

VIDEOS

Quadcopter Acrobatics

<https://youtu.be/XxFZ-VStApo>

Inverted Pendulum

<https://youtu.be/a4c7AwHFkT8>

Questions to consider

- How are these similar?
- How are these different?
- How are these tasks accomplished?
- How will you accomplish these tasks?

MODELING PHYSICAL SYSTEMS

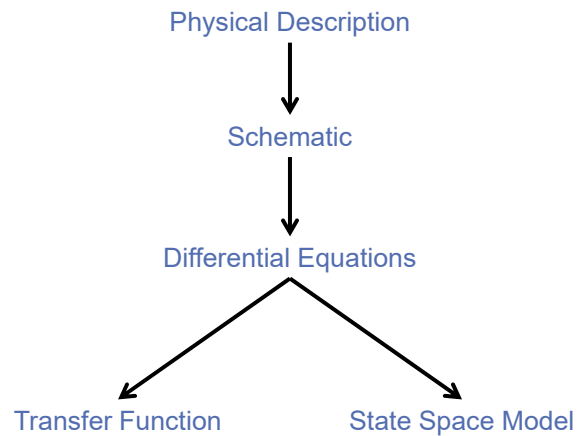
Topics

- Why we model dynamic systems
- How we model dynamic systems

At the end of this section, students should be able to:

- Model dynamic systems with ODEs.
- Model dynamic systems with transfer functions.
- Model dynamic systems with state-space representation.

HOW DO WE MODEL A PHYSICAL SYSTEM FOR CONTROL?



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WE LIKE TO MAKE SEVERAL MODELING ASSUMPTIONS

Linear – obeys linear superposition

Time Invariant – constant parameters

Lumped Parameters – characteristics are lumped into discrete elements

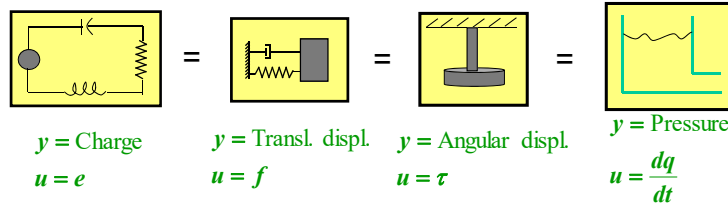
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DYNAMIC SYSTEMS

Models are the same regardless of the physical domain:



We only have to understand one model, but we have four different systems!

$$\ddot{y} + \dot{y} + y = u(t)$$

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IDEALIZED MODELING ELEMENTS

Basic Components

- Inertia
- Energy Storage
- Energy Dissipation

Mechanical – Force driven

- Mass
- Linear Spring
- Linear Damper

Electrical – Voltage driven

- Inductor
- Capacitor
- Resistor

Thermal

- Thermal capacitance (specific heat)
- Thermal resistance from conduction or convection (actually energy transfer, not dissipation)

Hydraulic

- Usually distributed (i.e. along a length of pipe)
- Lumped approximations may be used

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TERMINOLOGY

Nominal Model

- Simplified approximate model used for control design

Calibration (or Simulation) Model

- Model that captures all pertinent aspects of plant behavior to be used for controller validation

Model Error

- The difference between the nominal and calibration model

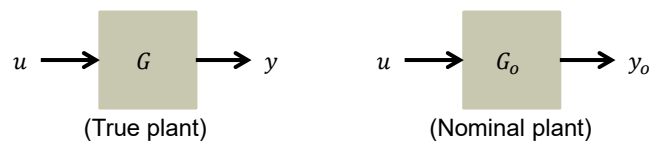
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MODEL ERRORS

Let y_o be the nominal system output and y be the actual system output



Additive Modeling Error (AME):

$$AME = y - y_o = [G - G_o]U(s) = \Delta G \cdot U(s)$$

Multiplicative Modeling Error (MME):

$$Y(s) = G(s)U(s) = [G_o + \Delta G]U(s) = G_o \left(1 + \frac{\Delta G}{G_o} \right) U(s)$$

MME

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BUILDING MODELS

Three approaches:

- From physical principles (white-box model)
- From experimental data (black-box model)
- Combination of the above

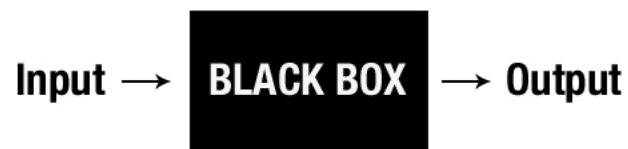
Always use the simplest model that captures the essential aspects of the process for control design.

WHAT PROBLEMS EXIST WITH A WHITE-BOX MODEL?



Damping coefficients (friction) can be difficult (or impossible) to estimate

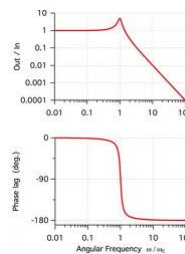
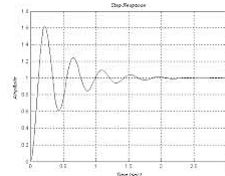
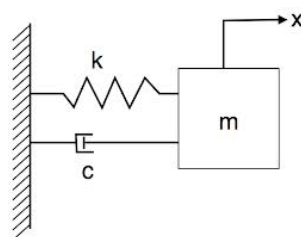
WHAT PROBLEMS EXIST WITH A BLACK-BOX MODEL?



How do you select a model structure?

Can you gain any intuition from the model?

CAN WE REMEDY THESE PROBLEMS WITH A GRAY-BOX MODEL?



$$G(s) = \frac{K_s \omega_n^2}{s^2 + 2\zeta \omega_n s + \omega_n^2}$$

Use what we know (intuition) as a starting point

Fill in the gaps using data

DEFINITIONS

State

- The smallest set of n variables (**state variables**) such that knowledge of these n variables at $t = t_0$, together with knowledge of the input for $t \geq t_0$, completely and uniquely determines system behavior for $t \geq t_0$.

State vector

- n^{th} order vector whose components are the state variables

State space

- n -dimensional space whose coordinate axes consist of the x_1 axis, x_2 axis, etc.

State trajectory

- Path produced in the state space by the state vector as it changes over time

FOR LTI SYSTEMS WE CAN SIMPLIFY THE GENERAL STATE SPACE FORM

General state space equations:

$$\begin{aligned}\dot{x}(t) &= f(x, u, t) \\ y(t) &= g(x, u, t)\end{aligned}$$

Assuming system is linear:

$$\begin{aligned}\dot{x}(t) &= A(t)x(t) + B(t)u(t) \\ y(t) &= C(t)x(t) + D(t)u(t)\end{aligned}$$

Assuming system is time-invariant:

$$\begin{aligned}\dot{x}(t) &= Ax(t) + Bu(t) \\ y(t) &= Cx(t) + Du(t)\end{aligned}$$

WHAT IF WE HAVE A NONLINEAR STATE SPACE MODEL?

$$\begin{aligned}\dot{x}(t) &= f(x, u, t) \\ y(t) &= g(x, u, t)\end{aligned}$$

Choose operating point: x_Q, u_Q

$$\dot{x}(t) \approx f(x_Q, u_Q) + \left. \frac{\partial f}{\partial x} \right|_{x=x_Q, u=u_Q} (x(t) - x_Q) + \left. \frac{\partial f}{\partial u} \right|_{x=x_Q, u=u_Q} (u(t) - u_Q)$$

$$y(t) \approx g(x_Q, u_Q) + \left. \frac{\partial g}{\partial x} \right|_{x=x_Q, u=u_Q} (x(t) - x_Q) + \left. \frac{\partial g}{\partial u} \right|_{x=x_Q, u=u_Q} (u(t) - u_Q)$$

WHEN LINEARIZING ABOUT THE ORIGIN, THE LINEARIZATION IS CLEAR

$$\begin{aligned}x_Q &= 0 \\ u_Q &= 0\end{aligned}$$

$$\dot{x}(t) \approx Ax(t) + Bu(t) + f(0,0)$$

$$y(t) \approx Cx(t) + Du(t) + g(0,0)$$

- For other operating points we might consider an incremental model (see textbook)

CONTROL THEORY

Classical Control Theory

- Single-Input Single-Output (SISO) systems
- Linear Time-Invariant (LTI) systems
- Developed without computers
- Graphical tools, algebraic manipulation

Modern Control Theory

- Multiple-Input Multiple-Output (MIMO) systems
- Nonlinear systems
- Easy access to computers
- Numeric solutions, matrix manipulation

COMMENTS ON MODEL TYPES

Transfer Function Models:

- Are conceptually simple
- Are easily converted to frequency domain transfer functions that are more intuitive to practicing engineers
- Are difficult to solve in the time domain (solution: Laplace transformation)

State Space Models:

- Consider the internal behavior of a system
- Can easily incorporate complicated output variables
- Have significant computation advantage for computer simulation
- Can represent multi-input multi-output (MIMO) systems and nonlinear systems

COMING UP...

Case Study: Hard disk drive read/write head

Transfer Functions

- Poles and zeros
- First and second order systems

Dynamic Response

- Types of inputs
- Free and Force responses
- Transient and Steady-State responses