

Report for ABS

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- ABS are meant to control the wheel slip in order to maintain the friction coefficient close to the optimal value
- wheel slip :- Relative motion between a wheel and the surface of the road, during vehicle movement.

consider vehicle moving in a straight direction under braking conditions.

So for horizontal direction

$$F_f = F_i \quad (1)$$

$F_f \rightarrow$ frictional force

$F_i \rightarrow$ inertial force

$$F_f = \mu \cdot m_v \cdot g \quad (2)$$

m_v - total vehicle mass

g - gravitational acceleration

$$F_i = m_v \cdot a_v \quad (a_v - \text{vehicle acceleration})$$

$$\text{where } a_v = \frac{dv_v}{dt} \quad (3)$$

\therefore vehicle acceleration (from 1, 2, 3)

$$\frac{dv_v}{dt} = \frac{1}{m_v} (\mu \cdot m_v \cdot g)$$

ABS system has to control on the wheel slip around an optimal target.

For slip,

$$S = 1 - \frac{\omega_w}{\omega_v}$$

$\omega_v \rightarrow$ equivalent angular speed of the vehicle

$$\omega_v = \frac{V_v}{r_w}$$

V_v (m/s) \rightarrow vehicle speed

- \rightarrow Friction coefficient depends on several factors like — wheel slip, vehicle speed, the type of road surface.
- For our purpose we are going to take into account only the variation on wheel slip.

Friction coefficient

Stability zone

- friction coeff increases with the wheel slip increases

unstable zone

- friction coeff decreases with the wheel slip increase

Friction coeff can be expressed as empirical function, where slip is functional argument

$$\mu(s) = A \cdot (B \cdot (1 - e^{-C \cdot s}) - D \cdot s)$$

s [-] - wheel slip

A, B, C, D [-] - empirical coefficients.

Depending upon the value of coeff A, B, C, D empirical formula can be used to represent the friction coefficient for diff road types.

	Type			
	dry	wet	snow	ice
A	0.9	0.7	0.3	0.1
B	1.07	1.07	1.07	1.07
C	0.2773	0.5	0.1773	0.38
D	0.0026	0.003	0.006	0.007

→ wheel model :-

wheel speed ω_w is initialized integrator is initialized. The linear wheel speed V_w is obtained by multiplying the angular speed with wheel radius. distance covered obtained by integrating linear speed

i/p :- braking torque T_b

friction force f_t

o/p :- angular wheel speed ω_w

linear wheel speed V_w

wheel distance d_w

→ Controller :-

bang-bang type controller, reaching on wheel slip feedback. max value of friction coeff is obtained around slip of 20% (0.2). Thus target is 0.2. Slip error is the diff between actual slip & target slip.

Hydraulic system modeled as first order transfer function, amplification factor K

& time const is T . o/p is braking torque

T_b accumulated over time by integrator.

- $iip \rightarrow$ wheel slip $S[-]$

$slp \rightarrow$ vehicle speed $V_v [m/s]$

\rightarrow Simulation :-

- Simulation run for 20 sec and results are stored in data inspector.

- When ABS disabled braking torque ramps up to maximum value & causes slip.

With ABS braking torque modulated to maintain optimal slip ratio.

- When ABS deactivated wheel slip climbs to 1 as torque increases. When ABS is activated slip is controlled by controlling braking torque.

- \therefore When ABS is deactivated wheel locks before coming to complete halt. With ABS active wheel is prevented from locking thus reducing slip.