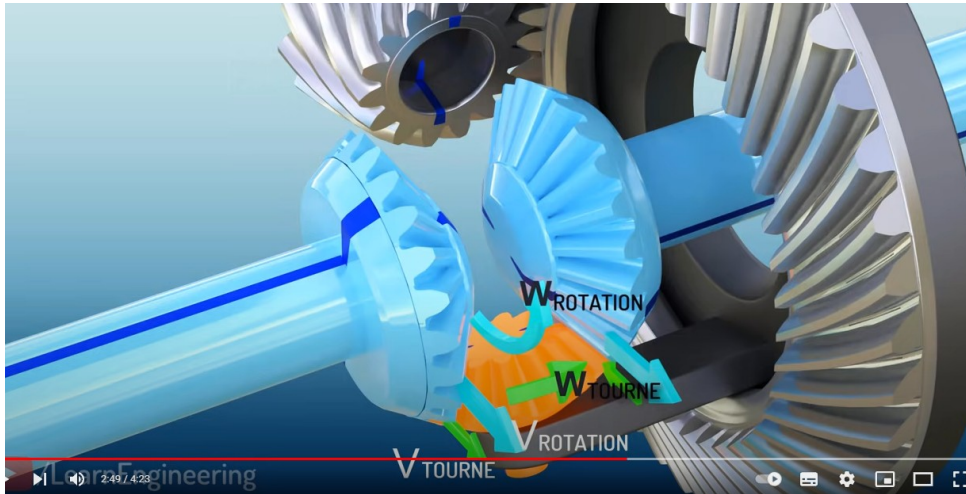


Point technique n°1

- 1) Différentiel mécanique
- 2) Commande électrique des moteurs
- 3) Principes et modèles existants
- 4) Avancement modèle Simulink
- 5) Questionnements techniques

1) Différentiel mécanique



<https://www.youtube.com/watch?v=qXoCHETmJ0k>

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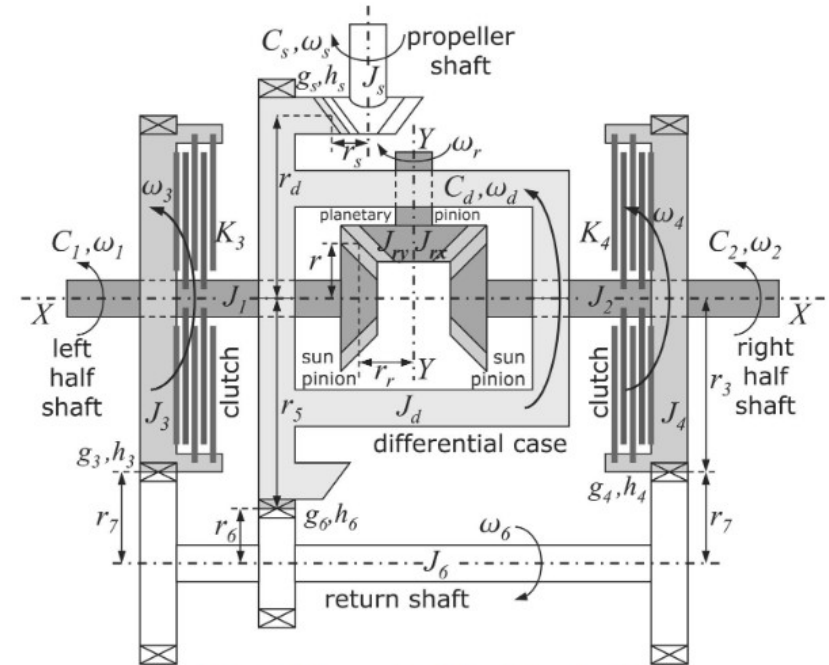


Figure 2. Mechanical scheme of the steering differential.

<https://www.tandfonline.com/doi/epdf/10.1080/13873950500066959?needAccess=true>

1) Différentiel mécanique

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R. Morselli et al.

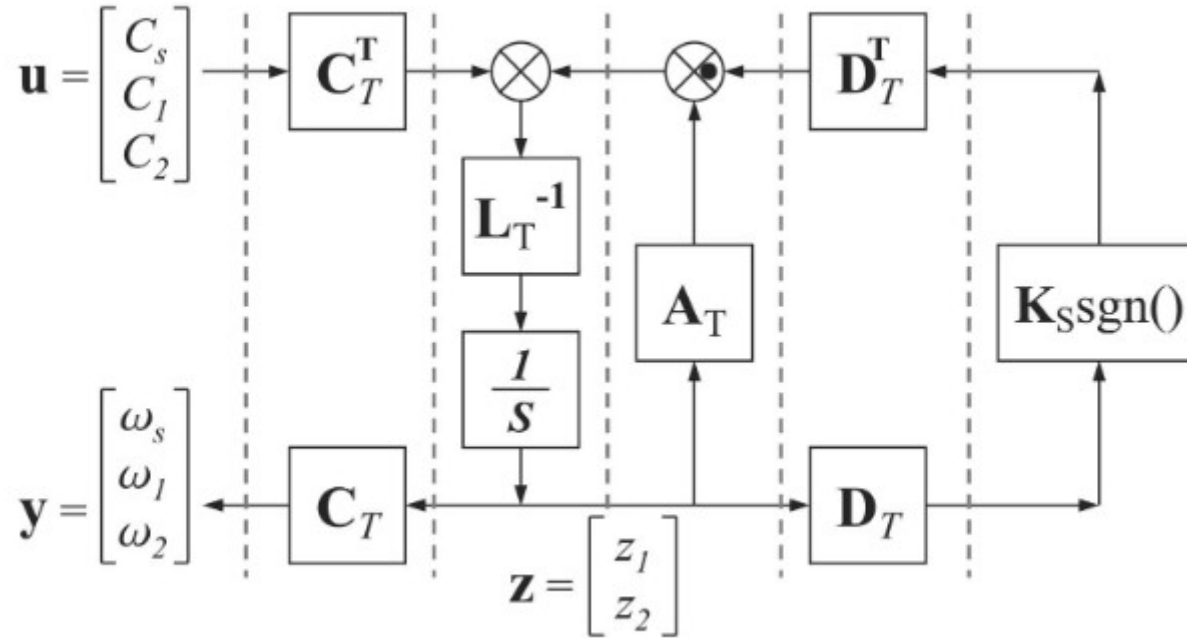
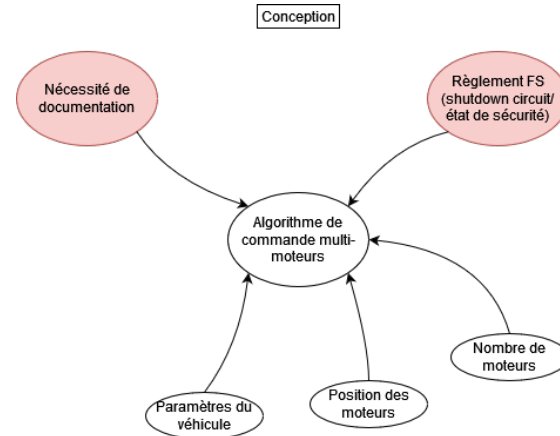
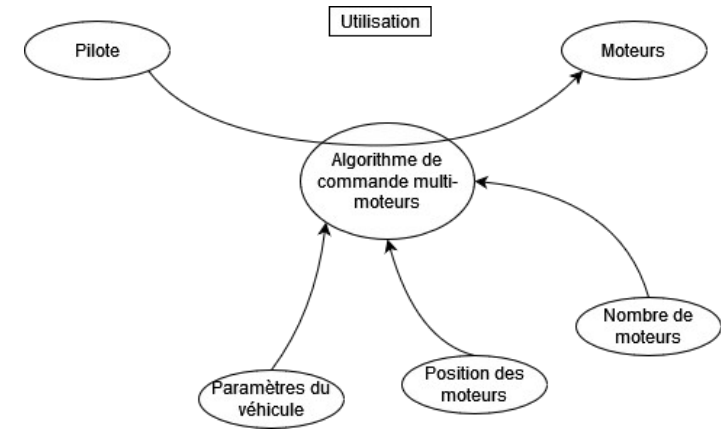


Figure 4. Scheme for the simulation of the reduced transformed system (7).

2) Commande électrique des moteurs

- Liberté d'implémentation
- Liberté sur la stratégie de commande → objet du stage
- Choix des grandeurs à contrôler à faire
- Dépendance aux moteurs utilisés et à leur nombre



3) Principes existants

- Vitesse de rotation des moteurs
- Slip-ratio
- Couple
- Moment de lacet

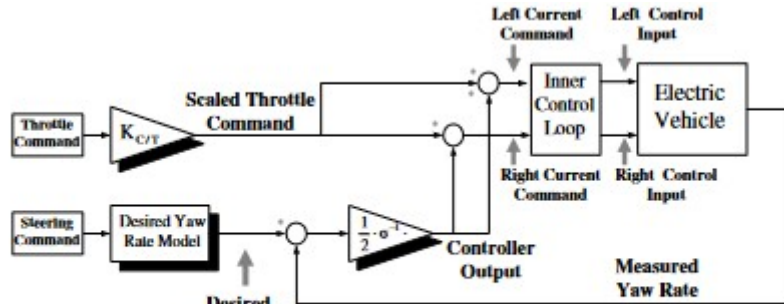


Fig. 3. Block diagram of yaw rate control loop.

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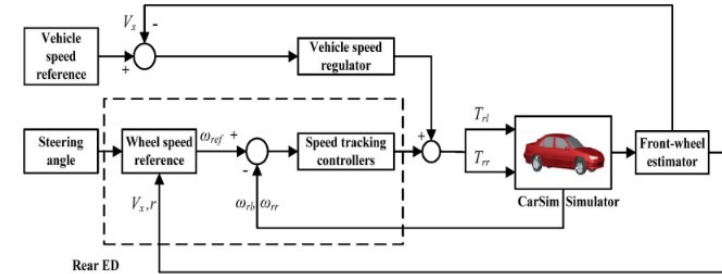


Fig. 7. Configuration of the rear ED design.

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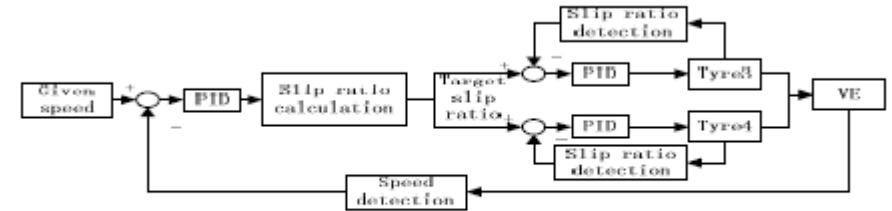
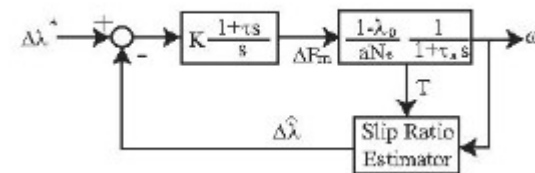


Fig. 4. Block diagram of the closed loop electric differential control strategy

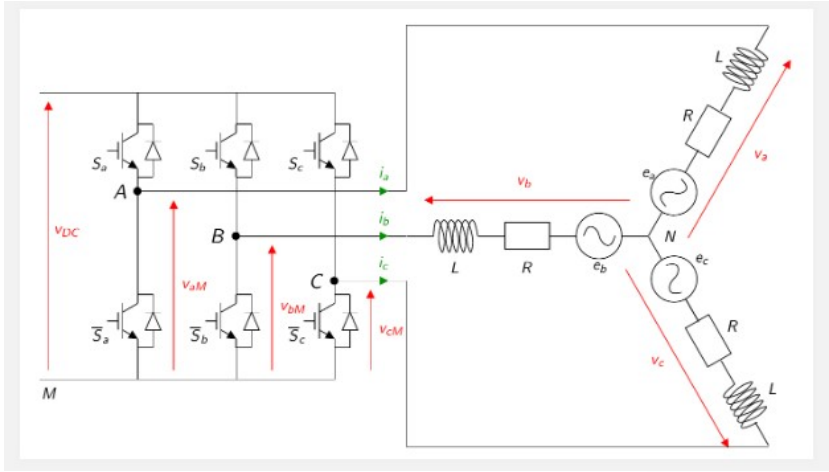
Research on Electric Differential for Steering Electric Vehicles

Zitong Wang, Wei Yao, and Wei Zhang

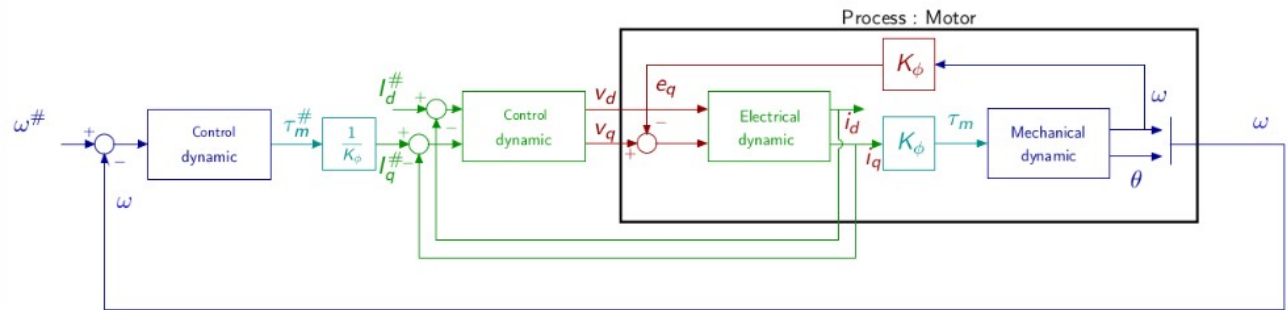


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3) Modèles existants - MSAP



rcp.ctrl-elec.fr



3- Modèles existants - Virage

- Modèle Ackermann-Jeantand (géométrique)

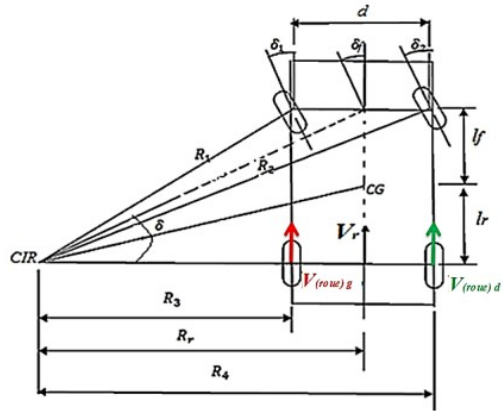


Figure 2. 10 : Modèle d'une trajectoire courbée [29]

D'où :

$$V_{(roue)g} = V - \left(\frac{\tan(\delta) \cdot d / 2}{L} \right) V \quad (2.53)$$

$$V_{(roue)d} = V + \left(\frac{\tan(\delta) \cdot d / 2}{L} \right) V \quad (2.54)$$

3- Modèles existants - Véhicule

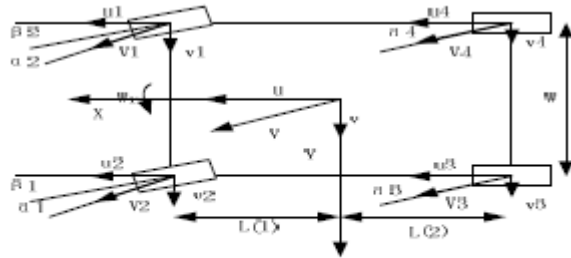


Fig. 1. The speed model of the steering of EV

$$F_{x1} = -f_1 \cos \beta_1 - P_1 \sin \beta_1 .$$

$$F_{x2} = -f_2 \cos \beta_2 - P_1 \sin \beta_2 .$$

$$F_{y1} = P_1 \cos \beta_1 - f_1 \sin \beta_1 .$$

$$F_{y2} = P_2 \cos \beta_2 - f_1 \sin \beta_2 .$$

$$F_{x1} + F_{x2} + F_3 + F_4 + M \cdot v \cdot W_r = M \cdot \dot{u} .$$

$$F_{y1} + F_{y2} + P_3 + P_4 - M \cdot u \cdot W_r = M \cdot \dot{v} .$$

$$(F_{y1} + F_{y2}) \cdot L(1) + \frac{(F_{x2} + F_4 - F_{x1} - F_2) \cdot W}{2} - (P_3 + P_4) \cdot L(2) = I_z \cdot \frac{dW_r}{dt}$$

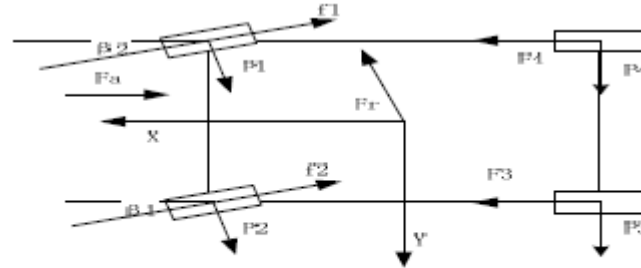


Fig. 3. The force model of the steering EV

$$\begin{cases} u_1 = u - W_r \cdot W / 2 \\ u_2 = u + W_r \cdot W / 2 \\ u_3 = u - W_r \cdot W / 2 \\ u_4 = u + W_r \cdot W / 2 \end{cases} \begin{cases} v_1 = v + W_r \cdot L(1) \\ v_2 = v + W_r \cdot L(1) \\ v_3 = v - W_r \cdot L(2) \\ v_4 = v - W_r \cdot L(2) \end{cases}$$

$$S = 1 - \frac{u_w}{\omega R}$$

$$T_e - M_r - F \cdot R = \frac{d\omega}{dt}$$

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3- Modèles existants - Pneus

The « Magic Formula » by Pacejka

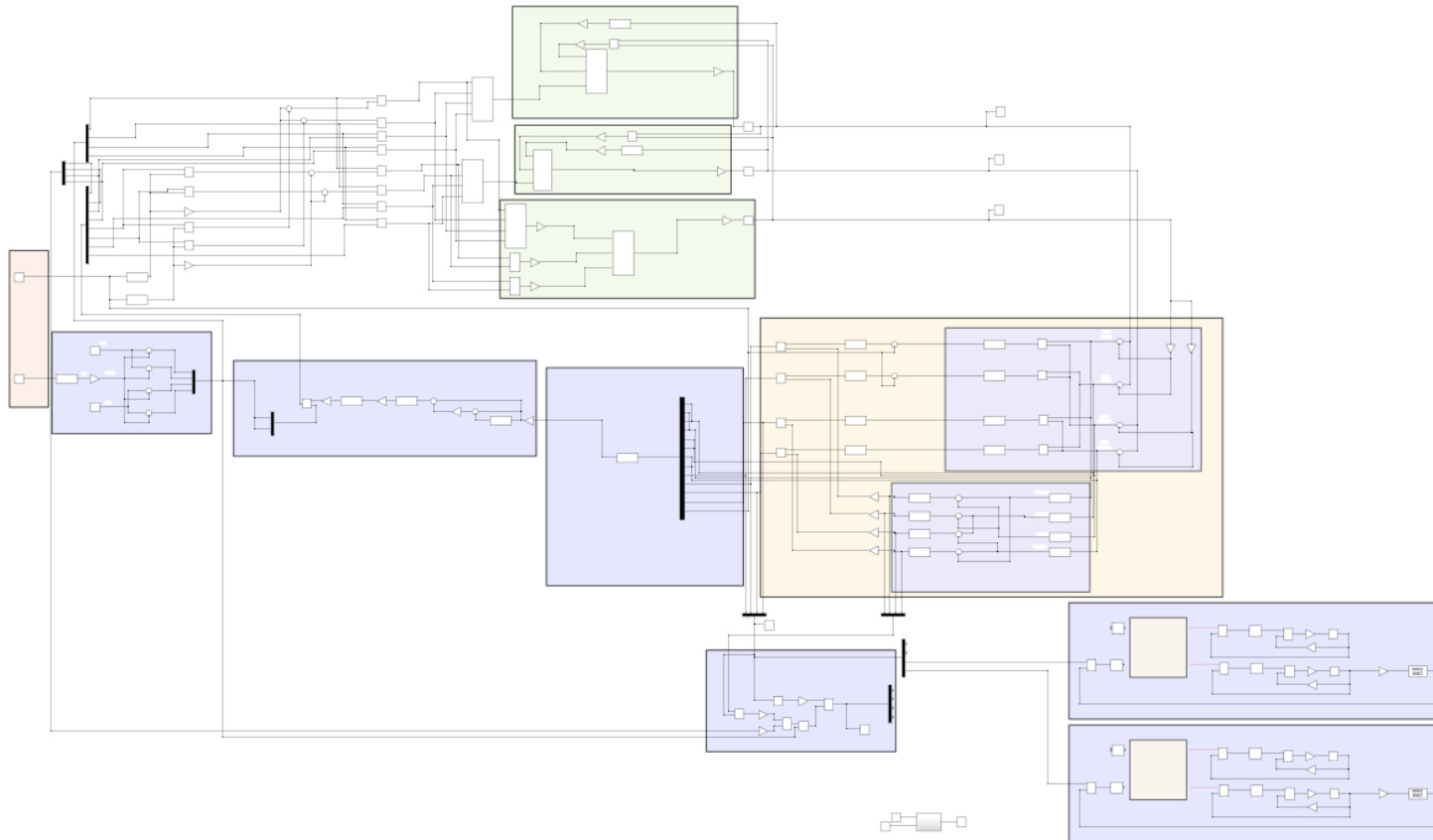
Equation (1) shows the general equation of the Magic Formula tire model [1].

$$F(x) = D \cos(C \arctan(Bx - E(Bx - \arctan(Bx)))) \quad (1)$$

$F(x)$ in the equation is either the longitudinal force F_x with x being represented by the longitudinal slip κ , or the lateral force F_y with x being represented by the lateral slip α . The coefficients B , C , D , and E define the characteristics of the curve and are calculated with additional equations using parameters found in the tire data files.

<https://ieeexplore.ieee.org/stamp/stamp.jsp?tp=&arnumber=9071136>

4) Avancement modèle Simulink



5) Questionnements techniques

- Implémentation à quel endroit ? Dans le VCU ? Si oui quelle puissance de calcul est nécessaire ?
- Quelle limite à l'autre extrémité ? Envoyer un consigne de couple à chaque onduleur ? Un consigne de courant ? Sous quelle forme ?
- A quelles questions doit-on se limiter dans le cadre de ce stage ?
- Comment définir des critères de stabilité, précision, rapidité,... pour une commande de commande ?