# MECH 360 Formula Sheet

## Stress & Strain

Average normal stress:

$$\sigma = \frac{P}{A}$$

Average shear stress:

$$\tau = \frac{V}{\Delta}$$

Double shear:

$$\tau = \frac{P}{2A}$$

Bearing stress:  

$$\sigma_b = \frac{P}{A} = \frac{P}{td}$$
Stresses on a 2-force member

( $\theta$  measured from vertical):

$$\sigma = \frac{P}{A_{\perp}} \cos^2 \theta \qquad \tau = \frac{P}{A_{\perp}} \sin \theta \cos \theta$$
Factor of safety:

$$\tau = \frac{P}{A_{\perp}} \sin \theta \cos \theta$$

Factor of safety.

Factor of safety = 
$$\frac{\text{Ultimate Load}}{\text{Allowable Load}}$$

Normal strain:

 $\epsilon = \frac{\delta}{L} = \frac{d\delta}{dx}$ 

Local shear strain (Change of  $\pi/2$ ):

$$\epsilon = \frac{\delta}{L} = \frac{d\delta}{dx}$$

$$\gamma = \pi/2 - \theta$$

## **Axial Load**

Hooke's Law and Modulus of Elasticity:

$$\sigma = E\epsilon$$

Elastic deformation under axial loading:

$$\delta = \frac{FL}{AE} = \sum_{i} \frac{F_{i}L_{i}}{A_{i}E_{i}}$$

Temperature change:

$$\delta_T = L_{\alpha} \alpha \Delta T$$

Poisson's Ration:

$$\nu = -\frac{\epsilon_{\text{lat}}}{\epsilon_{\text{long}}}$$

Shear Stress-Strain Diagrams:  

$$G = \frac{E}{2(1 + \nu)}$$

$$\tau = G\gamma$$
 (elastic region)

Elastic Strain Energy:

$$u = \int_0^\sigma \sigma \, d\epsilon = \frac{1}{2} \frac{\sigma^2}{E}$$

## **Torsion**

Polar Moment of Inertia:

ar Moment of Inertia:  

$$J = \int r^2 dA$$

$$J = \frac{\pi c^4}{2}$$
 (full tube)

$$J = \frac{\pi}{2}(c^4 - a^4)$$
 (hollow tube)

$$au = \frac{T\rho}{J}$$
  $au_{\text{max}} = \frac{T\rho}{J}$ 

$$P = T\omega$$

Angle of Twist:

$$\phi = \frac{TL}{JG} = \int_0^L \frac{T(x)}{J(x)G(x)} dx$$

Stress Concentrations:  $\tau_{\text{max}} = K \frac{Tc}{I}$ 

$$\tau_{\text{max}} = K^{\frac{T}{2}}$$

## Bending

Distributed Load Intensity at each point:

$$w = \frac{dV}{dx}$$

Shear at each point:

$$V = \frac{dM}{dx}$$

Normal Strain:

$$\epsilon_x = -\frac{y}{p} = -\frac{y}{c}\epsilon_{\text{max}}$$
Normal Stress:

$$\sigma = -\frac{y}{c}\sigma_{\text{max}}$$

$$\sigma = \frac{My}{I}$$

$$\sigma_{\text{max}} = \frac{Mc}{I}$$

Second Moment of Inertia:

$$I = \int y^2 \, dA$$

Circle: 
$$I = \frac{\pi}{4}r^4$$

Rectangle: 
$$I = \frac{1}{12}bh^3$$

Neutral Axis:

$$\int y dA = 0$$

Section Modulus:

$$S = I/c$$

Parallel Axis Theorem:

$$I_{\parallel} = I_G + Md^2$$

## **Stress Transformations**

$$\sigma_x' = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\sigma_y' = \frac{\sigma_x + \sigma_y}{2} - \frac{\sigma_x - \sigma_y}{2} \cos 2\theta - \tau_{xy} \sin 2\theta$$

$$\tau_{x'y'} = -\frac{\sigma_x - \sigma_y}{2} \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

Maximum In-Plane Shear Stress:

$$\tau_{\text{max}} = R = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

Angle of Principal In-Plane Stresses:

$$\tan 2\theta_p = \frac{2\tau_{xy}}{(\sigma_x - \sigma_y)}$$

Angle of Maximum In-Plane Stresses:

$$\tan 2\theta_s = -\frac{(\sigma_x - \sigma_y)}{2\tau_{xy}}$$

3D Analysis:

$$\tau_{\text{max}} = \frac{1}{2} |\sigma_{\text{max}} - \sigma_{\text{min}}|$$

## Theories of Failure

Maximum-Shearing-Stress Criterion for a yield strength  $\sigma_Y$  and principal stresses  $\sigma_a$ ,  $\sigma_b$ :

Same sign:  $|\sigma_a| < \sigma_Y$  and  $|\sigma_b| < \sigma_Y$ 

Opposite sign: 
$$|\sigma_a - \sigma_b| < \sigma_Y$$

Distortion Energy per unit volume:

$$u_d = \frac{1}{6G}(\sigma_a^2 - \sigma_a \sigma_b + \sigma_b^2)$$

Maximum Distortion Energy Criterion:

$$\sigma_a^2 - \sigma_a \sigma_b + \sigma_b^2 < \sigma_V^2$$

## Thin-Walled Pressure Vessel Stress

## Cylindrical

Hoop Stress:

$$\sigma_1 = \frac{pr}{t}$$

Longitudinal Stress:

$$\sigma_2 = \frac{pr}{2t}$$

Maximum In-Plane Shearing Stress:

$$\tau_{\text{max}} = \frac{1}{2}\sigma_2 = \frac{pr}{4t}$$

Maximum Out-of-Plane Shearing Stress (45° rotation around a longitudinal axis):

$$\tau_{\max} = \frac{pr}{2t}$$

#### Circle

$$\sigma_1 = \sigma_2 \frac{pr}{2t}$$

Maximum In-Plane Shearing Stress:

$$\tau_{\text{max}} = 0$$
 (reduces to a point)

Maximum Out-of-Plane Shearing Stress (45°rotation around a longitudinal axis):

$$\tau_{\text{max}} = \frac{1}{2}\sigma_1 = \frac{pr}{4t}$$

Updated October 2, 2018

https://github.com/DonneyF/formula-sheets