

Ducted hydrokinetic turbine design optimization using DAFoam

07. 25. 2024

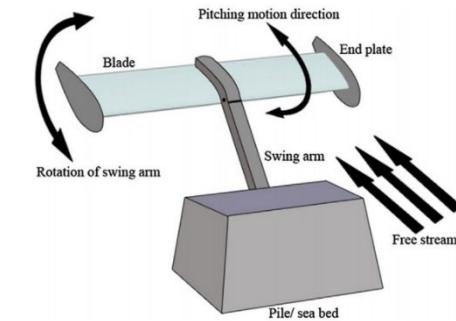
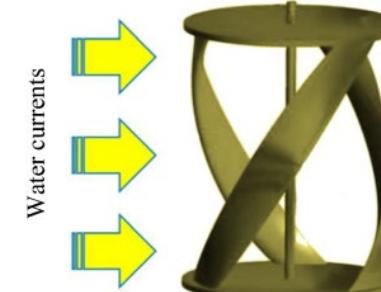
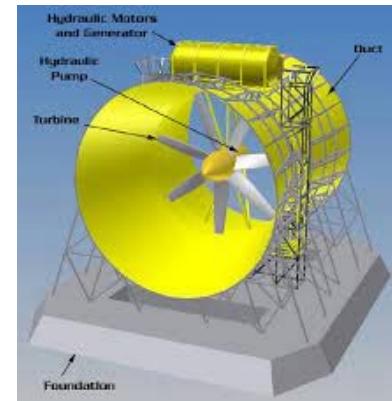


Outline

- Problem statement
- Work 1: Thin-walled ducted turbine optimization
 - Free-Form Deformation (FFD)
- Work 2: Foil-shaped ducted turbine optimization
 - Engineering Sketch Pad (ESP)

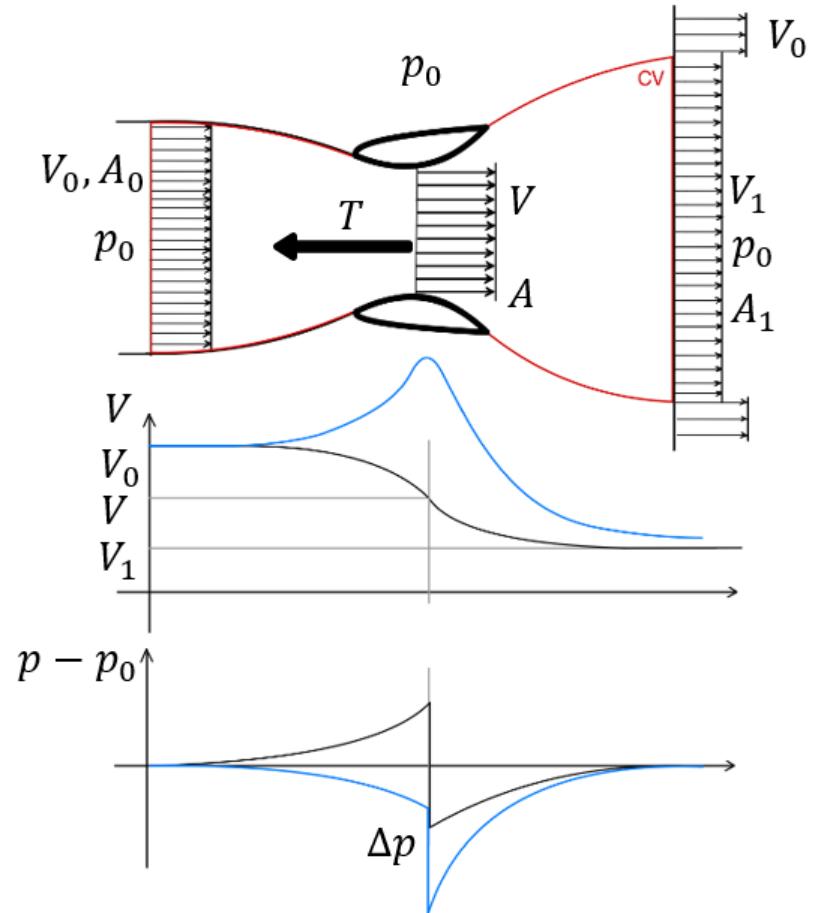
Hydrokinetic turbine

- Device that extracts energy from natural water flows
- Various types
 - **Horizontal-axis**
 - Popular benchmark: Bahaj turbine (efficiency: 46%)
 - Vertical-axis (cross-flow)
 - Oscillating foil



Improving efficiency of turbines

- Betz limit from 1D momentum theory
 - mass, momentum conservation + Bernoulli
 - Efficiency limited by 59.3%
- Introducing ‘Duct’
 - Accelerate & condition fluid flow
 - Improve overall energy extraction efficiency



Motivation and goal

- No substantial evidence provided for the effects of ducts
 - Difficult analytical explanation
 - Experimental/computational approaches are necessary
 - Lack of systematic optimization with high-fidelity approach
-
- Find an efficient ducted turbine design
Corroborate the benefit of using a duct
- Design optimization using DAFoam

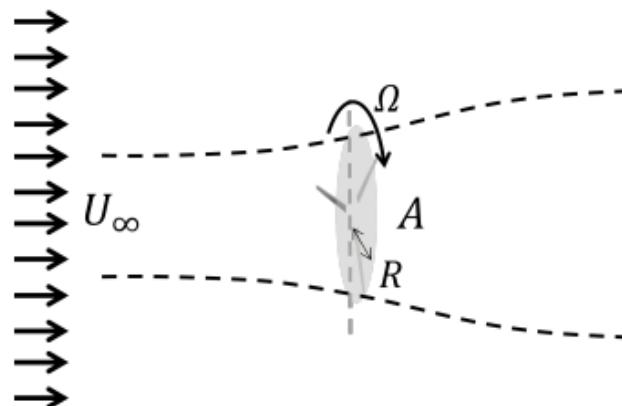
Physical problem

- **Find efficient horizontal-axis ducted turbine design**

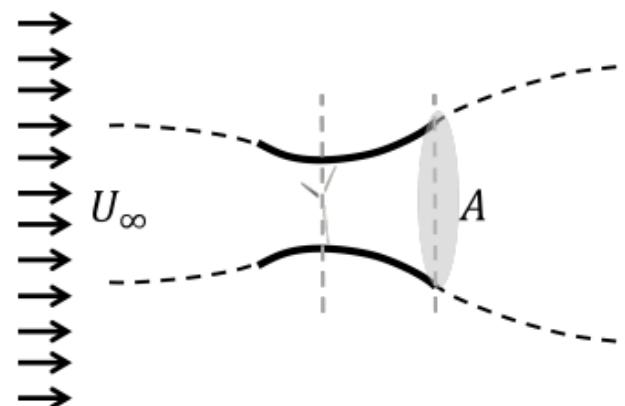
- Given U_∞ and Ω

- measuring Power coefficient (efficiency): $C_P = \frac{\text{Extracted power}}{\text{Available power in flow}} = \frac{Q\Omega}{\frac{1}{2}\rho U_\infty^3 A}$
- which is a function of Tip Speed Ratio: $\lambda = \frac{\Omega R}{U_\infty}$

- Measure C_P with maximum frontal area of a whole device



(a) Unducted turbine



(b) Ducted turbine

Optimization problem

maximize C_P

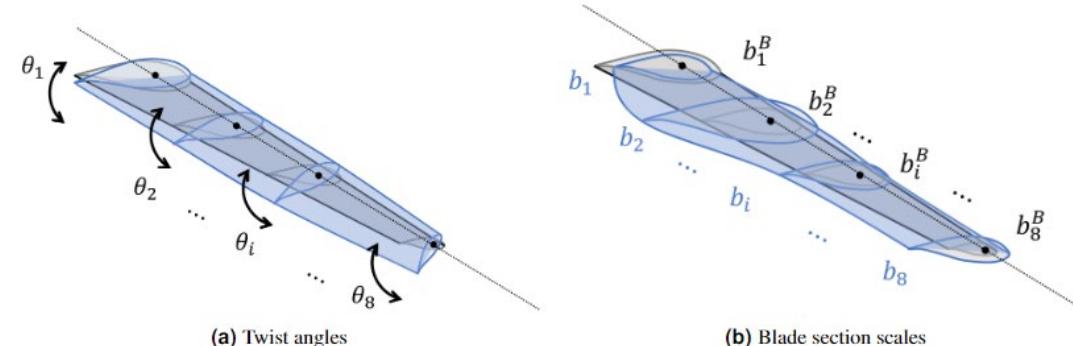
by varying $-30^\circ \leq \{\theta_i\}_{i=1}^8 \leq 30^\circ$, **Blade twists**

$$0.8 \leq \left\{ \frac{b_i}{b_i^B} \right\}_{i=1}^8 \leq 1.2, \quad \text{Blade section scales}$$

$$0 \leq d_3 \leq \{d_j\}_{j=1,2,4} \leq D_{\text{exit}}, \quad \text{Duct/blade radial scales}$$

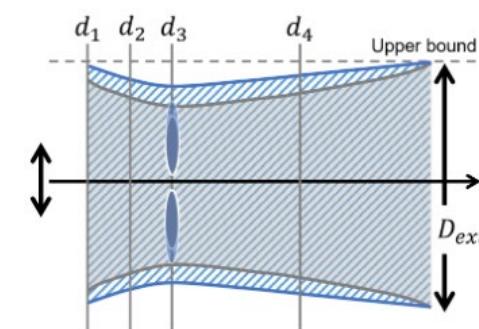
$$0.3 \leq \frac{l}{l^B} \leq 1.5, \quad \text{Duct axial scale}$$

subject to $\frac{2R}{d_3} = 0.91$, **Tip gap ratio**

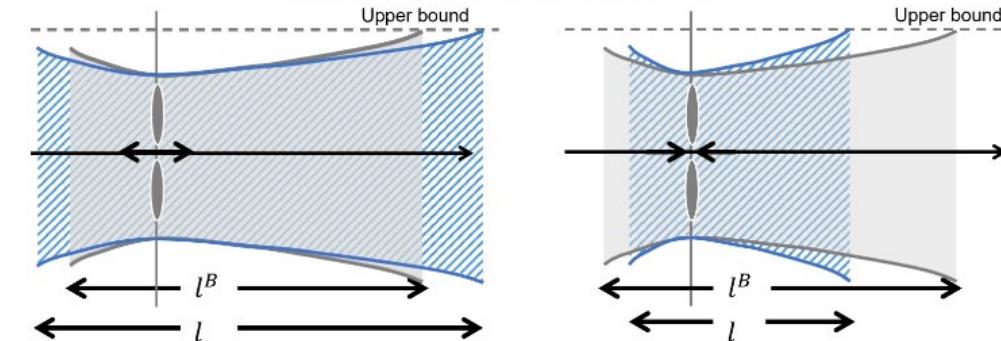
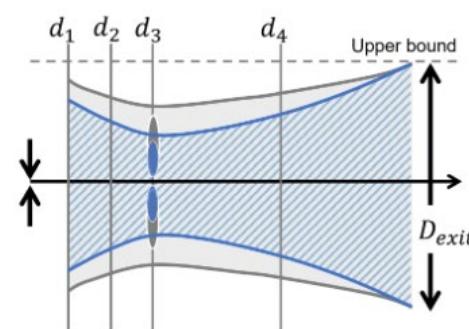


(a) Twist angles

(b) Blade section scales



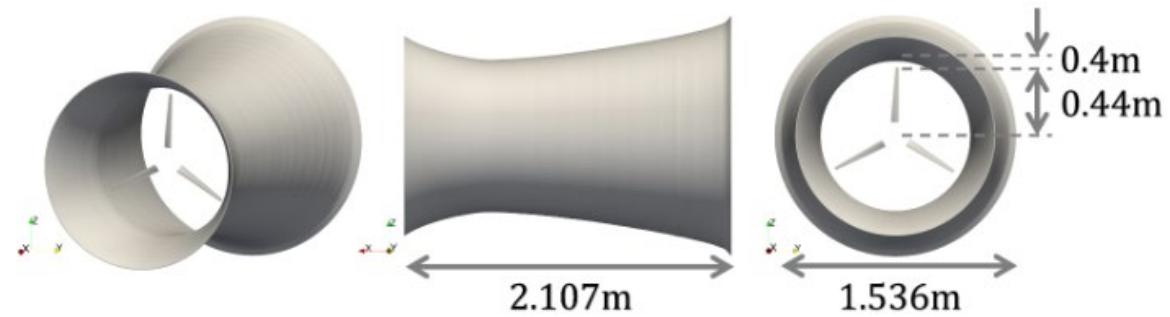
(a) Duct sectional radii change. Expansion and contraction



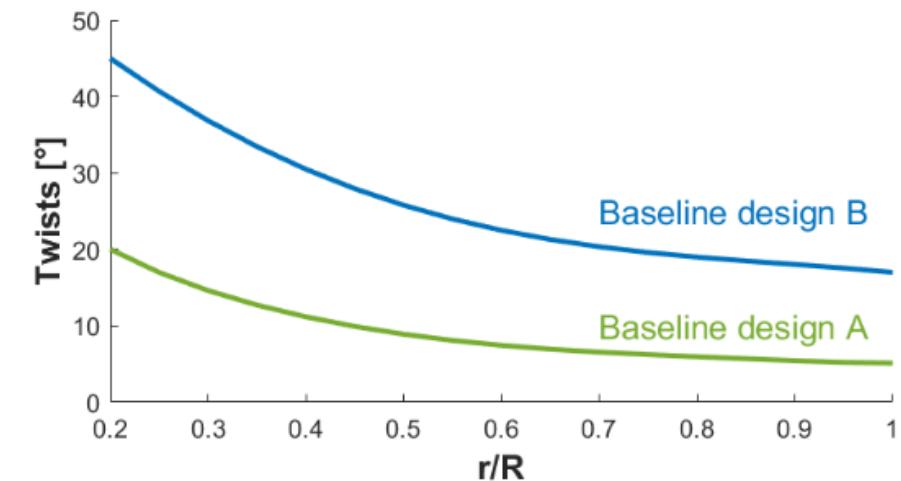
(b) Duct length change. Elongation and shortening

Thin-walled ducted turbine baseline design

- Baseline C_P
 - Design A: $C_P \sim 28\%$
 - Design B: $C_P \sim 45\%$
- Blades: Bahaj et al. [2007] turbine
 - Original twist profile (Design A)
 - Modified twist profile (Design B)
 - No hub included
- Duct: Knight et al. [2018]
 - Thin wall



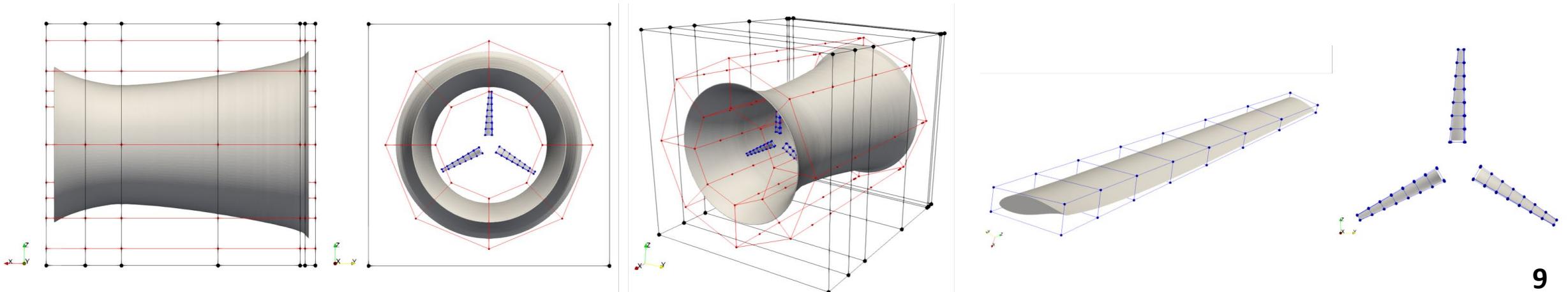
(a) Different views of baseline ducted turbine



(b) Twist distributions of baseline designs A and B

Ducted turbine FFD setup

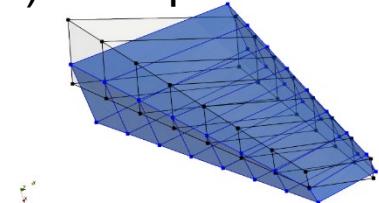
- 2 Layers of FFD boxes:
 - Parent box
 - FFD points control – Radial scale (black)
 - Blade and duct throat radii scales together, keeping the same tip gap ratio
 - Last 3 sections densely located at exit to fix the exit radius
 - Children box
 - FFD points control - Duct length (red)
 - FFD points control - Blade geometry (blue)



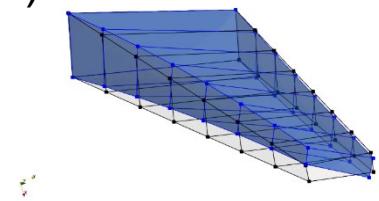
Design variable change through FFD

- Blade

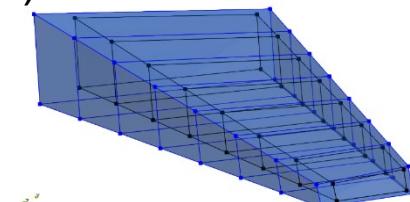
(a) Root pitch



(b) Twists

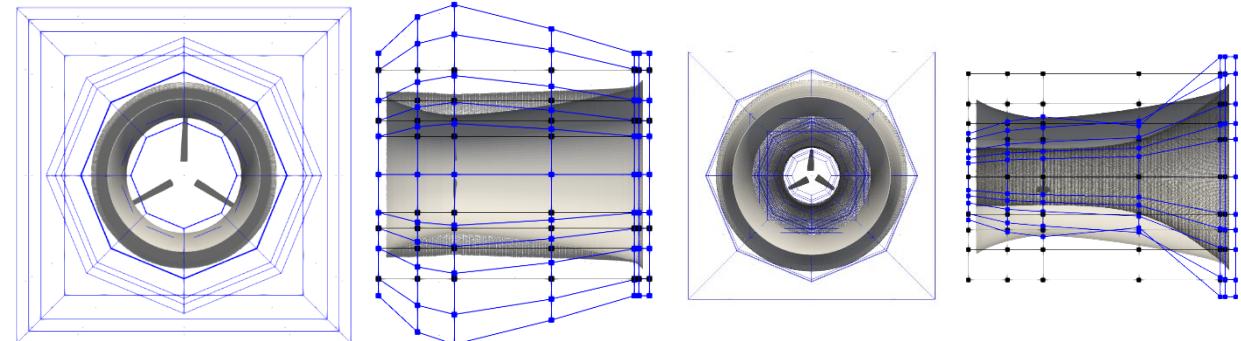


(c) Blade section scales

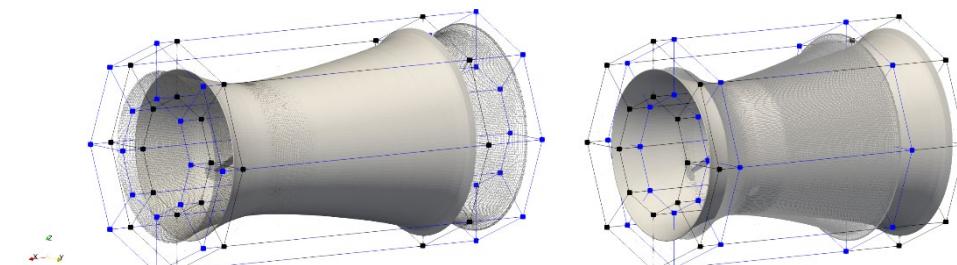


- Duct

(a) Duct radial scales. Expansion and contraction



(b) Duct length scale. Elongation and shortening

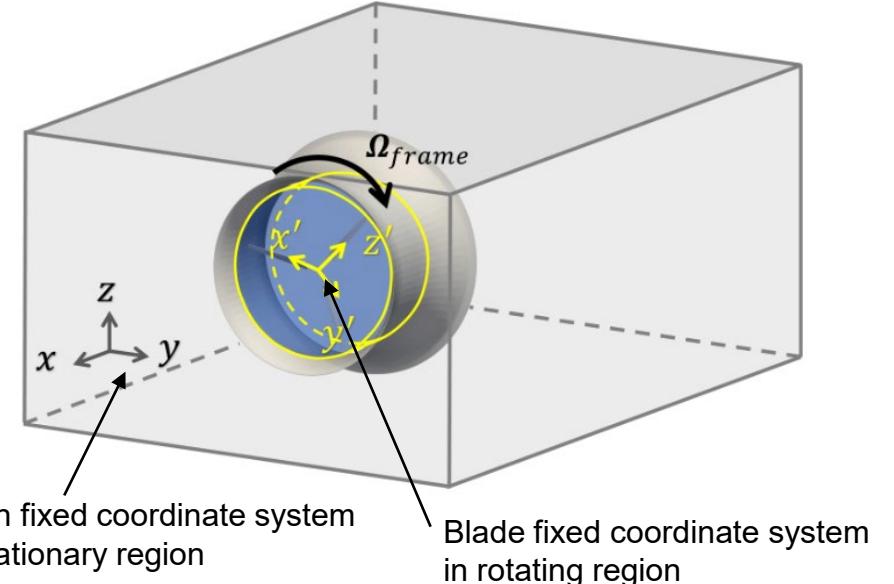


CFD simulation : Turbine rotation modeling

- Steady RANS + Multiple Reference Frame method (MRF)

- Multiple coordinate systems
 - Efficient, fair accuracy
 - Low-fidelity approach

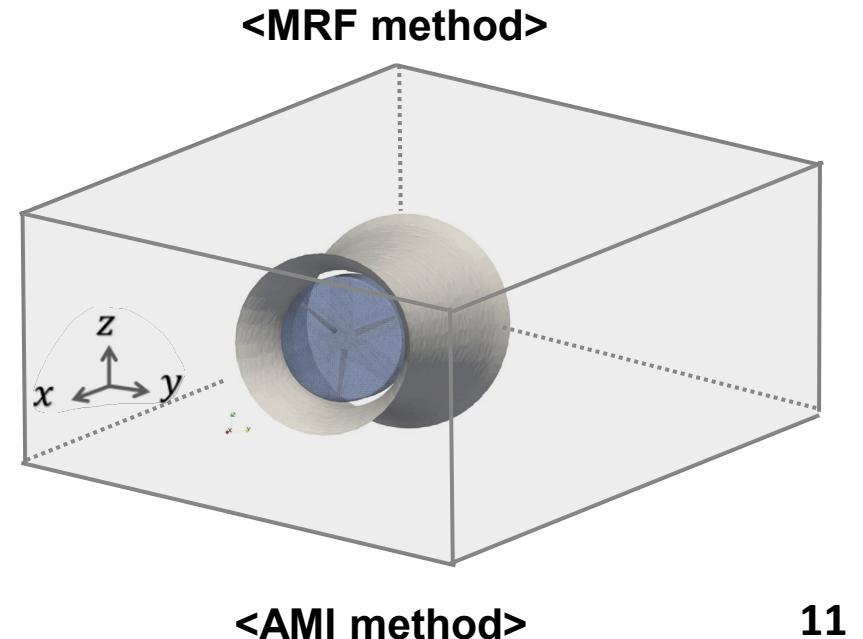
→ Optimization



- Unsteady RANS + Arbitrary Mesh Interface (AMI)

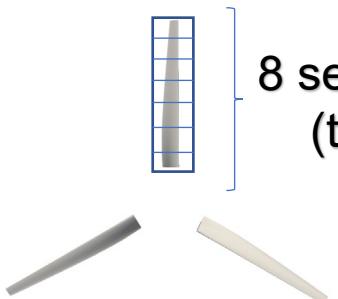
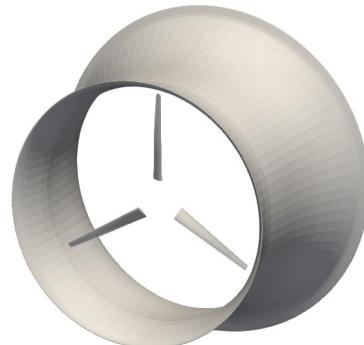
- Rotating-sliding mesh
 - Computational mesh rotation at each time step
 - Relatively accurate, expensive
 - High-fidelity approach

→ Re-evaluation

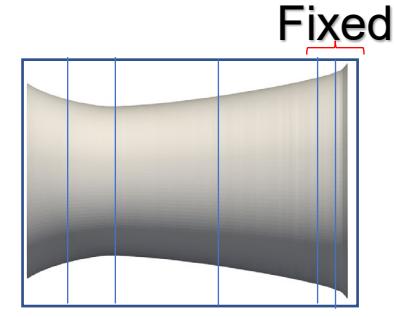


Ducted turbine optimization setup

		OPT A (from 28%)	OPT B (from 45%)
Baseline Design	Blade	Original-twist Bahaj	Twist-modified Bahaj
	Duct	Thin-walled duct from Knight et al. [2018]	
Objective		Maximize C_P @ fixed U_∞ & Ω	
Design variables	Blade	Root pitch/ twists/ chords (16 vars)	
	Duct	Length / radii (5 vars)	
Constraints		Fixed blade-duct gap ratio	



8 sections spanwise
(twists, chords)

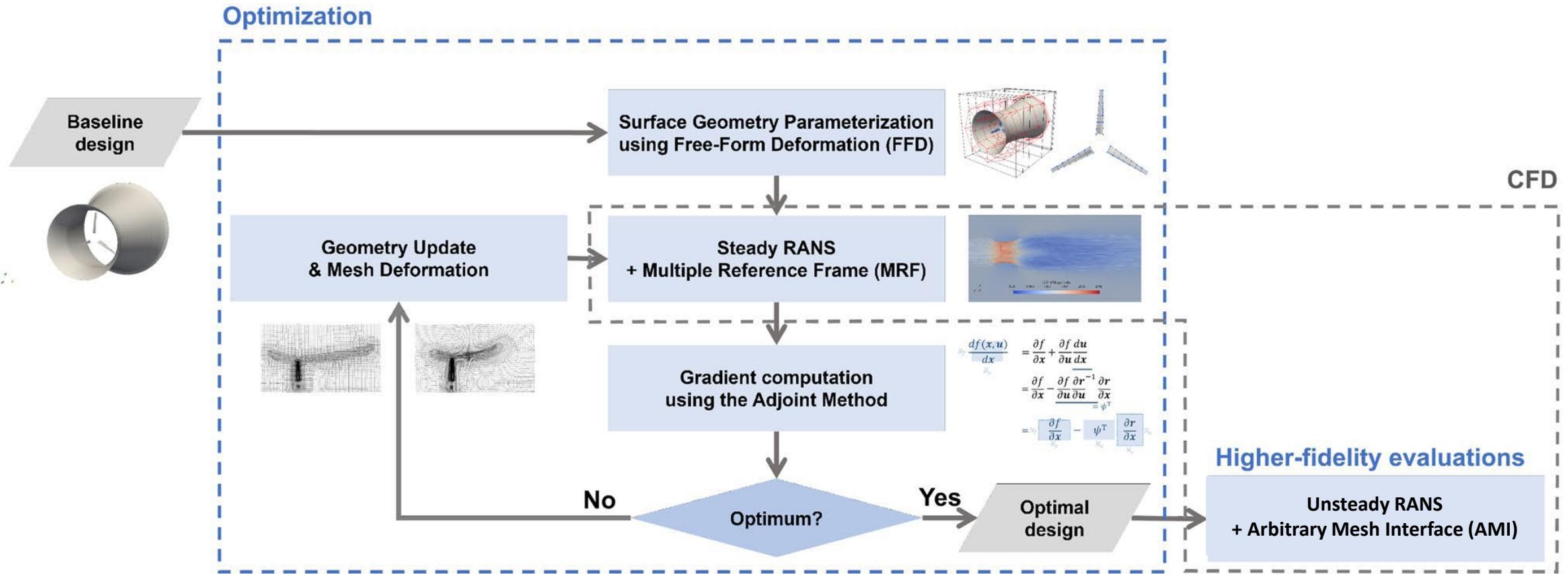


7 sections (radii)

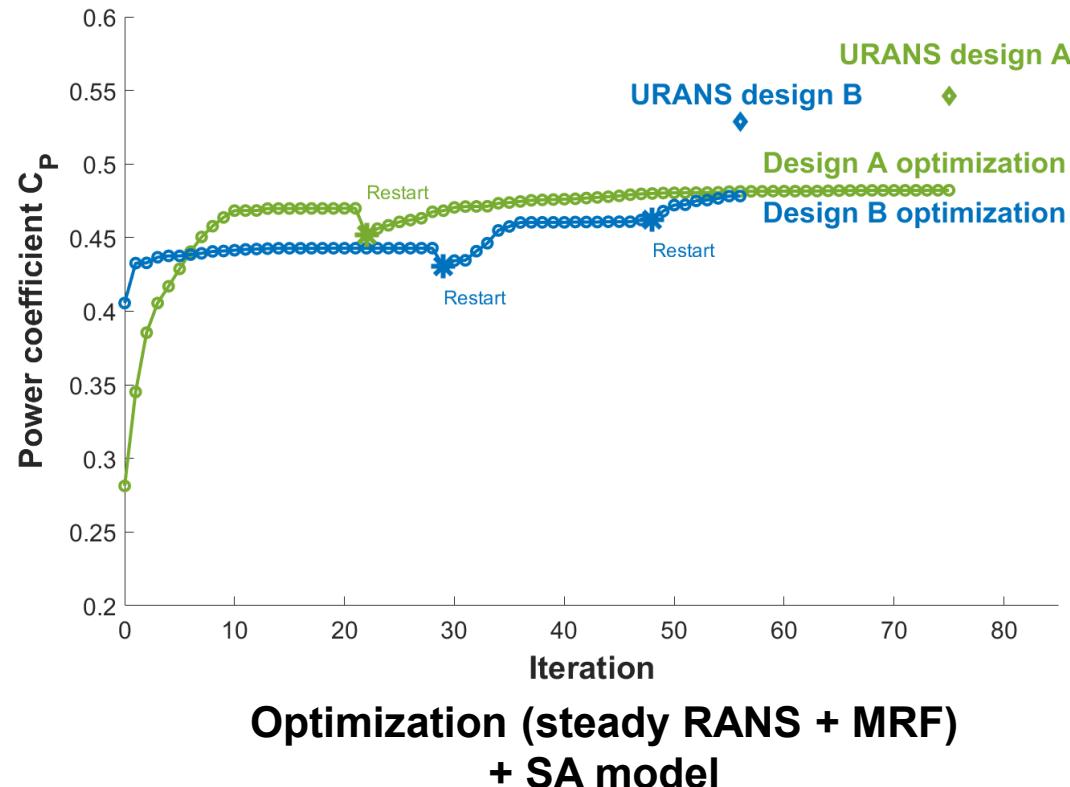
A.S. Bahaj, A.F. Molland, J.R. Chaplin, W.M.J. Batten, Power and thrust measurements of marine current turbines under various hydrodynamic flow conditions in a cavitation tunnel and a towing tank, Renewable Energy, Volume 32, Issue 3, 2007, Pages 407-426, ISSN 0960-1481

Knight B, Freda R, Young YL, Maki K. Coupling Numerical Methods and Analytical Models for Ducted Turbines to Evaluate Designs. *Journal of Marine Science and Engineering*. 2018; 6(2):43. <https://doi.org/10.3390/jmse6020043>

Optimization process



Optimization results: C_P



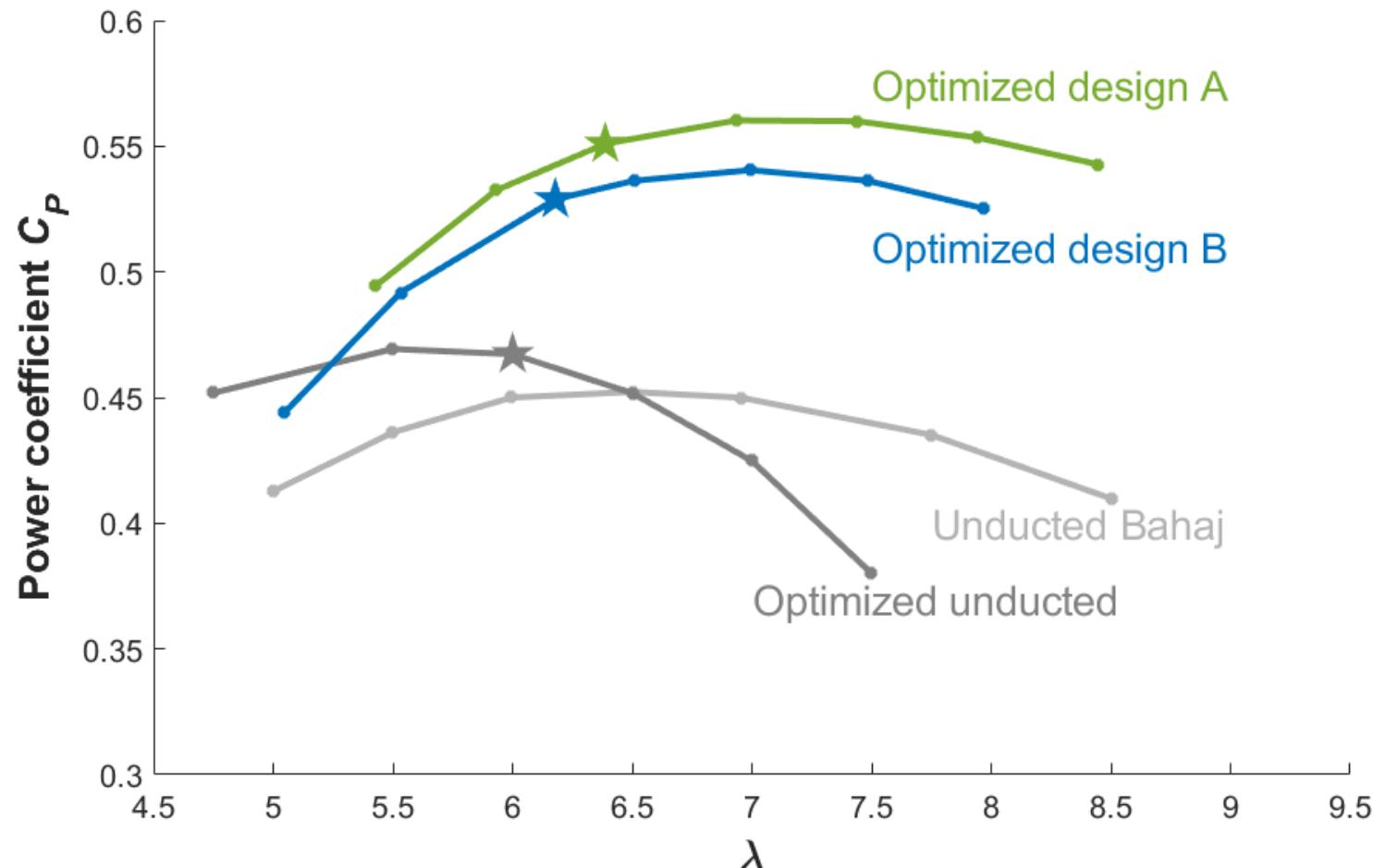
Design A				
	Num of cells	y^+ Blade	y^+ Duct	C_P
M0	2,741,276	47.75	200.2	0.5462
M1	4,697,325	40.82	191.1	0.5511
M2	7,280,862	33.30	147.7	0.5381

Design B				
	Num of cells	y^+ Blade	y^+ Duct	C_P
M0	3,066,956	49.32	221.1	0.5287
M1	5,358,847	40.81	173.5	0.5335
M2	8,453,165	34.33	155.4	0.5337

- Optimizations started from different starting points converge to very similar C_P
- Re-evaluation
 - Optimization A: 0.2759 ($\lambda = 5.5$) \rightarrow 0.5381 ($\lambda = 6.39$)
 - Optimization B: 0.4508 ($\lambda = 5.5$) \rightarrow 0.5337 ($\lambda = 6.18$)

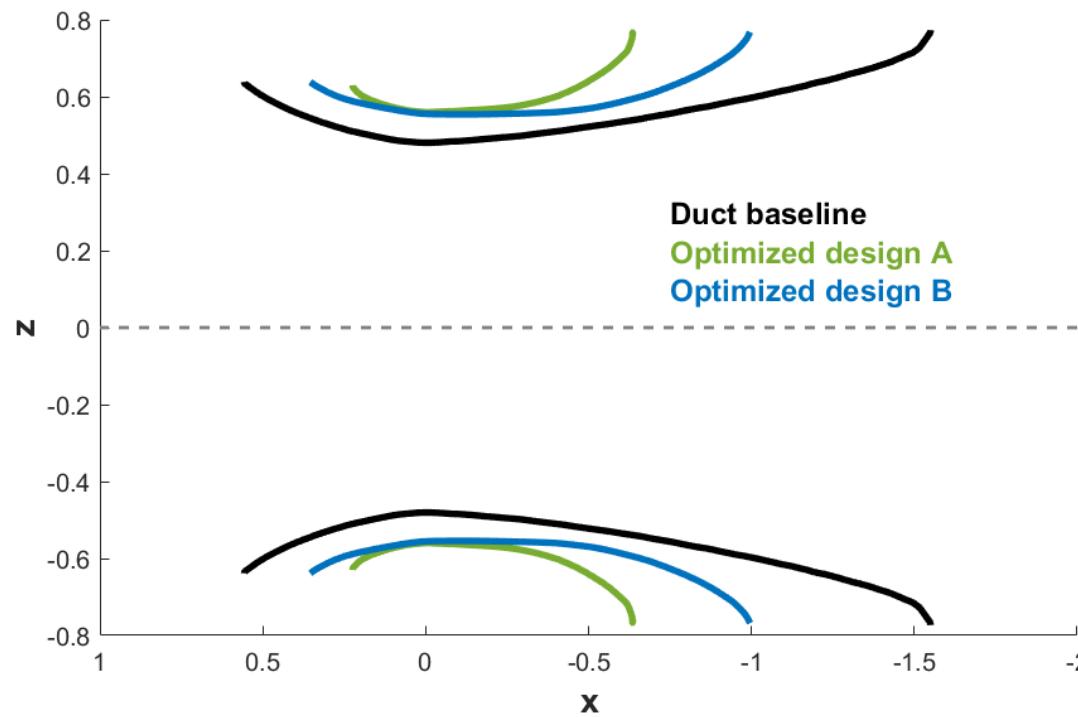
Optimization results : comparison with unducted turbines

- Ducted turbines can outperform unducted turbines for a range of TSRs

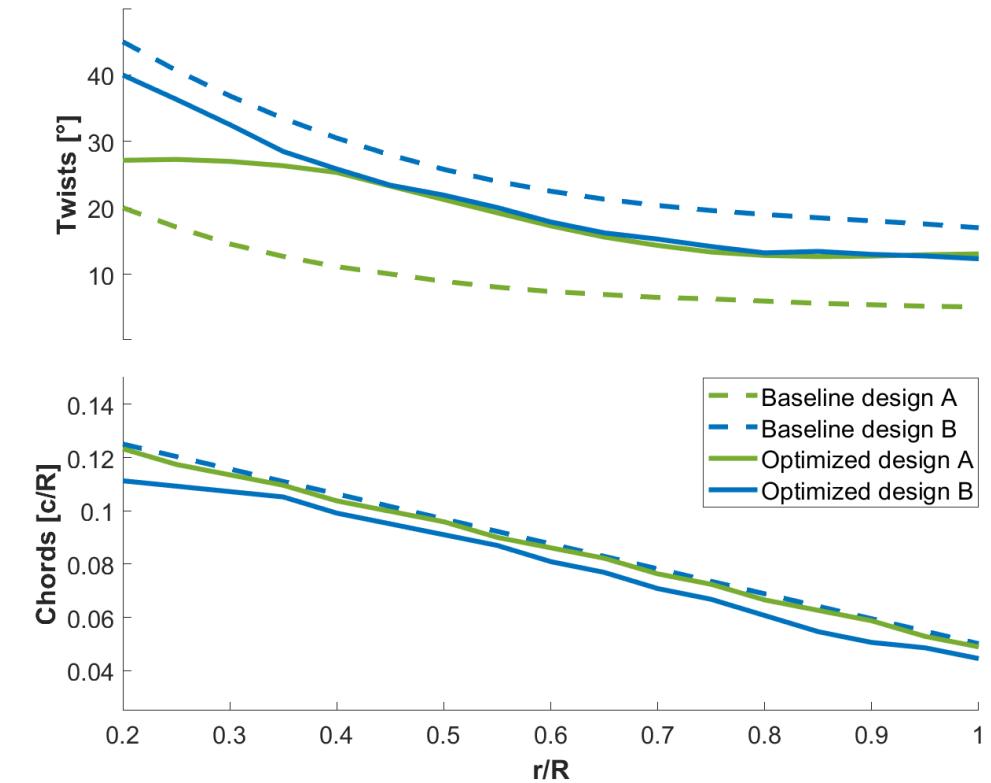


Optimization results: geometry

- Duct
 - C_P is more sensitive to the throat radius
 - Opt A) $0.409L$, $1.165R_{throat}$
 - Opt B) $0.641L$, $1.155R_{throat}$
- Blades
 - Radii are scaled with the throat radii
 - Converge to the same twist profile ($> 0.35R$)



- Blades
 - Radii are scaled with the throat radii
 - Converge to the same twist profile ($> 0.35R$)

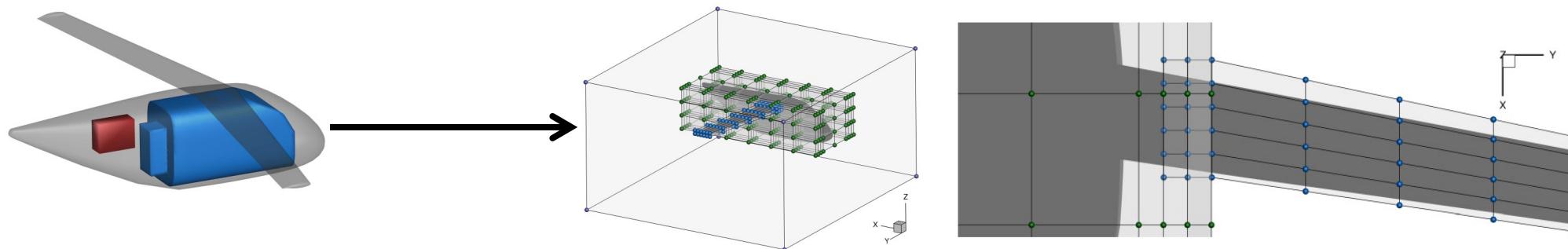
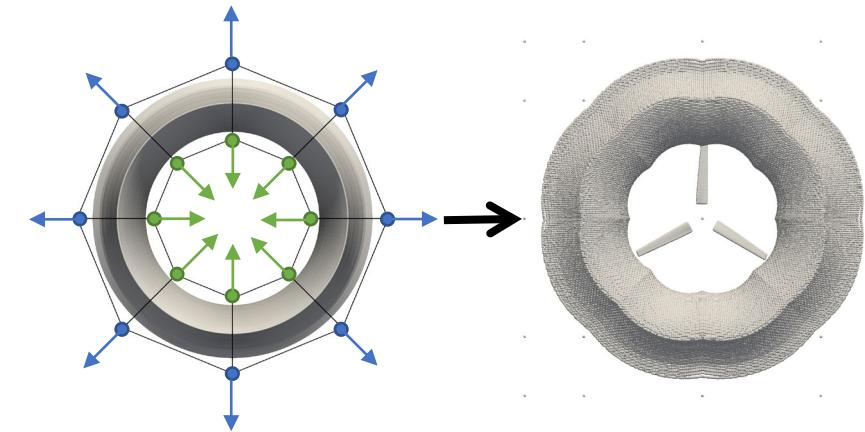


Summary of work 1

- Design optimization of thin-walled ducted hydrokinetic turbines is conducted
- Optimizations from different baselines converge to similar result
 - $C_P \sim 54\%$ (URANS solver)
 - Similar geometrical features: duct throat area, blade twist profile ($> 0.35R$)
- Ducted turbine can outperform unducted turbine for a given area
 - Ducted $C_P \sim 54\%$ vs. unducted $C_P \sim 47\%$
- Further improvement is needed

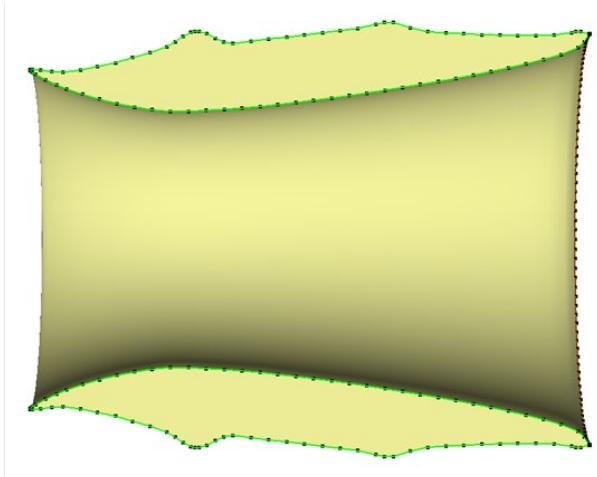
Further improvement is needed

- Thin-walled duct
 - Difficult to maintain a circular and axisymmetric shape
- No hub included
 - Complex setup is needed (Hajdik et al. 2023)

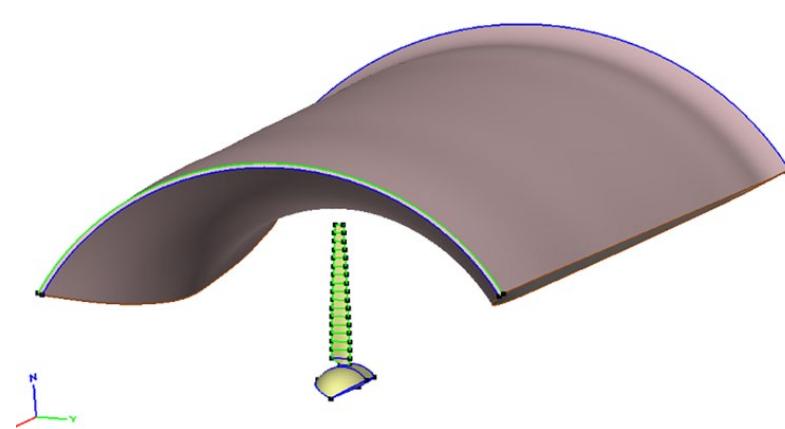


Engineering Sketch Pad (ESP)

- CAD-based geometry creation and manipulation system
- Geometry parameters can be directly used as design variables
 - Example of parametric design



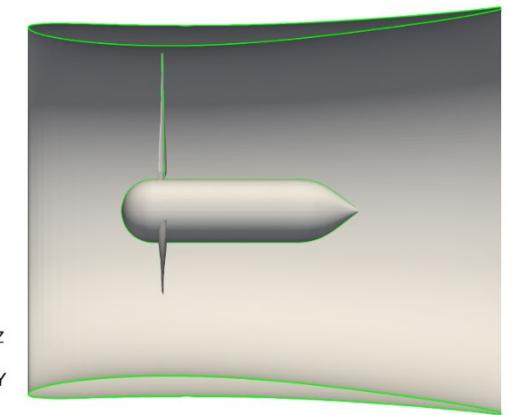
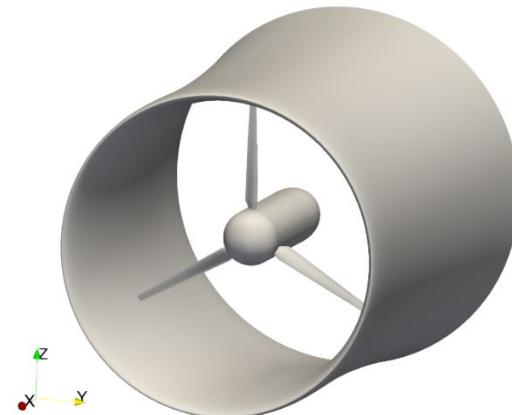
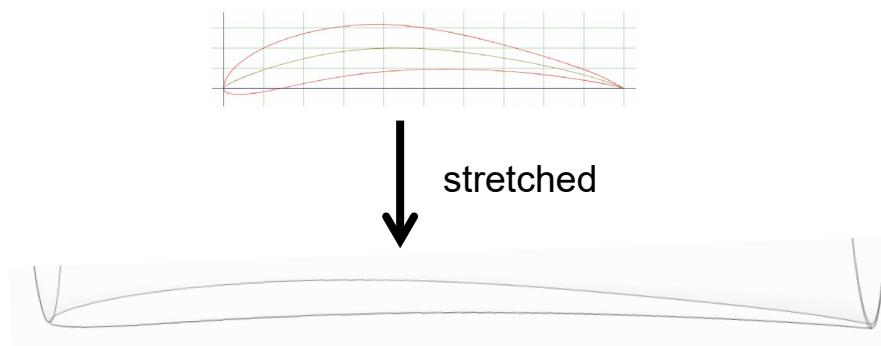
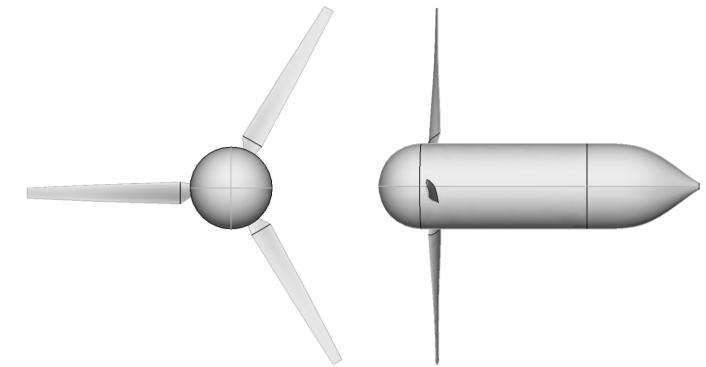
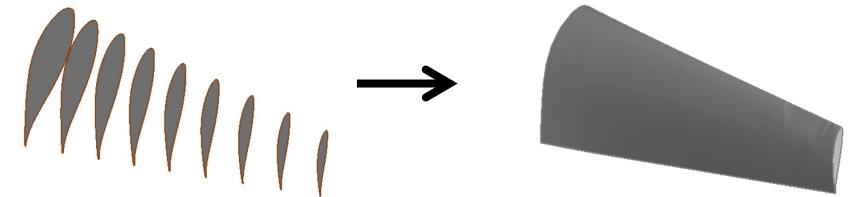
Design variables: Points locations
(duct shape is a cubic spline curve)



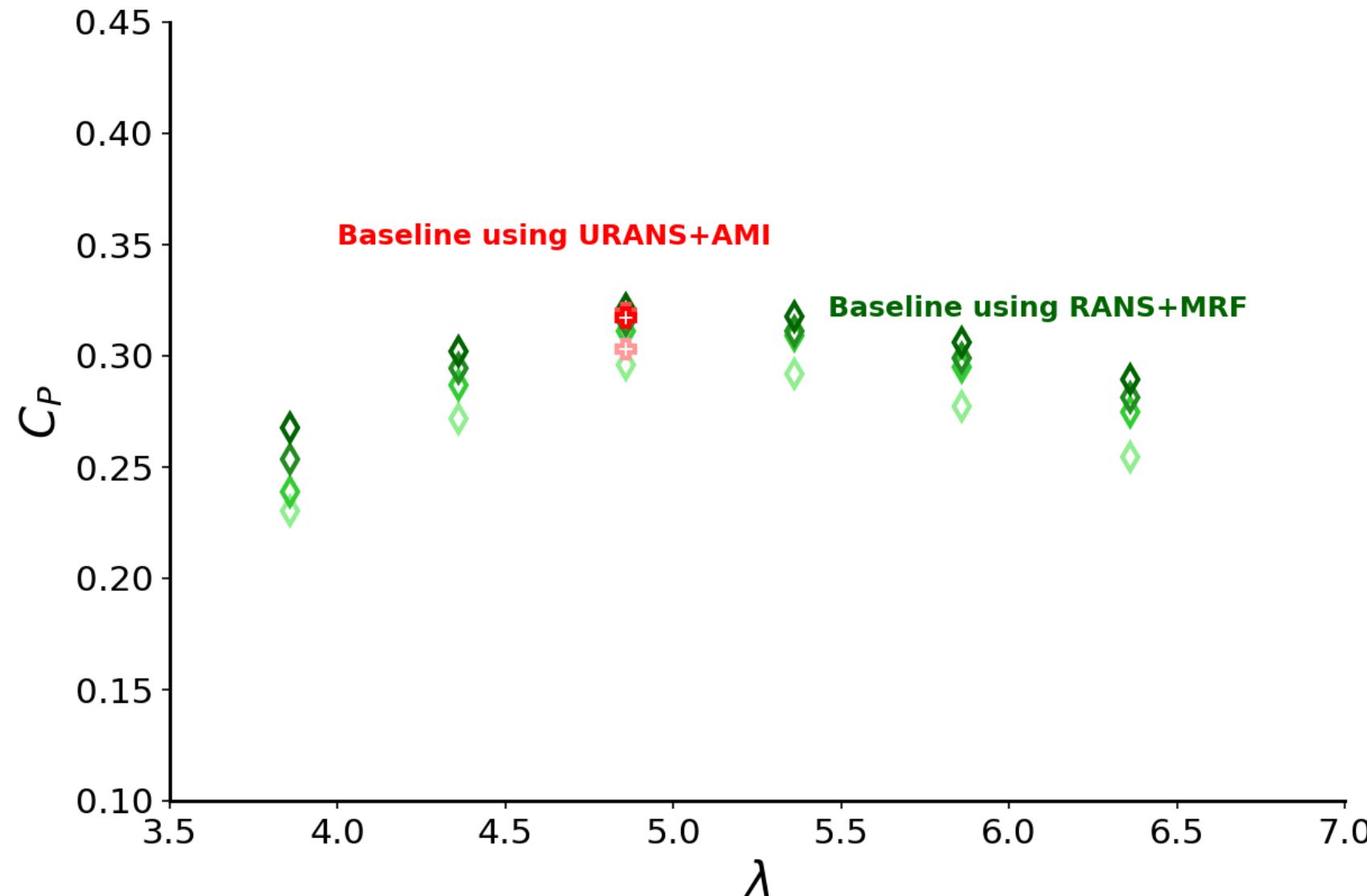
Design variables:
NACA foil thickness, camber, chord length,
blade radius, hub radius, tip gap, etc.

Baseline ducted turbine design

- Blade geometry
 - Twist/chord from previous optimization
 - 9 spanwise sections – Class-Shape function Transformation (CST)
- Hub geometry
 - Cylinder + Sphere + Cone with slight modification
- Duct geometry
 - Stretched E423 foil
 - CST parametrization



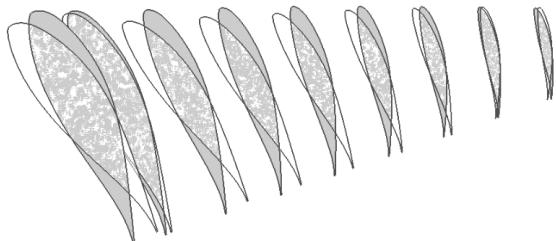
Baseline ducted turbine design



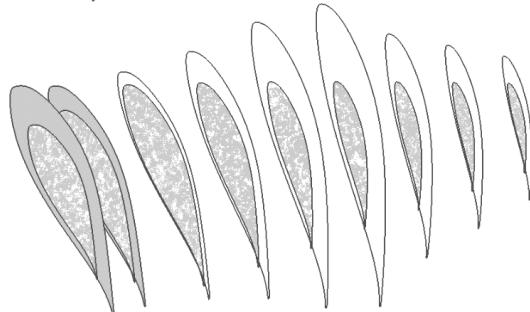
Design variable change through ESP

- Blade

(a) Twists

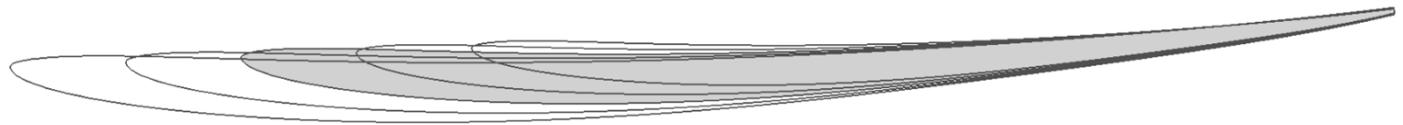


(b) Chords

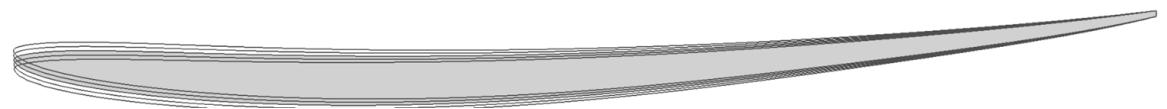


- Duct

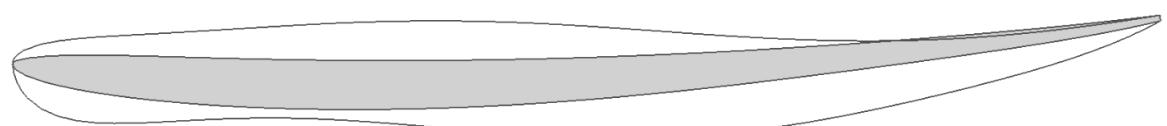
(a) Duct scale



(b) Angle of attack



(C) Duct shape



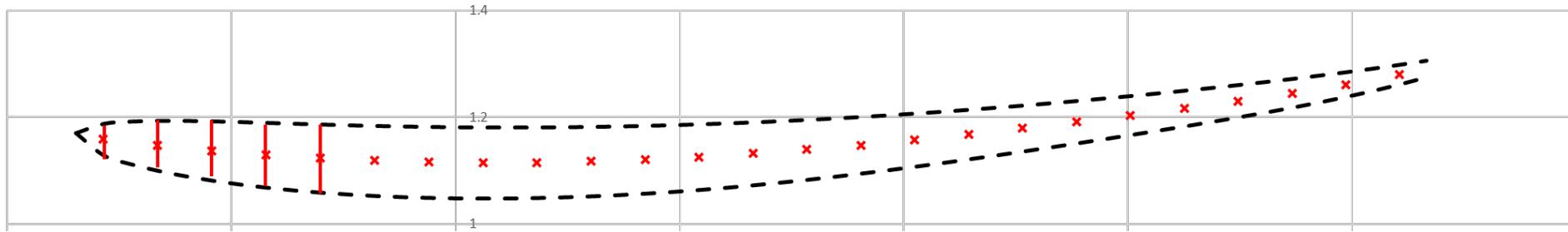
- Hub

(a) Translations of 4 points

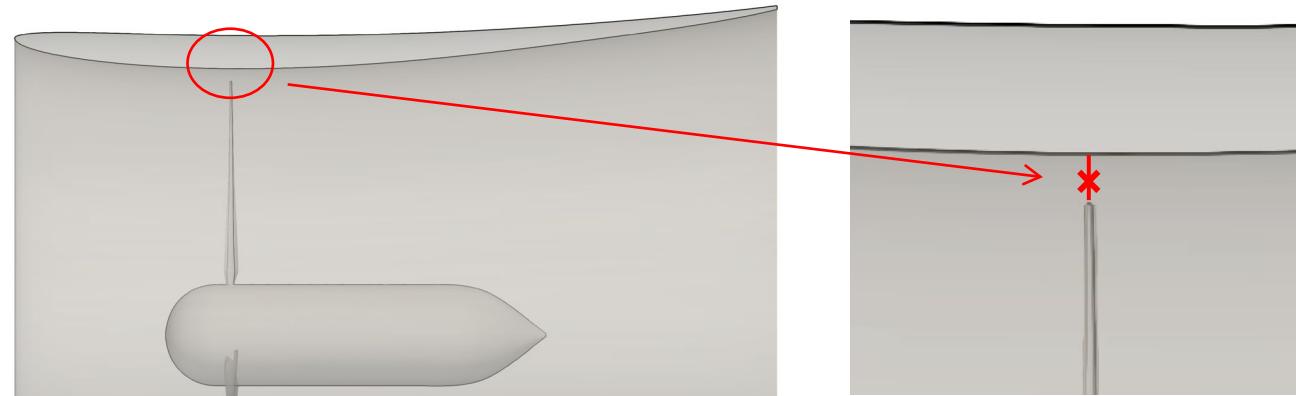


Constraints

- Duct thickness: 1D thickness constraint

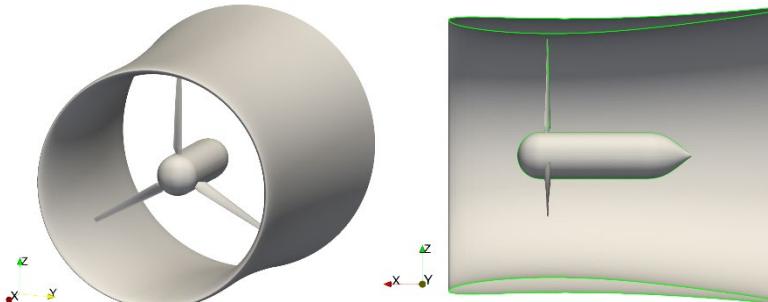


- Tip gap: proximity constraint

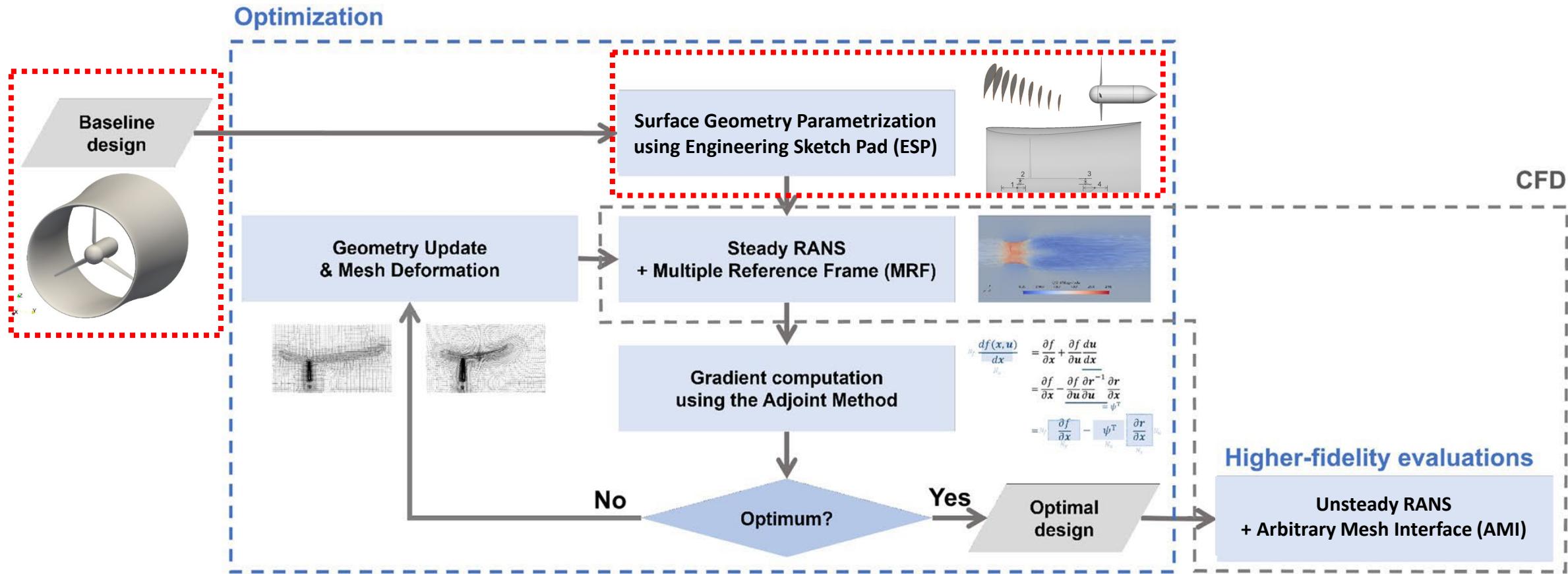


- Duct exit radius
- Duct LE curvature
- Hub cylindrical part

Optimization setup

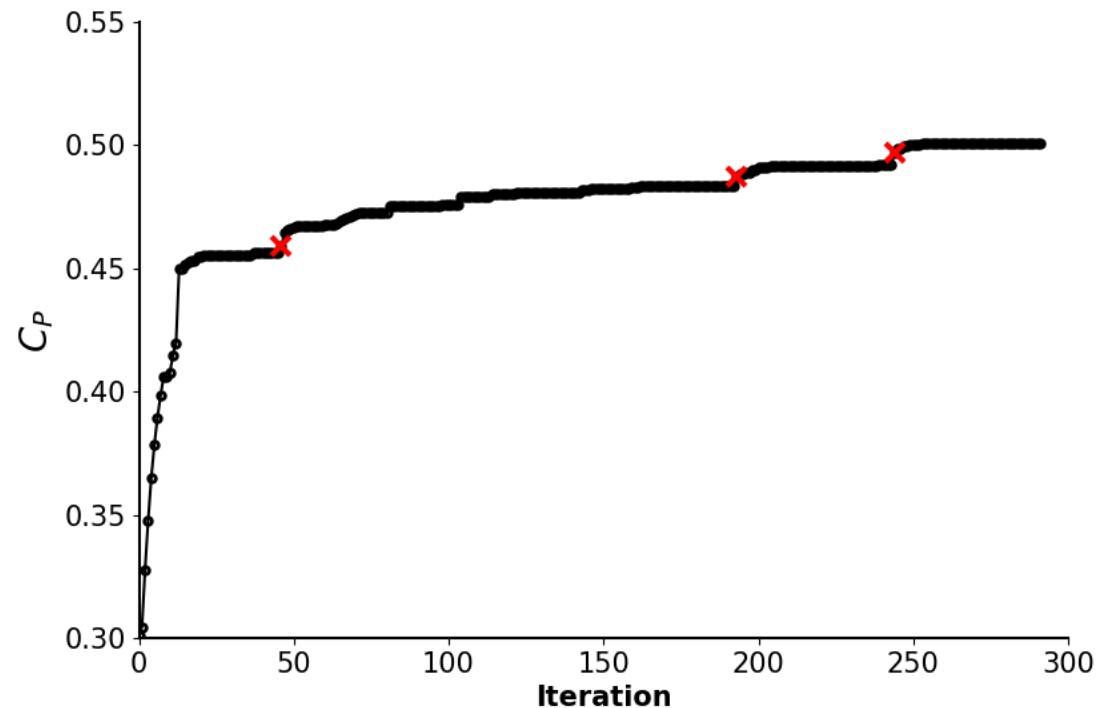
Baseline design		 Two 3D CAD models are shown side-by-side. The left model is a cylindrical duct containing a three-bladed wind turbine. The right model shows a cross-section of the duct, revealing the internal airfoil profile and the hub area.
Objective		Maximize C_P @ fixed U_∞ & Ω
Design variables	Turbine	Radius/ Chords/ Twists/ Hub shape (13 vars)
	Duct	Duct shape, Duct scale, Angle of attack (13 vars)
Constraints		Duct thickness Tip gap

Optimization setup

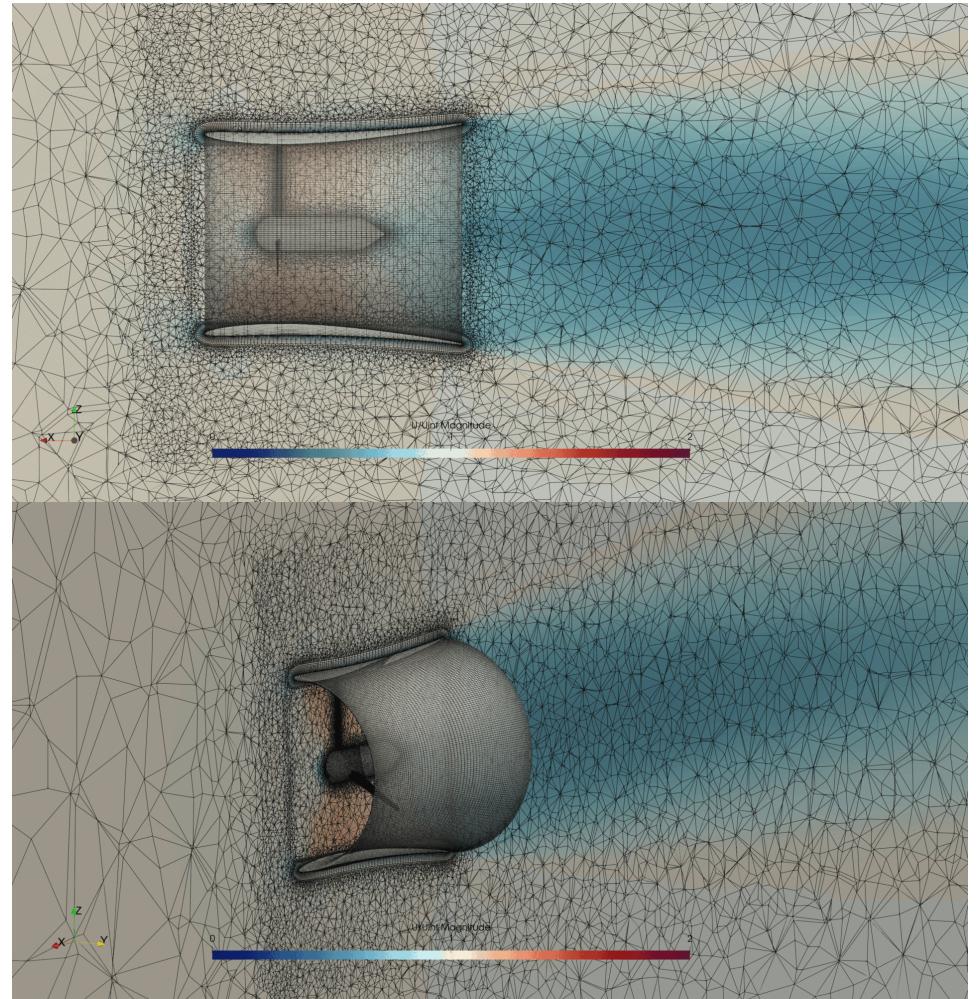


Optimization results: C_P

- Optimization: $0.3011 (\lambda = 4.86) \rightarrow 0.5009 (\lambda = 4.65)$
- Plan to run re-evaluation using URANS-AMI

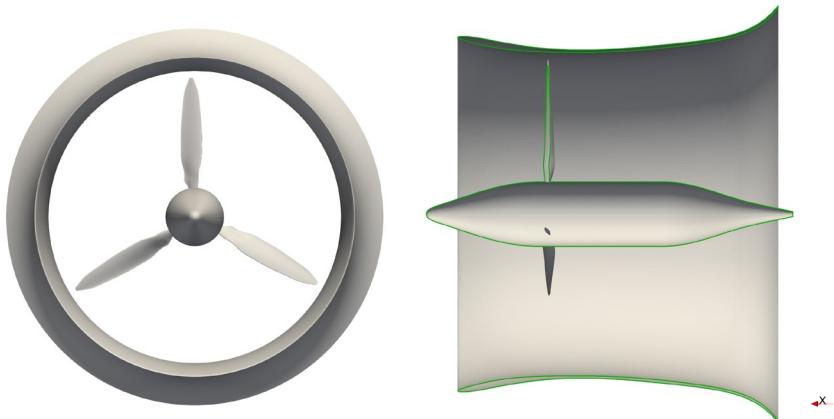


**Optimization (steady RANS + MRF)
+ SA model**

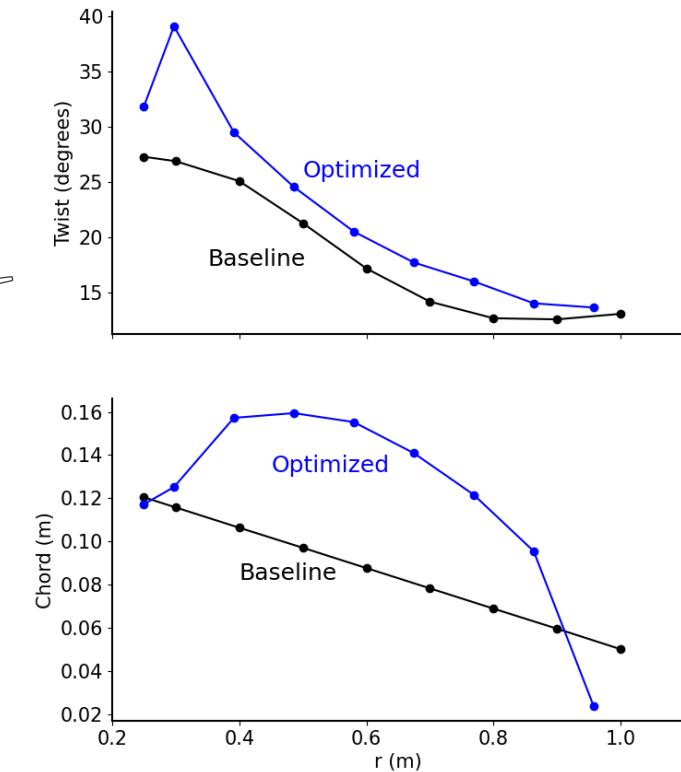
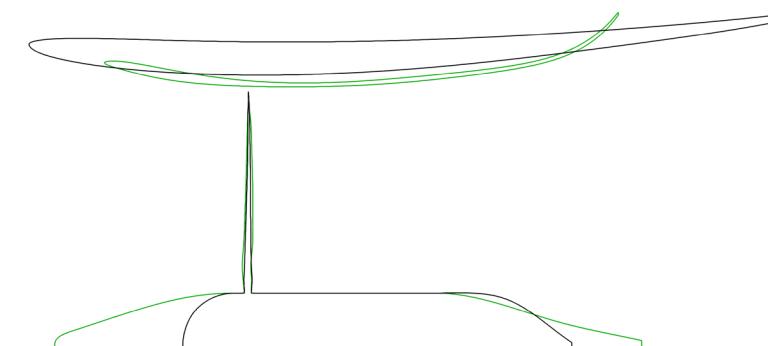


Optimization result

- Duct
 - Shorter
 - Thin and cambered
- Hub
 - Longer
 - Protruded
 - Bigger chords



- Blade
 - Smaller radius
 - Bigger chords



Summary of work 2

- Design optimization of a ducted turbine featuring a foil-shaped duct and a bulky hub is conducted
 - Geometry is parametrized using ESP
- Obtained ducted turbine has roughly 50% efficiency with unique geometrical features
- Further re-evaluation is needed

Q&A