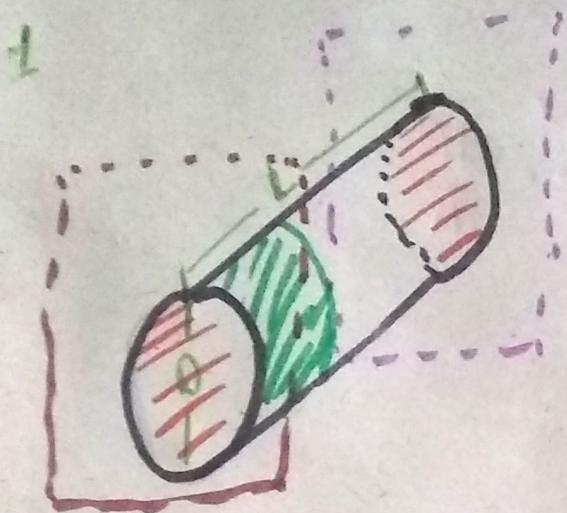
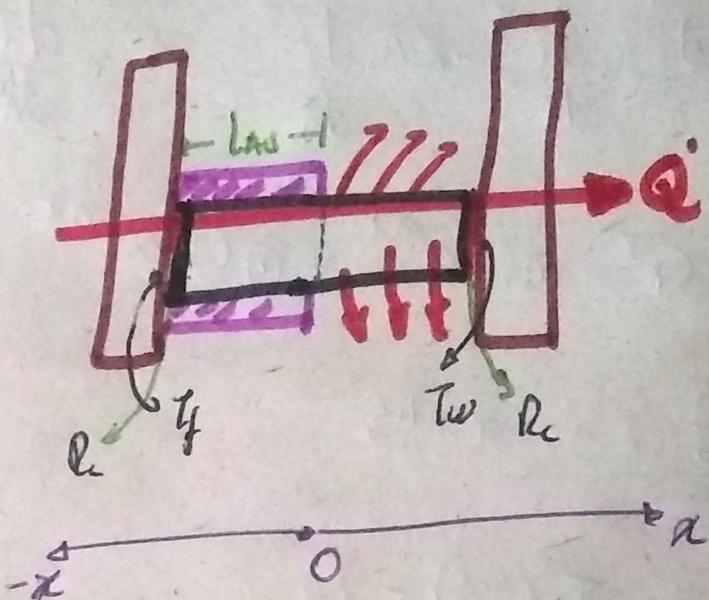


Fenómenos Transporte



Conductividad K



a)

Ecación General de Calor

Sistema I Autocodo.

$$\text{B.C. 1} \Rightarrow x = -L_c/2$$

$$\dot{Q} = \frac{T_f - T(-L_c/2)}{R_c}$$

$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} + \frac{\rho q u n}{K} = \frac{\partial T}{\partial t}$$

$$\frac{dT^I}{dx} = G^I$$

$$T^I(x) = C_1^I x + C_2^I$$

$$\dot{Q} = -KA_s \frac{dT}{dx} \rightarrow$$

$$-KA_s G^I = \frac{T_f - T^I(-L_c/2)}{R_c}$$

$$-KA_s C_1^I = \frac{T_f - C_1^I x + C_2^I}{R_c}$$

$$\text{B.C. 2} \Rightarrow \frac{x=0}{T^I(x) = T^{II}(x)} \rightarrow ??$$

$$C_1^I x + C_2^I = C_1^{II} e^{mx} + C_2^{II} e^{-mx}$$

$$C_2^I = C_1^{II} + C_2^{II}$$

Situacion II

$$\underline{\text{B.C1}} \rightarrow \underline{x=0}$$

$$\dot{Q}_{in} = Q_{out}$$

$$-\kappa A_1 \frac{dT^I}{dx} = -\kappa A_2 \frac{dT^{II}}{dx} \Rightarrow \frac{dT^I}{dx} = \frac{dT^{II}}{dx}$$

$$C_I^I = m C_1^I e^{mx} - m C_2^I e^{-mx}$$

$$\underline{\text{B.C2}} \rightarrow \underline{x=L_2}$$

$$\dot{Q} = \frac{T^I(L_2) - T_w}{R_c}$$

$$-\kappa A_2 \frac{dT^{II}}{dx} = \frac{T^{II}(L_2) - T_w}{R_c}$$

$$-\kappa A_2 (m C_1^I e^{mx} - m C_2^I e^{-mx}) = \frac{C_1^I e^{mx} + C_2^I e^{-mx} - T_w}{R_c}$$

In a Matrix System:

$$1) -KA_1 R_C C_1^{(I)} = T_f (-C_1^{(I)}(\omega_h) + C_2^{(I)})$$
$$C_1^{(I)}(-KA_1 R_C - \frac{L}{\omega_h}) - C_2^{(I)} = T_f$$

$$2) C_1^{(I)} - C_1^{(II)} - C_2^{(II)} = 0$$
$$\alpha[1] \quad \alpha[2] \quad \alpha[3]$$

$$3) C_1^{(I)} - m e^{m x} C_1^{(II)} + m e^{-m x} C_2^{(II)} = 0$$
$$\alpha[0] \quad \alpha[2] \quad \alpha[3]$$

$$4) R_f K A_1 m e^{m x} C_1^{(I)} + R_C K A_1 m e^{-m x} C_2^{(II)} = e^{m x} C_1^{(II)} + e^{-m x} C_2^{(III)} - T_f$$
$$C_1^{(I)}(-K R_C m e^{-m x} - e^{m x}) + C_2^{(II)}(R_C K A_1 m e^{-m x} - e^{-m x}) = -T_f$$
$$\alpha[2] \quad \alpha[3]$$

Recall:

$$m = \sqrt{\frac{hp}{KA_c}}$$

$p \rightarrow$ perimeter

$h \rightarrow$ convection

$K \rightarrow$ conduction

$A_c \rightarrow$ cross Area.

20 cm. 8m reboilants

$$D = 2.2 \text{ cm}$$

$$p = 2\pi r K_{c,1/2}$$

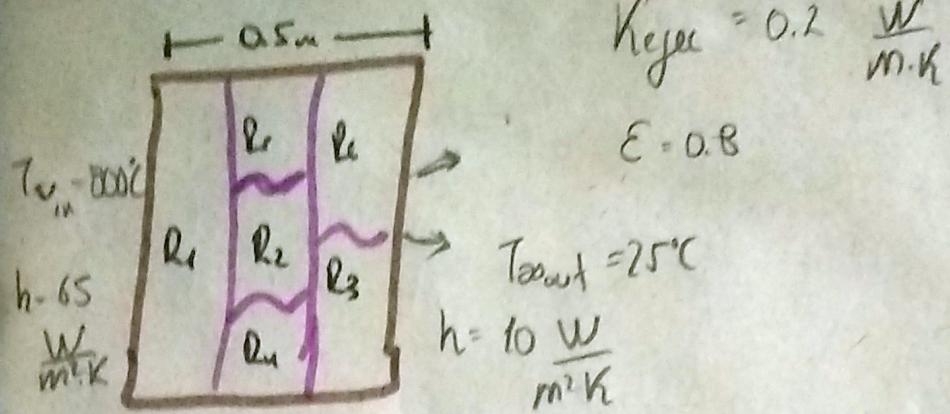
$$p = \pi D K_{c,1/2}$$

$$A_c = \frac{\pi D^2}{4}$$

$$m = \sqrt{\frac{\pi D L_{c,1/2} \cdot h}{K \cdot \frac{\pi D^2}{4}}}$$

$$m = \sqrt{\frac{4 \cdot L_{c,1/2} \cdot h}{K \cdot D}}$$

2^{do} PONTO.



a) Resistencia Térmica equivalente

Sabemos por sistema de Resistencias Térmicas

$$\dot{Q} = \frac{T_{amb,in} - T_{amb,out}}{R_{eq} + R_{paralelo} + R_{conductivo}}$$

Donde

$$R_{eq} = R_1 + R_2 + R_3 + \dots + R_n$$

donde

$$R_{eq, paralelo} = \frac{R_1 \cdot R_2 \cdot R_3 \cdots R_n}{R_1 + R_2 + R_3 + R_n}$$

$$R_{eq, total} = R_{eq, serie} + R_{eq, paralelo}$$

Si están en Serie.

Si están en Paralelo.

Conductivo

$$\text{Donde } R_i = \frac{L_i}{R_i \cdot A_s}$$

$$\dot{Q} = \frac{T_{amb,in} - T_{amb,out}}{R_{eq, total}}$$

$$R_{eq, total} = \frac{L_T}{R_{eq} \cdot A_s}$$

$$Q = \frac{T_{sur,m} - T_{sur,out}}{R_{conv,1} + R_{eq} + R_{conv,2}}$$

mm:80

$$\frac{dI}{dx} = C_1$$

$$T(x) = C_1 x + C_2$$

25°C

$$(\Delta I) / \left(\frac{1}{h_{conv,1}} + \frac{0.5}{0.2} + \frac{R_{conv,2} R_{rad}}{R_{conv,2} + R_{rad}} \right)$$

$$\left(\frac{1}{10} \cdot \frac{1}{\epsilon \sigma (T_{sur,out} + T_{sur,m}) (T_{sur,out}^2 + T_{sur,m}^2)} \right)$$

$$\frac{\frac{1}{10} \cdot \frac{1}{\epsilon \sigma (C_1(0.5) + C_2 + 298.15) ((C_1(0.5) + C_2)^2 + 298.15^2)}}{\frac{1}{10} + \frac{1}{h_{conv,1}}}$$

Constraints given by boundary condition

$$B.C.1 \rightarrow x=0$$

$$\dot{Q}^I = \dot{Q}^{II} \rightarrow h_{conv,1} (T_{sur,in} - T_{sur,m}) = -K C_1^{II}$$

$$h_{conv,1} (1073.15 - C_1^I(0) + C_2^{II}) = -K C_1^{II}$$

$$h_{conv,1} (1073.15 + C_2^{II}) + K C_1^{II} = 0$$

$$B.C.2 \rightarrow x=L$$

$$Q = \frac{T_{sur,m} - T_{sur,out}}{R_{conv,1} + R_{eq}} = h_{conv,1} (T_{sur,m}^{II} - T_{sur,in}^{II}) \left(\frac{1}{h_{conv,1}} + \frac{0.5}{0.2} \right)$$

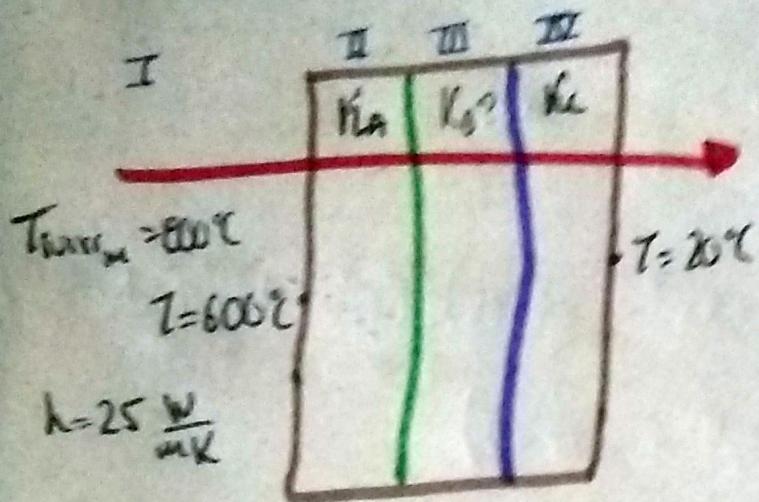
$$\left. \begin{aligned} & \Rightarrow (1073.15 + C_2^{II}) + h_{conv,1} \frac{0.5}{0.2} (1073.15 - C_2^{II}) \\ & = (1073.15 - 0.5 C_2^{II} - C_2^{II}) \end{aligned} \right\}$$

$BCL \rightarrow x=L$?? \rightarrow overheat
 $Q^{\text{in}} = Q^{\text{III}}$

$$-K \frac{dT}{dx} = h_{\text{comb}} (T_{\text{out, out}}^{\text{II}} - T_{\text{out, in}}^{\text{III}})$$

$$-K C_1^{\text{II}} = h_{\text{conv}} + \epsilon \sigma (C_1^{\text{II}} 0.5 + C_2^{\text{II}} + 298.15) ((C_1^{\text{II}} 0.5 + C_2^{\text{II}})^2 + 298.15) \\ (C_1^{\text{II}} 0.5 + C_2^{\text{II}} - 298.15)$$

Übung 10



$$K_A = 20 \frac{W}{m \cdot K}$$

$$K_C = 50 \frac{W}{m \cdot K}$$

$$L_A = 0.30 \text{ m}$$

$$L_C = 0.15 \text{ m}$$

$$L_B = 0.15 \text{ m}$$

$$K_B = ??$$

- Steady State
- Point Contact

• Kontakt Area

$$I \quad Q = \frac{T_{infty,A} - T_{out,ext}}{R_{cond}}$$

$$I \quad Q = \frac{1073.15 - 873.15}{\frac{1}{25}} \rightarrow \dot{Q} = 5000 \text{ [W]}$$

II

$$Q^I = Q^Z \rightarrow Q^I = \frac{T_{out,ext} - T_{out,ext,A}}{R_{cond}^I}$$

$$Q^I = 5000 = \frac{873.15 - ??}{\frac{0.30}{20}}$$

$$\Rightarrow T_{out,ext,A} = 793.15 \text{ [K]}$$

IV

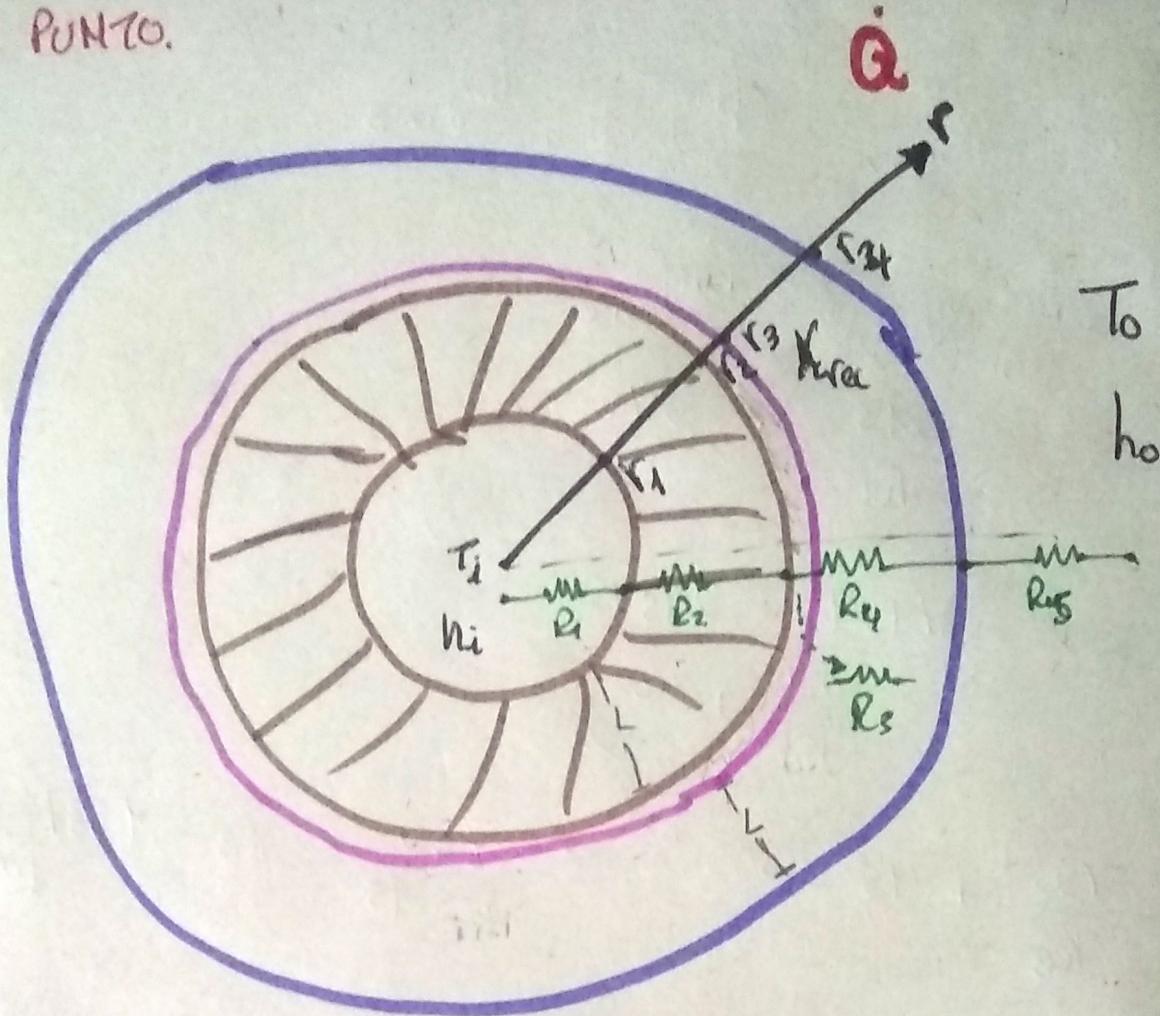
$$Q^{\text{III}} = Q^{\text{II}} \quad T_{\text{vrf}_{\text{outA}}} = T_{\text{vrf}_{\text{inB}}}$$

$$\dot{Q} = \frac{T_{\text{vrf}_{\text{outA}}} - T_{\text{vrf}_{\text{outC}}}}{\frac{0.15}{K_B} + \frac{0.15}{50}}$$

$$5000 = \frac{798.15 - 293.15}{\frac{0.15}{K_B} + \frac{0.15}{50}}$$

$$\Rightarrow \frac{750}{K_B} + 15 = 505 \quad \Rightarrow \quad \frac{750}{505-15} = K_B = 1.531 \left[\frac{W}{m \cdot K} \right]$$

3 PUNTO.



dissipa el flujo de calor uniforme Q''_h $\left[\frac{W}{m^2} \right]$

$$R_1 = \frac{1}{h_i A_s} = \frac{1}{h_i 2\pi r_1} ; \quad R_2 = \frac{\ln(r_2/r_1)}{2\pi(r_2 - r_1) K_{PIACA}}$$

$$R_3 = \frac{\ln(r_3/r_2)}{2\pi(r_3 - r_2) K_{PIACA}} ; \quad R_4 = \frac{\ln(r_4/r_3)}{2\pi(r_4 - r_3) K_{auslante}}$$

$$R_5 = \frac{1}{h_o A_s} = \frac{1}{h_o 2\pi r_4}$$

$$\dot{Q} = \frac{T_{\text{ext,in}} - T_{\text{ext,out}}}{R_{\text{tot}}}$$

se desprecia radiación.

$$R_{\text{tot}} = R_1 + R_2 + R_3 + R_4 + R_5$$

Suposición \Rightarrow si la resistencia drena $q''_h \left[\frac{\text{W}}{\text{m}^2} \right]$ por tanto el flujo \dot{Q}

$$\dot{Q} = q''_h * \text{Volumen de lámina} \Rightarrow \dot{Q} = q''_h \cdot 2\pi(r_3 - r_2) \cdot L \rightarrow \begin{matrix} \text{longitud} \\ \text{tubería} \end{matrix}$$

$$\frac{dT}{dr} = \frac{C_1^{III}}{r} ; \quad T(r) = C_1^{IV} \ln r + C_2^{IV}$$

$$\text{BC1} \Rightarrow r = r_2$$

$$\dot{Q}^{II} = Q^{III} \rightarrow -K A_s \frac{dT^{II}}{dx} = -K A_s \frac{dT^{III}}{dx}$$

$$Q^{II} = Q^I = Q^{III} = -K A_s \frac{dT}{dx}^{III}$$

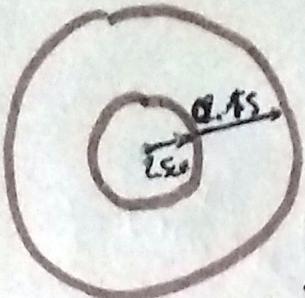
$$q''_h \cdot 2\pi(r_3 - r_2) \cdot L = -K A_s \frac{C_1^{III}}{r} \Rightarrow q''_h L = -K \frac{C_1^{III}}{r_2}$$

$$\text{BC2} \Rightarrow r = r_1$$

$$q''_h 2\pi(r_3 - r_2) \cdot L = \frac{T_{\text{surf,ext,in}}^{II} - T_{\text{surf,ext,in}}^{III}}{R_1 R_2 + R_2 R_3} \Rightarrow$$

$$\dot{Q} = \frac{T_{\text{surf,ext,in}}^{I} - C_1^{III} \ln r_2 + C_2^{IV}}{\frac{1}{h_1' 2\pi r_1} + \frac{\ln(r_2/r_1)}{2\pi(r_2-r_1)K}}$$

$P = 20 \text{ bar} \rightarrow \text{vapor saturado}$



$$h_i = 30 \frac{\text{W}}{\text{m}^2\text{K}}$$

$$D = 5 \text{ cm} \approx 0.05 \text{ m}$$

$$\text{espacio} = 15 \text{ mm} \\ \approx 0.015 \text{ m}$$

$$T_{amb} = 25^\circ\text{C}$$

$$h_o = 5 \frac{\text{W}}{\text{m}^2\text{K}}$$

$$h_{max} = 15.6 \frac{\text{W}}{\text{m}^2\text{K}}$$

$$\epsilon = 0.5$$

$$P \approx 2000 \text{ kPa}$$

rad
conv,1 conv,2
conv,1 conv,2

Temperaturas para el Vapor Saturado.

T	200 °C	1553.8 kPa
	220 °C	2318

$$T \Rightarrow y = mx + b \rightarrow y - y_1 = m(x - x_1)$$

$$y = mx - mx_1 + y_1$$

Ecación General de Calor.

$$\frac{dT}{dr} = \frac{C_1}{r} ; \quad T(r) = C_1 \ln(r) + C_2$$

$$\text{B.C. 1} \rightarrow a = 0.025 \text{ m}$$

$$\frac{dT^I}{dx} = \frac{dT^II}{dx}$$

$$h(T_{conv,in}^I - T_{surf,in}^I) = -k \frac{C_1}{r}$$

$$h(T_{conv,in}^I - C_1 \ln(r) - C_2) = -k \frac{C_1}{r}$$

$$\alpha = -k \frac{C_1}{r}$$

$$\text{B.C. 2. } \alpha = 0.025 + 0.015$$

$$\alpha = \frac{T_{in}^I - T_{surf}^III}{R_{conv,1} + R_{cond} + R_{comb}}$$

$$\alpha = \frac{T_{in}^I - T_{surf}^III}{\frac{1}{h_1 A_s} + \frac{\ln(r_2 - r_1)}{2\pi k_s} + \frac{1}{h_{comb} A_s}}$$

$$h_{comb} = h_{conv,2} + \epsilon \sigma (T_{surf} + T_{surf,2})^4 / (T_{surf}^4 + T_{surf,2}^4)$$

$$h_{\text{comb}} = h_{\text{conv}} + 0.5(\sigma)(C_1 \ln(T_{\text{max}}) + C_2 - T_{\text{sur}}^2) \\ * [(C_1 \ln(T_{\text{max}}) + C_2)^2 - T_{\text{sur}}^2]$$

Suppose the Tsugane ^{II}

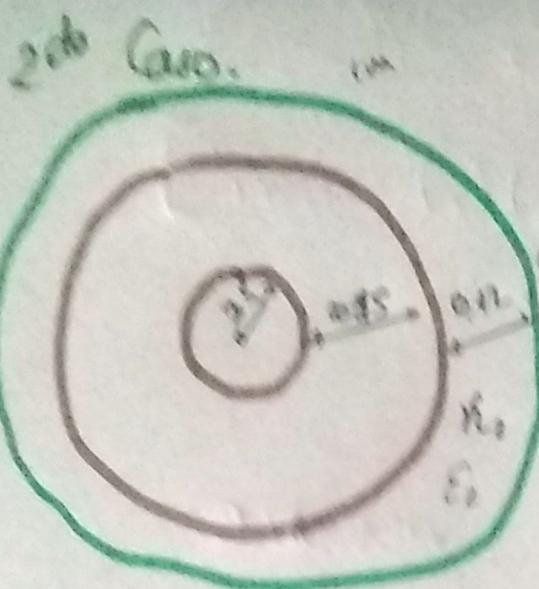
$$T_{\text{sur}} - C_1 \ln(r) - C_2 = \Delta T \quad \text{minimize}$$

_{suppose}

B.C. $\rightarrow x=0.05$ y B.C. en i. solo.

$$-\frac{\kappa C_1}{0.035} = \frac{T_{\text{in}}^{\text{I}} - T_{\text{sur}}^{\text{III}}}{\frac{1}{h u A_s} + \frac{\ln(0.05 - 0.035)}{2\pi K_i} + \frac{1}{A_s(h_{\text{conv}} + \epsilon\sigma)}} -$$

_{$2\pi r$} _{0.035} _{$A_s(h_{\text{conv}} + \epsilon\sigma)$} _{$2\pi(0.05)$}



some heat loss, heat gain

Suppose the degree T

we suppose $429.15 - C_1 \ln(0.025 + 0.015 + 0.012) - C_2 = \Delta T$

B.C. 1 B.C. 2

$$-\frac{K(C_1)}{0.025} = \frac{T_m^I - T_{env}^{IV}}{\frac{1}{h_1 A_s} + \frac{\ln(1/h_e)}{2\pi K_1}} + \frac{\ln(1/h_e)}{2\pi K_2} + \frac{1}{A_s(h_{env} + \epsilon 0 \dots)}$$

$$\frac{B.C.}{0.025} = \frac{2\pi(0.025 + 0.015 + 0.012)}{2\pi(0.025 + 0.015 + 0.012)}$$