

ENPH 257 LAB REPORT

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Abstract

hello world

Chapter 1

Introduction

1.1 Conduction

Heat diffusion due to conduction is given by the heat equation

$$\left(\frac{\partial}{\partial t} u(\mathbf{r}, t) \right)_{\text{cond}} = \frac{k}{c\rho} \nabla^2 u(\mathbf{r}, t), \quad (1.1)$$

where $u(\mathbf{r}, t)$ is the temperature at position \mathbf{r} and time t , k is the thermal conductivity, c is the specific heat of the material, and ρ is the mass density of the material.

1.2 Convection

Power gain due to convection for a differential surface element at position \mathbf{r} may be approximated by

$$P_{\text{conv}}(\mathbf{r}, t) = -k_c \delta S(\mathbf{r}) (u(\mathbf{r}, t) - u_{\text{amb}}), \quad (1.2)$$

where k_c is the convection coefficient, $\delta S(\mathbf{r})$ is the differential surface element at \mathbf{r} , and u_{amb} is the ambient temperature of the surrounding fluid.

Now temperature change due to power for a differential mass element is given by

$$P = \frac{\partial}{\partial t} Q = c\rho \delta V \frac{\partial}{\partial t} u, \quad (1.3)$$

so the rate of change of temperature due to convection is given by

$$\left(\frac{\partial}{\partial t} u(\mathbf{r}, t) \right)_{\text{conv}} = -\frac{k_c \delta S(\mathbf{r})}{c\rho \delta V(\mathbf{r})} (u(\mathbf{r}, t) - u_{\text{amb}}). \quad (1.4)$$

1.3 Radiation

Power gain due to radiation for a differential surface element at position \mathbf{r} is given by Planck's Law to be

$$P_{\text{rad}}(\mathbf{r}, t) = -\epsilon \sigma \delta S(\mathbf{r}) (u(\mathbf{r}, t)^4 - u_{\text{amb}}^4), \quad (1.5)$$

where ϵ is the emissivity and σ is the Stefan-Boltzmann constant.

Again from Equation (1.3), we can obtain the contribution to the rate of change of temperature due to radiation

$$\left(\frac{\partial}{\partial t} u(\mathbf{r}, t) \right)_{\text{rad}} = -\frac{\epsilon \sigma \delta S(\mathbf{r})}{c\rho \delta V(\mathbf{r})} (u(\mathbf{r}, t)^4 - u_{\text{amb}}^4). \quad (1.6)$$

1.4 Heat diffusion equation for a cylindrical rod

For a one-dimensional cylindrical rod of length L and radius R , the heat diffusion equation is

$$\frac{\partial}{\partial t} u(x, t) = \frac{1}{c\rho} \left(k \frac{\partial^2}{\partial x^2} u(x, t) - k_c \frac{\delta S(x)}{\delta V(x)} (u(x, t) - u_{\text{amb}}) - \epsilon \sigma \frac{\delta S(x)}{\delta V(x)} (u(x, t)^4 - u_{\text{amb}}^4) + \frac{P_{\text{in}}(x, t)}{\delta V(x)} \right), \quad (1.7)$$

where P_{in} is the power input function.

Chapter 2

Objective

Chapter 3

Method

3.1 Experimental setup

3.1.1 Rod dimensions

3.1.2 Thermocouple circuit

3.2 Calibration

3.3 Data aquisition

Chapter 4

Results

Chapter 5

Simulation and optimization

5.1 One-dimensional explicit-step simulation

The following is the finite difference equation to simulate heat diffusion based on Equation (1.7) for a cylindrical rod of length L and radius R . Based on the formalism in Appendix A and supposing x points $\{x_i : i \in [0, X]\}$ and t points $\{t_n : n \in [0, T]\}$, the explicit finite difference form of Equation (1.7) is

$$\mathbf{u}^{n+1} = \mathbf{u}^n + \delta t (\delta \mathbf{u}_{\text{cond}}^n + \delta \mathbf{u}_{\text{conv}}^n + \delta \mathbf{u}_{\text{rad}}^n + \delta \mathbf{u}_{\text{pow}}^n), \quad (5.1)$$

where

$$\begin{aligned} (\delta \mathbf{u}_{\text{cond}}^n)_i &= \frac{k}{c\rho} \cdot \frac{1}{\delta x^2} \cdot \begin{cases} (u_1^n - u_0^n), & i = 0 \\ (u_{X-1}^n - u_X^n), & i = X \\ (u_{i-1}^n - 2u_i^n + u_{i+1}^n), & \text{else} \end{cases} \\ (\delta \mathbf{u}_{\text{conv}}^n)_i &= -\frac{k_c}{c\rho} \cdot \frac{\delta S_i}{\delta V_i} (u_i^n - u_{\text{amb}}) \\ (\delta \mathbf{u}_{\text{rad}}^n)_i &= -\frac{\epsilon \sigma}{c\rho} \cdot \frac{\delta S_i}{\delta V_i} ((u_i^n)^4 - u_{\text{amb}}^4) \\ (\delta \mathbf{u}_{\text{pow}}^n)_i &= \frac{P_i^n}{c\rho \delta V_i} \end{aligned}$$

The differential volume at any point x_i is the same

$$\delta V_i = \pi R^2 \delta x.$$

The exposed surface is different at the ends, however, with

$$\delta S_i = 2\pi R \delta x + \begin{cases} \pi R^2, & i = 0 \text{ or } i = X \\ 0, & \text{else} \end{cases}$$

Finally power input occurs only at one end. It is assumed that power is a constant P_1 during heating and P_2 during cooling. Heating occurs until t_{stop} at which point cooling begins. Thus we have

$$P_i^n = \begin{cases} P_1, & t_0 < t_n < t_{\text{stop}} \\ P_2, & t_{\text{stop}} \leq t_n \leq t_T \end{cases}$$

5.2 Optimization of one-dimensional simulation

A numerical optimization routine was used to optimize thermodynamical parameters by repeatedly varying these parameters, running the one-dimensional simulation, and comparing the results to experimental data until optimal parameters were achieved. This optimization was run for each dataset. For each optimization, 67 positional points and a time step of .5 seconds were used.

5.3 High-level simulation

Chapter 6

Analysis

Chapter 7

Conclusion and Summary

Appendix A

Comparison of Finite Differencing Methods

The following are methods of finite differencing, demonstrated using the simple example of the one-dimensional heat conduction equation

$$\frac{\partial}{\partial t}u(x, t) = \alpha \frac{\partial^2}{\partial x^2}u(x, t). \quad (\text{A.1})$$

A.1 Explicit method

Finite difference equation

In the explicit method for finite differencing, the temperature for each spatial point can be independently determined from the previous step. That is,

$$u_i^{n+1} = u_i^n + \alpha \cdot \frac{\delta t}{\delta x^2} \cdot (u_{i-1}^n - 2u_i^n + u_{i+1}^n), \quad (\text{A.2})$$

where the superscript $n \in [0, T]$ signifies the step in time t_n and the subscript $i \in [0, X]$ represents the position x_i . δt and δx are the time steps in time and position respectively.

If we define

$$\zeta = \alpha \cdot \frac{\delta t}{\delta x^2},$$

then this can be written as a linear relationship

$$\mathbf{u}^{n+1} = \mathbf{D}\mathbf{u}^n, \quad (\text{A.3})$$

where \mathbf{D} is the matrix defined by

$$\mathbf{D}_{ij} = \begin{cases} \text{BC}, & i = 0 \text{ or } i = X \\ \zeta, & 0 < i < X \text{ and } |i - j| = 1 \\ 1 - 2\zeta, & 0 < i < X \text{ and } i - j = 0 \\ 0, & \text{else} \end{cases} \quad (\text{A.4})$$

The top and bottom rows ($i = 0$ and $i = X$) are determined from boundary conditions. Note that as ζ approaches 0, \mathbf{D} approaches the identity matrix. Thus ζ should be as small as possible for the best approximation. The condition for stability is

$$0 < \zeta < 1.$$

Error in time

The explicit method uses a forward difference in time. The expression for the derivative approximation is derived from the power series expansion in δt

$$u(x, t + \delta t) = u(x, t) + \frac{\partial}{\partial t} u(x, t) \cdot \delta t + \frac{\partial^2}{\partial t^2} u(x, t) \cdot \frac{\delta t^2}{2!} + \dots \quad (\text{A.5})$$

Thus we have an approximation for the time derivative

$$\frac{u(x, t + \delta t) - u(x, t)}{\delta t} = \frac{\partial}{\partial t} u(x, t) + O(\delta t). \quad (\text{A.6})$$

The error in time due to this approximation is linear in δt .

Error in position

The explicit method uses a central second derivative in space. The expression for the second derivative approximation is derived from the power series expansion in δx .

$$u(x + \delta x, t) = u(x, t) + \frac{\partial}{\partial x} u(x, t) \cdot \delta x + \frac{\partial^2}{\partial x^2} u(x, t) \cdot \frac{\delta x^2}{2!} + \frac{\partial^3}{\partial x^3} u(x, t) \cdot \frac{\delta x^3}{3!} + \dots \quad (\text{A.7})$$

At $-\delta x$, this is

$$u(x - \delta x, t) = u(x, t) - \frac{\partial}{\partial x} u(x, t) \cdot \delta x + \frac{\partial^2}{\partial x^2} u(x, t) \cdot \frac{\delta x^2}{2!} - \frac{\partial^3}{\partial x^3} u(x, t) \cdot \frac{\delta x^3}{3!} + \dots$$

Thus the approximation for the second spatial derivative is

$$\frac{u(x - \delta x, t) - 2u(x, t) + u(x + \delta x, t)}{\delta x^2} = \frac{\partial^2}{\partial x^2} u(x, t) + O(\delta x^2). \quad (\text{A.8})$$

The error in position due to this approximation is quadratic in δx .

A.2 Implicit method

A.3 Crank-Nicolson method