CONTROL SYSTEMS THEORY

Transfer Function of Physical Systems

CHAPTER 2 STB 35103

Objectives

- To learn about transfer functions.
- To develop mathematical models from schematics of physical system.

Overview

- Review on Laplace transform
- Learn about transfer function
 - Electric network
 - Translational mechanical system
 - Rotational mechanical system
- You will learn how to develop mathematical model.
- Present mathematical representation where the input, output and system are different and separate.
- Solving problems in group and individual

- A differential equation
- An equation that involves the derivatives of a function as well as the function itself. If partial derivatives are involved, the equation is called a partial differential equation;

If only ordinary derivatives are present, the equation is called an ordinary differential equation.

Differential equations. How to obtain?

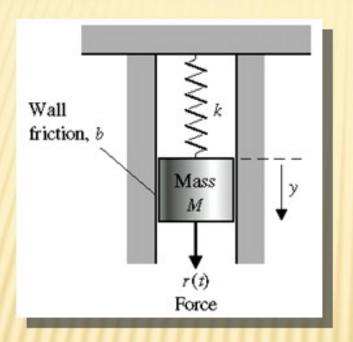
Physical law of the process — Differential equation

Examples:

Mechanical system (Newton's laws)

Electrical system (Kirchhoff's laws)

Example: Springer-mass-damper system



The time function of r(t) sometimes called forcing function

Assumption: Wall friction is a viscous force.

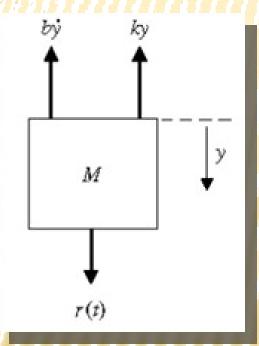
$$f(t) = -bv(t)$$



Linearly proportional to the velocity

Example: Springer-mass-damper system

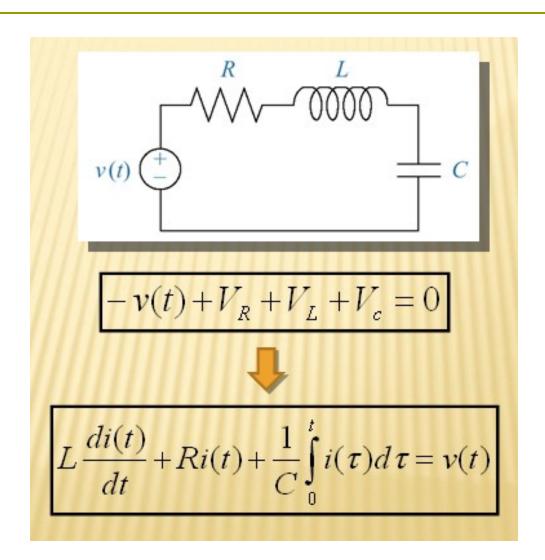
Newton's 2nd Law:



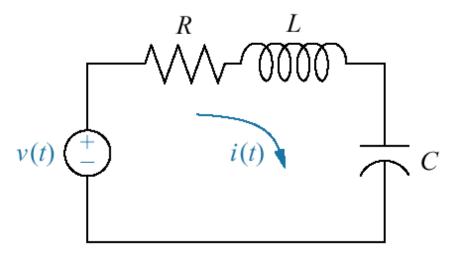
$$-bv(t) - ky(t) + r(t) = Ma(t)$$

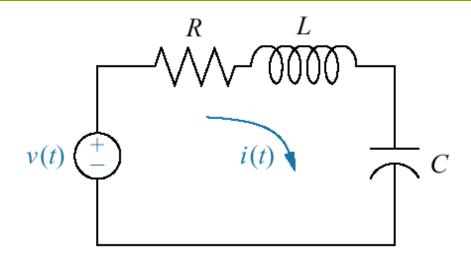
$$M \frac{d^2y(t)}{dt^2} + b \frac{dy(t)}{dt} + ky(t) = r(t)$$

RLC circuit~KVL~Ohm's law



- Control system can be represented using a mathematical model.
 - E.g. LED circuit
- Mathematical model is based on the schematic of physical systems.





Differential equation

$$L\frac{di(t)}{dt} + Ri(t) + \frac{1}{C} \int_{0}^{t} i(\tau)d\tau = v(t)$$

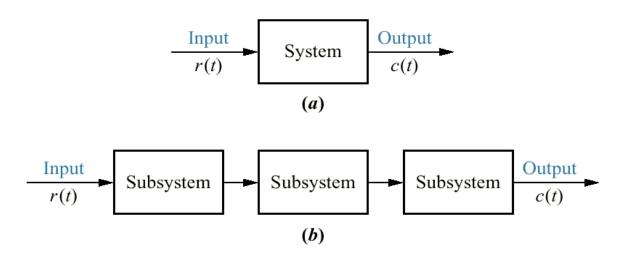
 Differential equation describes the relationship between the input and output of a system.

$$\frac{d^{n}c(t)}{dt^{n}} + a_{n-1}\frac{d^{n-1}c(t)}{dt^{n-1}} + \times \times + a_{0}c(t) = b_{m}\frac{d^{m}r(t)}{dt^{m}} + b_{m-1}\frac{d^{m-1}r(t)}{dt^{m-1}} + \times \times + b_{0}r(t)$$

c(t) output

r(*t*) input

It is easier if we can see the input and the output clearly such as in the figure below.



Note: The input, r(t), stands for reference input. The output, c(t), stands for controlled variable.

Type of input test

	Input	Function	Description	Sketch	Use
	Impulse	$\delta(t)$	$\delta(t) = \infty \text{ for } 0 - < t < 0 +$ $= 0 \text{ elsewhere}$ $\int_{0}^{0+} \delta(t) dt = 1$	f(t)	Transient response Modeling
Test waveforms used in control systems	Step	u(t)	u(t) = 1 for t > 0 $= 0 for t < 0$	$ \begin{array}{c c} & \delta(t) \\ \hline & f(t) \\ \hline \end{array} $	Transient response Steady-state error
	Ramp	tu(t)	$tu(t) = t \text{ for } t \ge 0$ = 0 elsewhere	f(t)	Steady-state error
	Parabola	$\frac{1}{2}t^2u(t)$	$\frac{1}{2}t^2u(t) = \frac{1}{2}t^2 \text{ for } t \ge 0$ = 0 elsewhere	f(t)	Steady-state error
	Sinusoid	sin ωt		f(t)	Transient response Modeling Steady-state error
				t t	

Laplace transform review

ltem no.	f(t)	F(s)	
1.	$\delta(t)$	1	
2.	u(t)	$\frac{1}{s}$	
3.	tu(t)	$\frac{1}{s^2}$	
4.	$t^n u(t)$	$\frac{n!}{s^{n+1}}$	
5.	$e^{-at}u(t)$	$\frac{1}{s+a}$	
6.	$\sin \omega t u(t)$	$\frac{\omega}{s^2 + \omega^2}$	
7.	$\cos \omega t u(t)$	$\frac{s}{s^2 + \omega^2}$	

Table 2.1 Laplace transform table

Laplace transform review

1.	$\mathcal{L}[f(t)] = F(t)$	$s) = \int_{0-}^{\infty} f(t)e^{-st}dt$	Definition
2.	$\mathcal{L}[kf(t)]$	= kF(s)	Linearity theorem
3.	$\mathcal{L}[f_1(t) + f_2(t$	$)] = F_1(s) + F_2(s)$	Linearity theorem
4.	$\mathcal{L}[e^{-at}f(t)]$	= F(s+a)	Frequency shift theo
5.	$\mathcal{L}[f(t-T)]$	$= e^{-sT}F(s)$	Time shift theorem
6.	$\mathcal{L}[f(at)]$	$=\frac{1}{a}F\left(\frac{s}{a}\right)$	Scaling theorem
7.	$\mathcal{L}\left[\frac{df}{dt}\right]$	= sF(s) - f(0-)	Differentiation theor
8.	$\mathcal{L}\left[\frac{d^2f}{dt^2}\right]$	$= s^2 F(s) - sf(0-) - \dot{f}(0-)$	Differentiation theor
9.	$\mathcal{L}\left[\frac{d^n f}{dt^n}\right]$	$= s^{n}F(s) - \sum_{k=1}^{n} s^{n-k}f^{k-1}(0 -$) Differentiation theor
10.	$\mathscr{L}\left[\int_{0-}^{t} f(\tau) d\tau\right]$		Integration theorem
11.	$f(\infty)$	$= \lim_{s \to 0} sF(s)$	Final value theorem
12.	<i>f</i> (0+)	$= \lim_{s \to \infty} sF(s)$	Initial value theorem

Name

Theorem

Item no.

Table 2.2 Laplace transform theorems

¹ For this theorem to yield correct finite results, all roots of the denominator of F(s) must have negative real parts and no more than one can be at the origin.

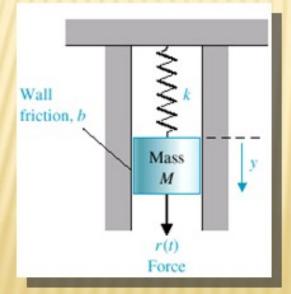
² For this theorem to be valid, f(t) must be continuous or have a step discontinuity at t = 0 (i.e., no impulses or their derivatives at t = 0).

The transfer function of a linear system is the ratio of the Laplace Transform of the output to the Laplace Transform of the input variable.

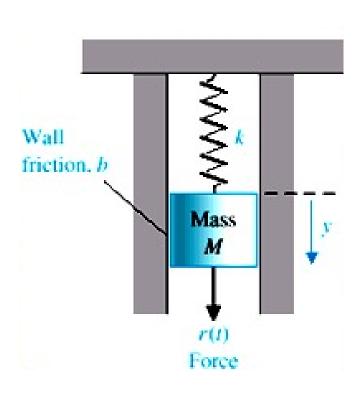
$$G(s) = \frac{Output(s)}{Input(s)}$$

Consider a spring-mass-damper dynamic equation with initial zero

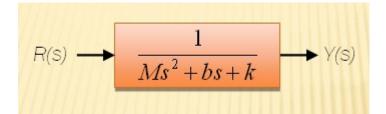
condition.



$$Ms^2Y(s) + bsY(s) + kY(s) = R(s)$$

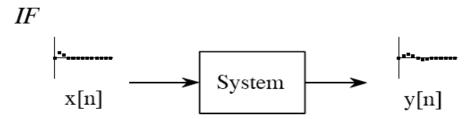


$$Ms^2Y(s) + bsY(s) + kY(s) = R(s)$$

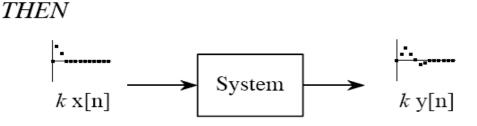


$$G(s) = \frac{Y(s)}{R(s)} = \frac{1}{Ms^2 + bs + k}$$

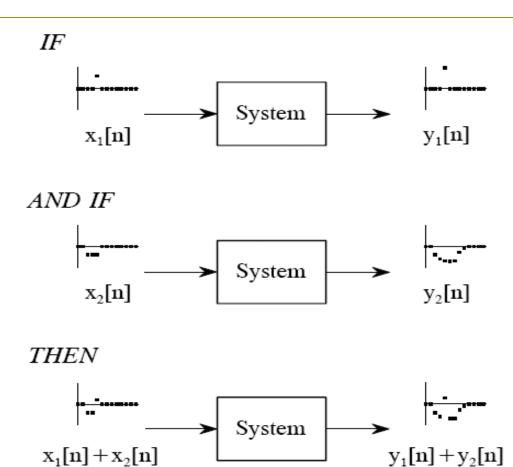
- Limited to linear system
 - What is linear system?
 - A system that has two mathematical properties: Homogeneity and additivity.



Homogeneity



Definition of homogeneity. A system is said to be *homogeneous* if an amplitude change in the input results in an identical amplitude change in the output. That is, if x[n] results in y[n], then kx[n] results in ky[n], for any signal, x[n], and any constant, k.



Additivity

Definition of additivity. A system is said to be *additive* if added signals pass through it without interacting. Formally, if $x_1[n]$ results in $y_1[n]$, and if $x_2[n]$ results in $y_2[n]$, then $x_1[n]+x_2[n]$ results in $y_1[n]+y_2[n]$.

Example:

Find the transfer function represented by

$$\frac{dc(t)}{dt} + 2c(t) = r(t)$$

Solution:

$$sC(s) + 2C(s) = R(s)$$

The transfer function G(s) is

$$G(s) = \frac{C(s)}{R(s)} = \frac{1}{s+2}$$

Example:

Find the transfer function represented by

$$\frac{dc(t)}{dt} + 0.5c(t) = 0.3r(t)$$

Solution:

$$sC(s) + 0.5C(s) = 0.3R(s)$$

The transfer function G(s) is

$$G(s) = \frac{C(s)}{R(s)} = \frac{0.3}{s + 0.5}$$

Example:

Given the transfer function for a system is

$$G(s) = \frac{1}{(s+2)}$$

The input for the system, r(t), is a unit step. r(t)=u(t), assuming zero initial conditions.

What is the output, c(t), of the system?

Solution:

Refer to Table 2.1. Laplace transform for a unit step input is 1/s. We know that

$$\frac{Output}{Input} = Transfer\ Function$$

SO

$$\frac{C(s)}{R(s)} = G(s)$$

$$C(s) = R(s)G(s)$$

$$= \frac{1}{s} \left[\frac{1}{s+2} \right]$$

$$= \frac{1}{s(s+2)}$$

Expanding the partial fraction, we get

$$C(s) = \frac{1/2}{s} - \frac{1/2}{s+2}$$

Taking the inverse Laplace transform (use Table 2.2) of each term,

$$c(t) = \frac{1}{2} - \frac{1}{2}e^{-2t}$$

- You are going to apply transfer function in three types of mathematical modeling.
 - Electric network
 - Translational mechanical system
 - Rotational mechanical system

- We are only going to apply transfer function to the mathematical modeling of electric circuits for passive networks (resistor, capacitor and inductor).
- We will look at a circuit and decide the input and the output.
- We will use Kirchhoff's laws as our guiding principles.
 - Mesh analysis
 - Nodal analysis

Component

Voltage-Current

Currentvoltage

Voltagecharge

Impedance Z(s)=

V(s)/I(s)

Y(s) =I(s)/V(s)

Admittance

$$v(t) = \frac{1}{C} \int_0^t i(\tau) d\tau \qquad i(t) = C \frac{dv(t)}{dt}$$

$$i(t) = C \frac{dv(t)}{dt}$$

$$v(t) = \frac{1}{C}q(t)$$

$$\frac{1}{Cs}$$

$$v(t) = Ri(t)$$

$$i(t) = \frac{1}{R}v(t)$$

$$v(t) = R \frac{dq(t)}{dt}$$

$$\frac{1}{R} = G$$

$$v(t) = L\frac{di(t)}{dt}$$

$$i(t) = \frac{1}{L} \int_0^t v(\tau) d\tau \qquad v(t) = L \frac{d^2 q(t)}{dt^2}$$

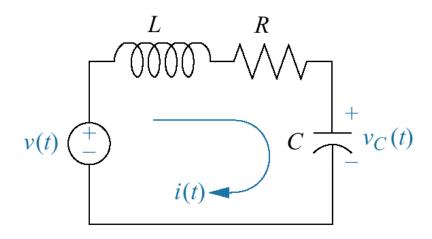
$$v(t) = L \frac{d^2 q(t)}{dt^2}$$

$$\frac{1}{I_{s}}$$

Note: The following set of symbols and units is used throughout this book: v(t) = V (volts), i(t) = A (amps), q(t) = Q (coulombs), C = F (farads), $R = \Omega$ (ohms), $G = \mathcal{U}$ (mhos), L = H (henries).

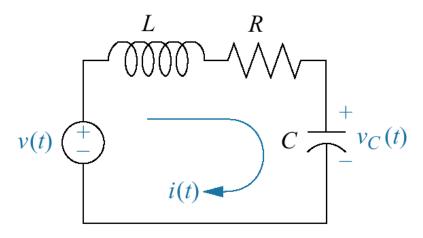
Simple circuit via mesh analysis

We can obtain transfer function using Kirchhoff's voltage law and summing voltages around loops or meshes. This method is called *loop* or *mesh analysis*.



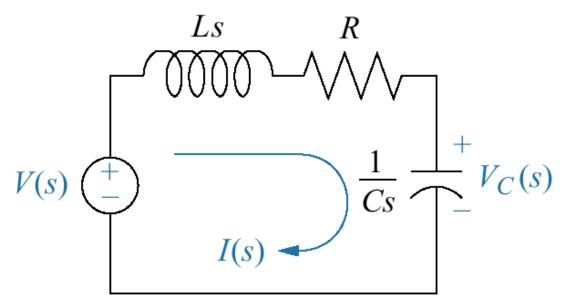
Example:

Find the transfer function relating the capacitor voltage, Vc(s), to the input voltage, V(s), in figure below



Solution:

Redraw the circuit using Laplace transform. Replace the component values with their impedance values.

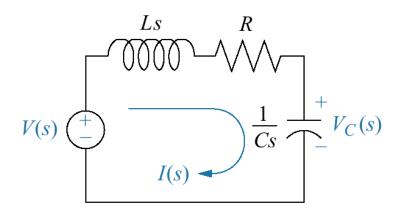


Determine the input and the output for the circuit. For this circuit,

Input is V(s)

Output is $V_c(s)$

Next, we write a mesh equation using the impedance as we would use resistor values in a purely resistive circuit.



We obtain

$$\left(Ls + R + \frac{1}{Cs}\right)I(s) = V(s) \tag{2.7}$$

If we look at Eq.2.7, we can only find the input, V(s). In order to calculate the transfer function we must have the output which is $V_c(s)$.

From the circuit,

$$V(s) \stackrel{Ls}{\leftarrow} V_C(s) \qquad V_C(s) = I(s) \frac{1}{Cs}$$

$$V_C(s) = I(s) \frac{1}{Cs}$$

$$I(s) = V_C(s)Cs$$
(2.8)

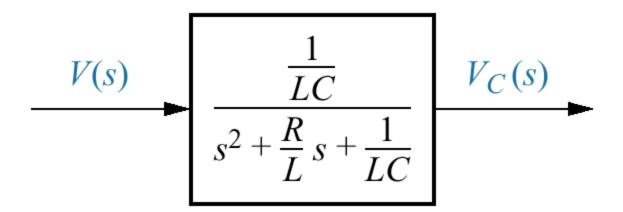
Substituting Eq.2.8 into Eq.2.7.

$$\left(Ls + R + \frac{1}{Cs} \right) V_C(s) Cs = V(s)$$

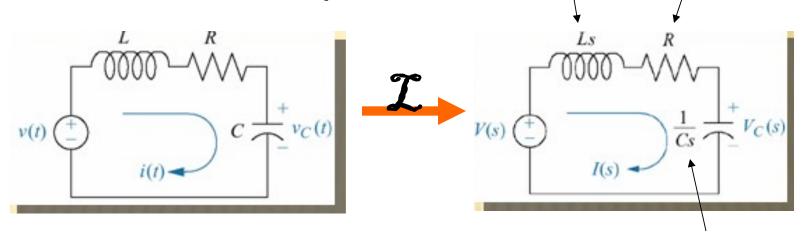
$$\frac{V_C(s)}{V(s)} = \frac{1}{Cs \left(Ls + R + \frac{1}{Cs} \right)} = \frac{1}{CLs^2 + RCs + 1}$$

$$= \frac{1/LC}{s^2 + \frac{R}{L}s + \frac{1}{LC}}$$

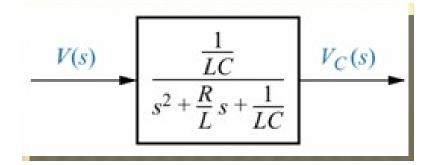
We can also present our answer in block diagram



Solution summary

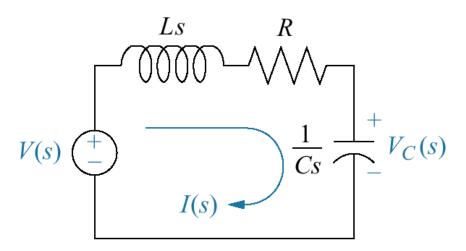


Using mesh analysis

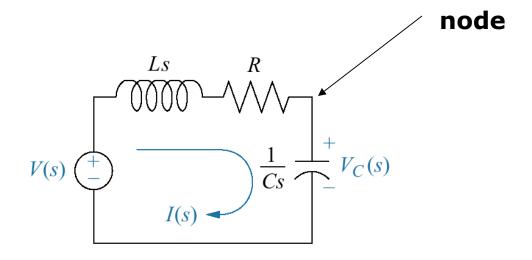


Simple circuit via nodal analysis

We obtain the transfer function using Kirchhoff's current law and summing current flowing from nodes. This method is called nodal analysis.



Example:



We will try to solve the previous example but this time using nodal analysis.

Solution:

We will look at the current flowing in and out of the node whose voltage is $V_c(s)$. We assume the current leaving the node is positive and current entering the node is negative.

$$\frac{V_{C}(s)}{1/Cs} = \frac{V(s) - V_{C}(s)}{R + Ls}$$

$$\frac{V_{C}(s)}{1/Cs} - \frac{V(s) - V_{C}(s)}{R + Ls} = 0$$

$$\vdots$$

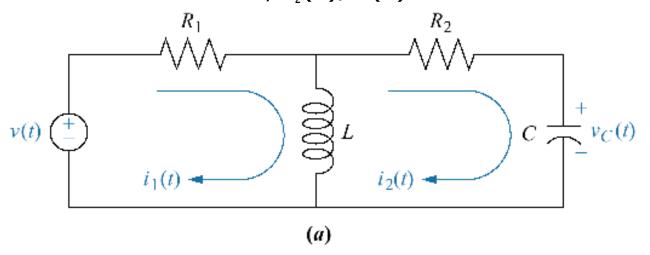
$$\frac{V_{C}(s)}{V(s)} = \frac{1/LC}{s^{2} + \frac{R}{L}s + \frac{1}{LC}}$$

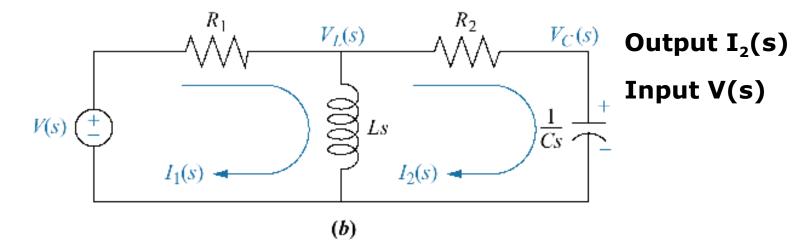
Complex circuit via mesh analysis

To solve a complex circuit we will perform the following steps.

- 1. Replace passive elements values with their impedances.
- Replace all sources and time variables with their Laplace transform.
- 3. Assume a transform current and a current direction in each mesh.
- 4. Write Kirchhoff's voltage law around each mesh.
- 5. Solve the simultaneous equations for the output.
- 6. Form the transfer function.

 \square Find the transfer function, $I_2(s)/V(s)$





$$\begin{bmatrix} \text{Sum of impedances around Mesh 1} \end{bmatrix} I_1(s) - \begin{bmatrix} \text{Sum of impedances common to the two meshes} \end{bmatrix} I_2(s) = \begin{bmatrix} \text{Sum of applied voltages around Mesh 1} \end{bmatrix}$$

$$\begin{bmatrix} \text{Sum of} \\ \text{impedances} \\ \text{common to the} \\ \text{two meshes} \end{bmatrix} I_1(s) - \begin{bmatrix} \text{Sum of} \\ \text{impedances} \\ \text{around Mesh 2} \end{bmatrix} I_2(s) = \begin{bmatrix} \text{Sum of applied} \\ \text{voltages around} \\ \text{Mesh 2} \end{bmatrix}$$

$$(R_1 + L_S)I_1(s) - L_SI_2(s) = V(s)$$
 (1)

$$-LsI_1(s) + \left(Ls + R_2 + \frac{1}{Cs}\right)I_2(s) = 0$$
 (2)

We need to solve both equation (1) and (2) to get the value of $I_2(s)$ and V(s).

You can use substitution method or Cramer's rule

It is easier if we use Cramer's rule

$$I_2(s) = \frac{\begin{vmatrix} (R_1 + Ls) & V(s) \\ -Ls & 0 \end{vmatrix}}{\Delta} = \frac{LsV(s)}{\Delta}$$

where

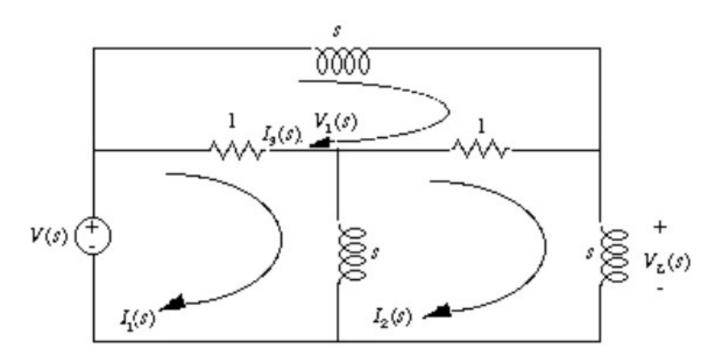
$$\Delta = \begin{vmatrix} (R_1 + Ls) & -Ls \\ -Ls & \left(Ls - R_2 + \frac{1}{Cs} \right) \end{vmatrix}$$

Forming the transfer function, G(s), yields

$$G(s) = \frac{I_2(s)}{V(s)} = \frac{Ls}{\Delta} = \frac{LCs^2}{(R_1 + R_2)LCs^2 + (R_1R_2C + L)s + R_1}$$

Example

Solve using mesh analysis and Nodal analysis



Mesh analysis method

Write the mesh equations :

$$(s+1)I_1(s) - sI_2(s) - I_3(s) = V(s)$$

$$-sI_1(s) + (2s+1)I_2(s) - I_3(s) = 0$$

$$-I_1(s) - I_2(s) + (s+2)I_3(s) = 0$$

 \square Solving the equations for $I_2(s)$:

$$I_2(s) = \frac{\begin{vmatrix} (s+1) & V(s) & -1 \\ -s & 0 & -1 \end{vmatrix}}{\begin{vmatrix} (s+1) & -s & -1 \\ -s & (2s+1) & -1 \\ -1 & -1 & (s+2) \end{vmatrix}} = \frac{(s^2 + 2s + 1)V(s)}{s(s^2 + 5s + 2)}$$

Mesh analysis method

But,
$$V_L(s) = sI_2(s)$$

Hence,

$$V_L(s) = \frac{(s^2 + 2s + 1)V(s)}{(s^2 + 5s + 2)}$$

Of

$$\frac{V_L(s)}{V(s)} = \frac{s^2 + 2s + 1}{s^2 + 5s + 2}$$

Nodal analysis method

Write the nodal equations:

$$(\frac{1}{s} + 2)V_1(s) - V_L(s) = V(s)$$
$$-V_1(s) + (\frac{2}{s} + 1)V_L(s) = \frac{1}{s}V(s)$$

Nodal analysis method

Solving for $V_{\tau}(s)$,

$$V_{L}(s) = \frac{\begin{vmatrix} (\frac{1}{s} + 2) & V(s) \\ -1 & \frac{1}{s}V(s) \\ \hline (\frac{1}{s} + 2) & -1 \\ -1 & (\frac{2}{s} + 1) \end{vmatrix}}{\begin{vmatrix} (\frac{1}{s} + 2) & -1 \\ -1 & (\frac{2}{s} + 1) \end{vmatrix}} = \frac{(s^{2} + 2s + 1)V(s)}{(s^{2} + 5s + 2)}$$

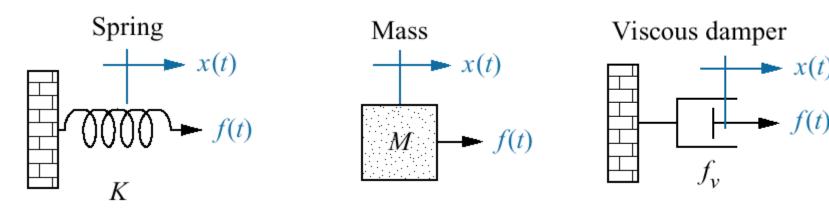
OI

$$\frac{V_L(s)}{V(s)} = \frac{s^2 + 2s + 1}{s^2 + 5s + 2}$$

Mechanical Systems

- Types of Mechanical Systems
 - Translational Systems
 - Rotational Systems

- We are going to model translational mechanical system by a transfer function.
- In electrical we have three passive elements, resistor, capacitor and inductor. In mechanical we have spring, mass and viscous damper.



- □ In electrical we have resistance, capacitance and inductance but in mechanical we have, spring constant (K), viscous damper (f_{ν}) and mass (M).
- We are going to find the transfer function for a mechanical system in term of force-displacement (i.e. forces are written in terms of displacement)

Translational Mechanical System

Mass

f(t) represents the applied force, x(t) represents the displacement, and M represents the mass. Then, in accordance with Newton's second law,

$$f(t) = Ma(t) = \frac{Mdv(t)}{dt} = \frac{Mdx(t)}{dt^2}$$

Where v(t) is velocity and a(t) is acceleration. It is assumed that the mass is rigid at the top connection point and that cannot move relative to the bottom connection point.

Translational Mechanical System

Damper

- Damper is the damping elements and damping is the friction existing in physical systems whenever mechanical system moves on sliding surface. The friction encountered is of many types, namely stiction, coulomb friction and viscous friction force
- In friction elements, the top connection point can move relative to the bottom connection point. Hence two displacement variables are required to describe the motion of these elements, where B is the damping coefficient

$$f(t) = B\left(\frac{dx_1(t)}{dt} - \frac{dx_2(t)}{dt}\right)$$

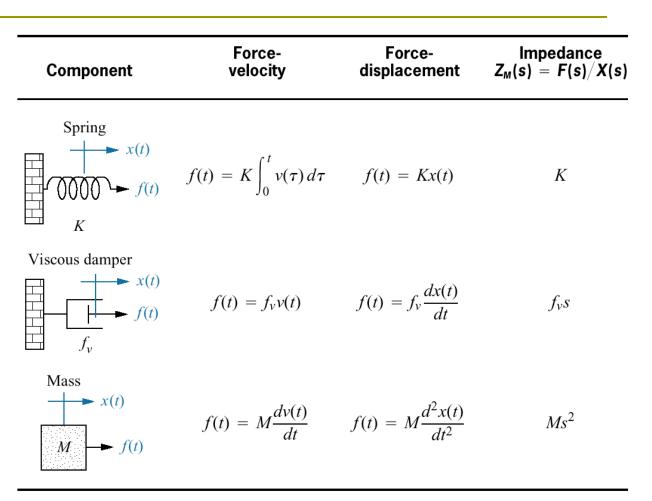
Translational Mechanical System

Spring

The final translational mechanical element is a spring. The ideal spring gives the elastic deformation of a body. The defining equation from Hooke's law, is given by

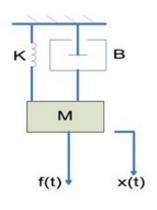
$$f(t) = \mathbf{K}(x_1(t) - x_2(t))$$

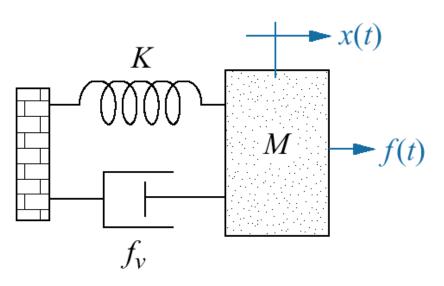
Force-velocity, force-displacement, and impedance translational relationships for springs, viscous dampers, and mass



Note: The following set of symbols and units is used throughout this book: f(t) = N (newtons), x(t) = m (meters), v(t) = m/s (meters/second), K = N/m (newtons/meter), $f_v = N-s/m$ (newton-seconds/meter), $f_v = N-s/m$ (newton-seconds/meter), $f_v = N-s/m$ (newton-seconds/meter).

Simple system



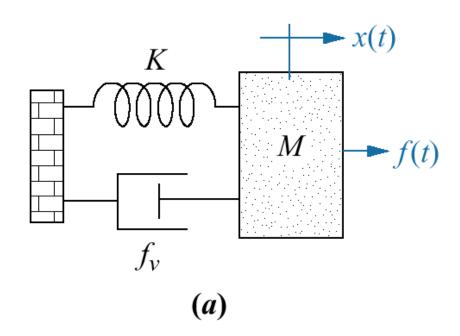


Assumption

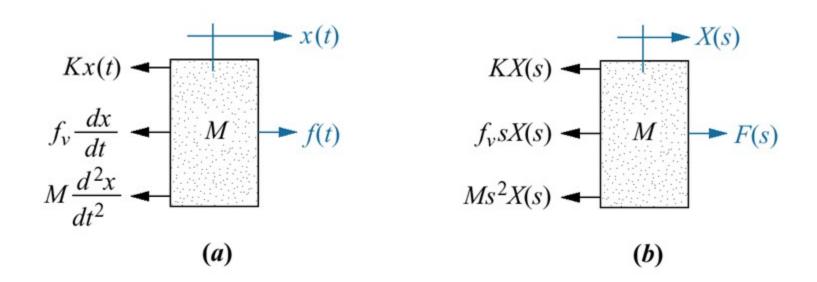
- (a)
- Movement to the left is assumed to be positive.
- positive <----- +ve</pre>

- Draw a free body diagram, placing on the body all forces that act on the body either in the direction of motion or opposite to it.
- Use Newton's law to form a differential equation of motion by summing the forces and setting the sum equal to zero.
- Assume zero initial conditions, we change the differential equation into Laplace form.

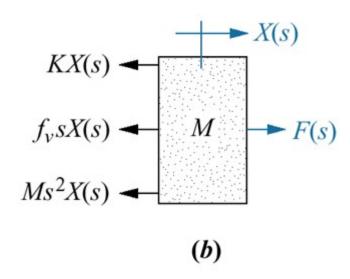
Example:



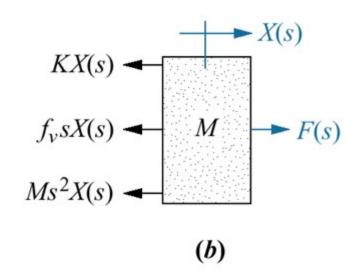
Find the transfer function, X(s)/F(s), for the system in Figure (a).



- Draw the free body diagram
- Place on the mass all forces felt by the mass.
- Assume the mass is travelling toward the right.
- Use Laplace transform.

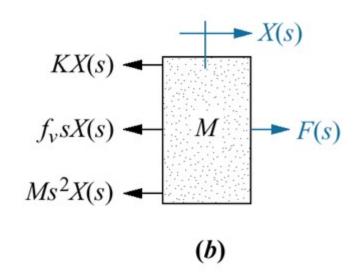


- How do we get the equation?
- Answer:
 - [sum of impedances]X(s) = [sum of applied forces]
 - Movement to the left is positive



The equation in Laplace form is

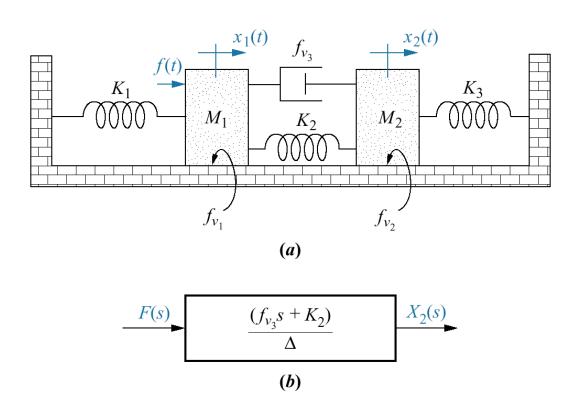
$$(Ms^2 + f_v s + K)X(s) = F(s)$$



Solving for the transfer function

$$G(s) = \frac{X(s)}{F(s)} = \frac{1}{Ms^2 + f_v s + K}$$

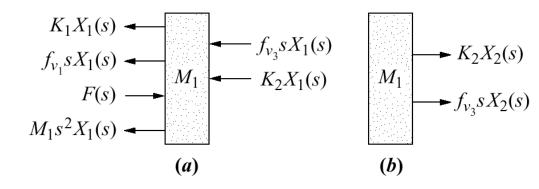
- Complex system
- \square Find transfer function, $X_2(s)/F(s)$

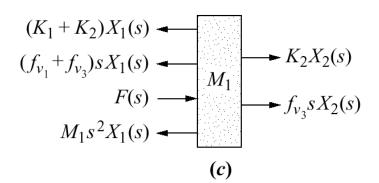


a. Forces on M₁ due **only** to motion of M₁

b. forces on M₁ due **only** to motion of M₂(sharing);

c. all forces on M₁

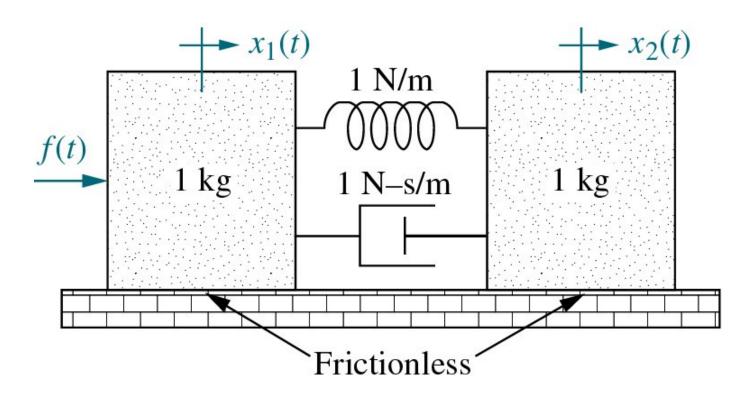




a. Forces on M₂ due **only** to motio $K_2X_2(s)$ of M_2 ; $f_{v_2}sX_2(s) \longrightarrow M_2$ $f_{v_3}sX_2(s) \longrightarrow M_2$ $f_{v_3}sX_1(s) \longrightarrow f_{v_3}sX_1(s) \longrightarrow f_{v_3}sX_1(s)$ **(b)** (a) only to motion of M₁(sharing); $(K_2 + K_3)X_2(s) \longrightarrow f_{v_3}sX_1(s)$ $(f_{v_2} + f_{v_3})sX_2(s) \longrightarrow M_2$ $M_2s^2X_2(s) \longrightarrow K_2X_1(s)$ **c.** all forces on M,

Example

□ Solve for $G(s) = X_2(s)/F(s)$



Writing the equations of motion,

$$(s^{2} + s + 1)X_{1}(s) - (s + 1)X_{2}(s) = F(s)$$
$$-(s + 1)X_{1}(s) + (s^{2} + s + 1)X_{2}(s) = 0$$

Solving for X2(s),

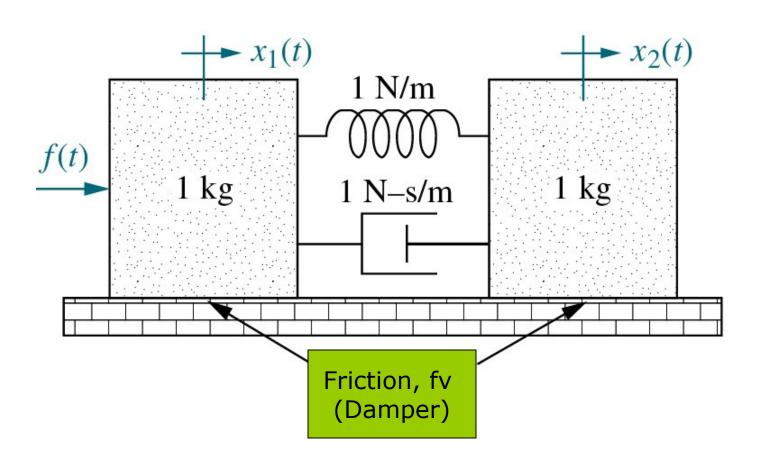
$$X_{2}(s) = \frac{\begin{bmatrix} (s^{2} + s + 1) & F(s) \\ -(s + 1) & 0 \end{bmatrix}}{\begin{bmatrix} (s^{2} + s + 1) & -(s + 1) \\ -(s + 1) & (s^{2} + s + 1) \end{bmatrix}} = \frac{(s + 1)F(s)}{s^{2}(s^{2} + 2s + 2)}$$

From which,

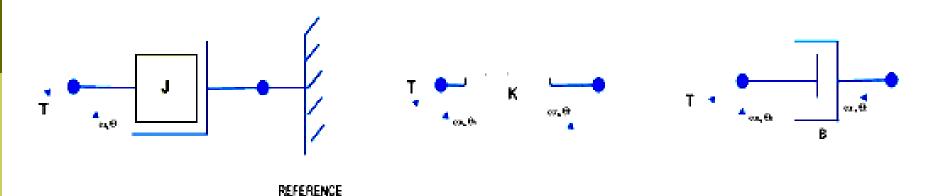
$$\frac{X_2(s)}{F(s)} = \frac{(s+1)}{s^2(s^2+2s+2)}.$$

Exercise

□ Solve for $G(s) = X_2(s)/F(s)$



Rotational Mechanical System



$$T = \frac{Jd^{2}\theta}{dt^{2}} = \frac{Jd\omega}{dt^{2}}$$

$$T = K(\theta - \theta)$$

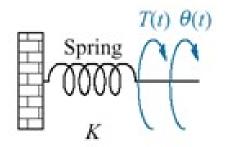
$$T = B\left(\frac{d\theta_{1}}{dt} - \frac{d\theta_{2}}{dt}\right) = B(\omega_{1} - \omega_{2})$$

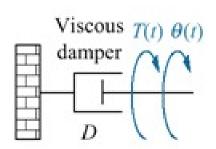
Rotational Mechanical System Transfer Functions

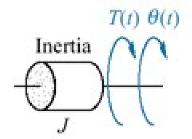
We are going to solve for rotational mechanical system using the same way as the translational mechanical systems except

- Torque replaces force
- Angular displacement replaces translational displacement

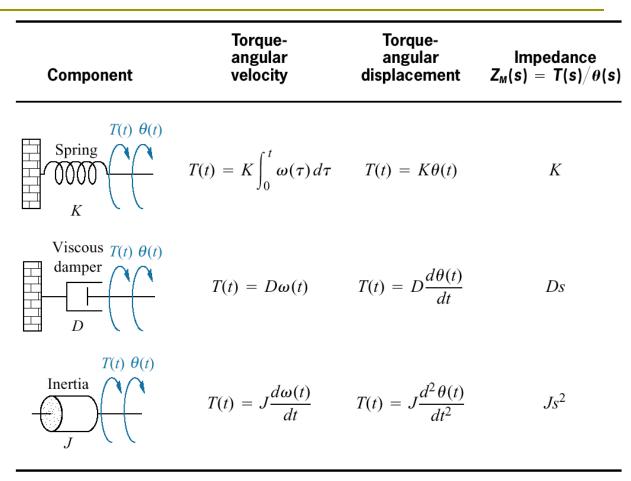
In translational mechanical system we have three elements; spring, damper and mass. In rotational mechanical system we have; spring, damper and inertia.





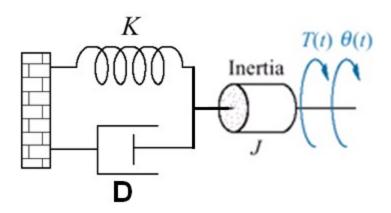


Torque-angular velocity, torque-angular displacement, and impedance rotational relationships for springs, viscous dampers, and inertia



Note: The following set of symbols and units is used throughout this book: T(t) = N-m (newton-meters), $\theta(t) = rad$ (radians), $\omega(t) = rad/s$ (radians/second), K = N-m/rad (newton-meters/radian), D = N-m-s/rad (newton-meters-seconds/radian), $J = kg-m^2$ (kilogram-meters² = newton-meters-seconds²/radian).

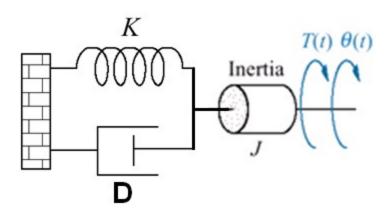
Simple system



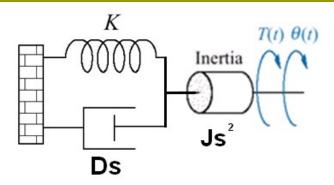
Assumption

Anti-clockwise movement is assumed to be positive

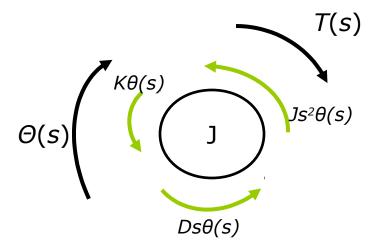
Example:



Find the transfer function, $\theta(s)/T(s)$



Draw the free body diagram



- We will use
 - \square [sum of impedances] $\theta(s)$ = [sum of applied torque]
- Based on the free body diagram, the equation of motion is

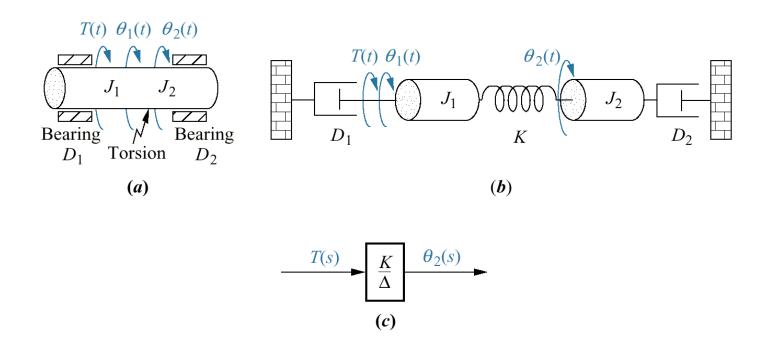
$$\left(Js^2 + Ds + K\right)\theta(s) = T(s)$$

we know transfer function, G(s), is $\frac{\theta(s)}{T(s)}$

$$\frac{\theta(s)}{T(s)} = \frac{1}{Js^2 + Ds + K}$$

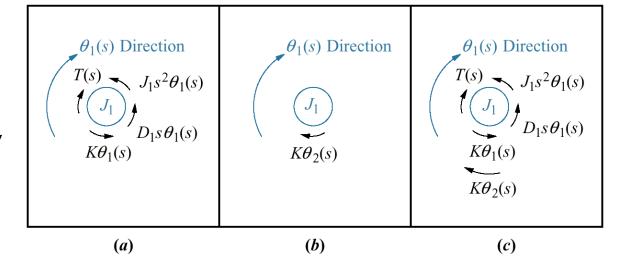
$$=\frac{\frac{1}{J}}{s^2 + \frac{D}{J}s + \frac{K}{J}}$$

- Complex system
- □ Find transfer function $\theta^2(s)/T(s)$

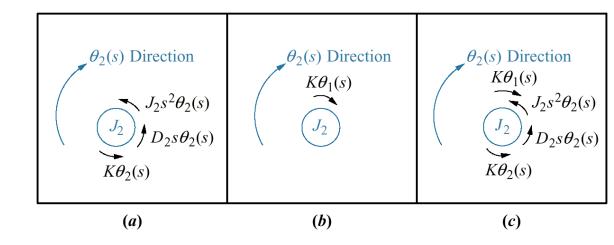


- **a.** Torques on J₁ due only to the motion of J₁ **b.** torques on J₁ due only to the
- **c.** final free-body diagram for J₁

motion of J₂



a. Torques on J₂ due only to the motion of J₂; **b.** torques on J₂ due only to the motion of J₁ **c.** final free-body diagram for J₂



The Laplace transform is defined as

$$\mathcal{D}[f(t)] = F(s) = \int_{0-}^{\infty} f(t)e^{-st}dt$$

Where $s = \sigma + j\omega$

Inverse Laplace transform, to get f(t) given F(s), is $\sigma_{+j\infty}$

$$\mathscr{Q}^{-1}[F(s)] = \frac{1}{2\pi j} \int_{\sigma - j\infty}^{\sigma + j\infty} F(s)e^{st}ds$$

ltem no.	f(t)	F(s)
1.	$\delta(t)$	1
2.	u(t)	$\frac{1}{s}$
3.	tu(t)	$\frac{1}{s^2}$
4.	$t^n u(t)$	$\frac{n!}{s^{n+1}}$
5.	$e^{-at}u(t)$	$\frac{1}{s+a}$
6.	$\sin \omega t u(t)$	$\frac{\omega}{s^2+\omega^2}$
7.	$\cos \omega t u(t)$	$\frac{s}{s^2+\omega^2}$

Table 2.1 Laplace transform table

ltem no.	Theorem		Name
1.	$\mathcal{L}[f(t)] = F(s)$	$= \int_{0-}^{\infty} f(t)e^{-st}dt$	Definition
2.	$\mathcal{L}[kf(t)]$	= kF(s)	Linearity theorem
3.	$\mathcal{L}[f_1(t) + f_2(t)]$	$= F_1(s) + F_2(s)$	Linearity theorem
4.	$\mathcal{L}[e^{-at}f(t)]$	= F(s+a)	Frequency shift theorem
5.	$\mathcal{L}[f(t-T)]$	$= e^{-sT}F(s)$	Time shift theorem
6.	$\mathcal{L}[f(at)]$	$=\frac{1}{a}F\left(\frac{s}{a}\right)$	Scaling theorem
7.	$\mathcal{L}\left[\frac{df}{dt}\right]$	= sF(s) - f(0-)	Differentiation theorem
8.	$\mathscr{L}\left[\frac{d^2f}{dt^2}\right]$	$= s^2 F(s) - s f(0-) - \dot{f}(0-)$	Differentiation theorer
9.	$\mathscr{L}\left[\frac{d^nf}{dt^n}\right]$	$= s^{n}F(s) - \sum_{k=1}^{n} s^{n-k}f^{k-1}(0-)$	Differentiation theorer
10.	$\mathscr{L}\left[\int_{0-}^{t} f(\tau) d\tau\right]$	$=\frac{F(s)}{s}$	Integration theorem
11.	$f(\infty)$	$= \lim_{s \to 0} sF(s)$	Final value theorem ¹
12.	<i>f</i> (0+)	$= \lim_{s \to \infty} sF(s)$	Initial value theorem ²

Table 2.2 Laplace transform theorems

¹ For this theorem to yield correct finite results, all roots of the denominator of F(s) must have negative real parts and no more than one can be at the origin.

² For this theorem to be valid, f(t) must be continuous or have a step discontinuity at t = 0 (i.e., no impulses or their derivatives at t = 0).

Example

Find the inverse Laplace transform of

$$F(s)=1/s$$

$$F(s)=1/s^2$$

$$F(s) = 1/(s+3)$$

Example:

Find the inverse Laplace transform of

$$F(s) = \frac{1}{s+3}$$

Answer

We use frequency shift theorem, item 4 in Table 2.2, and Laplace transform of f(t)=u(t), item 3 in Table 2.1

If inverse transform of $F(s)=1/s^2$ is tu(t), the inverse transform of $F(s+a)=1/(s+a)^2$ is $e^{-at}tu(t)$.

Hence, $f(t) = e^{-3t}tu(t)$

Example:

Find the inverse Laplace transform of

$$F(s) = 1/(s+3)^2$$

Answer

We use frequency shift theorem, item 4 in Table 2.2, and Laplace transform of f(t)=tu(t), item 3 in Table 2.1

If inverse transform of $F(s)=1/s^2$ is tu(t), the inverse transform of $F(s+a)=1/(s+a)^2$ is $e^{-at}tu(t)$.

Hence, $f(t) = e^{-3t}tu(t)$

Can you solve this question using Table 2.1 and Table 2.2?

$$F(s) = 1/(s+3)^3$$

Or this question

$$F(s) = \frac{s^3 + 2s^2 + 6s + 7}{s^2 + s + 5}$$

We can find the inverse Laplace transform for complicated function by changing the function to a sum of simpler term for which we know the Laplace transform of each term.

The result is called *partial-fraction expansion*

Let's take F(s)=N(s)/D(s)

In order for us to do the partial-fraction expansion we need to make sure the order of N(s) is smaller than or equal to the order of D(s).

$$N(s) \leq D(s)$$

What will we do if

$$N(s) >= D(s)$$
?

Answer.

We will divide N(s) by D(s) successively until the result has a remainder whose numerator is of order less that its denominator (polynomial division).

Polynomial long division

Example:

Divide $x^2+9x+14$ by x+7

Answer

$$x + 2$$

$$x + 7) x^{2} + 9x + 14$$

$$-x^{2} + 7x$$

$$2x + 14$$

$$-2x + 14$$

$$0$$

Divide
$$3x^3 - 5x^2 + 10x - 3$$
 by $3x + 1$

Answer:

$$x^2-2x+4+\frac{-7}{3x+1}$$

Can be divided into 3 cases.

Case 1:

Roots of denominator A(s) are real and distinct.

Case 2:

Roots of denominator A(s) are real and repeated.

Case 3:

Roots of denominator A(s) are complex conjugate.

Roots of the denominator of F(s) are real and distinct

Example of an F(s) with real and distinct roots in the denominator is

$$F(s) = \frac{2}{(s+1)(s+2)}$$

Solution

$$F(s) = \frac{2}{(s+1)(s+2)} = \frac{K_1}{(s+1)} + \frac{K_2}{(s+2)}$$
(2.1)

To find K¹, we multiply Eq. (2.1) by (s+1), which isolates K¹. Thus

$$K_{1} = \frac{2}{(s+2)} \Big|_{s \to -1} = 2$$

$$K_{2} = \frac{2}{(s+1)} \Big|_{s \to -2} = -2$$

Change value K₁ and K₂ gives

$$F(s) = \frac{2}{(s+1)(s+2)} = \frac{K_1}{(s+1)} + \frac{K_2}{(s+2)}$$
$$F(s) = \frac{2}{(s+1)(s+2)} = \frac{2}{(s+1)} - \frac{2}{(s+2)}$$

Using Table 2.1, the inverse Laplace transform is ??

$$f(s) = (2e^{-t} - 2e^{-2t})u(t)$$

Problem: Given the following differential equation, solve for y(t) if all initial condition are zero. Use Laplace transform.

$$\frac{d^2y}{dt^2} + 12\frac{dy}{dt} + 32y = 32u(t)$$

Roots of the denominator of F(s) are real and repeated

Example of an F(s) with real and repeated roots in the denominator is

$$F(s) = \frac{2}{(s+1)(s+2)^2}$$

Roots of $(s+2)^2$ in the denominator are repeated.

- We write the partial-fraction expansion as a sum of terms. Each factor of the denominator forms the denominator of each term.
- Each multiple root generates additional terms consisting of denominator factors of reduced multiplicity.

$$F(s) = \frac{2}{(s+1)(s+2)^2} = \frac{K_1}{(s+1)} + \frac{K_2}{(s+2)^2} + \frac{K_3}{(s+2)}$$
(2.2)

■ We can solve for K₁ using method in case 1 in Eq.2.2.

$$K_1 = \frac{2}{(s+2)^2} \Big|_{s \to -1} = 2$$

Next step is to isolate K2 by multiplying Eq. 2.2 by (s+2)²

$$\frac{2}{s+1} = (s+2)^2 \frac{K_1}{(s+1)} + K_2 + (s+2) K_3$$
 (2.3)

Solve for K2 using the same method as Case 1 in Eq.2.3.

$$K_2 = \frac{2}{(s+1)} \Big|_{s \to -2} = -2$$

□ To get the value of K₃ we need to differentiate Eq.2.3 with respect to s,

$$\frac{-2}{(s+1)^2} = \frac{(s+2)s}{(s+1)^2} K_1 + K_3$$
 (2.4)

Using the method in case 1. K₃ can be found if we let s approach -2. (-2 comes from the denominator of K₃)

$$K_3 = \frac{-2}{(s+1)^2} \Big|_{s \to -2} = -2$$

Changing K₁, K₂ and K₃ with their respective values

$$F(s) = \frac{2}{(s+1)(s+2)^2} = \frac{K_1}{(s+1)} + \frac{K_2}{(s+2)^2} + \frac{K_3}{(s+2)}$$
$$= \frac{2}{(s+1)} - \frac{2}{(s+2)^2} - \frac{2}{(s+2)}$$

Roots of the denominator of F(s) are complex or imaginary

Example of an F(s) with complex roots in the denominator is

$$F(s) = \frac{3}{s(s^2 + 2s + 5)}$$

We can expand the function in partial fractions as

$$F(s) = \frac{3}{s(s^2 + 2s + 5)} = \frac{3}{s(s+1+j2)(s+1-j2)}$$
$$= \frac{K_1}{s} + \frac{K_2}{s+1+j2} + \frac{K_3}{s+1-j1}$$

Solve for K1 using similar method in case 1.

$$K_1 = \frac{3}{\left(s^2 + 2s + 5\right)}\Big|_{s \to 0} = \frac{3}{5}$$

Solve for K2 using similar method in case 1.

$$K_2 = \frac{3}{s(s+1-j2)} \Big|_{s \to -1-j2} = -\frac{3}{20}(2+j1)$$

K3 is found to be the complex conjugate of K2

$$K_3 = \frac{3}{s(s+1+j2)} \Big|_{s \to -1+j2} = \frac{3}{20} (2+j1)$$

Changing K₁, K₂ and K₃ with their respective values

$$F(s) = \frac{3/5}{s} - \frac{3}{20} \left(\frac{2+j1}{s+1+j2} + \frac{2-j1}{s+1-j2} \frac{1}{j} \right)$$
 (2.5)

Inverse Laplace transform for Eq.2.5 is

$$f(t) = \frac{3}{5} - \frac{3}{20} \left[(2+j1) e^{-(1+j2)t} + (2-j1) e^{-(1-j2)t} \right]$$

$$= \frac{3}{5} - \frac{3}{20} e^{-t} \left[4 \left(\frac{e^{j2t} + e^{-j2t}}{2} \frac{1}{j} + 2 \left(\frac{e^{j2t} + e^{-j2t}}{2j} \frac{1}{j} \right) \right]$$
(2.6)

Using equation

$$\frac{e^{j\theta} + e^{-j\theta}}{2} = \cos\theta \quad \text{and} \quad \frac{e^{j\theta} - e^{-j\theta}}{2j} = \sin\theta$$

Into Eq.2.6

The final result is

$$f(t) = \frac{3}{5} - \frac{3}{5}e^{-t} \left(\cos 2t + \frac{1}{2}\sin 2t\right) = 0.6 - 0.671e^{-t}\cos(2t - \phi)$$

where ϕ = arctan $0.5 = 26.57^{\circ}$