

# Lateral Dynamic Bicycle Model

Course 1, Module 4, Lesson 5



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# Learning Objectives

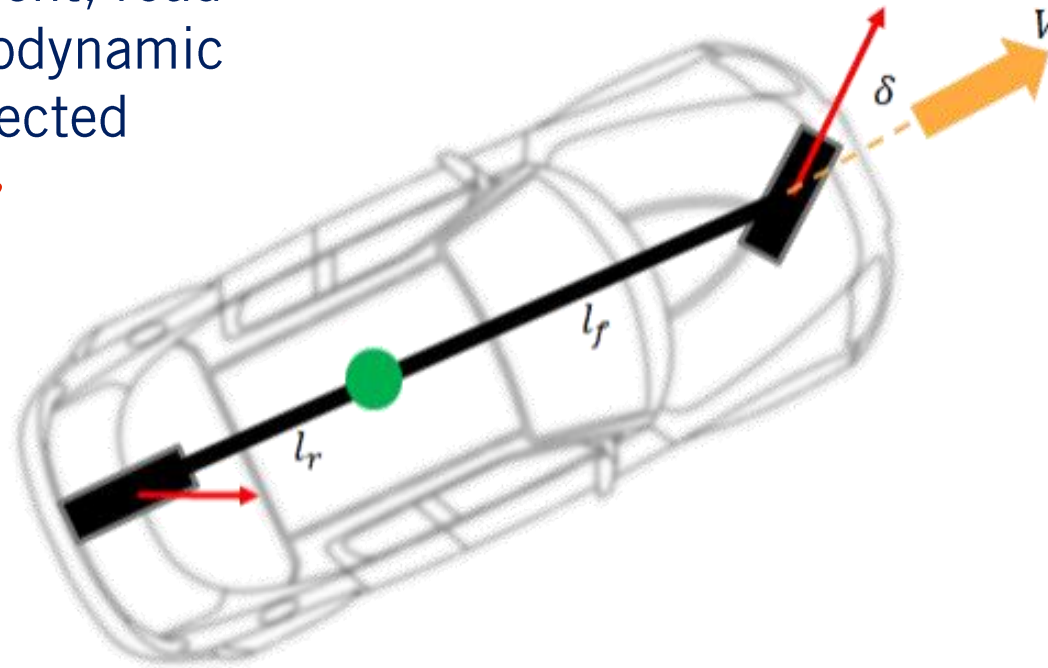
- Build a dynamic model of a car using the kinematic bicycle model as a starting point
- Convert to standard state space representation

# Vehicle Model to Bicycle Model

- Assumptions

- Longitudinal velocity is constant
- Left and right axle are lumped into a single wheel (bicycle model)
- Suspension movement, road inclination and aerodynamic influences are neglected

*nonlinear dynamics*



# Lateral Dynamics

- Lateral dynamics can be written as

$\beta$ : vehicle slip angle

$$V = \omega R$$

$$\omega = \dot{\psi}$$

Lateral acceleration

$$a_y = \ddot{y} + \omega^2 R$$

$$= V\dot{\beta} + V\dot{\psi}$$

heading rate of change

vehicle mass

$$mV(\dot{\beta} + \dot{\psi}) = F_{yf} + F_{yr}$$

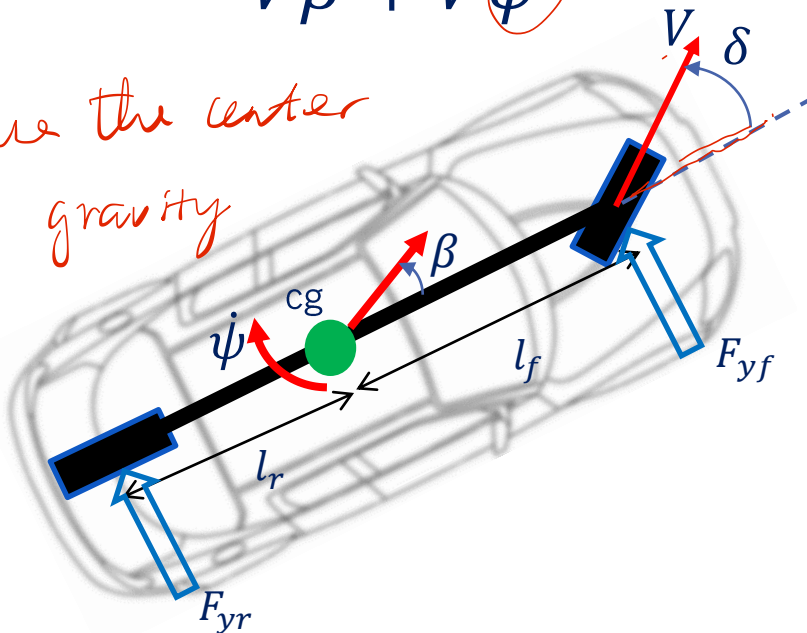
side slip rate

yaw rate

front and rear tire forces

vehicle velocity

use the center of gravity



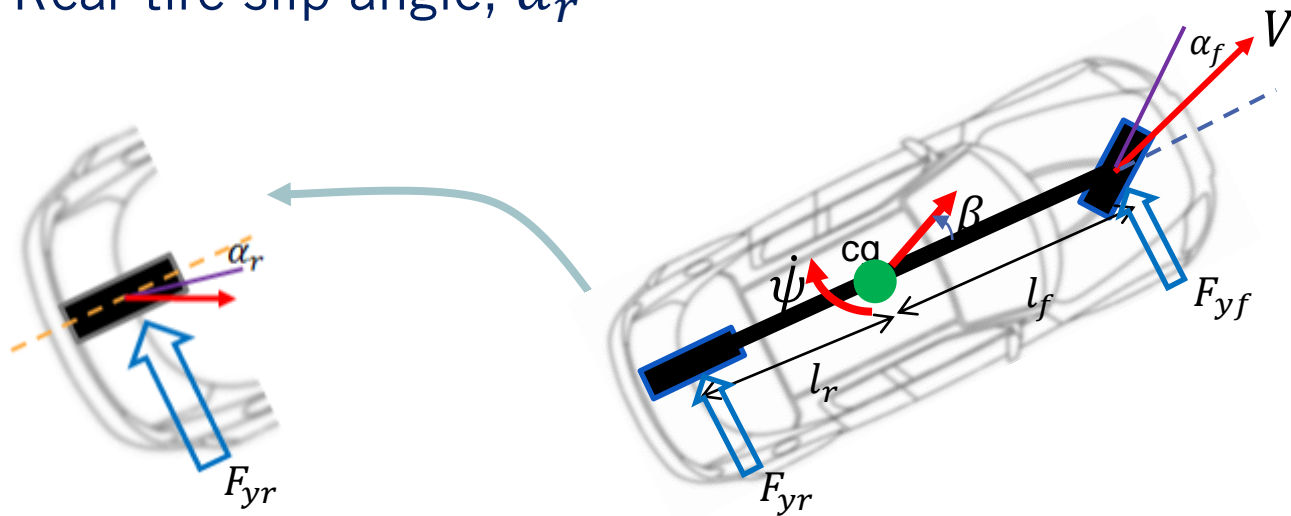
$$I_z \ddot{\psi} = l_f F_{yf} - l_r F_{yr}$$

vehicle inertia

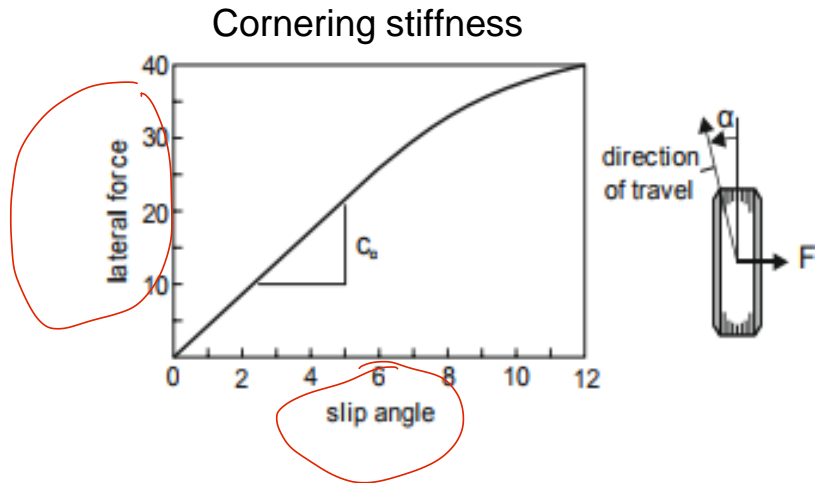
center of gravity distance from front and rear tires

# Tire Slip Angles

- Many different tire slip models
- For **small tire slip angles**, the lateral tire forces are approximated as a linear function of tire slip angle
- Tire variables
  - Front tire slip angle,  $\alpha_f$
  - Rear tire slip angle,  $\alpha_r$



# Front and Rear Tire Forces



- $C_f$  : linearized cornering stiffness of the front wheel

$$F_{yf} = C_f \alpha_f = C_f \left( \underset{\triangle}{\delta} - \underset{\triangle}{\beta} - \frac{l_f \dot{\psi}}{V} \right)$$

steering angle

- $C_r$  : linearized cornering stiffness of the rear wheel

$$F_{yr} = C_r \alpha_r = C_r \left( -\beta + \frac{l_r \dot{\psi}}{V} \right)$$

# Lateral and Yaw Dynamics

- From the previous slide formulations:

$$F_{yf} = C_f \alpha_f = C_f \left( \delta - \beta - \frac{l_f \dot{\psi}}{V} \right)$$

$$F_{yr} = C_r \alpha_r = C_r \left( -\beta + \frac{l_r \dot{\psi}}{V} \right)$$

Substitute the lateral forces

$$mV(\dot{\beta} + \dot{\psi}) = F_{yf} + F_{yr}$$

$$I_z \ddot{\psi} = l_f F_{yf} - l_r F_{yr}$$

Rearranging the equations

$$\dot{\beta} = \frac{-(C_r + C_f)}{mV} \beta + \left( \frac{C_r l_r - C_f l_f}{mV^2} - 1 \right) \dot{\psi} + \frac{C_f}{mV} \delta$$

rate of change  
of yaw rate

$$\ddot{\psi} = \frac{C_r l_r - C_f l_f}{I_z} \beta - \frac{C_r l_r^2 + C_f l_f^2}{I_z V} \dot{\psi} + \frac{C_f l_f}{I_z} \delta$$

# Standard State Space Representation

- State Vector:  $X_{lat} = [y \quad \beta \quad \psi \quad \dot{\psi}]^T$   
lateral position    side slip angle    yaw angle    yaw rate

$$\dot{X}_{lat} = A_{lat}X_{lat} + B_{lat}\delta$$

$$A_{lat} = \begin{bmatrix} 0 & V & V & 0 \\ 0 & -\frac{C_r + C_f}{mV} & 0 & \frac{C_r l_r - C_f l_f}{mV^2} - 1 \\ 0 & 0 & 0 & 1 \\ 0 & \frac{C_r l_r - C_f l_f}{I_z} & 0 & -\frac{C_r l_r^2 + C_f l_f^2}{I_z V} \end{bmatrix}$$

$$B_{lat} = \begin{bmatrix} 0 \\ \frac{C_f}{mV} \\ 0 \\ \frac{C_f l_f}{I_z} \end{bmatrix}$$



# Summary

What we have learned from this lesson?

- Formulated the lateral dynamics of a bicycle model
- Defined a state space representation of lateral model

What is next?

- Vehicle actuation system models
  - Throttle, brake & steering