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AUTOMATIC TRANSMISSION SERVICE GROUP

1989 SEMINAR

TRANSMISSION TECHNOLOGY SIMPLIFIED

This booklet has compiled an introduction to the new CHRYSLER A-500 overdrive read drive automatic transmission, the new A-604 automatic overdrive transaxle and the FORD E40D truck automatic overdrive transmission.

In answer to the many requests for power flow of the different Domestic and Foreign automatic transmissions we have covered the most common units. These charts show what components are on in the different drive ranges. This should be of great help in diagnosing transmission malfunctions.

We have added a section showing the cooler flow from the transmission, the outlet side to the cooler, and the return line fitting in the transmission.

A thought: It might be helpful to arrange transmission problems as follows:

COMPLAINT	or	problem
CAUSE the part or condition creating	the	problem
CORRECTION what is needed to be done to fix	the	problem

Keep in mind you have the need to be able to diagnose whether the transmission problem is ELECTRICAL – HYDRAULIC – MECHANICAL if it is not the computer . . . again we welcome you.

ROBERT D. CHERRNAY

DALE ENGLAND

Technical Director Field Service Consultant



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CHRYSLER A-500

This bulletin provides a description of the A500 four-speed automatic overdrive transmission as well as diagnosis and repair procedures to service this transmission --- initially released for use on the subject model vehicles.

The front portion of this unit is a modified A999 three-speed loadflite automatic transmission. The rear unit, or overdrive, replaces the extension housing and provides a fourth gear with an economy gear ratio (.69 to 1.0) (Figure 1).

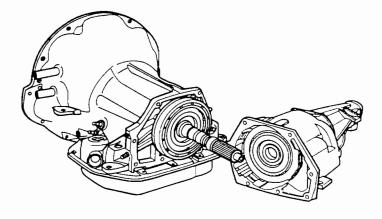


FIGURE 1

A lockup torque converter which is electronically controlled and hydraulically activated, will also be used with this transmission. Fourth gear (overdrive) and lockup will only occur during certain conditions determined by the SMEC (Single Module Engine Controller).

When the vehicle is traveling in third gear over 25 miles per hour, the SMEC uses the following information to allow the shift: The SMEC checks the coolant sensor signal for a 60-degree fahrenheit minimum temperature. It also checks the engine speed sensor, the vehicle speed sensor, the throttle position sensor, and the map sensor.



There is an overdrive off switch located on the instrument panel. The switch is marked O/D OFF and incorporates a pilot light to indicate when the switch has been activated. When activated, the switch will prevent shifts into overdrive and lockup. This is desirable when towing, driving in hilly terrain, or any other driving conditions that make overdrive unsuitable. Activating the switch again will restore the automatic overdrive operation. The switch will reset for operation when the key is turned off.

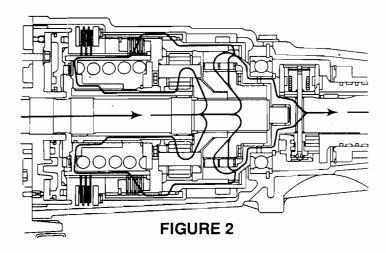
Other features in conjunction with this unit include:

- The output shaft of the three-speed section will now be referred to as the intermediate shaft.
- The three-speed section rear drum is retained on the support with a snap ring.
- The governor and speedometer drive have been relocated to the rear of the output shaft.
- The overdrive case now contains two output shaft bearings. One is the output shaft front bearing, and the other is the output shaft rear bearing.
- There are no rotating seal rings or pressurized oil for the overdrive and direct clutches in the overdrive housing. The governor is the only component receiving pressurized oil through slip-fit tubes. Pressurized oil for the overdrive lubrication circuit is supplied through the intermediate shaft.
- Governor pressure and overdrive pressure taps are provided in the rear of the main case for in-vehicle transmission pressure testing.
- The valve body has been modified by adding several new valves. There's an overdrive solenoid, a 3-4 shift valve, a 3-4 timing valve, a 3-4 accumulator, and a 3-4 shuttle valve. Once in fourth gear, the lockup solenoid, lockup valve, and lockup timing valve accomplish the hydraulics to lock the converter turbine to the torque converter housing.
- The direct drive and overdrive gear ratios are supplied by a third planetary gear set, a direct clutch, an overdrive clutch, and an overrunning clutch. A very strong spring, rated at up to 800 pounds (5,516 KPA), holds the sun gear to the annulus for direct drive. For coasting or reverse gear, power flows only through the direct clutch.

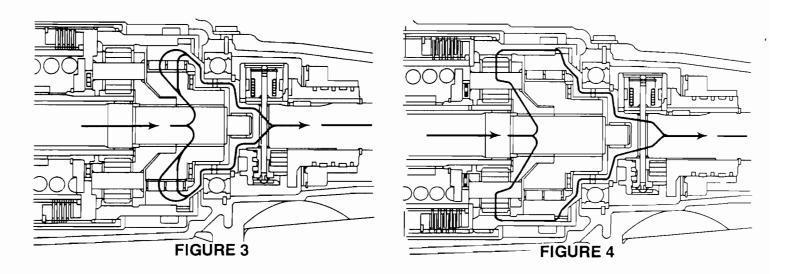
A500 Power Flow

In third gear, the power flow comes off the intermediate shaft simultaneously into the planet carrier and the overrunning clutch. From the carrier, the power flows into the sun gear, then to the inner sliding hub splines. The direct clutch holds the sliding hub to the direct clutch drum under spring pressure. The drum engages directly with the annulus for a one-to-one ratio. The annulus is splined directly to the output shaft (Figure 2).





The overrunning clutch carries the power flow during the shift into fourth gear. When the SMEC energizes the overdrive solenoid, the third gear line pressure starts to move the overdrive piston rearward. The piston pushes the sliding hub and begins to compress the direct clutch spring. Now, with spring pressure relieved, the direct clutch is disengaged. Power flows only through the overrunning clutch (Figure 3).



As the sliding hub is forced further rearward, the overdrive clutch engages, stopping the sliding hub and sun gear and hold them to the overdrive case. As power enters the planet carrier, its gears move around the stationary sun gear and the engine can now turn less for one revolution of the output shaft (Figure 4).

The lockup torque converter is also controlled by the SMEC. Once in fourth gear, the SMEC signals the lockup solenoid to close where the existing pressure builds to move the lockup valve. The lockup valve directs pressure to the torque converter where the turbine is clamped to the spinning torque converter housing. This eliminates any slipping normally attributed to automatic transmissions.



The lockup timing valve releases the torque converter to normal operation prior to the 4--3 downshift.

Additional A-500 Information

- All closed throttle 3-4 upshifts will occur at 25-28 mph, regardless of axle ratio.
- All closed throttle 4-3 downshifts will occur at 25 mph, regardless of axle ratio.
- No 3-4 upshift can be achieved, regardless of vehicle speed, if throttle opening is greater than 70% approximately.

ELEMENTS IN USE AT EACH POSITION OF THE SELECTOR LEVER

	A500			TRANSMISSION						OVERDRIVE		
LEVER	OVER-	START	PARKING	CLUTCHES				В	ANDS	CLUTCHES		
POSITION	DRIVE	SAFETY	SPRAG	FRONT	REAR	O'RUNNING	LOCKUP	K/D FRONT	REVERSE/REAR	0/D	O'RUNNING	DIRECT
P-Park		Х	Х								70121210	D CLEOT
R-Reverse	2.21			Х					х			X
O-Drive					-							
First	2.74				X	х			ļ		x ,	l x
Second	1.54				Х			x	J.	,	X	x
Third	1.00	l ,)	Х	Х		Х	-			X	x
0/D	.69			Х	х		X			х	^	x
2-Second												
First	2.74	i j			x	x					х	х
Second	1.54				x		j	Х			X	X
1-Low	2.74				Х	Х			X		<u>x</u>	



OVERDRIVE DIAGNOSIS

Condition	Possible Cause
• No Reverse or Slips in "R".	 Failed Direct Clutch Overdrive Spring Lost Load Wrong Overdrive Piston Bearing Spacer Selected
• No Overdrive Shift	 Blown Fuse Faulty Overdrive Solenoid Faulty Wiring or Connectors Faulty Overdrive Off Switch Faulty SMEC Failed Overdrive Clutch Wrong Overdrive Piston Bearing Spacer Selected Low Overdrive Pressure Lower Valve Body Malfunction
Runaway Overdrive Shift	• Failed Overdrive Overrunning Clutch
• Overdrive Shift Occurs Immediately Every 2-3 Shift	 Faulty Overdrive Solenoid - Not Venting Lower Valve Body Malfunction Faulty Wiring Faulty SMEC
• Excessively Delayed Overdrive Shift	 Incorrect Overdrive Piston Bearing Spacer Faulty Sensor
• No 4-3 Downshift	 Faulty Lockup Solenoid - Not Venting Lower Valve Body Malfunction Faulty Wiring Faulty SMEC
 No 4-3 Downshift With Overdrive Off Switch 	 Faulty Overdrive Off Switch Faulty SMEC Faulty Lockup Solenoid - Not Venting Faulty Wiring
 Torque Converter Locks Up In 2nd and 3rd Gears 	• Faulty Lockup Solenoid - Not Venting
• Harsh Shifts 1-2, 2-3, & 3-2	• Faulty Lockup Solenoid - Not Venting
• Low Governor Pressure	 Leaking Governor Tubes Bent Loose Fit Governor Seal Rings Broken or Worn



OVERDRIVE DIAGNOSIS - CONT'D.

Condition

Noisy

Possible Cause

- Failed Overdrive Piston Bearing
- Failed Gear Train Needle Thrust Bearings
- Failed Overdrive Planetary
- Failed Overdrive Overrunning Clutch



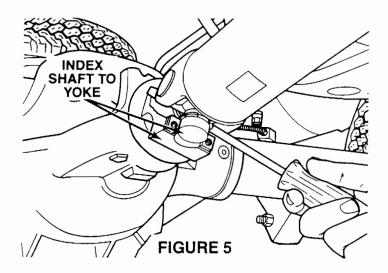
REPAIR PROCEDURE

Removal

This procedure involves removal, disassembly and assembly of the A500 transmission overdrive unit.

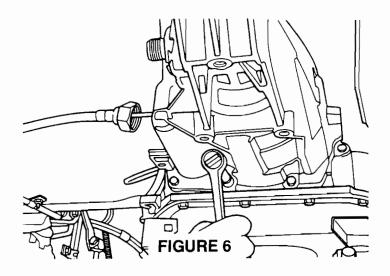
Before raising the vehicle, place the gear shift selector in park.

- 1. To start the overdrive removal, remove the transmission oil pan, remove the gasket, drain the oil and, using a new gasket, reinstall the pan. If there is a failure within the overdrive unit or if the fluid is contaminated, then remove the entire transmission. If the diagnosis indicates clutch or governor problems only, then remove only the overdrive unit.
- 2. Index the drive shaft universal joint to the pinion yoke for proper orientation during reassembly. Then remove the drive shaft (Figure 5).



- Using a transmission jack, support the transmission and raise it enough to remove the crossmember. Mark the position of the crossmember for exact reassembly.
- 4. Remove the speedometer cable.
- 5. Remove the seven bolts securing the overdrive to the transmission (Figure 6).





6. Very carefully pull the unit off of the intermediate shaft. A bearing and a select spacer may be either on the overdrive piston on the rear of the main case, on the sliding hub, or on the intermediate shaft.

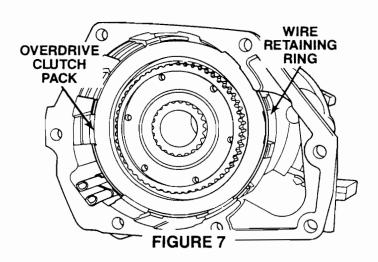
NOTE: ONCE THE OVERDRIVE UNIT IS PULLED BACK APPROXIMATELY 1" IT IS FREE TO FALL IF IT IS UNSUPPORTED.

7. Place several clean shop towels on a bench.

Tip the unit so that any excess oil drains. Watch for any sign of abnormal wear, such as clutch material or metal fragments in the oil.

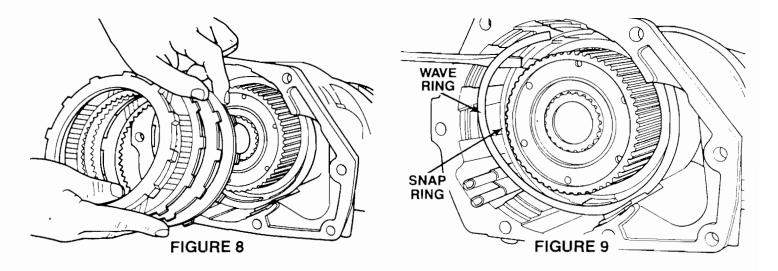
Disassembly

1. Remove the overdrive clutch wire retaining ring, then pull out the alternating metal and friction clutch components. Note that the heaviest metal plate is placed in the front of the clutch pack. This is the pressure plate (Figure 7).

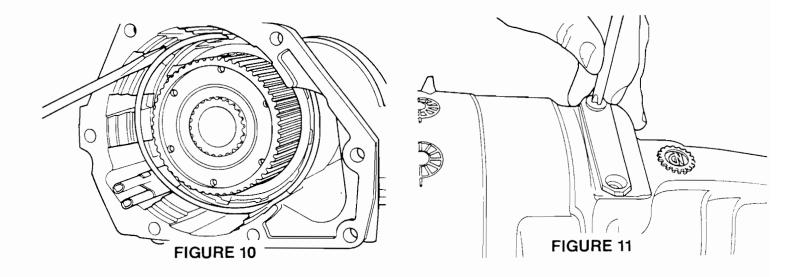




2. Take a close look at each clutch component for signs of wear. Replace if necessary (Figure 8).

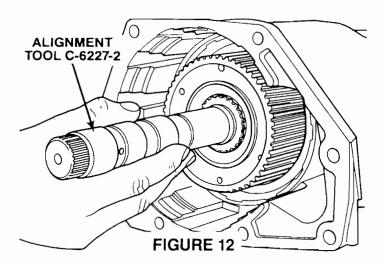


- 3. Next, take out the wave ring. This special ring acts as a cushion to absorb the shock when the overdrive clutch engages (Figure 9).
 - In the same groove, there is another large flat snap ring to be removed (Figure 10).
- 4. Unscrew the two Phillips screws and remove the access plate from the top of the case (Figure 11). The snap ring holds the output shaft front bearing in place.

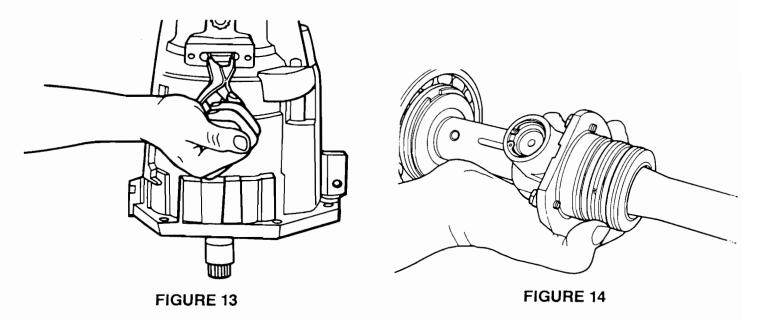




5. Because the entire case must be inverted to remove the gear train, insert the alignment tool (Miller special tool C-6227-2) into the sun gear. After seating it (Figure 12), invert the case on the alignment tool (Figure 13). Use expanding snap ring pliers to expand the output shaft front bearing snap ring and carefully lift the case off the gear train.

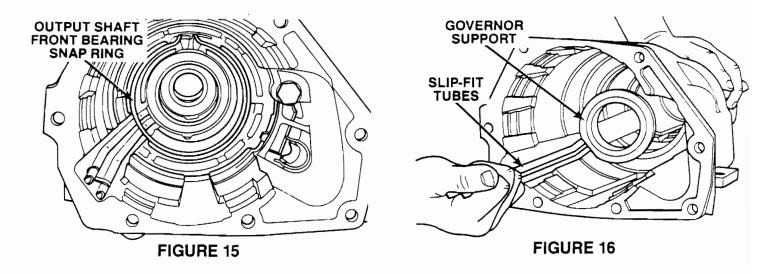


6. Remove governor retaining snap ring. Remove the governor and shaft key (Figure 14). This will prevent damaging the governor when the direct clutch spring is compressed in the arbor press. Set the gear train aside and continue disassembly of the case components.

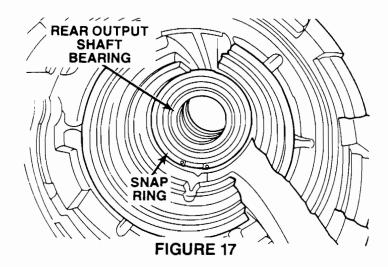




7. Remove the output shaft front bearing snap ring (Figure 15), then remove the governor support snap ring. Take the governor support with slip-fit tubes out of the case (Figure 16).



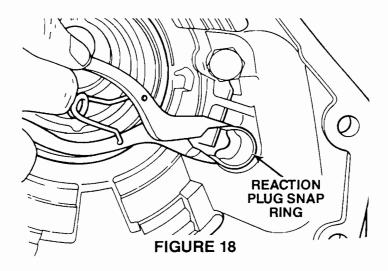
8. Using locking snap ring pliers, remove the output shaft rear bearing snap ring (Figure 17).



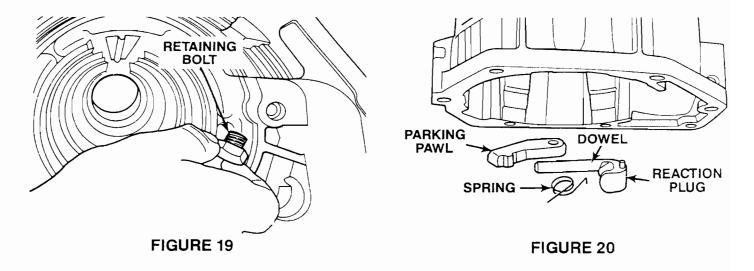
Now, by tapping the case downward on the bench, the bearing should drop out.



9. To remove the parking mechanism, first remove the reaction plug snap ring (Figure 18). Care should be taken to compress the snap ring only enough to allow its removal.



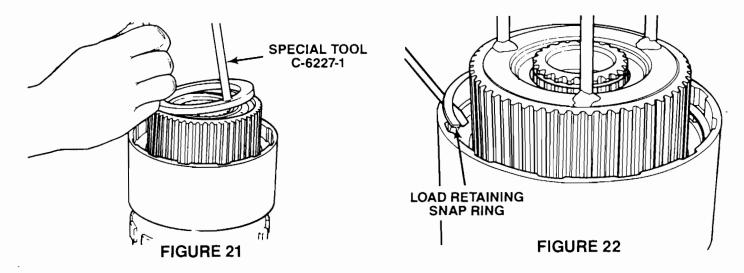
Next, unscrew the bolt securing the dowel and parking pawl (Figure 19). Now, another light tap to the case on the bench will cause the dowel, the parking pawl, and the reaction plug to drop out on the bench (Figure 20).



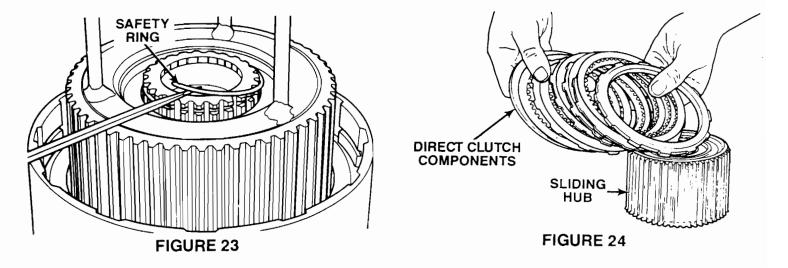
- 10. At the rear of the case, a standard A-727 rear transmission bushing and seal are used at the output shaft. Now that the case is disassembled you may continue disassembly of the gear train.
- 11. It is very important that you use a press capable of three inches of travel to compress the direct clutch spring. A press must have the three-inch travel required to perform this critical step safely. The spring exerts over 800 pounds of force on the sliding hub. Spring tension must be released slowly and completely to avoid personal injury.



12. Place the output shaft in a fixture that will support the output shaft flange. With the assembly properly supported in the press, place the special tool, C-6227-1, in place (Figure 21). Have a helper operate the press to compress the direct clutch spring.



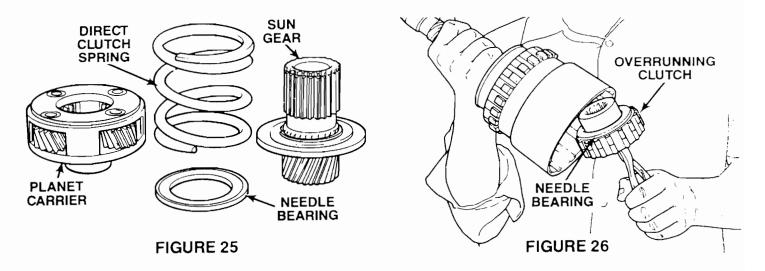
13. When the hub is compressed, you will now be able to safely remove the large load-retaining ring (Figure 22), and the small load-retaining safety ring (Figure 23). When your helper unloads the press, the direct clutch spring tension is relieved. The rest of the unit can be disassembled.



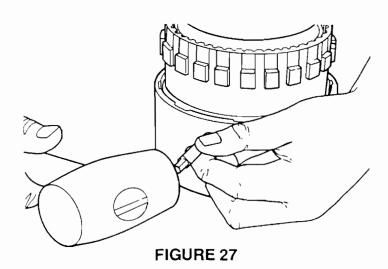
14. Remove the sliding hub with the direct clutch on it. Remove the components of the clutch from the hub and inspect them one at a time (Figure 24).



15. Next, remove the direct clutch spring, the sun gear, a needle bearing pack and the planet carrier (Figure 25).

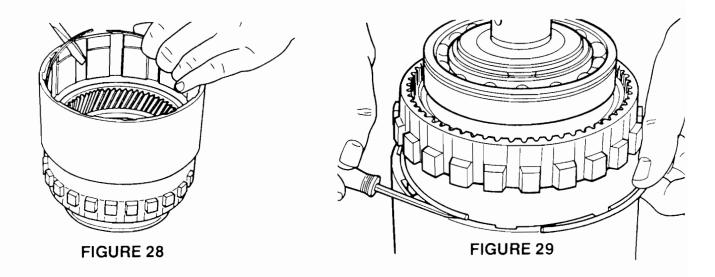


- 16. Next, to remove the overrunning clutch, invert the assembly and, with expanding snap ring pliers, reach into the inner splines of the clutch. You can remove the overrunning clutch intact with a quick counterclockwise twist (Figure 26). Also remove the needle bearing.
- 17. Mark the direct clutch drum and the annulus for exact reassembly (Figure 27).

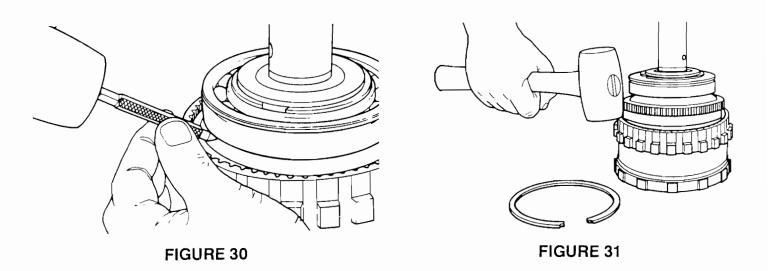




18. Two wire retaining rings secure the direct clutch drum to the annulus. Remove the inner one first (Figure 28), then the one behind the rear of the drum (Figure 29). Now slide the drum from the annulus.



19. Mark the annulus and the output shaft for exact reassembly (Figure 30).



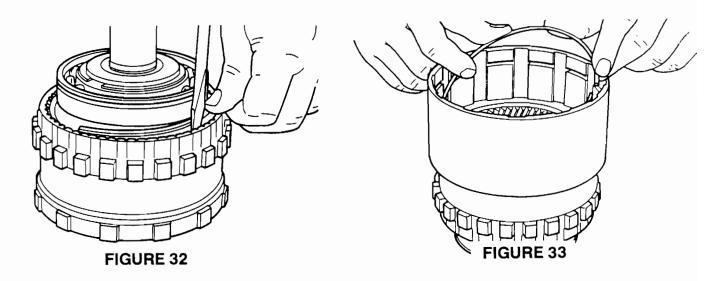
20. To remove the annulus (one snap ring secures it to the output shaft) a light tap with a soft mallet will pop it off the shaft (Figure 31). Now the only component left is the output shaft front bearing which is a slip fit on the shaft.



Assembly

Before assembling, clean all the parts and dry them with compressed air.
 Never clean or dry parts with shop towels as lint deposits could plug oil filter.

To assemble the overdrive unit, align the mating marks, then insert the shaft through the back of the annulus and secure it with the snap ring (Figure 32).

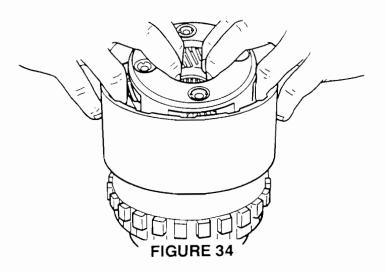


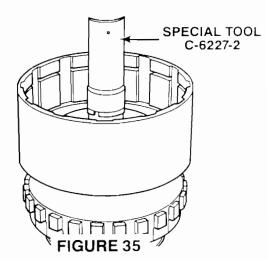
- Next, set the direct clutch drum face down and align the other set of mating marks. Insert the annulus lugs into the slots inside the drum. Install the rear wire retaining ring first. Then invert the assembly. Slide the drum forward to expose the retaining ring groove. Then install the front wire retaining ring to secure the drum (Figure 33).
- 3. Hold the overrunning clutch upside down with expanding snap ring pliers. Place the needle bearing against the back face of the clutch. Now, hold the shaft assembly upside down, and with an upward counterclockwise twisting motion, install the overrunning clutch.

Continued . . .

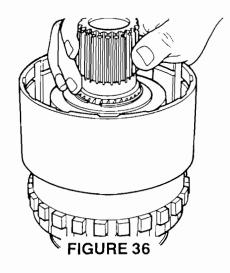


4. Carefully set the carrier assembly into the annulus (Figure 34) and align the splines using special tool C-6227-2 (Figure 35).

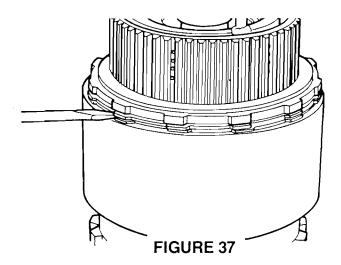




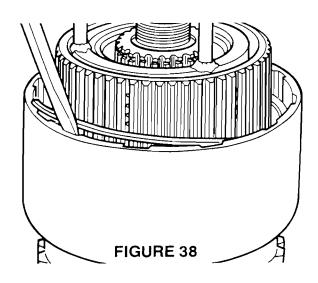
5. Set the needle bearing in place, remove the tool, and install the sun gear (Figure 36). Set the spring on the sun gear, then place the sliding hub on the spring. Reinstall the alignment tool and install the direct clutch plates one at a time on the hub.

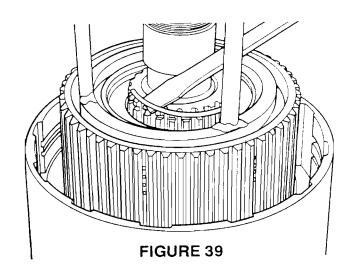






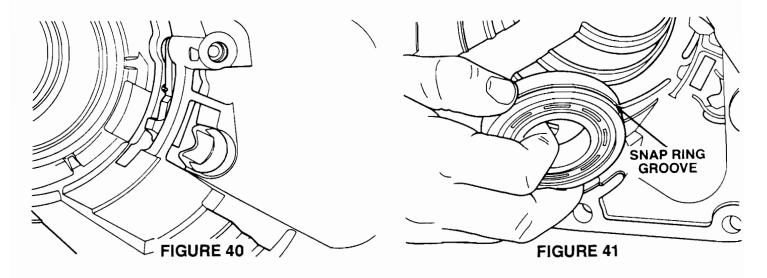
6. This step involves compressing the direct clutch spring with the arbor press. Set special tool, C-6227-1, on the hub and compress the spring. Remember, you'll need a helper to operate the press because, as the spring is compressed, you must slide the clutch plates into their grooves and down the hub (Figure 37.) Then install the large load-retaining snap ring, seating it firmly with a large screwdriver (Figure 38). Now install the safety ring (Figure 39). Slowly release the press, ensuring that the load-retaining rings are properly seated. Install governor and snap ring.



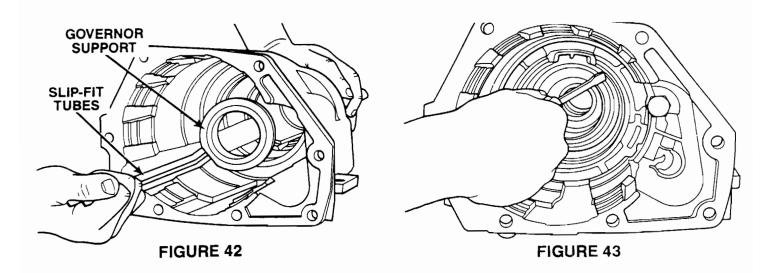




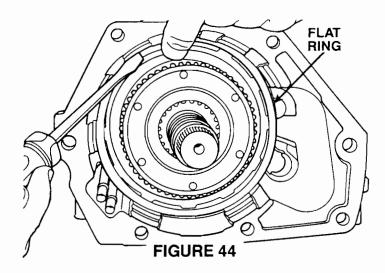
- 7. Set the gear train aside, with special alignment tool still in place, then reassemble the parking mechanism into the case.
- 8. Install the dowel, parking pawl and spring back into the case (Figure 40). Then reinstall the retaining bolt and torque it to 20 foot-pounds (27 N·m). Next, position the reaction plug in place and secure it with its snap ring. Use care to squeeze the snap ring only enough to install it.



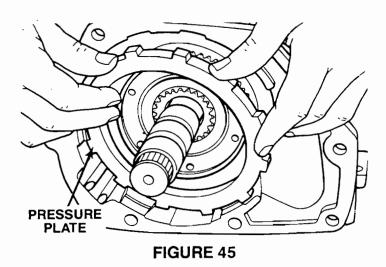
- 9. Set the output shaft rear bearing in the case, ensuring that the groove is toward the front of the case (Figure 41).
- 10. Then install the governor support (Figure 42) and secure it with its snap ring into the overdrive housing (Figure 43).

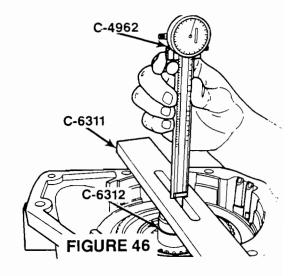


- 11. Install the output shaft front bearing snap ring in the case. Make sure the alignment tool is reinstalled in the gear train. Invert the gear train and slip the case over the shaft.
- 12. Expand the snap ring and slip the case down until the ring locks in the bearing groove. Release the snap ring.
- 13. After reinstalling the access plate and gasket, install the flat snap ring. Use a large screwdriver to be sure that it's seated properly. Then reinstall the wave ring (Figure 44) using the same seating technique.



- 14. One by one, install the overdrive clutch plates, be certain to put the thickest plate in last (Figure 45).
- 15. Position the overdrive unit vertically in a large vise. To determine the proper intermediate shaft spacer thickness, insert special tool (C-6312) through the sun gear. Be sure that the tool bottoms against the carrier spline shoulder. Position special tool (C-6311) across overdrive case face. Using dial caliper tool (C-4962), positioned over tool C-6311, measure distance to the top of tool C-6312 (Figure 46).





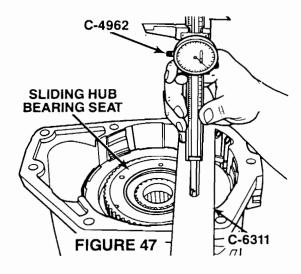


Using this measurement, select the proper thickness spacer from the chart.

Intermediate Shaft Spacer Chart

Measurement (Inches)	Use Spacer No.	Spacer Thickness (Inches)
.73367505	4431916	.159158
.75067675	4431917	.176175
.76767855	4431918	.194193
.78568011	4431919	.212211

16. To determine the proper shim thickness for the overdrive piston: Position special tool, C-6311, across overdrive case face. Then using dial caliper tool (C-4962) positioned over tool C-6311, measure distance to the sliding hub bearing seat (Figure 47).





This measurement should be taken at four locations 90° apart. Add all measurements together and divide by four (4). Using this measurement, select the proper thickness shim from the chart.

Overdrive Piston Shim Chart

Measurement (Inches)	Use Shim No.	Shim Thickness (Inches)
1.7500-1.7649	4431730	.108110
1.7650-1.7799	4431585	.123125
1.7800-1.7949	4431731	.138140
1.7950-1.8099	4431586	.153155
1.8100-1.8249	4431732	.168170
1.8250-1.8399	4431587	.183185
1.8400-1.8549	4431733	.198200
1.8550-1.8699	4431588	.213215
1.8700-1.8849	4431734	.228230
1.8850-1.8999	4431590	.243245

Installation

1. Before installing the overdrive unit, it will be necessary to cut out the old gasket. Using a sharp knife, cut out the old gasket around the piston (Figure 48). Place the old gasket on the new one for a template and trim the new gasket to fit.

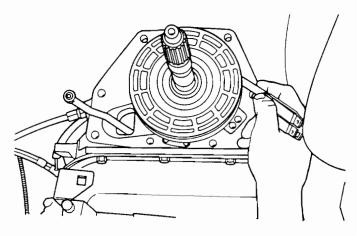


FIGURE 48



- 2. Place the new gasket in place on the rear of the main case. Install the spacer selected in Step 15 of the Assembly Procedure on the intermediate shaft.
- 3. Place the shim selected in Step 16 of the Assembly Procedure in position over the piston on the main portion of the transmission. Install the sliding hub bearing over the intermediate shaft, against the sliding hub.

NOTE: THE SHOULDER ON THE INSIDE DIAMETER OF THE BEARING MUST FACE FORWARD.

A small amount of petroleum jelly can be used to hold the shim and bearing in position.

- 4. Carefully lift the overdrive unit and slide it onto the intermediate shaft. Insert parking rod into reaction plug. Extreme care must be used not to tilt the unit as this could cause the carrier and overrunning clutch splines to rotate out of alignment. If this happens, it will be necessary to remove the overdrive unit and align them with tool C-6227-2. Align the slip-fit tubes and push the unit forward until it touches the main case.
- 5. Install the seven attaching bolts. Tighten in a crisscross pattern to 25 foot-pounds (34 N·m).
- 6. Install the crossmember, speedometer cable and drive shaft using marks made at disassembly.
- 7. Refill to proper level with Mopar ATF Plus (Automatic Transmission Fluid Type 7176) or equivalent.



TRANSMISSION - AXLE - TIRE AVAILABILITY

		Axle Rat	tios
	<u>Tires</u>	3.9L Eng.	5.2L Eng.
B150	P195/75R15 Min. P235/75R15 Max.	3.9L Only	3.55 Std. 3.91 Opt.
B250	P225/75R15 Min. P235/75R15 Max.	3.9L Only	3.55 Std. 3.91 Opt.
B350		N/A	3.9 4.1 Opt.
N1	P185/75R14 LT 215	3.55 3.91 Opt.	N/A
N 5	P195/75R15 P235/75R15	3.55	N/A

A500 FLUID CAPACITY

		Quarts	Liters
A500 - With 3.9L	or 5.2L Engine	10.2	9.6



CHRYSLER A-604

ELECTRONIC, FOUR-SPEED AUTOMATIC TRANSAXLE

Features and Benefits: An all-new, electronically-controlled, four-speed automatic transaxle is available with the 3.0 L V-6 engine in New Yorker, New Yorker Landau, Dynasty, Spirit ES, Acclaim LX, Caravan LE, all Grand Caravan models, Voyager LE, and all Grand Voyager models.

The customer will find this all-new transaxle to be very smooth and unobtrusive while providing reduced noise, improved highway fuel economy, faster and smoother response, and improved shift quality.

The transaxle provides faster acceleration in conjunction with a 3.43:1 final drive ratio, a ratio 6% to 23% higher than the ratio used with previous three-speed transmissions. When the transmission shifts to fourth gear, which is overdrive, the overall ratio drops to 2.36:1 to provide quieter operation. To provide good fuel economy in conjunction with the lower overall ratio, the torque converter locks in fourth gear, thereby eliminating slippage.

Shifts are very smooth due to fully adaptive electronic control which senses the speed changes between components within the gear train as shifts occur and adjusts hydraulic pressure as needed. This control method contrasts sharply with conventional automatic transmissions which shift by applying hydraulic pressure through orifices and mechanical accumulators based on a predetermined set of assumptions about engine output and friction material characteristics.

Chrysler Motors' use of fully-adaptive electronic transmission controls in its new four-speed automatic transaxle :

A unique feature of electronic control is partial lock up of the torque converter which produces a smooth transition to full lock up. The speed differential between the input and output shafts of the transmission prior to lock up is typically 250 rpm. Partial lock up brings that differential into the range of 50-100 rpm, then completes the lock up. Adaptive control can do this because it can sense the speed differential and apply just enough pressure to the lock-up clutch to achieve the small slippage. After partial lock-up is achieved, pressure is increased incrementally until full lock-up is complete.

The electronic adaptive controls provide kick-down shifts with a smoothness that is unmatched by any previous unit, and in so doing, make the powertrain feel more responsive without increasing harshness. Being adaptive, these controls inherently compensate for changes in engine or friction element torque and provide good, consistent shift quality for the life of the transmission.

AUTOMATIC TRANSMISSION SERVICE GROUP

ATSG

Technical Service Information

ELECTRONIC FOUR-SPEED AUTOMATIC TRANSAXLE

Function: The transmission provides forward ratios of 2.84, 1.57, 1.0, and 0.69 with lockup available in 4th gear; the Reverse ratio is 2.21. The shift quadrant has six positions: P, R, N, OD, D, and L. The OD position is actually a "D" inside an "O" to indicate overdrive operation. When OD is selected, the transmission shifts normally through all four speeds. It is recommended for most driving. The D position is used for hilly or mountainous driving. When D is selected, the transmission uses only 1st, 2nd, and 3rd gears. When operating in D or L positions torque converter lock-up occurs in third gear for improved transmission cooling when towing trailers on steep grades. If high engine coolant temperature occurs, the torque converter will also lock up in 2nd gear. The L position provides maximum engine braking for descending steep grades. Unlike most current transmissions, up-shifts are provided to 2nd or 3rd at peak engine speeds if the accelerator is depressed. This provides engine over-speed protection and maximum performance.

Description: The electronic controls make the transmission unique in a number of ways. First, the adaptive controls are used to significantly reduce complexity. Relative to today's three-speed unit, the new transaxle requires no additional gearing, one less overrunning clutch, and only one more friction element. It has 20 fewer part numbers than today's three-speed unit. The resulting compactness allows the new four-speed transaxle to package in the same vehicles and with the same ground clearance as the three-speed unit and the lockup torque converter is similar to the one used in other front-wheel drive units. Torque capacity for future vehicles and engines has been assured by using larger gearing throughout than in the present three-speed transaxle; yet it weighs only 5.9 kg (13 pounds) more and is only 13 mm (0.5 in.) longer. For manufacturing simplification, the planetary gears are the same diameter and length and have the same number of teeth as those used in our three-speed reardrive passenger car unit. Also, the engine-to-differential center line dimension has been preserved, simplifying installation in existing vehicles.

ELECTRONIC FOUR-SPEED AUTOMATIC TRANSAXLE AND FINAL DRIVE

The transmission uses only clutches to change ratios. Clutches provide smooth, consistent shifts whereas bands, which are used in some transmissions, are harder to control and less consistent.

Actuation and release of the clutches is controlled by ball-type solenoid valves, which were chosen for maximum reliability in the transmission operating environment. Moreover, the solenoids operate the valves directly without any intermediate element-

Further simplification is achieved through a unique logic-controlled switching valve which permits one solenoid to control the application of two friction elements. Any selection of 2nd, 3rd, or 4th gear elements causes this valve to

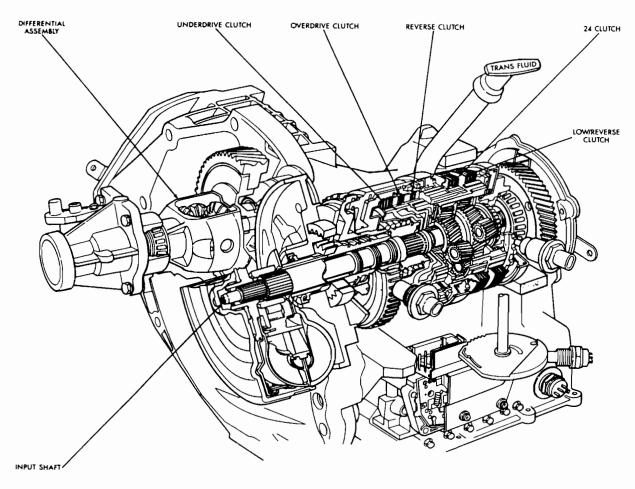


release Low Gear. A failure cannot reselect it. When a shift to Low Gear is appropriate, the logic first determines that no malfunctions exist, then a specific sequence of solenoid commands is used to shift the valve and again select Low Gear.

The control electronics are located underhood in a potted, die cast aluminum housing with a sealed, 40-way connector. On New Yorker, and New Yorker Landau, Dynasty, Spirit and Acclaim, the control computer is located on the right fender side shield. On Caravan and Voyager it is located on the right side of the dash panel.

The transmission control computer for the Dodge Caravan and Plymouth Voyager includes control logic to protect the transmission from overheating. In trailer towing situations where the vehicle is operated in "D" which does not permit a shift to overdrive, the torque converter will lock up in 2nd as well as 3rd gear if the coolant temperature becomes moderately high. This reduces transmission heat rejection by reducing torque converter slippage and reduces engine heat rejection by lowering engine speed.

Through the use of SMD's (surface mount devices) and ASIC's (application-specific integrated circuits), the controller size was minimized. The electrical power requirements of the control system have been minimized by using switch-mode, current-controlled solenoid drivers and an efficient CMOS (complementary metal-oxide semi-conductor) microprocessor. These features combine to provide a state-of-the-art control system for the transmission.





Because the A-604 has fully adaptive electronic controls, it provides superb performance with a relatively simple mechanical design. For example, there are no bands and no low-gear overrunning clutch.

Compared with typical competitive four-speed automatic transaxles, the A-604 has substantially fewer parts.

Because of its compact design, it can be used in the same applications as the three-speed automatic transaxle.

INPUT CLUTCH ASSEMBLY

The three input clutches that supply input power through the transaxle are contained in the input clutch assembly (Fig. 1). These are the underdrive clutch, the over-

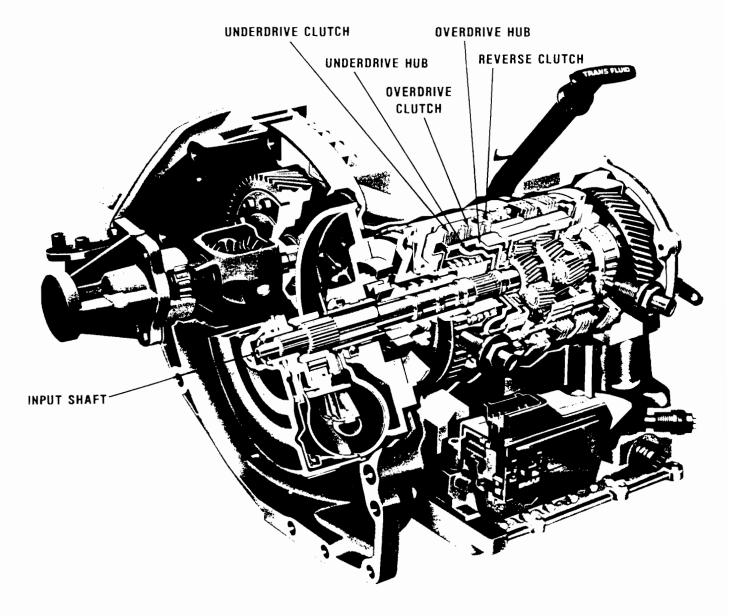


Fig. 1 — The input clutch assembly includes the input shaft, the overdrive hub and the underdrive hub, as well as the three input clutches.



drive clutch and the reverse clutch. The input clutch assembly also includes the input shaft, the overdrive hub and the underdrive hub.

TWO-FOUR CLUTCH AND LOW-REVERSE CLUTCH

In addition to the three input clutches, there are two other clutches: the two-four clutch and the low-reverse

clutch (Fig. 2). These are splined to the case and provide reaction torque by holding various components.

OTHER COMPONENTS

Other major components of the A-604 transaxle include the planetary gear sets, the front annulus/rear carrier assembly and the rear annulus/front carrier assembly.

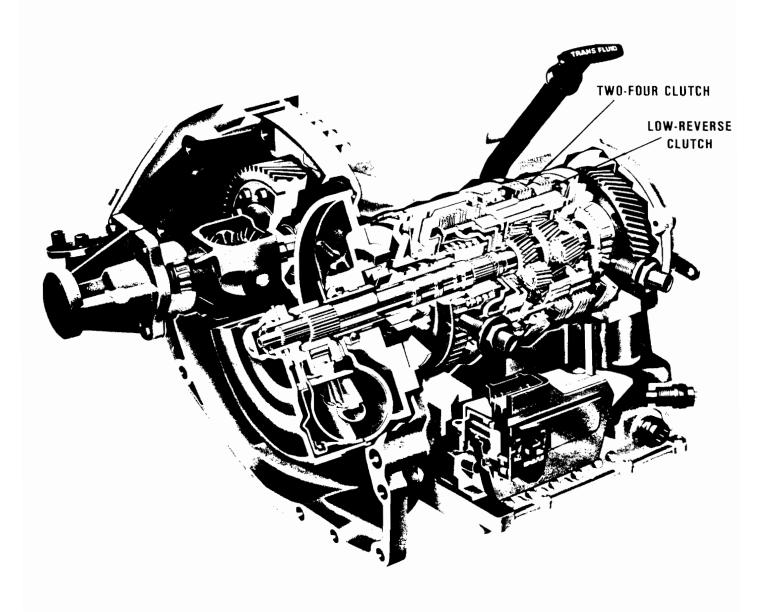
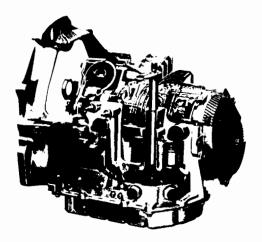


Fig. 2 — The low-reverse and two-four clutches provide reaction torque by holding various elements of the planetary and carrier assemblies.

AUTOMATIC TRANSMISSION SERVICE GROUP



The elements in use chart (Fig. 3) shows mechanical elements are involved in the transaxle power flow in each gear.

NEUTRAL/PARK

In neutral or park, there is no power flow through the transaxle — none of the input clutches are applied. However, the low-reverse clutch (Fig. 4) is applied to supply reaction torque in preparation for a shift into drive or reverse.

ELEMENTS			CLUTCHES					
IN USE SHIFT LEVER POSITION	Start Safety	Park Sprag	Underdrive	Overdrive	Reverse	2/4	Low/Reverse	
P — PARK	Χ	X					X	
R — REVERSE					Χ		X	
N — NEUTRAL	Χ						X	
OD — OVERDRIVE First			Χ				Х	
Second			Χ			X		
Direct			Χ	Χ				
Overdrive				Χ		X		
D — DRIVE* First			Χ				Х	
Second			Χ			Χ		
Direct			Χ	Χ				
L — LOW* First			Х				Х	
Second			Χ			X		
Direct			Χ	Χ			_	

Vehicle upshift and downshift speeds are increased when in these selector positions.

Fig. 3 — This chart shows the A-604 elements in use in each

gear.

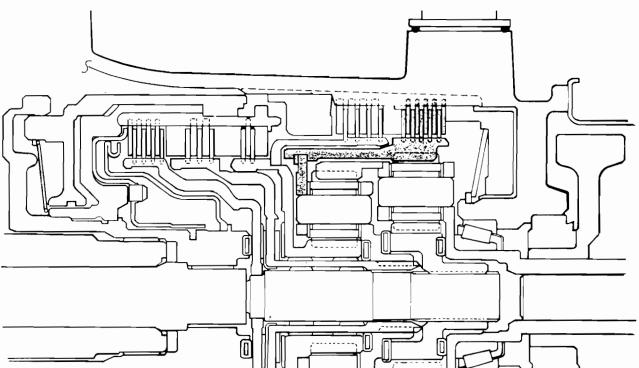


Fig. 4 — In neutral or park, there is no power flow through the transaxle.



LOW GEAR

In low gear, the ratio is 2.84:1. Torque input is through the underdrive clutch to the underdrive hub assembly, which turns the rear sun gear. The low-reverse clutch is applied to hold the rear annulus/front carrier assembly. The rear sun gear drives the pinions around the stationary rear annulus, causing the rear carrier to rotate and provide output torque (Fig. 5). In this gear, the rest of the planetary set is freewheeling.

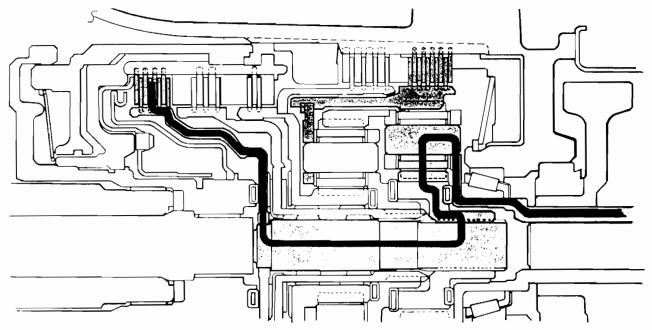


Fig. 5 — Power flow in low gear.

SECOND GEAR

In second gear, the ratio is 1.57:1. This ratio is achieved by having both planetary gear sets contribute to torque multiplication (Fig. 6). As in low gear, torque input is through the underdrive clutch to the rear sun gear. The two-four clutch is also applied, holding the front sun gear stationary. The rotating rear sun gear turns the rear pinions. The rotating rear pinions transmit torque to the rear annulus/front carrier assembly while the front pinions rotate around the stationary front sun gear. This transmits torque to the front annulus/rear carrier assembly, which provides the output torque.

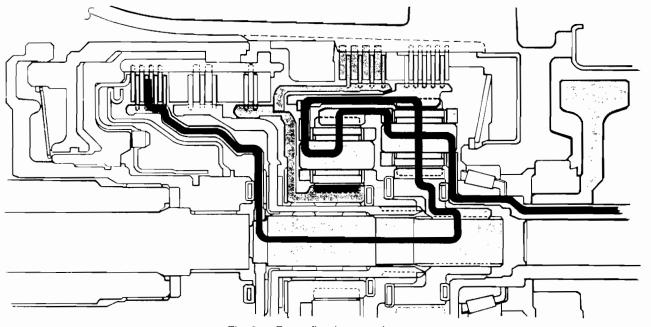


Fig. 6 — Power flow in second gear.

THIRD GEAR

In third gear, the ratio is one to one. Two input clutches — the underdrive clutch and the overdrive clutch — are applied to provide input torque (Fig. 7). The underdrive

clutch rotates the rear sun gear while the overdrive clutch rotates the front carrier/rear annulus assembly. In effect, this locks the entire planetary gear set so that it rotates as a unit.

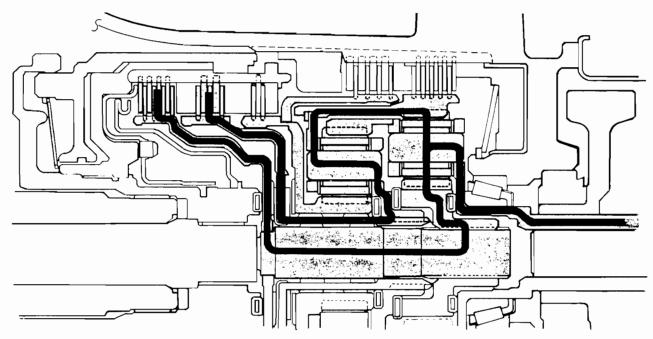


Fig. 7 — Power flow in third gear.

FOURTH GEAR

In fourth gear, which is overdrive, the ratio is 0.69:1. This means that output speed is greater than input speed. Input is through the overdrive clutch while the two-four clutch is applied to hold the front sun gear (Fig.

8). As the front carrier rotates, it causes the pinions to "walk around" the stationary front sun gear and turn the front annulus/rear carrier assembly, which provides output.

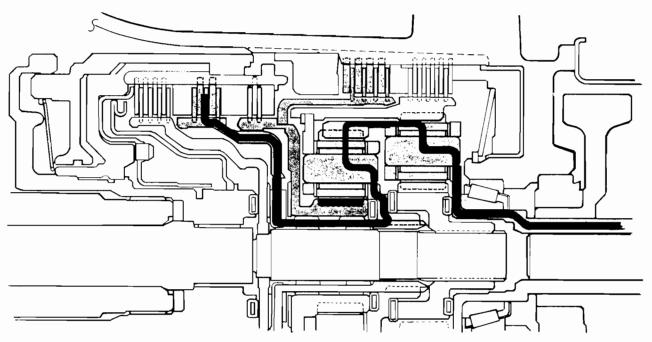


Fig. 8 — Power flow in fourth gear.



REVERSE

In reverse, the ratio is -2.21:1. Input is through the reverse clutch, which drives the front sun gear (Fig. 9). The low-reverse clutch is applied to hold the front car-

rier/rear annulus stationary. The front sun gear rotates the front pinions, which in turn rotate the front annulus/rear carrier assembly — providing output in the reverse direction.

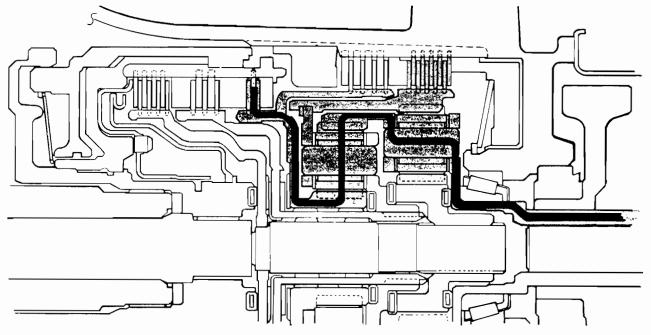
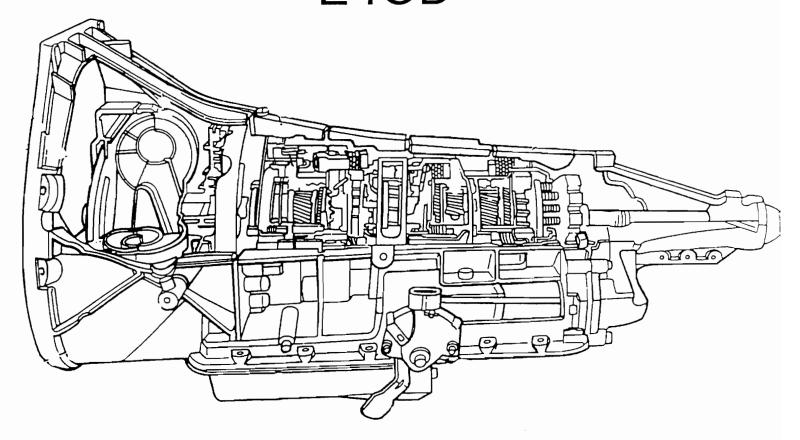


Fig. 9 — Power flow in reverse gear.



FORD E4OD



E40D Automatic Transmission

The E4OD Transmission is a fully, automatic, electronically controlled, four-speed unit with a three element locking torque converter. The main operating components of the E4OD transmission include a converter clutch, six multiple-disc friction clutches, one band, two sprag one-way clutches and a roller one-way clutch which provide for the desired function of three planetary gear sets.

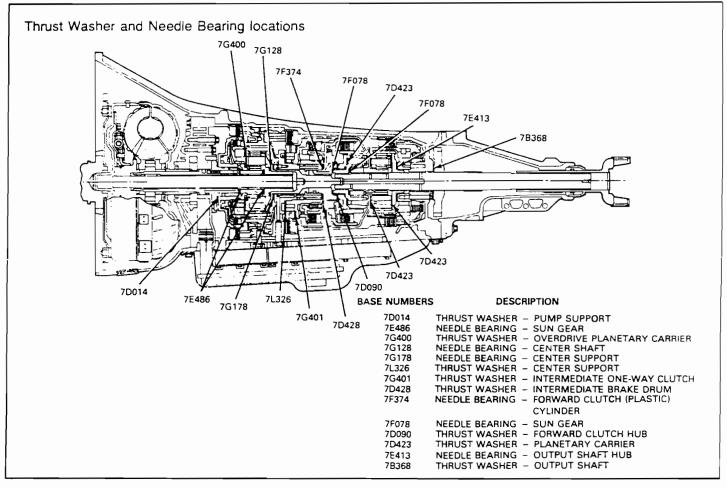
Transmission gear selection in the (b) range and converter clutch operation is controlled by the EEC-IV control system. Operating conditions are relayed to EEC-IV by various sensors throughout the vehicle. The EEC-IV compares these conditions with electronically stored parameters and logically determines the state that the transmission should operate at.

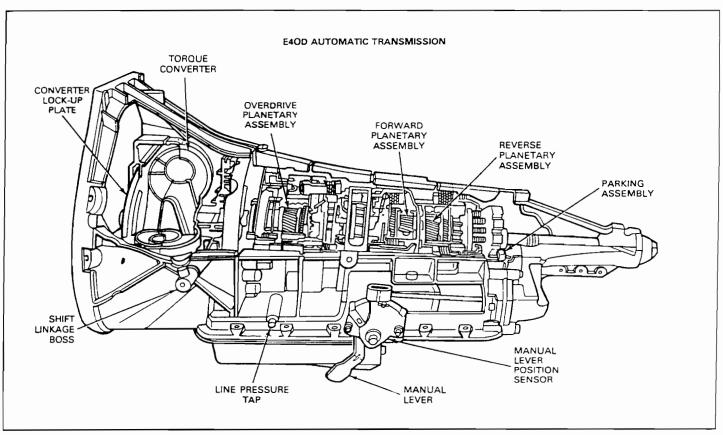
In the (1) range, automatic operation of all four gears is possible. The Overdrive Cancel Switch, located on the vehicle's dashboard, disables overdrive operation and enables automatic operation through the first three gears.

Manual gear selection is available in the 1 and 2 range. Second gear is commanded when the gear selector is in the 2 range and when downshifted into the 1 range at speeds above approximately 56 Km/h (35 mph) (for diesel 48 Km/h (30 mph). First gear is commanded in the 1 range at startups and when downshifted into 1 range below approximately 56 Km/h (35 mph) (for diesel 48 Km/h (30 mph).

NOTE: Any reference to Intermediate Brake Drum and Direct Clutch Cylinder are one and the same.







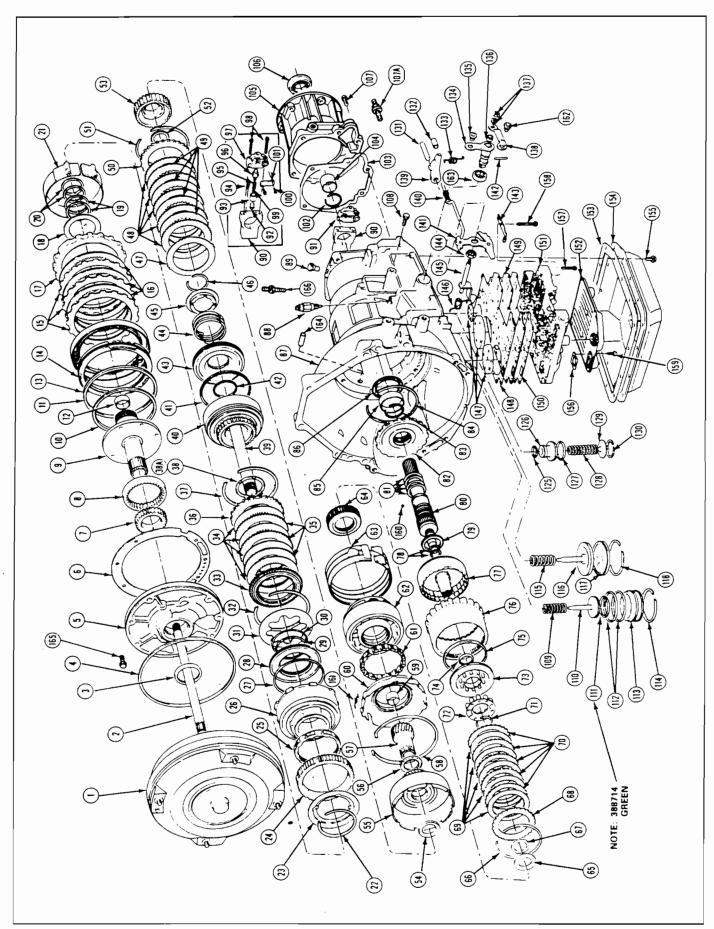
AUTOMATIC TRANSMISSION SERVICE GROUP



3	L		2								
₹	₹ %	Description	₹ 5 8	₹ 9.	Description	ਨੂੰ ਵੇਂ ਦੇ	Part No.	Description	다. 왕 -	æ æ. F.	Description
- 0	7902	Torque converter	\$	7F207	Forward clutch cylinder & turbine	26	7F273	Output shaft large (4) steel seal rings	125	7F250	2-3 accumulator valve seal (small)
~ •	7.50	Cred orve shart	3	,	snatt	;		47, 48, 49, 410	8	7F251	2-3 accumulator valve
2	77.44	Front pump seal	- 5	75.22	Forward dutch piston seal (outer)	38	20107	Output shaft hub	127	7F249	2-3 accumulator valve seal (large)
r 10	74103	Front pump body	4 3	7,140	Forward clutch piston	3	5176	Hetaining snap and (O.P.S. nub to O.P.S.)	<u>s</u> \$	75252	2-3 accumulator valve return spring
9	7A136	Front pump gasket	1	7F230	Forward clutch piston return spring	2	70122	Relaining span find (O P.S. hub to	3 5	388492	Retaining soon time (2.3 postum to
7	70010	Front pump drive gear	45	7.523	Relum spring retainer	;	1	ning gear)	3	1	Case)
60	25	Front pump driven gear	4	388099	Retaining snap nng	88	7E110	Rear case bushing	131	7D071	Park pawl shaft
თ	7A108	Stator support — front pump	4	75085	Waved spring	88	7F242	#9 needle bearing (rear case)	132	7D419	Guide cup
유	7F225	Interm. clutch piston inner lip seal	\$	7B442	Forward clutch external spline steel	87	7005	Case assy.	£	7D070	Park pawl return spring
= :	7F224	Interm. clutch piston outer lip seal	•	į	plate	88	7A247	Neutral start switch	\$	7F338	Manual lever
2 5	78258	Front pump bushing	4 9	/E311	Forward & reverse clutch internal	æ :	7034	Vent cap	£5.	734	Grommet
2 ;	1535	Inform dutch pictor	5	25070	spine inclion plate	86 8	7A189	Governor counterweight	8	7F337	Throttle lever oil seal
<u>4</u>	77.	menn. Guidh pision return springs a	8	0/7/	rorward & reverse dutch pressure	5 8	7,0063	Body assy. — governor	137	M62004-S51	Attaching nut & lock washer — M8 x 1.25
55	7F220	Interm. clutch external soline steel	5	70483	Retaining snap ring (selective)	8 8	74304	Sleeve covernor	8 5	7441	Infome lever (outer)
!	i :	plates (sel.)	25	76040	#3 needle bearing (Md. clutch)	35.	7E242	Screen assv. — oov. oil	3	7A232	Park pays actuation rod
16	7F219	Interm. dutch internal spline friction	ន	7D051	Forward clutch hub	፠	7A302	Spring gov, valve	141	7A115	Manual lever (inner)
		plates	Z,	7F244	#4 needle bearing	88	70054	Valve governor	142	7B210	Roll pin — 1/8 x 0.95 grooved
17	7F226	Interm. clutch pressure plate	22	7A019	Reverse sun gear & drive shell assy.	93	7A300	Body governor	143	7E332	Detent spring
8 2	7001 4	#1 thrust washer (front pump)	%	7F244	#5 needle bearing	88	N800273	Bolt (governor body to counterweight)	<u>‡</u>	N800287-S51	Attaching nut (manual LVR) —
,		selective	2	78399	Forward sun gear	8	7A305	Clip — governor spring retainer			M14 x 1.5 HEX
6	7F275	Stator support seal nings (rev. dutch)	8 8	38850	Center support retaining ring	5	N800274	Bolt (governor cover to governor	145	7F290	Throttle lever (inner)
8	95035	# 1 and # 2	25	/F209	Porward sun gear precision bushing	Ş		(Apoq	146	7F292	Throttle torsion spring
₹	115/0	Statos Support Seat Imas (IMC. Ciutor)	3 2	3 5	Certier support planetary Planetary OWC case sector 1 collect	≥ §	1930	Cover — governor valve body	147	7F282	Valve body reinforcement plate
21	7F196	Overdrive band	5	1	assv.	2	5	Indiaming shap ing (governor ass).	9 7	7400	Separator plate gasket (upper)
ខ	389790-S	Interm. OWC retaining snap	છ	7A398	Planetary assy.	ā	2086	Extension housing gasket	3 53	20107	Separator plate packet (lower)
ន	7F262	Interm. OWC retaining plate	ន	7D095	Reverse band	ই	7A034	Extension housing bushing	151	7A100	Valve body (main control)
72, 5	7F221	Interm. OWC outer race	Z :	7F236	Direct clutch hub	ই	7A039	Extension housing	152	7F003	Filter & grommet assy oil pan
8 8	7F271	Inferm. one-way dutch assy.	65	71-243	#7 needle bearing (direct clutch	\$ 5	7052	Extension housing seal	<u>s</u>	7A191	Oil pan gasket
2 8	70403	Beverse Auth pistos seal (order)	8	289765.6.7.9	Relation and the (caloritie)	2	MB03/4/-	Boff (ext. hsg. to case) M6-12.5 x 30	<u> </u>	7.4264	Oil pan
; æ	7E079	Reverse clutch piston	8 29	7F237	Thrust spacer	107A	Managas.	(bried d) Stud (extribed to case) MR-125 x 54	2	390233-32	bort (oil pain to case) MB x 1.25 x
R	70407	Reverse clutch piston seal (inner)	88	78477	Direct clutch pressure plate	•	S100	(2 red'd holes 1 and 6)(2)	156	7E062	Oi file pasket
8	7D256	Thrust ring	69	7E313	Direct clutch internal spline plates	\$	376649	Pipe plug — 1/8-27 dry seal	157	N605775-S	Boft (valve body to case) — M6-1.0 x
ا	500	Reverse clutch piston return spring	۶;	7F238	Direct clutch external spline plates	<u>8</u>	7F201	Overdrive servo piston return spring			30 (8 req'd).
3 8	78066	Reverse clutch frod pressure plate	2 2	7525	Return soring & retainer	2 :	7F200	Overdrive servo piston	<u>8</u>	N606022-S	Boft (valve body to case) — M6-1.0
3 25	7E31	Beverse clutch internal spline friction	3.2	75.54	Direct clatch piston	= \$	286515.C.IM	Overdine servo piston seal	150	ALCOUNTY C	X 40 (17 red d)
		plate	74	7F234	Direct clutch piston seal (inner)	13	75204	Overdrive servo cover	S.	6-27/6001	M6-1 0 x 16 (3 recid)
೫	7B442	Reverse clutch external spline steel	75	7C000	Direct clutch piston seal (outer)	=	388216	Retaining snap ring (O/D servo to	991	353351-5	Ball (governor drive)
		plate	92	7A153	Ring gear & park gear			case)	191	7F277	Spring (anti clunk)
8	7F278	Forward and reverse clutch	F 8	7F283	Direct cylinder	= 2	7D031	Reverse servo piston return spring	162	7F434	Grommet
	70,482	pressure plate	82	/F284	Output shaft small (2) Tellon seal	<u>9</u> :	20030 20030	Reverse servo piston (selective)	<u>হ</u>	78498	Oil seal assy. Manual lever
ج ج ح	7007	#2 thoust washer free clutch)	70	7E240	#8 poodle beans (drest clark	= =	70281	Heverse servo cover	<u> </u>	1 0000	Overdrive anchor-pin — not serviced
8 8	7A166	#2 Wildst #asile! (rev. clutch)()	2	747 L	ouler)	=	388215	Hetaining snap ring (rev. servo to	18	N605/89-5100 N804/795-5100	Bolts (front pump to case — 7 req'd)
æ	7F212	Turbine shaft	8	7060	Output shaft			(actr)	3	M04/33/0W	5/16 tube x 1.4 external pipe
Š	(i) Some Applications	ions									

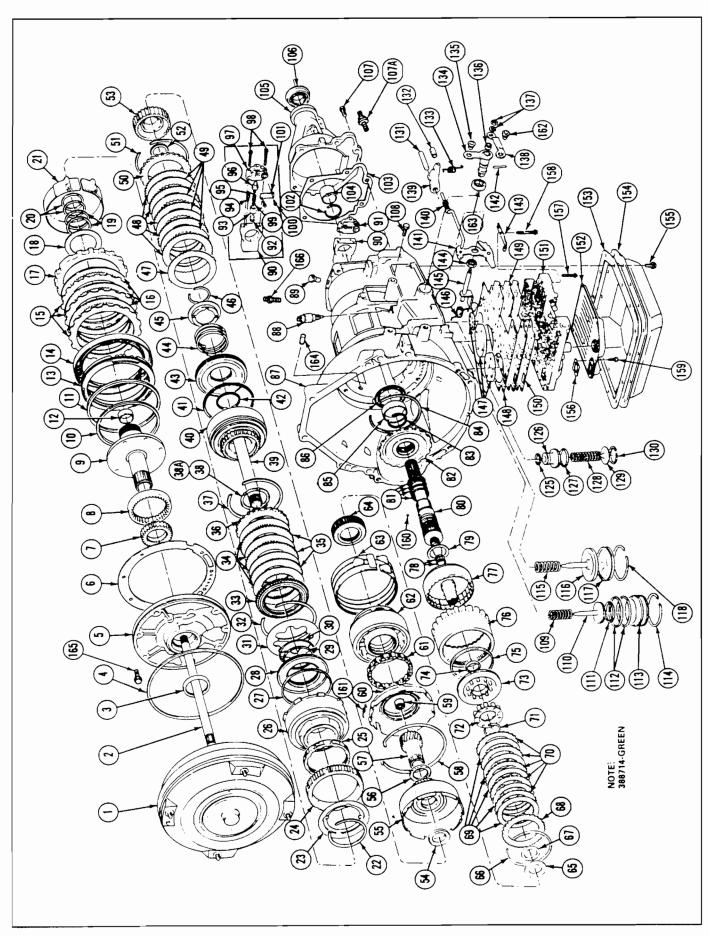
Automatic Overdrive Transmission—AOD—Exploded View—4x2 Vehicles





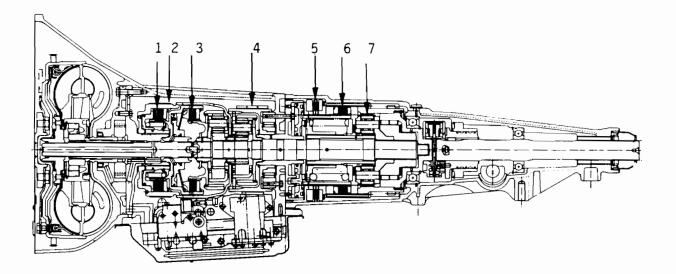
Automatic Overdrive Transmission-Nomenclature





Automatic Overdrive Transmission—AOD—Exploded View 4x4 Vehicles



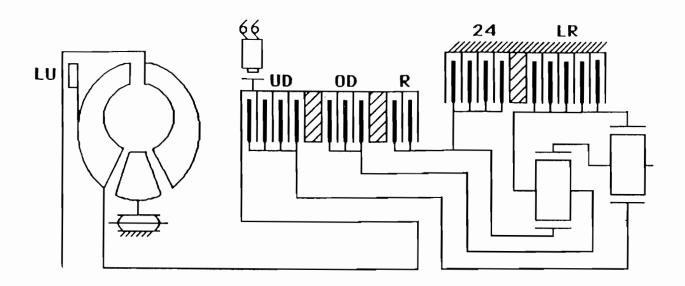


- 1. FRONT CLUTCH
- 2. KICKDOWN BAND
- 3. REAR CLUTCH
- 4. REVERSE BAND
- 5. OVERDRIVE CLUTCH
- 6. DIRECT CLUTCH
- 7. OVERRUN CLUTCH

	A 500			·			TRANSMIS	SSION			OVERDRIVE	
LEVER	OVER-	START	PARKING		CL	JTCHKS		В.	ANDS		CLUTCHES	
POSITION	DRIVE	SAFETY	SPRAG	FRONT	REAR	O'RUNNING	LOCKUP	K/D FRONT	REVERSE/REAR	o/p	O'RUNNING	DIRECT
P-Park		Х	Х									
R-Reverse	2.21			Х					X			Х
O-Drive												
First	2.74			1	Х	X				l	X	X
Second	1.54				Х			X			Х	X
Third	1.00	1		х	Х		X				X	X
O/D	.69			х	х		Х			Х		х
2-Second				_			-	_				
First	2.74				х	Х	i				x	x
Second	1.54				х			х			х	x
i-Low	2.74				Х	Х		-	х		Х	Х



CHRYSLER A-604



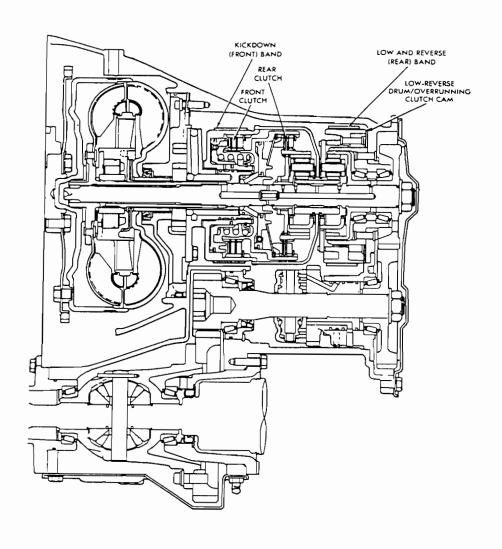
A604 4-SPEED EATX

UD=UNDERDRIVE CLUTCH
OD=OYERDRIVE CLUTCH
R =REYERSE CLUTCH
24=2-4 CLUTCH
LR=LOW REYERSE CLUTCH
LU=LOCKUP CLUTCH

GEAR	TORQUE	CLUTCHES
	RATIO	APPLIED
REYERSE	2.21	R,LR
NEUTRAL	[-	LR
FIRST	2.84	UD,LR
SECOND	1.57	UD,24
DIRECT	1.00	UD,OD
OYERDRIYE	0.69	0D,24



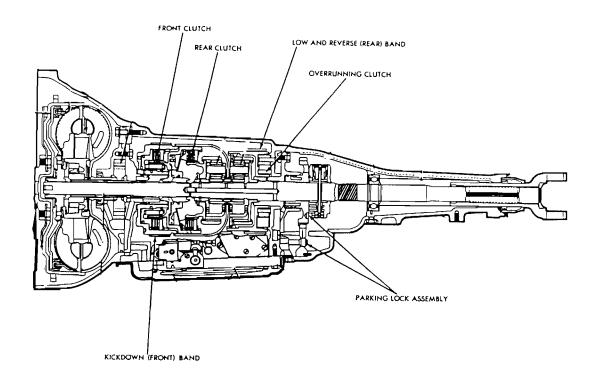
CHRYSLER A-670



RANGE	DIRECT CLUTCH	FORWARD CLUTCH	ROLLER CLUTCH	KICK-DOWN BAND	LOW-REV BAND
					
REV	ON				ON
D 1ST		ON	ON		
D 2ND		ON		ON	
D 3RD	ON	ON			
S 1ST		ON	ON		
S 2ND		ON		ON	
LOW		ON			ON

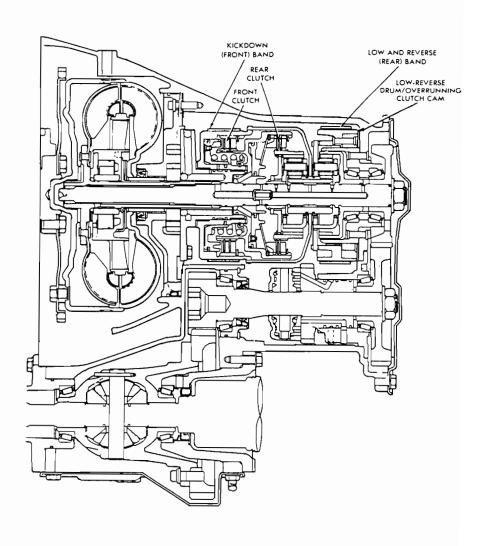
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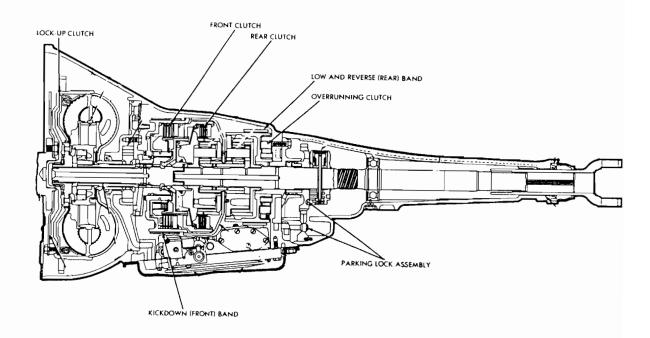
RANGE	DIRECT	FORWARD	ROLLER	KICK-DOWN	LOW-REV
	CLUTCH	CLUTCH	CLUTCH	BAND	BAND
REV D 1ST D 2ND D 3RD S 1ST S 2ND LOW	ON	ON ON ON ON ON	ON ON	ON ON	ON ON





		Clutches-		Bar	ıds
Lever Position	Front	Rear	Over- running	(Kickdown) Front	(Low-Rev.) Rear
P—PARK					
R-REVERSE	X				X
N-NEUTRAL					
D-DRIVE					
First		X	X		
Second		X		X	
Direct	X	X			
2—SECOND					
First		X	X		
Second		X		X	
1—LOW (First)		X			X

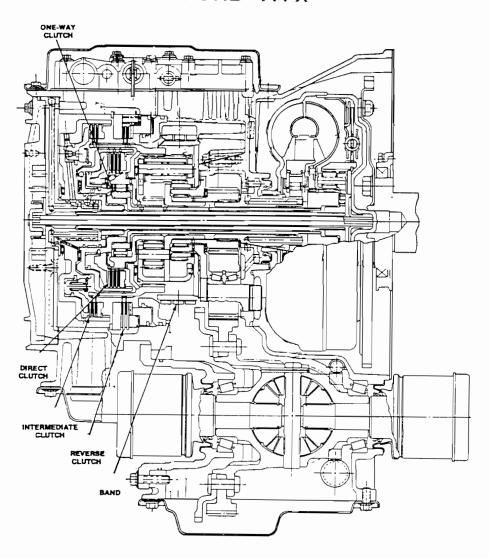




		_ Clu	utches	Bai	nds
Lever Position	Front	Rear	Over- running	(Kickdown) Front	(Low-Rev.) Rear
P—PARK					
R-REVERSE	Χ				Х
N-NEUTRAL					
D:—DRIVE First Second Direct	X	X X X	x	x	
2—SECOND First Second		X X	Х	X	
1—LOW (First)		X			Χ



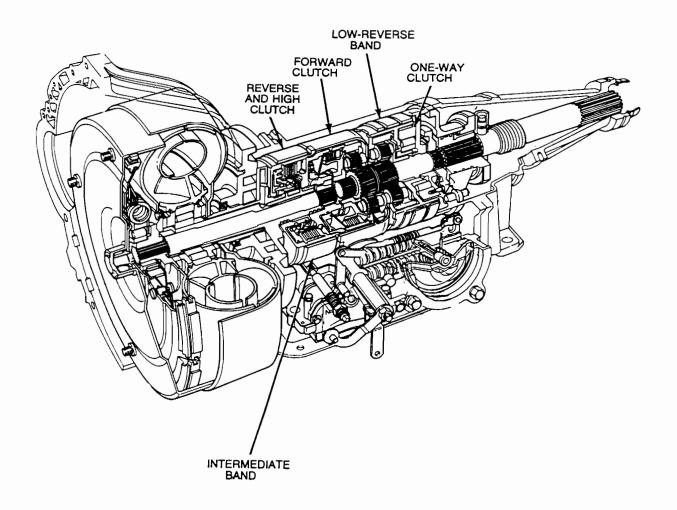
FORD ATX



Gear	Band	Direct Clutch	Intermediate Clutch	Reverse Clutch	Intermediate One-Way Clutch
1st Gear Manual Low	Applied	Applied			Holding
2nd Gear Manual Low	Applied		Applied		
1st Gear (Drive)	Applied				Holding
2nd Gear (Drive)	Applied		Applied		
3rd Gear (Drive)		Applied	Applied		
Reverse (R)		Applied		Applied	Holding
Neutral (N)					Holding
Park (P)					Holoing

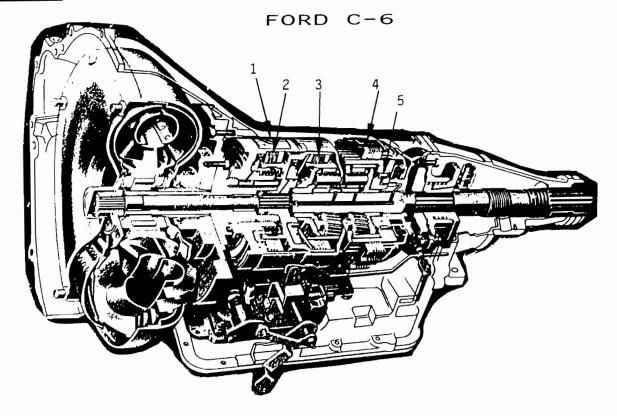


FORD C-5



Gear	Reverse and High Clutch	Forward Clutch	One-Way Clutch	Intermediate Band	Low-Reverse Band
1st (D Range)		Applied	Applied		
1st (1 Range)		Applied			Applied
2nd		Applied		Applied	
3rd	Applied	Applied			
Reverse	Applied				Applied



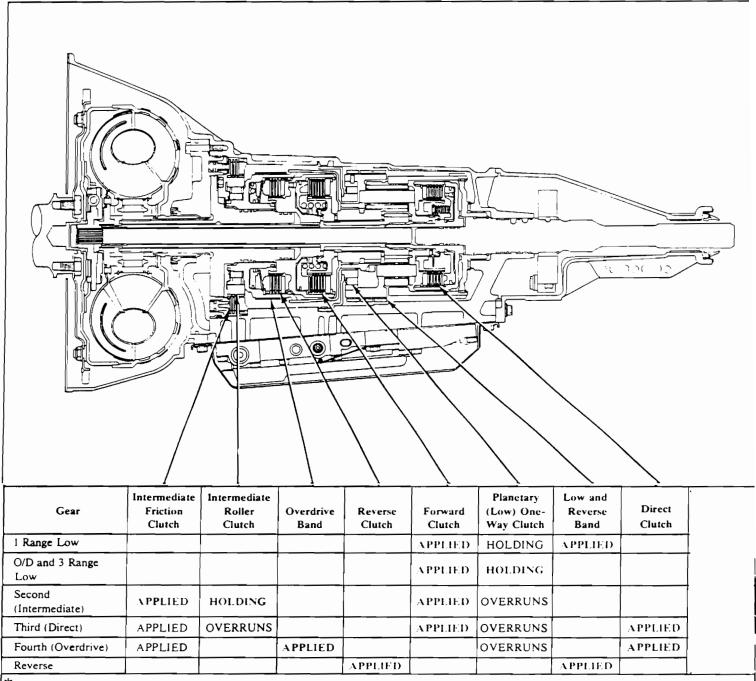


- 1. INTERMEDIATE BAND
- 2. REV/HIGH CLUTCH
- 3. FORWARD CLUTCH
- 4. LOW/REV CLUTCH
- 5. ONE-WAY CLUTCH

RANGE	FORWARD	REV/HIGH	LOW/REV	INTERMEDIATE	ONE-WAY
	CLUTCH	CLUTCH	CLUTCH	BAND	CLUTCH
D 1 D 2 D 3 LOW REV	ON ON ON	ON ON	ON ON	ON	HOLDING

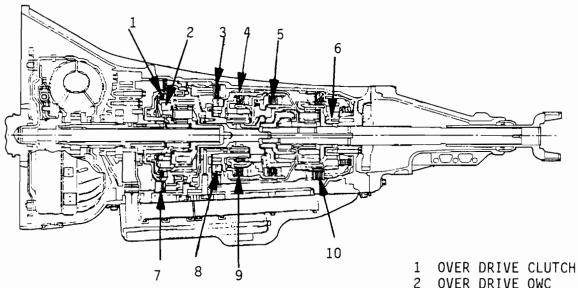


FORD AOD



^{*}Not including torque converter reduction in 1st. Second and Reverse.

FORD E40D



- OVER DRIVE OWC
- INTERMEDIATE CLUTCH
- BAND
- FORWARD CLUTCH
- LOW/REVERSE OWC
- COAST CLUTCH
- INTERMEDIATE OWC
- 9 DIRECT CLUTCH
- 10 REVERSE CLUTCH

											One-Wa	y Clutch		
				Fri	ction Eleme	ents				Drive			Coast	
(Gear	Coast	Inter- mediate	Direct	Forward	Reverse	Over- Drive	Band	O/D OWC	inter- mediate OWC	Low Reverse OWC	O/D OWC	inter- mediate OWC	Low Reverse OWC
		①	2	3	•	⑤	6	7	8	9	100	8	9	100
0	first	•			apply				hold		hold	o/r*		o/r
000	second	•	apply		apply				hold	hold	o/r	o/r*	o/r	o/r
(third	•	apply	apply	apply				hold	o/r	o/r	o/r*	o/r	o/r
(D)	fourth		apply	apply	apply		apply		o/r	o/r	o/r	o/r	o/r	o/r
	1	apply			apply	apply								
	2	apply	apply		apply			apply			o/r			o/r
Re	verse	apply		apply		apply				o/r			o/r	

O/D — Overdrive

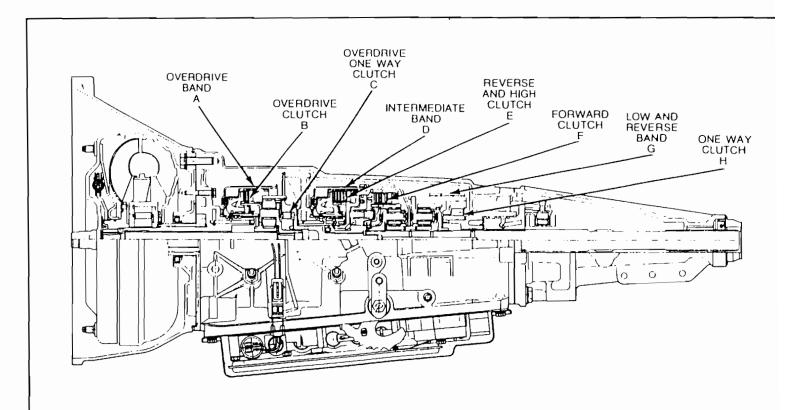
OWC - One-Way Clutch

O/R — Overrunning

In D Range with the Overdrive Cancel Switch pressed, the coast clutch is applied and the O/D one-way clutch is bypassed.



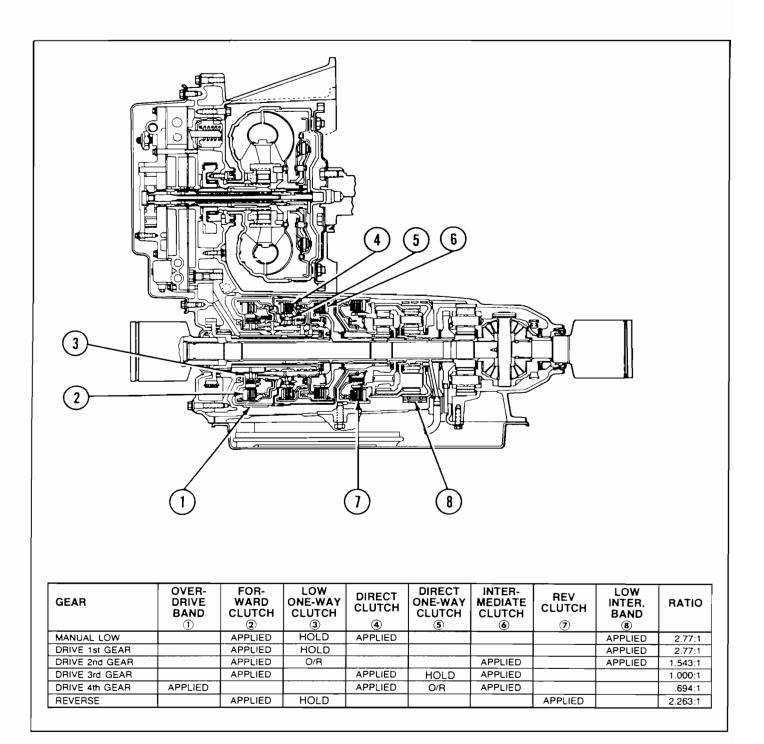
FORD A4LD



GEAR	OVER- DRIVE BAND A	OVER- DRIVE CLUTCH B	OVER- DRIVE ONE WAY CLUTCH C	INTERMEDIATE BAND D	REVERSE AND HIGH CLUTCH E	FORWARD CLUTCH F	LOW AND REVERSE BAND G	ONE WAY	GEAR RATIO
1 — MANUAL FIRST GEAR (LOW)		APPLIED	HOLDING			APPLIED	APPLIED	HOLDING	2.47:1
2 MANUAL SECOND GEAR		APPLIED	HOLDING	APPLIED		APPLIED			1.47:1
D - DRIVE AUTO 1ST. GEAR		APPLIED	HOLDING			APPLIED		HOLDING	2.47:1
D- 0/D AUTO 1ST. GEAR			HOLDING			APPLIED		HOLDING	2.47:1
D — DRIVE AUTO. — 2ND. GEAR		APPLIED	HOLDING	APPLIED		APPLIED			1.47:1
(D) O/D AUTO 2ND. GEAR			HOLDING	APPLIED		APPLIED			1.47:1
D DRIVE AUTO 3RD. GEAR		APPLIED	HOLDING		APPLIED	APPLIED			1.0:1
D- 0/D AUTO 3RD. GEAR			HOLDING		APPLIED	APPLIED			1.0:1
D-OVERDRIVE AUTOMATIC FOURTH GEAR	APPLIED				APPLIED	APPLIED			0.75:1
REVERSE		APPLIED	HOLDING		APPLIED		APPLIED		2.1:1

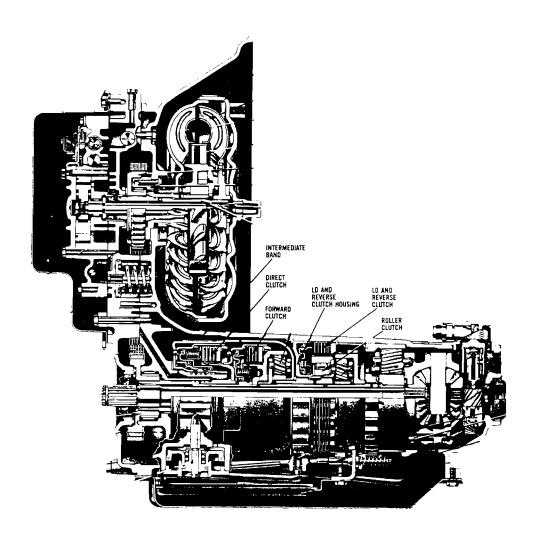


FORD AXOD





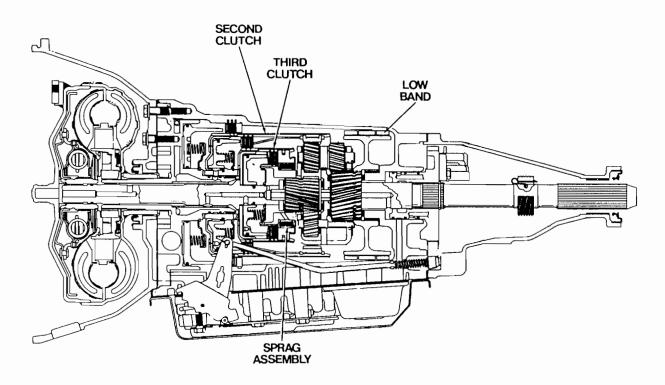
GM 125-C



RANGE REFERENCE CHART											
RANGE GEAR DIRECT INTERMEDIATE FORWARD ROLLER CLUTCH											
PARK - NEUT.											
	FIRST			APPLIED	HOLDING						
DRIVE	SECOND		APPLIED	APPLIED							
	THIRD	APPLIED		APPLIED							
INTERMEDIATE	FIRST			APPLIED	HOLDING						
INTERMEDIATE	SECDND		APPLIED	APPLIED							
LO	FIRST			APPLIED	HOLDING	APPLIED					
REVERSE		APPLIED				APPLIED					

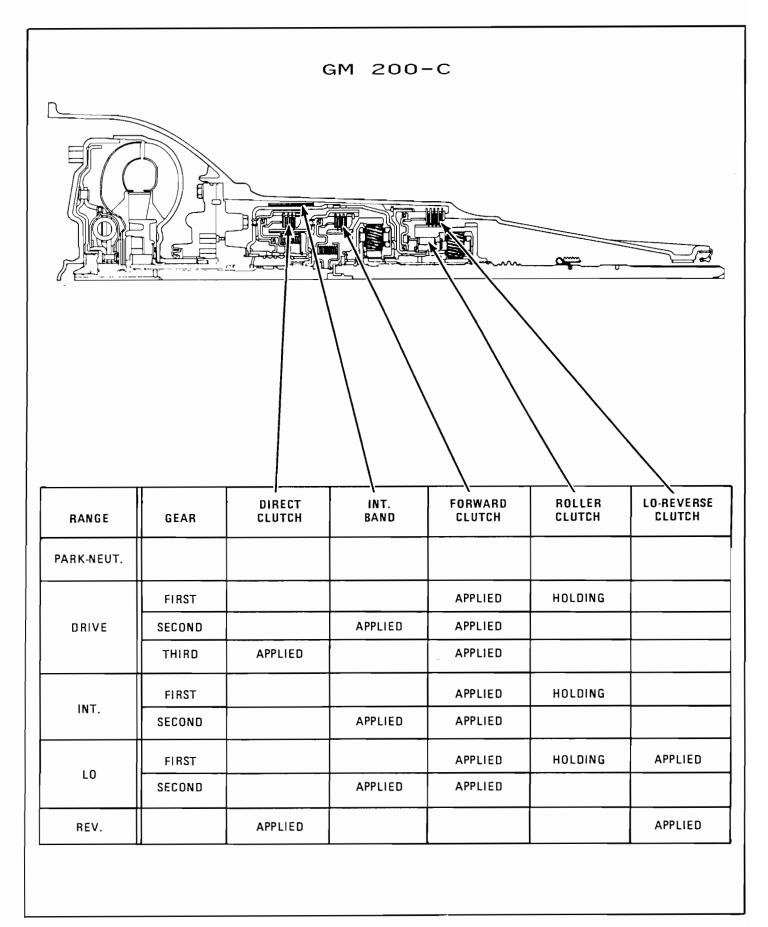


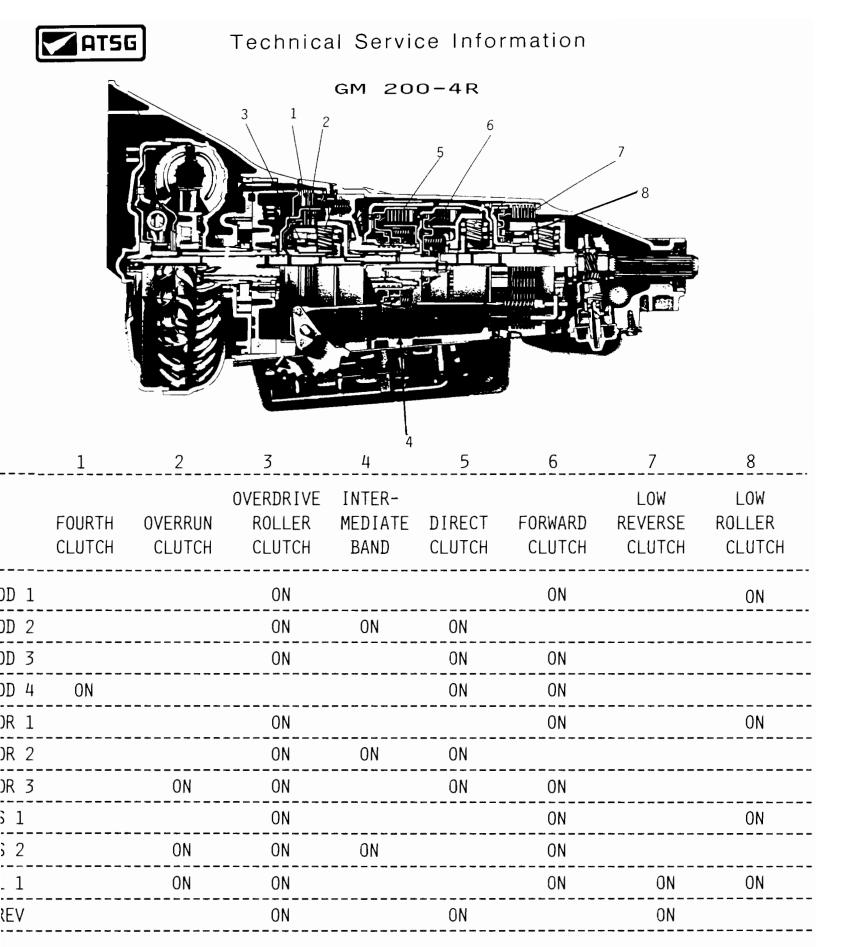
GM 180-C



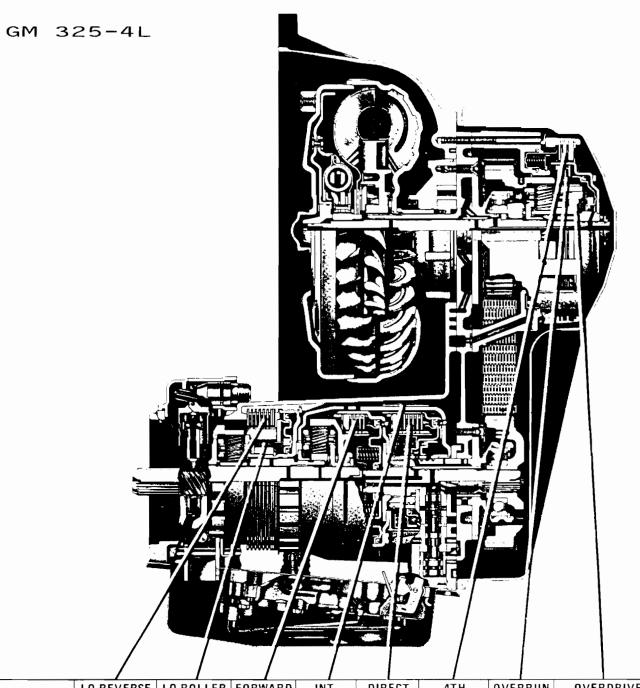
		REVERSE CLUTCH	SECOND CLUTCH	CLUTCH	LOW BAND	SPRAG
A.	NEUTRAL/PARK	Released	Released	Released	Released	Locked
В.	DRIVE RANGE, First Gear	Released	Released	Released	Applied	Locked
C.	L ₁ RANGE	Released	Released	Applied	Applied	Locked
D.	DRIVE RANGE, Second Gear	Released	Applied	Released	Applied	Overrunning
E.	L ₂ RANGE	Released	Applied	Released	Applied	Overrunning
F.	DRIVE RANGE, Third Gear	Released	Applied	Applied	Released	Locked
G.	REVERSE RANGE	Applied	Released	Applied	Released	Locked







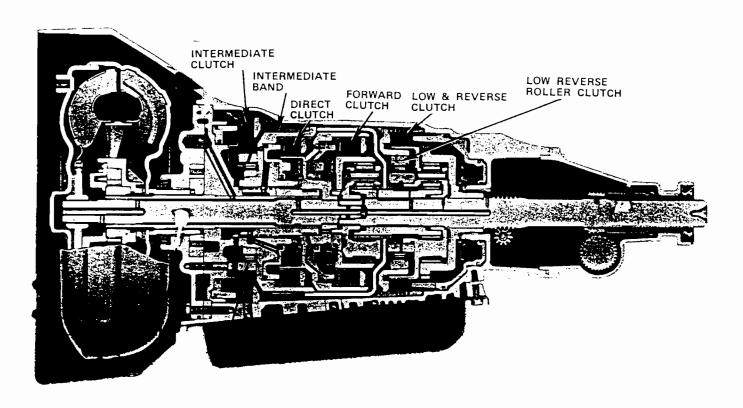




RANGE	E/GEAR	LO-REVERSE CLUTCH	LO-ROLLER CLUTCH	FORWARD CLUTCH	INT. Band	DIRECT CLUTCH	4TH CLUTCH	OVERRUN CLUTCH	OVERDRIVE ROLLER CLUTCH
P.A	ARK								
REV	/ERSE	APPLIED				APPLIED			HOLDING
NEU	TRAL								HOLDING
DRIVE 4	1ST		HOLDING	APPLIED					, 1.
	2ND		·	APPLIED	APPLIED				HOLDING
	3RD			APPLIED		APPLIED			HOLDING
Ī	4TH			APPLIED		APPLIED	APPLIED		
DRIVE 3	1ST		HOLDING	APPLIED				APPLIED	
Ì	2ND			APPLIED	APPLIED			APPLIED	
Ī	3RD			APPLIED		APPLIED		APPLIED	
DRIVE 2	1ST		HOLDING	APPLIED				APPLIED	
	2ND			APPLIED	APPLIED			APPLIED	
LO	1ST	APPLIED		APPLIED				APPLIED	

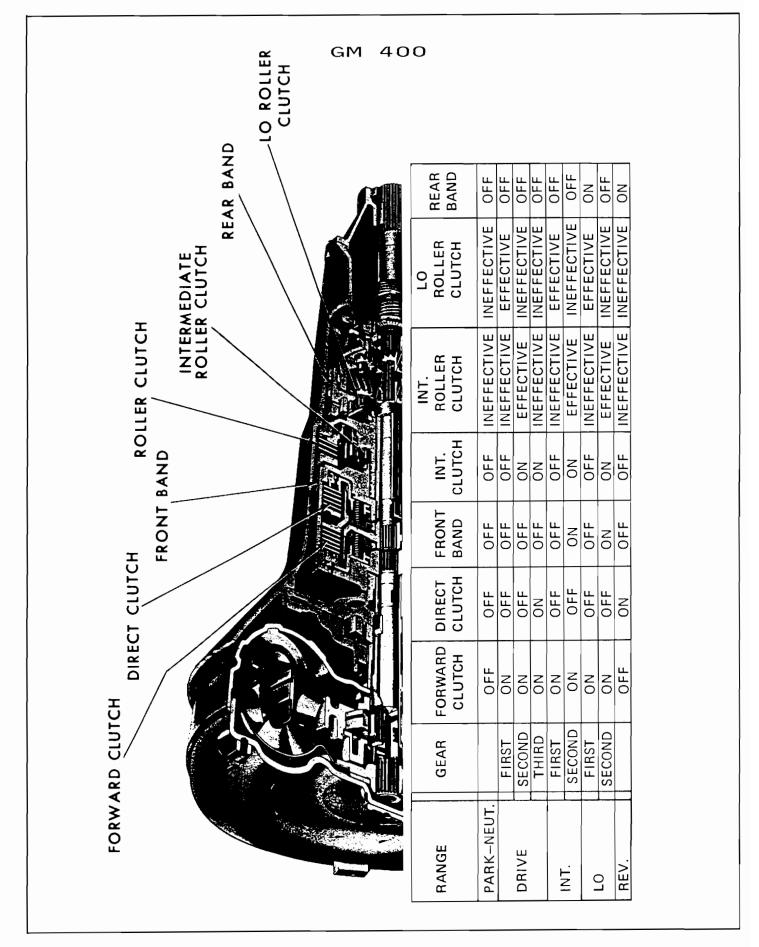


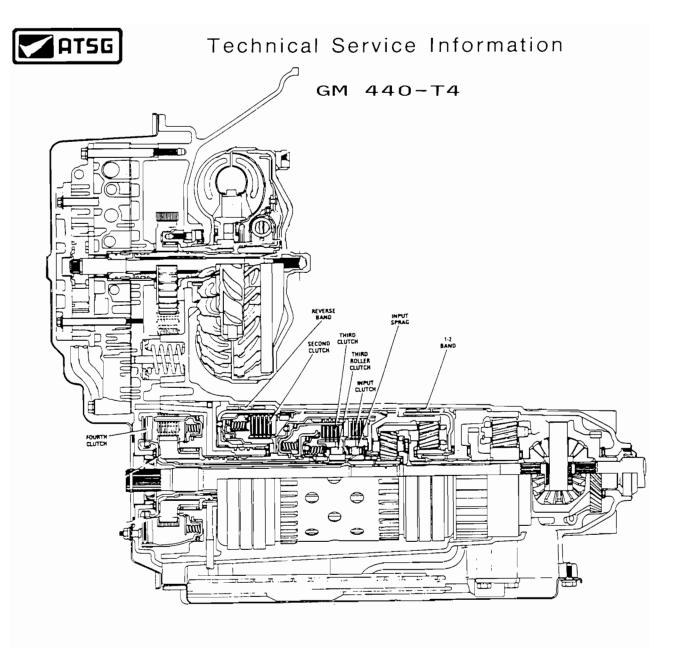
GM 350-C



RANGE	LOW ONE-WAY CLUTCH	FORWARD CLUTCH	REVERSE HIGH CLUTCH	INTERMED CLUTCH	INTERMED BAND	LOW REVERSE CLUTCH	INTERMEAD ONE-WAY CLUTCH
DR 1	ON	ON					
DR 2		ON		ON			ON
DR 3		ON	ON	ON			
LOW 2		ON		ON	ON		ON
LOW 1	ON	ON				ON	
REV			ON			ON	

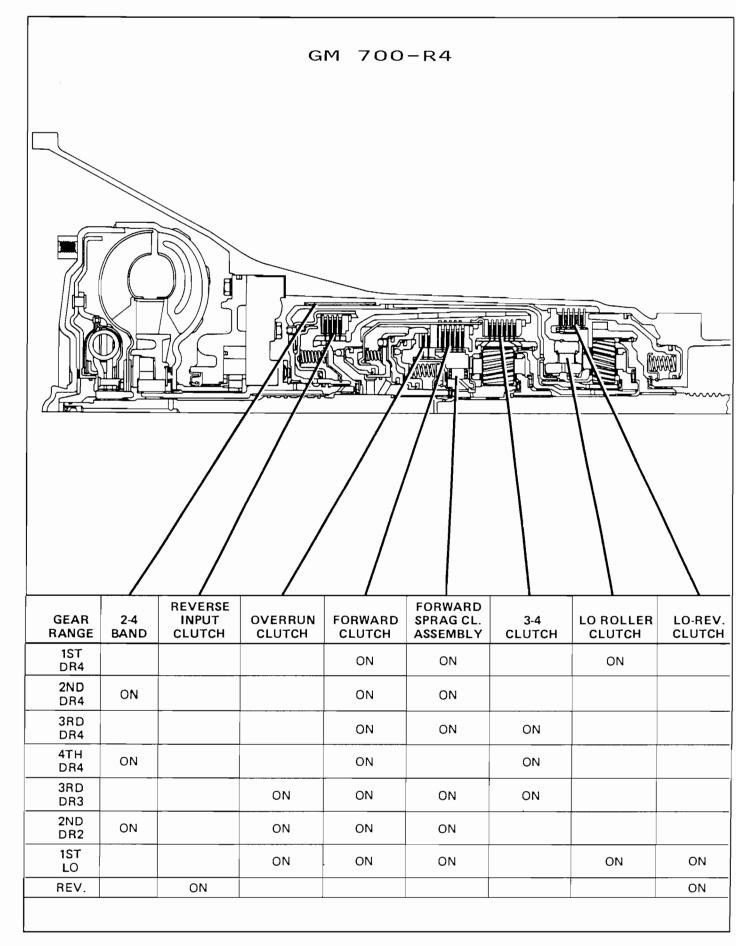






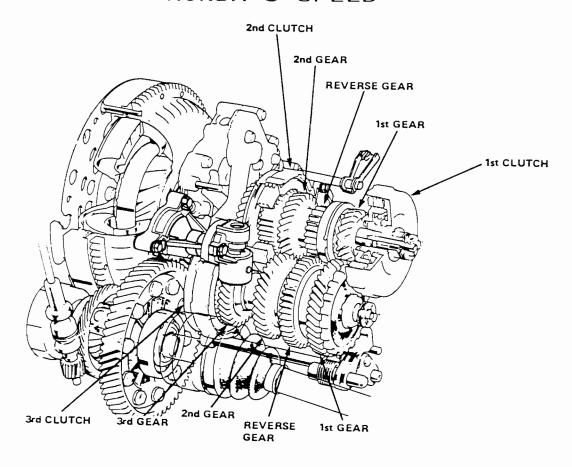
RANGE		4TH CLUTCH	REVERSE BAND	2ND CLUTCH	3RD CLUTCH	3RD ROLLER CLUTCH	INPUT	INPUT SPRAG	1-2 BAND
NEUTRAL PARK							*	*	
DRIVE	1						00	HOLD	ON
	2			ON			*	OVER'	ON
	3			ON	ON	HOLD	OFF		
	4	ON		ОИ	*	OVER			
MANUAL	3			00	ON	HOLD	ON	HOLD	
	2			00			*	OVER- RUNNING	ON
	1				07	HOLD	ON	HOLD	ON
REVERSE			011				ON	HOLD	







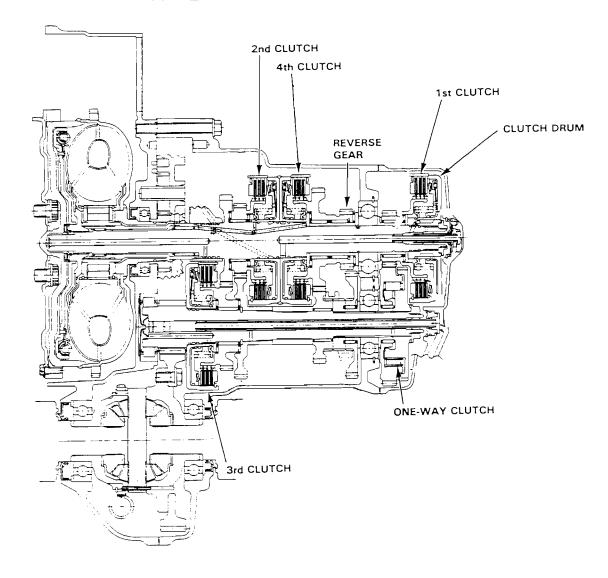
HONDA 3 SPEED



GEAR	1ST CLUTCH	2ND CLUTCH	3RD CLUTCH	SERV0
PARK/NEUTRAL				
DRIVE 1	APPLIED			
DRIVE 2		APPLIED		
DRIVE 3			APPLIED	
REVERSE		APPLIED		APPLIED



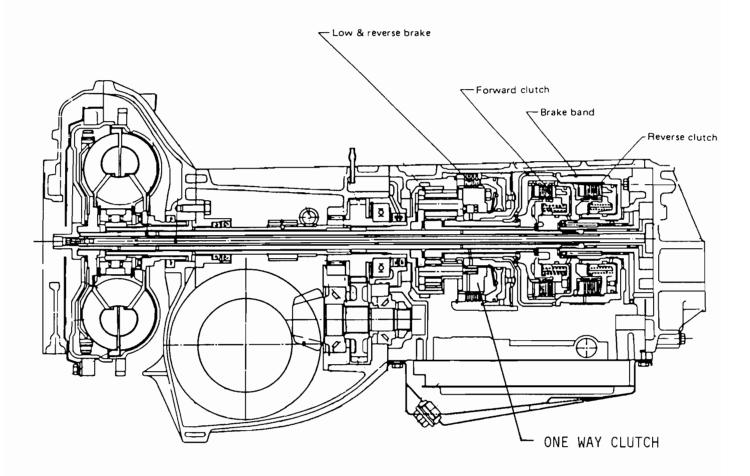
HONDA 4 SPEED



1ST CLUTCH	2ND CLUTCH	3RD CLUTCH	4TH CLUTCH	SERV0
APPLIED				
	APPLIED			
		APPLIED		
			APPLIED	
	APPLIED			
			APPLIED	APPLIED
	CLUTCH	CLUTCH CLUTCH APPLIED APPLIED	CLUTCH CLUTCH APPLIED APPLIED APPLIED	CLUTCH CLUTCH CLUTCH APPLIED APPLIED APPLIED APPLIED APPLIED

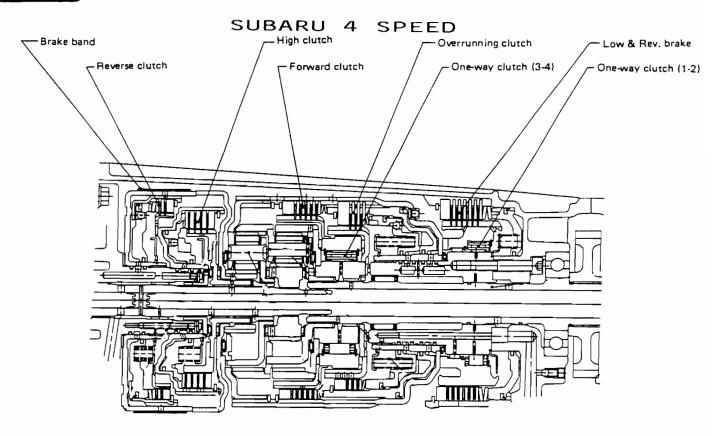


SUBARU 3 SPEED



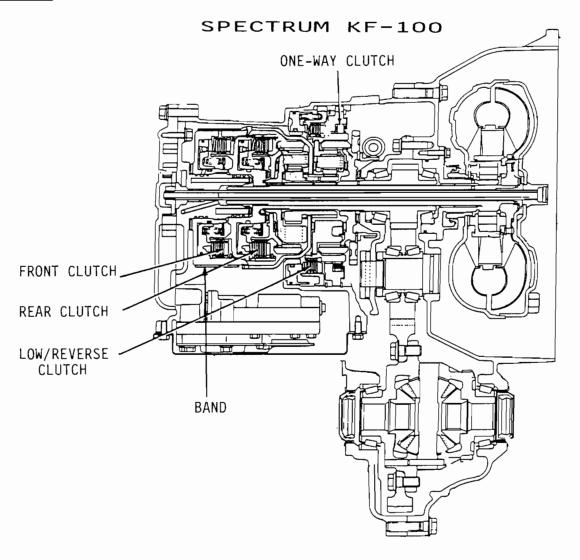
Panga	Gear	Clu	tch	Low & Re- verse brake	Brake band	One-way
Range	Gear	Forward C ₁	Reverse C ₂	B ₁	B ₂	clutch C ₃
Р	Parking			0		
R	Reverse		0	0		
N	Neutral					
	1st	0				0
D	2nd	0			0	
	3rd	0	0			
2	2nd (locked)	0			0	
1	2nd	0			0	
	1st	0		0		





•1: F	har	itchbut indling	ion		Selector lever operation									7			
Only w	-				(2)			(a)			(0	•)		(z)	(R		
when se	38(<u>)</u> į	ä() N =	151	3 A D	2ND	157	4TH	3RD	2ND	1ST				
Only when selector lev For prevention of over-r Engine brake ineffective		١٠.	:		:										0		Rev./C
Only when selector lever is in "2". For prevention of over-revolution Engine brake ineffective		0			0			0		0		0					8/8
	0			0			0			0	0						Häh/C
and 1st hold button is ON.	0	0	0	0	0	0	0	0	0	0	0	0	0				FWD/C
button i	0	0	0	0	0	0	0	0	0		0	0	0				0WC
ON.	0	0	0	0	0	0	0	0	0								OVR/C
			O			O									0		Lo/ Rev./B
									O.				0				OWC (1-2)

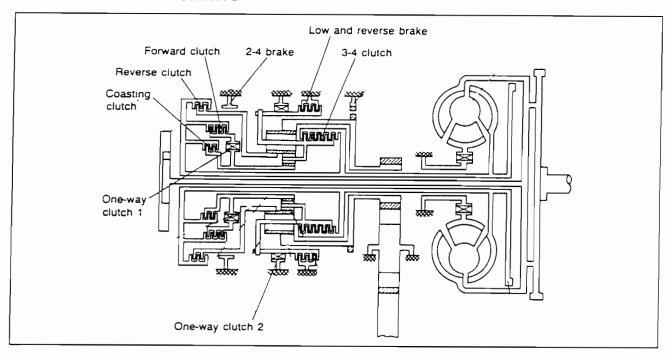




s	Shift	Clu	tch	Low and Reverse	Band S	Servo	One-way
	sition	Front	Rear	Brake	Operation	Release	Clutch
	Р			0	, , , , , , , , , , , , , , , , , , ,		
	R	0		o	-	x	
	N					_	
	1st		0				0
D	2nd		0		0		
	3rd	o	0		x	x	
	2nd		0		0		
1	2nd		0		0		
	1st		0	0			

MAZDA 4 SPEED

OPERATION OF COMPONENTS



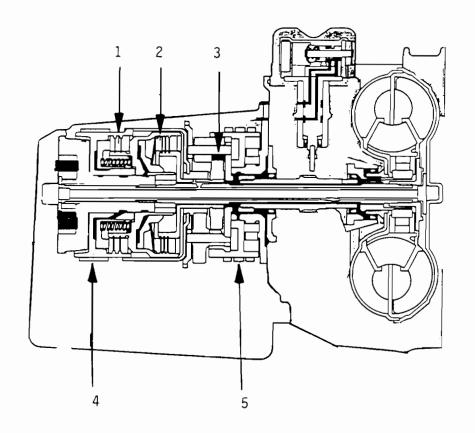
Operation Table (G4A-HL)

					C	peration	elemen	ts			
Range	Gear	Engine	Forward	Coasting	3-4	Reverse		orake	Low &	One-way	One-way
		effect	clutch	clutch	clutch	clutch	Applied Released		reverse brake	clutch 1	clutch 2
Р	_	_									
R	_	Yes				0			0		
N	_							_			
	1st	No	0							0	0
D	2nd	No	0				0			0	
	3rd	Yes	0	0	0		⊗	0		0	
	OD	Yes	0		0		0				
2	2nd	Yes	0	0			0	_		0	
1	1st	Yes	0	0				0	0	0	
	2nd	Yes	0	0			0			0	

- 10 : Indicates fluid pressure to servo but band not applied due to pressure difference in servo.
- © : Indicates that it does not function to transmit power.



VOKSWAGON 003 / 010



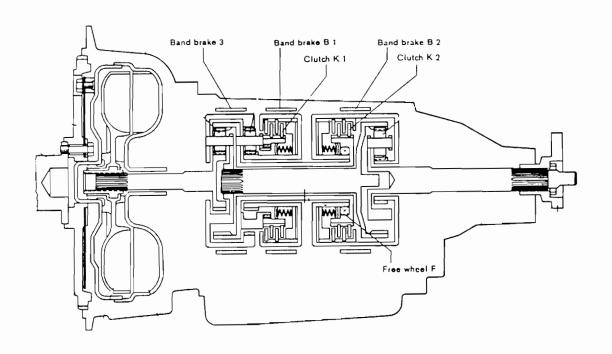
- 1 DIRECT/REVERSE CLUTCH
- 2 FORWARD CLUTCH
- 3 ONE-WAY CLUTCH
- 4 INTERMEDIATE BRAKE
- 5 1ST / REVERSE BRAKE

RANGE	LOW/REV BAND	INT. BAND	FORARD CLUTCH	DIRECT CLUTCH	ONE-WAY CLUTCH
D 1ST D 2ND		ON	ON ON		HOLDING
D 3RD S 1ST		011	ON ON	ON	HOLDING
S 2ND	ON	ON	ON		HOLDING
LOW REV	ON O N		ON	ON	

NOTE: THE LOW/REVERSE BAND HAS BEEN ELIMINATED AND A CLUTCH SUBSTITUTED IN THE 010 SERIES.



MERCEDES 722.1 / 722.2



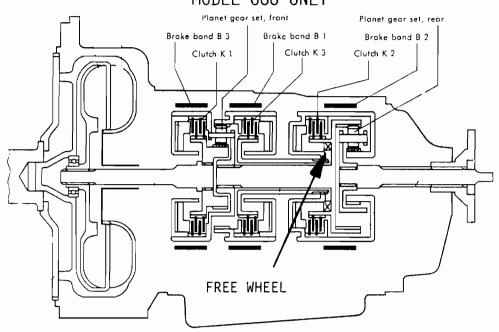
Gear	B 1	K 1	B 2	K 2	В 3	F
1			×			×
2	x		×			
3		x	x			
4	i 1	x		x		
R					×	×



MERCEDES DB TRANSMISSION

SOLENOID CONTROL

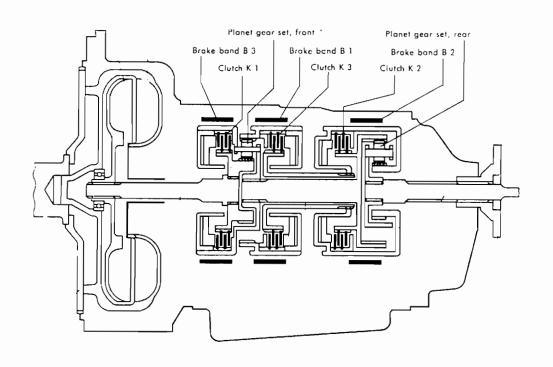
MODEL 600 ONLY



				 -			
RANGE	K1 CLUTCH	K2 CLUTCH	K3 CLUTCH	B1 BAND	B2 BAND	B3 BAND	FREE WHEEL
1ST				ON	ON		
2ND		ON		ON			
3RD	ON				ON		
4TH	ON	ON					
REV						ON	HOLDING
			 -		- 		



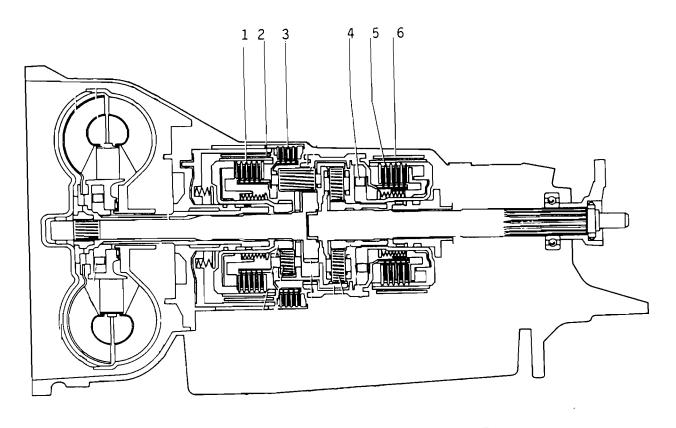
MERCEDES DB TRANSMISSION



RANGE	K1 CLUTCH	K2 CLUTCH	K3 CLUTCH	B1 BAND	B2 BAND	B3 BAND
1ST 2ND		ON		ON ON	ON	
3RD	ON	• • • • • • • • • • • • • • • • • • • •		0,1	ON	
4TH	ON	ON				
REV			ON			ON



MERCEDES 722.3 / 722..4



1 CLUTCH K-1

4 ONE WAY CLUTCH F

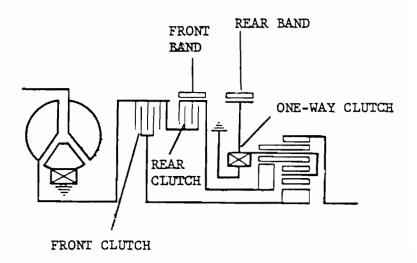
2 BAND B-1 5 CLUTCH K-2 3 BRAKE B-3 6 BAND B-2

Speed	B 1	B 2	В 3	K 1	K 2	F	Reduction
1		X			(X)	X	3 68
2	X	X					2 4 1
3		Х		X			1 44
4				×	X		î
R			X		(X)	×	5 14

(X) K 2 bridges the one-way clutch during deceleration (coasting)



BW 65-66

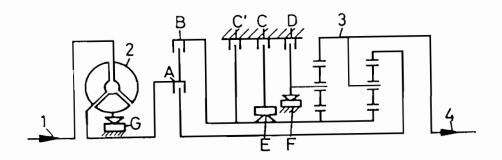


	FC	RC	FB	RB	1-way clutch
l (first gear)	•			•	
D (first gear)	•				•
2 & D (sec. gear)	•		•		
D (third gear)	•	•			
R (reverse gear)		•		•	
*P (park)				•	

^{*} When engine running

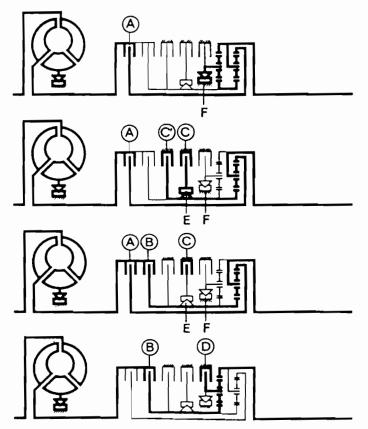


POWER FLOW 3HP22



Layout of Transmission 3 HP-22:

- 1 Input
- 2 Torque converter
- 3 Planet gear set
- 4 Output



Power Flow Diagrams for 3 HP-22

1st Gear

Clutch A is engaged.

Planet gear carrier bears on one-way clutch F during acceleration and is cancelled while coasting. With selector lever in position 1 clutch D also engages in 1st gear, so that engine braking force can be utilized.

2nd Gear

Clutches A, C' and C are engaged.
One-way clutch F is cancelled.
Hollow shaft is fixed with sun gear.

3rd Gear

Clutches A, B and C are engaged.

One-way clutches E and F are cancelled.

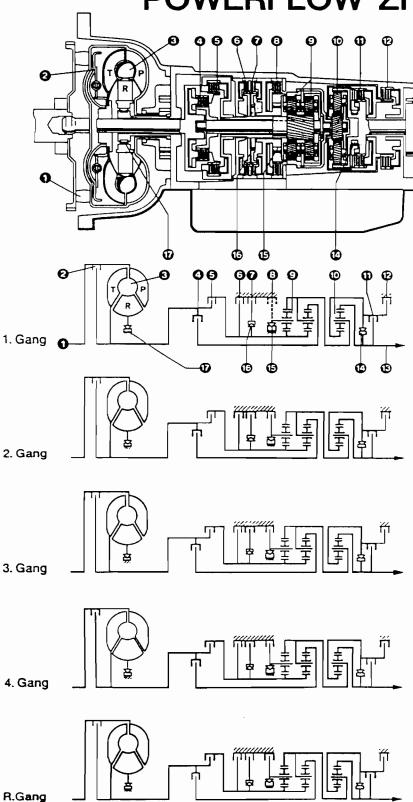
The entire set of planet gears turns as an unit at a ratio of 1:1.

Reverse Gear

Clutches B and D are engaged.
The output shaft's direction of rotation is reversed by way of the locked planet gear carrier. Power flow is 1st, 2nd, 3rd and reverse gears by way of emphasized parts.







Clutches 4 and 11 are engaged. The front planet gear carrier of gear set 9 is locked against the housing through freewheel 15 when the engine is pulling, but is overrun when the engine is coasting. Epicyclic gear set 10 rotates as a solid block, with the front planet gear carrier. In addition, in selector lever position 1 and in speed range 1, clutch 8 is engaged to permit engine braking.

Clutches 4,6,7 and 11 are engaged. Freewheel 15 overruns. The hollow shaft with the sun wheel of epicyclic gear set 9 is locked. Epicyclic gear set 10 also rotates, as a solid block.

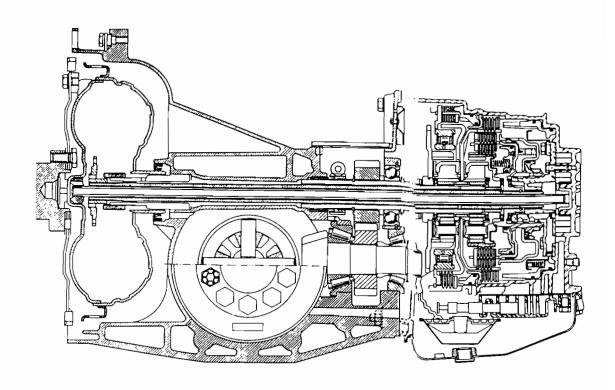
Clutches 4,5,7 and 11 are engaged. Freewheels 15 and 16 are overrun. Epicyclic gear sets 9 and 10 rotate as a solid block at a ratio of 1:1.

Clutches 4,5,7 and 12 are engaged. Freewheels 14, 15 and 16 are overrun. Epicyclic gear set 9 rotates as a solid block. The hollow shaft with the sun wheel of epicyclic gear set 10 is locked. Above a predetermined road speed, clutch 2 locks torque converter 3 solid to prevent slip.

Clutches 5,8 and 11 are engaged. Since the front planet gear carrier of epicyclic gear set 9 is locked, the direction of output-shaft rotation is reversed. Epicyclic gear set 10 also rotates, as a solid block.



RENAULT MB1-MJ3



lev	ector ver ition	RL	E1	E2	F1	F2	EL1	EL2
	Р							
F	۹			X	\times			X
?	N							\times
	1	X	X					X
Α	2		\times			\times	\times	\times
	3		X	X				
2nd H	OLD		X			\times	X	\times
1st HO	OLD		X		X			\times

RL = Freewheel

E1 = Clutch 1

E2 = Clutch 2 F1 = Brake 1

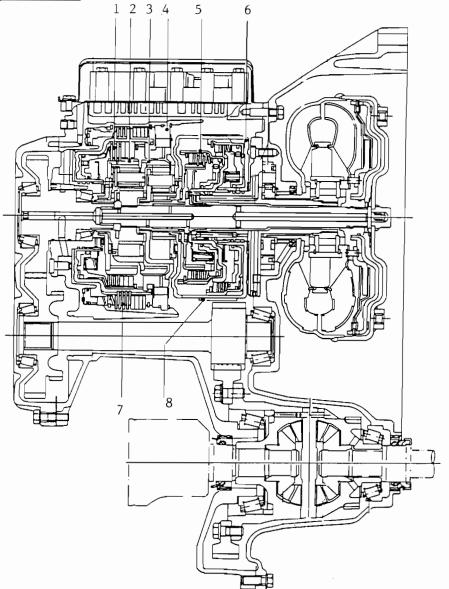
F2 = Brake 2

EL1 = Solenoid valve 1

EL2 = Solenoid valve 2

AUTOMATIC TRANSMISSION SERVICE GROUP





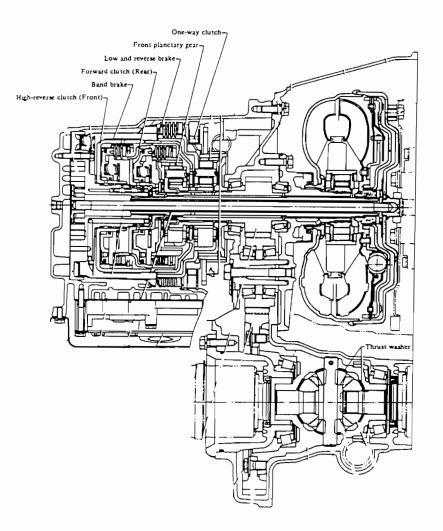
NISSAN RL4F02A

- 1 LOW CLUTCH
- 2 REAR PLANETARY
- 3 FRONT PLANETARY
- 4 ONE-WAY CLUTCH
- 5 HIGH CLUTCH
- 6 REVERSE CLUTCH
- 7 LOW/REVERSE BRAKE
- 8 BAND BRAKE

RL4F02A

	P		Gear	Reverse	High	Low	Band	servo	Low &	One-way	Parking	l sale is
	Rang	ge .	ratio	clutch	clutch	clutch	Operation	Release	reverse brake	clutch	pawl	Lock-up
Park			-								on	
Reverse			2.272	on					on			
Neutral			1			1						
	D,	Low	2.785			00				on		
Drive	D,	Second	1.545			on	on					
Dilve	D,	Top (3rd)	1.000		on	an	(00)	00				
	D.	O.D. (4th)	0.694		on		on					on
2	2,	Low	2.785			on				on		
	2,	Second	1.545			on	on					
,	1,	Low	2 785			on			on	on		
•	1,	Second	1.545			on	on					



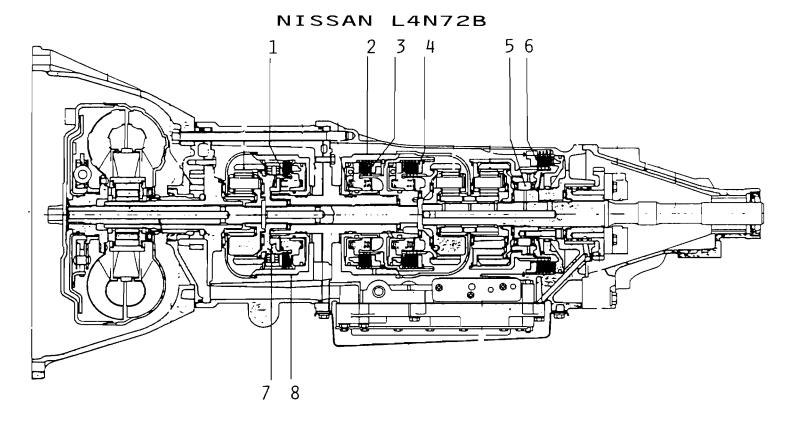


NISSAN RL3F01A

			Cit	utch	Low &		Band	servo	One-	
	Range	Gear ratio	High-reverse clutch (Front)	Forward clutch (Rear)	reverse	Lock-up	Operation	Release	way	Parking pawl
Park										on
Rever	se	2.364	on		on					
Neutr	al									
	D ₁ Low	2.826		on					on	
Drive	D ₂ Second	1.543		on			on			
	D ₃ Top (3rd)	1.000	on	on		on	(on)	on		
2	2 ₁ Low	2.826		on					on	
2	2 ₂ Second	1.543		on			on			
1	1 ₁ Low	2.826		on	on				on	
'	1 ₂ Second	1.543		on			on			

The low & reverse brake is applied in "1," range to prevent free wheeling when coasting and allows engine braking.





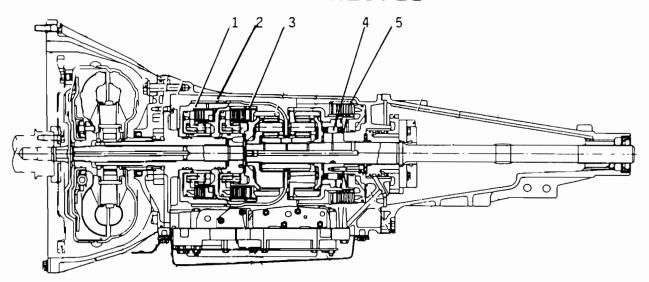
	D	Direct	O.D. ba	and servo	High- reverse	Forward	Low &	2nd ba	nd servo	One-way	.
	Range	clutch	Apply	Release	clutch (Front)	clutch (Rear)	reverse brake	Арріу	Release	clutch	pawl
Par	k	ON	(ON)	ON			ON				ON
Rev	/erse	ON	(ON)	ON	ON		ON		OŃ		
Net	ıtral	ON	(ON)	ON							
	D ₁ (Low)	ON	(ON)	ON		ON				ON	
۵	D ₂ (Second)	ON	(ON)	ON		ON		ON			
	D ₃ (Top)	ON	(ON)	ON	ON	ON		(ON)	ON		
	D ₄ (O.D.)		ON		ON	ON		(ON)	ON		-
2	Second	ON	(ON)	ON		ON		ON			
,	1 ₂ (Second)	ON	(ON)	ON .		ON	_	ON			
	1 ₁ (Low)	ON	(ON)	ON		ON	ON			ON	

The low & reverse brake is applied in "11" range to prevent free wheeling when coasting and allows engine braking.

1 DIRECT CLUTCH
2 2ND BAND
3 FORWARD CLUTH
4 REAR CLUTCH
5 ONE-WAY CLUTCH
6 LOW/REVERSE CLUTCH
7 O/D ONE WAY CLUTCH
8 O/D BAND



NISSAN RL371B



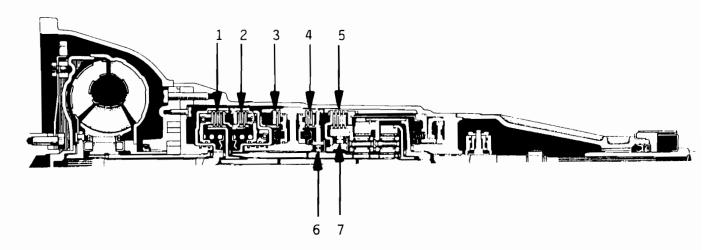
- 1 HIGH/REVERSE CLUTCH
- 2 BAND BRAKE
- 3 FORWARD CLUTCH
- 4 ONE-WAY CLUTCH
- 5 LOW/REVERSE BAND

				Clu	tch	Low &		Band	servo	One	
	Rang	e	Gear ratio	High- reverse (Front)	Forward (Rear)	reverse brake	Lock-up	Operation		way clutch	Parking pawl
Park						on					on
Reverse	e		2.182	on		on			on		
Neutral	l										
	D1	Low	2.458		on					on	
Drive	D2	Second	1.458		on			on			
	D3	Тор	1.000	on	on		on	(on)	on		
2		Second	1.458		on			on			
1	12	Second	1.458		on			on			
	11	Low	2.458		on	on					

The low & reverse brake is applied in "1₁" range to prevent free wheeling when coasting and allows engine braking.



TOYOTA A-40



Shift Lever	Gear	C ₁	С	2	В1	B ₂	В	3	F ₁	F ₂
Position	Gear		I.P.	O.P.	01	52	I.P.	O.P.	' 1	12
Р	Neutral							0		
R	Reverse		0	0			0	0		
N	Neutral									
D & 2	First	0								0
D	Second	0				0			0	
D	Third	0	0			0				
2	Second	0			Δ	0			0	
L	First	0					△ only for 4M	Δ		0

 $1 \quad C_1 : Front clutch$

2 C₂ : Rear clutch

3 B₁ : Brake No.14 B₂ : Brake No.2

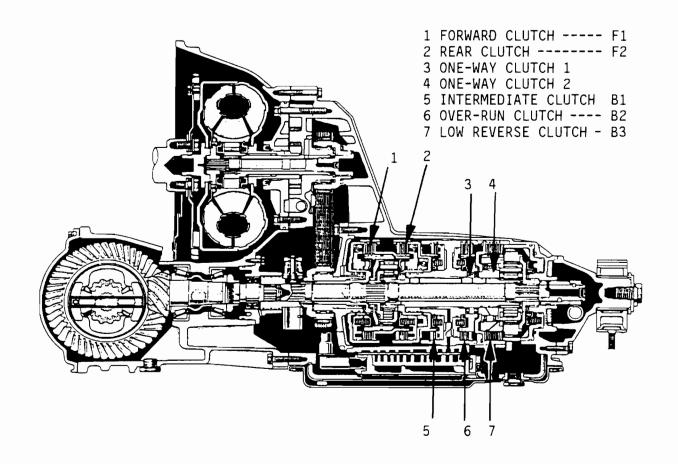
5 B₃ : Brake No.3

6 F₁: One-way clutch No.17 F₂: One-way clutch No.2

O.P.: Outer piston I.P.: Inner piston

Note: \triangle indicates operation when braking by engine

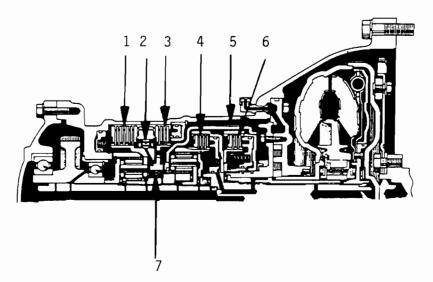
TOYOTA A-55



FORWARD CLUTCH	REAR CLUTCH	ONE-WAY CLUTCH-1	ONE-WAY CLUTCH-2	OVER-RUN CLUTCH	INTERMED CLUTCH	LOW-REV CLUTCH
	ON					ON
ON			HOLD			
ON		HOLD		ON		
ON	ON			ON		
ON			HOLD			
ON			HOLD	ON	ON	
ON			HOLD			ON
	ON ON ON ON ON	CLUTCH CLUTCH ON ON ON ON ON ON ON	CLUTCH CLUTCH CLUTCH-1 ON ON ON ON ON ON ON ON	CLUTCH CLUTCH-1 CLUTCH-2 ON ON HOLD ON ON ON HOLD ON HOLD ON HOLD	CLUTCH CLUTCH-1 CLUTCH-2 CLUTCH ON ON HOLD ON ON ON ON ON HOLD ON HOLD ON HOLD ON ON HOLD ON	CLUTCH CLUTCH-1 CLUTCH-2 CLUTCH CLUTCH ON ON HOLD ON ON ON ON ON HOLD ON HOLD ON HOLD ON ON HOLD ON ON ON HOLD ON ON ON ON ON ON ON ON ON O



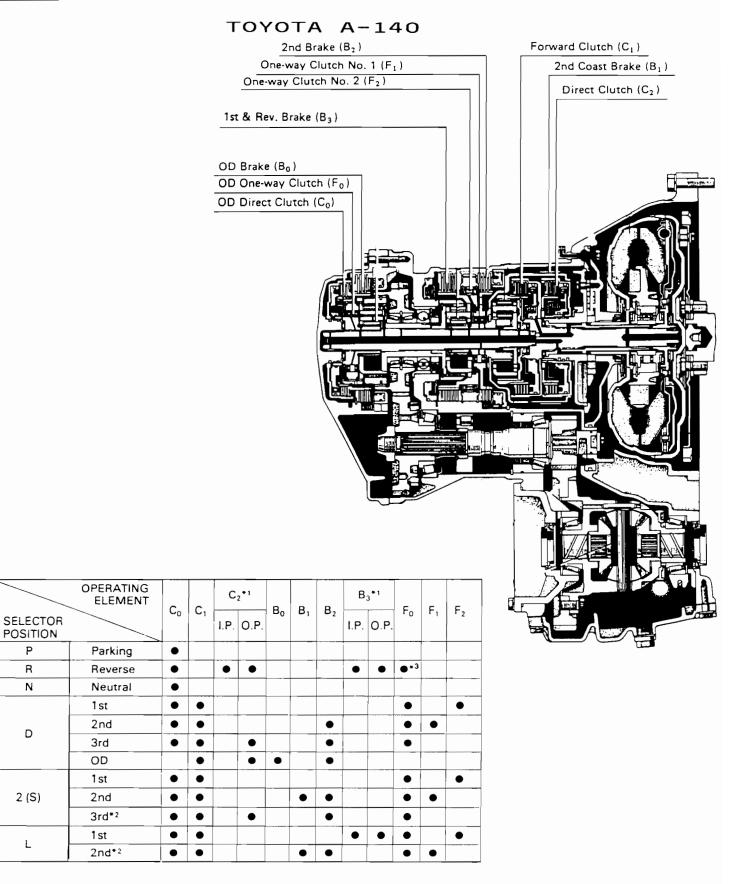
TOYOTA A-130



- 1 B3 1ST & REV BRAKE
- 2 F2 ONE-WAY CLUTCH #2
- 3 B2 2ND BRAKE
- 4 C1 FORWARD CLUTCH
- 5 B1 COAST BRAKE
- 6 C2 DIRECT CLUTCH
- 7 F1 ONW-WAY CLUTCH #1

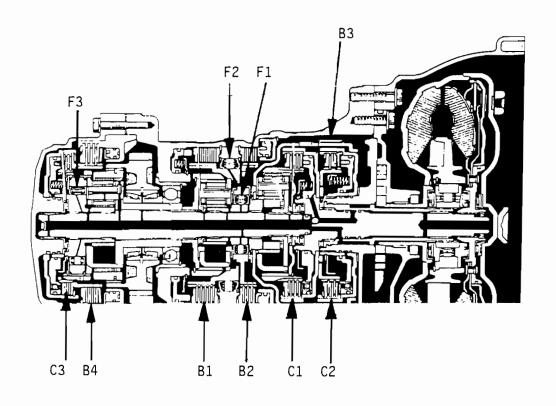
SHIFT POSITION	GEAR	C ₁	C ₂	B ₁	B_2	B_3	F ₁	F ₂
P	PARKING							
R	REVERSE		0			0		
N	NEUTRAL							
D, 2	FIRST	0						0
D	SECOND	0			0		0	
D	THIRD	0	0		0			
2	SECOND	0		0	0		0	
L	FIRST	0				0		0
	FINAL							







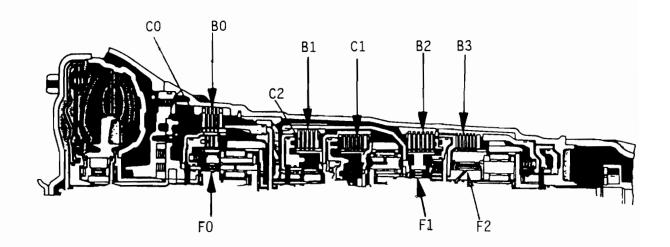
TOYOTA A-240



Shift lever position	Gear position	Cı	C ₂	C ₃	В	В2	В3	В4	F ₁	F ₂	F ₃
Р	Parking							0			
R	Reverse		0				0	0			
N	Neutral							0			
	1st	0						0		0	0
D	2nd	0				0		0	0		0
	3rd	0	0			0		0			0
	OD	0	0	0		0					
2	1st	0						0		0	0
	2nd	0			0	0		0	0		0
L	1st	0					0	0		0	0



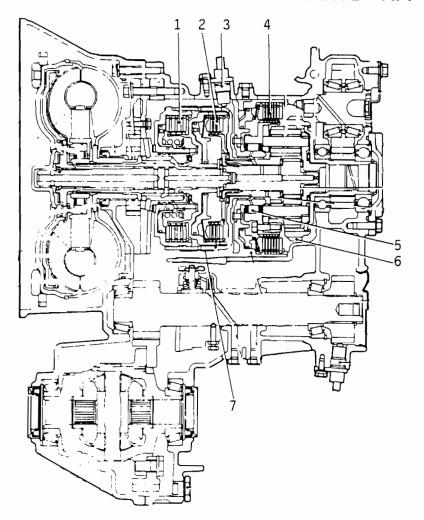
TOYOTA A-340



RANGE	CO	C1	C2	В0	B1	В2	В3	F0	F1	F2
REV D 1 D 2 D 3	ON ON ON ON	ON ON ON	C2	O.U.		ON ON	В3	ON ON ON	ON	ON
D 4 S 1 S 2 S 3	ON ON ON	ON ON ON ON	ON ON	ON	ON	ON ON ON		ON ON ON ON		ON
ĽOŴ	ÖÑ	ÖÑ					ON	ŎŇ		ON



MITSUBISHI KM-171

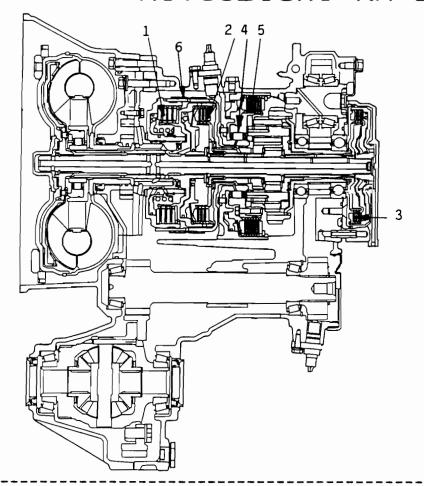


- 1 FORWARD CLUTCH
- 2 REAR CLUTCH
- 3 PULSE GENERATOR
- 4 LOW/REVERSE BRAKE
- 5 ONE-WAY CLUTCH
- 6 PARKIBG SPRAG
- 7 KICKDOWN BAND

Lever position	Gear ' ratio	Engine starting	Parking sprag	Front clutch	Rear clutch	One-way clutch	Kickdown band	Low- reverse brake
P - PARKING		0	0				_	
R - REVERSE	2.502			0				0
N - NEUTRAL		0						
D - DRIVE								
First	3.273				0	0		
Second	1.818				0		0	
Third	1.150			0	0			
2 – SECOND								
First	3.273				0	0		
Second	1.818				0		0	
L – LOCK-UP (First)	3.273				0			0



MITSUBISHI KM-175



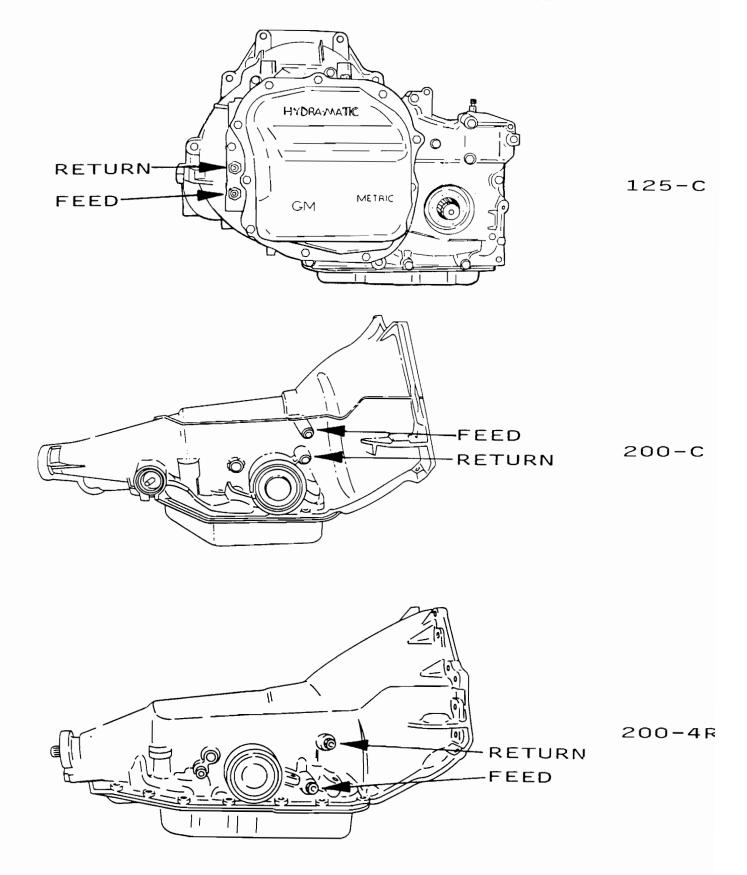
- 1. FRONT CLUTCH
- 2. REAR CLUTCH
- 3. END CLUTCH
- 4. ONE-WAY CLUTCH
- 5. LOW-REV CLUTCH
- 6. KICK DOWN BAND

RANGE	FRONT CLUTCH	REAR CLUTCH	END CLUTCH	ONE-WAY CLUTCH	LOW/REV CLUTCH	KICKDOWN BAND
D 1		ON		ON		
D 2		ON				ON
D 3	ON	ON	ON			
D 4			ON			ON
S 1		ON		ON		
S 2		ON				ON
LOW		ON				ON
REV	ON				ON	

NOTE: CONVERTER CLUTCH CAN APPLY IN SECOND, THIRD AND FORTH.

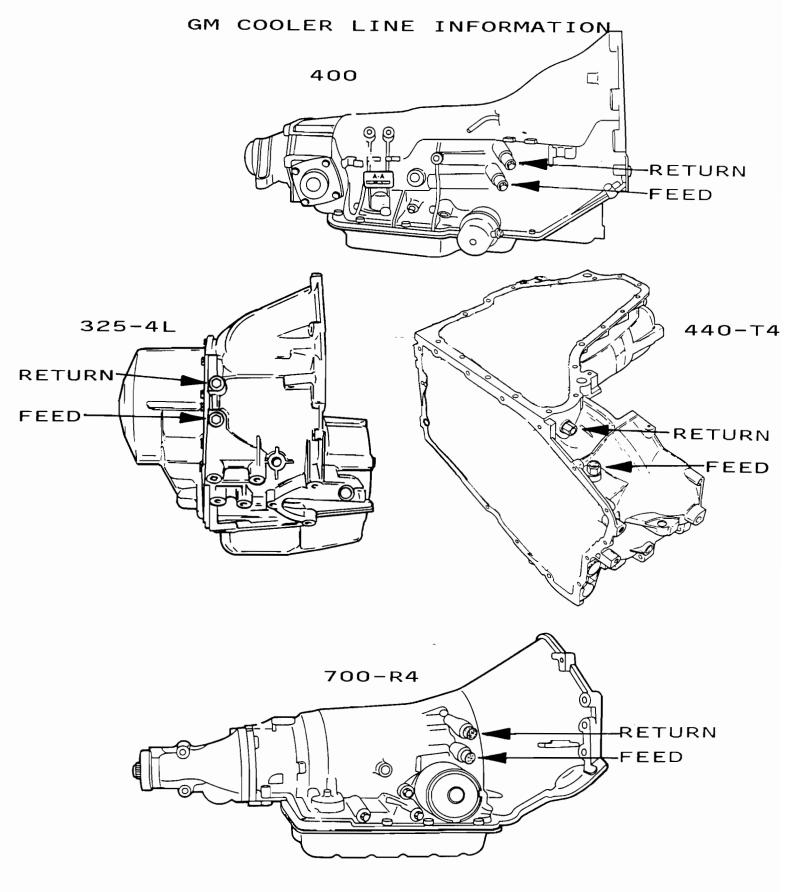


GM COOLER LINE INFORMATION



AUTOMATIC TRANSMISSION SERVICE GROUP

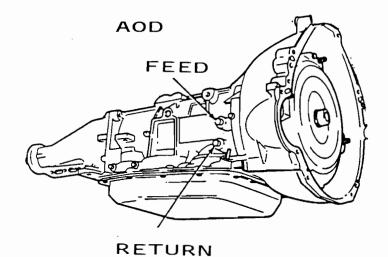


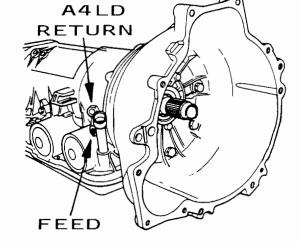


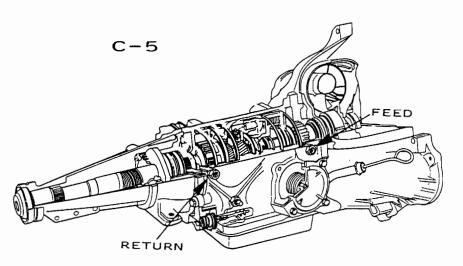
AUTOMATIC TRANSMISSION SERVICE GROUP

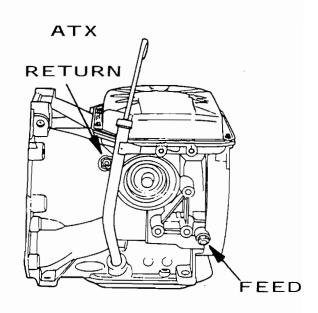


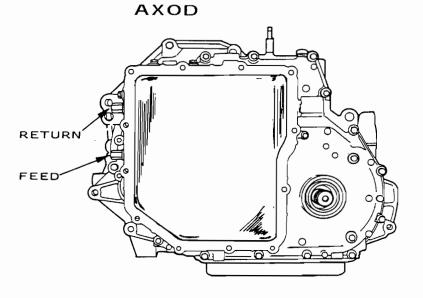
FORD COOLER LINE INFORMATION





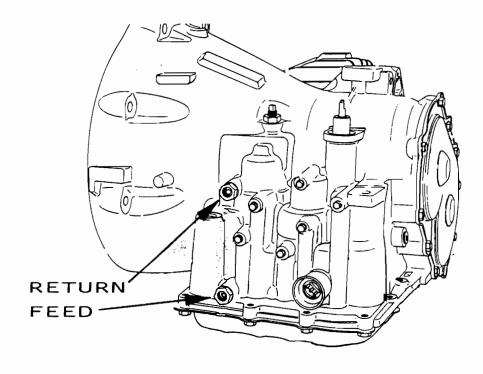




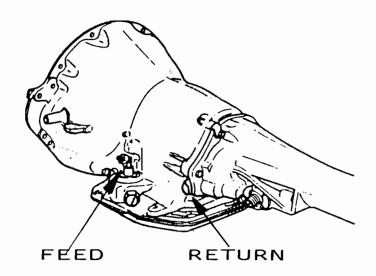




CHRYSLER COOLER LINE INFORMATION



A404/413/415/470 A-670



A-727/A-904/A-999



L4N71B

COMPLAINT: Transmission shifts into O.D. regardless of O.D. control switch position.

CAUSE: The cause may be the small "0" ring missing, or mis-installed in the

groove of the O.D. cancel solenoid. The small "O" ring goes <u>BEHIND</u> the O.D. cancel solenoid, as shown in Figure 1, and seals against the bottom

of the case bore (See Figure 1).

CORRECTION: Install "O" ring in proper location as shown in Figure 1.

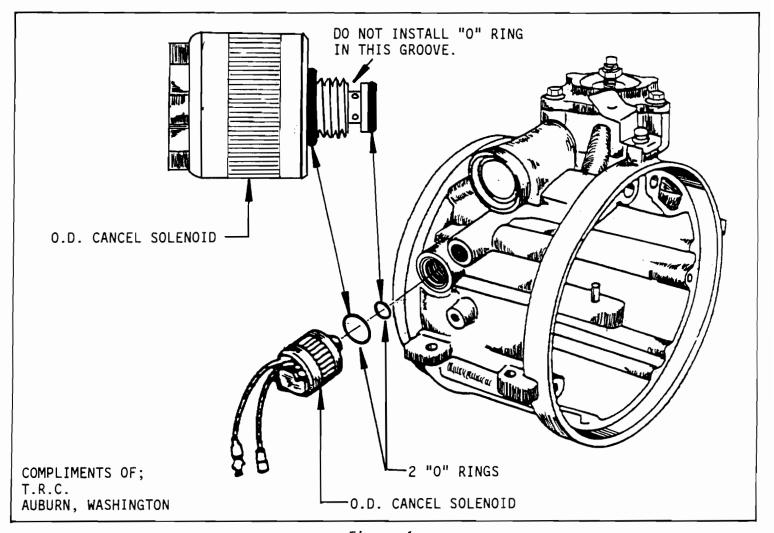


Figure 1



A43DL AND A44DL HARSH 1-2 SHIFT

COMPLAINT: Harsh 1-2 shift at all throttle positions.

CAUSE: Viton check ball missing, or off location.

CORRECTION: Install check ball in proper location (See Figure 1).

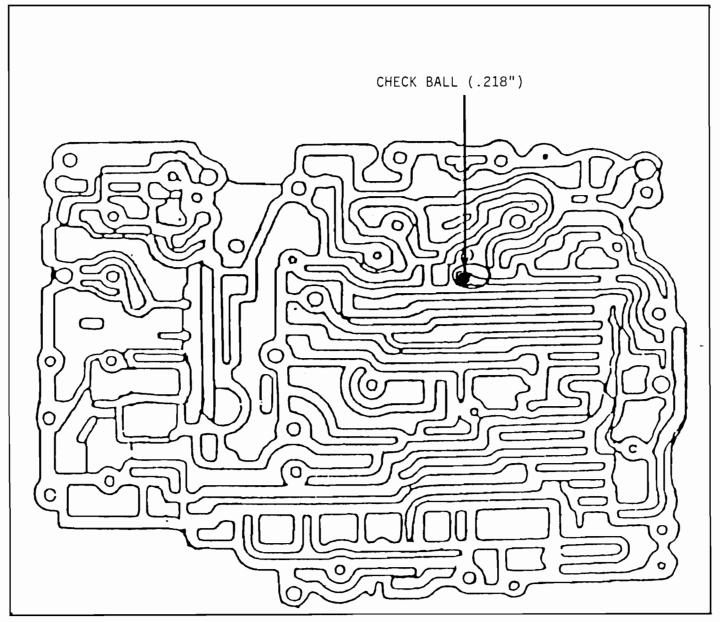


Figure 1