

**Control System Design**

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# DC motor and brake

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### Context

We have been tasked to design a numeric controller for a given system which has to meet some requirements. The system is called *DC motor and brake* and it consists of two current controlled motors connected to a brake through a shaft.

Our objective is to send the best signal to the motors to ensure that:

- the shaft can reach any reasonable speed in less than half a second
- the shaft's speed is maintained as smooth as possible
- the system can reject a step disturbance as fast as possible
- the shaft's speed can follow a feasible speed reference of  $4Hz$  from any feasible speed

In addition to this, we could implement a strategy to reject various brake disturbances profile and magnitude, while maintaining the speed variations as smooth as possible.

### Plant description

As explained in the introduction, the plant consists of two motors and one brake acting on the same shaft. There are three sensors available: one speed sensor and two position sensors.



Figure 1: Picture of the plant

## System identification

Based on the mechanical equations of the DC motor, the form of the expected transfer function was already known:

$$G(s) = \frac{A_0}{\tau s + 1} \quad (1.1)$$

Because it has no pole in 0 (or equivalently it is a non-integrator system), the response of the system to a step command is enough to find both  $A_0$  and  $\tau$ .

In the following section, the identification of motor 1 will be explained in detail. The same method has also been done with motor 2 to find its own transfer function.

### 1.1 Step response

The starting point of the step response experiment is the static characteristic of the motor:

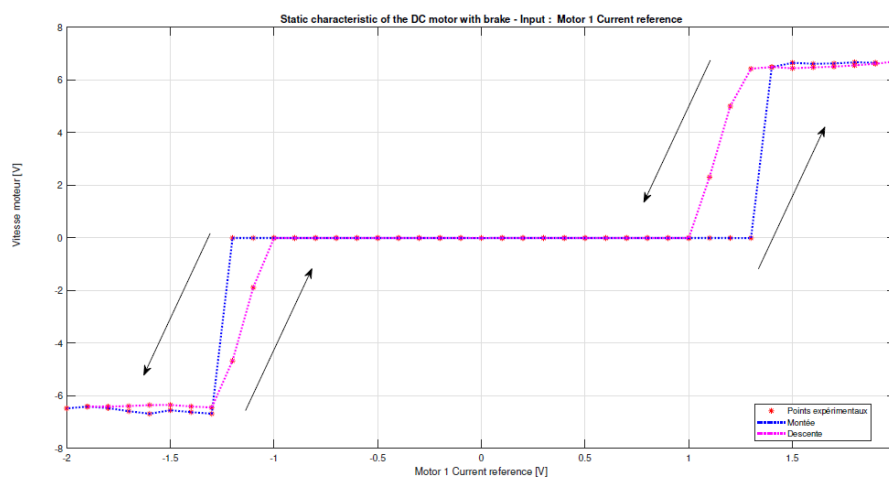


Figure 1.1: Static characteristic of the system driven by the motor 1

By analyzing figure 1.1, a first obvious deduction to be made is that the velocity cannot reach a value

greater than approximately  $6.5V^1$  (and  $-6.5V$  is minimum value). This clarifies the use of "*feasible*" in the requirements as a speed of  $7V$  could never been achieved, no matter what the input is.

What is really interesting to understand is that the static characteristic gives the speed of the shaft after a sufficient amount of time (in theory, after an infinite amount of time bt in practice, once the transient response vanishes). By looking at it, it is clear that to have a speed that is different from  $0V$  and that is outside of the saturated region, a high command should be send to reach saturation and it must then be lowered to a point where the speed is no longer saturating (a command between  $1.1V$  and  $1.3V$  approximately).

With this in mind, the following step response has been recorded:

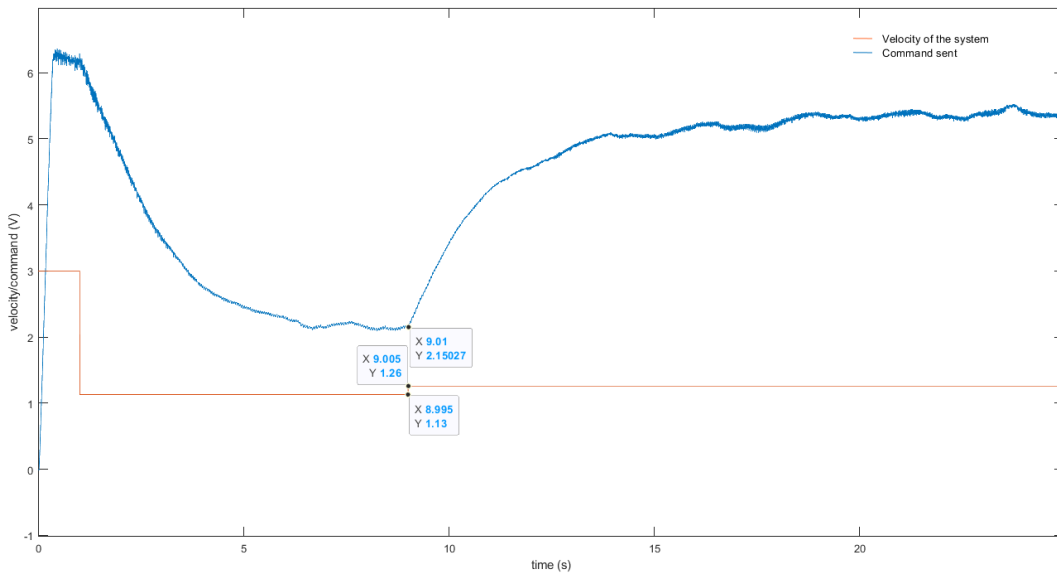


Figure 1.2: Step response of the system when the motor 1 has a step as command

Where the command is first set at a high enough voltage ( $\geq 1.5V$ , based on 1.1). After the velocity saturates, it lowers to a value where the velocity will stabilize. This operating point  $OP_{1+}$  will correspond to the transfer function  $G(s)$  that we are trying to estimate here.

$$OP_{1+} = \begin{bmatrix} \text{command} = 1.13V \\ \text{velocity} = 2.15V \end{bmatrix} \quad (1.2)$$

With an coordinates change for easier visualization, the data plot 1.3 is used for the determination of  $A_0$  and  $\tau$ . It is indeed known that for a transfer function in the form of 1.1, the parameter  $A_0$  is equal to the asymptotic value of the step response divided by the height of the step.  $\tau$  on the other hand is equal to the

<sup>1</sup>Any speed will be expressed in volts ( $V$ ) as speed is measured by a sensor that returns a voltage

time after which the step response reaches  $1/e$  ( $\approx 0.63$ ) times the asymptotic step response. This gives as transfer function:

$$G_{1+} = \frac{24.88}{19.15s + 1} \quad (1.3)$$

The name  $G_{1+}$  has been chosen because the "1" indicates that it corresponds to the motor 1 and the "+" indicates that it is used for a positive speed. As shown on the static characteristic 1.1, the operating point at which the system will be operating to have a negative speed ( $OP_{1-}$ ) is quite far from  $OP_{1+}$ , which means that another model will be needed to describe the behavior of the system in this zone (see discussion in section 1.2).

With the model 1.3, the simulated step response can be computed and as shown in figure 1.3 it can be seen that it matches quite well with the experimental results.

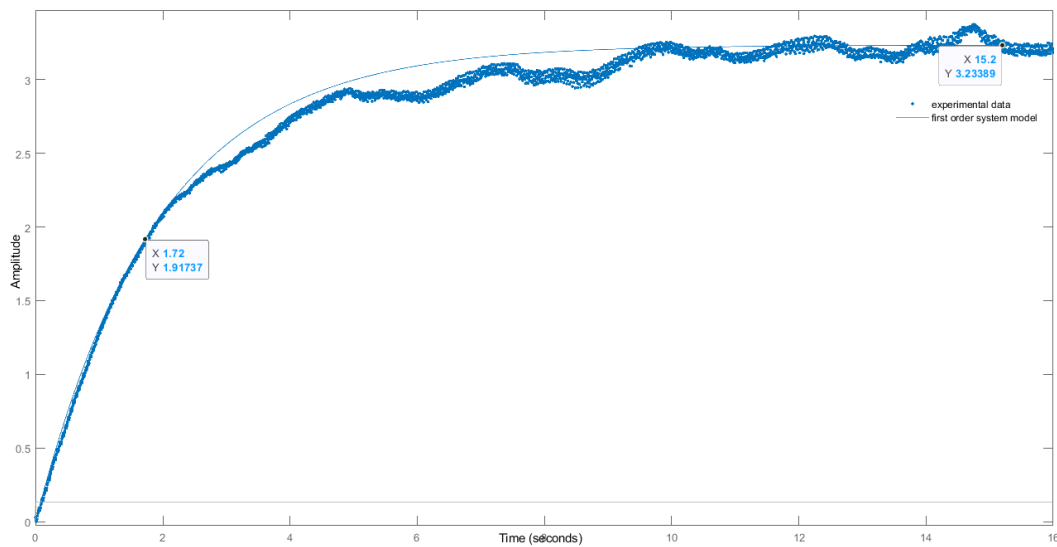


Figure 1.3: Estimation of the step response compared to the real one

## 1.2 validation

