# Multiphase flows Lecture 2

Equation of motion of a single particle, forces on individual particles

Lagrangian framework: evolution of properties associated with the *i-th* particle (*X, V, R (radius)* – Lagrangian coordinates)

$$\frac{\mathrm{d}\boldsymbol{X}_{(i)}}{dt} = \boldsymbol{V}_{(i)} \qquad \frac{\mathrm{d}\boldsymbol{V}_{(i)}}{dt} = \boldsymbol{A}_{(i)} \qquad \frac{\mathrm{d}\boldsymbol{R}_{(i)}}{dt} = \boldsymbol{\Theta}_{(i)}$$

 $A_{(i)}$  – acceleration experienced by the particle

 $\Theta_{(i)}$  - rate of change of radius due to interphase mass transfer (e.g. vaporization)

### Lagrangian vs. Eulerian reference frames

Eulerian reference frame

Lagrangian reference frame

$$v(x,t)$$
 $p(x,t)$ 

Velocity v and pressure p location x at time t

• Position  $\boldsymbol{X}$  of element  $\boldsymbol{a}$  at time t

$$v(X(a,t),t) = \frac{\partial X}{\partial t}(a,t)$$

Relation between the Eulerian and the Lagrangian descriptions

# Assumptions: Incompressible flow, translational, unbounded, small sphere, slow variation of characteristics, $Re_P << 1$

Maxey and Riley (1983)

$$m_{P} \frac{d\mathbf{u}_{P}}{dt} = \sum \mathbf{F}_{i} \iff m_{P} \frac{d\mathbf{u}_{P}}{dt} = -\int_{S_{P}} \boldsymbol{\sigma}_{fij} n_{pj} dS + m_{P} \mathbf{g}$$

$$m_{P} \frac{d\mathbf{u}_{P}}{dt} = \sum_{i} \mathbf{F}_{i} \iff m_{P} \frac{du_{P,i}}{dt} = \oint_{S} \left[ -p \delta_{ij} + \mu \left( \frac{\partial u_{i}}{\partial x_{j}} + \frac{\partial u_{j}}{\partial x_{i}} \right) \right] n_{j} dS + m_{P} g_{i}$$

Stress tensor of the fluid phase (to be evaluated on the surface of the sphere)

## A possible way to go in the derivation - Decomposition of the stress tensor

$$-\int_{S_p} \sigma_{fij} n_{1j} dS = \int_{S_p}^o \sigma_{fij} n_{2j} dS + \int_{S_p} \delta \sigma_{fij} n_{2j} dS$$

- 1- Stress tensor on a surface of a particle (if the particle were not present)
- 2 Perturbation (due to the presence of a particle)

## Contributions – identification of different forces

- Drag force
- Lift force
- History force
- Added mass force
- Pressure gradient force
- Force due to Brownian motion

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### Important to have in mind:

- Analytical solution available only for the Stokes regime
- Consideration of heat and mass transfer requires solution of two additional PDE (e.g. droplet diameter and droplet temperature)
- What happens for higher particle Re numbers?

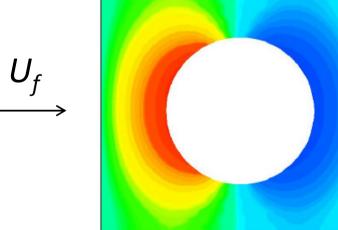
### Fluid – particle interaction force (normal and tangential component)

$$F_{\text{fluid},z} = F_{n,z} + F_{t,z}$$

$$= \int_0^{2\pi} \int_0^{\pi} (-p|_{r=R} \cos \theta) R^2 \sin \theta d\theta d\varphi$$

$$+ \int_0^{2\pi} \int_0^{\pi} (\tau_{r\theta}|_{r=R} \sin \theta) R^2 \sin \theta d\theta d\varphi,$$

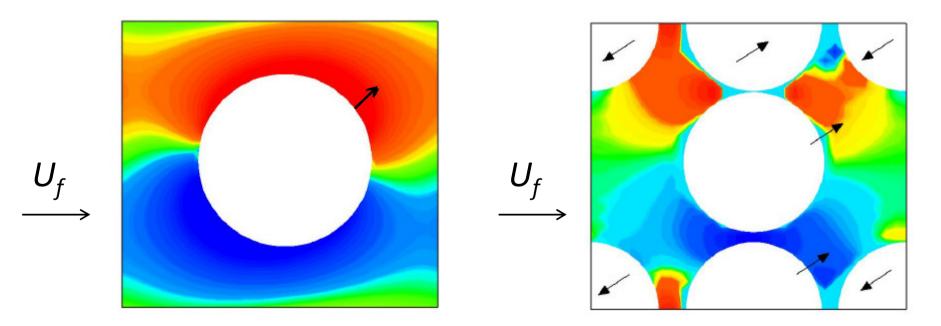
Distribution of pressure – flow over a stationary particle



## What happens in other cases (contours of pressure are shown)?

A particle moving in a different direction than the mean fluid flow

Collection of particles moving in a different direction than the mean fluid flow



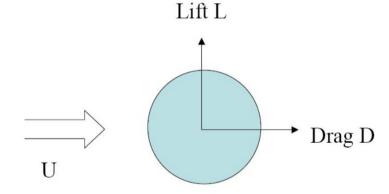
### **Drag force**

For highly viscous flows: Viscous drag (Stokes law)

$$F_D = 3\pi \,\mu_f \,d_P \big(\mathbf{u}_f - \mathbf{u}_P\big)$$

Form drag:  $1/3 F_D$ 

Skin drag:  $2/3 F_D$ 



### **Drag force**

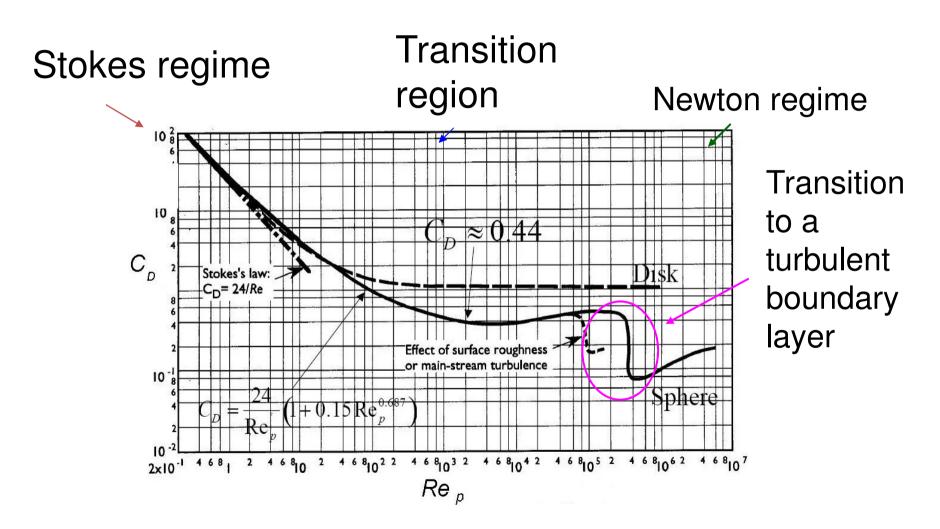
Generalized expression

$$F_{D} = \frac{1}{2} \rho_{f} \frac{d_{p}^{2} \pi}{4} C_{D} |u_{f} - u_{p}| (u_{f} - u_{p})$$

Comparison with the Stokes drag

$$C_D = \frac{24}{Re_P} \qquad Re_P = \frac{\rho_f (u_f - u_P) d_p}{\mu_f}$$

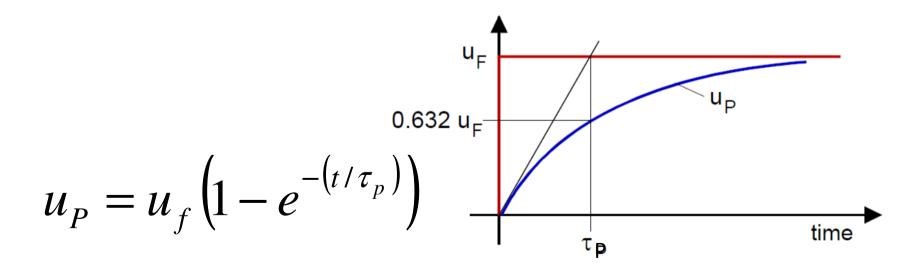
### **Drag force**: friction + form drag



### Particle response time - derivation

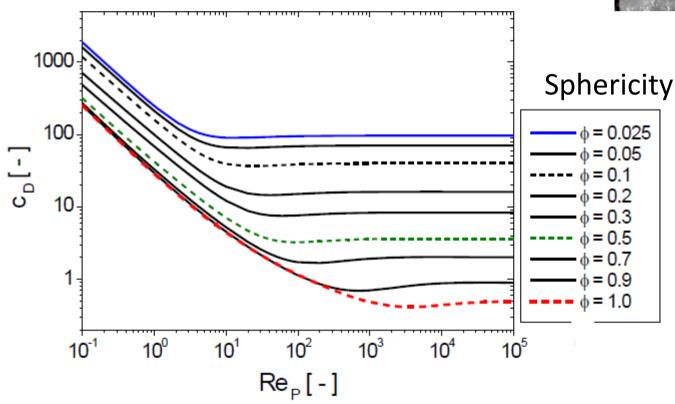
$$m_{P} \frac{du_{P}}{dt} = \frac{1}{2} \rho_{f} \frac{d_{p}^{2} \pi}{4} C_{D} |u_{f} - u_{p}| (u_{f} - u_{p})$$

$$\frac{du_P}{dt} = \frac{18\mu_f}{\rho_p d_p^2} \frac{C_D \operatorname{Re}_p}{24} \left( u_f - u_p \right) \qquad \Longrightarrow \qquad \frac{du_P}{dt} = \frac{\left( u_f - u_p \right)}{\tau_p}$$



### Drag force - Effect of particle shape?



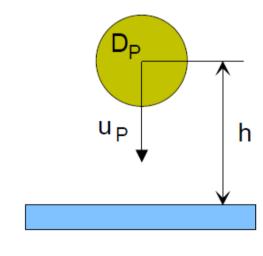


### Drag force - Effect of wall?



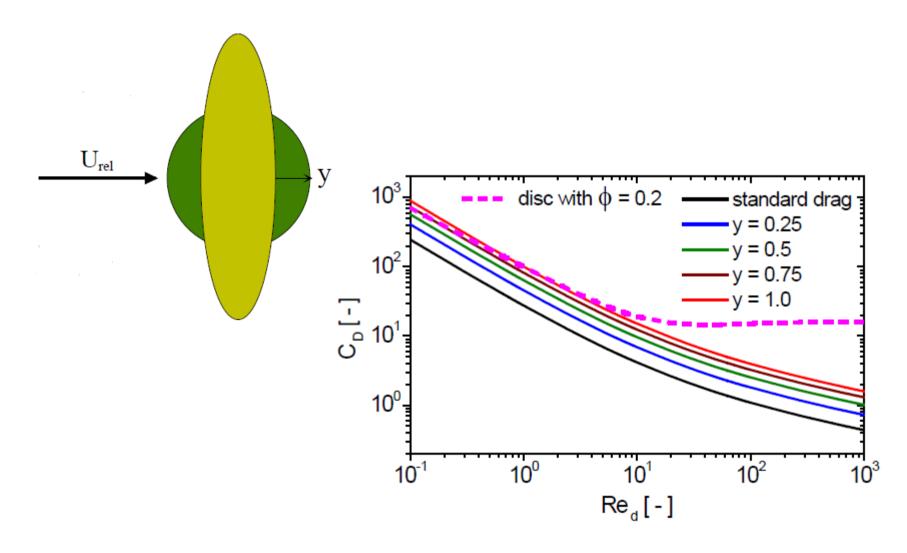
Drag force increases – introduction of correction coefficients in modelling

$$\frac{C_D}{C_{D,Stokes}} = 1 + const \frac{d_p}{h}$$

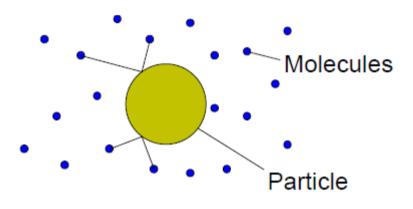


wall

### Drag force – Oscillating and/or distorted particles

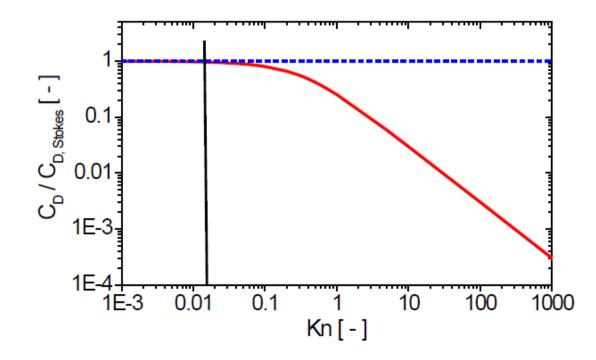


### Drag force – what if the flow is not a continuum?



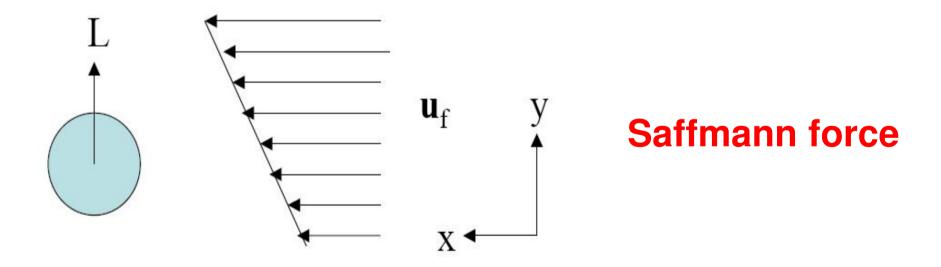
A particle "sees" individual molecules

Kn – particleKnudsen number



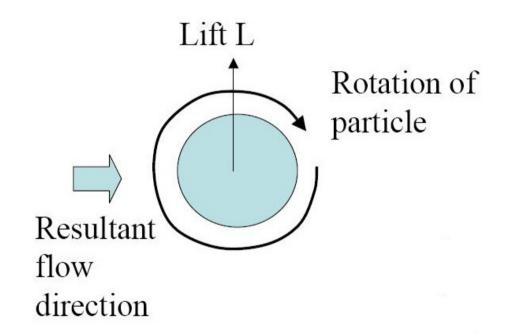
### Transverse forces

- if the flow is non uniform presence of a velocity gradient
- if the particle is rotating (velocity gradient, collisions)
- if the particle moves in the vicinity of a wall



### Transverse forces

### **Magnus force**



**Saffmann lift force** – presence of a velocity gradient in the direction normal to the main flow

$$\mathbf{F}_{saff} = 1.61 d_P^2 \sqrt{\rho_f \mu_f} \frac{(\mathbf{u}_f - \mathbf{u}_P) \times \mathbf{\omega}_f}{\sqrt{|\boldsymbol{\omega}_f|}}; \quad \mathbf{\omega}_f = rot \, \mathbf{u}_f$$

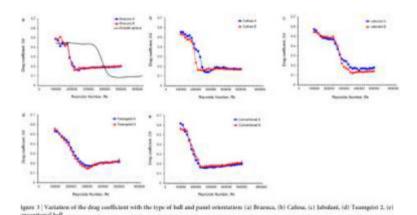
Magnus lift force – particle rotating in a fluid flow (rotation introduced due to, e.g., collisions)

$$\mathbf{F}_{mag} = \frac{\pi}{8} d_P^3 \, \boldsymbol{\rho}_f \left( \frac{1}{2} \nabla \times \mathbf{u}_f - \boldsymbol{\omega}_P \right) \times \left( \mathbf{u}_f - \mathbf{u}_P \right)$$

### Magnus force example: Freekicks!



Figure 1 | Photograph of the wind tunnel test setup.



Two identical freekicks made at the 2013 FIFA Confederations 25m from the goal with velocity 30 m/s would reach the goal three meters from each other in the vertical direction. Why?

Because the ball was rotated 45 degrees before the second freekick

Hong & Asai, Scientific Reports 4 (2014)



Figure 2 | Soccer balls used for the test and their panel orientations. (a, b) Adidas Brazuca: small dimple and six panels, (c, d) Adidas Cafusa: small grip texture and 32 modified panels, (e, f) Adidas Jabulani: small ridges or protrusions and eight panels, (g, h) Adidas Teamgeist 2: small protuberances and 14 panels; (i, j) Molten Vantaggio (conventional soccer ball): smooth surface and 32 pentagonal and hexagonal panels. (Photo by

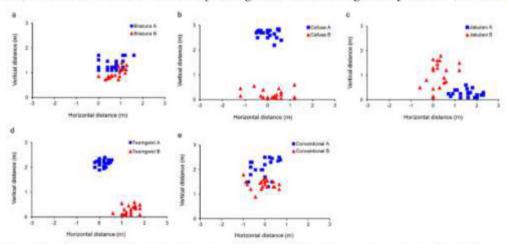


Figure 7 | Comparison of the flight characteristics (points of impact) of the different balls for different panel orientations (initial launch velocity of

### Added mass force – acceleration of a certain fraction of the surrounding fluid

$$F_A = \frac{1}{2} m_P \frac{\rho_f}{\rho_P} \frac{d}{dt} (u_f - u_p)$$

Inertia added to the system

When can this force be neglected?

History (Basset) force – delay in the boundary layer development with changing relative velocity

$$F_{H} = \sqrt{\pi \rho_{f} \mu_{f}} \frac{m_{P}}{\rho_{P} d_{P}} \int_{0}^{t} \frac{1}{\sqrt{1-\tau}} \frac{d}{dt} \left( \mathbf{u}_{f} - \mathbf{u}_{P} \right) d\tau$$

Integration along the entire trajectory for each time step of calculation

### **History (Basset) force**

Inclusion of the history force transforms Newton's second law for the particle from an ODE to an integro-differential equation (not explicit in  $u_p$  or  $du_p/dt$ )

Effects most pronounced for high frequency unsteady flows when the fluid-to-particle density ratio is high, and for acceleration from rest in a quiescent fluid Pressure gradient force (presence of a local pressure gradient) and the force due to shear

$$F_{P} = \frac{m_{P}}{\rho_{P}} \left( -\nabla p + \mu_{f} \nabla \tau_{shear} \right)$$

from the Navier-Stokes equations

$$-\nabla p + \nabla \tau_{shear} = \rho_f \left( \frac{D u_f}{D t} - g \right)$$

The total pressure force is:

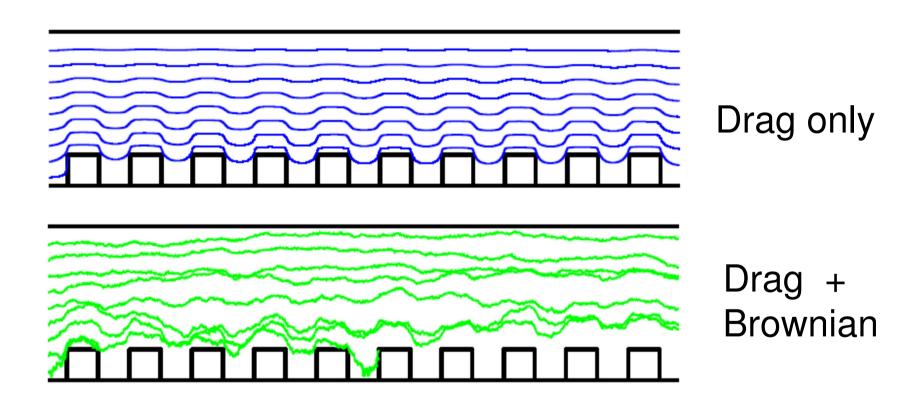
$$F_{P} = m_{P} \frac{\rho_{f}}{\rho_{P}} \left( \frac{D u_{f}}{D t} - g \right)$$

When is this force Srdjan Sasic important?

### **Pressure gradient force**

"For a small sphere, small compared to the scale of the spatial variations of the undisturbed flow, the effect of the undisturbed fluid stresses both from pressure and viscosity is to produce the same net force as would act on a fluid sphere of the same size. This force must equal the product of the fluid mass and local fluid acceleration."

### Force due to Brownian motion (sub-micron particles)



#### Force due to Brownian motion

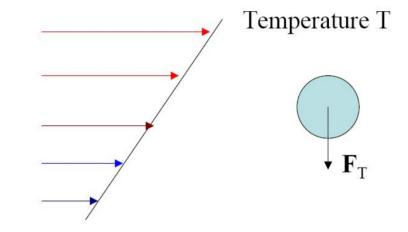
$$F_{Brownian} = \xi m_p \sqrt{\frac{216\mu k_B T}{\pi d_p^5 \rho_p^2 C_c \Delta t}}$$

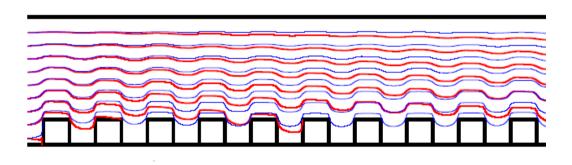
- Can be modelled as a white-noise processes using random numbers
- Transforms Newton's second law for the particle from an ODE to a stochastic differential equation
- Requires very short time steps and many particles

### Thermophoretic force

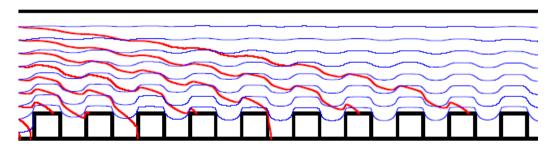
$$F_{T} = -\frac{6\pi d_{p}\mu^{2}C_{s}\left(K + C_{t}\mathrm{Kn}\right)}{\rho\left(1 + 3C_{m}\mathrm{Kn}\right)\left(1 + 2K + 2C_{t}\mathrm{Kn}\right)}\frac{1}{T}\frac{\partial T}{\partial x}$$

Talbot et al., 1980





$$\frac{dT}{dh} = 1 K/mm$$



$$\frac{dT}{dh} = 10 \ K/mm$$

✓ For 
$$Re_p > 1$$
 and  $Re_p >> 1$ 

- 1. The same forces as recognized before, but with coefficients introduced
- 2. Drag coefficient, Lift coefficients, Added mass coefficients, History coefficients
- 3. For the drag force successful
- 4. Other forces with varying success

Specific features related to bubbles

### Important to have in mind

> Absence of a rigid interface between a bubble and fluid

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✓ Consequence: Reduction of the drag force (compared to the corresponding solid particle)

> Contamination of the surface of a bubble



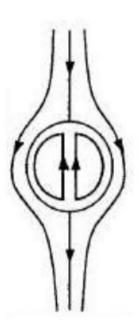
✓ Consequence: Rigid interface again created?

# Internal circulation as a fundamental phenomenon

Tendency to internal circulation given by:

$$\frac{3\kappa+2}{3\kappa+3}$$
 1 = particle behaves as a rigid body 2/3 = "full" internal circulation

$$\kappa = \frac{viscosity \ of \ particle \ fluid}{viscosity \ of \ carrier \ fluid}$$



### Drag force – what happens for bubbles and drops?

Hadamard-Rybczynski drag law – stresses on the surface induce internal motion - drag coefficient decreased

$$C_{D,HR} = \frac{24}{Re_p} \left( \frac{2/3 + \mu_d/\mu_c}{1 + \mu_d/\mu_c} \right)$$

Droplet in air: the Stokes law is recovered

Bubble in liquid: drag is reduced by 1/3

#### Bubble deformation and oscillation

#### Weber number:

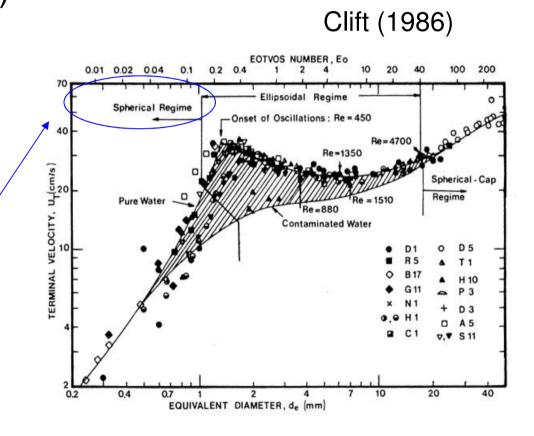
(inertia/surface tension)

$$We_b = \frac{\rho_f \ u_{rel}^2 \ d_{eq}}{\sigma}$$

Equivalent hydraulic diameter

Can bubbles maintain spherical shape?

$$E_O = \frac{We_b}{Fr_b} = g \frac{\left| \rho_f - \rho_b \right| d_{eq}^2}{\sigma}$$



#### Eötvös number

(buoyancy/surface tension) (buoyancy/surface tension)

### Bubble-shape diagram

### Capillary number

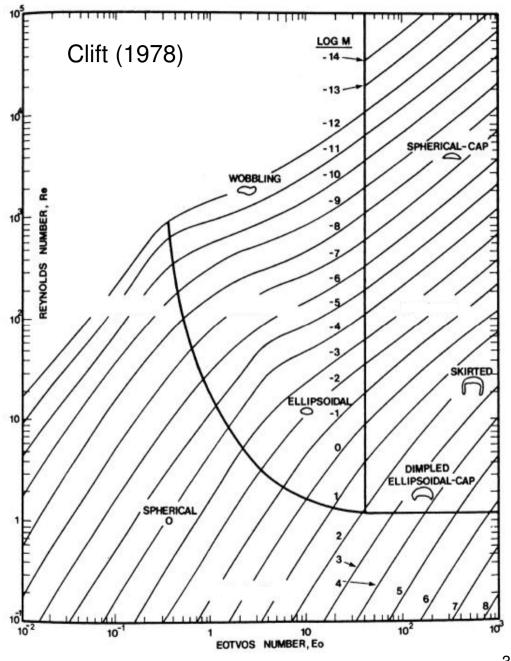
(viscous/surface tension)

$$Ca = \frac{\mu_f U}{\sigma} = \frac{We}{Re}$$

#### Morton number

$$Mo = \frac{g\mu_f^4 |\rho_f - \rho_b|}{\rho_f^2 \sigma^3}$$
$$Mo = \frac{EoWe^2}{Re^4}$$

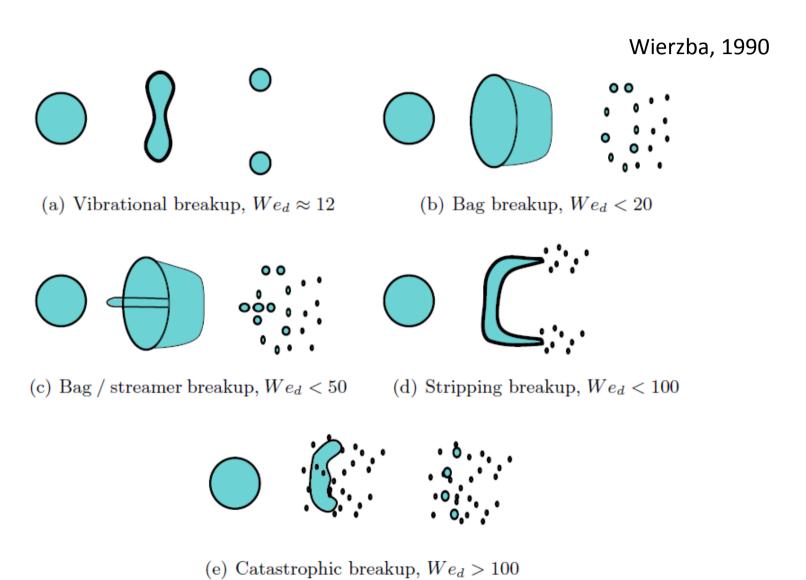
$$Mo = \frac{EoWe^2}{Re^4}$$



Multiphase flow course

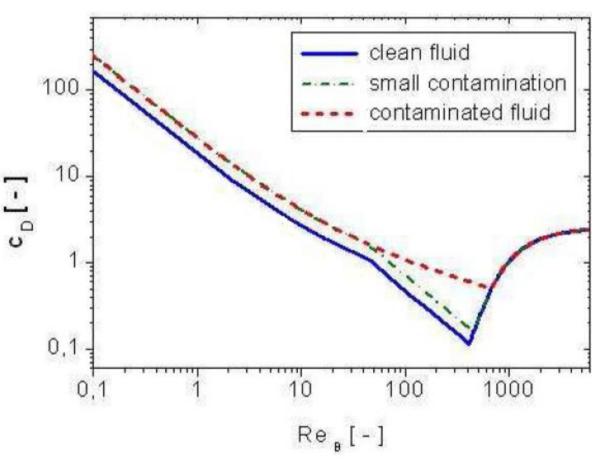
Srdjan Sasic

### Example: (secondary) breakup of droplets as a function of *We*



### Drag force as a function of shape and level of contamination

Tomyama (1998)



#### Tomyama (1998)

Shape	spherical rectilinear		non-spherical
Motion			fluctuating
Purity	pure	contaminated	both
Flow pattern			
Governing effects	viscosity	viscosity	surface tension and gravity
Relevant dimension- less number	Re	Re	Eo

Rigid interface re-created?