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*Abstract*—This manual is an introduction to control systems based on GATE problems. Links to sample Python codes are available in the text.

## 1 STABILITY

## 1.1 Second order System

## 2 ROUTH HURWITZ CRITERION

## 2.1 Marginal Stability

## 2.2 Stability

## 3 STATE-SPACE MODEL

## 4 NYQUIST PLOT

## 5 COMPENSATORS

## 6 DAMPING RATIO AND UNDAMPED NATURAL FREQUENCY

## 6.1. Consider a state-variable model of a system

$$\begin{pmatrix} \dot{x}_1 \\ \dot{x}_2 \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -\alpha & -2\beta \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} + \begin{pmatrix} b_1 \\ b_2 \end{pmatrix} r \quad (6.1.1)$$

$$y = \begin{pmatrix} 1 & 0 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \end{pmatrix} \quad (6.1.2)$$

where  $y$  is the output, and  $r$  is the input. Find the Damping ratio  $\zeta$  and the Undamped natural frequency  $\omega_n$  of the system.

**Solution:** The state space model is given by

$$\dot{X} = AX + BU \quad (6.1.3)$$

$$Y = CX + DU \quad (6.1.4)$$

The transfer function for the state space model is:

$$H(s) = C(sI - A)^{-1}B + D \quad (6.1.5)$$

$$\Rightarrow H(s) = \frac{\begin{pmatrix} 1 & 0 \end{pmatrix} \begin{pmatrix} s + 2\beta & 1 \\ -\alpha & s \end{pmatrix} \begin{pmatrix} b_1 \\ b_2 \end{pmatrix}}{s(s + 2\beta) + \alpha} \quad (6.1.6)$$

$$= \frac{b_1(s + 2\beta) + b_2}{s^2 + 2s\beta + \alpha} \quad (6.1.7)$$

$$\Rightarrow H(s) = \frac{b_1 s}{s^2 + 2s\beta + \alpha} + \frac{2b_1\beta + b_2}{s^2 + 2s\beta + \alpha} \quad (6.1.8)$$

Generally for a second order system the trans-

Damping ratio( $\zeta$ )	Undamped natural frequency( $\omega_n$ )
Damping ratio basically indicates the amount of damping present in the overall system denoted by zeta, where damping is a counter force. It is a dimensionless measure describing how oscillations in a system decay after a disturbance.	The frequency of oscillation of the system without damping. A system may or may not have an associated natural frequency.
The damping ratio is a system parameter, denoted by $\zeta$ , that can vary from undamped ( $\zeta = 0$ ), underdamped ( $\zeta < 1$ ) through critically damped ( $\zeta = 1$ ) to overdamped ( $\zeta > 1$ ).	Only systems with $\zeta < 1$ have a natural frequency $\omega$ and only in the case that $\zeta = 0$ will the natural frequency $\omega = \omega_n$ , the undamped natural frequency.

TABLE 6.1

fer function is given by

$$H(s) = \frac{\omega_n^2}{s^2 + 2s\zeta\omega_n + \omega_n^2} \quad (6.1.9)$$

Now from the transfer function we got we can see that our system is bandpass and lowpass combination but the comparison of denominator of our transfer function to the general transfer function is still valid.

$$\therefore 2\zeta\omega_n = 2\beta, \quad (6.1.10)$$

$$\omega_n^2 = \alpha \quad (6.1.11)$$

$$\Rightarrow \zeta = \frac{\beta}{\sqrt{\alpha}}, \omega_n = \sqrt{\alpha} \quad (6.1.12)$$