trocador

September 8, 2021

```
[65]: from CoolProp.CoolProp import PropsSI
     import numpy as np
     import matplotlib.pyplot as plt
[66]: "########### Parâmetros fixos ##########"
     "############ chute inicial de temp. #########"
     T_f_{ent_1} = 28 + 273.15
     T_f_{sai_chut_1} = 700 + 273.15
     T_f_{sai_chut_5} = (T_f_{sai_chut_1} - T_f_{ent_1})/2 + T_f_{ent_1}
     T_q_{ent_2} = 665 + 273.15
     T_q_sai_chut_2 = 50 + 273.15
     T_q_sai_chut_4 = (T_q_sai_chut_2 - T_q_ent_2)/3 + T_q_ent_2
     T_q_sai_chut_5 = 2*(T_q_sai_chut_2 - T_q_ent_2)/3 + T_q_ent_2
     "########### chute inicial de comprimentos. ##########"
     L_chut_5 = 0.6
     L = 2
     L_chut_4 = L/3
     L_chut_5 = L*2/3
     "########### parametros do trocador ##########"
     L = 2
     D = 76e-3
     n_{tubos} = 19
     P_T = 15e-3
     B_{chi} = L / 8
     d_i = 10e-3
     d_e = d_i + 1e-3
```

```
A_i = np.pi * d_i
A_e = np.pi * d_e
P_e_m = np.pi * d_e
A_{i_t} = (np.pi * d_i**2)/4
A_c_t = np.pi * D**2/4
k_{aco} = 401
"########### parâmetros de entrada dos flúidos ###########"
m_dot_f_eval = np.linspace(0.00005,0.0002,num=100)
m_dot_q = 0.0016
P_q = 101325
P_f = 101325
parameters_2 = \{'q' : [],
                'D_P' : [],
                'epsilon' : [],
                'Re_D_i' : [],
                'h_i' : [],
                'T_q_sai' : [],
                'T_f_sai' : [],
                'L' : []}
parameters_4 = \{'q' : [],
                'D_P' : [],
                'epsilon' : [],
                'Re_D_i' : [],
                'h_i' : [],
                'T_q_sai' : [],
                'T_f_sai' : [],
                'L' : []}
parameters_5 = \{'q' : [],
                'D_P' : [],
                'epsilon' : [],
                'Re_D_i' : [],
                'h_i' : [],
                'T_q_sai' : [],
                'T_f_sai' : [],
                'L' : []}
parameters_6 = \{'q' : [],
                'D_P' : [],
                'epsilon' : [],
```

```
'Re_D_i' : [],
'h_i' : [],
'T_q_sai' : [],
'T_f_sai' : [],
'L' : []}
```

```
[67]: ############ Volumes de Controle II #############
      for m_dot_f in m_dot_f_eval:
          while True:
              A_c = np.pi * d_e * L
              #"parametros do fluido frio"
              T_{med_f_1} = (T_{f_ent_1} + T_{f_sai_chut_1}) / 2
              c_p_f_1 = PropsSI('CPMASS', 'P', P_f, 'T', T_med_f_1, 'water')
              I_f_ent_1 = PropsSI('H', 'P', P_f, 'T', T_f_ent_1, 'water')
              I_f_sai_1 = PropsSI('H','P',P_f,'T',T_med_f_1,'water')
              mu_f_1 = PropsSI('viscosity', 'P', P_f, 'T', T_med_f_1, 'water')
              Pr_f_1 = PropsSI('Prandtl','P',P_f,'T',T_med_f_1,'water')
              k_f_1 = PropsSI('conductivity', 'P', P_f, 'T', T_med_f_1, 'water')
              rho_f_1 = PropsSI('D', 'P', P_f, 'T', T_med_f_1, 'water')
              # parametros do fluido quente
              T_{med_q_2} = (T_{q_ent_2} + T_{q_sai_chut_2})/2
              c_p_q_2 = PropsSI('CPMASS', 'P', P_q, 'T', T_med_q_2, 'air')
              mu_q_2 = PropsSI('viscosity', 'P', P_q, 'T', T_med_q_2, 'air')
              Pr_q_2 = PropsSI('Prandtl','P',P_q,'T',T_med_q_2,'air')
              k_q_2 = PropsSI('conductivity', 'P', P_q, 'T', T_med_q_2, 'air')
              rho_q_2 = PropsSI('D', 'P', P_q, 'T', T_med_q_2, 'air')
              # Balanço de massas e energias
              q_2 = m_dot_f * (I_f_sai_1 - I_f_ent_1)
              # calculo da efetividade
              c_q_2 = m_dot_q * c_p_q_2
              c_{\min_2} = c_{q_2}
              c_r_2 = 0
```

```
q_{max_2} = c_{min_2} * (T_q_{ent_2} - T_f_{ent_1})
       epsilon_2 = q_2 / q_max_2
       T_q_{sai_2} = T_q_{ent_2} - q_{max_2} * epsilon_2/(m_dot_q * c_p_q_2)
       if epsilon_2 >= 1:
           print(m_dot_f)
           break
       NUT_5 = -np.log(1-epsilon_2)
       # Estudo do escoamento externo
       D_e = kern = \frac{4*((P_T**2 * 3**(1/2))/4 - (np.pi * d_e**2)/8)}{(np.pi *_U)}
\rightarrowd_e/2)
       C_{kern} = P_T - d_e
       A_c = (D * C_kern * B_chi)/P_T
       # Numero de Reynolds
       Re_c = (m_dot_q * D_e_kern)/(A_c * mu_q_2)
       # Coeficiente convectivo
       h_e = (0.36 * k_q_2)/D_e_kern * Re_c**0.55 * Pr_q_2**(1/3)
       # Fator de atrito
       if Re_c >= 2300:
           f = 0.0375
       else:
           f = Re_c/64
       # queda de pressão
       D_P_2 = n_{tubos*f*(m_dot_q/A_c)*(7)*D_e_kern/(2*rho_q_2*D)}
       #"########### parametros do loop ##########"
       error_q_2 = ((T_q_sai_chut_2 - T_q_sai_2)**2)**0.5
```

```
for m_dot_f in m_dot_f_eval:
    while True:
        A_i_4 = A_i *L_chut_4
        #"parametros do fluido frio"
        T_f_{ent_4} = T_f_{ent_1}
        T_f_{sai_4} = 373.15
        T_{med_f_4} = (T_{f_ent_4} + T_{f_sai_4}) / 2
        c_p_f_4 = PropsSI('CPMASS', 'P', P_f, 'T', T_med_f_4, 'water')
        I_f_ent_4 = PropsSI('H','P',P_f,'T',T_f_ent_1,'water')
        I_f_sai_4 = PropsSI('H', 'P', P_f, 'T', T_f_sai_4, 'water')
        mu_f_4 = PropsSI('viscosity', 'P', P_f, 'T', T_med_f_4, 'water')
        Pr_f_4 = PropsSI('Prandtl', 'P', P_f, 'T', T_med_f_4, 'water')
        k_f_4 = PropsSI('conductivity', 'P', P_f, 'T', T_med_f_4, 'water')
        rho_f_4 = PropsSI('D','P',P_f,'T',T_med_f_4,'water')
        #"parametros do fluido quente"
        T_q_{ent_4} = T_q_{ent_2}
        T_{med_q_4} = (T_{q_ent_4} + T_{q_sai_chut_4})/2
        c_p_q_4 = PropsSI('CPMASS', 'P', P_q, 'T', T_med_q_4, 'air')
        #"Balanço de massas e energias"
```

```
T_q_{sai_4} = T_q_{ent_4} - m_{dot_f} * c_p_f_4*(T_f_{sai_4}-T_f_{ent_4})/
\rightarrow (m_dot_q * c_p_q_4)
       q_4 = m_{dot_f} * c_p_{f_4} * (T_f_{sai_4} - T_f_{ent_4})
       #"calculo da efetividade"
       c_q_4 = m_dot_q * c_p_q_4
       c_f_4 = m_dot_f * c_p_f_4
       if c_q_4<c_f_4:
            c_{\min_4} = c_{q_4}
            c_{max_4} = c_f_4
       else:
           c_{\min_4} = c_f_4
           c_{max_4} = c_{q_4}
       c_r_4 = c_{min_4/c_{max_4}}
       q_{max_4} = c_{min_4} * (T_q_{ent_4} - T_f_{ent_4})
       epsilon_4 = q_4 / q_max_4
       if epsilon_4 >= 1:
           print(m_dot_f)
           break
       E_c_4 = ((2/epsilon_4) - (1 + c_r_4))/((1 + c_r_4**2)**(1/2))
       NUT_4 = -(1 + c_r_4**2)**(-1/2) * np.log((E_c_4 - 1)/(E_c_4 + 1))
       *NUT_4 = (c_r_4-1)**(-1) * np.log((epsilon_4 - 1)/(epsilon_4*c_r_4 - 1))
       "Numero de Reynolds"
       Re_D_i_4 = (m_dot_f*d_i)/(A_i_t*mu_f_4)
       "numero de Nusselt"
       if Re_D_i_4 >= 2300:
           f_f = 0.0375
            Nu_D_4 = ((f_f/8)*(Re_D_i_4-1e3)*Pr_f_4)/(1 + 12.7 * (f_f/8)**0.5 *_U
\hookrightarrow (Pr f 4**(2/3)-1))
```

```
elif Re_D_i_4 \Rightarrow 10000 and Pr_f_4 \Rightarrow 0.6 and Pr_f_4 \Leftarrow 160:
    Nu_D_4 = 0.023 * Re_D_i_4**(4/5) * Pr_f_4 * 0.4
else:
    Nu_D_4 = 4.36
"coeficiente de convecção interna"
h_i_4 = (Nu_D_4*k_f_4)/(d_i)
"coeficientes globais"
R_d_{i_4} = 0.0002
R_d_e = 0.0009
U_{i_4} = (1/h_{i_4} + R_{d_{i_4}} + d_{i_4} * n_{tubos} * (np.log(d_e/d_i))/
        (2 * k_aco) + (d_i * R_d_e)/d_e + d_i * 1/(d_e * h_e))**(-1)
"encontrando comprimento do volume de controle"
L_4 = (NUT_4 * c_f_4)/(U_i_4 * A_i_4)
"Queda de pressão"
f_4 = Re_D_i_4/64
vm_4_f = m_dot_f/(rho_f_4*A_i_t)
D_P_4 = rho_f_4 * vm_4_f**2 * L_4 * n_tubos/(2*d_i)
"########### parametros do loop ###########"
error_q_4 = ((T_q_sai_chut_4 - T_q_sai_4)**2)**0.5
T_q_{sai_chut_4} = (T_q_{sai_chut_4} + T_q_{sai_4})/2
if error_q_4 < (T_q_sai_4 * 0.01):</pre>
    break
```

```
parameters_4['q'].append(q_4)
parameters_4['D_P'].append(D_P_4)
parameters_4['epsilon'].append(epsilon_4)
parameters_4['Re_D_i'].append(Re_D_i_4)
parameters_4['h_i'].append(h_i_4)
parameters_4['T_q_sai'].append(T_q_sai_4)
parameters_4['T_f_sai'].append(T_f_sai_4)
parameters_4['L'].append(L_4)
```

```
[69]: ############ Volume de Controle V ##############
      for m_dot_f,T_q_sai_4,h_i_4 in_
      →zip(m_dot_f_eval,parameters_4['T_q_sai'],parameters_4['h_i']):
          while True:
              A_i_5 = A_i * L_chut_5
              A_e_5 = A_e * L_chut_5
              "parametros do fluido frio"
              T_f_{ent_5} = 373.15
              T_f_{sai_5} = 373.15
              I_f_ent_5 = PropsSI('H','Q',0,'T',T_f_ent_5,'water')
              I_f_sai_5 = PropsSI('H','Q',1,'T',T_f_sai_5,'water')
              mu_f_a = PropsSI('viscosity','Q',0,'T',T_f_ent_5,'water')
              Pr_f_a = PropsSI('Prandtl','Q',0,'T',T_f_ent_5,'water')
              k_f_a = PropsSI('conductivity','Q',0,'T',T_f_ent_5,'water')
              c_p_f_a = PropsSI('CPMASS','Q',0,'T',T_f_ent_5,'water')
              I_f_a = PropsSI('H', 'Q', 0, 'T', T_f_ent_5, 'water')
              rho_f_a = PropsSI('D','Q',0,'T',T_f_ent_5,'water')
              mu_f_v = PropsSI('viscosity','Q',1,'T',T_f_ent_5,'water')
              Pr_f_v = PropsSI('Prandtl','Q',1,'T',T_f_ent_5,'water')
              k_f_v = PropsSI('conductivity','Q',1,'T',T_f_ent_5,'water')
              c_p_f_v = PropsSI('CPMASS','Q',1,'T',T_f_ent_5,'water')
              I_f_v = PropsSI('H','Q',1,'T',T_f_ent_5,'water')
              rho_f_v = PropsSI('D','Q',0,'T',T_f_ent_5,'water')
              "parametros do fluido quente"
              T q ent 5 = T q sai 4
              T_{med_q_5} = (T_{q_ent_5} + T_{q_sai_chut_5})/2
              c_p_q_5 = PropsSI('CPMASS', 'P', P_q, 'T', T_med_q_5, 'air')
```

```
"parametros adicionais"
mu_f_s_v = PropsSI('viscosity','Q',0,'T',647.096,'water')
"Balanço de massas e energias"
q_5 = m_dot_f * (I_f_sai_5 - I_f_ent_5)
"calculo da efetividade"
c_q_5 = m_dot_q * c_p_q_5
c_{\min}5 = c_{q}5
c_r_5 = 0
q_{max_5} = c_{min_5} * (T_q_{ent_5} - T_f_{ent_5})
epsilon_5 = q_5 / q_max_5
T_q_{sai_5} = T_q_{ent_5} - q_{max_5} * epsilon_5/(m_{dot_q} * c_p_q_5)
if epsilon_5 >= 1:
    print(m_dot_f)
    break
NUT_5 = -np.log(1-epsilon_5)
"Numero de Reynolds"
Re_D_i_5_v = (m_dot_f*d_i)/(A_i_t*mu_f_v)
Re_D_i_5_a = (m_dot_f*d_i)/(A_i_t*mu_f_a)
"Calculo do coeficiente convectivo"
q_s_flux = q_5/(A_i_5)
x = np.linspace(0,L_chut_5,num=100)
X_{eval} = (q_s_{flux} * np.pi * d_i * x)/(m_dot_f*(I_f_v-I_f_a))
```

```
G_s_f = 1
       Fr = ((m_dot_f/A_i_t)**2 / rho_f_a)/(10*d_i)
       if Fr >= 0.04:
           f_Fr =1
       else:
           f_Fr = 2.63*Fr**0.3
       "coeficiente de convecção interna"
       h_mf = h_i_4
       aux = []
       for X in X_eval:
            if (X > 0) and (X \le 0.8):
                aux.append((1.136*(rho_f_a/rho_f_v)**0.45 * X**0.72 * (1-X)**0.
→08 * f Fr +
                667*(q_s_flux/((m_dot_f/A_i_t)*(I_f_v-I_f_a)))**0.7*(1-X)**0.
\rightarrow8 * G_s_f))
            else:
                r = 0.6683*((rho_f_a/rho_f_v)**0.1) * X**0.16 * (1-X)**0.64 *_
\hookrightarrow f_Fr + 1058*((q_s_flux/((m_dot_f/A_i_t)*(I_f_v-I_f_a)))**0.7) * (1-X)**0.8 *__
\hookrightarrowG_s_f
                if np.isnan(r) == True:
                    aux.append(0)
                else:
                    aux.append(r)
       h_i_5 = np.mean(aux) * h_mf
       "coeficientes globais"
       R_d_{i_5} = 0.0002
```

```
U_e_5 = (d_e/(d_i*h_i_5) + R_d_i_5 * d_e/d_i + d_e * n_tubos * (np.)
\rightarrow \log(d_e/d_i))/(2 * k_aco) +
               R_d_e + 1/h_e)**(-1)
       "encontrando comprimento do volume de controle"
       L_5 = (NUT_5 * c_q_5)/(U_e_5 * A_e_5)
       "Queda de pressão"
       f_5 = Re_D_i_5_v/64
       vm_5_f = m_dot_f/(rho_f_v*A_i_t)
       D_P_5 = rho_f_v * vm_5_f **2 * L_5 * n_tubos/(2*d_i)
       "########### parametros do loop ##########"
       error L 5 = ((L \text{ chut } 5 - L 5)**2)**0.5
       L_chut_5 = (L_chut_5 + L_5)/2
       if error_L_5 < (L_5 * 0.01):</pre>
           break
  parameters_5['q'].append(q_5)
  parameters_5['D_P'].append(D_P_5)
  parameters_5['epsilon'].append(epsilon_5)
  parameters_5['Re_D_i'].append(Re_D_i_5_v)
  parameters_5['h_i'].append(h_i_5)
  parameters_5['T_q_sai'].append(T_q_sai_5)
  parameters_5['T_f_sai'].append(T_f_sai_5)
  parameters_5['L'].append(L_5)
```

/home/thiago/yes/envs/tf/lib/python3.7/site-packages/ipykernel_launcher.py:111: RuntimeWarning: invalid value encountered in double_scalars

```
L_6 = L - (L_4 + L_5)
A_i_6 = A_i * L_6
"parametros do fluido frio"
T_f_{ent_6} = 373.15
T_{med_f_6} = (T_{f_ent_6} + T_{f_sai_chut_1}) / 2
c_p_f_6 = PropsSI('CPMASS', 'P', P_f, 'T', T_med_f_6, 'water')
I_f_ent_6 = PropsSI('H', 'P', P_f, 'T', T_f_ent_6, 'water')
I_f_sai_6 = PropsSI('H','P',P_f,'T',T_f_sai_chut_1,'water')
mu_f_6 = PropsSI('viscosity', 'P', P_f, 'T', T_med_f_6, 'water')
Pr_f_6 = PropsSI('Prandtl','P',P_f,'T',T_med_f_6,'water')
k_f_6 = PropsSI('conductivity', 'P', P_f, 'T', T_med_f_6, 'water')
"parametros do fluido quente"
T_q_{ent_6} = T_q_{sai_5}
T_{med_q_6} = (T_{q_ent_6} + T_{q_sai_2})/2
c_p_q_6 = PropsSI('CPMASS', 'P', P_q, 'T', T_med_q_6, 'air')
"parametros adicionais"
mu_f_s_6 = PropsSI('viscosity', 'P', P_f, 'T', T_med_f_6, 'water')
"Balanço de massas e energias"
q_6 = m_{dot_q} * c_p_{q_6} * (T_q_{ent_6} - T_q_{sai_2})
"calculo da efetividade"
c_q_6 = m_dot_q * c_p_q_6
c_f_6 = m_dot_f * c_p_f_6
if (c_q_6<c_f_6):
    c_{\min}6 = c_{q}6
    c_{max_6} = c_f_6
else:
    c_{\min_6} = c_f_6
    c_{max_6} = c_{q_6}
```

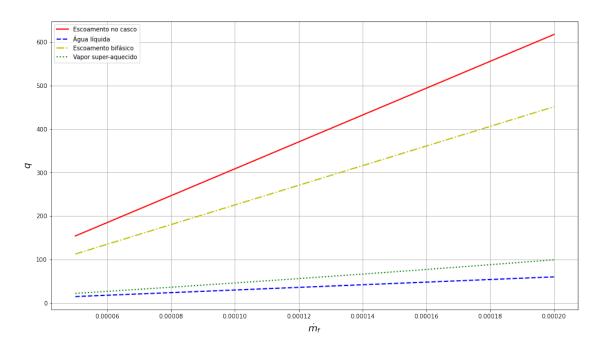
```
c_r_6 = c_min_6/c_max_6
       q_max_6 = c_min_6 * (T_q_ent_6 - T_f_ent_6)
       epsilon_6 = q_6 / q_max_6
       T_f_{sai_6} = T_f_{ent_6} + q_{max_6} * epsilon_6/(m_dot_f * c_p_f_6)
       \#NUT_6 = (c_r_6 - 1)*np.log((epsilon_6-1)/(epsilon_6*c_r_4-1))
       e_c_6 = (2/epsilon_6 - 1 + c_r_6)/(1 + c_r_6**2)**(1/2)
       NUT_6 = -(1 + c_r_6**2)**(-1/2) * np.log((e_c_6 - 1)/(e_c_6 + 1))
       "Numero de Reynolds"
       Re_D_i_6 = (m_dot_f*d_i)/(A_i_t*mu_f_6)
       "numero de Nusselt"
       if Re_D_i_6 >= 2300:
           f f = 0.0375
           Nu_D_6 = ((f_f/8)*(Re_D_i_6-1e3)*Pr_f_6)/(1 + 12.7 * (f_f/8)**0.5 *_U
\hookrightarrow (\Pr_f_6 **(2/3)-1))
       elif Re_D_i_6 >= 10000 and Pr_f_6 >= 0.6 and Pr_f_6 <= 160:
           Nu_D_6 = 0.023 * Re_D_i_6**(4/5) * Pr_f_6 * 0.4
       else:
           Nu_D_6 = 4.36
       "coeficiente de convecção interna"
       h_i_6 = (Nu_D_6*k_f_6)/(d_i)
       "coeficientes globais"
       R_d_{i_6} = 0.0002
       U_{i_6} = (1/h_{i_6} + R_d_{i_6} + d_{i_8} * n_{tubos} * (np.log(d_e/d_i))/(2 *_U)
\rightarrowk_aco) +
                (d_i/d_e) * R_d_e + d_i/(d_e * h_e))**(-1)
```

```
"Queda de pressão"
    f_6 = Re_D_i_6/64
    vm_6_f = m_dot_f/(rho_f_v*A_i_t)
    D_P_6 = rho_f_v * vm_6_f**2 * L_6 * n_tubos/(2*d_i)
    "############ parametros do loop ##########"
    error_f_6 = ((T_f_sai_chut_1 - T_f_sai_6)**2)**0.5
    T_f_{sai_chut_1} = (T_f_{sai_chut_1} + T_f_{sai_6})/2
    if (error_f_6 < (T_f_sai_6 * 0.01)):</pre>
        break
parameters_6['q'].append(q_6)
parameters_6['D_P'].append(D_P_6)
parameters_6['epsilon'].append(epsilon_6)
parameters_6['Re_D_i'].append(Re_D_i_6)
parameters_6['h_i'].append(h_i_6)
parameters_2['T_f_sai'].append(T_f_sai_6)
parameters_6['T_q_sai'].append(T_q_sai_2)
parameters_6['T_f_sai'].append(T_f_sai_6)
parameters_6['L'].append(L_6)
```

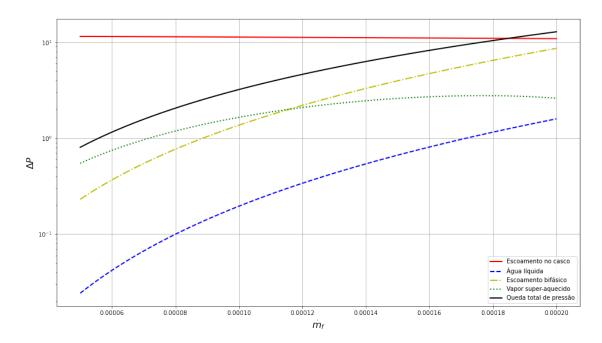
```
]
colors = [
    'r',
    'b',
    'y',
    'g'
]
styles = [
   '-',
    '--',
    '-.',
    1:1
]
names = [
    'Taxa de transferência de calor',
    'Queda de Pressão',
    'Efetividade',
    'Número de Reynolds',
    'Coeficiente de transferência convectiva',
    'Temperatura de saída do fluido quente',
    'Temperatura de saída do fluido frio',
    'Comprimento da zona'
]
symbols = [
    '$q$',
    '$\Delta P$',
    '$\epsilon$',
    '$Re_D$',
    '$h_i$',
    '$T_{q,sai}$',
    '$T_{f,sai}$',
    '$L$'
]
for parameter, name , symbol in zip(parameters, names, symbols):
    fig = plt.figure(figsize=[16,9])
    fig.suptitle((name + ' em função de $\dot{m}_f$'), fontsize=16)
    ax = fig.add_subplot(1, 1, 1)
    ax.set_xlabel('$\dot{m}_f$',fontsize=16)
```

```
ax.set_ylabel(symbol,fontsize=16)
   for zone,z_name,color,style in zip(zones,z_names,colors,styles):
       ax.plot(m_dot_f_eval,zone[parameter],color,_
→linewidth=2,linestyle=style,label = z_name)
       if name == 'Queda de Pressão':
           plt.yscale("log")
       elif name == 'Número de Reynolds':
           plt.yscale("log")
       elif name == 'Temperatura de saída do fluido frio':
           plt.yscale("log")
       elif name == 'Coeficiente de transferência convectiva':
           plt.yscale("log")
   if name == 'Queda de Pressão':
       tot = [p1+p2+p3 for p1,p2,p3 in zip(parameters_4[parameter],
                                           parameters_5[parameter],
                                           parameters_6[parameter])]
       ax.plot(m_dot_f_eval,tot,'k', linewidth=2,linestyle='-',label = 'Queda_u
→total de pressão')
   ax.grid()
   ax.legend()
   plt.show()
   fig.savefig('plots/' + name + '.png', bbox_inches='tight')
   del fig
   del ax
   plt.close()
```

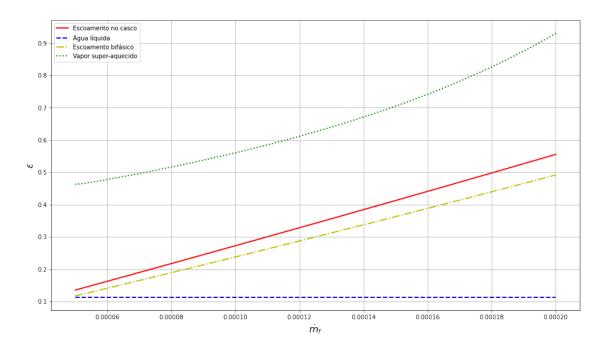
Taxa de transferência de calor em função de $\dot{m_f}$



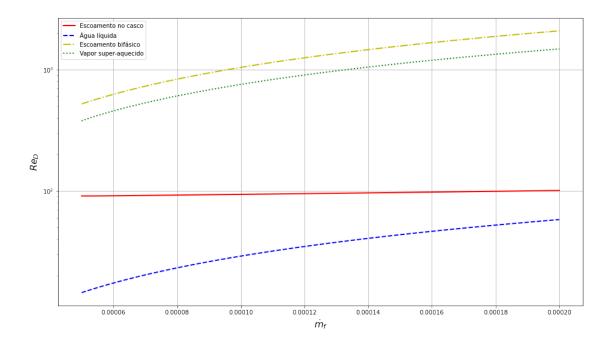
Queda de Pressão em função de $\dot{m}_{\rm f}$



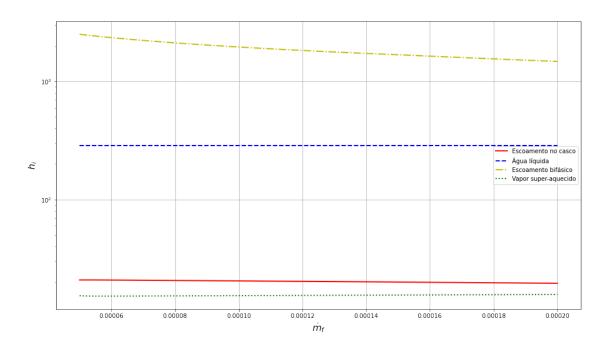
Efetividade em função de $\dot{m}_{\it f}$



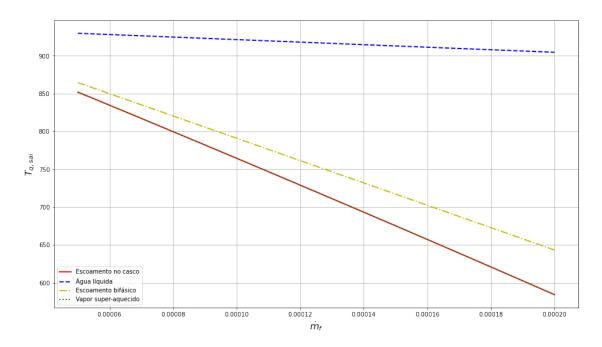
Número de Reynolds em função de $\dot{m}_{\it f}$



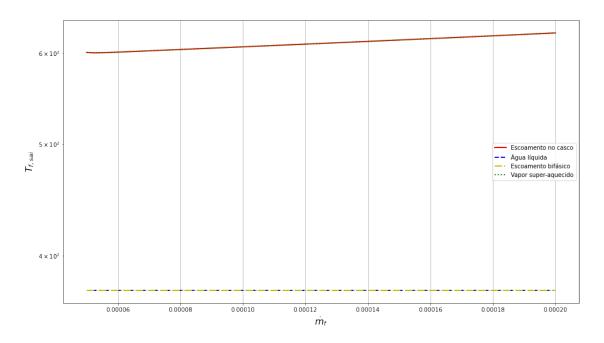
Coeficiente de transferência convectiva em função de \dot{m}_{f}



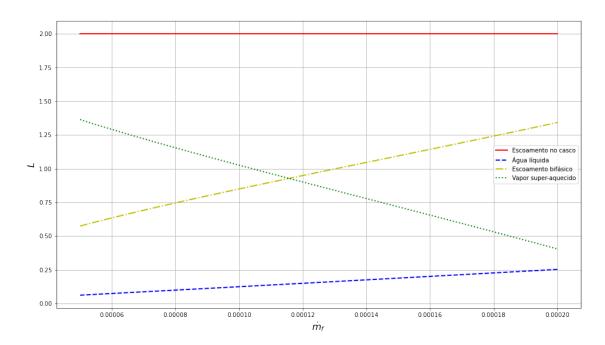
Temperatura de saída do fluido quente em função de \dot{m}_{f}



Temperatura de saída do fluido frio em função de $\dot{m}_{\it f}$



Comprimento da zona em função de $\dot{m_f}$



[]:

[]:[