

Spacecraft and Aircraft Dynamics

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Lecture 11: Longitudinal Dynamics

Aircraft Dynamics

Lecture 11

In this Lecture we will cover:

Longitudinal Dynamics:

- Finding dimensional coefficients from non-dimensional coefficients
- Eigenvalue Analysis
- Approximate modal behavior
 - ▶ short period mode
 - ▶ phugoid mode

Review: Longitudinal Dynamics

Combined Terms

$$\Delta \dot{u} + \Delta \theta g \cos \theta_0 = X_u \Delta u + X_w \Delta w + X_{\delta_e} \delta_e + X_{\delta_T} \delta_T$$

$$\Delta \dot{w} + \Delta \theta g \sin \theta_0 - u_0 \Delta \dot{\theta} = Z_u \Delta u + Z_w \Delta w + Z_{\dot{w}} \Delta \dot{w} + Z_q \Delta \dot{\theta} + Z_{\delta_e} \delta_e + Z_{\delta_T} \delta_T$$

$$\Delta \ddot{\theta} = M_u \Delta u + M_w \Delta w + M_{\dot{w}} \Delta \dot{w} + M_q \Delta \dot{\theta} + M_{\delta_e} \delta_e + M_{\delta_T} \delta_T$$

Force Coefficients

Force/Moment Coefficients can be found in Table 3.5 of Nelson

TABLE 3.5
Summary of longitudinal derivatives

$$X_u = \frac{-(C_{D_a} + 2C_{D_0})QS}{mu_0} (s^{-1})$$

$$X_w = \frac{-(C_{D_a} - C_{L_0})QS}{mu_0} (s^{-1})$$

$$Z_u = \frac{-(C_{L_u} + 2C_{L_0})QS}{mu_0} (s^{-1})$$

$$Z_w = \frac{-(C_{L_a} + C_{D_0})QS}{mu_0} (s^{-1})$$

$$Z_w = C_{z_a} \frac{c}{2u_0} QS / (u_0 m)$$

$$Z_\alpha = u_0 Z_w (\text{ft/s}^2) \text{ or } (\text{m/s}^2)$$

$$Z_{\dot{\alpha}} = u_0 Z_w (\text{ft/s}) \text{ or } (\text{m/s})$$

$$Z_q = C_{Z_q} \frac{c}{2u_0} QS/m (\text{ft/s}) \text{ or } (\text{m/s})$$

$$Z_{\delta_e} = C_{Z_{\delta_e}} QS/m (\text{ft/s}^2)$$

$$M_u = C_{m_u} \frac{(QSc)}{u_0 I_y} \left(\frac{1}{\text{ft} \cdot \text{s}} \right) \text{ or } \left(\frac{1}{\text{m} \cdot \text{s}} \right)$$

$$M_w = C_{m_a} \frac{(QSc)}{u_0 I_y} \left(\frac{1}{\text{ft} \cdot \text{s}} \right) \text{ or } \left(\frac{1}{\text{m} \cdot \text{s}} \right) \quad M_w = C_{m_{\dot{\alpha}}} \frac{\bar{c}}{2u_0} \frac{QS\bar{c}}{u_0 I_y} (\text{ft}^{-1})$$

$$M_\alpha = u_0 M_w (\text{s}^{-2})$$

$$M_{\dot{\alpha}} = u_0 M_w (\text{s}^{-1})$$

$$M_q = C_{m_q} \frac{\bar{c}}{2u_0} (QSc)/I_y (\text{s}^{-1})$$

$$M_{\delta_e} = C_{m_{\delta_e}} (QSc)/I_y (\text{s}^{-2})$$

Nondimensional Force Coefficients

Nondimensional Force/Moment Coefficients can be found in Table 3.3 of Nelson

TABLE 3.3

Equations for estimating the longitudinal stability coefficients

	X-force derivatives	Z-force derivatives	Pitching moment derivatives
u	$C_{X_u} = -[C_{D_0} + 2C_{D_0}] + C_{T_u}$	$C_{Z_u} = -\frac{M^2}{1-M^2}C_{L_0} - 2C_{L_0}$	$C_{m_u} = \frac{\partial C_m}{\partial M}M_0$
α	$C_{X_\alpha} = C_{L_0} - \frac{2C_{L_0}}{\pi e} \frac{C_{L_0}}{AR}$	$C_{Z_\alpha} = -(C_{L_\alpha} + C_{D_0})$	$C_{m_\alpha} = C_{L_{\alpha w}} \left(\frac{X_{cg}}{c} - \frac{X_{ac}}{c} \right) + C_{m_{\alpha fus}} - \eta V_H C_{L_{\alpha t}} \left(1 - \frac{d\varepsilon}{d\alpha} \right)$
$\dot{\alpha}$	0	$C_{Z_{\dot{\alpha}}} = -2\eta C_{L_{\alpha t}} V_H \frac{d\varepsilon}{d\alpha}$	$C_{m_{\dot{\alpha}}} = -2\eta C_{L_{\alpha t}} V_H \frac{l_t}{c} \frac{d\varepsilon}{d\alpha}$
q	0	$C_{Z_q} = -2\eta C_{L_{\alpha t}} V_H$	$C_{m_q} = -2\eta C_{L_{\alpha t}} V_H \frac{l_t}{c}$
α_e	0	$C_{Z_{\delta_e}} = -C_{L_{\delta_e}} = -\frac{S_t}{S} \eta \frac{dC_{L_t}}{d\delta_e}$	$C_{m_{\delta_e}} = -\eta V_H \frac{dC_{L_t}}{d\delta_e}$

Subscript 0 indicates reference values and M is the Mach number.

AR	Aspect ratio	V_H	Horizontal tail volume ratio
C_{D_0}	Reference drag coefficient	M	Flight mach number
C_{L_0}	Reference lift coefficient	S	Wing area
C_{L_α}	Airplane lift curve slope	S_t	Horizontal tail area
$C_{L_{\alpha w}}$	Wing lift curve slope	$\frac{d\varepsilon}{d\alpha}$	Change in downwash due to a change in angle of attack
$C_{L_{\alpha t}}$	Tail lift curve slope	η	Efficiency factor of the horizontal tail
\bar{c}	Mean aerodynamic chord		
e	Oswald's span efficiency factor		
l_t	Distance from center of gravity to tail quarter chord		

State-Space

From the homework, we have a state-space representation of form

$$\begin{bmatrix} \Delta \dot{u} \\ \Delta \dot{w} \\ \Delta \dot{\theta} \\ \Delta \dot{q} \end{bmatrix} = [A] \begin{bmatrix} u \\ w \\ \theta \\ q \end{bmatrix} + [B] \begin{bmatrix} \delta_e \\ \delta_T \end{bmatrix}$$

Where we get A and B from X_u , Z_u , etc.

Recall:

- Eigenvalues of A define stability of $\dot{x} = Ax$.
- A is 4×4 , so A has 4 eigenvalues.
- Stable if eigenvalues all have negative real part.

Natural Motion

We mentioned that A is

- *Stable* if eigenvalues all have negative real part.

Now we say more: Eigenvalues have the form

$$\lambda = \lambda_R \pm \lambda_I i$$

If we have a **pair** of complex eigenvalues, then we have two more concepts:

1. Natural Frequency:

$$\omega_n = \sqrt{\lambda_R^2 + \lambda_I^2}$$

2. Damping Ratio:

$$d = -\frac{\lambda_R}{\omega_n}$$

Natural Frequency

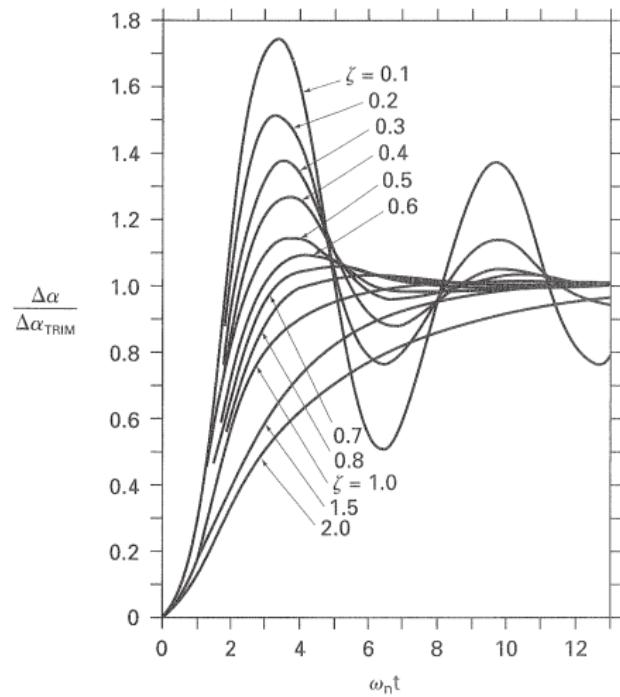
Natural frequency is how fast the motion oscillates.

Closely related is the

Definition 1.

The **Period** is the time take to complete one oscillation

$$\tau = \frac{2\pi}{\omega_n}$$



Damping Ratio

Damping ratio is how much amplitude decays per oscillation.

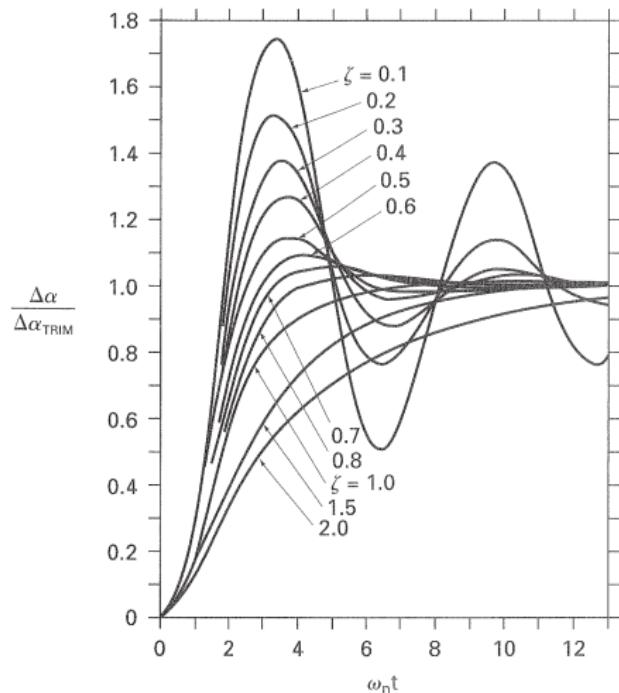
- Even if d is large, may decay slowly if ω_n is small

Closely related is

Definition 2.

The **Half-Life** is the time taken for the amplitude to decay by half.

$$\gamma = \frac{.693}{|\lambda_R|}$$



State-Space

Example: Uncontrolled Motion

C172: $V_0 = 132kt$, 5000ft.

$$\begin{bmatrix} \Delta\dot{u} \\ \Delta\dot{w} \\ \Delta\dot{q} \\ \Delta\dot{\theta} \end{bmatrix} = \begin{bmatrix} -.0442 & 18.7 & 0 & -32.2 \\ -.0013 & -2.18 & .97 & 0 \\ .0024 & -23.8 & -6.08 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} u \\ w \\ q \\ \theta \end{bmatrix}$$

State-Space

Example: Uncontrolled Motion

Using the Matlab command $[u, V] = \text{eigs}(A)$, we find the eigenvalues as
Phugoid (Long-Period) Mode

$$\lambda_{1,2} = -0.0209 \pm .18i$$

and Eigenvectors

$$v_{1,2} = \begin{bmatrix} -.1717 \\ -.0748 \\ .9131 \\ -.1038 \end{bmatrix} \pm \begin{bmatrix} .2826 \\ .1685 \\ 0 \\ .1103 \end{bmatrix} i$$

Short-Period Mode

$$\lambda_{3,4} = -4.13 \pm 4.39i$$

and Eigenvectors

$$v_{3,4} = \begin{bmatrix} 1 \\ -.0002 \\ .001 \\ -.0008 \end{bmatrix} \pm \begin{bmatrix} 0 \\ .0000001 \\ .0000011 \\ .0055 \end{bmatrix} i$$

Notice that this is hard to interpret. Lets scale u and q by equilibrium values.

State-Space

Example: Uncontrolled Motion

After scaling the state by the equilibrium values, we find the eigenvalues unchanged (Why?) as

Phugoid (Long-Period) Mode

$$\lambda_{1,2} = -.0209 \pm .18i$$

but clearer Eigenvectors

$$v_{1,2} = \begin{bmatrix} -.629 \\ .0218 \\ -.0016 \\ .138 \end{bmatrix} \pm \begin{bmatrix} .0213 \\ .0007 \\ .0001 \\ .765 \end{bmatrix} i$$

Natural Frequency: $\omega_n = .181 rad/s$

Damping Ratio: $d = .115$

Period: $\tau = 34.7s$

Half-Life: $\gamma = 33.16s$

Motion dominated by variables u and θ .

Modal Illustration

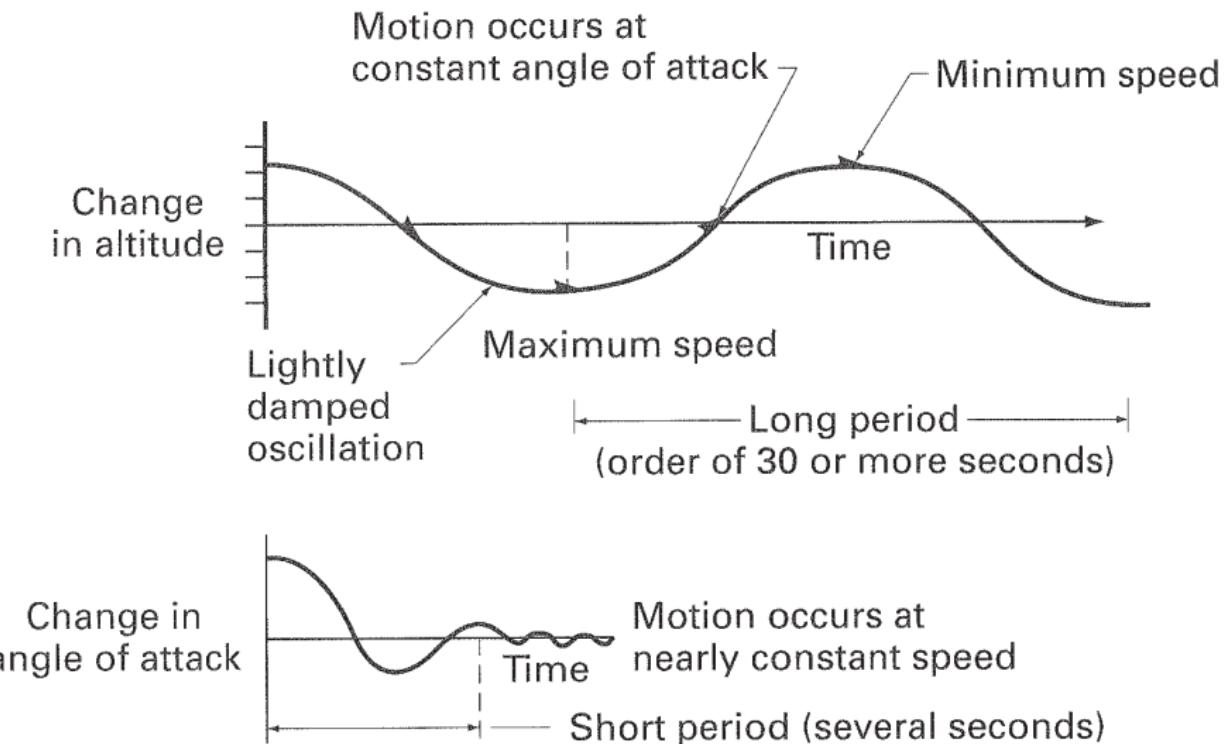


FIGURE 4.10

State-Space

Example: Long Period Mode

State-Space

Example: Uncontrolled Motion

Using the Matlab command $[u, V] = \text{eigs}(A)$, we find the eigenvalues as
Short-Period Mode

$$\lambda_{3,4} = -4.13 \pm 4.39i$$

and Eigenvectors

$$v_{3,4} = \begin{bmatrix} -.0049 \\ -.655 \\ -.396 \\ -.006 \end{bmatrix} \pm \begin{bmatrix} .004 \\ .409 \\ .495 \\ .0423 \end{bmatrix} i$$

Natural Frequency: $\omega_n = 6.03\text{rad/s}$

Damping Ratio: $d = .685$

Period: $\tau = 1.04\text{s}$

Half-Life: $\gamma = .167\text{s}$

Motion dominated by variables w and q .

Modal Illustration

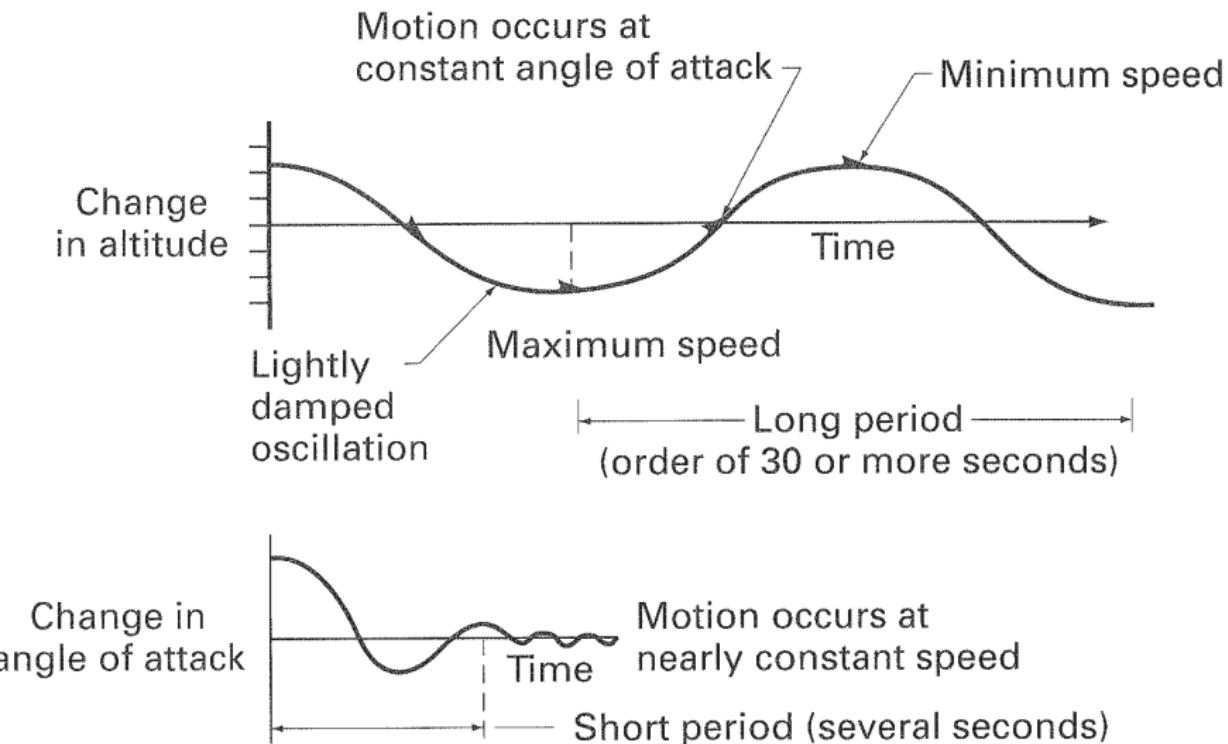


FIGURE 4.10

State-Space

Example: Short Period Mode

State-Space

Modal Approximations

Now that we know that longitudinal dynamics have two modes:

- Short Period Mode
- Phugoid Mode (Long-Period Mode)

Short Period Mode:

- fix $u = 0$ and $w = 0$.
- study variation in θ and q .
- Similar to Static Longitudinal Stability

Long Period Mode:

- fix $q = 0$ and $w = 0$.
- study variation in θ and u .

Now we develop some simplified expressions to study these modes.

Short Period Approximation

For the short period mode, we have the following dynamics:

$$\begin{bmatrix} \frac{\dot{w}}{u_0} \\ \dot{q} \end{bmatrix} = \begin{bmatrix} \frac{Z_\alpha}{u_0} & 1 \\ M_\alpha + \frac{M_{\dot{\alpha}} Z_\alpha}{u_0} & M_q + M_{\dot{\alpha}} \end{bmatrix} \begin{bmatrix} \frac{w}{u_0} \\ q \end{bmatrix} + \begin{bmatrix} \frac{Z_{\delta_e}}{u_0} \\ M_{\delta_e} + \frac{Z_{\delta_e}}{u_0} \end{bmatrix} \delta_e \\ = A_{sp} \begin{bmatrix} \theta \\ q \end{bmatrix} + B_{sp} \delta_e \end{math>$$

- To understand stability, we need the eigenvalues of A_{sp} .
- Eigenvalues are solutions of $\det(\lambda I - A_{sp}) = 0$.

Thus we want to solve

$$\det(\lambda I - A_{sp}) = \lambda^2 - (M_q + \frac{Z_\alpha}{u_0} + M_{\dot{\alpha}})\lambda + (\frac{Z_\alpha M_q}{u_0} - M_\alpha) = 0$$

We use the quadratic formula (Lecture 1):

$$\lambda_{3,4} = \frac{1}{2}(M_q + M_{\dot{\alpha}} + \frac{Z_\alpha}{u_0}) \pm \frac{1}{2}\sqrt{(M_q + M_{\dot{\alpha}} + \frac{Z_\alpha}{u_0})^2 - 4(m_q \frac{Z_\alpha}{u_0} - M_\alpha)}$$

Short Period Approximation

Frequency and Damping Ratio

$$\lambda_{3,4} = \frac{1}{2}(M_q + M_{\dot{\alpha}} + \frac{Z_{\alpha}}{u_0}) \pm \frac{1}{2}\sqrt{(M_q + M_{\dot{\alpha}} + \frac{Z_{\alpha}}{u_0})^2 - 4(m_q \frac{Z_{\alpha}}{u_0} - M_{\alpha})}$$

This leads to the *Approximation Equations*:

- **Natural Frequency:**

$$\omega_{sp} = M_q \frac{Z_{\alpha}}{u_0}$$

- **Damping Ratio:**

$$d_{sp} = -\frac{1}{2} \frac{M_q + M_{\dot{\alpha}} + \frac{Z_{\alpha}}{u_0}}{\omega_{sp}}$$

Short Period Mode Approximations

Example: C127

Approximate Natural Frequency:

$$\omega_{sp} = \sqrt{\frac{-481 * -431}{219} + 27.7} = 6.10 \text{ rad/s}$$

True Natural Frequency:

$$\omega_{sp} = 6.03 \text{ rad/s}$$

Approximate Damping Ratio:

$$d_{sp} = -\frac{4.32 - 2.20 - 1.81}{2 * 6.10} = .683 \text{ rad/s}$$

True Damping Ratio:

$$d_{sp} = .685 \text{ rad/s}$$

So, generally good agreement.

Long Period Approximation

Long period motion considers only motion in u and θ .

$$\begin{bmatrix} \dot{u} \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} X_u & -g \\ -\frac{Z_u}{u_0} & 0 \end{bmatrix} \begin{bmatrix} u \\ \theta \end{bmatrix}$$

This time, we must solve the simple expression

$$\det(\lambda I - A) = \lambda^2 - X_u \lambda - \frac{Z_u}{u_0} g = 0$$

Using the quadratic formula, we get

$$\lambda_{1,2} = \frac{X_u \pm \sqrt{X_u^2 + 4 \frac{Z_u}{u_0} g}}{2}$$

Long Period Approximation

$$\lambda_{1,2} = \frac{X_u \pm \sqrt{X_u^2 + 4\frac{Z_u}{u_0}g}}{2}$$

This leads to the *Approximation Equations*:

- **Natural Frequency:**

$$\omega_{lp} = \sqrt{-\frac{Z_u g}{u_0}}$$

- **Damping Ratio:**

$$d_{lp} = -\frac{X_u}{2\omega_{lp}}$$

Long Period Mode Approximations

Example: C127

Approximate Natural Frequency:

$$\omega_{lp} = .181 \text{ rad/s}$$

True Natural Frequency:

$$\omega_{lp} = .208 \text{ rad/s}$$

Approximate Damping Ratio:

$$d_{lp} = .115 \text{ rad/s}$$

True Damping Ratio:

$$d_{lp} = .106 \text{ rad/s}$$

So, generally good, but not as good. Why?

Conclusion

In this lecture, we covered:

- How to find and interpret the eigenvalues and eigenvectors of a state-space matrix
 - ▶ Natural Frequency
 - ▶ Damping Ratio
- How to identify
 - ▶ Long Period Eigenvalues/Motion
 - ▶ Short Period Eigenvalues/Motion
- Modal Approximations
 - ▶ Phugoid and Short-Period Modes
 - ▶ Formulas for natural frequency
 - ▶ Formulae for damping ratio