

Kinematic Model of Ackerman Steering Vehicles

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Relation between the Control Input and Speed of Wheels

Let:

- v denote the linear velocity of the vehicle;
- v_r the linear velocity of the right wheel;
- v_l the linear velocity of the left wheel;
- ω (omega) the angular velocity of the vehicle;
- ϕ (phi) the steering angle of the vehicle;
- L the distance between front and rear axles (or wheel base);
- l the distance between left and right wheels.

Figure 1 shows the approximate kinematic model of an Ackerman steering vehicle.

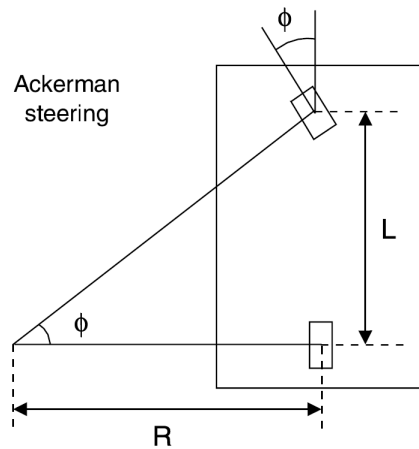


Figure 1: Approximate kinematic model of an Ackerman steering vehicle [SOT03].

The linear velocity of the right wheel, v_r , and the linear velocity of the left wheel, v_l , can be computed from the linear velocity of the vehicle, v , and the steering angle of the vehicle, ϕ , as follows. The linear velocity of the vehicle, v , is given by:

$$v = \frac{v_r + v_l}{2} \quad (1)$$

The angular velocity of the vehicle, ω , is given by:

$$\omega = \frac{v_r - v_l}{l} \quad (2)$$

The steering angle of the vehicle, ϕ , is given by:

$$\phi = \arctan\left(L \frac{\omega}{v}\right) \quad (3)$$

Isolating v_r in Equation 1:

$$v = \frac{v_r + v_l}{2} \rightarrow v_r + v_l = 2v \rightarrow v_r = 2v - v_l \quad (4)$$

Isolating v_r in Equation 2:

$$\omega = \frac{v_r - v_l}{l} \rightarrow v_r - v_l = \omega l \rightarrow v_r = \omega l + v_l \quad (5)$$

Isolating ω in Equation 3:

$$\phi = \arctan\left(L \frac{\omega}{v}\right) \rightarrow \tan(\phi) = L \frac{\omega}{v} \rightarrow \omega = \frac{v \tan(\phi)}{L} \quad (6)$$

Equating the Equations 4 and 5 and isolating v_l :

$$2v - v_l = \omega l + v_l \rightarrow 2v_l = 2v - \omega l \rightarrow v_l = \frac{2v - \omega l}{2} \quad (7)$$

Substituting the Equation 6 in Equation 7, we finally obtain the v_l from v and ϕ :

$$v_l = \frac{2v - \omega l}{2} \rightarrow v_l = \frac{2v - \left(\frac{v \tan(\phi)}{L}\right)l}{2} \rightarrow v_l = v - \frac{lv \tan(\phi)}{2L} \quad (8)$$

Isolating v_r in Equation 1:

$$v = \frac{v_r + v_l}{2} \rightarrow v_r + v_l = 2v \rightarrow v_r = 2v - v_l \quad (9)$$

Substituting Equation 8 in Equation 9, we finally obtain v_r from v and ϕ :

$$v_r = 2v - v_l \rightarrow v_r = 2v - \left(v - \frac{lv \tan(\phi)}{2L}\right) \rightarrow v_r = v + \frac{lv \tan(\phi)}{2L} \quad (10)$$

The steering angle of the vehicle, ϕ , can be computed from the linear velocity of the right wheel, v_r , and the linear velocity of the left wheel, v_l , by substituting the Equations 1 and 2 in Equation 3:

$$\phi = \arctan\left(L \frac{\omega}{v}\right) \rightarrow \phi = \arctan\left(L \frac{\left(\frac{v_r - v_l}{l}\right)}{\left(\frac{v_r + v_l}{2}\right)}\right) \rightarrow \phi = \arctan\left(\frac{2L(v_r - v_l)}{l(v_r + v_l)}\right) \quad (11)$$

Dynamics of the Global Position and Orientation Variables

Let:

- (x, y) denote the pose of the vehicle in a global frame of coordinates;
- θ the orientation of the vehicle in a global frame of coordinates;

The dynamics of the global position and orientation variables, (x, y, θ) , are given by:

$$\frac{dx}{dt} = v \cos(\theta) \quad (12)$$

$$\frac{dy}{dt} = v \sin(\theta) \quad (13)$$

$$\frac{d\theta}{dt} = v \frac{\tan(\phi)}{L} \quad (14)$$

References

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