

## MEC 361 Assignment

Fidelugwuowo Dilibe. Reg No: 2018/248767

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## Question 2.1

An 80-m-long wire of 5-mm diameter is made of a steel with  $E = 200 \text{ GPa}$  and an ultimate tensile strength of  $400 \text{ MPa}$ . If a factor of safety of 3.2 is desired, determine

- the largest allowable tension in the wire
- the corresponding elongation of the wire

### Solution

Given:

$$L = 80m$$

$$E = 200GPa$$

$$d = 5mm$$

$$n = 3.2$$

$$\text{Formular : } E = \frac{\sigma}{A}$$

$$\text{Allowable stress, } \sigma_{allow} = \frac{\text{yieldstress}}{N}$$

$$\sigma_{allow} = \frac{400 \times 10^6}{3.2} = 125MPa$$

To find the extension,

$$e = \frac{\sigma}{E} = \frac{400 \times 10^6}{200 \times 10^9} = 2 \times 10^{-3}$$

The value for the extension has no unit

## Question 2.9

An aluminum control rod must stretch 0.08 in. when a 500-lb tensile load is applied to it. Knowing that  $\sigma_{ult} = 22ksi$  and  $E = 10.1 \times 10^6 psi$ , determine the smallest diameter and shortest length that can be selected for the rod.

### Solution

Given:

$$\delta = 0.08in$$

$$\sigma = 22 \times 10^6$$

$$E = 10.1 \times 10^6 psi$$

$$P = 500lb$$

For the diameter:

$$A = \frac{\pi}{4}d^2$$

$$\sigma = \frac{P}{A}$$

Therefore

$$A = \frac{P}{\sigma} = \frac{500}{22 \times 10^6} = 0.0227 \text{ in}^2$$

making d the subject of the formula, we get

$$d = \sqrt{\frac{4 \times 2.27 \times 10^{-2}}{\pi}} = 0.1701 \text{ in}$$

For the shortest length, we have

$$E = \frac{\sigma}{\epsilon}, \epsilon = \frac{\delta}{L}$$

Therefore the equation above transcribe to

$$E = \frac{\sigma L}{\delta}$$

Making L the subject of the formula,

$$L = \frac{E\delta}{\sigma} = \frac{10.1 \times 10^6 \times 0.08}{22 \times 10^3} = 36.72 \text{ in}$$

## Question 2.15

A 4-ft section of aluminum pipe of cross-sectional area  $1.75 \text{ in}^2$  rests on a fixed support at A. The  $\frac{5}{8}$ -in.-diameter steel rod BC hangs from a rigid bar that rests on the top of the pipe at B. Knowing that the modulus of elasticity is  $29 \times 10^6$  psi for steel and  $10.4 \times 10^6$  psi for aluminum, determine the deflection of point C when a 15-kip force is applied at C.

### Solution

Given:

Diameter of the Steel rod  $d_s = \frac{5}{8} \text{ in}$

Length of the aluminium rod,  $L_A = 4 \text{ ft} = 48 \text{ in}$

Length of the steel rod,  $L_S = 7 \text{ ft} = 84 \text{ in}$

Cross sectional Area of the aluminium,  $A_A = 1.75 \text{ in}^2$

Modulus of elasticity for aluminum,  $E_A = 10.4 \times 10^6$

Modulus of elasticity for steel.  $E_S = 29 \times 10^6$

The force that  $P = 15 \times 10^3$

we will now find the cross sectional area of steel

$$A_S = \frac{\pi}{4}d_S = \frac{\pi}{4} \times \left(\frac{5}{8}\right)^2 = 0.3068 \text{ in}^2$$

Deflection at c :

$$\delta_c = \delta_A + \delta_S$$

$$\delta_A = \frac{PL_A}{A_A E_A} = \frac{15 \times 10^3 \times 48}{1.75 \times 10.4 \times 10^6} = 0.03956 \text{ in}$$

$$\delta_S = \frac{PL_S}{A_S E_S} = \frac{15 \times 10^3 \times 84}{6.3068 \times 29 \times 10^6} = 0.14162 \text{ in}$$

$$\delta_c = \delta_A + \delta_S$$

$$\delta_c = 0.03956 + 0.14162 = 0.181 \text{ in}$$

## Question 2.19

Both portions of the rod ABC are made of an aluminum for which  $E = 70 \text{ GPa}$ . Knowing that the magnitude of P is 4 kN, determine

1. the value of Q so that the deflection at A is zero
2. the corresponding deflection of B

### Solution

Given:

$$E = 70 \text{ GPa} = 70 \times 10^9 \text{ Pa}$$

$$P = 4 \text{ kN} = 4 \times 10^3 \text{ N}$$

$$d_{DB} = 20 \text{ mm}, d_{BC} = 6 \text{ mm}, d_{AB} = 0.4 \text{ mm}, d_{B2} = 0.5 \text{ m}.$$

Calculating the area of different parts of aluminum,

$$A_{AB} = \frac{\pi}{4}d^2 = \frac{\pi}{4}20 \times 10^{-3^2} = 3.24 \times 10^{-4} \text{ m}^2$$

$$A_{BC} = \frac{\pi}{4}d^2 = \frac{\pi}{4}60 \times 10^{-3^2} = 2.827 \times 10^{-3} \text{ m}^2$$

Using the method of superposition, we can assume that the forces acting at different points in the metal, sum up to zero.

Now for there to be zero deflection at A,

$$\delta_{AB} = -\delta_{BC}$$

At section AB,

$$\delta_{AB} = \frac{PL_{AB}}{A_{AB}E} = \frac{4 \times 10^3 \times 0.4}{3.14 \times 10^{-4} \times 70 \times 10^9} = 7.279 \times 10^{-6}$$

At section BC

$$\begin{aligned}\delta_{BC} &= \frac{(P - Q)L_{BC}}{A_{BC}E} = \frac{(P - Q) \times 0.5}{2.827 \times 10^{-3} \times 70 \times 10^9} \\ &= \frac{(P - Q)0.5}{2.125 \times 10^{-3}} = (P - Q)2.52 \times 10^{-9}\end{aligned}$$

Following the statement we made for a zero deflection at A,

$$\begin{aligned}7.275 \times 10^{-5} &= -(P - Q)2.52 \times 10^{-9} \\ 7.275 \times 10^{-5} &= -(1.010 \times 10^{-5}) + Q(2.52 \times 10^{-9}) \\ 8.285 \times 10^{-5} &= (Q)2.52 \times 10^{-9} \\ Q &= \frac{8.285 \times 10^{-5}}{2.52 \times 10^{-9}} = 32.97 \times 10^3 N\end{aligned}$$

For Deflection to occur at B,

$$\delta_{AB} = \delta_B = 7.275 \times 10^{-5}$$

## Question 2.26

The length of the  $\frac{3}{32}$  in.-diameter steel wire CD has been adjusted so that with no load applied, a gap of  $\frac{1}{16}$  in. exists between the end B of the rigid beam ACB and a contact point E. Knowing that  $E = 29 \times 10^6$  psi, determine where a 50-lb block should be placed on the beam in order to cause contact between B and E.

Solution

Given:

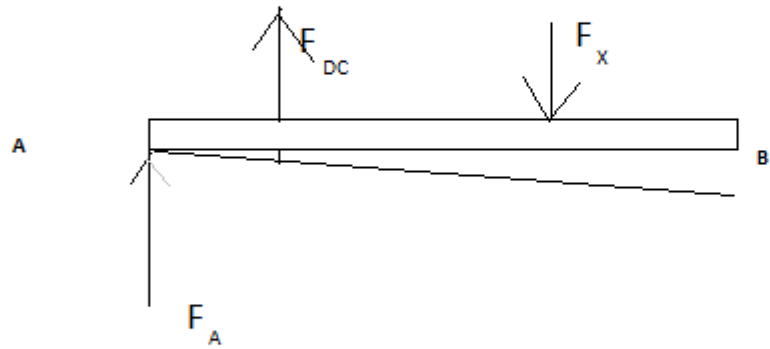
$$\text{Length of string, } L_s = 12.5 \text{ in}$$

$$\text{diameter of string, } d_s = \frac{3}{32} \text{ in}$$

$$E_S = 29 \times 10^6$$

$$P = 50 \text{ lb}$$

Drawing a free body diagram of the system, we have



### The Free Body Diagram of the question

Taking moment about point A, we have

$$M_A = 4f_s - 50(20 - x) = 0$$

but  $f_s = ?$ , However

$$\delta_s = \frac{f_s L_s}{A_s E}. \text{ This implies that } f_s = \frac{\delta_s A_s E}{L_s}$$

$$\delta_s = 4\theta$$

where 4 is the distance from the string to the point A  
 $\theta$  is the angle covered by the beam

$$\theta = \frac{\frac{1}{16}}{20} = 3.125 \times 10^{-3} \text{ rads}$$

$$\delta_s = 4 \times 3.235 \times 10^{-2} = 0.065 \text{ in}$$

$$A_s = \frac{\pi}{4} \times \left(\frac{3}{32}\right)^2 = 6.903 \times 10^{-3} in^2$$

$$F = \frac{0.0125 \times 6.903 \times 10^{-3} \times 29 \times 10^6}{12.5} = 200187 lbs$$

Hence,

$$4f_s - 50(20 - x) = 0$$

$$(4 \times 200.187) - 1000 + 50x = 0$$

$$800.748 - 1000 + 50x = 0$$

$$50x = 199.252$$

$$x = \frac{199.252}{50}$$

$$x = 3.985 in$$

### Question 2.35

A 4-ft concrete post is reinforced with four steel bars, each with a  $\frac{3}{4}$  in diameter. Knowing that  $E_s = 29 \times 10^6 psi$  and  $E_c = 3.6 \times 10^6 psi$ , determine the normal stresses in the steel and in the concrete when a 150-kip axial centric force P is applied to the post.

Solution

$$P = P_c + P_s$$

deformation of a steel bar and that of a concrete are equal, hence

$$\delta_c = \delta_s$$

$$\frac{P_c L_c}{A_c E_c} = \frac{P_s L_s}{A_s E_s}$$

making  $P_c$  the subject of the formular, we have

$$P_c = \frac{P_s A_c E_c}{A_s E_s}$$

$$A_s = 4 \times \frac{\pi}{4} \times \left(\frac{3}{4}\right)^2 = 1.767 in^2$$

$$A_s = 8.2 - A_s = 64 - 1.767 = 62.233 in^2$$

Since  $P_c = \frac{P_s A_c E_c}{A_s E_s}$

$$P = P_c + P_s$$

becomes

$$P = \frac{P_s A_c E_c}{A_s E_s} + P_s$$

This then becomes

$$P = P_s \frac{A_c E_c}{A_s E_s} + 1$$

$$P = 150 Kips = 180 \times 10^3 lbs$$

$$150 \times 10^3 = P_s \left( \frac{62.233 \times 3.6 \times 10^6}{1.767 \times 29 \times 10^6} + 1 \right)$$

$$150 \times 10^3 = P_s (4.372 + 1)$$

$$P_s = \frac{150 \times 10^3}{5.372}$$

$$P_s = 27922.56 lbs$$

Therefore  $P_c = 50000 - 27922.56 = 122077.44$  Normal Stress:

$$\sigma_c = \frac{P_c}{A_c} = \frac{122077.434}{62.233} = 1961.619 psi$$

$$\sigma_s = \frac{P_s}{A_s} = \frac{27922.56}{1767} = 15802.24 psi$$

## Question 2.41

### Solution

The total deformation of the bar is

$$\delta = \delta_{AD} + \delta_E = 0$$

Where  $\delta_{AD}$  is as a result of the forces 60KN and 40KN and  $\delta_E$  is as a result of the reaction at E

$$\delta_{AD} = \frac{P_1 L_1}{A_B E_B} + \frac{P_2 L_2}{A_B E_B} + \frac{P_3 L_3}{A_S E_S} + \frac{P_4 L_4}{A_S E_S}$$

$$P_1 = 0, P_2 = P_3 = 40 KN = 40 \times 10^3, P_4 = 100 \times 10^3$$

$$A_1 = A_2 = \frac{\pi}{4} \times (30 \times 10^{-3})^2 = 7.069 \times 10^{-4}$$

$$A_3 = A_4 = \frac{\pi}{4} \times (40 \times 10^{-3})^2 = 1.257 \times 10^{-4}$$

$$l_1 = l_2 = 100 mm = 0.1$$

$$l_3 = 120 mm = 0.12$$

$$l_4 = 180 mm = 0.18$$



$$\begin{aligned}
E_s &= 200 \times 10^9, E_b = 205 \times 10^{-3} \\
\delta_{AD} &= 0 + \frac{40 \times 0.1 \times 10^3}{7.069 \times 10^{-3} \times 105 \times 10^9} + \frac{40 \times 10^3 \times 0.12}{1.259 \times 10^{-3} \times 200 \times 10^4} + \frac{100 \times 10^3 \times 0.18}{1.257 \times 10^{-3} \times 200 \times 10^4} \\
&= (5.384 \times 10^{-5}) + (1.909 \times 10^{-5}) + (7.16 \times 10^{-5}) = 14.458 \times 10^{-5} m \\
\delta_E &= \frac{R_E \times 0.2}{7.069 \times 10^{-3} \times 105 \times 10^9} + \frac{R_E \times 0.3}{1.259 \times 10^{-3} \times 200 \times 10^4} \\
&= (6.695 \times 10^{-9} R_E) + (1.193 \times 10^{-3} R_E) = 3.886 \times 10^{-9} R_E
\end{aligned}$$

Equating  $\delta_E$  and  $\delta_{AD}$

$$\begin{aligned}
\delta_E &= \delta_{AD} \\
14.458 \times 10^{-5} &= 3.886 \times 10^{-9} R_E \\
R_E &= \frac{14.458 \times 10^{-5}}{3.886 \times 10^{-9}} = 37186.21 = 37.19 KN
\end{aligned}$$

For reaction at A,  $R_A$  we have

$$\begin{aligned}
R_E + R_A &= (40 \times 10^3) + (60 \times 10^3) \\
R_A &= (100 \times 10^3) = 37186.21 \\
&= 62813.786 N
\end{aligned}$$

Deflection at point C

$$\begin{aligned}
\delta_C &= \delta_{AB} + \delta_{BC} \\
\delta_C &= \frac{R_A L_4}{A_B E_B} + \frac{P_A L_3}{A_S E_S}
\end{aligned}$$

Where  $P = R_A - 60 \times 10^{-3}$

$$\begin{aligned}
&\frac{62813.786 \times 0.18}{1.257 \times 10^{-3} \times 200 \times 10^9} + \frac{(62813.786 - 60000) \times 0.12}{1.259 \times 10^{-3} \times 200 \times 10^4} \\
&(4.497 \times 10^{-5}) + (1.34 \times 10^{-6}) = 4.63 \times 10^{-5} m
\end{aligned}$$

## Question 2.58

Solution

Given

$$\sigma = -11ksi = -11 \times 10^3$$

$$T_1 = 75^\circ F$$

$$Z_b = 14in, L_{AC} = 18in$$

$$A_B = 2.4in^2$$

$$E_B = 15 \times 10^6 psi$$

$$\alpha_B = 15 \times 10^{-6} F^{-1}$$

$$A_{AC} = 2.8in^2$$

$$E_{AC} = 10.6 \times 10^6 psi$$

$$\alpha_{AC} = 12.9 \times 10^{-6} F^{-1}$$

$$\text{Pressure, } P = -\sigma_{AL} A_{AL} = -(11 \times 10^3)(2.8) = -30800N$$

$$p_1 = P_2 = P$$

deformation as a result of force, P

$$\begin{aligned} F_P &= \frac{P_1 L_B}{A_B E_B} + \frac{P_2 L_{AL}}{A_{AL} E_{AL}} \\ &= \frac{-30800 \times 14}{2.4 \times 15 \times 10^6} + \frac{-30800 \times 18}{2.8 \times 10.6 \times 10^6} \\ &= -0.012 - 0.0187 = -0.0307in \end{aligned}$$

Deformation due to the thermal stress,

$$f_T = \delta_P + \delta = 0.0307 + 0.02 = 0.0507$$

$$\text{But } F_T = \alpha_B \Delta T L_B + \alpha_{AL} \Delta T L_{AL}$$

$$= \Delta T (\alpha_B L_B + \alpha_{AL} L_{AL})$$

$$\Delta T (12 \times 10^{-6} \times 14 + 12.9 \times 10^{-6} \times 18)$$

$$= \Delta T (4.002 \times 10^{-4})$$

Equating it  $\delta_T = 0.0507$ , we have

$$\Delta T(4.002 \times 10^{-4}) = 0.0507$$

$$\Delta T = \frac{0.0507}{4.002 \times 10^{-4}} = 126.69$$

$T_2$ , new temperature for stress of  $-11ksi$  to be experienced by aluminium =  $\Delta T + T_1$

$$= 126.69 + 75$$

$$T_2 = 201.68^\circ F$$

$$F_A = L_{AL}\alpha_{AL}(\Delta T) - \frac{PL_{AL}}{E_{AL}A_{AL}}$$

$$= (18)(2.9 \times 10^{-6})(26.6) = \frac{30800 \times 18}{10.6 \times 10^6 \times 2.8} = 10.712 \times 10^{-3}$$

$$L_{exact} = 18 + 10.712 \times 10^{-3} = 18.0107in$$

## Question 2.65

Solution

$$length, L = 2.5m$$

$$Outsidediameter, d_0 = 300mm = 0.3m$$

$$thickness, t = 15mm = 0.015m$$

$$E = 200GPa = 200 \times 10^9 Pa$$

$$v = 0.30$$

$$P = 700KN = 700 \times 10^3 N$$

- Change in length of the pipe,  $\delta$

$$\delta = \frac{-PL}{AE}$$

$$Area, A = \frac{\pi}{4} \times (d_0^2 - d_1^2)$$

$$d_1 = d_0 - 2t = 0.3 - 2(0.015) = 0.27$$

$$A = \frac{\pi}{4}(0.3^2 - 0.27^2) = 0.0134m^2$$

$$\delta = \frac{(700 \times 10^3)(2.5)}{(0.0134)(200 \times 10^9)} = -6.53 \times 10^{-4}$$

- Change in the outer diameter  $\Delta d_0$

$$\Delta d_0 = E_{lateral} \times d_0$$

but

$$E_{lateral} = -\nu E_{axial}$$

$$E_{axial} = \frac{\delta}{L} = \frac{-6.53 \times 10^{-4}}{2.5} = -2.612 \times 10^{-4}$$

$$E_{lateral} = -(0.30)(-2.612 \times 10^{-4}) = 7.836 \times 10^{-6}$$

$$\Delta d_0 = 7.836 \times 10^{-5} \times 0.3 = 2.3508 \times 10^{-5} m$$

- Change in thickness,  $\Delta t$

$$\Delta t = E_{lateral} \times t_0 = 7.836 \times 10^{-5} \times 0.015 = 11.754 \times 10^{-6} m$$

## Question 2.81

### Solution

$$G = 12 MPa = 12 \times 10^6$$

$$L = 100 mm = 0.1 m$$

$$P = 45 KN = 45 \times 10^3 N$$

$$a = ?, b = ?$$

$$\tau = 1.4 MPa = 1.4 \times 10^6$$

$$\delta = 5 mm = 5 \times 10^{-3} m$$

$$F = \frac{1}{2} P = \frac{45 \times 10^3}{2} = 22.5 \times 10^3 N$$

Finding b, bc = A but A = ?

$$\tau = \frac{F}{A} \Rightarrow A = \frac{F}{\tau} = \frac{22.5 \times 10^3}{1.4 \times 10^6} = 0.0161 m^2$$

$$bc = A \Rightarrow b = \frac{A}{C} = \frac{0.0161}{0.1} = 0.1607 m$$

Finding a,

$$r = \frac{\delta}{a} \Rightarrow a = \frac{\delta}{r}$$

$$r = \frac{\tau}{G} = \frac{1.4 \times 10^6}{12 \times 10^6} = 0.1167$$

substituting into  $a = \frac{\delta}{r}$

$$a = \frac{5 \times 10^{-3}}{0.1167} = 0.0428 m$$