ENPH270: Mechanics II

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16 Planar Kinematics of a Rigid Body

16.1 Planar Rigid-Body Motion

planar motion: motion such that all particles move along paths equidistant from a fixed plane

16.2 Translation

translation: planar motion such that all lines remain parallel to their original orientations rectilinear translation: translation such that all points move along parallel paths curvilinear translation: translation such that all points move along curved paths base point: the origin of a relative coordinate system

The kinematic equations of point B and base point A on a translating rigid body are:

$$\mathbf{r}_{B} = \mathbf{r}_{A} + \mathbf{r}_{B/A}$$

$$\mathbf{v}_{B} = \mathbf{v}_{A}$$

$$\mathbf{a}_{B} = \mathbf{a}_{A}$$
(1)

16.3 Rotation about a Fixed Axis

rotation: planar motion such that all points travel in circular paths about a fixed axis. The kinematic equations for rotation with constant angular acceleration are:

$$\omega = \omega_0 + \alpha t$$

$$\theta = \theta_0 + \omega_0 t + \frac{1}{2} \alpha t^2$$

$$\omega^2 = \omega_0^2 + 2\alpha (\theta - \theta_0)$$
(2)

Let \mathbf{r}_P be the vector from any point on the axis of rotation to the position of the rotated point, \mathbf{a}_t be the tangential acceleration, and \mathbf{a}_n be the normal acceleration. Then the kinematic equations for general rotation about a fixed axis are:

$$\alpha d\theta = \omega d\omega$$

$$\mathbf{v} = \boldsymbol{\omega} \times \mathbf{r}_{P}$$

$$\mathbf{a} = \mathbf{a}_{t} + \mathbf{a}_{n} = \boldsymbol{\alpha} \times \mathbf{r}_{P} - \omega^{2} \mathbf{r}_{P}$$
(3)

16.4 Absolute Motion Analysis

general plane motion: planar motion that combines translation and rotation

Analyze general plane motion by relating a rectilinear position s to an angular position θ by geometry and trigonometry, then differentiate to relate velocity v to angular velocity ω .

centre of mass (G): the one point in a body such that all external forces whose lines of action pass through it create no rotation of the body

At the centre of mass of a circular object that rolls without slipping:

$$s_G = \theta r$$
 $v_G = \omega r$ $a_G = \alpha r$ (4)

16.5 Relative-Motion Analysis: Velocity

Let A be the base point of a translating coordinate system and ω be the angular velocity of B about A. Then the velocity of point B is:

$$\mathbf{v}_B = \mathbf{v}_A + \boldsymbol{\omega} \times \mathbf{r}_{B/A} \tag{5}$$

16.6 Instantaneous Center of Zero Velocity

instantaneous centre of zero velocity: a base point with zero velocity

On a rigid body, the velocity of every point is perpendicular to their displacement from the instantaneous centre of zero velocity, and every point purely rotates about it:

$$\mathbf{v}_B = \boldsymbol{\omega} \times \mathbf{r}_{B/IC} \tag{6}$$

centrode: the path traced by the instantaneous centre of zero velocity as it moves over time

16.7 Relative-Motion Analysis: Acceleration

Let A be the base point of a translating coordinate system, ω be the angular velocity, and α be the angular acceleration of B about A. Then the acceleration of point B is:

$$\mathbf{a}_B = \mathbf{a}_A + (\mathbf{a}_{B/A})_t + (\mathbf{a}_{B/A})_n = \mathbf{a}_A + \alpha \times \mathbf{r}_{B/A} - \omega^2 \mathbf{r}_{B/A}$$
 (7)

16.8 Relative-Motion Analysis using Rotating Axes

Let A be the base point of a translating and rotating coordinate system, $(\mathbf{v}_{B/A})_{rel}$ and $(\mathbf{a}_{B/A})_{rel}$ be the velocity and acceleration of B inside the coordinate system, and ω and α be the angular velocity and acceleration of the coordinate system. Then the kinematic equations of point B are:

$$\mathbf{r}_{B} = \mathbf{r}_{A} + \mathbf{r}_{B/A}$$

$$\mathbf{v}_{B} = \mathbf{v}_{A} + \boldsymbol{\omega} \times \mathbf{r}_{B/A} + (\mathbf{v}_{B/A})_{rel}$$

$$\mathbf{a}_{B} = \mathbf{a}_{A} + \boldsymbol{\alpha} \times \mathbf{r}_{B/A} - \boldsymbol{\omega}^{2} \mathbf{r}_{B/A} + 2\boldsymbol{\omega} \times (\mathbf{v}_{B/A})_{rel} + (\mathbf{a}_{B/A})_{rel}$$
(8)

Coriolis acceleration: the $2\omega \times (\mathbf{v}_{B/A})_{rel}$ term of acceleration in a rotating reference frame

17 Planar Kinetics of a Rigid Body: Force and Acceleration

17.1 Mass Moment of Inertia

mass moment of inertia (I): a measure of the resistance of a body to angular acceleration Let r be the moment arm and ρ be the density of a body. Then the mass moment of inertia is the integral of the second moment of mass about some axis:

$$I = \int_{m} r^2 \, \mathrm{d}m = \int_{V} r^2 \rho \, \mathrm{d}V \tag{9}$$

The usual mass moment of inertia I_G is about the axis normal to the plane of motion that passes through the centre of mass.

parallel-axis theorem: the moment of inertia about a parallel axis d away is:

$$I = I_G + md^2 (10)$$

radius of gyration (k): a measure of the resistance of a shape to angular acceleration

$$I = mk^2 (11)$$

17.2 Planar Kinetic Equations of Motion

The kinetic equations of a rigid body in general plane motion are:

$$\mathbf{F}_R = m\mathbf{a}_G$$

$$(\mathbf{M}_R)_G = I_G \boldsymbol{\alpha}$$
(12)

17.3 Equations of Motion: Translation

The kinetic equations of a translating rigid body are:

$$(F_R)_x = m(a_G)_x$$
 $(F_R)_n = m(a_G)_n$
 $(F_R)_y = m(a_G)_y$ $(F_R)_t = m(a_G)_t$
 $(M_R)_G = 0$ (13)

17.4 Equations of Motion: Rotation about a Fixed Axis

The kinetic equations of a rigid body rotating about a fixed axis O are:

$$(F_R)_n = m\omega^2 r_G$$

$$(F_R)_t = m\alpha r_G$$

$$(M_R)_G = I_G \alpha$$

$$(M_R)_O = I_O \alpha$$
(14)

17.5 Equations of Motion: General Plane Motion

The kinetic equations of a rigid body in general plane motion at a point A are:

$$\mathbf{F}_{R} = m\mathbf{a}_{G}$$

$$(\mathbf{M}_{R})_{A} = I_{G}\boldsymbol{\alpha} + \mathbf{r}_{G/A} \times (m\mathbf{a}_{G}) = I_{A}\boldsymbol{\alpha} + \mathbf{r}_{G/A} \times (m\mathbf{a}_{A}) = I_{IC}\boldsymbol{\alpha}$$
(15)

18 Planar Kinetics of a Rigid Body: Work and Energy

18.1 Kinetic Energy

The kinetic energy of a rigid body rotating about a fixed axis O is:

$$T = \frac{1}{2}I_O\omega^2 \tag{16}$$

The kinetic energy of a rigid body in general plane motion is:

$$T = \frac{1}{2}mv_G^2 + \frac{1}{2}I_G\omega^2 = \frac{1}{2}I_{IC}\omega^2$$
 (17)

18.2 The Work of a Force

The work done by an external force \mathbf{F} is:

$$U_F = \int \mathbf{F} \cdot d\mathbf{r} \tag{18}$$

The work done by a constant force with magnitude F and direction θ over a displacement s is:

$$U_F = (F\cos\theta) s \tag{19}$$

The work done by a weight W over a vertical displacement Δy is:

$$U_W = -W\Delta y \tag{20}$$

The work done by a spring force with spring constant k as it lengthens from s_1 to s_2 is:

$$U_s = -\left(\frac{1}{2}ks_2^2 - \frac{1}{2}ks_1^2\right) \tag{21}$$

Forces that act at fixed points on the body or perpendicular to their displacement do no work.

18.3 The Work of a Couple Moment

The work done by an external couple moment M is:

$$U_M = \int M \, \mathrm{d}\theta \tag{22}$$

The work done by a constant couple moment from θ_1 to θ_2 is:

$$U_M = M\left(\theta_2 - \theta_1\right) \tag{23}$$

18.4 Principle of Work and Energy

principle of work and energy: the work done on a rigid body is its change in energy

$$T_1 + \sum U_{1-2} = T_2 \tag{24}$$

18.5 Conservation of Energy

The gravitational potential energy of a weight W at a height y_G above the horizontal datum is:

$$V_q = W y_G \tag{25}$$

The elastic potential energy imparted by a spring with spring constant k and displacement s is:

$$V_e = \frac{1}{2}ks^2 \tag{26}$$

conservation of energy: if only conservative forces do work, then energy is conserved

$$T_1 + V_1 = T_2 + V_2 (27)$$