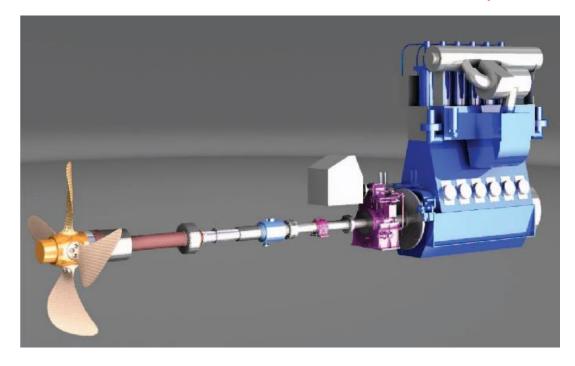
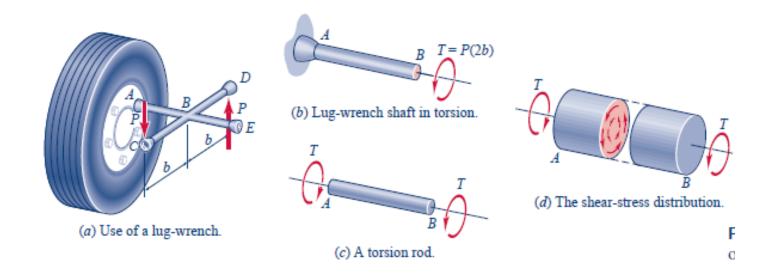
DEPARTAMENTO DE ENGEHARIA NAVAL E OCEÂNICA ESCOLA POLITÉCNICA DA USP

Torção Eixos Circulares: au_{xy}



PNV 3212 – Mecânica Dos Sólidos I 2020

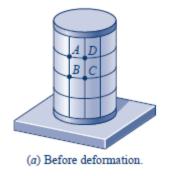
Shear Stress

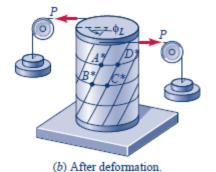


0,	
Axial Deformation	Torsion
Axial Force (F)	Torque (T)
Elongation (e)	Twist angle (ϕ)
Normal stress (σ)	Shear stress (τ)
Extensional strain (ϵ)	Shear strain (γ)
Modulus of elasticity (E)	Shear modulus (G)

Tensões de Cisalhamento

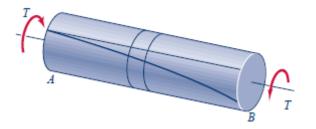
- Hipóteses (torsional-deformation assumptions)
 - o Problema é independente do tempo.
 - Eixo constituído de material linear-elástico.
 - The axis remains straight and remains inextensible.
 - Every cross section remains plane and remains perpendicular to the axis.
 - Radial lines remain straight and radial as the cross section rotates about the axis





Tensões de Cisalhamento

• Fórmula



$$\tau = \frac{T}{J} \gamma$$

Tensões de Cisalhamento

Caminho

- 1. Relação rotação-deformação
- 2. Lei de Hooke
- 3. Equilíbrio da Seção (Momentos)

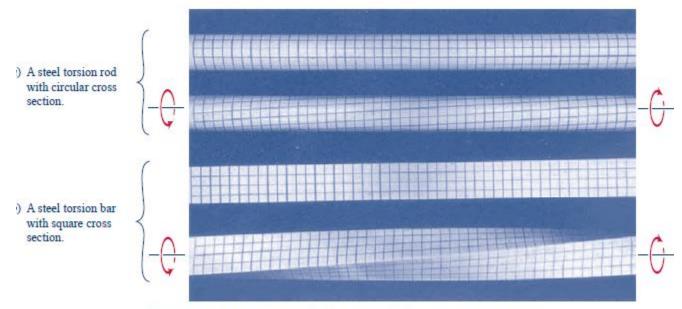
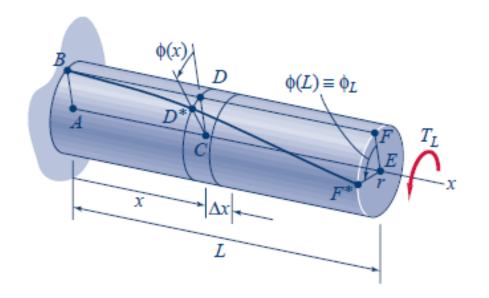
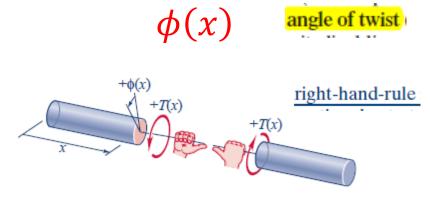


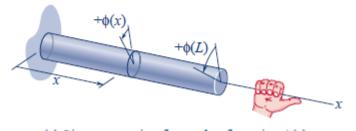
FIGURE 4.2 Examples of torsional deformation. (Roy Craig)

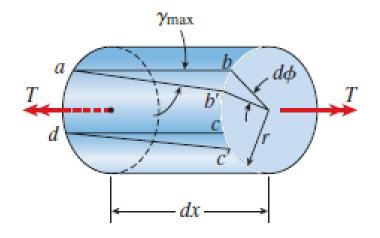


Cada seção transversal gira um ângulo



(b) Sign convention for internal (resisting) torque T(x).



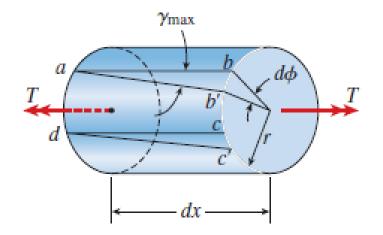


Movimento relativo entre as seções em x e x+dx

$$\phi(x) < \phi(x + dx)$$

Deformação angular

$$\gamma_{max} = \frac{bb'}{dx} + bb' = rd\phi$$
 \downarrow
 $\gamma_{max} = r\frac{d\phi}{dx}$



Deformação angular

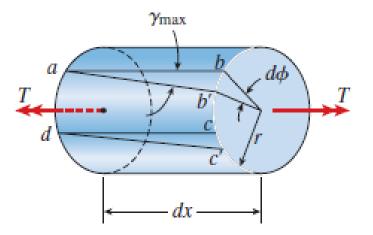
$$\gamma_{max} = r \frac{d\phi}{dx}$$

Razão de torção (razão de cambio do ângulo de torção)

$$\theta = \frac{d\phi}{dx}$$

$$\downarrow \qquad \qquad \qquad \downarrow$$

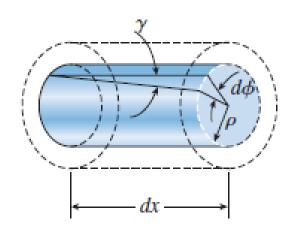
$$\gamma_{max} = r\theta \qquad \text{(rate of twist)}$$



Deformação angular na superfície

$$\gamma_{max} = r \frac{d\phi}{dx}$$

Deformação angular no interior

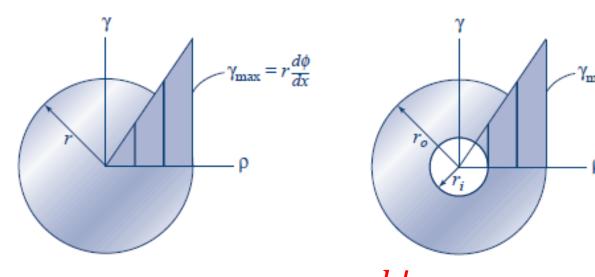


$$\gamma = \rho\theta + \frac{\gamma_{max}}{r} = \theta$$

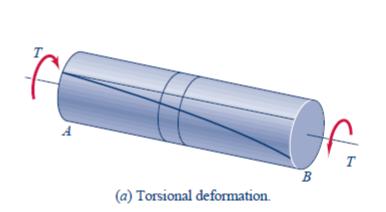
$$\gamma = \frac{\rho}{r} \gamma_{max}$$
 (variação linear com ρ)

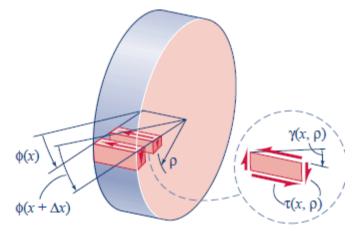
Deformação angular na superfície

$$\gamma = \frac{\rho}{r} \gamma_{max} \qquad \text{(variação linear com } \rho\text{)}$$

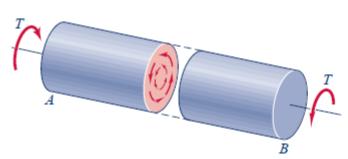


$$\gamma_{max} = r \frac{d\phi}{dx}$$

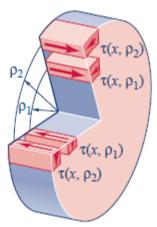




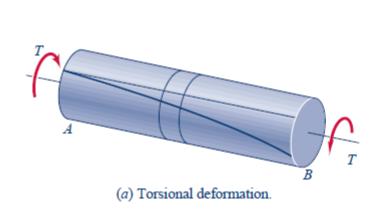
(c) Shear stress and shear strain at typical points.

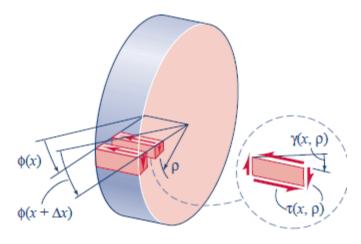


(b) Shear stress due to torsion.

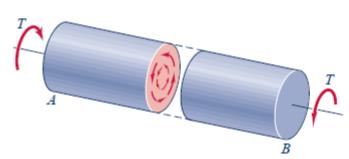


(d) Shear stresses along two typical radial lines in a cross section, and shear stress on radial planes.

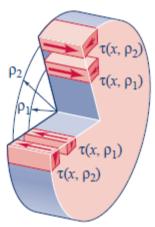




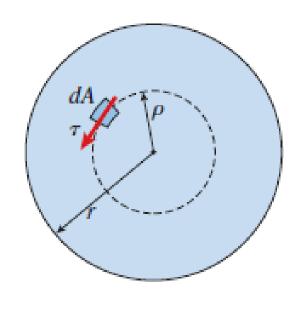
(c) Shear stress and shear strain at typical points.



(b) Shear stress due to torsion.



(d) Shear stresses along two typical radial lines in a cross section, and shear stress on radial planes.



Equilíbrio

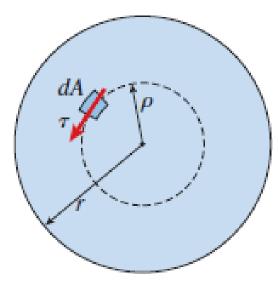
$$\sum M_{x}=0$$

$$T_{ext} = \int_{A} dM$$

Lei de Hooke

$$\tau = G\gamma$$

$$T_{ext} = \int_{A} \tau \rho dA$$



Lei de Hooke

$$\tau = G\gamma$$

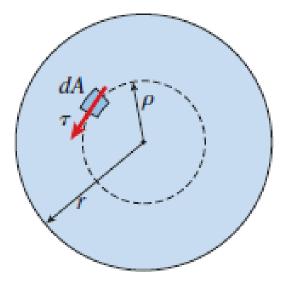
$$\gamma_{max} = r \frac{d\phi}{dx} \longrightarrow$$

Equilíbrio

$$T_{ext} = \int_{A} G\gamma \, \rho dA$$

$$T_{ext} = \int_{A} G \frac{\rho}{r} \gamma_{max} \rho dA$$

$$T_{ext} = G \frac{\gamma_{max}}{r} \int_{A} \rho^{2} dA$$



Lei de Hooke

$$\tau = G\gamma$$

Equilíbrio

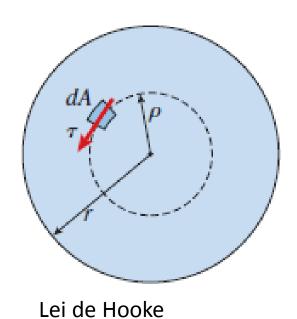
$$T_{ext} = G \frac{d\phi}{dx} \int_{A}^{\Phi} \rho^{2} dA$$

$$\int_{A}^{\pi r^{4}} \frac{\pi r^{4}}{2} = \frac{\pi d^{4}}{32}$$

$$T_{ext} = G \frac{d\phi}{dx} J$$

$$\frac{T_{ext}}{GI} = \frac{d\phi}{dx}$$

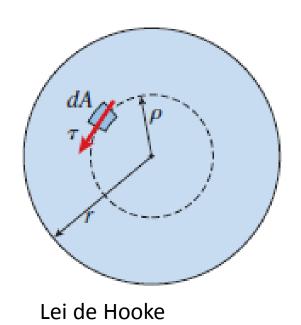
Eq. Torque-rotação



$$\frac{T_{ext}}{GJ} = \frac{d\phi}{dx}$$
 Eq. Torque-rotação
$$\tau = G\gamma \qquad \qquad \gamma = \rho \frac{d\phi}{dx}$$

$$\tau = G\rho \frac{d\phi}{dx}$$

$$\frac{\tau}{\sigma} = G \frac{d\phi}{dx}$$



$$\frac{T_{ext}}{J} = G \frac{d\phi}{dx} \qquad \frac{\tau}{\rho} = G \frac{d\phi}{dx}$$

$$\frac{\tau}{\rho} = G \frac{a\phi}{dx}$$

$$\frac{T_{ext}}{J} = \frac{\tau}{\rho}$$

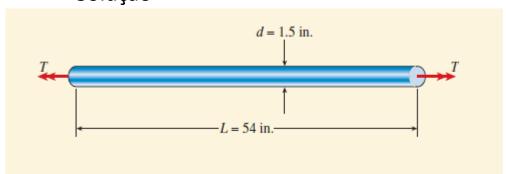
$$\tau = \frac{T_{ext}}{J}\rho \quad \text{vs} \quad \sigma = \frac{M}{I}y$$

Eixo sólido circular de diâmetro d=1.5 in, L =54 in. Aplica-se torque na extremidade T=250 lbf-ft. Determine a tensão de cisalhamento e o ângulo de torção



$$\tau = \frac{T_{ext}}{J}\rho$$

Solução



$$\tau = \frac{T_{ext}}{J}\rho$$

$$J = \frac{\pi d^4}{32}$$

$$J = \frac{\pi \times 1.5^4}{32}$$

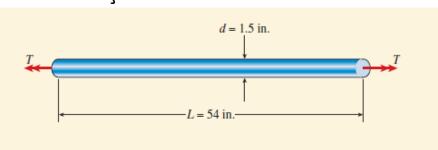
$$J = 0.497 in^4$$

$$\tau = \frac{250}{0.497} \frac{1.5}{2} \times \left(\frac{12in}{1ft}\right)$$

$$\tau = 4530 \text{ psi}$$
 (31.2 MPa)

$$(1 \text{ ksi} = 6.894 \text{ MPa})$$

Solução



Rotação

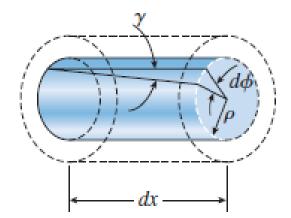
$$\frac{T_{ext}}{GI} = \frac{d\phi}{dx} \qquad \theta = \frac{d\phi}{dx}$$

$$\theta L = \phi$$

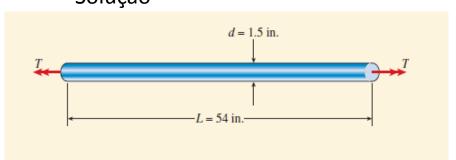
$$\frac{T_{ext}x}{GJ} = \phi(x)$$

$$\frac{250 \times 12 \times 54}{206000} = \phi$$

$$\frac{206000}{1 - 0.3^2} = 0.497$$

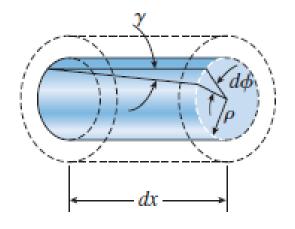


Solução



Rotação

$$\frac{250 \times 12 \times 54}{\frac{206000}{1 - 0.3^2} \cdot 0.497} = \phi$$



$$\phi = 0.028 \, \mathrm{rad}$$

$$\phi = 1.61^{\circ}$$