

Solution

Forces on x-axis

$$F_s, F_{\text{mag}}, mg_x, T, b$$

$$F_s = -k_1(x-d) - k_2(x-d)^3$$

Forces on y-axis

$$N - mg \cos \phi = 0$$

Total forces on x-axis

$$F_{\text{mag}} + mg \sin \phi - T - k_1(x-d) - k_2(x-d)^3 - b\dot{x} = m\ddot{x} \quad \text{--- (1)}$$

As we know that

$$M_T = T \cdot r = I \ddot{\theta}$$

$$= \frac{1}{2} m R^2 \ddot{\theta}$$

$$= \frac{1}{2} m \ddot{x} = T$$

(\because Intermix of linear motion)

$$F_{\text{mag}} + mg \sin \phi - T - k_1(x-d) - k_2(x-d)^3 - b\dot{x} = \frac{3}{2} m \ddot{x}$$

As given

$$F_{\text{mag}} = \frac{c I^2}{y^2} \quad \text{--- (2)}$$

Using KVL on the loop

$$V = IR + L \frac{dI}{dt}$$

Integrate on both sides w.r.t y

$$\int V \cdot dy = IR \int 1 \cdot dy + L \int \frac{dI}{dy} \cdot dy$$

$$V_y = IR_y + LI$$

$$V_y = I(R_y + L)$$

$$I = \frac{V_y}{R_y + L} \quad \text{--- (3)}$$

put eq. (3) in (2)

$$F_{\text{mag}} = \frac{c V_y^2}{(R_y + L)^2 y^2} = \frac{c}{(R_y + L)^2} \times V^2 \quad \text{--- (4)}$$

put eq. (4) in eq. (1)

Model

$$\frac{c}{(Ry+L)^2} (v^2) + mg \sin \phi - b \dot{x} - k_1(x-d) - k_2(x-d)^3 = \frac{3}{2} m \ddot{x}$$

multiply the eq with $\frac{2}{3m}$

$$\frac{2c}{3(Ry+L)^2 m} (v^2) + \frac{2}{3} g \sin \phi - \frac{2}{3m} b \dot{x} - \frac{2}{3m} k_1(x-d) - \frac{2}{3m} k_2(x-d)^3 = \ddot{x}$$

$$x_1 = x$$

$$x_2 = \dot{x}_1 = \dot{x}$$

$$\dot{x}_2 = \ddot{x}_1 = \ddot{x}$$

$$\frac{2c}{3(Ry+L)^2 m} (v^2) + \frac{2}{3} g \sin \phi - \frac{2}{3} b x_2 - \frac{2}{3m} k_1(x_1-d) - \frac{2}{3m} k_2(x_1-d)^3 = \dot{x}_2$$

State space representation

$$\begin{cases} \dot{x}_1 = x_2 \\ \dot{x}_2 = \frac{2c}{3(Ry+L)^2 m} (v^2) + \frac{2}{3} g \sin \phi - \frac{2}{3} b x_2 - \frac{2}{3m} k_1(x_1-d) - \frac{2}{3m} k_2(x_1-d)^3 \end{cases}$$

equilibrium

$$\dot{x} = f(x, v)$$

A pair (x^{ev}, v^{ev}) is an equilibrium point

$$\begin{cases} \dot{x}_2 = 0 \\ \frac{2c}{3(Ry+L)^2 m} (v^{ev})^2 + \frac{2}{3} g \sin \phi - \frac{2}{3} b x_2^{ev} - \frac{2}{3m} k_1(x_1^{ev}-d) - \frac{2}{3m} k_2(x_1^{ev}-d)^3 = 0 \end{cases}$$

Linearization

$$\frac{2c}{3(Ry+L)^2_m} (v)^2 + \frac{2}{3} g \sin \phi - \frac{2}{3m} bx_2 - \frac{2k_1}{3m} (x_1 - d) - \frac{2k_2}{3m} (x_1 - d)^3 = \ddot{x}_1$$

$$g(x_1) = (x_1 - d)^3$$

$$g'(x_1) = 3(x_1 - d)^2$$

$$g'(x_1^{ev}) = 3(x_1^{ev} - d)^2$$

$$g(x_1^{ev}) = (x_1^{ev} - d)^3$$

By using Taylor Approximation Theorem

$$g(x_1) \approx g(x_1^{ev}) + g'(x_1^{ev})(x_1 - x_1^{ev})$$

By putting all the values

$$(x_1 - d)^3 = (x_1^{ev} - d)^3 + 3(x_1^{ev} - d)(\bar{x}_1) - \textcircled{5} (\bar{x}_1 \text{ is deviation variable})$$

So,

$$\ddot{x}_1 = \frac{2c}{3(Ry+L)^2_m} (v^2) + \frac{2}{3} g \sin \phi - \frac{2}{3m} bx_2 - \frac{2k_1}{3m} (x_1 - d) - \frac{2k_2}{3m} (x_1 - d)^3$$

(-)

$$0 = \frac{2c}{3(Ry+L)^2_m} (v^{ev})^2 + \frac{2}{3} g \sin \phi - \frac{2}{3m} bx_2^{ev} - k_1(x_1^{ev} - d) - k_2(x_1^{ev} - d)^3$$

$$= \frac{2}{3(Ry+L)^2_m} (v^{ev})^2 - \frac{2}{3m} b(x_2 - x_2^{ev}) - \frac{2}{3m} k_1(x_1 - x_1^{ev}) - \frac{2k_2}{3m} [(x_1 - d)^3 - (x_1^{ev} - d)^3]$$

By using eq(5)

$$= \frac{2}{3(Ry+L)^2_m} (\bar{v})^2 - \frac{2}{3m} b(\bar{x}_2) - \frac{2}{3m} k_1(\bar{x}_1) - \frac{2k_2}{3m} (3(x_1^{ev} - d)^2) \bar{x}_1$$

$$\frac{2c}{3m(Ry+L)^2} (\bar{v}^2) - \frac{2}{3m} (b)(\bar{x}_2) - \frac{2}{3m} [k_1 + k_2(3(x_1^{eq} - d)^2)] \bar{x}_2$$

$$q = \frac{2c}{3m(Ry+L)^2}$$

$$u = \frac{2}{3m} b$$

$$w = \frac{2}{3m} [k_1 + k_2(3(x_1^{eq} - d)^2)]$$

So,

$$\dot{\bar{x}}_2 = q \bar{v}^2 - u \bar{x}_2 - \bar{x}_2 w$$

$$\bar{x} = ?$$

$$\bar{x}_1 = \bar{x}_2$$

$$\dot{\bar{x}}_2 = \dot{x}_2$$

$$\dot{\bar{x}}_2 = q [\bar{v}^2] - u \bar{x}_2 - \bar{x}_2 w$$

$$0 = \dot{x}_{eq}$$

$$\text{So, } \bar{x}_1 = \bar{x}_2$$

Transfer function

taking Laplace on both sides

$$s \bar{x}_2 = \dot{\bar{x}}_2$$

$$s \bar{x}_2 = q \bar{v}^2 - u \bar{x}_2 - \bar{x}_2 w$$

So,

$$s^2 \bar{x}_2 = q \bar{v}^2 - u \bar{x}_2 s - \bar{x}_2 w$$

$$\bar{x}_2 = \frac{q \bar{v}^2}{s^2 + us + w}$$

$$\frac{\bar{x}_2(s)}{\bar{v}^2(s)} = \frac{q}{s^2 + us + w}$$

Poles

Using quadratic formula

$$s = \frac{-u \pm \sqrt{u^2 - 4w}}{2}$$

$$s = \frac{-u - \sqrt{u^2 - 4w}}{2} ; \quad s = \frac{-u + \sqrt{u^2 - 4w}}{2}$$

So

Transfer function

$$\frac{q}{\left(\frac{s + \frac{u}{2} - \frac{\sqrt{u^2 - 4w}}{2}}{1} \right) \left(\frac{s + \frac{u}{2} + \frac{\sqrt{u^2 - 4w}}{2}}{1} \right)}$$

It has two poles at

~~$s = \dots$~~

Applying Impulse function

$$X_1(s) = \frac{q}{s^2 + us + w} \quad (1)$$

So function will be same

$$X_2(s) = \frac{q}{s^2 + us + w}$$

condition at which system is oscillatory

As we know that when there is no damper the impulse response of the transfer function will be oscillatory. i.e. $b=0$ or $u=0$