Chapter 7. Steady-State Errors

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Chapter 7. Steady-State Errors

Objectives

Steady-state error: difference between the input and the output for a test input as $t \to \infty$

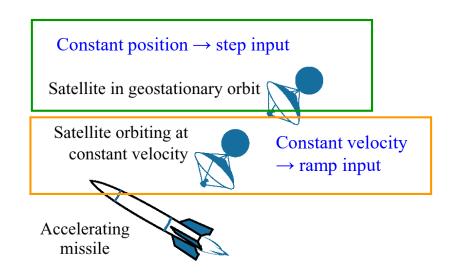
- How to find the *steady-state error* for a unity feedback system
- How to specify a system's steady-state error performance
- How to find the steady-state error for *disturbance inputs*
- How to find the steady-state error for *nonunity feedback systems*
- How to *design system performance* to meet steady-state error performance specifications
- How to find the steady-state error for systems represented in *state space*

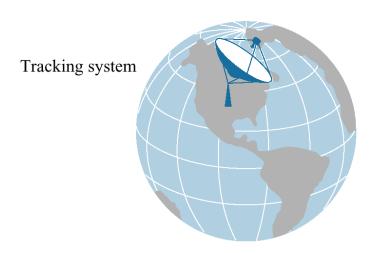
7.1 Introduction

- steady-state error $\Rightarrow \lim_{t \to \infty} e(t) = \lim_{t \to \infty} \{r(t) c(t)\}$
- Test waveforms for evaluating steady-state errors of position control systems

Waveform	Name	Physical interpretation	Time function	Laplace transform
r(t)	Step	Constant position	1	$\frac{1}{s}$
r(t)				S
1	Ramp	Constant velocity	t	$\frac{1}{s^2}$
r(t) t	Parabola	Constant acceleration	$\frac{1}{2}t^2$	$\frac{1}{s^3}$

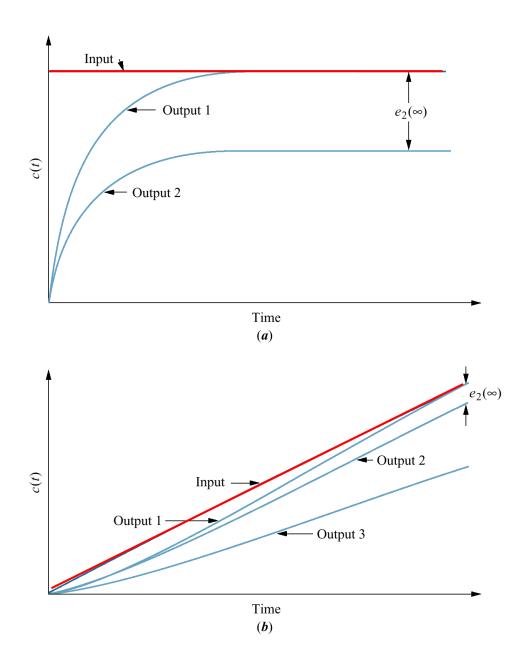
• Test inputs for steady-state error analysis and design vary with target type





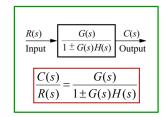
Evaluation steady-state error:

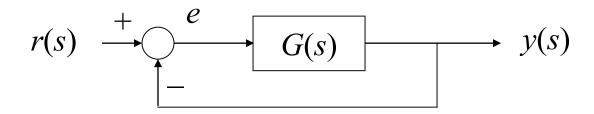
- a. step input;
- b. ramp input



Chapter 7. Steady-State Errors -5-

Unity Negative Feedback

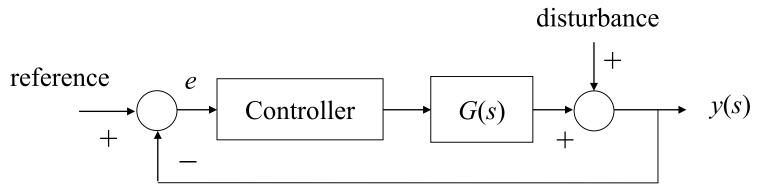




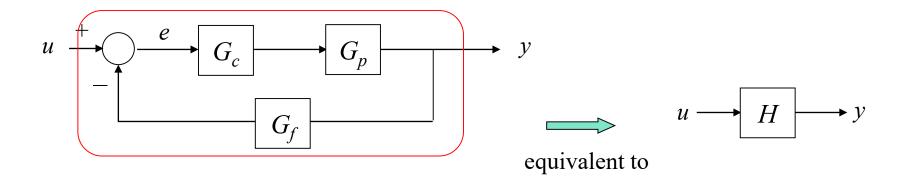
• The overall transfer function of the above system is:

$$T(s) = \frac{G(s)}{1 + G(s)}$$

• System Model:



Feedback



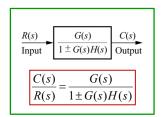
Find
$$H$$
?
$$Y(s) = G_p(s) \cdot G_c(s) \cdot E(s)$$

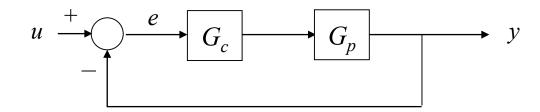
$$E(s) = U(s) - G_f(s) \cdot Y(s)$$

$$Y(s) = G_p(s) \cdot G_c(s) [U(s) - G_f(s) \cdot Y(s)]$$

$$Y(s) = \frac{G_p \cdot G_c}{1 + G_p \cdot G_c \cdot G_f} U(s) = H(s) \cdot U(s)$$

· Special case: unity feedback system



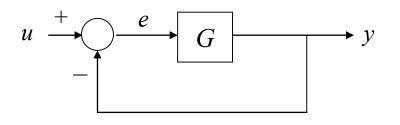


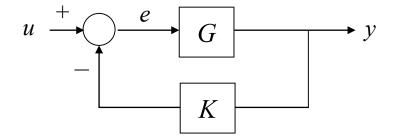
$$Y(s) = \frac{G_p(s) \cdot G_c(s)}{1 + G_p(s) \cdot G_c(s)} U(s) = H(s) \cdot U(s)$$

Purpose of feedback:

- 1. Change the dynamics of the system, stabilization
- 2. Tracking

Reference

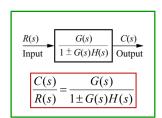




$$y = G(u - y)$$
$$Y(s) = \frac{G}{1 + G}U(s)$$

$$y = G(u - Ky)$$

$$Y(s) = \frac{G}{1 + GK}U(s)$$



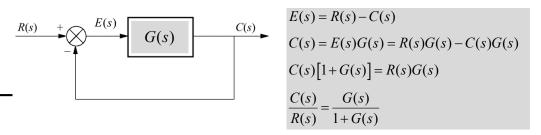
$$y = G(u - Ky)$$

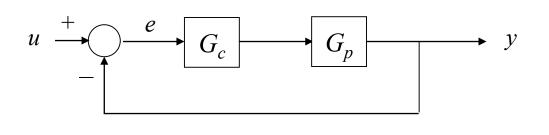
$$y + GKy = Gu$$

$$y(1 + GK) = Gu$$

$$Y(s) = \frac{G}{1 + GK}U(s)$$

Stabilization





$$G_p(s) = \frac{1}{s-1}$$

⇒ unstable

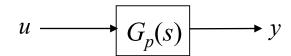
Let
$$G_c(s) = K$$
,

$$H(s) = \frac{KG_p(s)}{1 + KG_p(s)} = \frac{\frac{K}{s-1}}{1 + \frac{K}{s-1}} = \frac{K}{s-1+K}$$

Pole:
$$s-1+K=0$$
, $s=1-K$ $\rightarrow s=(1-K)<0$
 $\rightarrow K>1$

Tracking

• Open loop system



$$G_p(s) = \frac{1}{(s+2)(s+\frac{1}{2})}$$

You want to find *u* such that

$$y(\infty) = \lim_{t \to \infty} y(t) = 1$$
Final value theorem

1

• Final value theorem:
$$y(\infty) = \lim_{s \to 0} sY(s)$$

Try
$$u(t) = \text{step}(t)$$

$$Y(s) = G_p(s) \cdot U(s) = \frac{1}{(s+2)(s+\frac{1}{2})} \cdot \frac{1}{s}$$

By the final value theorem

$$y(\infty) = \lim_{s \to 0} sY(s) = \lim_{s \to 0} \frac{1}{(s+2)(s+\frac{1}{2})} = \boxed{1}$$

• Final value theorem:

$$x(\infty) = \lim_{s \to 0} sX(s)$$

• Initial value theorem:

$$x(0) = \lim_{s \to \infty} sX(s)$$

Ref: if u=step then

$$y(\infty) = \lim_{s \to 0} sG_p(s) \cdot \frac{1}{s} = G_p(0)$$
 \Rightarrow the d_c gain of the system

Now, suppose that
$$G_p(s) = \frac{1}{(s+2+\varepsilon)(s+\frac{1}{2})}$$

for some small, ε

What do you do $? \Rightarrow Use feedback !$

$$u \xrightarrow{+} G_c G_p$$
 y

$$H(s) = \frac{G_c(s)G_p(s)}{1 + G_c(s)G_p(s)} = \frac{\frac{1}{(s+2+\varepsilon)(s+\frac{1}{2})}G_c(s)}{1 + \frac{1}{(s+2+\varepsilon)(s+\frac{1}{2})}G_c(s)}$$

$$Y(s) = H(s)U(s) = H(s)\frac{1}{s} \quad (u(t) = step \text{ in problem})$$
$$y(\infty) = \lim_{s \to 0} sY(s) = \lim_{s \to 0} H(s)$$

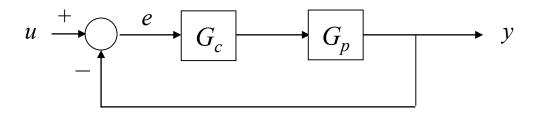
i) Try
$$G_c(s) = k$$

$$y(\infty) \neq 1$$
, but $y(\infty) \xrightarrow[k \to \infty]{} 1$

Need large gains

ii) Try
$$G_c(s) = \frac{1}{s}$$
, $H(s) = \frac{1}{(s+2+\varepsilon)(s+\frac{1}{2})\cdot s + 1}$

$$y(\infty) = \lim_{s \to 0} H(s) = 1$$



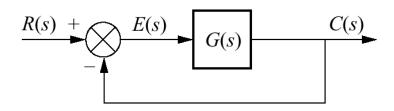
$$U(s) = \frac{1}{s} \qquad G(s) = \frac{1}{(s+2+\varepsilon)\left(s+\frac{1}{2}\right)}$$

$$G_c(s) = \frac{1}{s} \implies y(\infty) = 1$$

End of tracking

7.2 Steady-State Error for Unity Feedback Systems

• Steady-State Error for Unity Feedback Systems

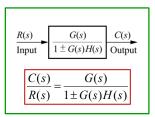


$$E(s) = R(s) - C(s)$$

$$C(s) = E(s)G(s)$$

$$\Rightarrow E(s) = \frac{1}{1 + G(s)}R(s)$$

$$e(\infty) = \lim_{s \to 0} sE(s) = \lim_{s \to 0} \frac{sR(s)}{1 + G(s)}$$



$$e(\infty) = \lim_{s \to 0} sE(s) = \lim_{s \to 0} \frac{sR(s)}{1 + G(s)}$$

• Step input:
$$R(s) = \frac{1}{s}$$

$$e(\infty) = \lim_{s \to 0} \frac{s(1/s)}{1 + G(s)} = \frac{1}{1 + \lim_{s \to 0} G(s)}$$

$$\to \infty \text{ for } n \ge 1$$

$$G(s) = \frac{(s+z_1)(s+z_2)\cdots}{s^n(s+p_1)(s+p_2)\cdots}$$

$$e(\infty) = \frac{1}{1 + \lim_{s \to 0} G(s)} = \frac{1}{1 + \infty} = 0$$

For
$$e(\infty) = 0$$
 $\implies n \ge 1$
 $(\lim_{s \to 0} G(s) = \infty)$ If $n = 0$ then $e(\infty)$ is finite.

• Ramp input:
$$R(s) = \frac{1}{s^2}$$

$$e(\infty) = \lim_{s \to 0} \frac{s(1/s^2)}{1 + G(s)} = \frac{1}{\lim_{s \to 0} sG(s)}$$

$$\to \infty \text{ for } n \ge 2$$

For
$$n = 1$$
: $e(\infty) = \frac{1}{\lim_{s \to 0} sG(s)} = \frac{1}{\lim_{s \to 0} sG(s)} = \frac{1}{\lim_{s \to 0} s\frac{(s+z_1)}{s(s+p_1)}} = \frac{1}{\lim_{s \to 0} \frac{(s+z_1)}{(s+p_1)}} = \frac{p_1}{z_1}$

For
$$e(\infty) = 0$$
 \Rightarrow $n \ge 2$
If $n = 1$, $e(\infty)$ is finite.
If $n = 0$, $e(\infty)$ is infinite.

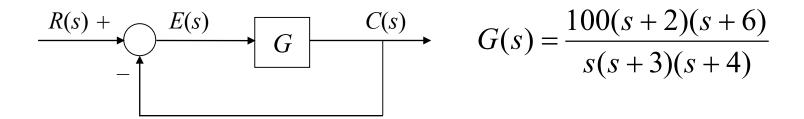
• Parabolic input:
$$R(s) = \frac{1}{s^3}$$

$$e(\infty) = \lim_{s \to 0} \frac{s(1/s^3)}{1 + G(s)} = \frac{1}{\lim_{s \to 0} s^2 G(s)}$$

For
$$e(\infty) = 0$$
 \Rightarrow $n \ge 3$
If $n = 2$, $e(\infty)$ is finite.
If $n = 0$ or 1, $e(\infty)$ is infinite.

$$e(\infty) = \lim_{s \to 0} sE(s) = \lim_{s \to 0} \frac{sR(s)}{1 + G(s)}$$

Example:



Find the steady-state error: $i(u(t), ii)tu(t), iii)\frac{1}{2}t^2u(t)$.

i)
$$R(s) = \frac{1}{s}$$

$$e(\infty) = \lim_{s \to 0} \frac{s\left(\frac{1}{s}\right)}{1 + G(s)} = \frac{1}{1 + \lim_{s \to 0} G(s)} = \frac{1}{\infty} = 0$$

$$e(\infty) = \lim_{s \to 0} sE(s) = \lim_{s \to 0} \frac{sR(s)}{1 + G(s)}$$

ii)
$$R(s) = \frac{1}{s^2}$$

$$e(\infty) = \lim_{s \to 0} \frac{s\left(\frac{1}{s^2}\right)}{1 + G(s)} = \lim_{s \to 0} \frac{1}{s + sG(s)} = \frac{1}{\lim_{s \to 0} sG(s)} = \frac{1}{100}$$

iii)
$$R(s) = \frac{1}{s^3}$$

$$e(\infty) = \lim_{s \to 0} \frac{s\left(\frac{1}{s^3}\right)}{1 + G(s)} = \lim_{s \to 0} \frac{1}{s^2 + s^2 G(s)} = \frac{1}{\lim_{s \to 0} s^2 G(s)} = \frac{1}{0} = \infty$$

7.3 Static Error Constants and System Type

- Static error constants
 - i) step input u(t), $e(\infty) = \frac{1}{1 + \lim_{s \to 0} G(s)} = \frac{1}{1 + K_p}$

position constant : $K_p = \lim_{s \to 0} G(s)$

ii) ramp input
$$tu(t)$$
, $e(\infty) = \frac{1}{\lim_{s \to 0} s G(s)} = \frac{1}{K_v}$

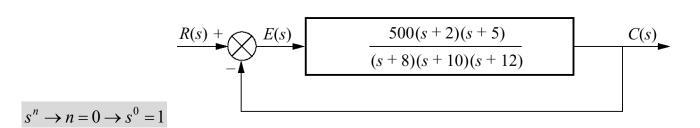
velocity constant : $K_v = \lim_{s \to 0} sG(s)$

iii) parabolic input
$$\frac{1}{2}t^2u(t)$$
, $e(\infty) = \frac{1}{\lim_{s\to 0} s^2 G(s)} = \frac{1}{K_a}$

acceleration constant: $K_a = \lim_{s \to 0} s^2 G(s)$

Example 7.4 (page 350): Steady-state error via static error constants

Evaluate the static error constants and find the expected error for the standard step, ramp, and parabolic inputs.



$$K_p = \lim_{s \to 0} G(s) = 5.208$$
 $K_v = \lim_{s \to 0} sG(s) = 0$ $K_a = \lim_{s \to 0} s^2G(s) = 0$

For a step input
$$\rightarrow e(\infty) = \frac{1}{1 + K_p} = 0.161$$

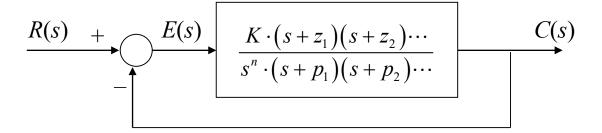
For a ramp input $\rightarrow e(\infty) = \frac{1}{K_p} = \infty$
For a parabolic input $\rightarrow e(\infty) = \frac{1}{K_p} = \infty$

End of example

• System type

$$n = 0 \rightarrow \text{Type } 0$$

 $n = 1 \rightarrow \text{Type } 1$
 $n = 2 \rightarrow \text{Type } 2$

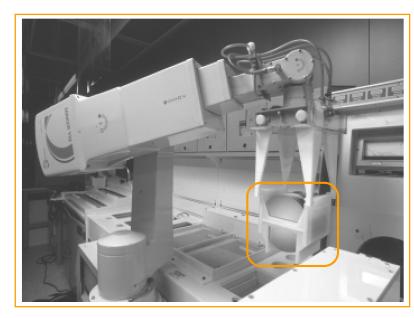


		Тур	Type 0		Type 1		Type 2	
Input	Steady-state error formula	Static error constant	Error	Static error constant	Error	Static error constant	Error	
Step, $u(t)$	$\frac{1}{1+K_p}$	$K_p =$ Constant	$\frac{1}{1+K_p}$	$K_p = \infty$	0	$K_p = \infty$	0	
Ramp, $tu(t)$	$\frac{1}{K_{\nu}}$	$K_{v}=0$	∞	$K_{\nu} =$ Constant	$\frac{1}{K_{v}}$	$K_v = \infty$	0	
Parabola, $\frac{1}{2}t^2u(t)$	$\frac{1}{K_a}$	$K_a = 0$	∞	$K_a = 0$	∞	$K_a =$ Constant	$\frac{1}{K_a}$	

Relationships between input, system type, static error constants, and steady-state errors

7.4 Steady-State Error Specification

- A robot used in the manufacturing of semiconductor randomaccess memories (RAMs) similar to those in personal computers.
- Steady-state error is an important design consideration for assembly-line robots.



© Westlight/ Charles O'Rear.



https://www.sumcosi.com/english/products/process/step_01.html

Example 7.6 (page 355): Gain design to meet a steady-state error specification Find the value of K so that there is 10% error in the steady state.

$$\begin{array}{c|c}
R(s) + E(s) \\
\hline
S(s+6)(s+7)(s+8)
\end{array}$$

$$C(s)$$

• The system is Type $1 \Rightarrow$ a ramp input is applied to the system.

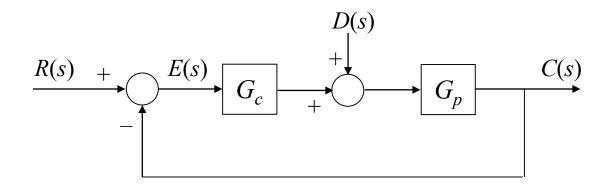
$$e(\infty) = \frac{1}{K_v} = 0.1$$

$$\to K_v = 10 = \lim_{s \to 0} sG(s) = \frac{K \times 5}{6 \times 7 \times 8}$$

$$\to K = 672$$

Input	Steady-state error formula	
Step, $u(t)$	$\frac{1}{1+K_p}$	
Ramp, tu(t)	$\frac{1}{K_{\nu}}$	
Parabola, $\frac{1}{2}t^2u(t)$	$\frac{1}{K_a}$	

7.5 Steady-State Error for Disturbances



$$\begin{split} E(s) &= R(s) - C(s) \\ C(s) &= \left[E(s) \cdot G_c(s) + D(s) \right] \cdot G_p(s) \\ E &= R - E \cdot G_c G_p - D \cdot G_p \quad , \quad E\left(1 + G_c G_p\right) = R - D \cdot G_p \end{split}$$

$$E(s) = \frac{R(s)}{1 + G_c(s) \cdot G_p(s)} - \frac{G_p(s)}{1 + G_c(s) \cdot G_p(s)} \cdot D(s)$$
$$= e_R(\infty) + e_D(\infty)$$

Example: Find G_c to eliminate the effect of ω on y in steady state.

$$Y_{\omega}(s) = \frac{G_{P}}{1 - G_{C}G_{P}}\omega$$

$$= \frac{(s+1)\frac{1}{s}}{(s+2)(s+0.51) - (s+1)G_{C}}$$

Let,
$$G_C(s) = \frac{1}{s}$$

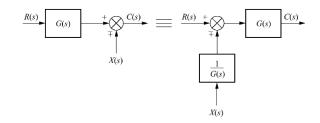
$$y_{\infty}(\infty) = \lim_{s \to 0} s Y_{\omega}(s) = \lim_{s \to 0} s \cdot \frac{(s+1)\frac{1}{s}}{(s+2)(s+0.51) - (s+1)\frac{1}{s}}$$

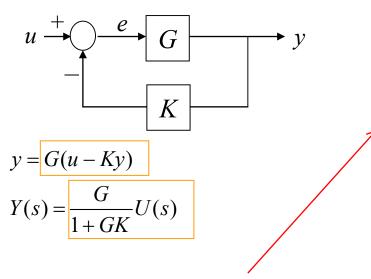
$$= \lim_{s \to 0} \frac{s(s+1)}{s(s+2)(s+0.51) - (s+1)}$$

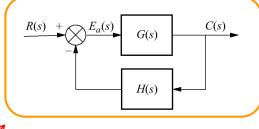
$$= 0$$

$$\therefore G_C(s) = \frac{1}{s} \rightarrow y_\infty(\infty) = \boxed{0}$$

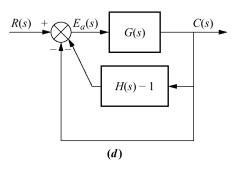
7.6 Steady-State Error for Nonunity Feedback Systems







$$G = G_1 G_2, H = \frac{H_1}{G_1}$$



(b)

$$G(s)$$

$$H(s)$$

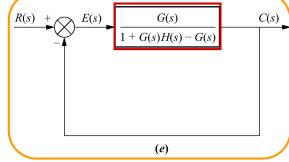
$$G(s)$$

$$G(s)$$

$$G(s)$$

$$G(s)$$

C(s)



$$C(s) = G_1 \frac{G_2}{1 + G_2 H_1} R(s) = \frac{G_1 G_2}{1 + G_1 G_2 \frac{H_1}{G_2}} R(s) = \frac{G}{1 + GH} R(s)$$

 $H_1(s)$

(a)

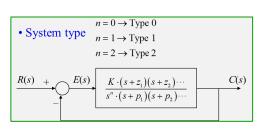
C(s)

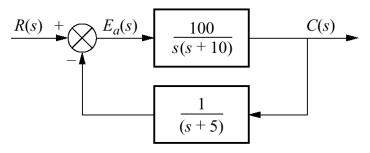
$$Y = \frac{\frac{G}{1+GH-G}}{1+\frac{G}{1+GH-G}}R$$
$$= \frac{G}{(1+GH-G)+G}R = \frac{G}{1+GH}R$$

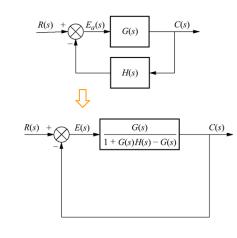
Chapter 7. Steady-State Errors -29-

Example 7.8 (page 359): Steady-state error for nonunity feedback systems

Find the system type, the appropriate static error constant, and the steadystate error for a unit step input.







$$G_e(s) = \frac{G(s)}{1 + G(s)H(s) - G(s)}$$

equivalent

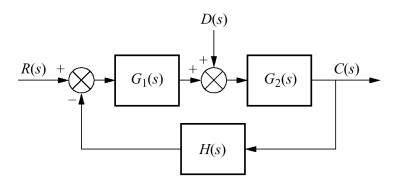
$$= \frac{\frac{100}{s(s+10)}}{1 + \frac{100}{s(s+10)} \cdot \frac{1}{s+5} - \frac{100}{s(s+10)}}$$

$$=\frac{100(s+5)}{s^3+15s^2-50s-400}$$

2) Error Constant:
$$K_p = \lim_{s \to 0} G_e(s) = \frac{500}{-400} = -\frac{5}{4}$$

3)
$$e(\infty) = \frac{1}{1+K_p} = \frac{1}{1-(5/4)} = -4$$

Non-Unity Feedback Control with Disturbance



Steady-state error: difference between the input and the output for a test input as $t \to \infty$

$$R(s) = D(s) = \frac{1}{s}$$

$$e(\infty) = \lim_{s \to 0} sE(s) = \left\{ \left[1 - \frac{\lim_{s \to 0} G_1(s)G_2(s)}{\lim_{s \to 0} \left[1 + G_1(s)G_2(s)H(s) \right]} \right] - \left[\frac{\lim_{s \to 0} G_2(s)}{\lim_{s \to 0} \left[\left[1 + G_1(s)G_2(s)H(s) \right] \right]} \right] \right\}$$

$$= \left(1 - \frac{G_1G_2}{1 + G_1G_2H} \right)R - \left(\frac{G_2}{1 + G_1G_2H} \right)D$$

For
$$e(\infty) = 0$$
,
$$\frac{\lim_{s \to 0} [G_1(s)G_2(s)]}{\lim_{s \to 0} [1 + G_1(s)G_2(s)H(s)]} = 1 \quad and \quad \frac{\lim_{s \to 0} G_2(s)}{\lim_{s \to 0} [1 + G_1(s)G_2(s)H(s)]} = 0$$

$$\Rightarrow \frac{1}{\lim_{s \to 0} G_1(s)G_2(s)} + \lim_{s \to 0} H(s) = 1, \quad \frac{1}{\lim_{s \to 0} G_2(s)} + \lim_{s \to 0} G_1(s)H(s) = 0$$

input of G_1 : R - CHinput of G_2 : $G_1E + D = G_1(R - CH) + D$ $C = G_2 \lceil G_1 (R - CH) + D \rceil \cdots (1)$ steady-state error E = R - C, $C = R - E \cdots (2)$ $(2) \to (1);$ $R - E = G_1G_2R - G_1G_2(CH) + G_2D$ $=G_1G_2R-G_1G_2(R-E)H+G_2D$ $= G_1G_2R - G_1G_2RH + G_1G_2EH + G_2D$ arrange for E, $E(1+G_1G_2H) = R - G_1G_2R + G_1G_2RH - G_2D$ $\therefore E = \frac{1 - G_1 G_2 + G_1 G_2 H}{1 + G_1 G_2 H} R - \frac{G_2}{1 + G_1 G_2 H} D$

7.7 Sensitivity

- System parameters are changed → transfer function
 - \Rightarrow performance
 - \Rightarrow sensibility

$$F = \frac{K}{K+a}$$
, $K = 10$, $a = 100 \rightarrow F = \frac{10}{10+100} = 0.091$

$$\frac{300-100}{100} \times 100$$

$$= 200\%$$

$$a = 300 \rightarrow F = \frac{10}{10+300} = 0.032$$

F has reduced sensitivity to changes in parameter a:

$$\begin{pmatrix} 200\% \text{ change in parameter '}a' \\ \rightarrow -65\% \text{ change in } F \end{pmatrix} \Rightarrow \frac{-65\%}{200\%} = -0.325$$

• Sensitivity:

$$S_{F:P} = \lim_{\Delta P \to 0} \frac{\frac{\Delta F}{F}}{\frac{\Delta P}{P}} = \lim_{\Delta P \to 0} \frac{P}{F} \frac{\Delta F}{\Delta P}$$

$$S_{F:P} = \frac{P}{F} \frac{\delta F}{\delta P}$$

Example 7.10 (page 362): Sensitivity of a closed-loop transfer function

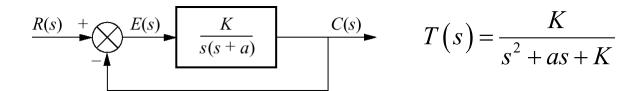
Calculate the sensitivity to change in the parameter *a*. How would you reduce the sensitivity?

(1)
$$S_{T:a} = \frac{a}{T} \frac{\delta T}{\delta a} = \frac{a}{\frac{K}{s^2 + as + K}} \cdot \frac{-sK}{(s^2 + as + K)^2} = \frac{-as}{(s^2 + as + K)}$$

(2)
$$\Rightarrow$$
 Increase $K \rightarrow$ reduce sensitivity

Example 7.11 (page 363): Sensitivity of steady-state error with ramp input

Find the sensitivity of the steady-state error to changes in parameters *K* and *a* with ramp input.



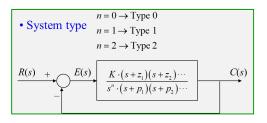
Type 1:

$$e = e(\infty) = \frac{1}{K_v} = \frac{1}{\lim_{s \to 0} sG(s)} = \frac{1}{K/a} = \frac{a}{K}$$

$$S_{e:a} = \frac{a}{e} \cdot \frac{\delta e}{\delta a} = \frac{a}{a/K} \left(\frac{1}{K}\right) = 1$$

$$S_{e:K} = \frac{K}{e} \frac{\delta e}{\delta K} = \frac{K}{a/K} \left[\frac{-a}{K^2} \right] = -1$$

$$\begin{split} S_{F:P} &= \lim_{\Delta P \to 0} \frac{\Delta F / F}{\Delta P / P} \\ &= \lim_{\Delta P \to 0} \frac{P}{F} \frac{\Delta F}{\Delta P} \\ &= \frac{P}{F} \frac{\delta F}{\delta P} \end{split}$$



Example: Sensitivity of the S.S. error in K, a

$$G = \frac{K(s) + C(s)}{G(s+a)(s+b)}, \quad R(s) = \frac{1}{s}$$

$$E(s) = R(s) - G \cdot E(s), \quad E(s) = \frac{1}{1 + G(s)} R(s)$$

$$e(\infty) = \lim_{s \to 0} sE(s) = \lim_{s \to 0} \frac{s\left(\frac{1}{s}\right)}{1 + G(s)} = \frac{1}{1 + \lim_{s \to 0} G(s)} = \frac{1}{1 + \frac{K}{ab}}$$

$$=\frac{ab}{ab+K}$$

$$e(\infty) = \lim_{s \to 0} sE(s) = \frac{ab}{ab + K}$$

(1)
$$S_{e:a} = \frac{a}{e} \frac{\delta e}{\delta a} = \frac{a}{\frac{ab}{ab+K}} \cdot \frac{b(ab+K)-ab(b)}{(ab+K)^2} = \frac{b \cdot K}{b(ab+K)}$$
$$= \frac{K}{ab+K}$$

$$(2) S_{e:K} = \frac{K}{e} \frac{\delta e}{\delta K} = \frac{K}{ab} \cdot \frac{-ab}{\left(ab + K\right)^2} = \frac{-K}{ab + K}.$$

→ feedback sensitivity

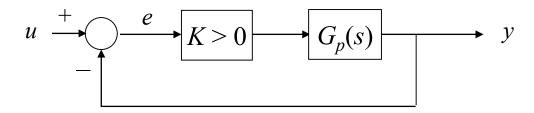
$$S_{F:P} = \lim_{\Delta P \to 0} \frac{\Delta F / F}{\Delta P / P}$$

$$= \lim_{\Delta P \to 0} \frac{P}{F} \frac{\Delta F}{\Delta P}$$

$$= \frac{P}{F} \frac{\delta F}{\delta P}$$

Next, Stability design.

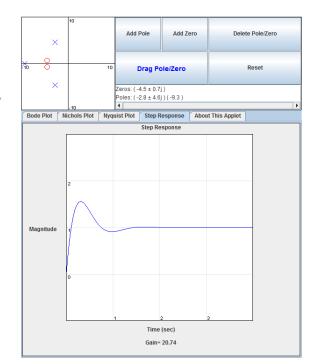
- Root locus, Nyquist Plot, Bode diagram

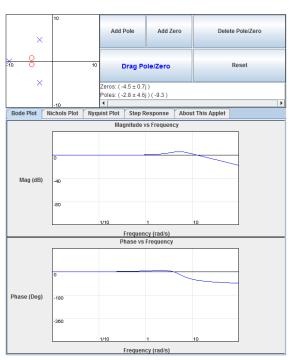


Question: For what value of K > 0 is the closed-loop system stable?

PoleZeroApplet:

http://web.mit.edu/6.302/www/pz/





Errors -38-

7.8 Steady-State Error for Systems in State Space

Analysis via final value theorem:

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}r & \Rightarrow \qquad E(s) = R(s) - Y(s), \quad Y(s) = R(s)T(s) \\ y = \mathbf{C}\mathbf{x} & E(s) = R(s) - R(s)T(s) \\ & = R(s) \left[1 - T(s)\right] \\ & = R(s) \left[1 - \mathbf{C}(sI - \mathbf{A})^{-1}\mathbf{B}\right] \end{cases}$$

 \Rightarrow Applying the final value theorem:

$$\lim_{s \to 0} sE(s) = \lim_{s \to 0} sR(s)[1 - C(sI - A)^{-1}B]$$