

Wind Turbine Aerodynamics

B. One-Dimensional Momentum Theory

RECALL WE HAD EXPRESSION FOR P

$$P = \dot{m} \left(\frac{u_{x,1}^2 - u_{x,4}^2}{2} \right)$$

$$u_{x,4} = u_{x,1} (1 - 2a)$$

$$P = \rho u_{x,1}^3 A_2 2a(1-a)^2$$

↑
UPSTREAM VELOCITY

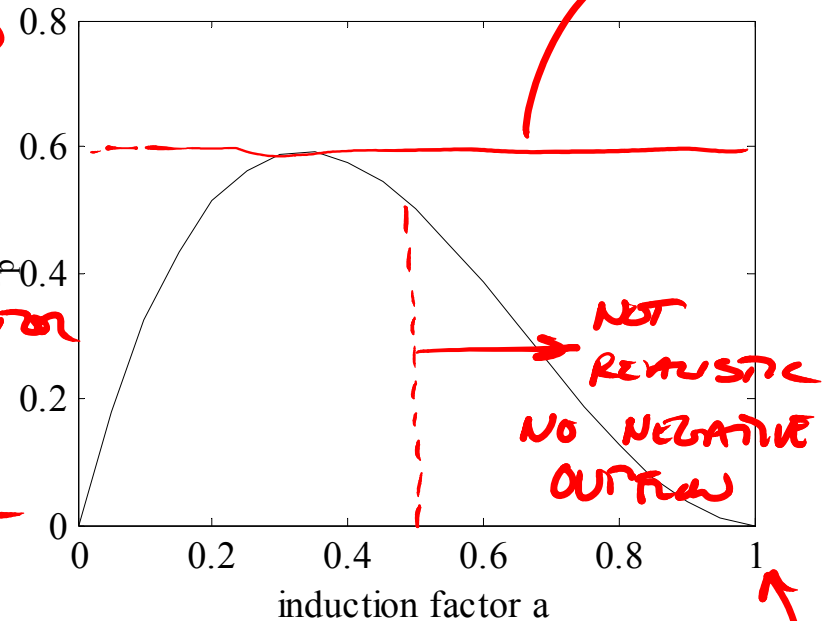
AREA OF ROTOR

EFFICIENCY

$$\eta = \frac{P}{P_{in}} = \frac{\rho u_{x,1}^3 A_2 (2a)(1-a)^2}{\frac{1}{2} \rho u_{x,1}^2 \underbrace{(u_{x,2})}_{\text{DEPENDS ON TURBINE}} A_2}$$

DEFINE POWER COEFFICIENT

$$C_p = \frac{P}{\frac{1}{2} \rho u_{x,1}^3 A_2}$$



$$C_p = \frac{P}{\frac{1}{2} \rho u_{x,1}^3 A_2} = 4a(1-a)^2$$

Wind Turbine Aerodynamics

C. Wake Rotation Effects

IN PREVIOUS ANALYSIS

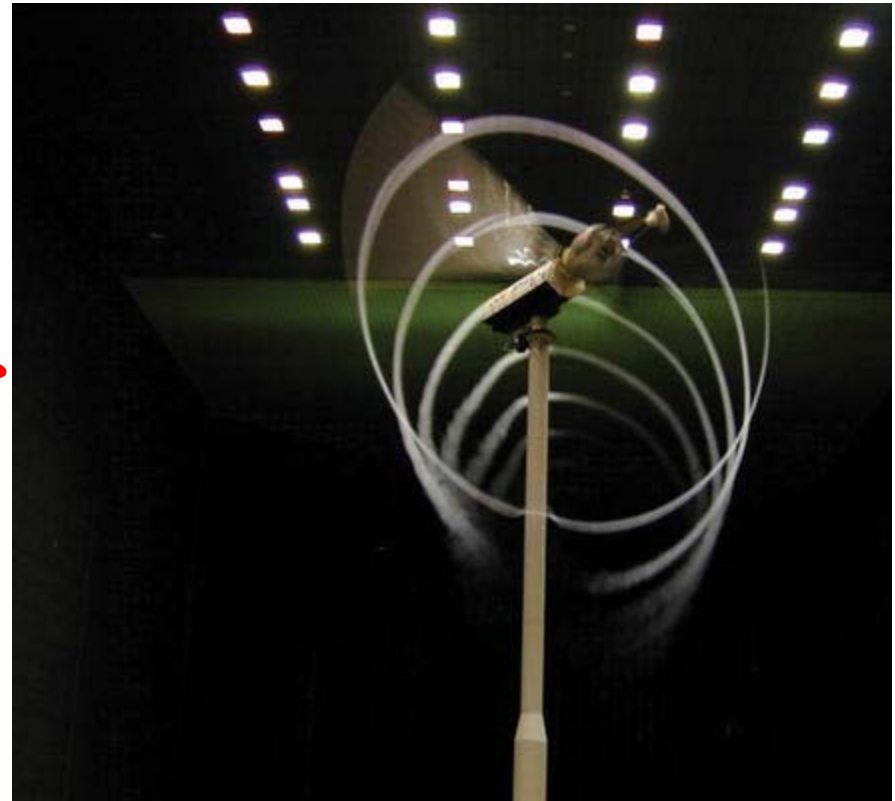
$$U_{0,y} \ll U_{x,y}$$

WE KNOW THE WIND PRODUCES
A TORQUE ON BLADES \rightarrow
BLADE EXERTS TORQUE ON
FLOW

FLOW BEHIND TURBINE ROTATES
IN DIRECTION OPPOSITE TO
BLADE ROTATION

KINETIC ENERGY IN ROTATION
RESULTS IN LESS ENERGY
EXTRACTED FROM THE FLOW

SLOW ROTATION TURBINES (LOW SPEED) / HIGH TORQUE \rightarrow MORE ROTATION
LESS EFFICIENT THAN FAST ROTATION / LOW TORQUE TURBINE



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1. Theory

CONSIDER TURBINE ROTATING
WITH ANGULAR VELOCITY Ω

AN ANGULAR VELOCITY IS
IMPARTED TO FLOW ω

THE ROTATION IMPARTED IS
NOW A FUNCTION OF r

SO PRESSURE $P_2 - P_3$

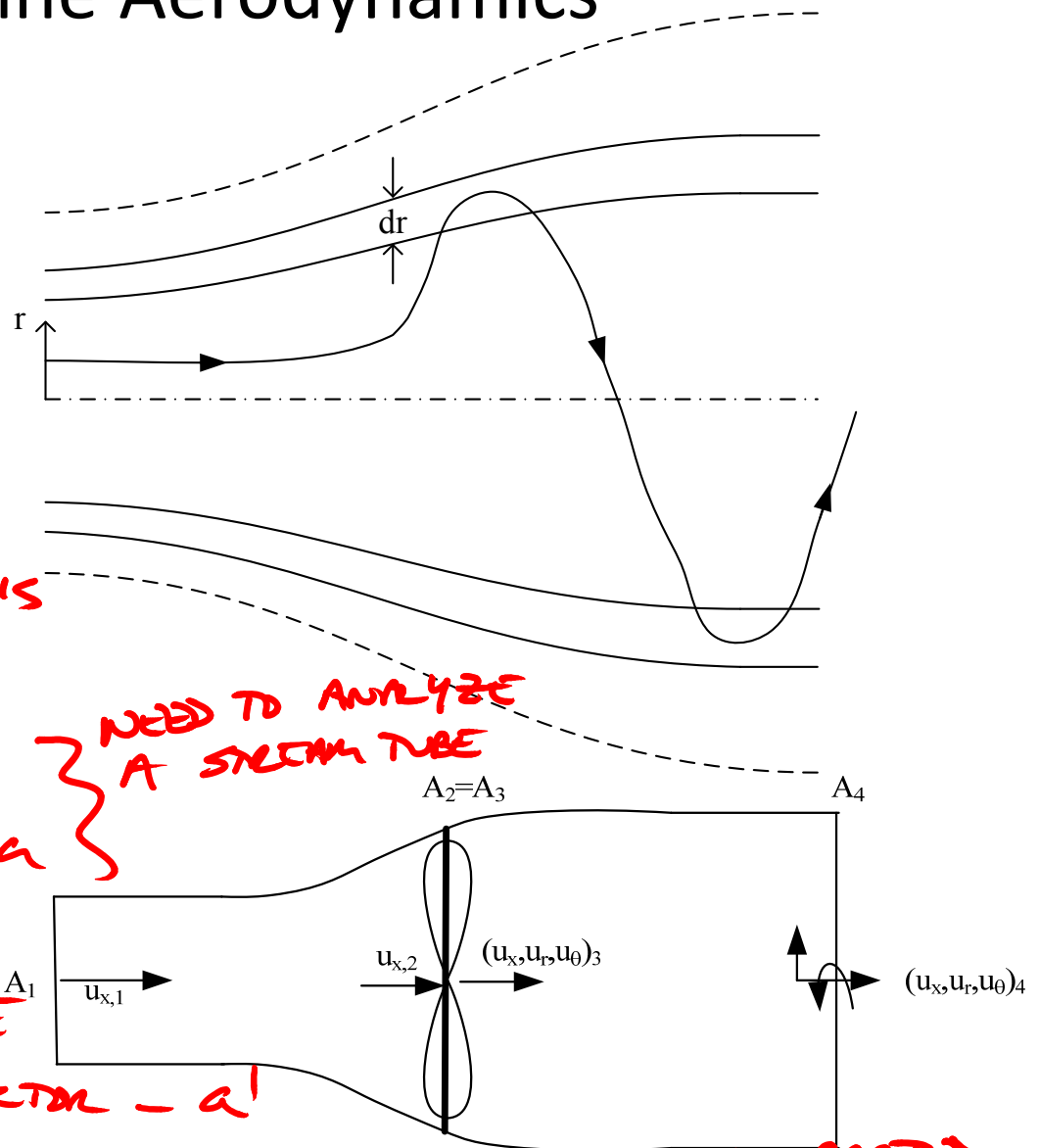
AXIAL INDUCTION FACTOR a

WILL ALSO VARY WITH r

INTRODUCE A NEW VARIABLE

ANGULAR INDUCTION FACTOR - a'

MEASURE OF ANGULAR VELOCITY IMPARTED
TO FLOW



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1. Theory

ENERGY APPLIED TO STREAM TUBE

$$\int_{CV} \rho \left(\frac{P}{\rho} + \frac{1}{2}(u_x^2 + u_\theta^2 + u_r^2) + \rho g z \right) (\vec{v} \cdot \hat{n}) dA = \dot{W} + \dot{Q}$$

$$\dot{W} = \frac{P_2 - P_3}{\rho} - \frac{1}{2} u_{\theta,3}^2$$

$$W = \frac{P_2 - P_3}{\rho} - \frac{1}{2} u_{\theta,3}^2$$

CONSIDER ANGULAR MOMENTUM

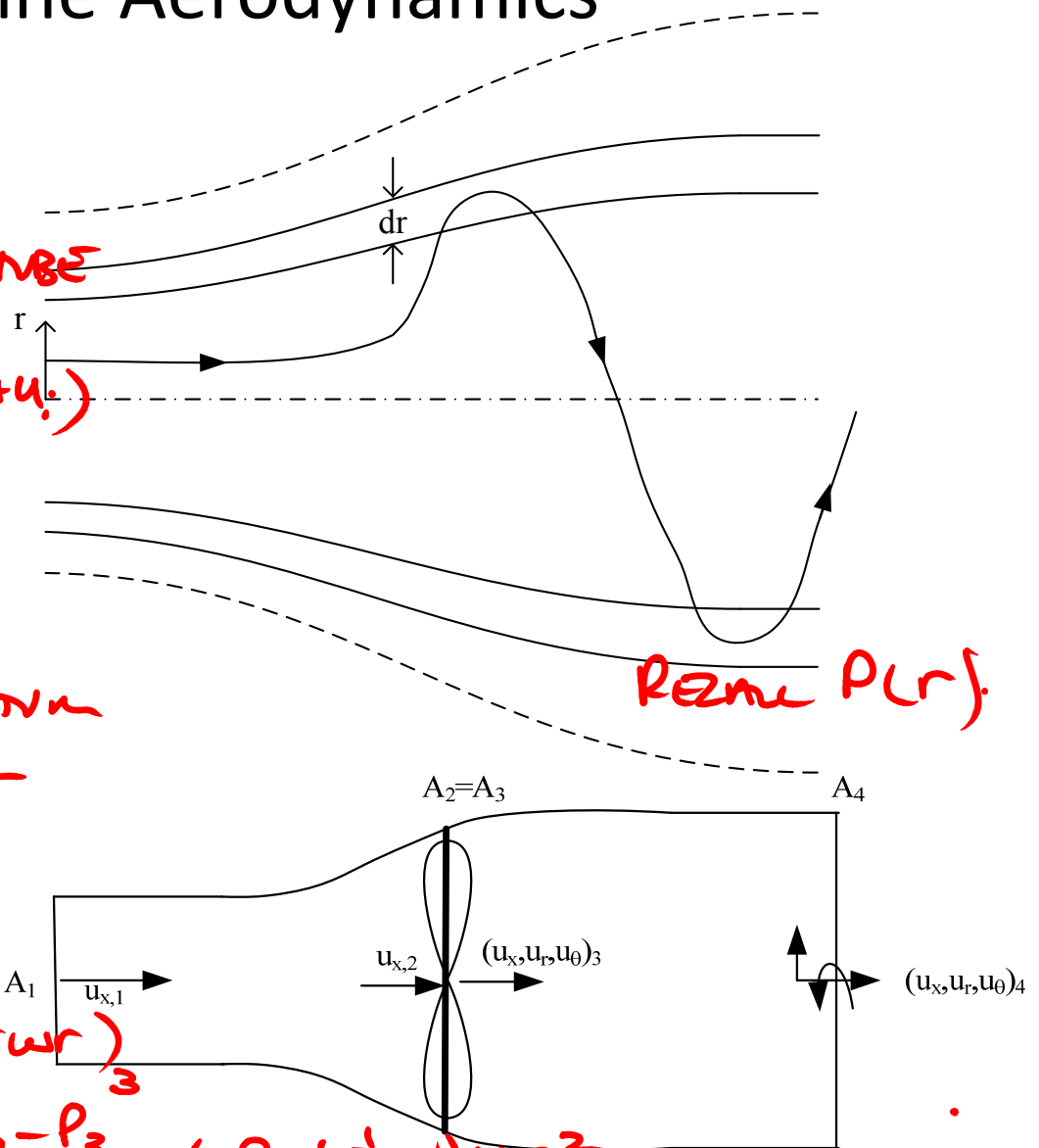
$$\int_{CV} \rho (\vec{r} \times \vec{v}) (\vec{v} \cdot \hat{n}) dA = T$$

$$\dot{W} = T\omega$$

$$W = \frac{\dot{W}}{\dot{m}}$$

$$W = - \left(\Omega r \frac{u_\theta}{r} \right)_2 + \left(\Omega r \frac{u_\theta}{r} \right)_3$$

$$\frac{P_2 - P_3}{\rho} = \left(\Omega + \frac{1}{2} \omega \right) \omega r^2$$



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1. Theory

NOW CONSIDER THE THRUST
IN EACH STREAM TUBE

$$dT = (P_2 - P_3) dA$$

$$dT = \left[\rho \left(\Omega r + \frac{1}{2} \omega \right) r^2 \omega \right] 2\pi r dr$$

DEFINE ANGULAR INDUCTION FACTOR

$$a' = \frac{\omega}{2\Omega}$$

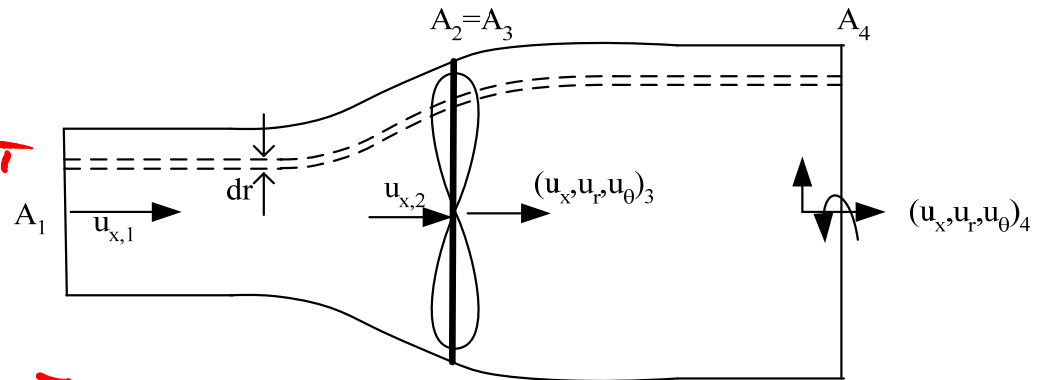
SUBSTITUTE INTO dT

$$dT = 2a'(1+a')\rho\Omega^2 r^2 2\pi r dr$$

$$\frac{a(1-a)}{a'(1+a')} = \frac{\Omega^2 r^2}{u_{1,x}^2} = \lambda_r$$

$$dT = \frac{1}{2} \rho u_{1,x}^2 4a(1-a) 2\pi r dr$$

$$\lambda_r = \frac{\text{ROTATION VELOCITY}}{\text{INFLow VELOCITY}} \rightarrow \text{LOCAL SPEED RATIO}$$



RECALL FROM EARLIER
ANALYSIS

$$dT = \frac{1}{2} \rho (u_{x,1}^2 - u_{x,4}^2) dA$$

SUB AXIAL INDUCTION
FUNCTION

$$a = \frac{u_{1,x} - u_{2,x}}{u_{1,x}}$$

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1. Theory

ONCE MORE BACK TO
ANGULAR MOMENTUM

$$\int_{CV} \rho (\vec{r} \times \vec{v}) (\vec{v} \cdot \hat{n}) dA = T_r$$

APPLY BETWEEN ② & ③ ON ANNUAL REGION

$$dT_r = \rho r u_{\theta,3} u_{x,3} 2\pi r dr$$

SUBSTITUTE INDUCTION FACTOR

$$dT_r = 4a'(1-a) \frac{1}{2} \rho u_{i,x} \Omega r^2 2\pi r dr$$

INCREMENTAL POWER FROM ANNULUS, $dP = \Omega dT_r$

$$dP = \frac{1}{2} \rho A_2 u_{i,x}^3 \left[8a'(1-a) \frac{dr^3}{\lambda^2} d\lambda \right] \quad \lambda = \frac{\Omega R}{u_{i,x}}$$

SO POWER DEPENDS ON a' , a & λ $\lambda = f(a, a')$

