

FROM LAST TIME...

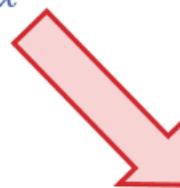
State Observers

- Rationale
- Observer Design
- Method 1 - "pole placement"
- Method 2 (Ackermann's Formula)

We know:

$$\dot{x} = Ax + Bu$$

$$y = Cx$$



Measure:

$$u \text{ and } y$$



Calculate:

$$x$$

$$L = T \begin{bmatrix} \alpha_0 - a_0 \\ \vdots \\ \alpha_{n-1} - a_{n-1} \end{bmatrix}$$

From ISI-AI
CHARACTERISTIC EQUATION

"pole placement"

A-LC

$$L = \phi(A)W_O^{-1} \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 1 \end{bmatrix}$$

ACKERMANN'S FORMULA

STATE OBSERVER + FEEDBACK

Topics

- Output feedback
- Reduced-order observer

At the end of this section, students should be able to:

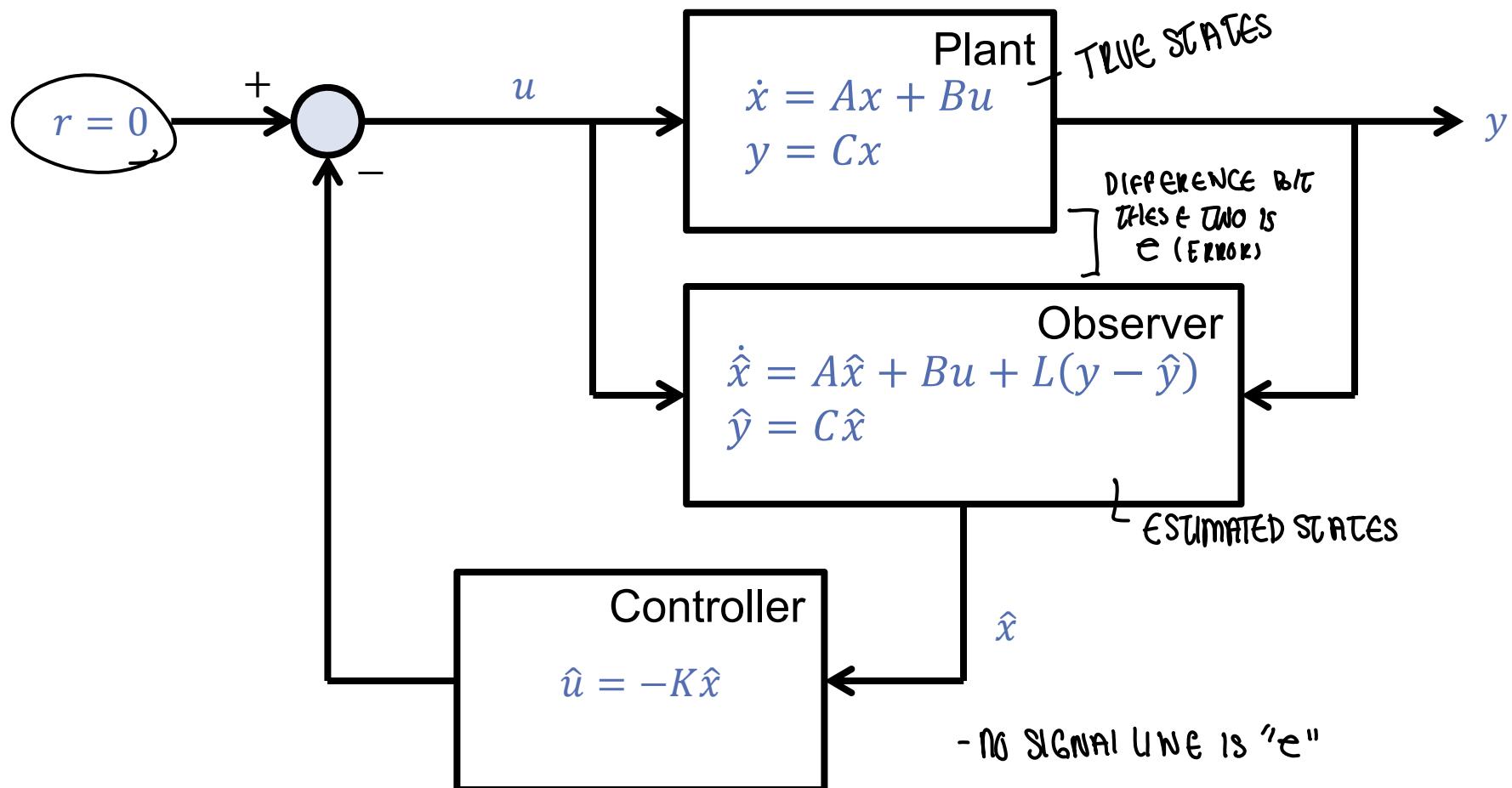
- Combine state observer and feedback into output feedback.
- Explain the benefit of a reduced-order state observer.
- Design a reduced-order state observer.

STATE OBSERVER + FEEDBACK

OUTPUT FEEDBACK

MEASURE INPUT
MEASURE OUTPUT
ESTIMATE STATES

LET'S COMBINE THE STATE OBSERVER WITH STATE FEEDBACK TO CREATE OUTPUT FEEDBACK



\wedge STATE DEFINE ESTIMATION ERROR AS

$$e = x - \hat{x} \quad U = -K \hat{x}$$

"WE NEED TO INVENT
MORE LETTERS"- MYNDERSE

$$\begin{aligned}\dot{\hat{x}} &= Ax + Bu = Ax - BK\hat{x} \\ &\quad + BKx - BKx \\ \dot{\hat{x}} &= (A - BK)\hat{x} + BKe\end{aligned}$$

FROM OBSERVER DYNAMICS $\dot{e} = (A - LC)e$

$$\begin{bmatrix} \dot{x} \\ \dot{e} \end{bmatrix} = \begin{bmatrix} A - BK & BK \\ 0 & A - LC \end{bmatrix} \begin{bmatrix} x \\ e \end{bmatrix}$$

REGULATOR CLOSED LOOP
DYNAMICS W/ OBSERVER

CLOSED-LOOP
FREE RESPONSE

CLOSED LOOP EIGEN VALUES

$$\det(SI - \begin{bmatrix} A - BK & BK \\ 0 & A - LC \end{bmatrix}) = |SI - A + BK| \cdot |SI - A + LC|$$

POLES FROM STATE FEEDBACK POLES FROM STATIC OBSERVER

DET(SI - A) =

$$(SI - A + BK)(SI - A + LC)$$

SYSTEM WILL BE A COMBINATION OF POLES

FIND THE CLOSED-LOOP EIGENVALUES

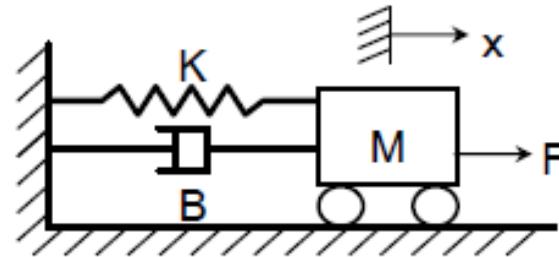
SEPARATION PROPERTY ALLOWS US TO SEPARATE THE STATE OBSERVER AND STATE FEEDBACK DESIGNS

In an output feedback controller, the state feedback controller and state observer can be independently designed provided that . . .

- 1.** A, B, and C are known exactly
- 2.** (A, B) controllable
- 3.** (A, C) observable

APPLY SEPARATION PROPERTY TO AN EXAMPLE

$$\begin{aligned}\dot{x} &= \begin{bmatrix} 0 & 1 \\ -1 & -1 \end{bmatrix}x + \begin{bmatrix} 0 \\ 1 \end{bmatrix}u \\ y &= [1 \ 0]x\end{aligned}$$



State feedback gains to give $\zeta_{CL} = 0.7$ and $\omega_{nCL} = 5$ r/s:

$$K = [24 \quad 6]$$

Observer poles at $s = -18$ requires observer gains:

$$L = \begin{bmatrix} 35 \\ 288 \end{bmatrix}$$

$$\dot{X} = (A - BK)X + BKe$$

$$= \left(\begin{bmatrix} 0 & 1 \\ -1 & -1 \end{bmatrix} - \begin{bmatrix} 0 \\ 1 \end{bmatrix} \begin{bmatrix} 24 & 6 \end{bmatrix} \right) \times \begin{bmatrix} 0 \\ 1 \end{bmatrix} \begin{bmatrix} 24 & 6 \end{bmatrix} e$$

$$\dot{e} = (A - \frac{1}{2}C)e$$

L DUE TO
OBSERVER

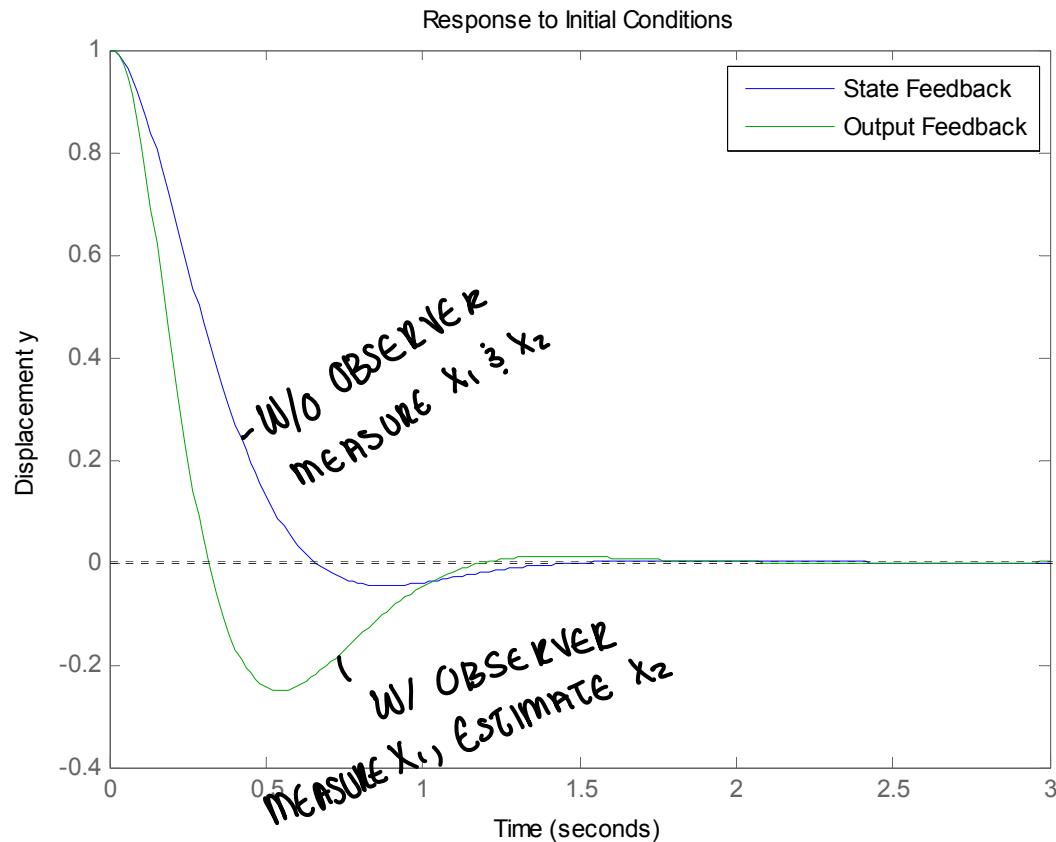
$$= \left(\begin{bmatrix} 0 & 1 \\ -1 & -1 \end{bmatrix} - \begin{bmatrix} 35 \\ 288 \end{bmatrix} \begin{bmatrix} 1 & 0 \end{bmatrix} \right) e$$

$e = x - \hat{x}$

FREE RESPONSE OF THE
CLOSED LOOP SYSTEM

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{e}_1 \\ \dot{e}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ -25 & -7 & 24 & 6 \\ 0 & 0 & -35 & 1 \\ 0 & 0 & -289 & -1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ e_1 \\ e_2 \end{bmatrix}$$

SIMULATE THE CLOSED-LOOP SYSTEM

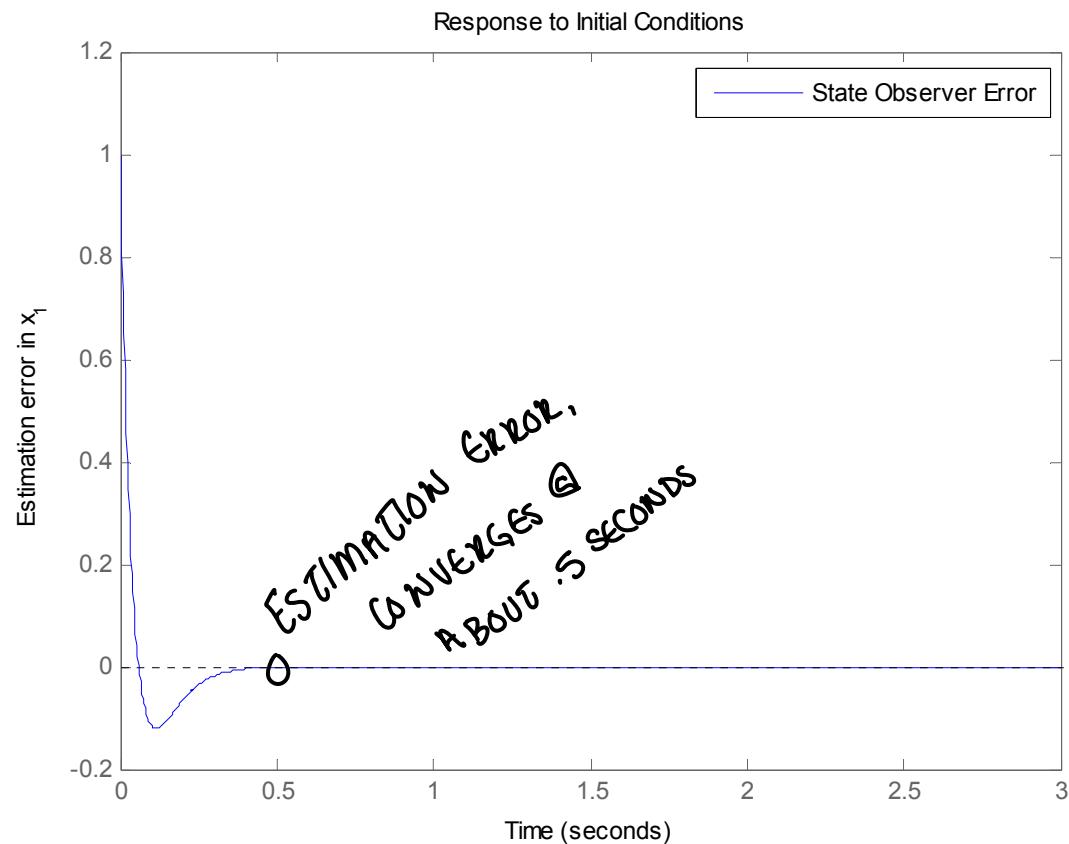


$$x(0) = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

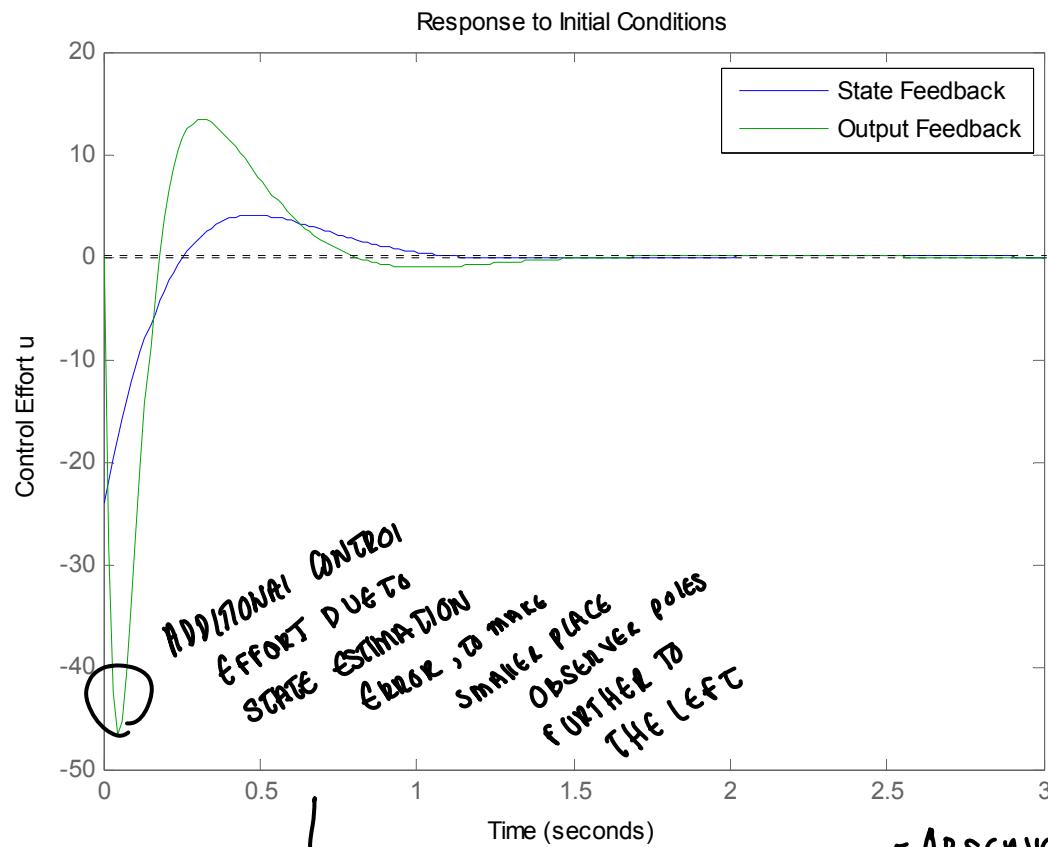
$$\hat{x}(0) = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$e(0) = \begin{bmatrix} 1 \\ 0 \end{bmatrix} - \begin{bmatrix} 0 \\ 0 \end{bmatrix} = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

HOW WELL DID THE STATE OBSERVER PERFORM?



WHAT CONTROL EFFORT WAS REQUIRED?



PLOTTED USING
INITIAL(sys, x_0)
STATE SPACE
INITIAL CONDITIONS

$$A = \begin{bmatrix} & \end{bmatrix} \quad B = \begin{bmatrix} & \\ & \end{bmatrix}$$

$$C = [-k_1, -k_2, k_1, k_2] X$$

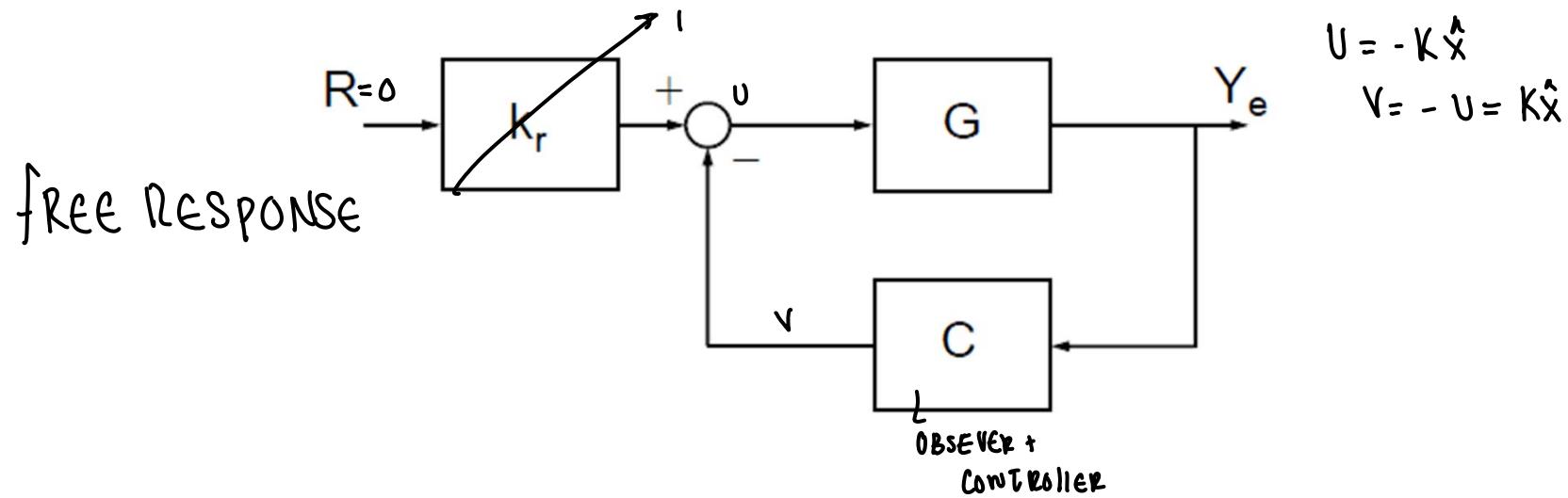
- OBSERVER POLES SHOULD BE 5X FASTER THAN THE CONTROLLER POLES

$$u = -K\hat{x}$$

$$= -K(x - e)$$

$$= [-K \quad K] \begin{bmatrix} x \\ e \end{bmatrix}$$

CAN WE CONVERT THE COMBINED OBSERVER + CONTROLLER TO TRANSFER FUNCTION?



OBSERVER DYNAMICS

$$S\hat{x}(s) = (A - LC)\hat{x}(s) + BU(s) + LV(s)$$

$$\hat{x}(s) = (sI - A + LC + BK)^{-1}LV(s)$$

WHERE $S\hat{x} = S(I - \hat{x})$

e.g. $\begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$

THE FEEDBACK COMPENSATOR IS GIVEN BY...



$$\dot{x}_C = (A - BK - LC)x_C + Ly$$
$$v = Kx_C$$

OUTPUT / INPUT
OF THE OUTPUT
FEEDBACK COMPENSATOR

$$\begin{aligned} \dot{x} &= Ax + Bu \\ y &= cx \\ u &= C(sI - A)^{-1}B \end{aligned}$$

$$C(s) = \frac{V(s)}{Y(s)} = K(sI - A + LC + BK)^{-1}L$$

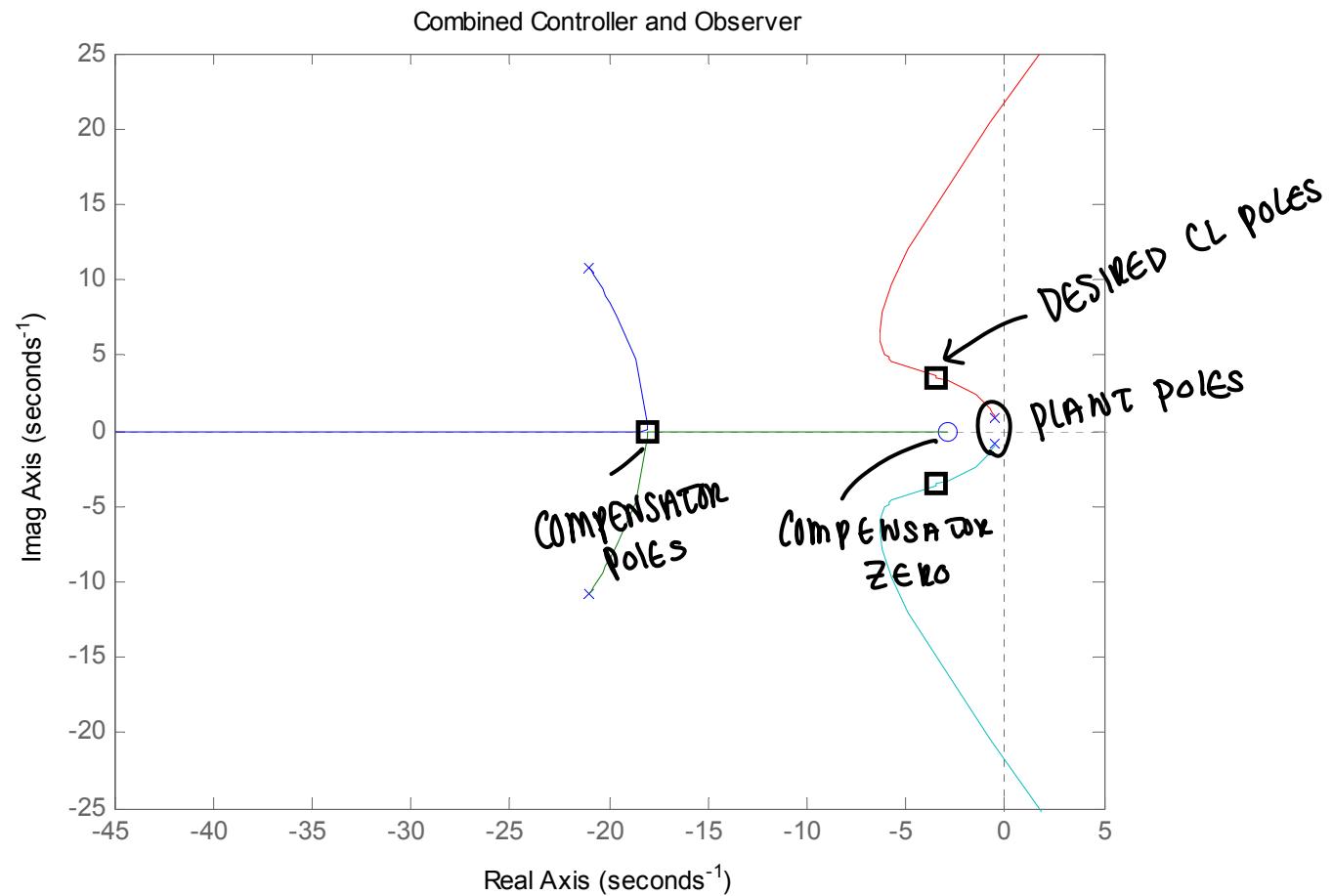
$$= [24 \quad 6] \begin{bmatrix} s + 35 & -1 \\ 313 & s + 7 \end{bmatrix}^{-1} \begin{bmatrix} 35 \\ 288 \end{bmatrix}$$

$$= \frac{2568s + 7542}{s^2 + 42s + 588}$$

WHY DO WE CARE

- IT'S A GOOD WAY TO LOOK @ THINGS - MYNDERSE

EXAMINE THE ROOT LOCUS OF THE CLOSED-LOOP SYSTEM



LET'S COMPARE STATE FEEDBACK AND OUTPUT FEEDBACK COMPENSATORS FURTHER

$$C(s) = 6s + 24$$

MEASURE

$$x_1, \dot{x}_2$$

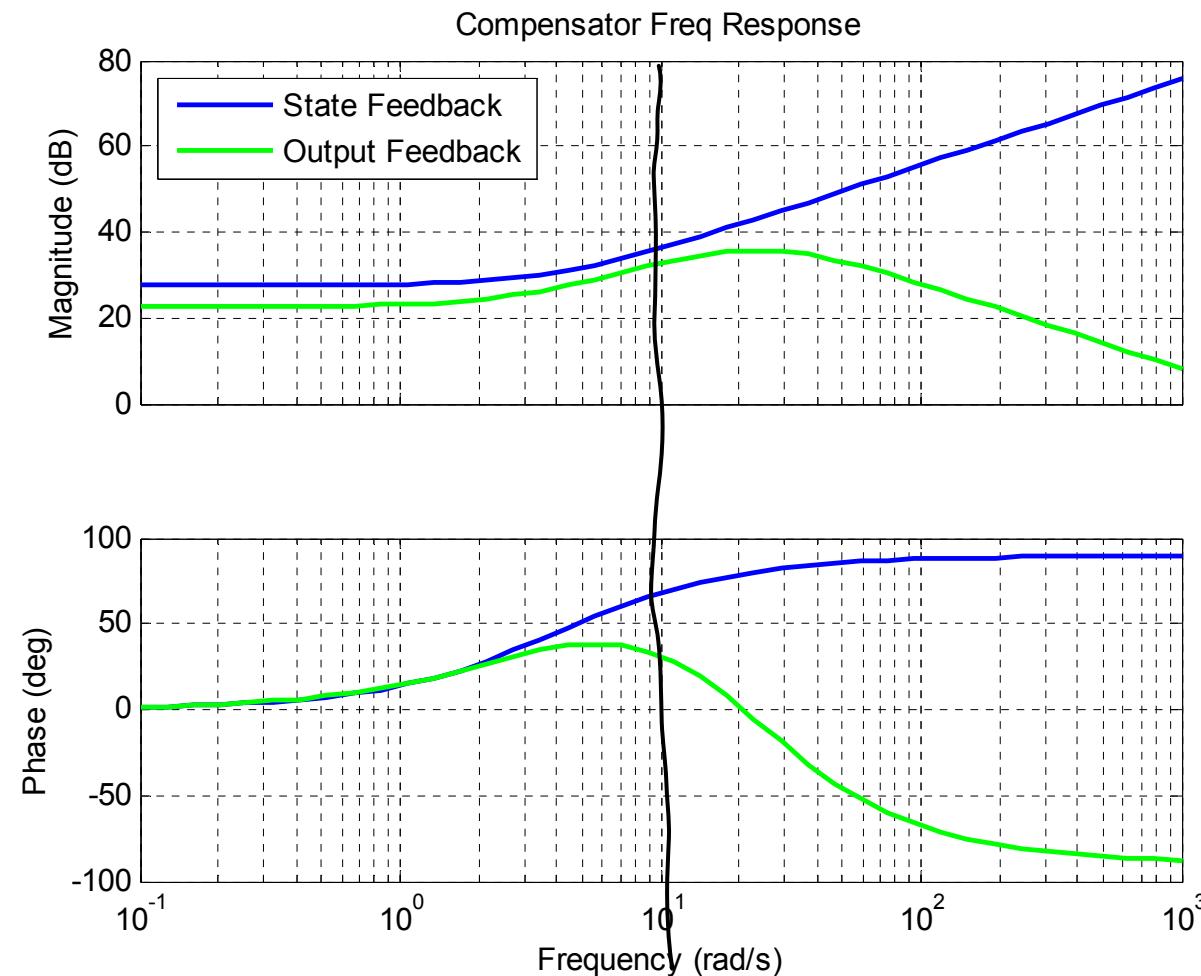
C WI OBSERVER

$$C(s) = \frac{2568s + 7542}{s^2 + 42s + 558}$$

MEASURE ONLY OUTPUT Y

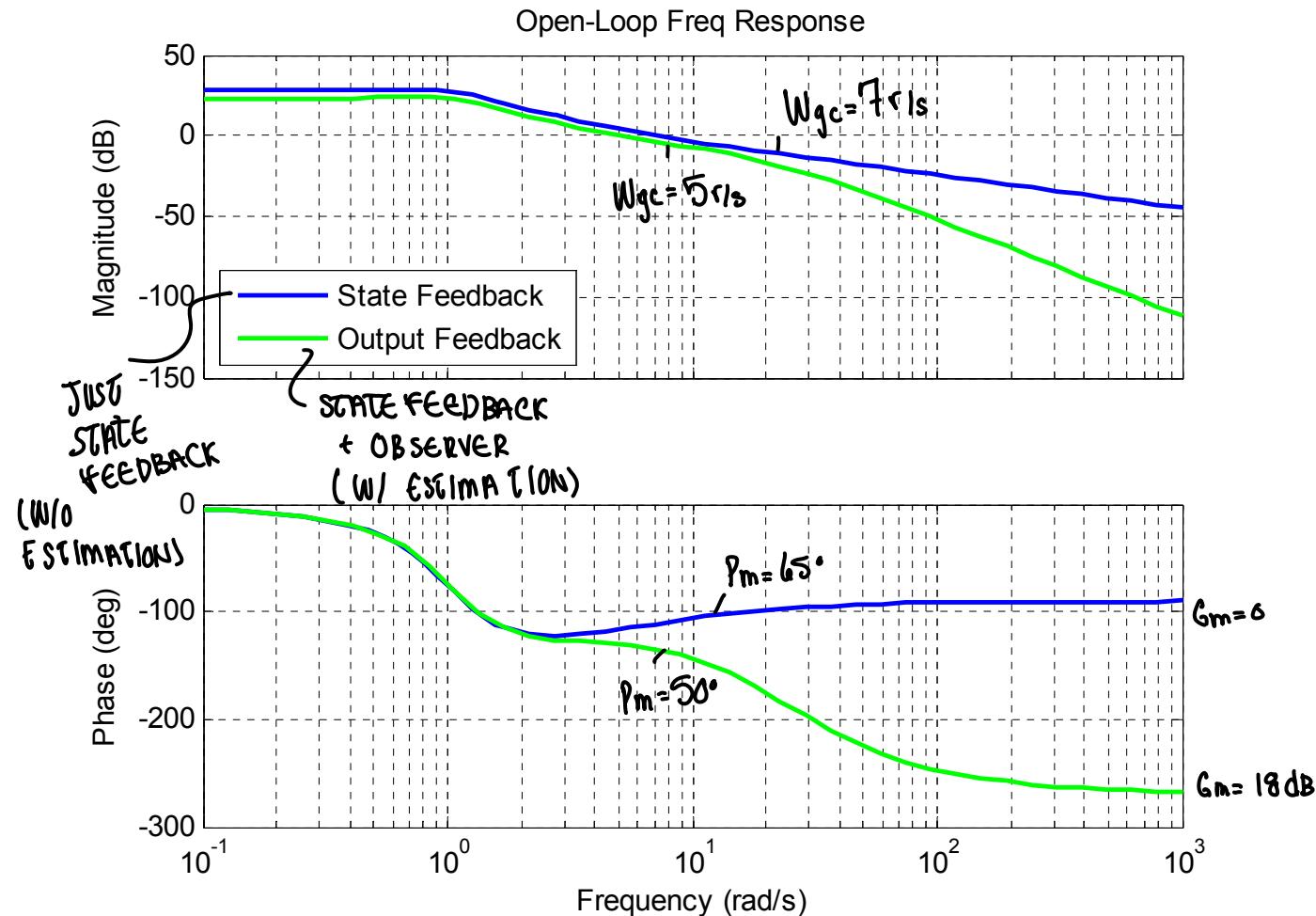
The output feedback compensator is strictly proper

COMPENSATOR FREQUENCY RESPONSE IS SIMILAR BELOW 10 r/s



SIMILARLY, THE OPEN LOOP ($L = CG$) FREQUENCY RESPONSES ARE COMPARABLE BELOW 10 r/s.

CONTROLLER
vs OBSERVER



WHAT IS THE EFFECT OF THE OBSERVER POLES?

Let: $|sI - A + LC| = (s + p)^2 = s^2 + 2ps + p^2$

$\xrightarrow{(s+18)^2}$
 $p = -18$ or $(s - (-p))$

$$L = \begin{bmatrix} \ell_1 \\ \ell_2 \end{bmatrix} = T \begin{bmatrix} \alpha_0 - a_0 \\ \alpha_1 - a_1 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 1 & -1 \end{bmatrix} \begin{bmatrix} p^2 - 1 \\ 2p - 1 \end{bmatrix} = \begin{bmatrix} 2p - 1 \\ p^2 - 2p \end{bmatrix}$$

L
L IS DIRECTLY RELATED TO P

$$C(s) = [24 \quad 6] \begin{bmatrix} s + \ell_1 & -1 \\ \ell_2 + 25 & s + 7 \end{bmatrix}^{-1} \begin{bmatrix} \ell_1 \\ \ell_2 \end{bmatrix}$$

$$= \frac{(24\ell_1 + 6\ell_2)s + (18\ell_1 + 24\ell_2)}{s^2 + (\ell_1 + 7)s + (7\ell_1 + \ell_2 + 25)}$$

Next, substitute the known observer gains to observe pole effects!

$$C(s) = \frac{6(p^2 + 6p - 4)s + (24p^2 - 12p - 18)}{s^2 + (2p + 6)s + (p^2 + 12p + 18)} \cdot \frac{\frac{1}{p^2}}{\frac{1}{p^2}}$$

NOT NECESSARY
 just makes the
 math easier
 to see
 , I CAN'T
 spell

$$C(s) = \frac{6\left(1 + \frac{6}{p} - \frac{4}{p^2}\right)s + \left(24 - \frac{12}{p} - \frac{18}{p^2}\right)}{\frac{1}{p^2}s^2 + \left(\frac{2}{p} + \frac{6}{p^2}\right)s + \left(1 + \frac{12}{p} + \frac{18}{p^2}\right)}$$

$p \rightarrow \infty$

$$p \rightarrow \infty, \quad C(s) \rightarrow 6s + 24$$

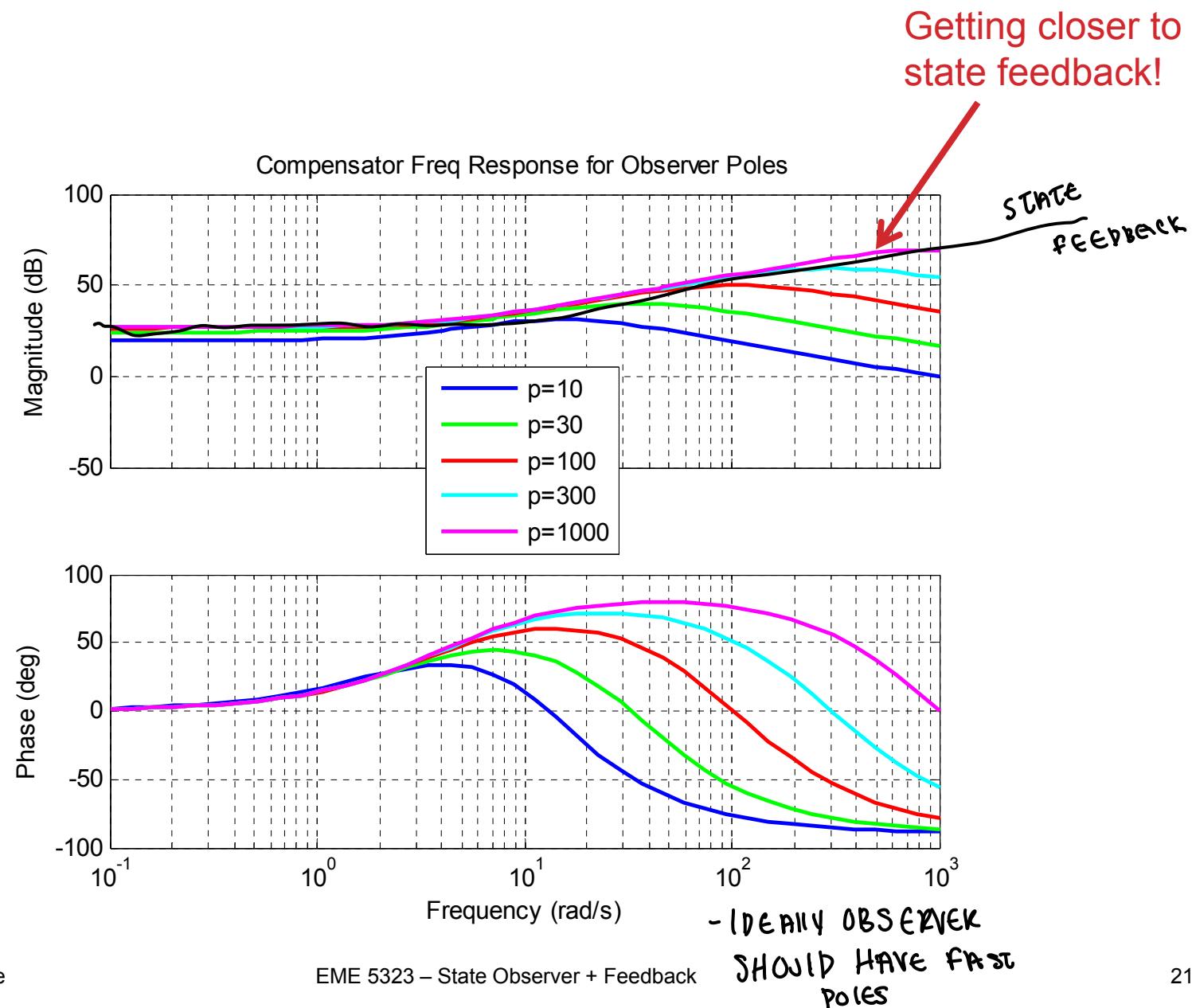
As the observer poles become infinitely fast, the observer error converges to zero in infinitesimal time.

Then we have state feedback!

↓ like unto
AN OBSERVER

LEARNER WILL
BE SMALL
BETWEEN TRUE &
ESTIMATED MODEI

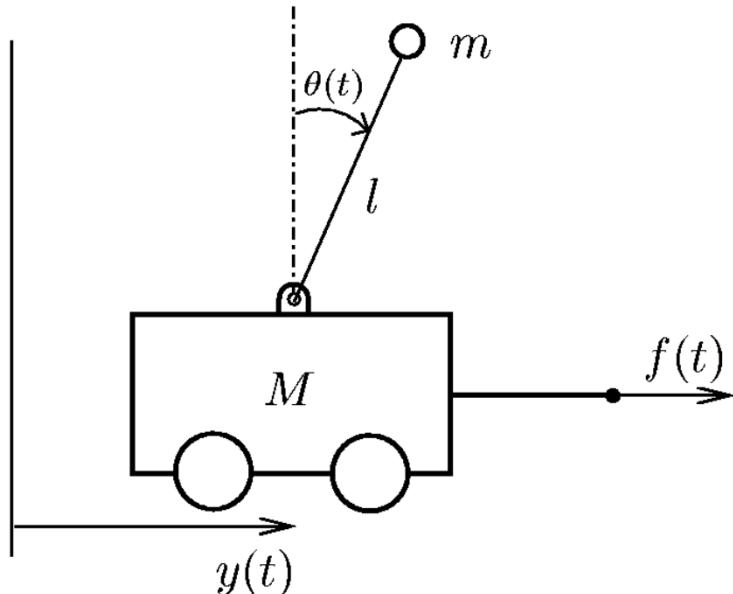
LET THE VALUE OF p VARY



STATE OBSERVER + FEEDBACK

REDUCED-ORDER OBSERVER

RECALL THE ROBOTIC WELDING PROBLEM

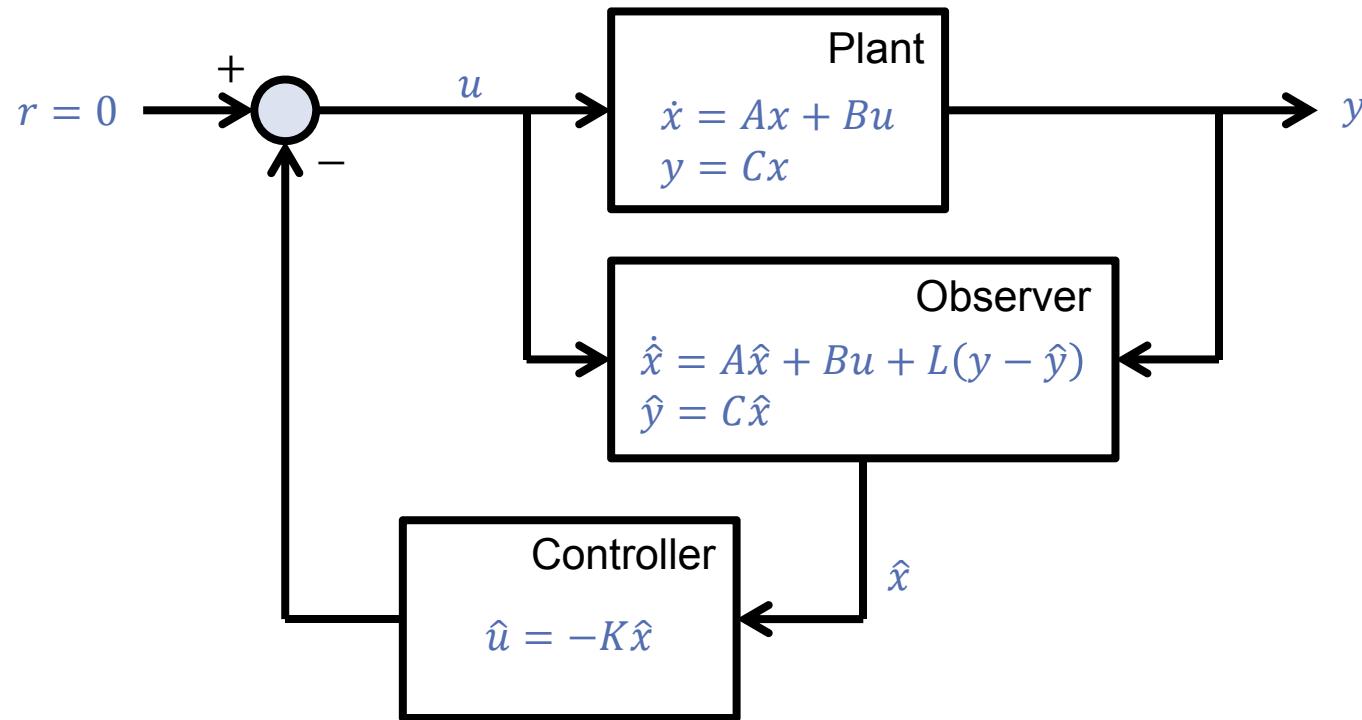


$$\begin{bmatrix} \dot{y} \\ \ddot{y} \\ \dot{\theta} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & -10 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & 20 & 0 \end{bmatrix} \begin{bmatrix} y \\ \dot{y} \\ \theta \\ \dot{\theta} \end{bmatrix} + \begin{bmatrix} 0 \\ 2 \\ 0 \\ -2 \end{bmatrix} f(t)$$
$$\begin{bmatrix} y \\ \dot{y} \\ \theta \\ \dot{\theta} \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} \dot{y} \\ \ddot{y} \\ \dot{\theta} \\ \ddot{\theta} \end{bmatrix}$$

System is fully state **observable** with output y

- Choose y for output feedback design

DESIGN A STATE OBSERVER AND STATE FEEDBACK



Notice that we can measure output $x_1 = y$, but our state feedback instead uses $\hat{x}_1 = \hat{y}$.

Why not use the measurement instead of the estimate?

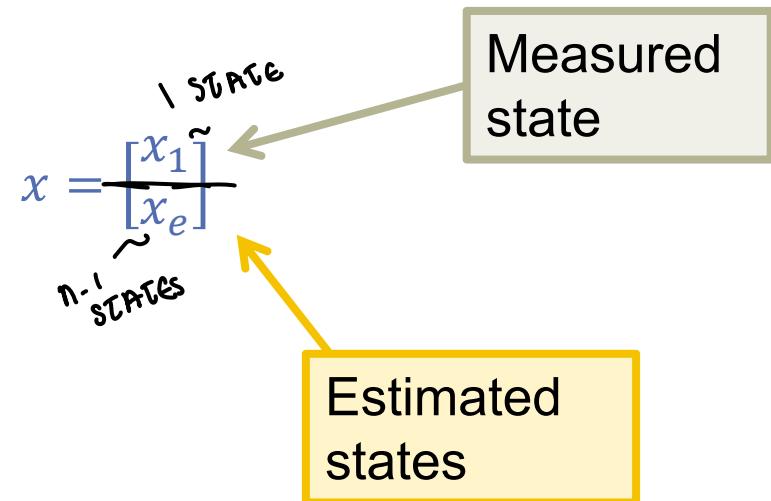
DEFINE A REDUCED ORDER OBSERVER USING m LINEARLY INDEPENDENT MEASURED OUTPUTS

n states
 m measured outputs
 $n - m$ estimated states

Given:
(assuming $m = 1$)

$$\dot{x} = Ax + Bu$$
$$y = [1 \ 0 \ \dots \ 0]x$$

Partition the state vector:



PARTITION THE STATE SPACE BASED ON STATE VECTOR PARTITION

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_e \end{bmatrix} = \begin{bmatrix} a_{11} & A_{1e} \\ A_{e1} & A_{ee} \end{bmatrix} \begin{bmatrix} x_1 \\ x_e \end{bmatrix} + \begin{bmatrix} b_1 \\ B_e \end{bmatrix} u$$

MEASURED STATES
 ↓
 ESTIMATED STATES

Express as plant in terms of states x_e :

$$\dot{x}_e = A_{ee}x_e + (A_{e1}x_1 + B_e u)$$

OUTPUT $\approx y$

$$(\dot{x}_1 - a_{11}x_1 - b_1 u) = A_{1e}x_e$$

l
Almost
AN INPUT
LIKE $\approx Bu$

$$\begin{aligned} \dot{x} &= Ax + Bu \\ y &\approx cx \end{aligned}$$

Reduced-order observer equation:

$$\dot{\hat{x}}_e = A_{ee}\hat{x}_e + (A_{e1}x_1 + B_e u) + L(\dot{x}_1 - a_{11}x_1 - b_1 u - A_{1e}\hat{x}_e)$$

y

$C\hat{x}$

$$\dot{\hat{x}}_e = (A_{ee} - LA_{1e})\hat{x}_e + A_{e1}x_1 + B_e u + L(\dot{x}_1 - a_{11}x_1 - b_1 u)$$

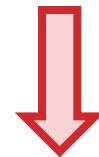
HOW DO WE DESIGN THE REDUCED-ORDER OBSERVER GAINS?

Define

$$e = x_e - \hat{x}_e$$

ESTIMATED STATES ONLY

$$\dot{e} = \dot{x}_e - \dot{\hat{x}}_e = (A_{ee} - L A_{1e}) e$$



$$L = \phi(A_{ee}) \begin{bmatrix} A_{1e} \\ A_{1e}A_{ee} \\ \vdots \\ A_{1e}A_{ee}^{n-2} \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 1 \end{bmatrix}$$

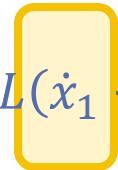
*WILL BE
OF SIZE
 $n-1 \times 1$*

ACKERMANS METHOD

This is a standard observer design method!

CONSIDER THE IMPLEMENTATION: THIS METHOD REQUIRES KNOWLEDGE OF THE OUTPUT DERIVATIVE

$$\dot{\hat{x}}_e = (A_{ee} - LA_{1e})\hat{x}_e + A_{e1}x_1 + B_e u + L(\dot{x}_1 - a_{11}x_1 - b_1 u)$$



Differentiation amplifies measurement noise.

We want to eliminate differentiation!

DEFINE A CHANGE OF VARIABLES:

$$\hat{x}_p = \hat{x}_e - Ly$$

L
P DOESN'T STAND
FOR ANYTHING
SIGNIFICANT

$$\Rightarrow \hat{x}_e = \hat{x}_p + Ly$$

$$\dot{\hat{x}}_e = \dot{\hat{x}}_p + L\dot{y} \rightarrow L\dot{x}_1$$

$$\dot{\hat{x}}_p + L\dot{y} = (A_{ee} - LA_{1e})(\hat{x}_p + Ly) + A_{e1}y + B_e u + L(y - a_{11}y - b_1u)$$

$$\dot{\hat{x}}_p = (A_{ee} - LA_{1e})\hat{x}_p + (A_{e1} - La_{11} + A_{ee}L - LA_{1e}L)y + (B_e - Lb_1)u$$

$\underbrace{\phantom{(A_{ee} - LA_{1e})}}$
 $A-LC$

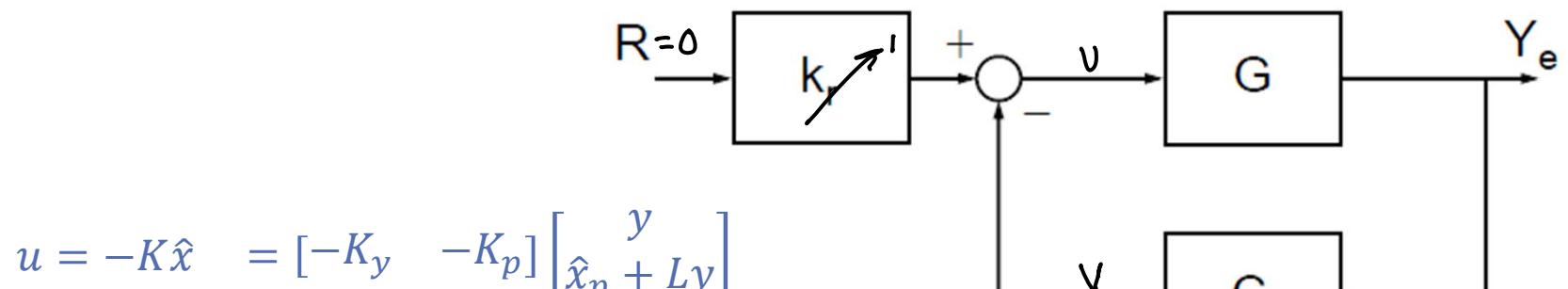
$\underbrace{\phantom{(A_{e1} - La_{11} + A_{ee}L - LA_{1e}L)y}}$
 L

$\underbrace{}$
 B

Recall that the full state vector is given by $(A - LC)\hat{x}_p + Ly + Bu$

$$\hat{x} = \begin{bmatrix} x_1 \\ \hat{x}_e \end{bmatrix} = \begin{bmatrix} y \\ \hat{x}_p + Ly \end{bmatrix} = \begin{bmatrix} 0 \\ I_{n-1} \end{bmatrix} \hat{x}_p + \begin{bmatrix} 1 \\ L \end{bmatrix} y$$

CAN WE MAKE A COMBINED REDUCED-ORDER OBSERVER + CONTROLLER SYSTEM?

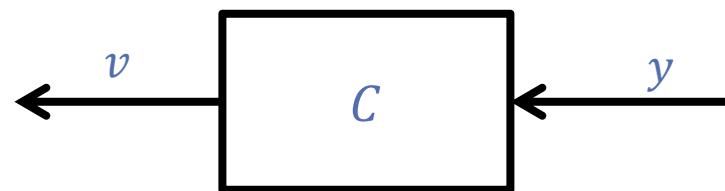


$$u = -K\hat{x} = [-K_y \quad -K_p] \begin{bmatrix} y \\ \hat{x}_p + Ly \end{bmatrix}$$

$$\dot{\hat{x}}_p = (A_{ee} - LA_{1e})\hat{x}_p + (A_{e1} - La_{11} + A_{ee}L - LA_{1e}L)y + (B_e - Lb_1)u$$

$$\begin{aligned} \dot{\hat{x}}_p &= (A_{ee} - LA_{1e})\hat{x}_p + (A_{e1} - La_{11} + A_{ee}L - LA_{1e}L)y \\ &\quad + (B_e - Lb_1)[-(K_y + K_pL)y - K_p\hat{x}_p] \end{aligned}$$

FEEDBACK CONTROLLER C IS A STATE-SPACE SYSTEM!



$$\begin{aligned}\dot{\hat{x}}_p &= (A_{ee} - LA_{1e} - (B_e - Lb_1)K_p)\hat{x}_p \\ &\quad + (A_{e1} - La_{11} + A_{ee}L - LA_{1e}L - (B_e - Lb_1)(K_y + K_pL))y \\ v &= K_p\hat{x}_p + (K_y + K_pL)y\end{aligned}$$

- CRIES

It may be more convenient to short-hand this controller as:

$$\begin{aligned}\dot{x}_C &= A_C x_C + B_C y \\ v &= C_C x_C + D_C y\end{aligned}$$

DEVELOP A REDUCED-ORDER OBSERVER FOR THE PREVIOUS EXAMPLE:

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ -1 & -1 \end{bmatrix}x + \begin{bmatrix} 0 \\ 1 \end{bmatrix}u$$

$$y = [1 \ 0]x$$

e denotes estimate

$$A = \begin{bmatrix} a_{11} & a_{12} \\ a_{21} & a_{22} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -1 & -1 \end{bmatrix} \quad B = \begin{bmatrix} b_1 \\ b_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$\hat{x}_2 = \hat{x}_p + 2y$$

$$\dot{\hat{x}}_p = (-1-L)\hat{x}_p + (-1-L \cdot 0 + (-1) \cdot (L - L^2(1)))y + (1 - L \cdot 0)u$$

$$\dot{e} = (-1-L)e$$

Poles @ -18

$$|SI - A + LC| = (s+18)$$

$$|SI - (-1+L)| = s+18$$

$$s+1+L = s+18$$

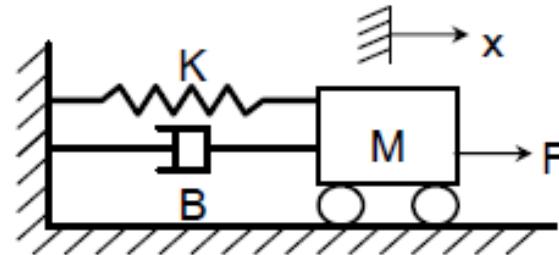
$$\therefore L = 17$$

SCALAR OBSR.

$$\begin{aligned} \dot{\hat{x}}_p &= -18\hat{x}_p - 307y + u \\ \dot{\hat{x}}_2 &= \hat{x}_p + 17y \end{aligned}$$

James A. Mynderse

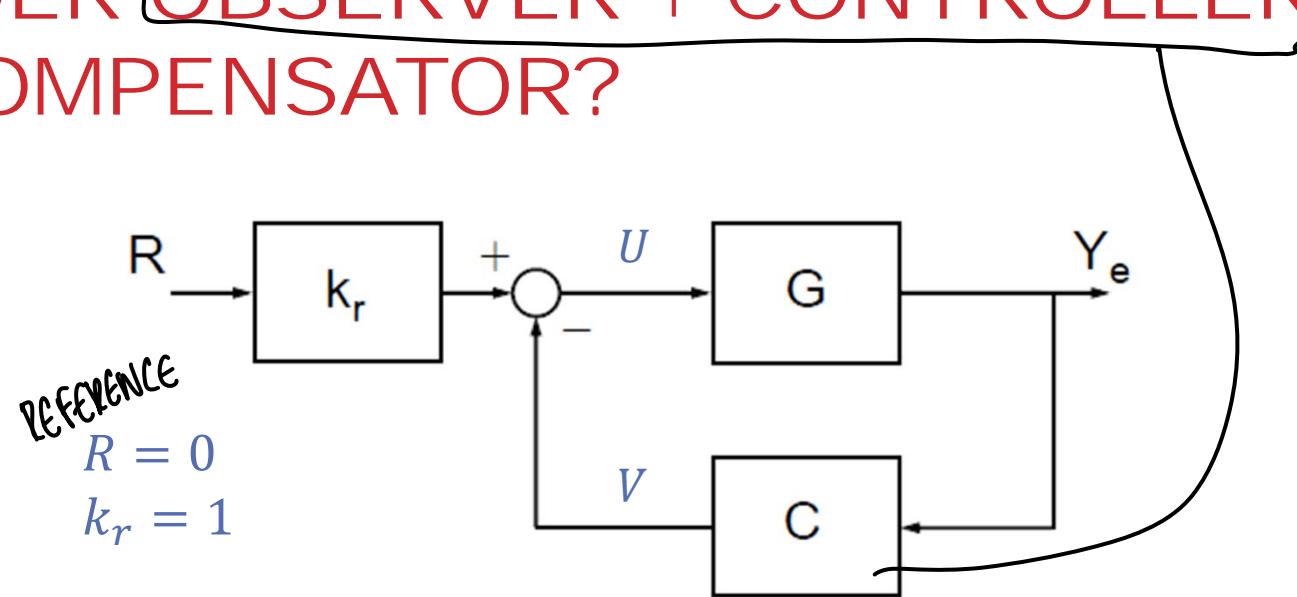
ESTIMATION OF
THE 2nd STATE



EME 5323 – State Observer + Feedback

CHOOSE THE REDUCED-ORDER OBSERVER POLE AT $s = -18$

CAN WE CONVERT THE REDUCED ORDER OBSERVER + CONTROLLER AS A COMPENSATOR?



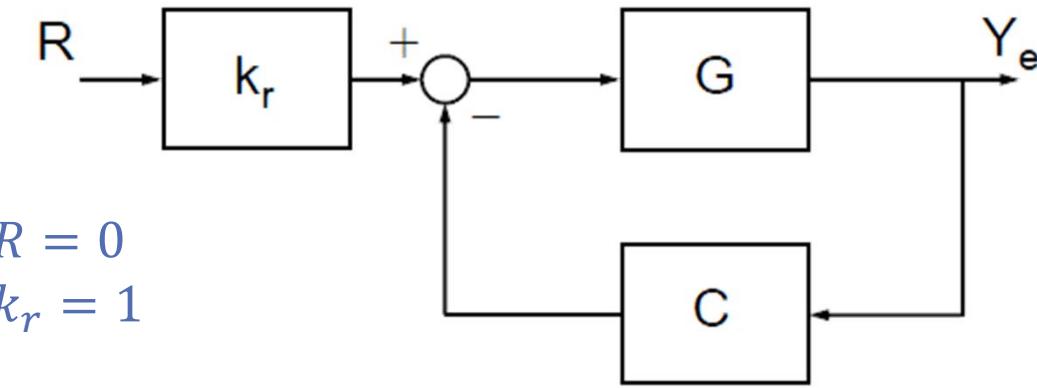
$$U(s) = -24Y(s) - 6\hat{X}_2(s) = -V(s)$$

STATE ESTIMATE

$$\begin{aligned}\hat{X}_2 &= \hat{X}_p(s) + 17Y(s) \\ \hat{X}_p(s) &= \frac{-307Y(s) + U(s)}{s + 18}\end{aligned}$$

$$\Rightarrow \hat{X}_2(s) = \frac{(17s - 1)Y(s) + U(s)}{s + 18}$$

MEASUREMENT



$$R = 0$$

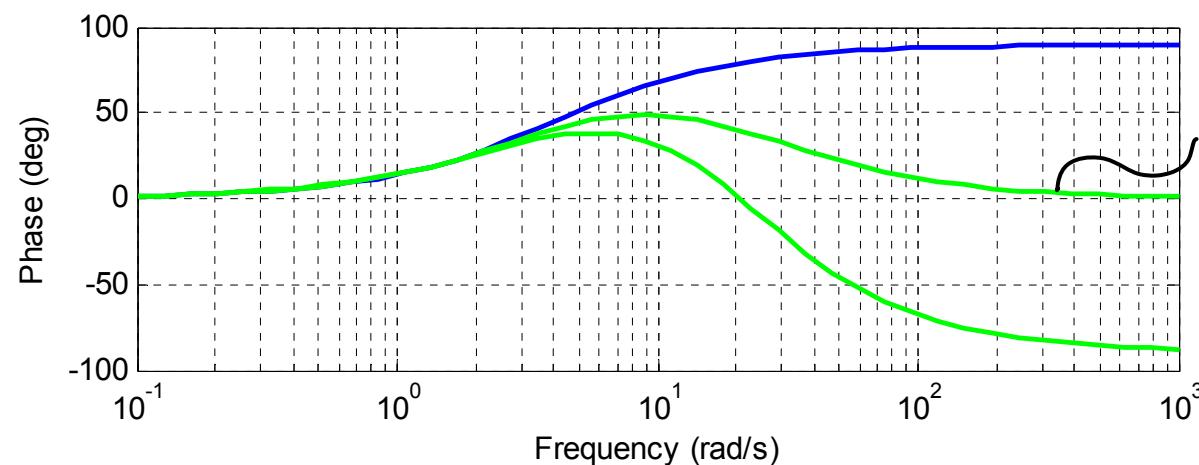
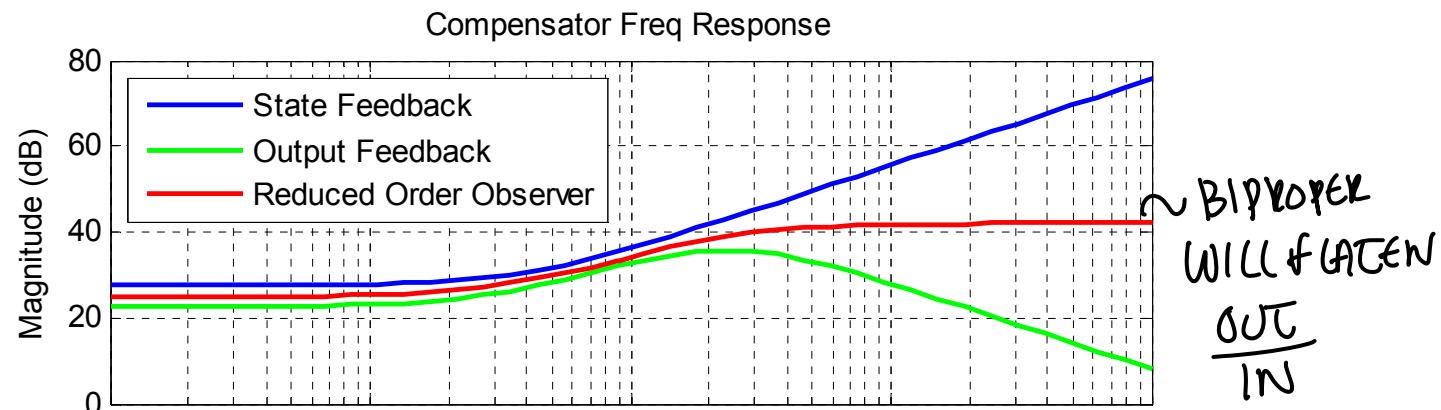
$$k_r = 1$$

$$C(s) = \frac{V(s)}{Y(s)} = \frac{6(2s + 7)}{s + 24}$$

*~ OUTPUT FEEDBACK
 COMPENSATOR
 IN TRANSFER
 FUNCTION FORM*

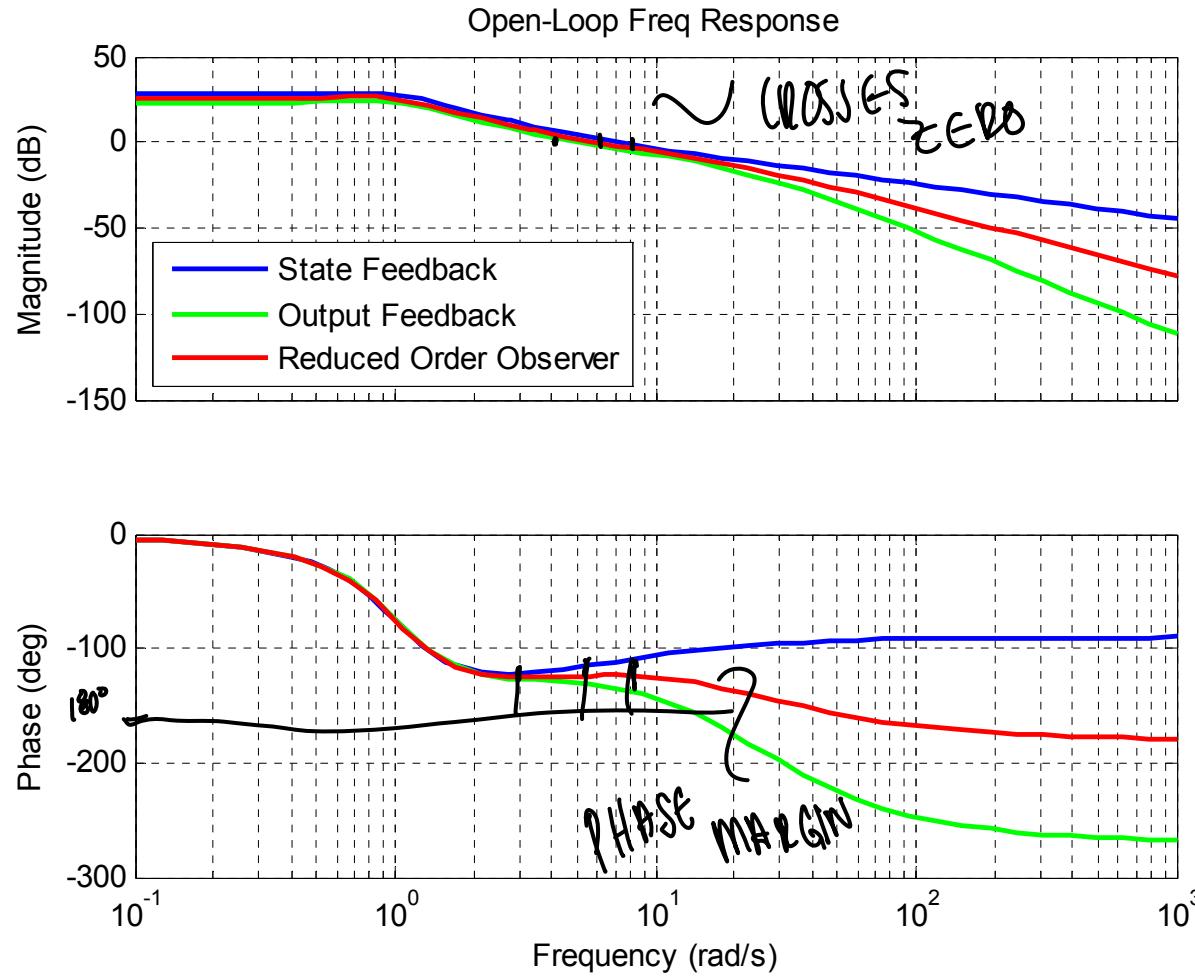
COMPENSATOR FREQUENCY RESPONSE LIES BETWEEN STATE AND OUTPUT FEEDBACK

FULL ORDER
- STRICTLY PROPER
REDUCED ORDER
- BI PROPER



SIMILAR BEHAVIOR FOR THE OPEN LOOP ($L = CG$)

(C PLANT
COMPENSATOR



COMING UP...

Case Study

Tracking and integral control