



## DEPARTMENT OF ENGINEERING CYBERNETICS

TTK4550 - SPECIALIZATION PROJECT

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# Design and Control of a Spring-actuated Jumping Quadruped in Earth Gravity

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## Abstract

This specialization project involves the development of a concept design for a small, lightweight, and low-cost jumping quadruped robot, which jumps by releasing the energy stored in torsional springs embedded in parallel to the robots knee motor. The purpose of the robot is to be capable of jumping long distances in earth gravity, and to be used as a platform for testing quadruped jumping control algorithms. This goal must be seen in the context of the potential utility of jumping quadrupeds in extraterrestrial exploration, where they could traverse obstacles that would be insurmountable to wheeled rovers. The robot is designed to be small and lightweight to reduce the risk of damage during testing, and low-cost to make it more accessible to researchers. In addition to the general concept design, this report presents a Simulink Simscape simulation of the robot, used to evaluate various design choices. The report also presents a detailed CAD model for the spring-actuated leg, as well as an experiment-based system identification of the gear-friction of the selected motors. Using the identified friction characteristics of the motor, our simulations indicate a potential center of mass jump height of 0.5m for a robot with a total leg length of 20cm. TODO add the precise newest number.

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## Abbreviations

<b>Abbreviation</b>	<b>Description</b>
TODO	To be filled in

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# 1 Introduction

## 1.1 Motivation

The exploration of extraterrestrial environments represents one of the most demanding frontiers of robotic systems, requiring exceptional autonomy, resilience, and adaptability to navigate complex and unpredictable terrain. On Mars, wheeled rovers have proven their utility, with six successful deployments to date [18], robots like Axel [9] and Reachbot [10] have also been designed, tailored towards specific tasks. One such task that has received much attention in recent years, is the exploration of potential Martian and Lunar lava tubes [3]. These tubes are hollow caverns hypothesized to exist beneath the surface of Mars and the Moon, formed by ancient lava flows. They are of particular interest to astrobiologists and planetary scientists, as they could provide shelter from cosmic radiation and micrometeorites, as well as stable temperatures and access to subsurface water ice [3].

The exploration of such lava tubes present a unique challenge to robotic systems, as they are believed to be characterized by rough, uneven terrain, sharp rocks, and steep slopes. This could present a challenge to traditional wheeled rovers. Further, the motion of wheeled robots is limited to the ground plane, and thus, inherently, they do not utilize the lower gravity of extraterrestrial objects such as asteroids, the Moon and Mars. Jumping quadrupeds, on the other hand, inherently utilize the lower gravity of such objects, and in low earth gravity could potentially jump to heights of several meters [12]. This could allow them to traverse obstacles that would be insurmountable to wheeled rovers, such as steep slopes, large rocks, and gaps in the terrain.

While recent years have seen great progress in the development of quadruped robots, most quadrupeds still struggle with jumping any significant distance in earth gravity. Since, additionally, low gravity environments are very hard to replicate on earth, it is difficult to test hardware and control algorithms intended for quadrupeds jumping in low gravity. Jumping also includes high velocity impacts, making damage to the often expensive hardware likely. This motivates the main goal of this project, which is to develop a design for a small, lightweight, and low-cost jumping quadruped robot. The robot's low weight is intended to reduce the risk of damage during testing, and the low cost to make it more accessible to researchers, as well as reduce the cost of potential damage. Special emphasis is placed on being able to jump long distances, without losing the general utility of the quadruped form factor, such as the ability to walk on rough terrains, flexibly adjust body pose, and potentially carry scientific payloads.

## 1.2 Scope

As described in the motivation section, section 1.1, the main goal of this project is to develop a design for a small, lightweight, and low-cost jumping quadruped robot. The work presented in this report is part of a specialization project, TTK4550 - Engineering Cybernetics, Specialization Project, pursued at the Norwegian University of Science and Technology (NTNU), as a preparation for a master's thesis. So while the scope of the specialization project is limited to the development of a design, the overall goal is for the design to be used as the basis for a master's thesis, where the robot will be built and tested. The master's thesis will also include the development of control algorithms for the robot, which is not included in this report.

More precisely, the scope of this project is limited to the following:

- Developing a simplified simulation for the robot in MATLAB/Simulink, to be used for verification and evaluation of various design choices.
- Choosing a specific method of actuation, such as motors, parallel torsional springs, parallel extension springs, or a combination of these.
- Identifying key hardware components, such as motors and springs.

- 
- Designing a CAD model for a single leg of the robot. The leg must adhere to geometric and mechanical constraints such as:
    - Accommodating chosen springs and motors.
    - Being easily manufacturable using 3D printing and the CNC facilities readily available at NTNU.
    - Sturdiness, ie. being able to withstand the forces and impacts of jumping.

### 1.3 Related Work

The problem of robotic jumping in earth and low gravity environments has been studied by several researchers, with various approaches taken. One unique example is the Olympus robot [11] [12] developed by NTNU's ARL (Autonomous Robots lab), which uses a 5-bar linkage spring assisted leg to jump. The robot weighs TODO kg, is capable of jumping to heights of up to TODO meters in earth gravity, and has been tested in simulated low gravity environments. Another example is the 600g robot RAVEN (Robotic Avian-inspired Vehicle for multiple ENvironments) [14] developed at EPFL, which uses its bird-inspired 2 DOF multifunctional legs to jump rapidly into flight, walk on the ground, and hop over obstacles and gaps similar to the multimodal locomotion of birds. Notable for RAVEN is its geared BLDC motors, which wind up embedded torsional springs, which then assist in jumping. Apart from the different topology of the legs and springs, the concept is quite similar to that of Olympus. The RAVEN robot can jump TODO (26 cm) cm in earth gravity. A third example is the Grillo robot [13], which weighs 15g and takes off at velocities of about 30 body lengths per second, ie. 1.5m/s.

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## 2 Theory

### 2.1 Actuator Modeling

#### 2.1.1 DC Motor Model

#### 2.1.2 BLDC Motor Model

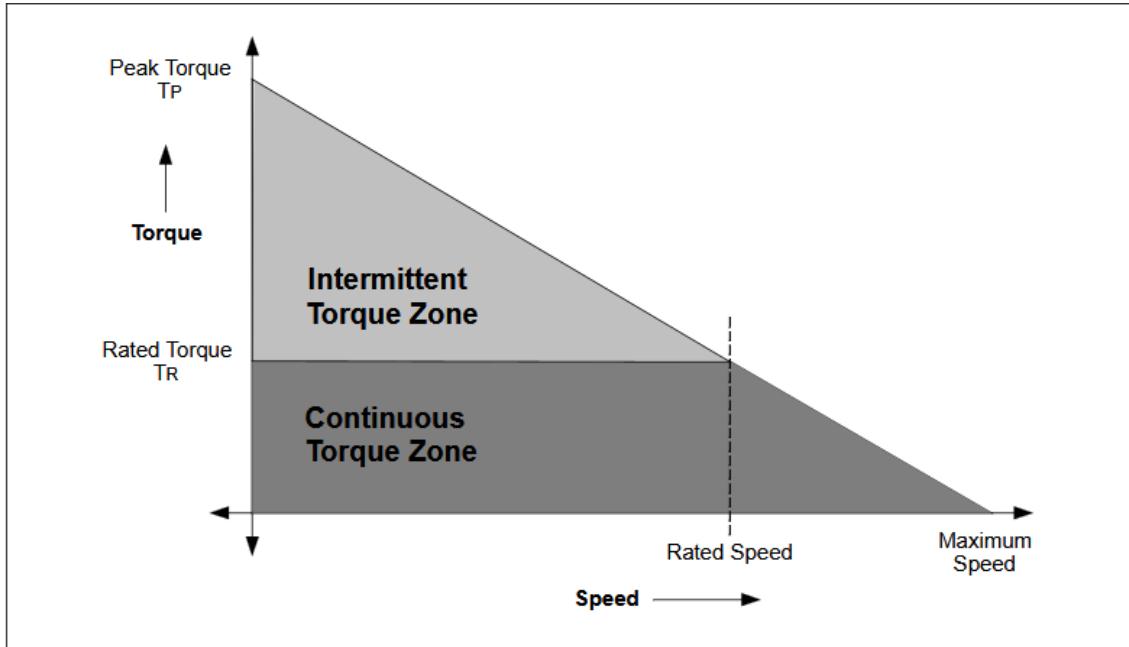


Figure 1: Torque-speed characteristics of a BLDC motor. TODO: Replace with an image that isn't stolen, and that we are allowed to use.

#### 2.1.3 Gear Transmission Friction Model

Although electric motors for robotics are available for a wide power range, they are often too high speed and low power to do any useful work. For this reason, it is often necessary to use a gear transmission to increase the torque and reduce the speed of the motor.

In the presence of a geared transmission, assuming no power loss, the output torque and velocity of a geared motor are given by equation 1 and equation 2, respectively, where  $N$  is the gear ratio,  $\tau_{in}$  is the input torque,  $\tau_{out}$  is the output torque,  $w_{in}$  is the input velocity, and  $w_{out}$  is the output velocity [7].

$$w_{out} = \frac{w_{in}}{N} \quad (1)$$

$$\tau_{out} = N\tau_{in} \quad (2)$$

In reality, however, there is always some power loss in the transmission. A common way to model this is to use friction model consisting of a viscous friction term and a Coulomb friction term [7]. The viscous friction term is proportional to the velocity of the transmission, and the Coulomb friction term is a constant friction torque that must be overcome before the transmission starts moving. The total friction torque is the sum of these two terms, as seen in equation 3. It is also possible to drop one or the other of these terms, depending on the application [7].

---


$$\tau_{friction} = b_{viscous}\dot{\theta} + b_{coulomb}\text{sign}(\dot{\theta}) \quad (3)$$

In addition to friction, heavily gearing motors can lead to a very high apparent rotor inertia. If one looks at equation 4, it is clear that the apparent rotor inertia is proportional to the square of the gear ratio [7]. This can lead to a very high apparent rotor inertia, which can often be problematic to robotic applications. This is especially the case for cases with contact forces, as the high apparent rotor inertia can lead to very stiff and damaging collisions [17].

$$K = \frac{1}{2}I_{rotor}(G\dot{\theta})^2 = \frac{1}{2}I_{rotor}G^2(\dot{\theta})^2 = \frac{1}{2}I_{apparent}(\dot{\theta})^2 \quad (4)$$

## 2.2 Spring Modeling

Springs are mechanical devices that store and release energy when subjected to displacement. There are two main types of springs: extension springs and torsion springs.

Extension springs are designed to operate with a tension load, meaning they extend as the load is applied. The force exerted by an extension spring is proportional to the displacement from its equilibrium position, following Hooke's Law, which is given by:

$$F = -kx \quad (5)$$

where  $F$  is the force exerted by the spring,  $k$  is the spring constant, and  $x$  is the displacement from the equilibrium position. The potential energy stored in an extension spring is given by:

$$U = \frac{1}{2}kx^2 \quad (6)$$

Torsion springs, on the other hand, are designed to operate with a rotational or twisting load. They exert a torque that is proportional to the angular displacement from their equilibrium position. The torque generated by a torsion spring is given by:

$$\tau = -k\theta \quad (7)$$

where  $\tau$  is the torque,  $k$  is the torsion spring constant, and  $\theta$  is the angular displacement. The potential energy stored in a torsion spring is given by:

$$U = \frac{1}{2}k\theta^2 \quad (8)$$

Both types of springs are widely used in various mechanical systems to provide force or torque, absorb shock, and store energy.

## 2.3 Kinematics

### 2.3.1 Robot Kinematics

Consider a robotic link arm existing in  $\mathbb{R}^2$  consisting of  $n$  links, each with a length  $l_i$  and a joint angle  $q_i$ . The position of the end-effector is given by the vector  $\mathbf{x} = [x, y]^T$ , where  $x$  and  $y$  are the coordinates of the end-effector in the global coordinate system. Using simple trigonometry, the position of the end-effector can be expressed as a function of the joint angles and link lengths as seen in equation 9. Axes and joint angles corresponding to the expression in equation 9 can be seen in figure 2.

$$\mathbf{x} = \begin{bmatrix} x \\ y \end{bmatrix} = \begin{bmatrix} \sum_{i=1}^n l_i \cos \left( \sum_{j=1}^i q_j \right) \\ \sum_{i=1}^n l_i \sin \left( \sum_{j=1}^i q_j \right) \end{bmatrix} \quad (9)$$


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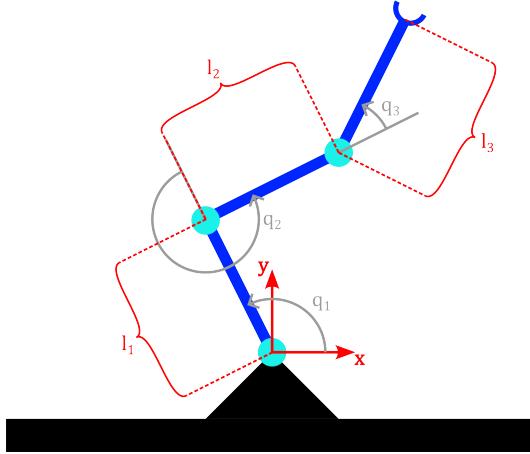


Figure 2: Illustration of a 3 link robotic link arm in  $\mathbb{R}^2$  with  $n$  links.

### 2.3.2 Jacobian Matrix

As described in section 2.3.1, the position of the end-effector can be expressed as a function of the joint angles and link lengths. In robotics, it is often useful to express the relationship between infinitesimal changes in the joint angles and the resulting change in the end-effector position. As can be seen in equation 10, infinitesimal changes in variables  $\delta y$  and  $\delta x$  can be described by means of the partial derivative [4]. If this is compared to the definition of the jacobian in equation 11, it is clear that the jacobian matrix  $\mathbf{J}$  can be used to map infinitesimal changes in joint angles to changes in the end-effector position, as illustrated in equation 12. The limit of an infinitesimal change over an infinitesimal time interval is a derivative, and thus by dividing each side in equation 12 by  $\delta t$ , one arrives at the expression in equation 13, by which the jacobian can be used to map joint velocities to end-effector velocities.

$$\delta y = \frac{\partial y}{\partial x} \delta x \quad (10)$$

$$\mathbf{J} = \begin{bmatrix} \frac{\partial x}{\partial q_1} & \frac{\partial x}{\partial q_2} & \dots & \frac{\partial x}{\partial q_n} \\ \frac{\partial y}{\partial q_1} & \frac{\partial y}{\partial q_2} & \dots & \frac{\partial y}{\partial q_n} \end{bmatrix} \quad (11)$$

$$\delta \mathbf{x} = \mathbf{J} \delta \mathbf{q} \quad (12)$$

$$\dot{\mathbf{x}} = \mathbf{J} \dot{\mathbf{q}} \quad (13)$$

### 2.3.3 Force/Torque Mapping

Consider a general robotic manipulator, such as the one illustrated in figure 2, but with an arbitrary amount,  $n$ , of joints and links. Using the principle of conservation of power, one arrives at the formulation found in equation 14.

$$\text{power at the joints} = (\text{power to move the robot}) + (\text{power at the end-effector}) \quad (14)$$

As the power used to move the robot approaches zero, ...

---

Consider a robotic manipulator with  $n$  joints, each with a joint angle  $q_i$  and a joint torque  $\tau_i$ . The position of the end effector for such a system is given by equation 9, and thus the formula in equation 13 can be used to map joint velocities to end effector velocities.

TODO: Derive equation 15 by either power balance, or virtual work.

$$\tau = J^T F \quad (15)$$

### 2.3.4 Inverse Kinematics

Inverse kinematics solves for joint angles that achieve a desired end-effector pose. For a planar two-link manipulator with link lengths  $L_1$  and  $L_2$ , given a desired end-effector position  $(x, y)$ , the joint angles  $\theta_1$  and  $\theta_2$  can be found analytically.

The forward kinematics equations are described by the 2-link case of equation 9.

$$x = L_1 \cos(\theta_1) + L_2 \cos(\theta_1 + \theta_2) \quad (16)$$

$$y = L_1 \sin(\theta_1) + L_2 \sin(\theta_1 + \theta_2) \quad (17)$$

From the law of cosines,  $\theta_2$  is:

$$\theta_2 = \pm \arccos \left( \frac{x^2 + y^2 - L_1^2 - L_2^2}{2L_1L_2} \right) \quad (18)$$

The angle  $\theta_1$  is then:

$$\theta_1 = \arctan 2(y, x) - \arctan 2(L_2 \sin(\theta_2), L_1 + L_2 \cos(\theta_2)) \quad (19)$$

The  $\pm$  in the equation for  $\theta_2$  indicates that two solutions exist for most end-effector positions—one with the second joint angle positive and one negative. When  $L_1 = L_2$ , a singularity occurs at  $(0, 0)$  where infinite solutions exist. At the workspace boundary where  $x^2 + y^2 = (L_1 + L_2)^2$ , only one solution exists.

## 2.4 Contact Friction

TODO

## 2.5 Numerical Solvers

Although a detailed review of numerical solvers and the associated theory is beyond the scope of this theory section, it is worth mentioning that numerical simulation of contact dynamics is particularly challenging. This is due to the discontinuities and high-frequency events that occur during contact. These dynamics often require the use of stiff solvers to accurately capture the rapid changes in forces and velocities [5][15]. Stiff solvers are designed to handle problems with widely varying timescales, ensuring stability and accuracy in the simulation of contact events. Without the use of stiff solvers, simulations can become unstable or fail to converge, leading to inaccurate results. Examples of stiff solvers include the well known ode15s and ode23s solvers in MATLAB, which are specifically designed to handle stiff ordinary differential equations [16].

---

## 2.6 Linear Least Squares Regression

Linear least squares regression is a method for finding the best-fitting line through a set of points by minimizing the sum of squared residuals. Given a set of observations  $(x_i, y_i)$  and a linear model  $y = X\beta$ , where  $X$  is the matrix of input variables and  $\beta$  contains the model parameters, the residual  $r_i$  for each observation is:

$$r_i = y_i - X_i\beta \quad (20)$$

The sum of squared residuals  $S$  is then:

$$S = \sum_{i=1}^n r_i^2 = (y - X\beta)^T(y - X\beta) \quad (21)$$

To minimize  $S$ , we take its derivative with respect to  $\beta$  and set it to zero:

$$\frac{\partial S}{\partial \beta} = -2X^T(y - X\beta) = 0 \quad (22)$$

Solving for  $\beta$  yields the normal equations:

$$X^T X \beta = X^T y \quad (23)$$

The solution is therefore:

$$\beta = (X^T X)^{-1} X^T y \quad (24)$$

This solution minimizes the sum of squared residuals and provides the optimal parameters  $\beta$  in the least squares sense.

## 2.7 Mean Square Error

Mean square error (MSE) measures the average squared difference between predicted values and actual values. For a set of  $n$  predictions  $\hat{y}_i$  and corresponding true values  $y_i$ , MSE is defined as:

$$\text{MSE} = \frac{1}{n} \sum_{i=1}^n (y_i - \hat{y}_i)^2 \quad (25)$$

MSE penalizes larger errors more heavily due to the squared term, making it useful for evaluating model fit quality. A lower MSE indicates better model performance.

## 2.8 Moving Average Filter

A moving average filter smooths data by replacing each point with the average of a window of neighboring points. For a window size  $w$ , the filtered value  $y_i$  at index  $i$  is:

$$y_i = \frac{1}{w} \sum_{j=i-\lfloor w/2 \rfloor}^{i+\lfloor w/2 \rfloor} x_j \quad (26)$$

---

where  $x_j$  are the input values. The filter reduces high-frequency noise while preserving lower-frequency trends in the data. Larger window sizes provide more smoothing but can attenuate rapid changes in the signal.

## 2.9 Centered Finite Differences

Centered finite differences approximate derivatives using symmetric sampling around each point. For time series data with constant time step  $\Delta t$ , the first and second derivatives at index  $i$  are approximated as:

$$\dot{x}_i = \frac{x_{i+1} - x_{i-1}}{2\Delta t} \quad (27)$$

$$\ddot{x}_i = \frac{x_{i+1} - 2x_i + x_{i-1}}{(\Delta t)^2} \quad (28)$$

## 2.10 Inertia

The moment of inertia,  $I$ , quantifies the rotational inertia of an object, indicating how much torque is required for a desired angular acceleration.

### 2.10.1 Parallel Axis Theorem

The Parallel Axis Theorem allows the calculation of the moment of inertia of a body about any axis, given its moment of inertia about a parallel axis through its center of mass. It is mathematically expressed as:

$$I = I_{\text{cm}} + md^2 \quad (29)$$

where:

- $I$  is the moment of inertia about the chosen axis,
- $I_{\text{cm}}$  is the moment of inertia about the center of mass axis,
- $m$  is the mass of the object,
- $d$  is the distance between the two parallel axes.

### 2.10.2 Thin Rod

For a thin rod of length  $l$  and mass  $m$ , rotating about an axis perpendicular to the rod and passing through its end, the moment of inertia is derived using the Parallel Axis Theorem. The moment of inertia about the center of mass is:

$$I_{\text{cm, rod}} = \frac{1}{12}ml^2 \quad (30)$$

Applying the Parallel Axis Theorem with  $d = \frac{l}{2}$ :

$$I_{\text{rod}} = I_{\text{cm, rod}} + m \left( \frac{l}{2} \right)^2 = \frac{1}{12}ml^2 + \frac{1}{4}ml^2 = \frac{1}{3}ml^2 \quad (31)$$

---

### 2.10.3 Disk

For a solid disk of radius  $r$  and mass  $m$ , the moment of inertia about its center is:

$$I_{\text{cm, disk}} = \frac{1}{2}mr^2 \quad (32)$$

Using the Parallel Axis Theorem to find the moment of inertia about an axis parallel and a distance  $d$  away:

$$I_{\text{disk}} = I_{\text{cm, disk}} + md^2 = \frac{1}{2}mr^2 + md^2 \quad (33)$$

## 3 Modeling and Simulation

For the purpose of doing design verification and optimization, a simplified model of the robot was created. The model was created in Simscape, a physical modeling toolbox integrated with MATLAB/Simulink.

### 3.1 Simscape

Simscape is a simulation tool that allows you to rapidly create models of physical systems within Mathworks' MATLAB/Simulink environment. With Simscape, physical systems are built by interconnecting blocks representing physical components, such as rigid bodies, joints and springs in a block diagram. The blocks are parameterized by physical properties, such as mass, inertia, and damping. Simscape automatically generates the equations of motion for the system, which can be solved numerically to simulate the system's behavior. Like you can do with Simulink without Simscape, you can also add ordinary Simulink blocks, including Matlab Function blocks, to the model. Simscape is also compatible with Simulink's multiple numerical solvers, such as ode15s, ode45, and ode23s.

An example of a typical SimScape block diagram can be found in figure 3. A visualization of the corresponding model can be seen in figure 4. As one can see, each element in a block diagram is typically either a rigid body, or joints connecting the various rigid bodies. Since a given body has multiple possible locations that a joint could be connected to, as well as axes it can act on, blocks can export different frames, with different origins and orientations, depending on the desired position and orientation of the joint. For example, for a block representing the robotic equivalent of a thigh, natural output frames would be the ones with origins at the top and bottom of the thigh, with a select axis aligned with the desired knee or hip axis of rotation. TODO: We want to use something other than the tutorial here, but our robot model is too split into submodels of submodels, so it loses the intuition you get from a "flatter" model. Will fix later.

### 3.2 Rigid Body Components of the Robot Model

Since the purpose of the Simscape model is not to facilitate the development of a complicated full degree of freedom feedback controller, nor to optimize every small detail of the design, a simplified model was selected. This model consists of a main body with four legs, each of which with two degrees of freedom. A visualization of the model, as well as an overview of the body's naming conventions can be found in figure 5. An overview of the body's angle conventions can be found in figure 6. Note the absence of a hip abduction/adduction joint. This is because the model's main purpose is to verify the design for jumping in the sagittal (forward-backward and upwards-downwards) plane, and the hip abduction/adduction joint is not necessary for this purpose.

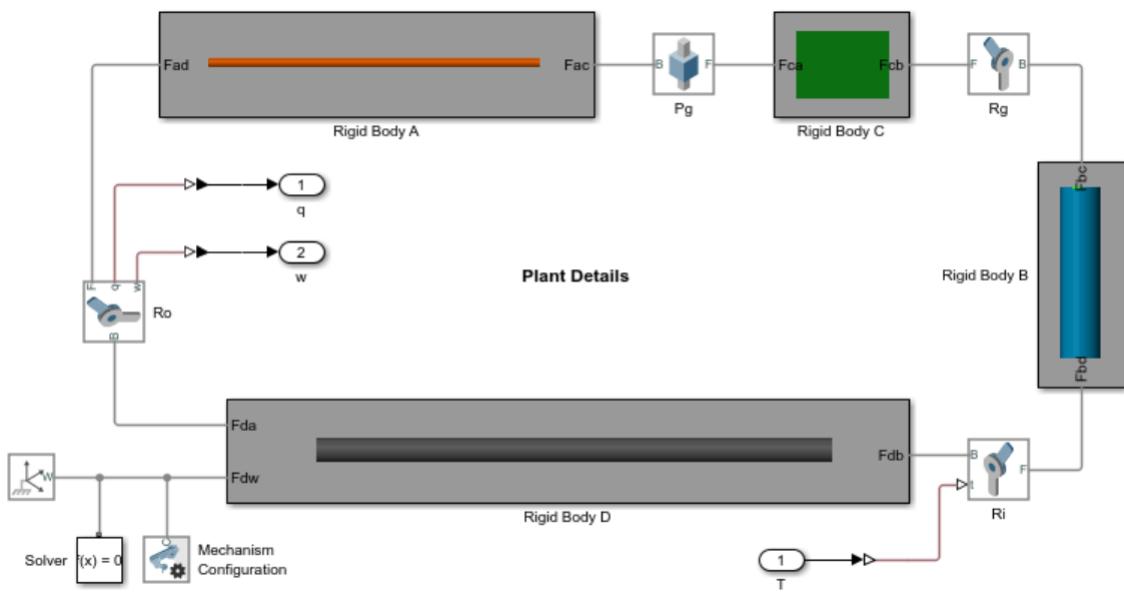


Figure 3: A typical Simscape block diagram.

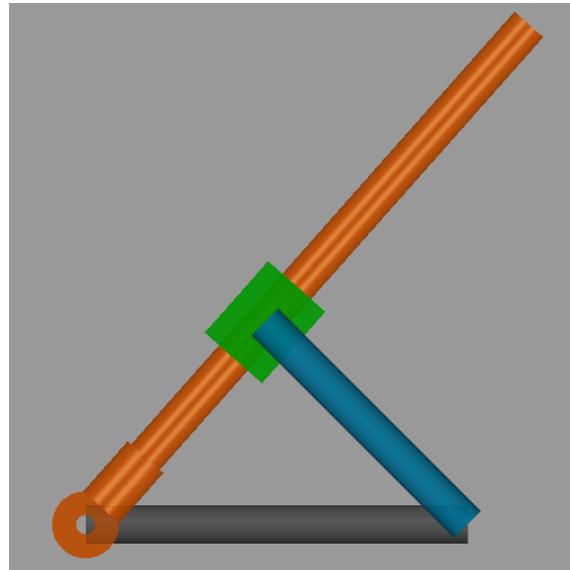


Figure 4: A visualization of the model in figure 3.

Component	Dimensions (cm)	Density (kg/m <sup>3</sup> )	Mass (g)
Thigh	L1 x 0.5 x 0.25	2700	Calculated
Shank	L2 x 0.5 x 0.25	2700	Calculated
Hip Motor (A06CLS V2)	2 x 0.8 x 1.78	Calculated	7g
Knee Motor (A20BHM)	2.3 x 1.2 x 2.75	Calculated	20g
Knee Motor (A80BHP-H)	4.0 x 2.0 x 3.75	Calculated	79g
Thigh Spine	2 x 0.5 x 0.25	2700	Calculated
Shank Spine	1 x 0.5 x 0.25	2700	Calculated

Table 2: Dimensions and density of the rigid bodies in the robot model. L1 and L2 have no set size, but are the variables we intend to optimize over.

Regarding the naming conventions presented in figure 5, note especially the naming of the different legs corresponding to location on the body, namely RH (Right Hind), RF (Right Front), LH (Left Hind), and LF (Left Front). Note also the naming of the joints hip (HIP) and knee (KNEE). If you see the angle conventions in figure 6, you can see that the angles of these joints correspond to the angles  $\theta_1$  and  $\theta_2$  respectively. Note that an orientation of zero degrees for the hip joint corresponds to the leg pointing straight downwards, and an orientation of zero degrees for the knee joint corresponds to the shank pointing in the same direction as the thigh. Note how each joint has its own coordinate system, with rotation being defined positive around the joints z axis. This is required for all Simscape Revolute joints. Note how the joints' z axes correspond to body y axis. This is the case for all the robots legs, both the right and the left side legs.

As can be seen in both figure 5 and figure 6, in addition to the main body and legs colored in grey, the robot model also contains large purple blocks. These blocks represent motor masses, and their mass can be adjusted to represent different motors. In the current figures, the hip motors are A06CLS V2 motors, and the knee motors are A80BHP-H motors.

### 3.3 Rigid Body Masses and Inertias

In Simscape, the mass and inertia properties of a rigid body can be specified by the user or automatically calculated based on the body's geometry and material properties. Hybrid solutions are also possible, where the user specifies some properties and Simscape calculates the rest [8].

In the case of this model, a summary of the origin of the mass and inertia properties of the rigid bodies can be found in table 3. Exceptions are the properties of the main body and the paws, which will be specified in more detail in the two next paragraphs. For the parts whose mass and inertia are calculated by the geometry, a summary can be found in table 2.

The mass properties of the main body are based on the hardware components used by the Eurepus robot constructed by Maurer and El Agroudi [1]. Since early in the design process a rough estimate was needed for the robot body mass, and it seemed likely that the electronic solution, apart from the motors, would be similar to the Eurepus robot, an approximate mass of the main body was calculated based on the mass of some of the Eurepus robot's electronics plus an approximate amount of Nylon body material, four motors, and a random chosen microcontroller (MCU) mass. The formula used for the approximate of the main body mass can be found in equations 34 to 36. The masses corresponding to variables in equations 34 to 36 can be found in table 4. The motor mass chosen in table 4 corresponds to the mass of the AGF-RC A20BHM motor. This motor was chosen because, although the current plan is to use the A06CLS V2 motor, which is lighter, we would rather overestimate the mass of the main body than underestimate it, and a change of motor is one of the more likely changes to the robot design, as well as one of the changes that would increase the mass of the robot the most. TODO: Add method used for the inertia as well.

The geometrical, mass and inertia properties of the paws differ based on which of two scenarios we intend to simulate. The first scenario is the normal jumping scenario, in which the paw mass is simply set to 1000 kg/m<sup>3</sup>, though choosing the actual density of rubber would maybe be more appropriate. The dimensions of the paw are in this scenario simply chosen so that the diameter

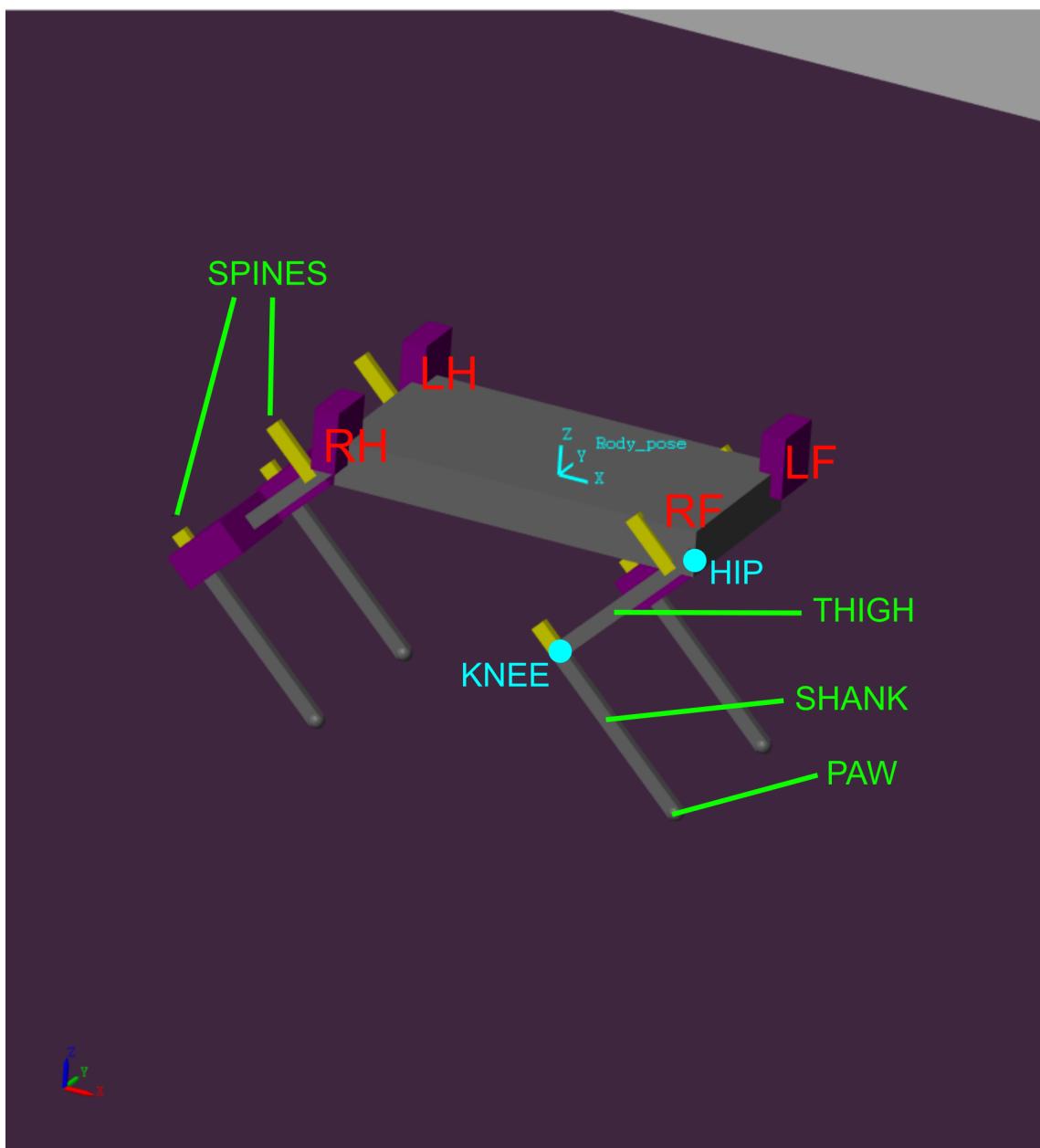


Figure 5: Naming conventions for the parts of the robot.

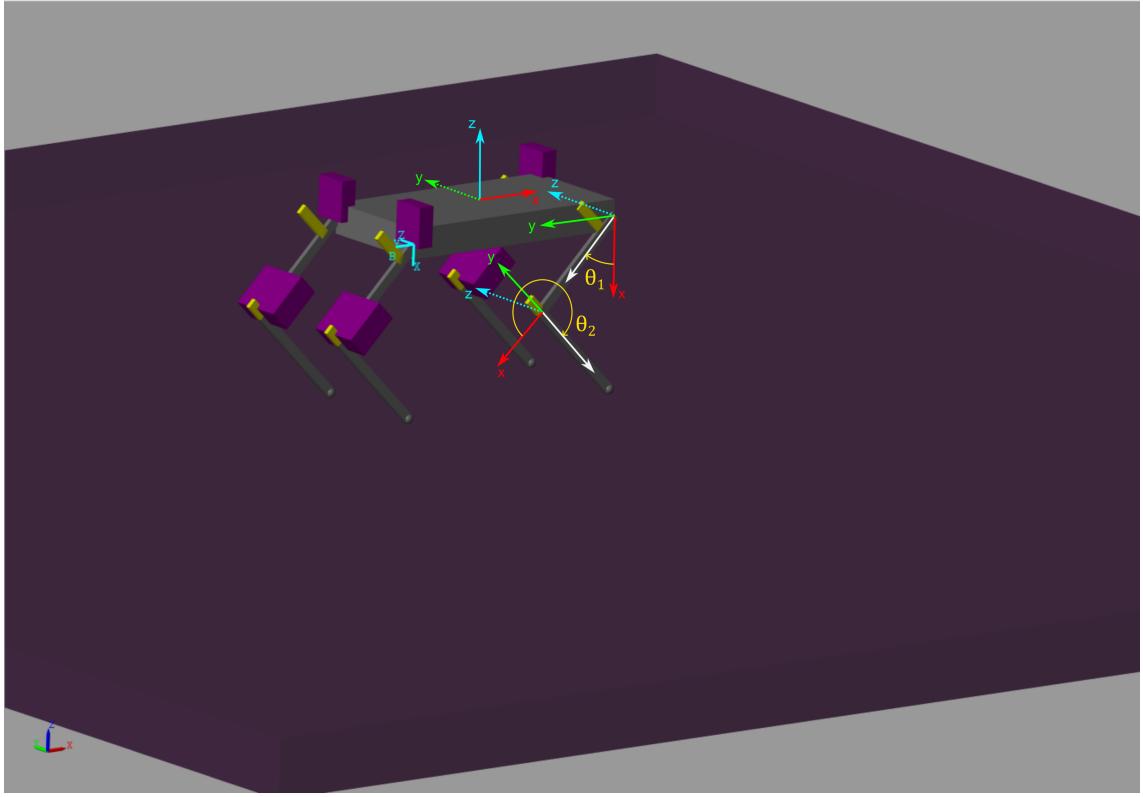


Figure 6: Angle conventions for the robot body.

coincides exactly with  $\max(\text{shank width}, \text{shank height})$ . The second scenario is the one described in section 4.5, where the paw mass and volume is increased to estimate hip flexor/extensor motor requirements.

$$m_{plate} = \rho_{nylon} \cdot V_{plate} = \rho_{nylon} \cdot l_{plate} \cdot w_{plate} \cdot h_{plate} \quad (34)$$

$$m_{eurepus\_electronics} = m_{battery} + m_{I2C} + 12 \cdot m_{ADC} + m_{PWM\_driver} \quad (35)$$

$$m_{main\_body} = m_{plate} + m_{eurepus\_electronics} + m_{MCU} + 4 \cdot m_{motor} \quad (36)$$

Component	Mass	Density ( $\text{kg/m}^3$ )	Inertia	Geometry
Main Body	See section 3.3	See section 3.3	See section 3.3	Rectangular Prism
Thigh	From geometry	2700 (Aluminium 6061)	From geometry	Rectangular Prism
Shank	From Geometry	2700 (Aluminium 6061)	From Geometry	Rectangular Prism
Law	From Geometry	2700 (Aluminium 6061)	From Geometry	Rectangular Prism
Hip Motor	Actual motor mass	From Geometry	From Geometry	Rectangular Prism
Knee Motor	Actual motor mass	From Geometry	From Geometry	Rectangular Prism
Thigh Spine	From Geometry	2700 (Aluminium 6061)	From Geometry	Rectangular Prism
Shank Spine	From Geometry	2700 (Aluminium 6061)	From Geometry	Rectangular Prism
Paw	See section 3.3	See section 3.3	See section 3.3	Sphere

Table 3: Mass and inertia properties of the rigid bodies in the robot model. A list of the Eurepus robot's electronics can be found in [1].

Variable	Description	Value
$\rho_{nylon}$	Density of Nylon	1520 kg/m <sup>3</sup>
$l_{plate}$	Length of the plate	10 cm
$w_{plate}$	Width of the plate	6 cm
$h_{plate}$	Height of the plate	1.67 cm
$m_{battery}$	Mass of the battery	27 g
$m_{motor}$	Mass of one motor	20 g
$m_{I2C}$	Mass of the I2C module	5.1 g
$m_{ADC}$	Mass of one ADC module	2.4 g
$m_{PWM\_driver}$	Mass of the PWM driver	8.5 g
$m_{MCU}$	Approximate mass of some microcontroller	30 g
$m_{main\_body}$	Resultant mass of the main body	332 g

Table 4: Masses and dimensions used in the main body mass calculation.

### 3.4 Elastic Components: Springs

In addition to the model’s many rigid bodies, we also implemented two different forms of spring based passive actuation, namely:

- **A torsional spring** acting in parallel with the knee joint, as illustrated in figure 7. This spring is at zero extension when the knee joint is at zero degrees, and applies a torque that is proportional to the knee joint angle, as covered in section 2.2.
- **An extension spring** acting in parallel with the knee joint, attached to the shank and thigh spine, as illustrated in figure 8. The force generated by the extension spring is proportional to its displacement, as covered in section 2.2. The spring is intended to be unloaded when the knee joint is at zero degrees.

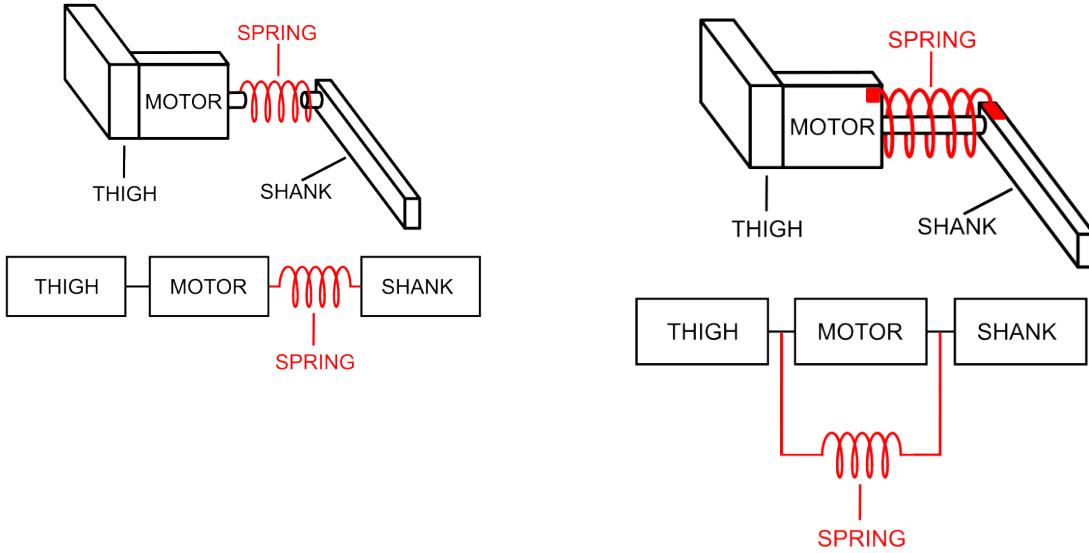


Figure 7: A figure displaying the conceptual difference between a parallel spring (right), as used in this work, and a series elastic spring (left) as used in the ANYmal robot [6].

The torsion spring was implemented in the model using Simscape’s prismatic joint option to add spring stiffness. The extension spring, on the other hand, was connected in parallel to the joint using Simscape’s natural block diagram functionality, as seen in figure 9.

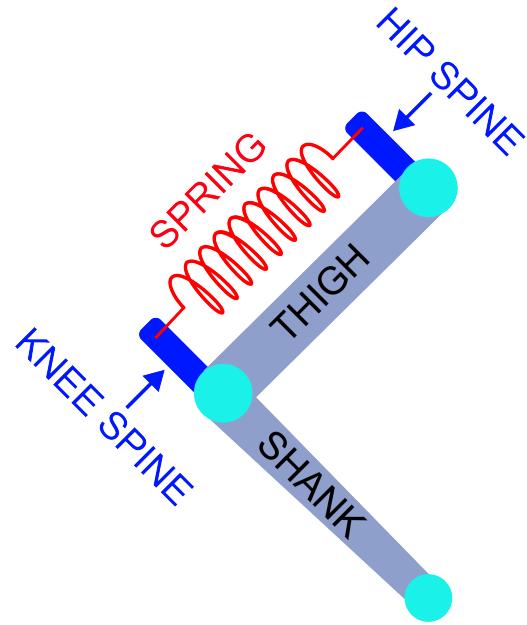


Figure 8: A figure displaying the concept of a parallel extension spring, note that the spring generates torque directly in parallel with the knee joint just like for the parallel torsional spring seen in figure 7. Mathematically, the main difference is the non-linear relationship between knee angle and spring displacement, giving a non-linear torque-angle relationship.

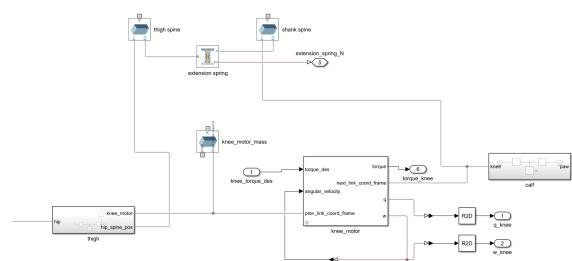


Figure 9: Extension spring implementation in Simscape. As can be seen, the spring is connected between the appropriate points (output frames) of the thigh and shank spine, and in parallel to the knee motor model, which contains the knee joint.

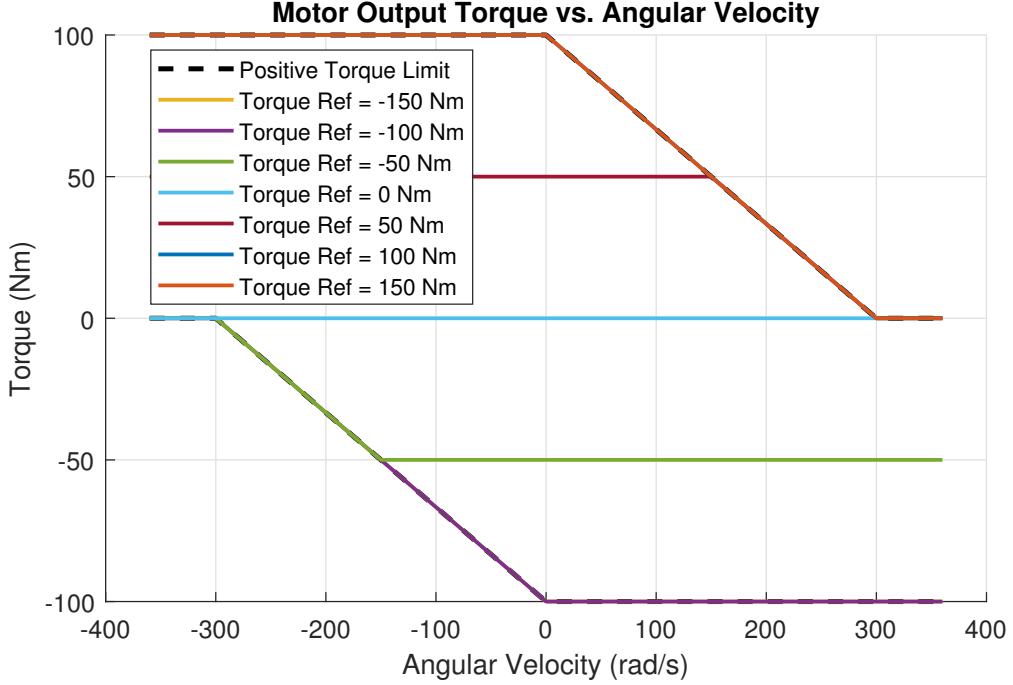


Figure 10: Torque-speed characteristics for a given motor using our implemented motor model.

### 3.5 Motor Modeling

To be able to more accurately judge the jumping capability of different motors, a motor model in the form of a torque speed curve was implemented. The BLDC torque-speed curve presented in figure 1 is characterized by four parameters, namely the stall torque, the operating torque, the rated speed, and the maximum speed. Since motor suppliers contacted (primarily agf-rc and T-motor) were unable to provide most of the desired parameters, we chose a motor model with only two parameters, namely stall torque and maximum speed. The torque-speed model thus became a simple model where torque decreases linearly from stall torque, at speeds smaller than or equal to zero, to zero at speeds greater than or equal to maximum speed. The model is identical for negative velocities and torques, but with opposite signs. An example of the relation between desired torques and achieved torques for a given motor max speed can be seen in figure 10.

### 3.6 Solver selection

As described in section 2.2, the potential energy of a loaded spring can be easily calculated. Similarly, the potential energy due to gravity of a robot at the peak of a jump can also be determined. For the reasons discussed in section 2.5 we chose a stiff numerical solver, initially, we chose the `ode15s` solver. However, it was eventually observed that simulations using `ode15s` occasionally resulted in jump trajectories where the gravitational potential energy at the robot's peak height exceeded the combined potential energy of the four fully loaded springs by a factor of 2, even for jumps with only passive (spring) actuation. To address this inaccuracy, we experimented with different numerical solvers and ultimately selected the `ode23s` solver. This solver provided accurate simulations without artificially generating excess energy, and it performed efficiently.

---

## 4 Robot Design

### 4.1 The Single Vertical Manipulator Jump Model

In the absence of air resistance, which is negligible for a robot like this during a jumping maneuver like this, the factor that determines jumping distance is body velocity at takeoff. To keep things simple, in the following will be given an example where the sole goal is to maximize vertical jump height. Further, only a one leg robot with a 2 DOF leg with equal link lengths is considered.

For a robot such as the one described above, the position of the paw relative to the body can be described by the standard 2 link manipulator equation as seen in equation 37, in accordance with the theory in section 2.3.1. For this simple robot, it is assumed that the optimal jump is one where the body center of mass and paw position move in opposite directions, with velocities strictly along the up/down y axis. This example is illustrated in figure 11. As is obvious from the figure and from the kinematics, for such a jump  $\theta_1 = -\frac{(\theta_2 + \pi)}{2}$ , and  $\dot{\theta}_2 = -2\dot{\theta}_1$ . If this is combined with the jacobian for the end effector, given in equation 4.1, one can plot the vertical velocity of the paw as a function of the knee angle  $\theta_2$ , this is shown in figure 12. This is done using the jacobian as in equation 13.

As can be seen in figure 12, the joint velocity of the leg much more readily translates to body velocity when the knee is crouched. Without giving a detailed derivation, the intuition meant to be gained from figure 12 is that how quickly the leg joints are accelerated can be just as important as the speed it is accelerated to. This has an important interpretation for the choice of motors, in the sense that, if a given motor is unable to accelerate the leg joints quickly enough, there is little sense in looking for a faster motor, unless it is also stronger.

$$\begin{aligned}x_{\text{end}} &= L_1 \cos(\theta_1) + L_2 \cos(\theta_1 + \theta_2) \\y_{\text{end}} &= L_1 \sin(\theta_1) + L_2 \sin(\theta_1 + \theta_2)\end{aligned}\tag{37}$$

$$J = \begin{bmatrix} -L_1 \sin(\theta_1) - L_2 \sin(\theta_1 + \theta_2) & -L_2 \sin(\theta_1 + \theta_2) \\ L_1 \cos(\theta_1) + L_2 \cos(\theta_1 + \theta_2) & L_2 \cos(\theta_1 + \theta_2) \end{bmatrix}$$

### 4.2 Actuation Method Selection: Motors Only

Initially, experiments were done utilizing motors alone. Due to the need for a light and strong motor, initially a series of AGF-RC motors were looked at, due to their high torque to weight ratio. As covered in section 3.5, a motor model consisting of a stall torque and a max velocity, with a linear decrease in max torque between these two, was chosen. For the AGF-RC motors, their provided stall torque and operating speed were used for the motor model parameters. Attempts were made to get more information about the motors' performance characteristics, but the supplier, AGF-RC, was unable to provide more detailed information. While extensive testing was done with various motors, the jumping performance was so far from acceptable that the idea was abandoned, and the full scope of the experiments are considered outside the scope of this report.

More details about the results of the motor only jumps can be found in section 8.4, but a brief overview of the reasoning involved will be given here. The most relevant of the motors chosen for these experiments was the A80BHP-H motor by AGF-RC, which was chosen because it had both the highest stall torque and operating speed of all the AGF-RC motors. Info about the A80BHP-H motor can be found in appendix D. Attempts were made to identify motors in the same weight range with similar stall torques and operating speeds, but none were found. The experiments using the A80BHP-H were therefore, to a certain degree, considered an "optimistic estimate". Since no satisfactory results were achieved with the A80BHP-H motor, as discussed in section 8.4, it was decided to explore alternatives to motor-only actuation.

The tests with the A80BHP-H motor were done by providing the knee motors with reference torques equal to their max torque, with the motor model limiting the resultant torque to more realistic

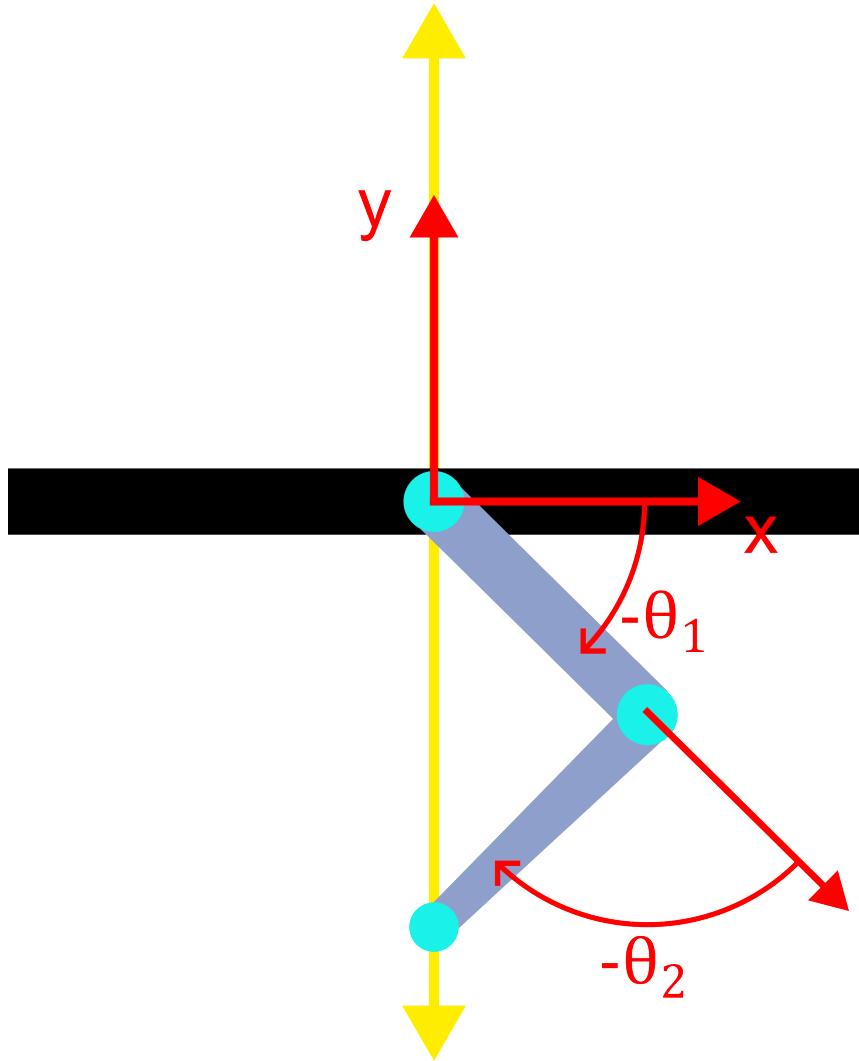


Figure 11: The manipulator corresponding to a vertical one leg jump.

values. Meanwhile, the hip joints were actuated according to the control law seen in equation 41 to limit slipping. This control law is based on equation 15, as derived in section 2.3.3, combined with the theory on friction presented in section 2.4. Equation 15 is derived with an assumption of zero velocity/equilibria (TODO), an assumption that is obviously not valid for this dynamic jumping scenario. Despite this, very little slipping is observed in practice.

$$N = \text{Normal force} \quad (38)$$

$$\mu = \text{friction coefficient} = 0.8 \quad (39)$$

$$\tau_{\text{friction cone limit}} = J^T \begin{bmatrix} N\mu \\ 0 \end{bmatrix} \quad (40)$$

$$\max(|\tau_{knee}|) = \tau_{\text{friction cone limit}} \quad (41)$$

Using this control law, jumping for variously dimensioned robots were attempted. As covered in section 8.4, none of them were satisfactory. Tests were eventually terminated, as no leg and body link length configuration was capable of jumping satisfactorily. By satisfactorily, it is meant that the robot center of mass cleared the ground by a distance more than twice that of the leg length, i.e., if the legs are 20cm long, the center of mass should clear the ground by at least 40cm.

For comparisons between this motor-only jumping and later spring+motor jumping, it is worth

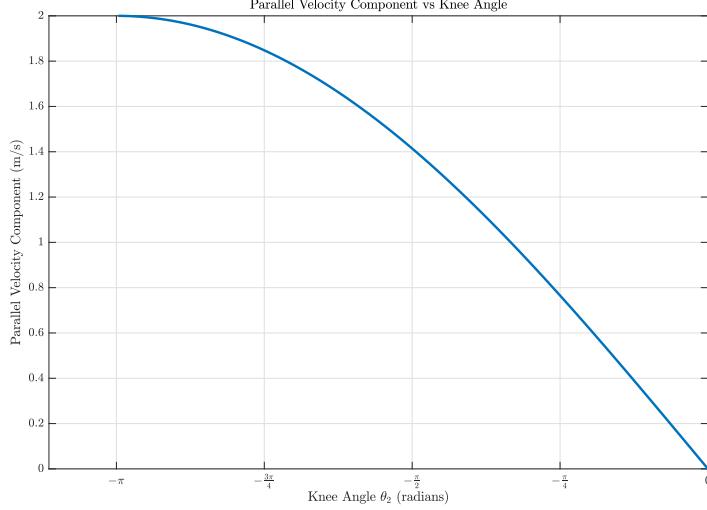


Figure 12: Vertical Paw velocity as a function of knee angle.

mentioning that the spring-motor jumping is done not with the A80BHP-H motors, but with weaker and slower motors, as well as with more realistic motor friction models. The reason the A80BHP-H motors were not used for the spring-motor jumping simulations, is that the motor supplier's website (AGF-RC) stated an operating travel range of 90 degrees. This was assumed to be correct, as other AGF-RC motors with stated operating ranges of 180 degrees have had actual mechanical limits of 220 degrees, so such a limitation was not considered out of the ordinary. For that reason, the motors initially purchased were not the A80BHP-H motors, and they are therefore not used for the rest of the experiments in this report. Later consultations with the supplier's website revealed conflicting information regarding the operating range of the A80BHP-H motor, and direct contact with the supplier revealed that the motor is in fact not limited to 90 degrees of travel. For this reason, potential use of the A80BHP-H motor for the spring-motor jumping is discussed in the future work section, section 10.

### 4.3 Actuation Method Selection: Motors and Torsional Spring

The next actuation method considered was to use a combination of motors and torsional springs, as this is the design we ended up choosing, a CAD model of this design can be seen in figure 19. With this design, the knee motors are used to compress the torsional springs, once the knee joint reaches a desired angle, ie. has stored the desired amount of energy, the motors are turned off, and the springs accelerate the legs' joints quickly enough for the robot to take off. Results from simulations are presented in section 8.2, but in short, jumping performance is superior to that of the motor-only jumps. For the spring-motor jumps, a friction model as derived in section 5 was used. Again, note the much weaker motor used in

### 4.4 Actuation Method Selection: Motors and Extension Spring

In addition to the torsional spring design, attempts were made to design a leg utilizing an extension spring. The resulting design is shown in figure 24. Although experiments akin to the ones done for torsional springs were done with comparable results, the extension spring design was ultimately abandoned. The reason for this was pure geometrical constraints, which are discussed in section 7.2. Because the extension spring design was abandoned, the details surrounding the extension spring simulations and simulation results are considered outside the scope of this report.

---

## 4.5 Hip Motor Strength Requirements

While the main purpose of the designed robot is to be proficient at jumping, the inclusion of torsional knee springs as a major part of the actuation method, and the subsequent lessened importance of the hip motors for jumping, complicated the choice of hip motors. For a conservative bound on the torque and speed required for in air attitude stabilization [1] uses a heuristic of three 90-degree back-and-forth lateral swings per second as a benchmark. Although the purpose of our robot is not to do in-air attitude stabilization, this heuristic was partially utilized for verification that the hip motors were strong enough. More specifically, the robot body was fixed in space, and the robot's paws were set to be 1cm diameter spheres of iron (ie. with a density of  $7800 \frac{kg}{cm^3}$ , for a total mass of  $\approx 32$  g). The hip motor was then commanded to follow a trajectory according to the heuristic specified in [1]. The results of this simulation are presented in section 8.3. While the paw masses used in [1] were 80g, the Eurepus robot's main body mass was also closer to 800g, compared to our body mass of approximately 300g. Our experimental paw masses of 32g were therefore considered appropriate, considering attitude stabilization is not strictly speaking a goal for the robot.

## 5 Motor Friction Estimation

This section details the estimation of parameters for the motor friction models of the hip and knee motors. The motors serve as pivot points for pendulums consisting of aluminum rods attached to the motor shafts with ballast masses. The pendulums are released from a horizontal position, and their angular velocity and acceleration are measured. These measurements facilitate the estimation of motor friction coefficients using linear regression. The experimental setup for both motors is depicted in Figure 13.

We adopt a linear friction model as described in Section ??, incorporating both viscous and Coulomb friction, to accurately fit the observed data.

### 5.1 Pendulum Modeling

The pendulum used in the motor friction tests consists of an aluminum rod of length  $l_{\text{arm}} = 0.21$  meters and a ballast mass  $m_{\text{ballast}} = 0.301$  kg with radius  $r_{\text{ballast}} = 0.03$  meters attached at a distance  $d_{\text{ballast}} = 0.20$  meters from the pivot. The total mass of the arm is  $m_{\text{arm}} = 0.034$  kg. The pendulum is modeled as a rigid body rotating about the motor shaft with a total moment of inertia  $I$  composed of the arm inertia  $I_{\text{arm}}$  and ballast inertia  $I_{\text{ballast}}$ . The arm is approximated as a thin rod, and the ballast as a disk, both with uniform density. Their inertias are calculated using the parallel axis theorem:

$$I_{\text{arm}} = \frac{1}{12}m_{\text{arm}}l_{\text{arm}}^2 + m_{\text{arm}}\left(\frac{l_{\text{arm}}}{2}\right)^2$$

$$I_{\text{ballast}} = \frac{1}{2}m_{\text{ballast}}r_{\text{ballast}}^2 + m_{\text{ballast}}d_{\text{ballast}}^2$$

$$I = I_{\text{arm}} + I_{\text{ballast}}$$

The torque due to gravity consists of contributions from both the arm and ballast mass:

$$\tau_{\text{gravity,arm}} = m_{\text{arm}} \frac{l_{\text{arm}}}{2} g \sin(\theta)$$

$$\tau_{\text{gravity,ballast}} = m_{\text{ballast}} d_{\text{ballast}} g \sin(\theta)$$

---


$$\tau_{\text{gravity}} = \tau_{\text{gravity,arm}} + \tau_{\text{gravity,ballast}}$$

The equations of motion, accounting for both viscous and Coulomb friction, are:

$$I\ddot{\theta} + b_v\dot{\theta} + b_c \text{sign}(\dot{\theta}) + \tau_{\text{gravity}} = 0$$

where:

- $\theta$  is the angular displacement (positive counterclockwise, zero at vertical down position)
- $\dot{\theta}$  and  $\ddot{\theta}$  are the angular velocity and acceleration, respectively
- $b_v$  is the viscous damping coefficient
- $b_c$  is the Coulomb damping coefficient
- $g = 9.81 \text{ m/s}^2$  is the acceleration due to gravity

## 5.2 Data Acquisition

Pendulum angles  $\theta$  were sampled by manually annotating video frames at 60 Hz using the Tracker program [2].

The angular velocity  $\dot{\theta}$  and acceleration  $\ddot{\theta}$  are computed using centered finite differences:

$$\dot{\theta}_i = \frac{\theta_{i+1} - \theta_{i-1}}{2\Delta t}$$

$$\ddot{\theta}_i = \frac{\theta_{i+1} - 2\theta_i + \theta_{i-1}}{(\Delta t)^2}$$

where  $\Delta t$  is the time step between measurements.

The angular position  $\theta$  was smoothed using a moving average filter with a window size of 5 before computing  $\dot{\theta}$ . Both  $\dot{\theta}$  and  $\ddot{\theta}$  were smoothed using a moving average filter with a window size of 3.

## 5.3 Linear Regression Derivation

Rearranging the equations of motion for linear regression:

$$I\ddot{\theta} + \tau_{\text{gravity}} = -b_v\dot{\theta} - b_c \text{sign}(\dot{\theta})$$

This can be expressed in matrix form:

$$Y = X\beta$$

where:

- $Y = -I\ddot{\theta} - \tau_{\text{gravity}}$ ,
- $X = [\dot{\theta} \quad \text{sign}(\dot{\theta})]$ ,
- $\beta = \begin{bmatrix} b_v \\ b_c \end{bmatrix}$ .



Figure 13: Experimental setup for motor friction estimation. An aluminum rod with ballast mass attached acts as a pendulum, with the motor shaft as the pivot point.

The linear least squares solution for  $\beta$  is:

$$\beta = (X^T X)^{-1} X^T Y$$

This yields both the viscous damping coefficient  $b_v$  and the Coulomb damping coefficient  $b_c$ .

To validate the model we forward simulate the pendulum motion using the estimated parameters with a runge-kutta solver and compare the results with the measured data. This and the results are shown in section 8.1.

## 6 Link Length Optimization

This section details the optimization of the robot's link lengths L1 and L2 to maximize jumping performance. The optimization ensures the robot can achieve sufficient jumping performance to clearly demonstrate future DRL control policies. A grid search approach using the Simscape robot model from section 3 systematically explores different link length configurations.

### 6.1 Impact of link lengths

Due to the lack of feedback control during the jumping maneuver, the jumping performance is determined fully by the link lengths and initial pose. By initial pose we mean the angles theta1 and theta2 of the knee and hip joints as the knee motor turns off and lets the loaded knee spring actuate the leg. Link lengths constrain the possible initial poses for any given jumping angle, while the initial pose determines spring compression through the knee angle theta2 and thus available potential spring energy. Additionally, link lengths affect the center of mass trajectory during jumps, influencing how gravitational and ground contact forces impact the robot's movement.

---

## 6.2 Problem simplification

First, we define the jump angle  $\theta_J$  as the angle of the velocity vector of the center of mass of the robot body at the moment the robot leaves the ground. While the robot should be able to jump both vertically and at angles sufficient to overcome obstacles, optimizing for angled jumps presents challenges. We want to directly compare the jumping performance of different link lengths, but it is not obvious how to place the initial pose of the robot to achieve a  $\theta_J$  across different link lengths.

To simplify the optimization, only the vertical jumping performance was considered, so that different link lengths can be compared directly. Vertical jumps are achieved across link lengths by flipping the front legs such that the legs are symmetric about the vertical axis, as shown in figure 17. This flipping transforms the asymmetric leg configuration in figure ?? where the front legs point forward into a symmetric configuration where both legs point backward. In this configuration, the movement of the legs during the jump is symmetric and the robot center of mass remains in the horizontal center of the robot, such that any horizontal component of the jump is canceled out. In practice, the robot will use the asymmetric configuration. The symmetric configuration simply provides an approximation for the jumping performance of a given link length configuration.

The metric used to evaluate the jumping performance is the maximum height reached by the center of the robot torso, minus the maximum standing height reached by the robot torso when the legs are fully extended vertically downwards and the paws are in contact with the ground.

The asymmetric leg configuration can also achieve vertical jumps by adjusting the angle between the hip-to-paw vector and vertical, as shown in figure ?? . However, finding or calculating the optimal offset for arbitrary link lengths is complex. We therefore simplify by placing paws directly beneath hips, except when  $L_2 > L_1$ . While this produces less realistic jump heights, it greatly simplifies the optimization.

For  $L_2 > L_1$  configurations, placing paws directly under hips results in near-vertical legs that slip during jumps. Slipping does not occur to the same extent for asymmetric configurations with the same link lengths where the hip-to-paw angle is adjusted to give a vertical jump. In order to make the symmetric configuration a better approximation of the asymmetric case, we wish to avoid the slipping behavior. We do so by adjusting the hip-to-paw angle such that paws are slightly inward towards the body by a constant offset of 0.3 radians. This increases the vertical component of the paw force and thus the friction force (figure 16). This exact offset was chosen because experimentation showed that this makes asymmetric configuration jumps generally more vertical across a range of link lengths. This adjusted paw placement better approximates the practical asymmetric case that requires similar adjustment for vertical jumps, although the exact angle offset will vary depending on the link lengths. To handle the increased opposing horizontal forces between paws in the symmetric configuration, we double the friction coefficients from 1.0/0.8 to 2.0/1.6 (static/kinetic), which further reduces slipping.

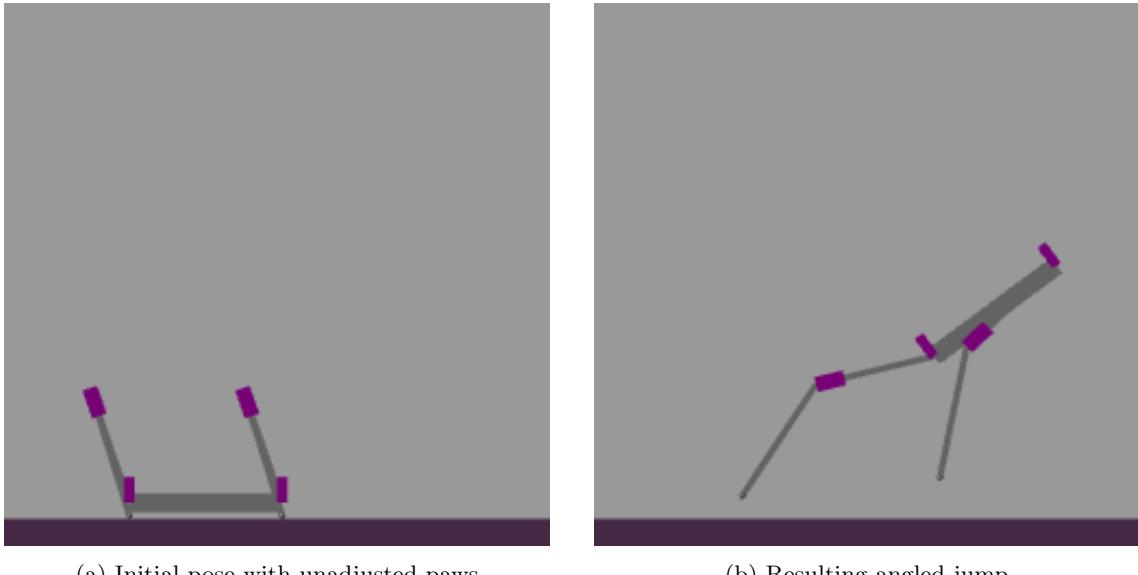
## 6.3 Initial Pose Calculation

For each set of link lengths, an initial pose must be calculated that satisfies several constraints:

- The paws must maintain ground contact
- Knee angle theta2 must be maximized to store maximum spring potential energy
- Knees cannot penetrate the ground
- Both knees bend outward rather than inward

The pose calculation considers three cases based on link length ratios:

1.  $L_1 = L_2$  (equal lengths)



(a) Initial pose with unadjusted paws

(b) Resulting angled jump

Figure 14: Unadjusted paw placement results in angled jump

2.  $L_2 < L_1$  (longer lower link)
3.  $L_1 > L_2$  (longer upper link)

For all cases, we calculate the distance  $d$  from hip joint to paw. For cases  $L_1 = L_2$  and  $L_1 > L_2$ , where paw is placed directly under hip,  $d$  is vertical from the hip. For case  $L_2 > L_1$ , where paw is adjusted inwards towards the body,  $d$  is along the adjusted hip-to-paw vector.

$d$  is minimized to maximize  $\theta_2$  and thus spring potential energy while satisfying the pose constraints. Inverse kinematics then determines  $\theta_1$  and  $\theta_2$ , selecting the solution where knees bend outward.  $d$  is calculated as follows:

- For  $L_1 = L_2$ :  $d = \epsilon$ , where  $\epsilon = 1\text{mm}$  ensures unique inverse kinematics solutions
- For  $L_2 > L_1$ :  $d = L_2 - L_1 + \epsilon$ , where  $\epsilon = 1\text{mm}$  ensures the paw position is reachable given the numerical precision of the inverse kinematics solver
- For  $L_1 > L_2$ :  $d = \sqrt{L_1^2 - L_2^2}$ , derived when  $L_2$  is horizontal (maximizing spring load) and forms a right triangle with  $L_1$  and the hip-to-paw vector

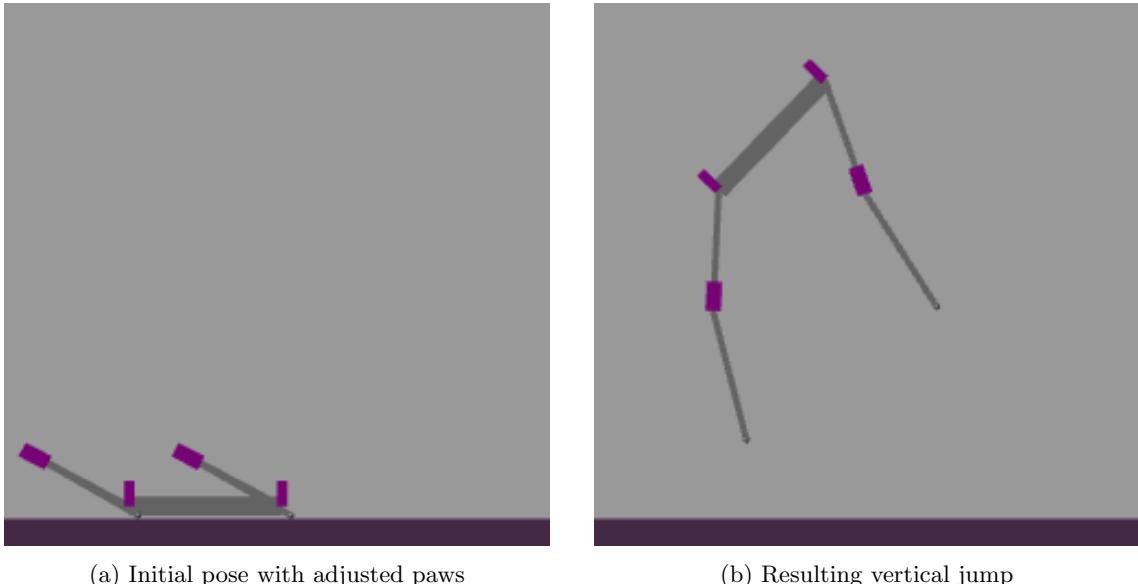
## 6.4 Grid Search

The grid search varies two parameters:  $L_2/L_1$  ratio, and total leg length  $L_1+L_2$ , focusing around  $L_2/L_1 \approx 1$  where preliminary tests indicated generally better performance. SimScape automatically updates the robot mass for each parameter set. Tests were run in both Earth ( $9.81\text{ m/s}^2$ ) and Mars ( $3.72\text{ m/s}^2$ ) gravity, with results in section 8.2.

# 7 Robot Hardware

## 7.1 Torsional Spring Leg

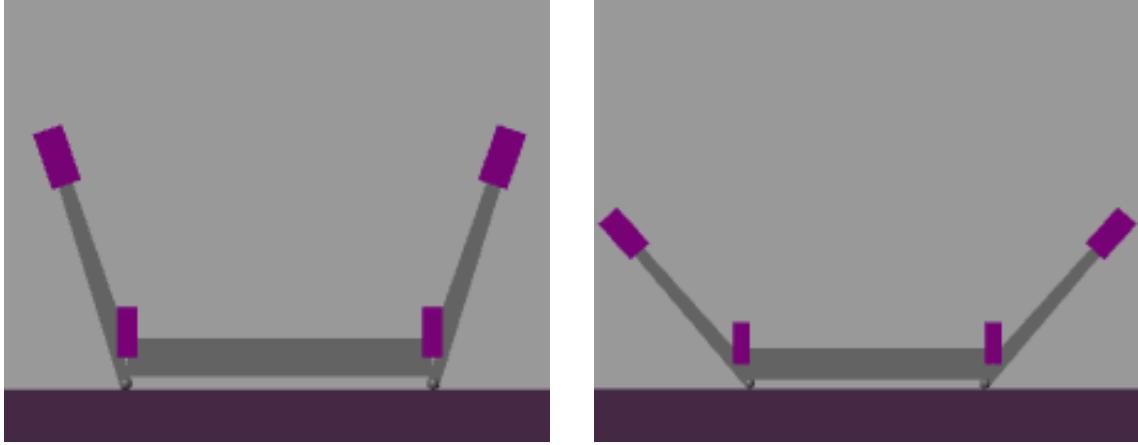
Figure 19 shows the CAD model for the torsional spring leg design, including both knee and hip extension/flexion motor, as well as the hip adduction/abduction motor. The CAD model was



(a) Initial pose with adjusted paws

(b) Resulting vertical jump

Figure 15: Adjusted paw placement enables vertical jump



(a) Paws directly under hips leading to near-vertical legs

(b) Paws translated inward to make legs more horizontal, which reduces slipping

Figure 16: Paw placement adjustment for  $L_2 > L_1$  configurations

created using the CAD software Solidworks. An annotated exploded view of the leg can be seen in figure 20, while an annotated exploded view for the hip motor housing can be seen in figure 21.

Figure 22 shows the components that are currently planned to manufacture in house. The axle that will be threaded and screwed directly into the motor shaft, and lead directly into a ball bearing, is emphasized in red. The leg has been designed to make it easily manufacturable in aluminum. Aluminum was chosen as a potential material due to its high strength-to-weight ratio, and the fact that it is easy to machine. Although many easily 3D-printable plastics are generally lighter, they are not as strong as aluminum, in fact aluminum has a much higher weight to yield ratio than 3D printable plastics, and thus, barring complications resulting from manufacture, a leg made in aluminum should be lighter than one made of plastic.

A 3D printed version of the leg thigh together with the purchased spring can be seen in figure 23. While the full leg has not been assembled, initial physical load tests seem to indicate that the thigh is able to withstand the load of the spring. This could indicate that the leg will be 3D printed rather than made in aluminum. The axle that will be screwed into the motor shaft, however, will still be made in aluminum.

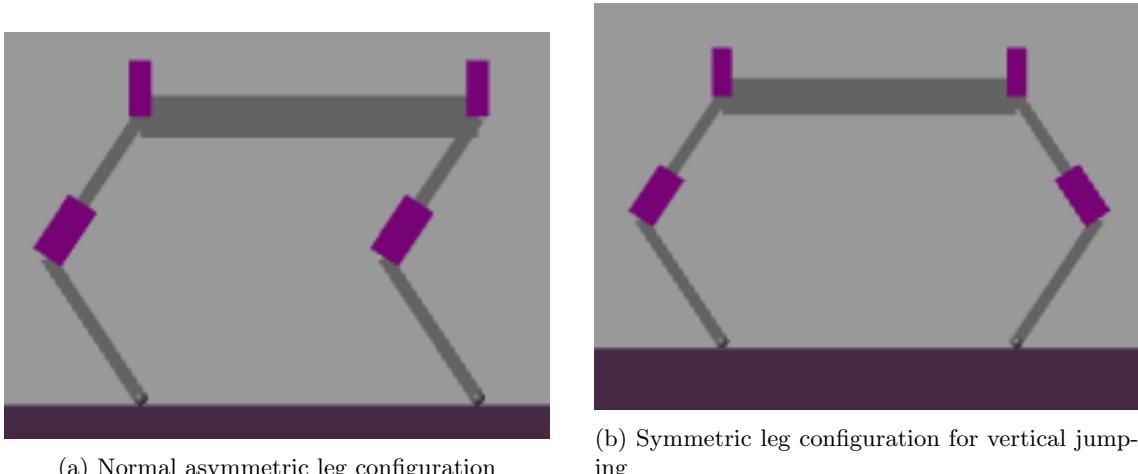


Figure 17: Comparison of normal and symmetric leg configurations

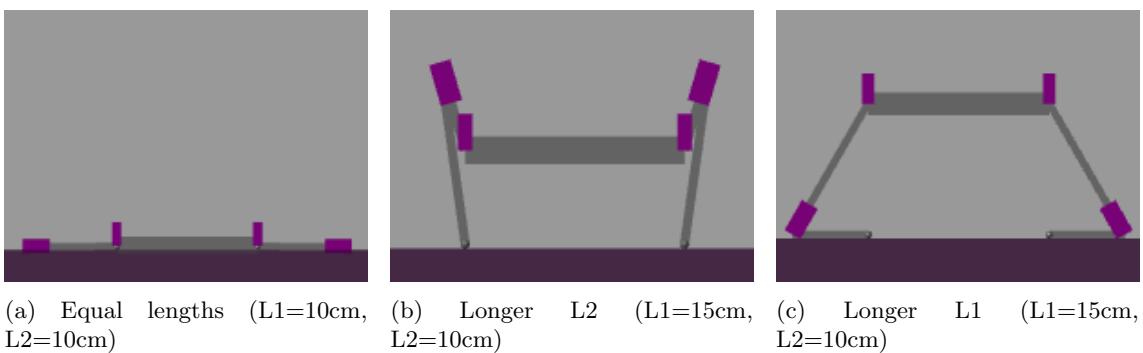


Figure 18: Initial poses for different link length ratios

## 7.2 Extension Spring Leg design

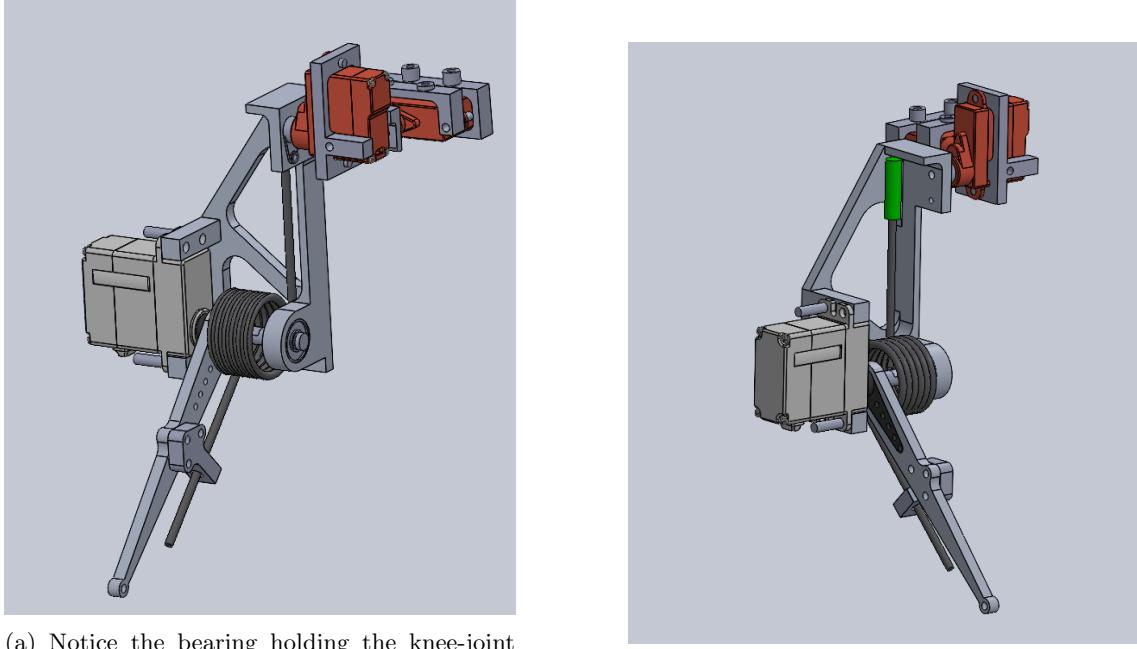
The extension spring leg design is shown in figure 24. As can be seen in the figure, the current design is such that the extension spring will collide with the robot shank as the knee angle approaches fully coiled (ie. -180 degrees). Despite efforts, no solutions were found for this problem, and this design direction was therefore abandoned. Among the suggested solutions was moving the shank-end attachment point of the spring inwards towards the body, so the extension spring could lie in parallel next to the shank when the leg is fully coiled. This would however introduce a significant moment arm acting directly on the motor shaft, and this design was therefore abandoned in favor of the torsional spring design.

## 7.3 Motor Selection

As discussed in section 4, the choice fell on AGF-RC motors due to their high torque to weight ratio, as well as our inability to find similarly fast motors of similar strength.

Our specific choice of motors can be found in table 5. Info about the specific motors can be found in appendices A to B.

The reason these motors in particular were chosen is the fact that no motors were found in a similar weight class that could provide the same torque and speed. For example, if considering AGF-RC motors, the next motor, strength-wise, after the A20BHM motor, is the A35CHM motor, info about which can be found in appendix C. Despite being significantly heavier, the A35CHM motor is only marginally stronger than the A20BHM motor. The reasoning behind the choice of the A06CLS V2 is detailed in section 4.5.



(a) Notice the bearing holding the knee-joint shaft in place. This is important to reduce the load the motor shaft suffers in directions other than the load direction.

(b) As can be seen, there is a green plastic (PLA) holster where the spring is in contact with the leg, this is to reduce friction.

Figure 19: An overview of the leg CAD model with a torsional spring.

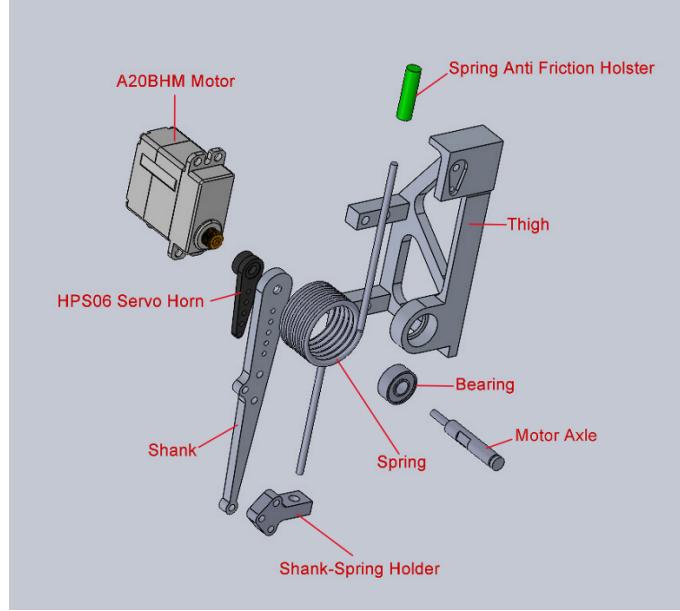


Figure 20: Annotated exploded view of the CAD leg design.

Corresponding Joint	Motor Name
Knee flexion/extension	A20BHM
Hip flexion/extension	A06CLS V2
Hip adduction/abduction	A06CLS V2

Table 5: Selected Motors

It is worth mentioning that the mass of the thigh shown in figure 22 is considerably higher than a similarly long thigh in the simulation, as described in section 3. The main difference is that

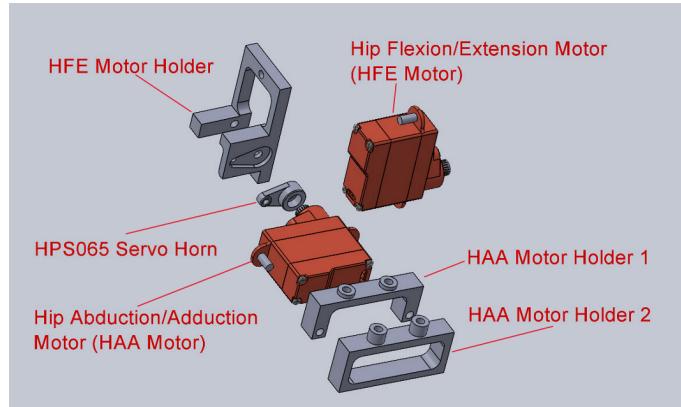


Figure 21: Exploded view of the motor housings for the hip joint.

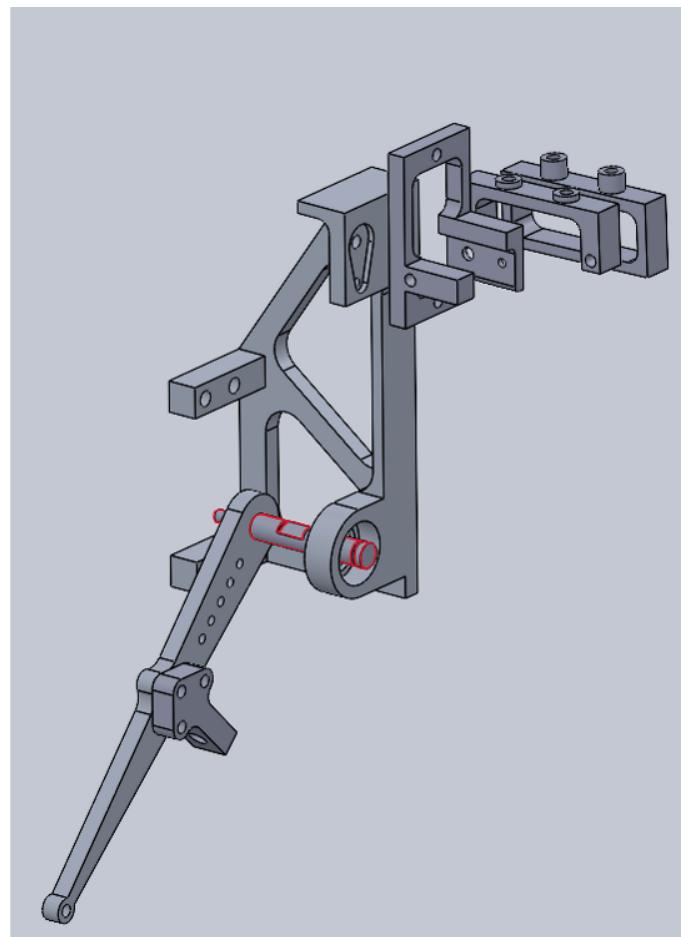


Figure 22: An overview of the CAD model of the leg design, but showing only the components that will be manufactured inhouse. Axe that will be threaded and screwed directly into the motor shaft, and lead directly into a ball bearing, is emphasized in red.

the thigh in the simulations is a long and thin rectangular prism, whereas the actual leg design is currently quite wide and bulky, so as to accommodate the motor and spring while maintaining structural integrity. This is a discrepancy between the simulations and the actual design, however, since the leg masses are so small compared to the mass of the motors and the main body, the discrepancy is not considered significant. For reference, the mass of the thigh for various material densities compared to the leg design used in simulation is shown in table 6.



Figure 23: A 3D printed version of the leg thigh with the torsional spring.

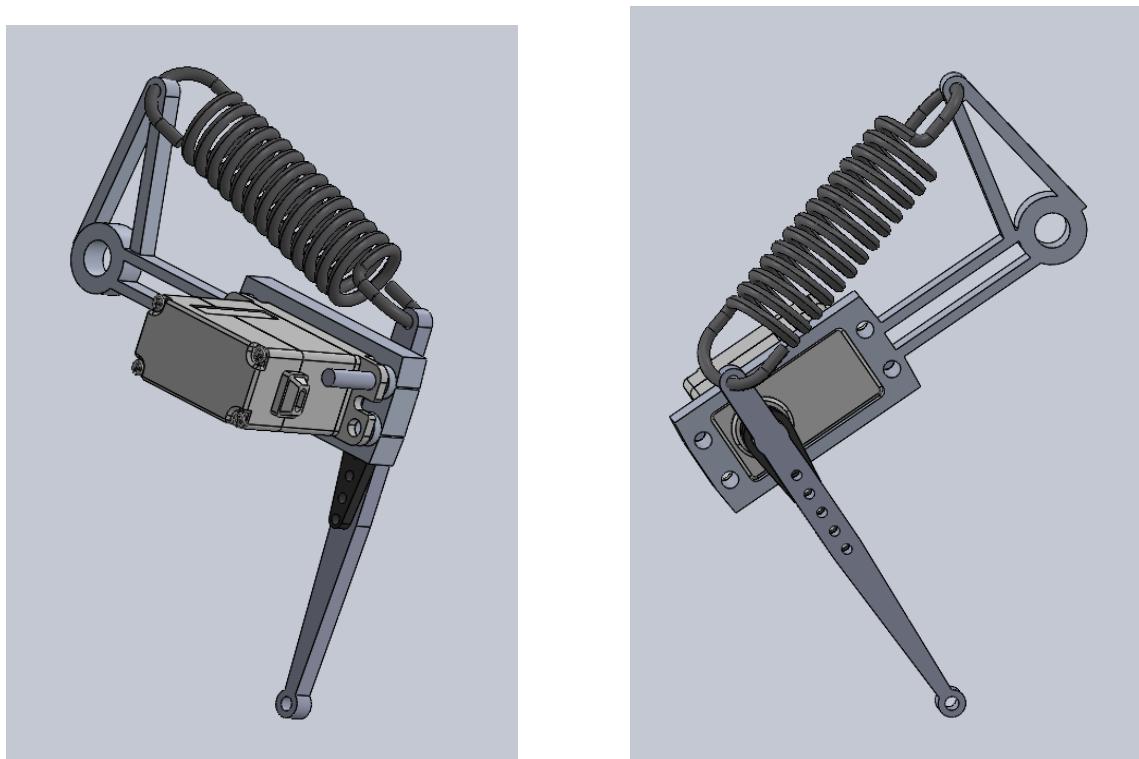


Figure 24: Comparison of extension spring configurations: outside (left) and inside (right) the leg.

Material Density	Mass of thigh in simulation	Mass of actual thigh design
1200 kg/m <sup>3</sup> (Tough PLA, see appendix E)	N/A	6.45
2700 kg/m <sup>3</sup> (Aluminum 6061)	2.26 g	14.27 g

Table 6: Mass of thigh for various material densities compared to the leg design used in simulation. For the real thigh, mass properties were calculated using Solidworks "Mass Properties" tool.

#### 7.4 Other Components, Spring, etc: TODO

Bearing, spring, paw must also be mentioned. Servo horns also.

## 8 Results

### 8.1 Motor Friction Estimation

Parameter	Hip Motor	Knee Motor
Viscous friction coefficient (without Coulomb) [Nm · s/rad]	0.003552	0.011612
Viscous friction coefficient (with Coulomb) [Nm · s/rad]	0.001703	0.006124
Coulomb friction [Nm]	0.010963	0.024127
MSE without Coulomb	0.003203	0.005989
MSE with Coulomb	0.000953	0.001967

Table 7: Friction model parameters and mean squared errors (MSE) for both motors, comparing models with and without Coulomb friction.

Mean squared errors (MSE) are calculated between the simulated theta values with estimated friction parameters and the measured theta values, as shown in figure ?? and ??.

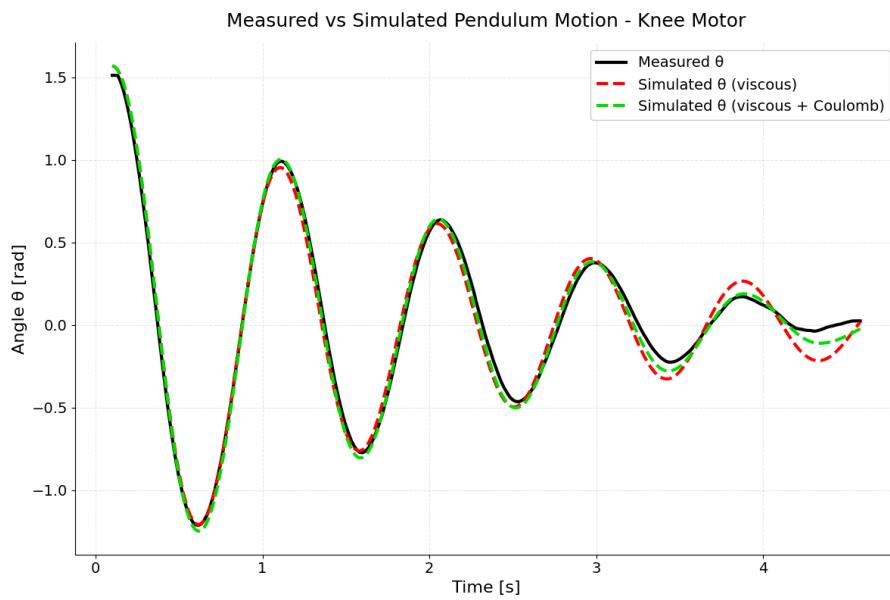


Figure 25: Linear regression fit of the pendulum data for the knee motor. Derived theta dot dot is the double derivative of the pendulum angle.

### 8.2 Link Length Optimization

The grid search results for both Earth and Mars gravity are shown in figures 27 and 28. The search explored link length ratios from 0.8 to 1.6 and total lengths from 8cm to 36cm, in increments of 0.1 and 0.5 cm respectively.

Best performing jumps are described in table 8.

Gravity	Jump Height (cm)	Ratio	Total Length (cm)	L1 (cm)	L2 (cm)
Earth	45.39	1.0	20	10	10
Mars	149.79	1.0	28	14	14

Table 8: Best performing link length configurations and their corresponding jump heights for Earth and Mars gravity.

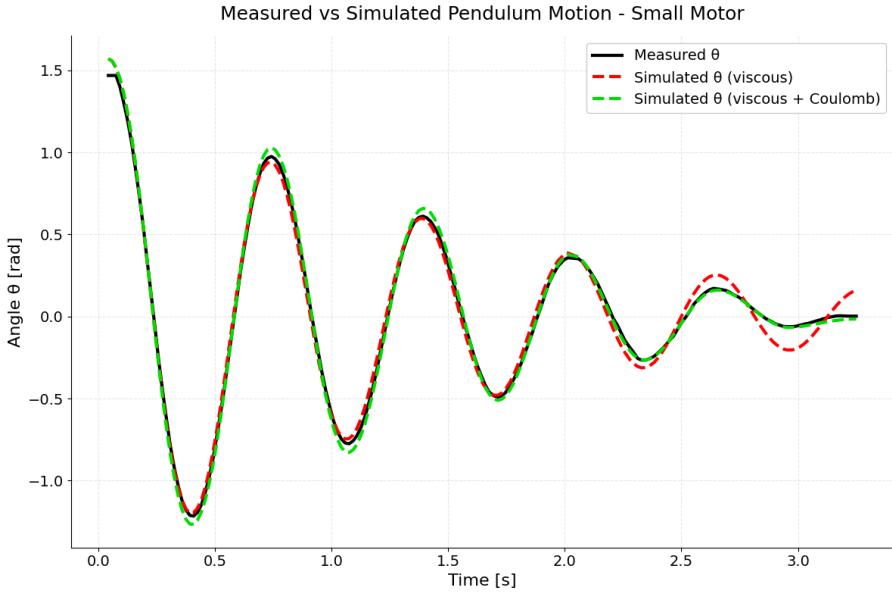


Figure 26: Linear regression fit of the pendulum data for the hip motor.

TODO: Correct optima The optimum total length is 8cm longer under Mars gravity. In both Mars and Earth gravity, the optimal link length ratio is 1.0 across all total lengths, with steep drops in jump height for ratios under 1.0 and less steep drops for ratios over 1.0.

### 8.3 Hip Motor Dimensioning Test

As can be seen in figure 29, the hip motors follow the angle reference well, achieving three back and forth swings of 90 degrees over a one second period. The torque output of the motors during this maneuver can be seen in figure 30. In other words, the hip motors satisfy the heuristic as presented in [1].

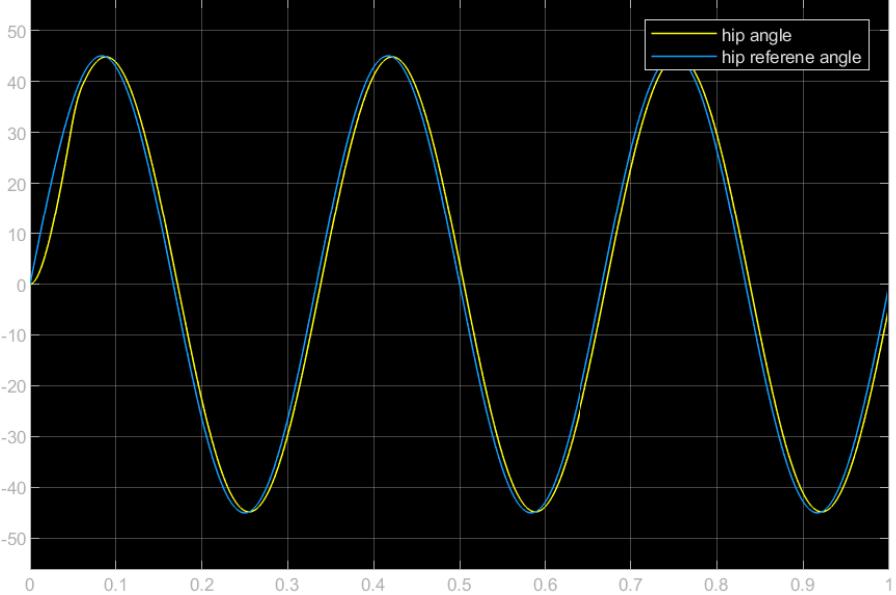


Figure 29: Commanded and actual hip joint angle achieved during the hip motor strength test simulation.

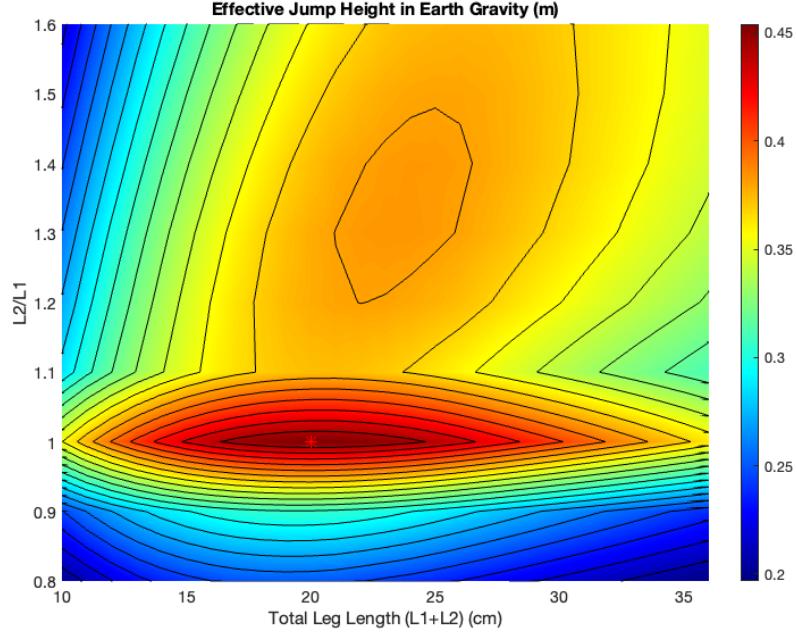


Figure 27: Grid search results showing jump height performance across different link length configurations under Earth gravity.

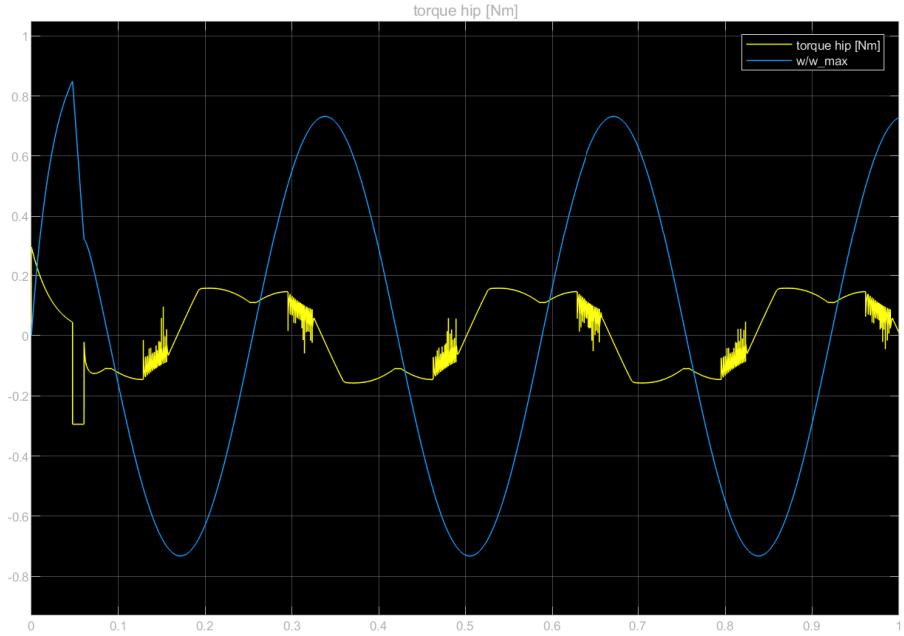


Figure 30: Torque output of the hip motors during the hip motor strength test simulation.

#### 8.4 Motor Only Jumping Results

As described in section 4.2, motor-only simulations were done using the assumed characteristics of the A80BHP-H motor. Because the results were so far from acceptable, only the results of one experiment are presented here, but simulations with shorter leg and body lengths were performed. Figure 31 shows the knee joint actuation torque, knee joint velocity and knee joint angle during takeoff for the A80BHP-H motor. The robot is not able to reach a high enough speed to achieve

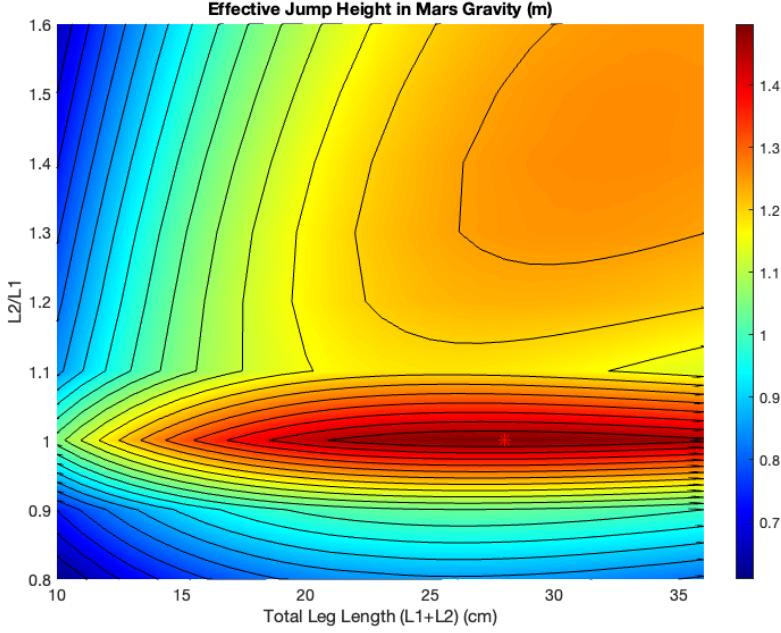


Figure 28: Grid search results showing jump height performance across different link length configurations under Mars gravity.

a good jump, and the peak body center of mass height reached is 70cm, despite a total leg length of 40cm.

## 9 Discussion

### 9.1 Link Length Optimization Trade-offs

The optimal link lengths found for both Earth and Mars gravity present practical challenges for the mechanical design. The longer lengths would require either elongating the robot body to prevent leg collisions, or translating the hip joints outward. Body elongation would increase overall weight, while outward hip translation raises collision risks during aerial stabilization. Additionally, longer legs increase the inertia that motors must overcome during aerial maneuvers, potentially slowing stabilization response.

Given these constraints, we opted for shorter link lengths that balance jumping performance with mechanical practicality. Future work could incorporate body dimensions into the optimization process. Additionally, activating hip motors during jumps may yield different optima that better align with design constraints.

### 9.2 Limitations of the Symmetric Jump Model

The symmetric jumping configuration has key limitations in approximating asymmetric jumping performance. For asymmetric legs, achieving a specific jump angle requires careful paw placement relative to the hip joints - a relationship that varies with link lengths. While the symmetric model places paws directly under hips, this same placement in the asymmetric case produces angled rather than vertical jumps.

Experiments show vertical jumps are possible with asymmetric legs by adjusting the hip-to-paw vector angle. However, the symmetric model only partially captures this through a constant angle

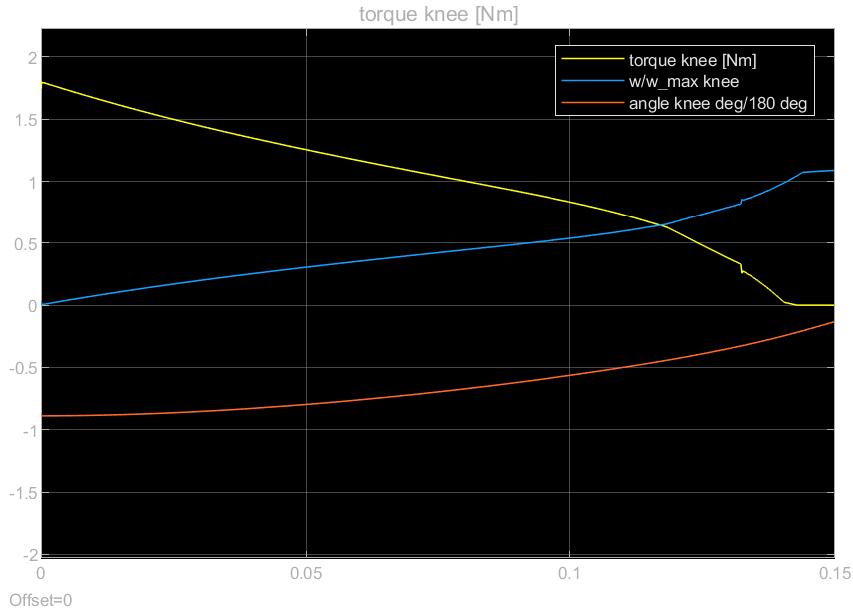


Figure 31: Knee speed until takeoff with A80BHP-H motor.

offset when  $L_2 > L_1$ , which fails to produce vertical jumps across all link lengths.

This approach introduces systematic biases:

- When  $L_1 \neq L_2$ : Underestimates jump height since asymmetric vertical jumps require paw positions that angle  $L_2$  and  $L_1$  more horizontally, better converting angular to vertical velocity with less slipping
- When  $L_1 = L_2$ : Overestimates jump height since achieving vertical jumps requires  $L_2$  to be less horizontal, reducing velocity conversion efficiency

These biases help explain the sharp performance peak at  $L_2 = L_1$  in the grid search results (figure ??) for both Earth and Mars gravity.

### 9.3 Control and Dynamic Challenges

Because the current plan is to turn the knee motors off during jumps, because the torsional springs' power is so much higher than the motors peak power, and it is unknown how the motors will handle that, the current design lacks knee feedback capability during jumps. This could limit the future RL controller's ability to compensate for parameter variations like spring stiffness. This could potentially widen the sim-to-real gap. As well as limit the robots ability to jump accurately.

When knees bend in the same direction, the shifted center of mass causes uneven leg loading. The back legs bear more weight while the front legs accelerate faster, resulting in backward rotation that complicates aerial stabilization. These challenges could be addressed by using stronger rear springs/motors or by having the hip motors on during the jump with more power provided to the back legs.

### 9.4 Design Considerations for Landing

The knee springs' equilibrium position at  $\theta_2 = 0$  (straight leg) may complicate landing. To achieve a crouched landing pose, knee motors must work against spring force, increasing response time. Further testing is needed to quantify motor response times for various knee angles.

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## 9.5 Lack of Hardware Tests

While simulation results are promising, the absence of hardware testing introduces several uncertainties. The simulated optimal link lengths may differ from real-world optima due to unmodeled dynamics and approximations. These include but are not limited to:

- Unmodeled frictions, for example friction between springs and motor housing
- The transversal motors' ability to withstand torque around their rotation axis during jumps
- The actual mass distribution likely differs from current models, which approximate legs and torso as uniform density rectangular prisms
- The torso mass and inertia properties are approximated as described in section 3.3, which may not be accurate
- Discrepancies in masses such as the leg mass. As described in section 7, the simulated and actual leg masses differ, but the difference in mass is considered negligible compared to motor and body masses
- Simplified viscous friction models for motors

These approximations, combined with simplified torso mass and inertia properties detailed in section 3.3, may lead to meaningful discrepancies between simulation and reality.

## 9.6 Motor Loading Uncertainty

A critical uncertainty is the knee motors' ability to maintain stall torque while loading the springs. As discussed in section 1, prolonged stall torque risks motor overheating. The time required to fully load springs at stall torque remains unknown, making it unclear whether motors can safely achieve maximum spring compression. This uncertainty could significantly impact the robot's jumping performance and motor longevity.

As detailed in section 3.5, the recommended safety margin for stall torque is about 20%. Our margin is smaller at 12%, which increases the risk that the motors will not be able to load the springs maximally.

## 9.7 Lack of feedback control during jumps

The current design lacks motor actuation and thus feedback control during jumps, which stops the future RL controller's ability to compensate for parameter variations like spring stiffness. This could potentially widen the sim-to-real gap. Introducing hip motor actuation during jumps would allow feedback control, which is why it will be investigated in the upcoming masters thesis.

## 9.8 Hip Abduction/Adduction Vector Arm

As can be seen in figure 19, there is a considerable distance in the  $z$  direction from the end of the shank, ie. where the paw will touch the ground, to the axis of rotation of the hip abduction/adduction motor. This distance will cause a significant and undesired load on the hip abduction/adduction motor. As the specialization project has primarily ignored abduction/adduction dynamics, adjusting the shank design to avoid this issue is considered an avenue of future work.

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## 10 Future work

The upcoming masters thesis will investigate adding hip motor actuation during jumps to enhance performance and enable feedback control. Then we will use our optimized design to construct the physical robot and create an accurate model in Nvidia Isaac Sim. This model will be used to train reinforcement learning controllers for jumping, aerial stabilization, and landing. Domain randomization techniques will help minimize the sim-to-real gap, and curriculum learning will be explored to make the learning process more tractable. Finally, we will evaluate the trained policies on the physical robot in practical jumping scenarios.

We will perform tests to evaluate the real-life jumping performance of the simulation-derived design and see if knee motors are able to provide stall torque for long enough to load springs maximally.

Due to the oversights addressed in section 4.2, the promising A80BHP-H motor was overlooked when selecting motors. This motor not only has almost twice the stall torque of the current knee motor, it also has more than twice the operating velocity. Future work will include investigating the jumping capabilities of a robot combining this motor with torsional springs.

As mentioned in section 9.8, adjusting the shank design to avoid a significant load on the hip abduction/adduction motor during jumping is something that will be addressed in future work.

Although the current robot incorporates springs in the actuation method, it does not incorporate series-elastic springs the way they are often used in legged robots, ie. to reduce the impact forces on the motors [17]. As the robot is intended to jump, future work will include adding elastic elements to the legs to reduce the impact forces on the motors.

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## 11 Conclusion

This project developed a concept design for a small and lightweight jumping quadruped robot to be used as a platform for training DRL controllers for jumping, aerial stabilization, and landing.

Through motor selection and Simulink Simscape simulations, motor-only actuation proved inadequate for achieving the desired jump performance. Consequently, integrating torsional springs was essential to store and release energy effectively during jumps.

Grid search optimization identified optimal link lengths that to maximize effective jump height. Optimal link lengths were found to be too long for practical robot construction, so link lengths were chosen according to a trade-off between jump height and robot size. System identification refined motor friction models, enhancing simulation accuracy.

Limitations include the symmetric jumping model's simplifications yielding potentially different optimal link lengths than a more realistic asymmetric jumping model. We have not yet performed hardware tests for jumping, thus we do not know how realistic the simulation results are, given mass, inertia, and friction approximations and unmodeled dynamics. We have not tested if the knee motors are able to give stall torque for long enough to load the springs, without overheating or damaging the motor, which is it assumed they can do in simulation.

Despite these limitations, the project demonstrates that the robot design is capable of sufficient jumping performance for demonstrating DRL controllers for jumping, aerial stabilization, and landing for traversing martian lava tubes.

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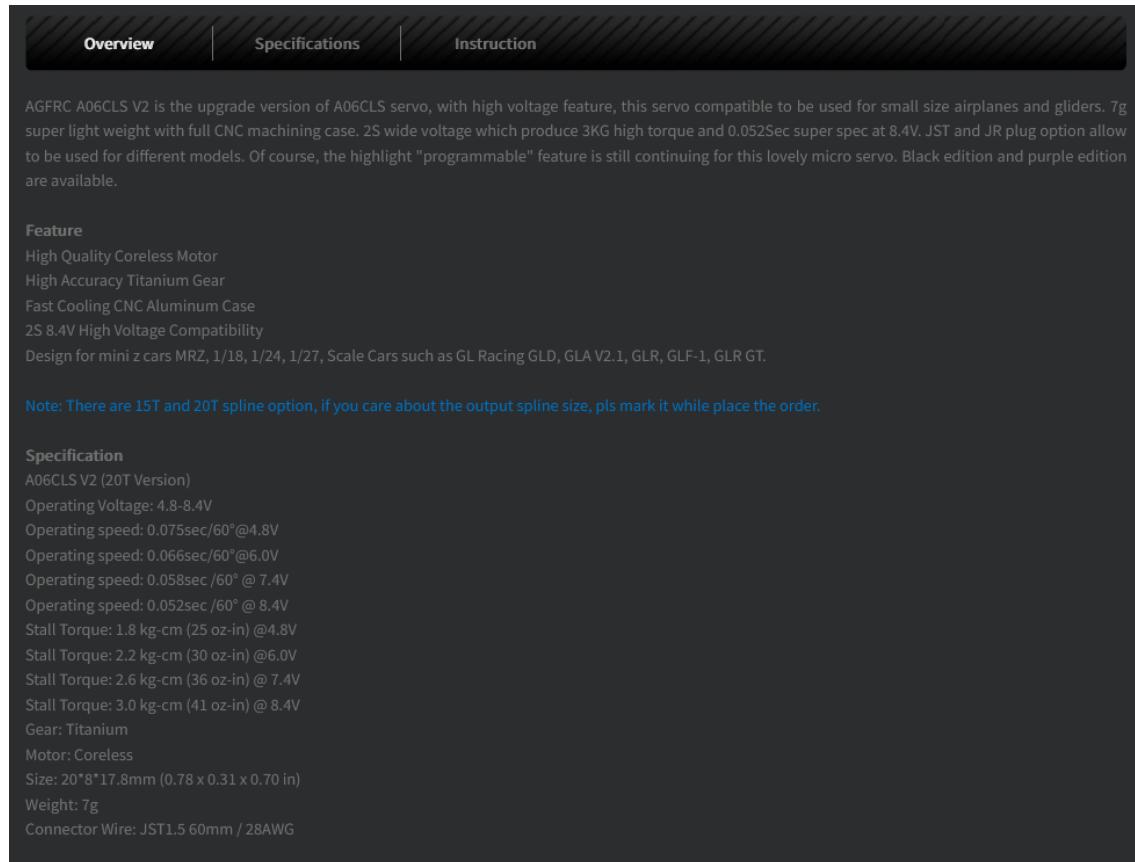
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## Appendix

### A A06CLS V2 Website Information

This information is taken from the website: <https://www.agfrc.com/index.php?id=2666>



AGFRC A06CLS V2 is the upgrade version of A06CLS servo, with high voltage feature, this servo compatible to be used for small size airplanes and gliders. 7g super light weight with full CNC machining case. 2S wide voltage which produce 3KG high torque and 0.052Sec super spec at 8.4V. JST and JR plug option allow to be used for different models. Of course, the highlight "programmable" feature is still continuing for this lovely micro servo. Black edition and purple edition are available.

**Feature**

- High Quality Coreless Motor
- High Accuracy Titanium Gear
- Fast Cooling CNC Aluminum Case
- 2S 8.4V High Voltage Compatibility
- Design for mini z cars MRZ, 1/18, 1/24, 1/27, Scale Cars such as GL Racing GLD, GLA V2.1, GLR, GLF-1, GLR GT.

**Note:** There are 15T and 20T spline option, if you care about the output spline size, pls mark it while place the order.

**Specification**

- A06CLS V2 (20T Version)
- Operating Voltage: 4.8-8.4V
- Operating speed: 0.075sec/60°@4.8V
- Operating speed: 0.066sec/60°@6.0V
- Operating speed: 0.058sec /60° @ 7.4V
- Operating speed: 0.052sec /60° @ 8.4V
- Stall Torque: 1.8 kg-cm (25 oz-in) @4.8V
- Stall Torque: 2.2 kg-cm (30 oz-in) @6.0V
- Stall Torque: 2.6 kg-cm (36 oz-in) @ 7.4V
- Stall Torque: 3.0 kg-cm (41 oz-in) @ 8.4V
- Gear: Titanium
- Motor: Coreless
- Size: 20\*8\*17.8mm (0.78 x 0.31 x 0.70 in)
- Weight: 7g
- Connector Wire: JST1.5 60mm / 28AWG

Figure 32: A06CLS V2 Motor Information (Curt)

### B A20BHM Website Information

This information is taken from the website:

<https://www.agf-rc.com/agfrc-a20bhm-21g-high-speed-0068sec-114kg-programmable-digital-brushless-48-84v-strength-steel-gear-micro-wing-servo-18010-for-airplane-aircraft-p4231199.html>

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**Description**[Item specifics](#)[Reviews \(0\)](#)**AGFRC Aluminum Case HV High Torque Brushless Micro Digital Servo (A20BHM)**

AGFrc A20BHM, is the 8.4V high voltage compatible performance digital mini servo, also the first launch brushless motor 12mm thickness RC servo. It is made in a robust CNC aluminum case with high accuracy metal gears, it particularly ideal for RC cars and other application.

**Feature**

- High Performance Digital Wing servo
- High Precision Metal Gears
- High Quality Brushless Motor
- Double Ball Bearings
- Full Aluminum Case

**Specification****A20BHM**

- Operating Voltage: 4.8-8.4 V
- Operating speed: 0.092sec/60° @ 6.0V
- Operating speed: 0.075sec/60° @ 7.4V
- Stall Torque: 8.5 kg-cm (118 oz-in) @ 6.0V
- Stall Torque: 10.2 kg-cm (142oz-in) @ 7.4V
- Gear: Strength Steel + Copper
- Motor: BLs
- Size: 23\*12\*27.5mm
- Weight: 21g
- Connector Wire: TYU 180 mm / 26AWG

## C A35CHM Motor Information

PRODUCT SPECIFICATION <b>A35CHM</b>			
<b>Control System</b>	Pulse width modulation control		
<b>Refresh Rate</b>	333Hz		
<b>Neutral Position</b>	1520uS		
<b>Signal Mode</b>	Digital		
<b>Dead band</b>	2 uSec		
<b>Operating Voltage</b>	4.8V ~ 8.4V		
<b>Operating Temperature</b>	-15C°~ +70C°		
<b>Bearing</b>	Dual Ball Bearing		
<b>Mechanical Limit Angle</b>	220°		
<b>Size</b>	35.5*15*29.2mm		
<b>Net Weight</b>	41g		
<b>Wire</b>	JR 180mm /22AGW		
<b>Operating Travel</b>	180°±10°		
<b>Signal Range</b>	500 to 2500 uSec		
<b>Stall Torque</b>	9.5 kg-cm (132 oz-in) @ 4.8V 10.5 kg-cm (146 oz-in) @ 6.0V 11.5 kg-cm (160 oz-in) @ 7.4V 12.5 kg-cm (174 oz-in) @ 8.4V	<b>Unload Current</b>	230mA @ 4.8V 300mA @ 6.0V 380mA @ 7.4V 420mA @ 8.4V
<b>Operating Speed</b>	0.145sec/ 60°@ 4.0V 0.125sec/60° @ 6.0V 0.095sec/60° @ 7.4V 0.085sec/60° @ 8.4V	<b>Loading Current</b>	2000mA @ 4.8V 2400mA @ 6.0V 2600mA @ 7.4V 2800mA @ 8.4V
<b>Direction</b>	<input checked="" type="checkbox"/> CCW <input type="checkbox"/> CW		
<b>Waterproof Level</b>	<input type="checkbox"/> IP65 <input type="checkbox"/> IP67		
<b>Angle Sensor</b>	<input checked="" type="checkbox"/> Potentiometer <input type="checkbox"/> Magnet Angle Sensor		
<b>Motor Type</b>	<input type="checkbox"/> Brushless <input checked="" type="checkbox"/> Coreless <input type="checkbox"/> DC		
<b>Motor Drive</b>	<input checked="" type="checkbox"/> FET Drive <input type="checkbox"/> IC Drive <input type="checkbox"/> Transistor Drive		
<b>Programmable</b>	<input checked="" type="checkbox"/> Yes <input type="checkbox"/> No		
<b>Gear Material</b>	<input type="checkbox"/> Strength Steel <input checked="" type="checkbox"/> Titanium <input type="checkbox"/> Copper <input type="checkbox"/> Plastic		
<b>Horn Gear Spline</b>	<input checked="" type="checkbox"/> 25T-φ5.92mm <input type="checkbox"/> 25T-φ4.94mm <input type="checkbox"/> Other _____		
<b>Case Material</b>	<input checked="" type="checkbox"/> AL6061T6 <input type="checkbox"/> AL+Plastic <input type="checkbox"/> Plastic		
<b>Bearing Material</b>	<input checked="" type="checkbox"/> Metal <input type="checkbox"/> Plastic		
<b>Horn Accessories</b>	<input type="checkbox"/> AL6061T6 <input checked="" type="checkbox"/> Plastic		
<b>Wire Color</b>	Negative: <input checked="" type="checkbox"/> Black <input type="checkbox"/> Brown Positive: <input type="checkbox"/> Black <input checked="" type="checkbox"/> Red Signal: <input type="checkbox"/> Grey <input checked="" type="checkbox"/> White <input type="checkbox"/> Orange		

Figure 33: A35CHM Motor Information

## D A80BHP-H Motor Information

Please note that this brochure gives an operating travel of 90 degrees, this was later confirmed to be false, the actual operating travel is 180 degrees.

# SERVO SPECIFICATION

Operating Voltage	Gear Material	Operating Temperature	Case Material
4.8V ~ 8.4V	Steel	-15C°~ +70C°	Full Aluminum
Operating Travel	Mechanical Angle	Net Weight	Pulse Width
90°±5°	220°	79g	452 to 1072 uSec
Motor Type	Dead Band	Frequency	Waterproof Level
Brushless	2 uSec	760uS / 666Hz	/
Size	Bearing	Wire	Programmable
40*20*37.5mm	3BB	TYU 300mm / 22AWG	Yes

Operating Speed
0.030Sec / 60° @ 8.4V
0.034Sec / 60° @ 7.4V
0.039Sec / 60° @ 6.0V
0.045Sec / 60° @ 4.8V

Stall Torque
18.5KG-CM (257 oz-in) @ 8.4V
15.5KG-CM (215 oz-in) @ 7.4V
14.0KG-CM (195 oz-in) @ 6.0V
12.0KG-CM (167 oz-in) @ 4.8V



Figure 34: A80BHP-H Motor Information

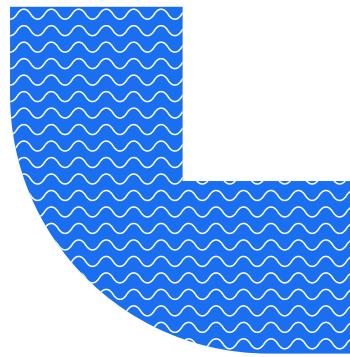
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## E Ultimaker Tough PLA Technical Data Sheet

Ultimaker

# Ultimaker Tough PLA

## Technical data sheet



### General overview

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Chemical composition	See Tough PLA safety data sheet, section 3
Description	Ultimaker Tough PLA is a technical PLA filament with toughness comparable to Ultimaker ABS. Ideal for reliably printing technical models at large sizes, our Tough PLA offers the same safe and easy use as regular PLA
Key features	<p>With an impact strength similar as and higher stiffness compared to Ultimaker ABS, Tough PLA is less brittle than regular PLA and gives a more matte surface finish quality. Heat resistance is similar to standard PLA filaments, so printed parts should not be exposed to temperature above 58 °C.</p> <p>More reliable than ABS for larger prints, with no delamination or warping. Ultimaker Tough PLA is compatible with Ultimaker support materials (PVA and Breakaway), giving full geometric freedom when designing parts</p>
Applications	Functional prototyping, tooling, manufacturing aids
Non-suitable for	Food contact and in vivo applications. Long term outdoor usage or applications where the printed part is exposed to temperatures higher than 58 °C.

### Filament specifications

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	Method (standard)	Value
Diameter	-	2.85 ± 0.05 mm
Max roundness deviation	-	0.05 mm
Net filament weight	-	750 g
Filament length	-	~96 m

### Color information

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Color	Color code
Black	RAL 9017
White	RAL 9003
Green	RAL 6038
Red	RAL 3018
Gray	RAL 7000
Yellow	RAL 1018
Blue	RAL 5019

Figure 35: UM220509 Tough PLA TDS