated by several factors: e.g. increasing the stroke rate, increasing surface roughness etc

As $Q = Vf$, if we pump faster will be a slight raise in hose pressure loss and the user would have it a bit easier to push down. However at low viscosity Re doesn't matter as much as inertia dominate flow over viscosity

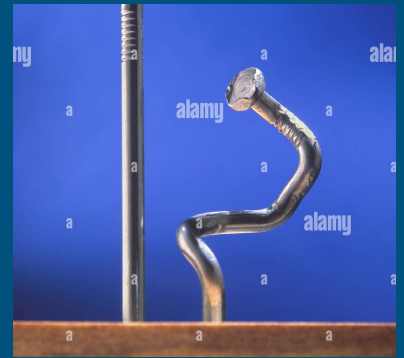
If the surface was smoother that would have decreased f_D and therefore pressure loss. However, due to the low air density, it would have not given a significant difference.



Change in design

The tube being straight vs curved vs have sharp bends

- Straight would be most ideal with least energy loss, but it would hinder its ease of use significantly as the position the hose can be at is limited
- Comparing a curved hose and a hose with sharp bending, with a curve there is less loss in energy, lower loss coefficient K compared to fluid experiencing extreme turns.
- $\Delta p_{\text{bend}} = K \cdot \rho U^2 / 2$ = if K goes from 0.2 \rightarrow 1.5, the pressure loss happens around 1pa \rightarrow 6pa, which is significantly less than board pressure



Change in design

If fluid were oil instead of water

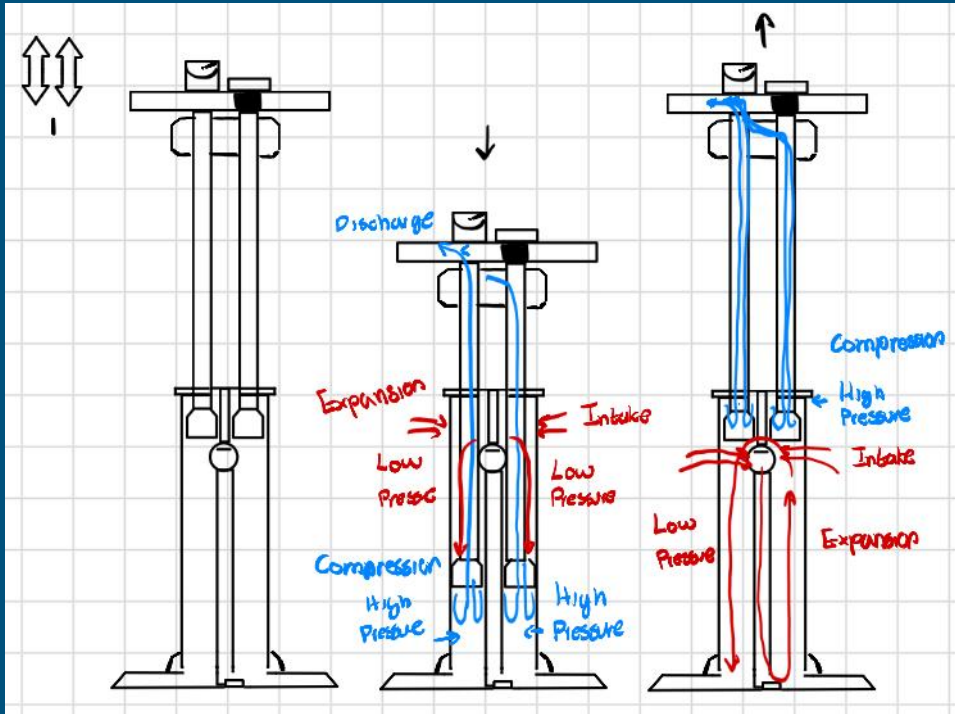
- Air has viscosity of $\mu = \sim 1.8 \times 10^{-5} \text{ Pa}\cdot\text{s}$
- Oil has viscosity near $\mu = \sim 0.3 \text{ Pa}\cdot\text{s}$

Re is related to viscosity as $Re \sim 1/\mu$

So honey decreases Re by factor of 10^4 it would be almost impossible to push. As force to push fluid is proportional to viscosity it would be 10^4 times harder to push

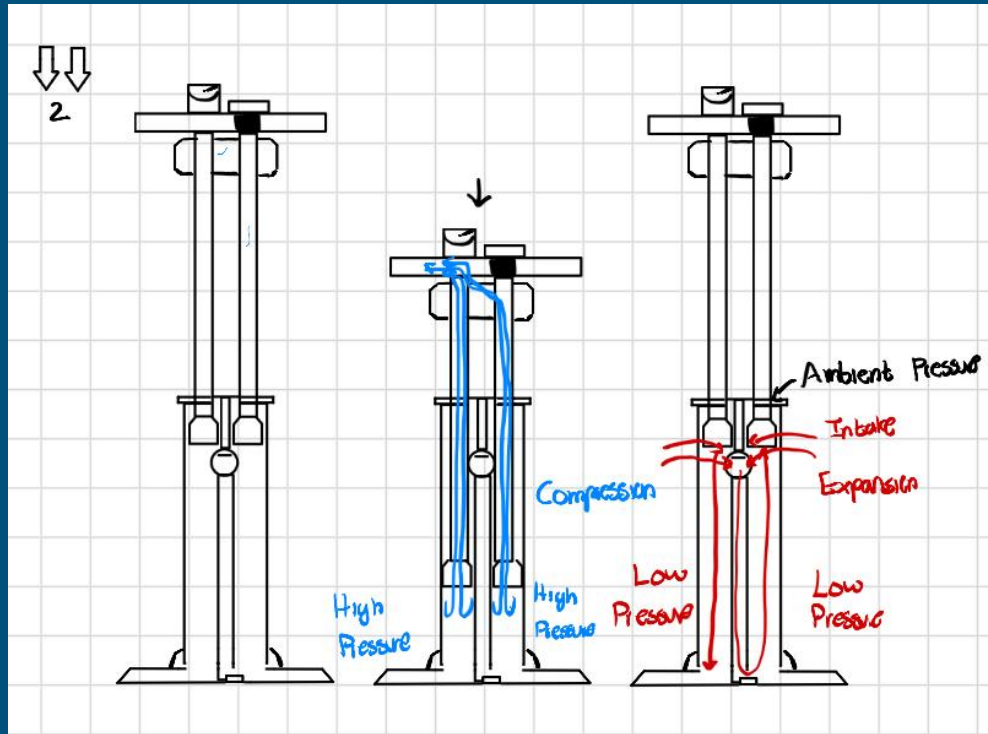


Mode 1: High Volume Low Pressure



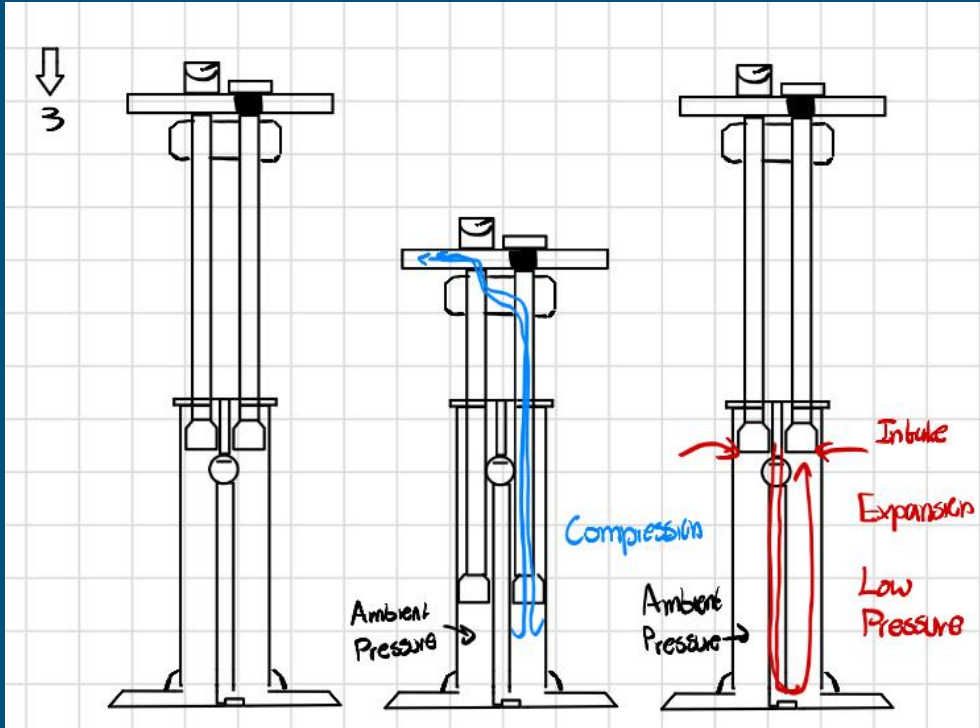
In this mode, each cycle displaces the volume of each barrel twice. Air flows onto the top and out the bottom of each piston on the down stroke and vice versa on the upstroke.

Mode 2: Medium Volume and Pressure



In this mode, air is not discharged during the up stroke. The dial exposes the top of the pistons to the environment so they are not pressurized. The up stroke intakes air only. For higher pressures this is desirable because it decreases the work required from the user on the upstroke.

Mode 3: Low Volume High Pressure



In this mode, only one barrel is utilized. The other barrel is at ambient pressure. It reduces both the volume displaced by the user on each pump but also the work required by the user. Up stroke is still only used for intake.

Thank you!
