This is an end-to-end, grey-box model for the vehicle dynamics. The model parameters are not measured directly, but optimized to best fit the vehicle behavior.

$$\mathbf{x} = [p_x, p_y, \psi, v]$$

 $\mathbf{u} = [f, \delta_{ref}, V]$

 $\begin{array}{c|c} p_x & \text{IPS x-position} \\ p_y & \text{IPS y-position} \\ \psi & \text{IPS yaw angle} \\ v & \text{Odometer Speed} \\ f & \text{Dimensionless motor command} \\ S_{ref} & \text{Dimensionless steering command} \\ V & \text{Battery voltage} \end{array}$

$$\dot{p}_x = p_1 \cdot v \cdot (1 + p_2 \cdot (\delta_{ref} + p_9)^2) \cdot \cos(\psi + p_3 \cdot (\delta_{ref} + p_9) + p_{10})$$

$$\dot{p}_y = p_1 \cdot v \cdot (1 + p_2 \cdot (\delta_{ref} + p_9)^2) \cdot \sin(\psi + p_3 \cdot (\delta_{ref} + p_9) + p_{10})$$

$$\dot{\psi} = p_4 \cdot v \cdot (\delta_{ref} + p_9)$$

$$\dot{v} = p_5 \cdot v + (p_6 + p_7 \cdot V) \cdot \text{sign}(f) \cdot |f|^{p_8}$$

This is a kinematic bicycle model with some added terms to account for various errors.

- p₁: Compensate calibration error between IPS speed and odometer speed.
- $(1+p_2\cdot\delta^2)$: Compensate for speed differences due to different reference points between the IPS and odometer. The formulation is simplified with a second-order Taylor approximation.
- p_3 : Side slip angle (Schwimmwinkel) due to steering.
- p_{10} : IPS Yaw calibration error.
- p_4 : Unit conversion for the steering state.

- p_5 : Speed low pass (PT1).
- p_6, p_7 : Motor strength depends on the battery voltage.
- $\bullet \ p_8 :$ Compensate non-linear steady-state speed.
- p_9 : Steering misalignment correction.