

第8章

弯曲内力

工程力学





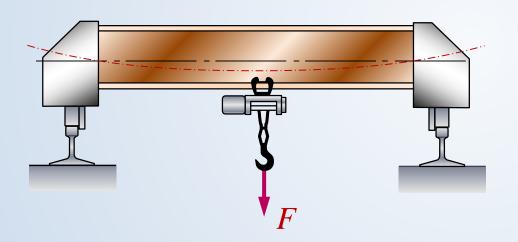
第8章 弯曲内力

- § 8.1 平面弯曲的概念和实例
- § 8.2 梁的支座及载荷的简化
- § 8.3 平面弯曲时梁横截面上的内力 —剪力和弯矩
- § 8.4 剪力方程和弯矩方程 剪力图和弯矩图
- § 8.5 分布载荷集、剪力和弯矩间的关系 及其应用
- § 8.6 用叠加法作弯矩图
- § 8.7 平面刚架和曲杆的弯内力



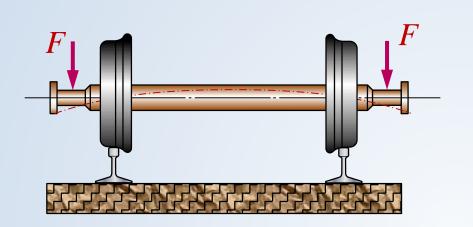
一. 平面弯曲的概念和实例

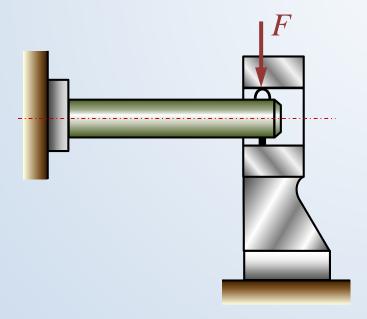
1.平面弯曲实例

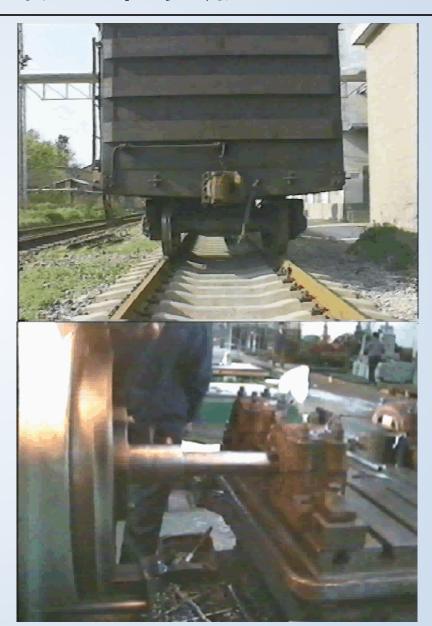










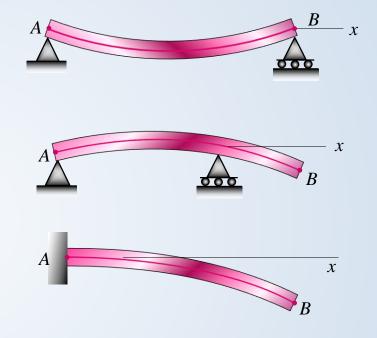




2. 弯曲受力特点:

外力垂直于杆件的轴 线,外力偶矩通过或平 行于轴线

3. 弯曲变形特点:



杆的轴线的曲率发生变化,相邻两横截面之间产生垂直轴线的相对转动

梁: 以弯曲变形为主的杆件



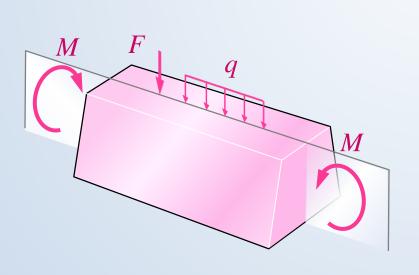
4. 产生平面弯曲的充分条件

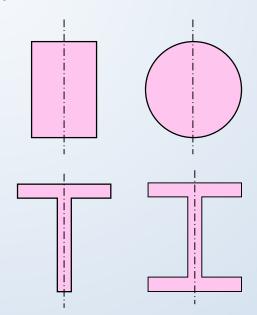
梁有纵向对称面{

截面纵向对称轴

梁轴线

全部外力作用于对称面

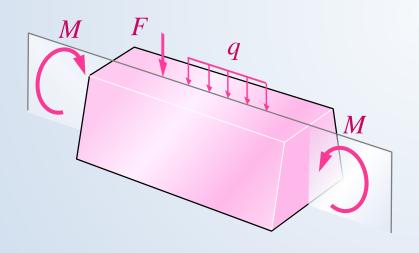


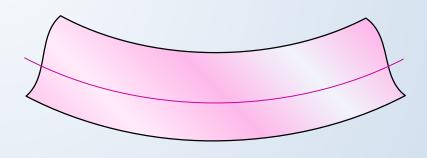




5. 平面弯曲的变形特点

变形前后梁的轴线位于同一平面内







1支座的基本形式

固定铰支座



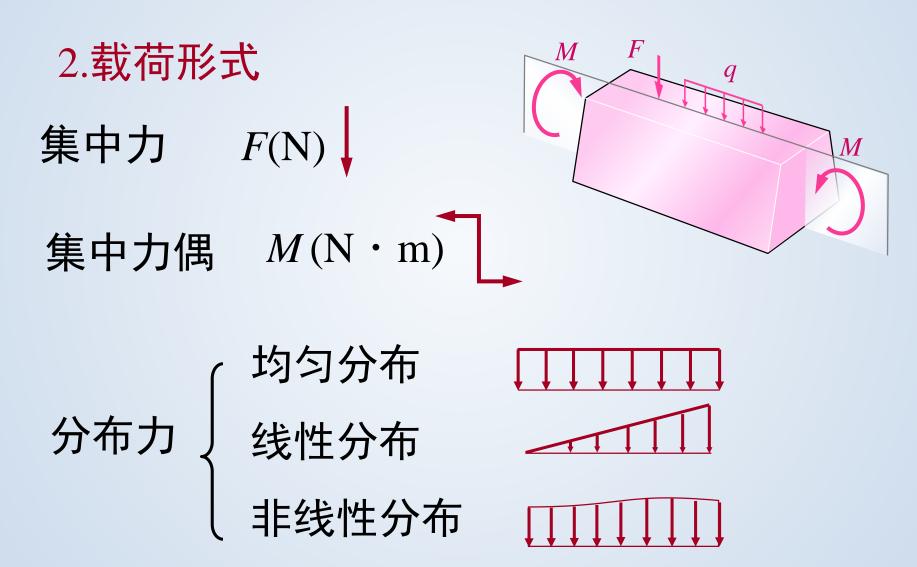
可动铰支座



固定端支座

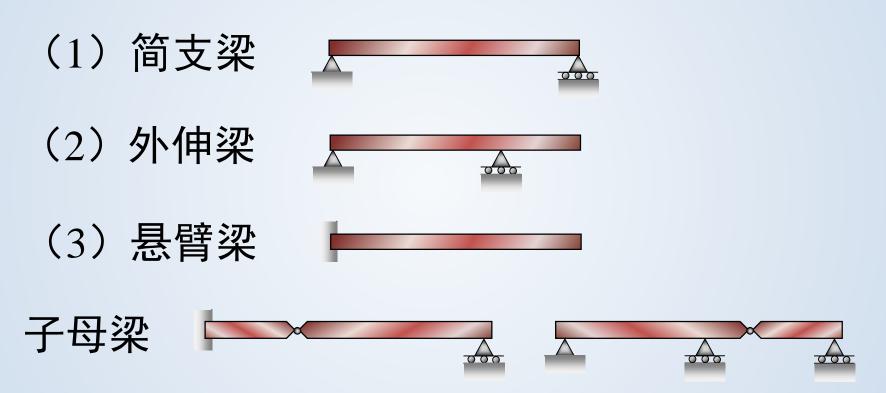








3. 静定梁的基本形式

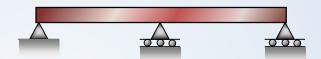


弯曲变形的主要研究对象为直梁。



4. 超静定梁的基本形式

(1) 连续梁



(2) 一端铰支梁

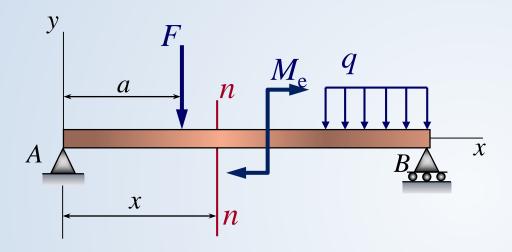


(3) 固定梁



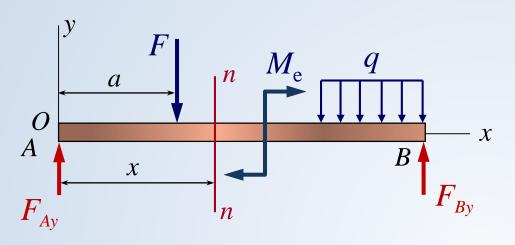


一.用截面法求内力



- 1) 首先正确求支反力
- 得到 F_{Ay} 和 F_{By}
- 2) 截面法求n-n内力





截、取、代、平

$$\sum F_{y} = 0 \quad F_{Ay} - F - F_{Q} = 0$$

解得 $F_{\mathbf{Q}} = F_{Ay} - F$

$$\sum M_c = 0 \quad M + F \cdot (x - a) - F_{Ay} \cdot x = 0$$

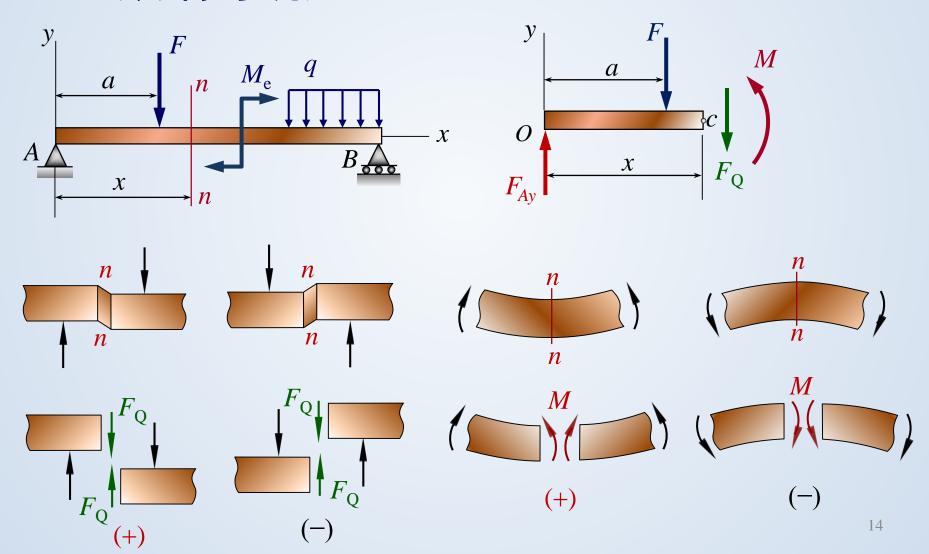
解得 $M = F_{ay}x - Fx + Fa$

F_Q— 剪力 M — 变矩

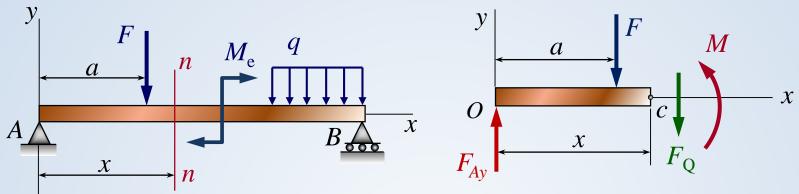
弯曲变形的内力



二.内力符号规定





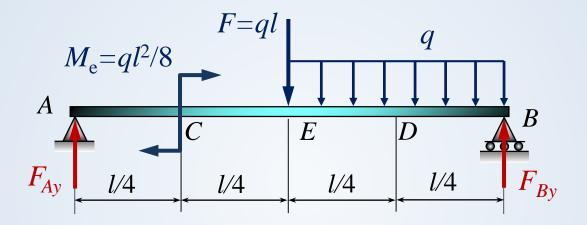


根据符号规定,可以得到下述两个规律:

- 1.任意横截面上的剪力,等于作用在此截面左侧(或右侧)梁上所有外力在y轴上投影的代数和;向上(下)的力产生正剪力,向下(上)的力产生负剪力。
- 2.任意横截面上的弯矩,等于作用在此横截面任一侧所有外力对该截面形心力矩的代数和;向上的力产生正弯矩,向下的力产生负弯矩。



例1 已知: m, F, q, l 求: D截面内力

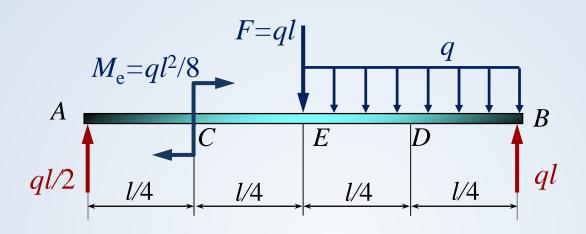


解:1.求支反力

$$\sum M_B = 0 - F_{Ay}l - M_e + \frac{Fl}{2} + \frac{ql}{2} \cdot \frac{l}{4} = 0 \qquad \text{##} F_{Ay} = \frac{ql}{2}$$

$$\sum M_A = 0$$
 $-F_{By}l - \frac{ql}{2} \cdot \frac{3l}{4} - \frac{Fl}{2} - M_e = 0$ 解得 $F_{By} = ql$





2.求D截面的内力

$$F_{\rm Q} = \frac{ql}{2} - ql - \frac{ql}{4} = -\frac{3}{4}ql$$

$$M = ql \times \frac{l}{4} - q \cdot \frac{l}{4} \times \frac{l}{8} = \frac{7ql^2}{32}$$

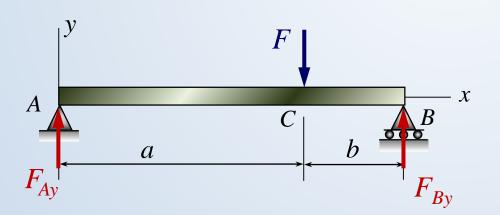


$$F_{Q} = F_{Q}(x)$$

 $M = M(x)$ 为剪力方程和弯矩方程

利用剪力方程弯矩方程画剪力图和弯矩图

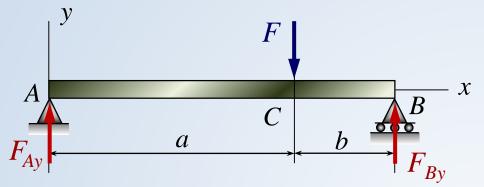
例2 列出下列梁的剪力方程和弯矩方程,并作剪力 图和弯矩图



解: 1.求支反力

$$F_{Ay} = \frac{Fb}{a+b}, \quad F_{By} = \frac{Fa}{a+b}$$





2. 写剪力方程和弯矩方程

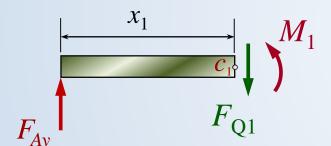
AC段

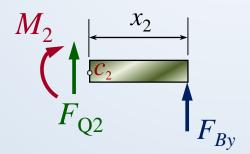
$$F_{Q1} = F_{Ay} \qquad M_1 = F_{Ay} x_1$$

CB段

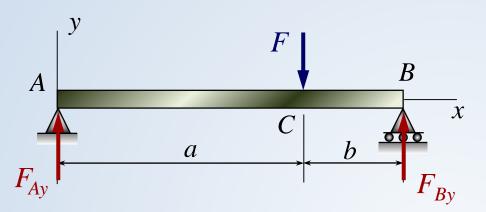
$$F_{\mathbf{Q}_2} = F_{A\mathbf{y}} - F = -F_{B\mathbf{y}}$$

$$M_2 = F_{Bv} x_2$$

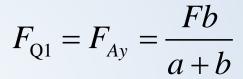








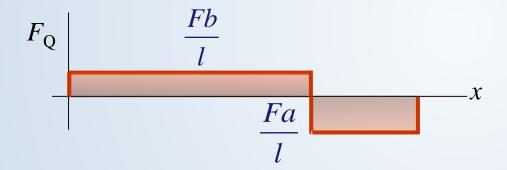


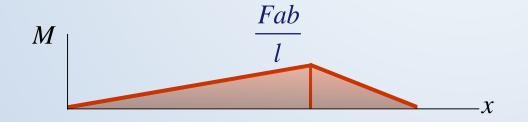


$$F_{Q_2} = -F_{By} = -\frac{Fa}{a+b}$$

$$M_1 = F_{Ay} x_1 = \frac{Fb}{a+b} x_1$$

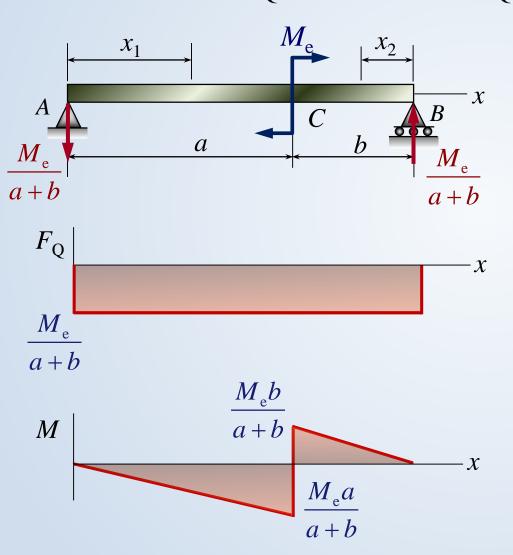
$$M_2 = F_{By} x_2 = \frac{Fa}{a+b} x_2$$







例3 列内力 F_0 ,M方程,作 F_0 M 图



解: 1.求支反力

校核支反力!

 $2.列F_{O}$ M 方程

$$F_{\rm O1} = -M_{\rm e}/(a+b)$$

$$M_1 = -M_e \cdot x_1/(a+b)$$

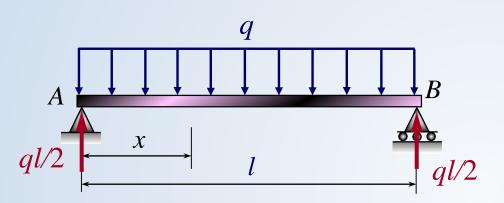
$$F_{\mathrm{Q2}} = -M_{\mathrm{e}}/(a+b)$$

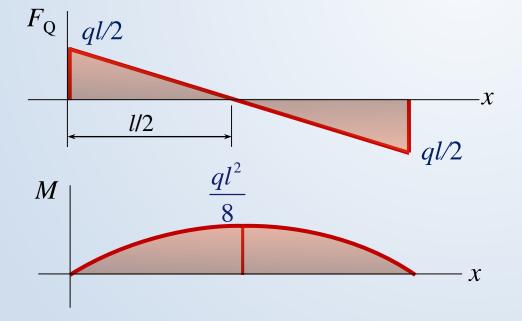
$$M_2 = M_e \cdot x_2 / (a+b)$$

 $3.作F_{O_{\bullet}}$ M图



例4、列图示内力 (F_0, M) 方程,作 F_0, M 图





解: 1.求支反力

 $2.列F_{O_{\cdot}}M$ 方程

$$F_{\mathcal{O}}(x) = ql/2 - qx$$

$$M(x) = qlx/2 - qx^2/2$$

3.作F_O, M 图

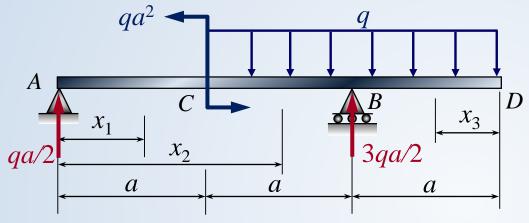
$$M'(x) = \frac{ql}{2} - qx$$

令
$$M'(x) = 0$$
, 得 $x_0 = \frac{l}{2}$

$$M(x_0) = \frac{ql^2}{2}$$



例5 列图示内力方程,作 F_{Q} M图



F_{Q} qa/2 qa/2 qa/2 qa/2 qa/2 $qa^2/2$ $qa^2/2$ $qa^2/2$ $qa^2/2$ $qa^2/2$

解:1.求支反力

 $2.列F_{O.}$ M 方程

$$AC$$
段 $F_{Q1} = qa/2$ $M_1 = qax_1/2$

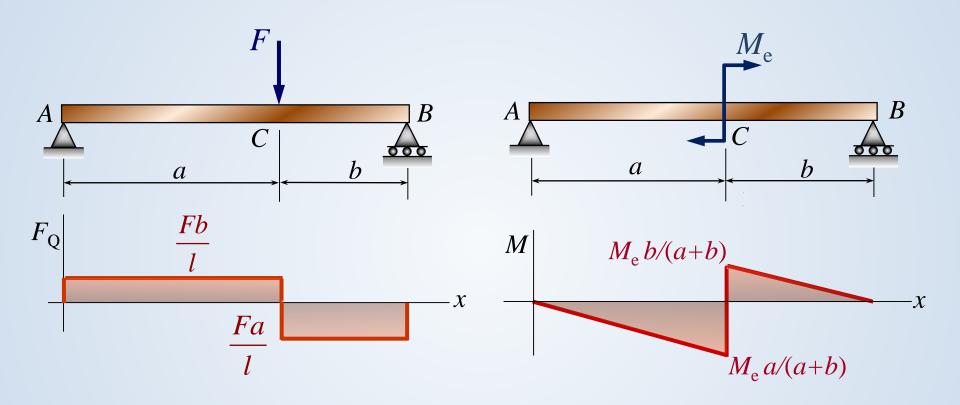
$$CB$$
 $F_{Q_2} = \frac{qa}{2} - q(x_2 - a)$

$$M_2 = \frac{qa}{2}x_2 - qa^2 - \frac{q(x_2 - a)^2}{2}$$

$$BD$$
段
$$F_{Q_3} = qx_3$$

$$M_3 = -\frac{qx_3^2}{2}$$



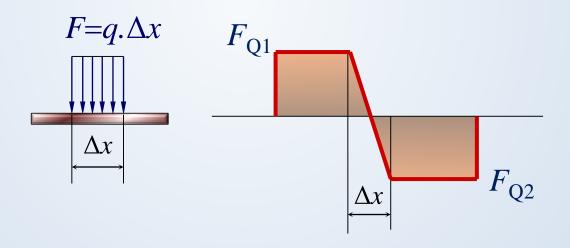


F 作用面两侧FQ图有突变,其突变值=F,M 作用面两侧M图有突变,其突变值=M。

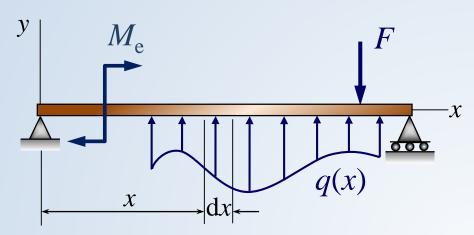


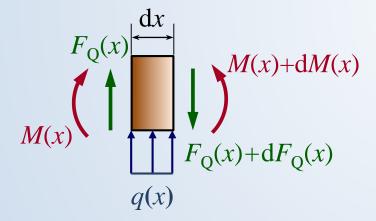
这使得F和M作用面没有确切的F。和M?

事实并非如此,因为F和M事实上不可能集中作用于一点,它实际分布于一个微段 Δx ,









q(x)向上为正

取dx, 由微段的平衡, 略去高阶微量,得:

$$\Sigma F_y = 0$$
 $dF_Q(x) / dx = q(x)$

$$\Sigma M = 0$$
 $dM(x) / dx = F_Q(x)$

推得 $d^2M(x)/dx^2 = q(x)$

经积分得:

$$F_{Q}(x_{2}) - F_{Q}(x_{1}) = \int_{x_{1}}^{x_{2}} q(x) dx$$

$$M(x_2) - M(x_1) = \int_{x_1}^{x_2} F_Q(x) dx$$



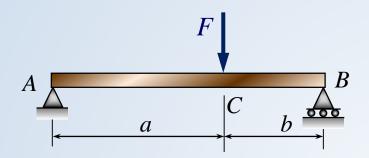
利用 $q(x)_{,}$ $F_{Q}(x)_{,}$ M(x)之间的微、积分关系可以帮助绘制校核 F_{Q} M 图。利用归纳的 $q_{,}$ $F_{,}$ M 作用下 F_{Q} M 图的特征,可以找出绘制 F_{Q} M 图的简便方法。

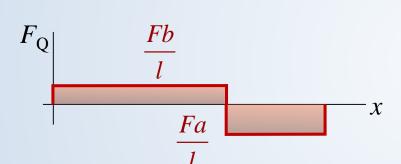


绘制 F_{O} 、M 图的简便方法

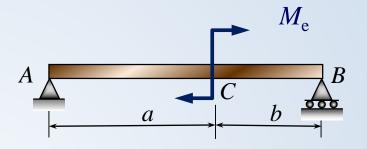
- 一正确求出支反力。
- 二 有集中力F 作用处, F_0 图有突变,
- 方向与F一致(左),突变值 =F,M 图有折线
- 三.有集中力偶M作用处,M图有突变,
- 方向与M一致(左), 突变值 =M, F_Q 图不变。



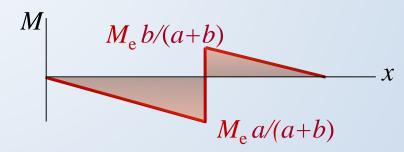






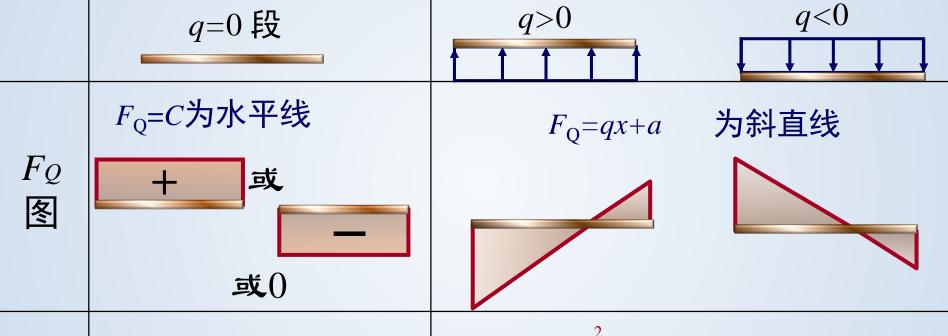








四. $q \supset F_0$, M 图的关系

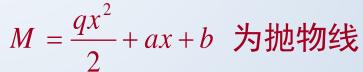


M

冬

 $F_0>0$ $F_0<0$

M=Cx+a 为斜直线





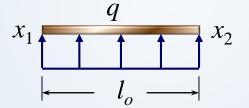


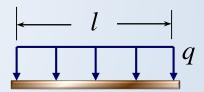
五. $F_0=0$ 处,M取得极值。

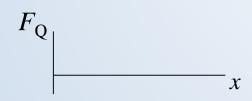
六.在 F_Q , M 图中标出F, M, 均布q起止点的 F_Q 和 M 的值,及M 取得极值处的值。

 F_0 , M 值的计算方法: 方法一; 方法二:

面积增量法



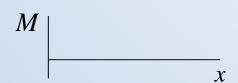






$$\left|\Delta F_{\rm Q}\right| = q l_o$$





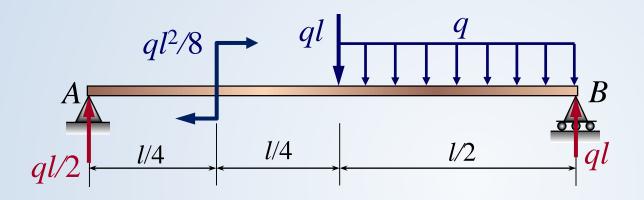


$$\left|\Delta M\right| = S_{F_{Q}}$$

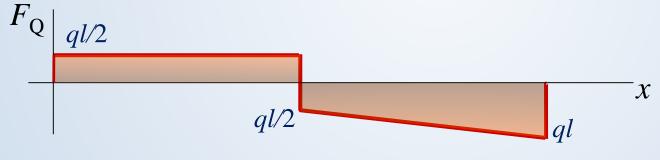




例 6 试画出图示梁的剪力图和弯矩图(作内力图)

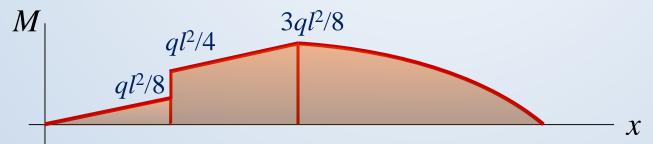


解:1.求支反力



$$F_{Ay} = ql/2$$

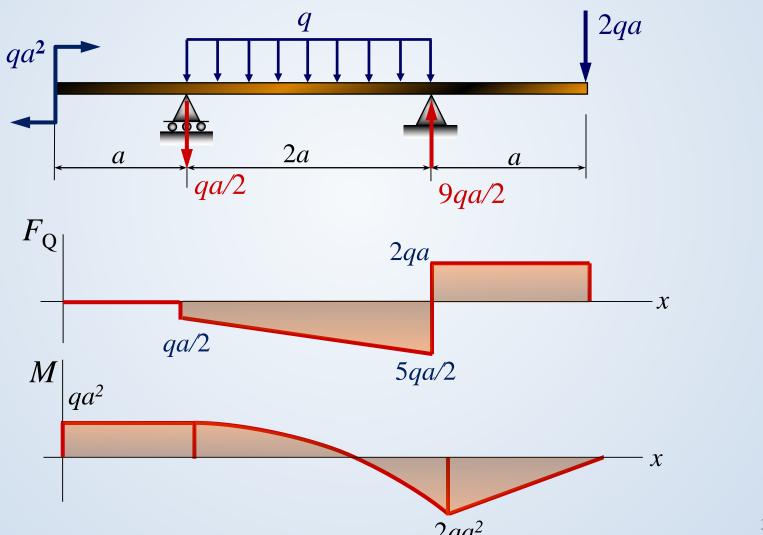
$$F_{By} = ql$$



2. 画 F_{Q} , M 图

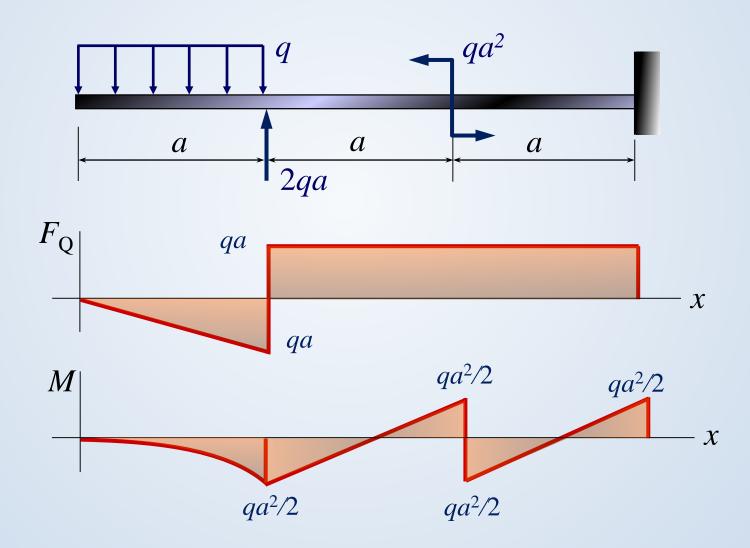


例7试画出图示梁的剪力图和弯矩图(作内力图)



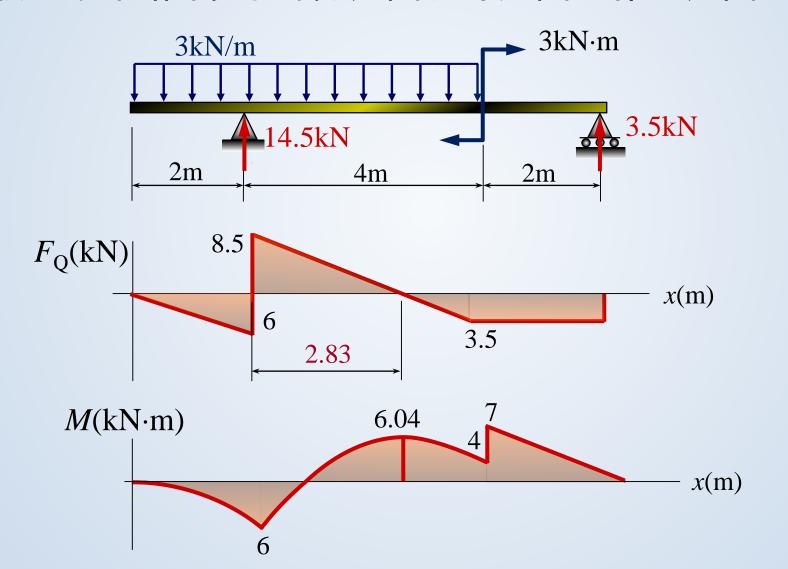


例8试画出图示梁的剪力图和弯矩图(作内力图)



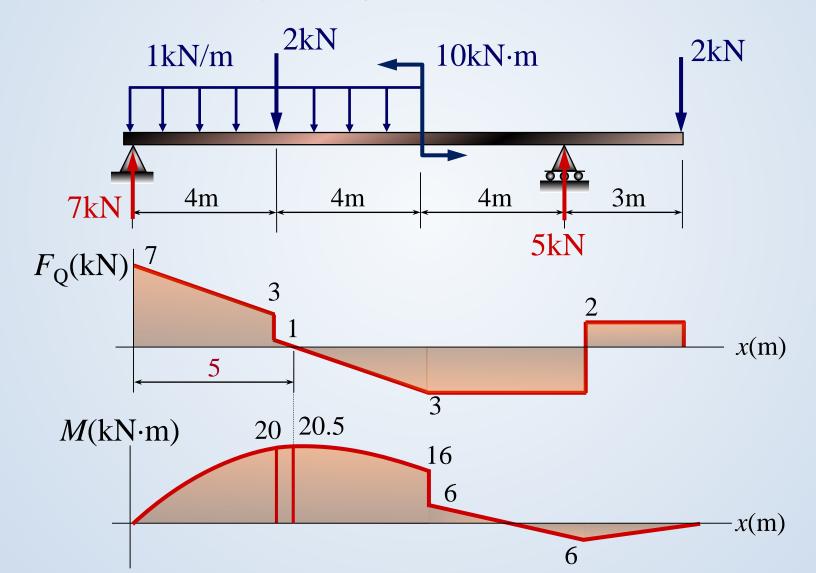


例9试画出图示梁的剪力图和弯矩图(作内力图)





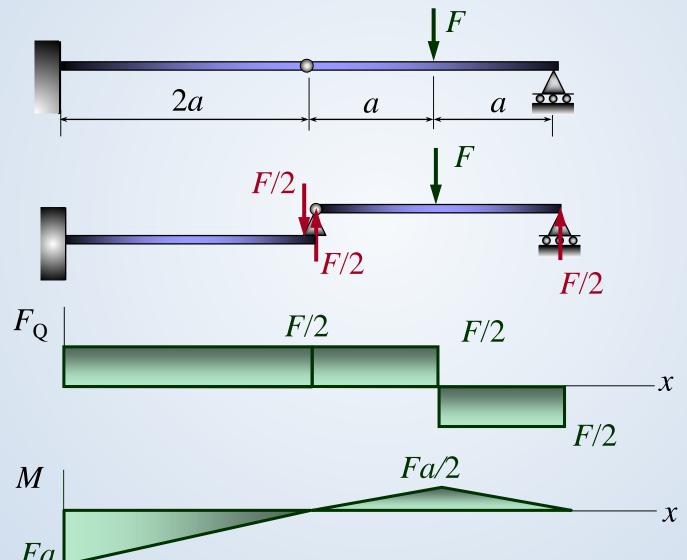
例10 试画出图示梁的剪力图和弯矩图





8.5 分布载荷集、剪力和弯矩间的关系及其应用

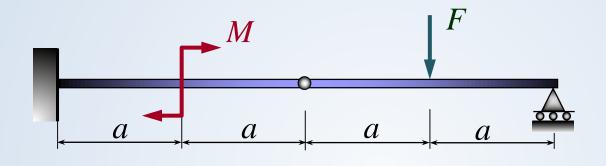
例12 试画出图示梁的剪力图和弯矩图(作内力图)

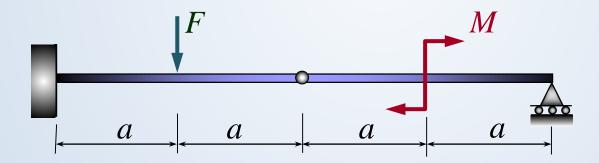




8.5 分布载荷集、剪力和弯矩间的关系及其应用

例 13 试画出图示梁的剪力图和弯矩图







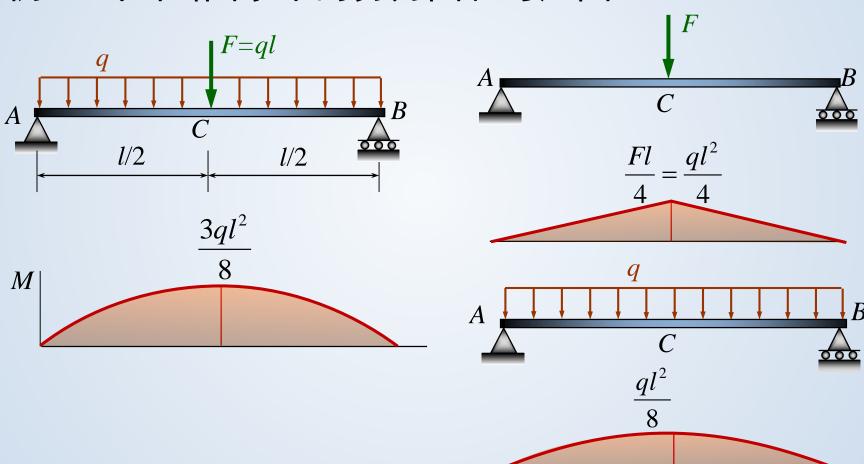
叠加原理:由几个载荷共同作用下所引起的某一物理量(内力,应力,应变或变形等),等于每一个载荷(主动)单独作用下所引起的该物理量的叠加(代数和)。

应用条件: 所求物理量(内力,应力,应变或形等)必须与载荷为线性关系(线弹性结构)

方法: 先分别画出每一载荷单独作用时梁的 弯矩图, 然后将同一截面相应的各纵坐标代 数叠加。

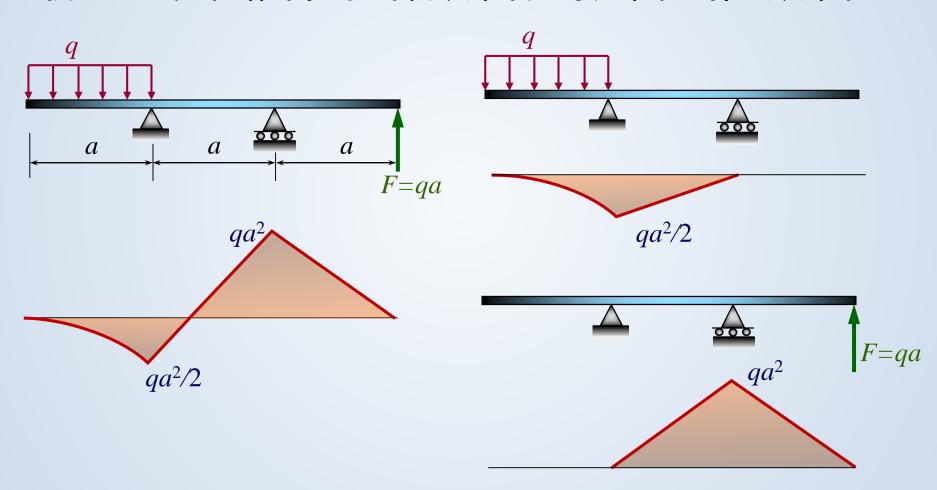


例14 试画出图示梁的剪力图和弯矩图



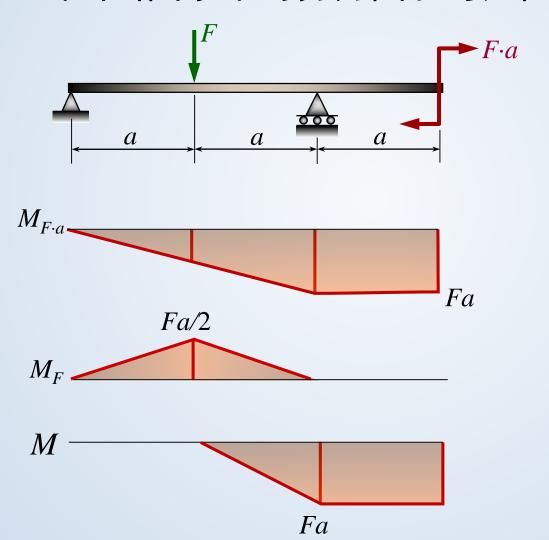


例 15 试画出图示梁的剪力图和弯矩图(作内力图)

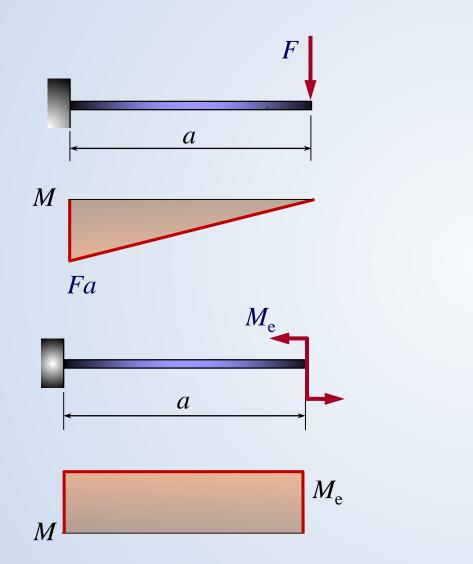


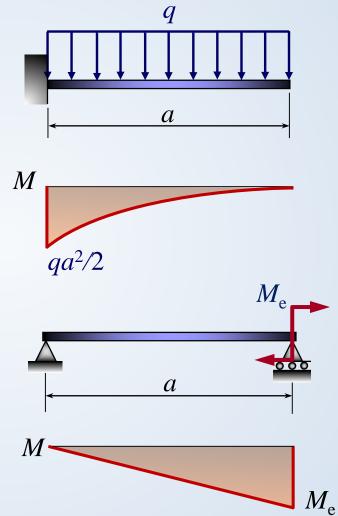


例 16 试画出图示梁的剪力图和弯矩图

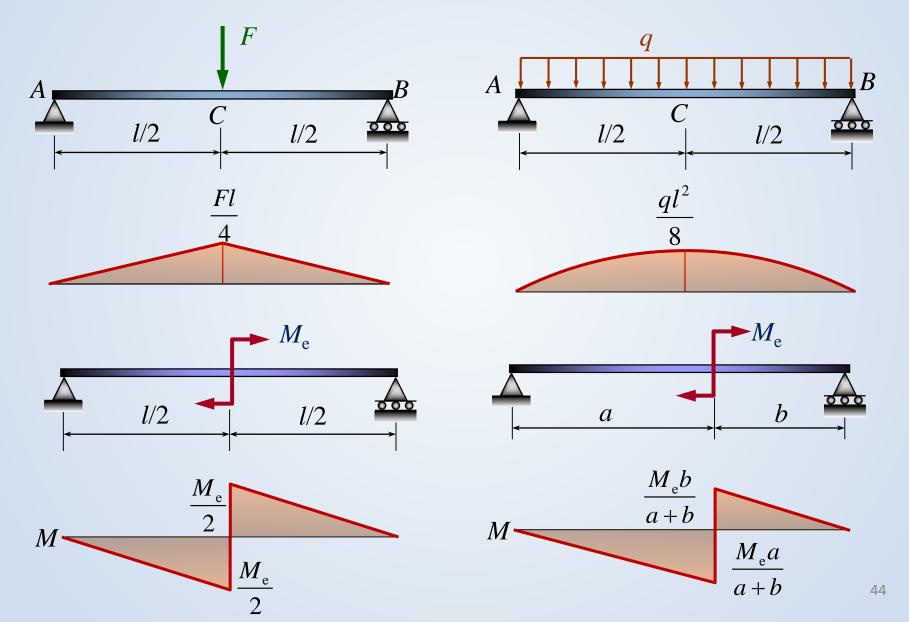




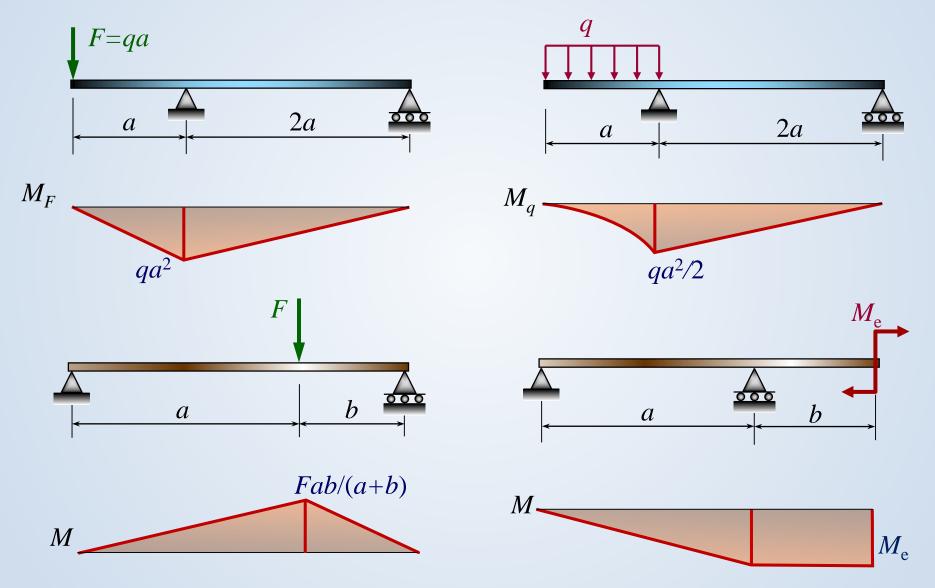














一. 平面刚架的内力

工程中某些结构的轴线是由几段直线组成的折线,这种结构的每两组成部分用刚节点联接。

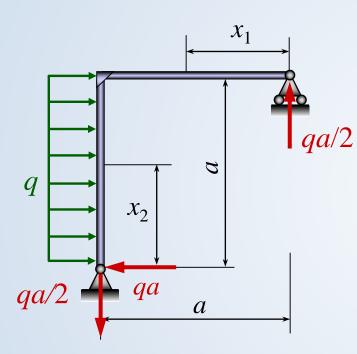
刚节点---刚性接头处,相连杆件间的夹角在 受力时不变化,刚节点不仅能传力,而且还 能传递力矩。

刚架---杆系在联接处用刚节点联接起来的结构。

平面刚架---刚架的各杆系位于同一平面内



例17 列出平面刚架的内力方程,并作内力图



解: 1. 求支反力

ga/2 2. 分段建立内力方程

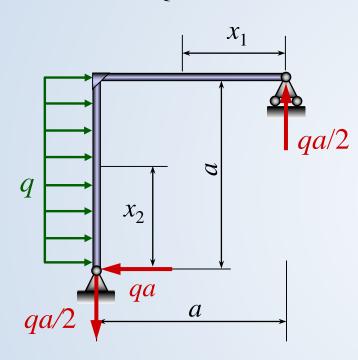
$$F_{N1} = 0$$
 $F_{O1} = -qa/2$ $M_1 = qax_1/2$

$$F_{N2} = qa/2$$
 $F_{O2} = qa - qx_2$

$$M_2 = qax_2 - qx_2^2/2$$



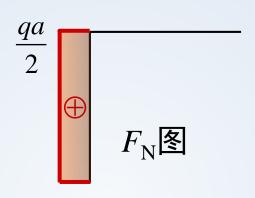
3、作 F_{Q} 、M、 F_{N} 图

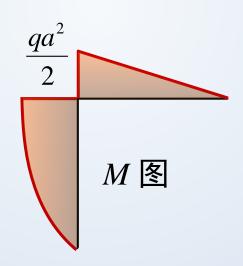


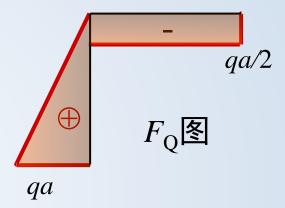
$$F_{\text{N1}} = 0$$
 $F_{\text{O1}} = -qa/2$ $M_1 = qax_1/2$

$$F_{\text{N2}} = qa/2 \quad F_{\text{O2}} = qa - qx_2$$

$$M_2 = qax_2 - qx_2^2/2$$



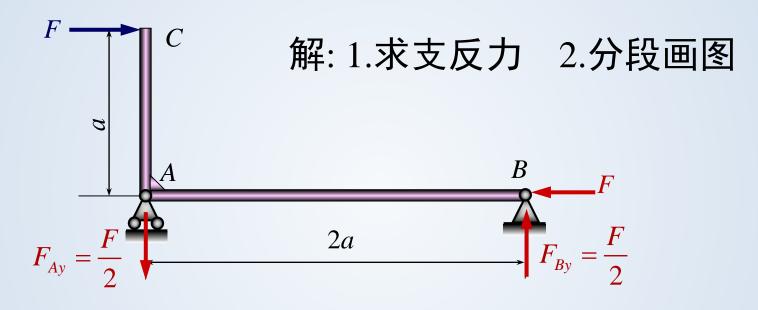


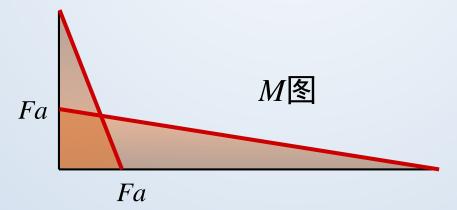


M图画在杆件受 压侧即和直梁的 规定一样



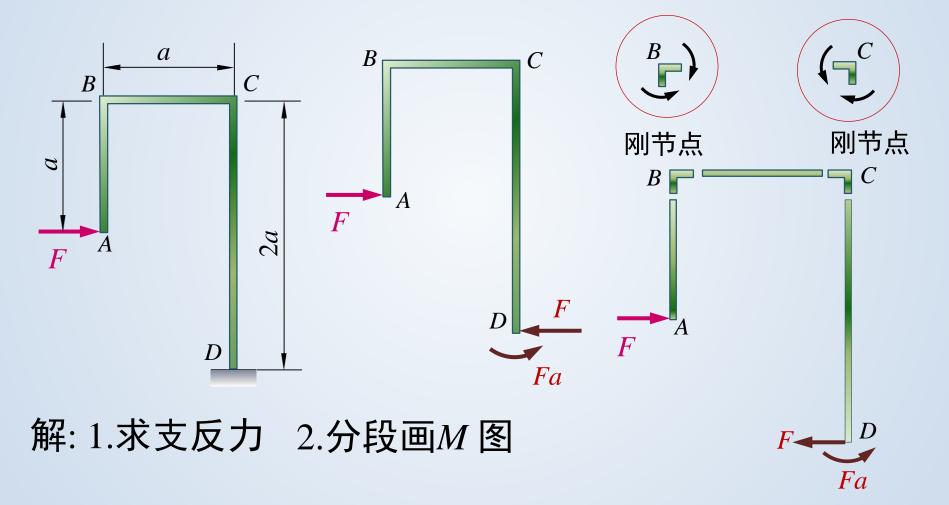
例 18 试画出图示梁的弯矩图。



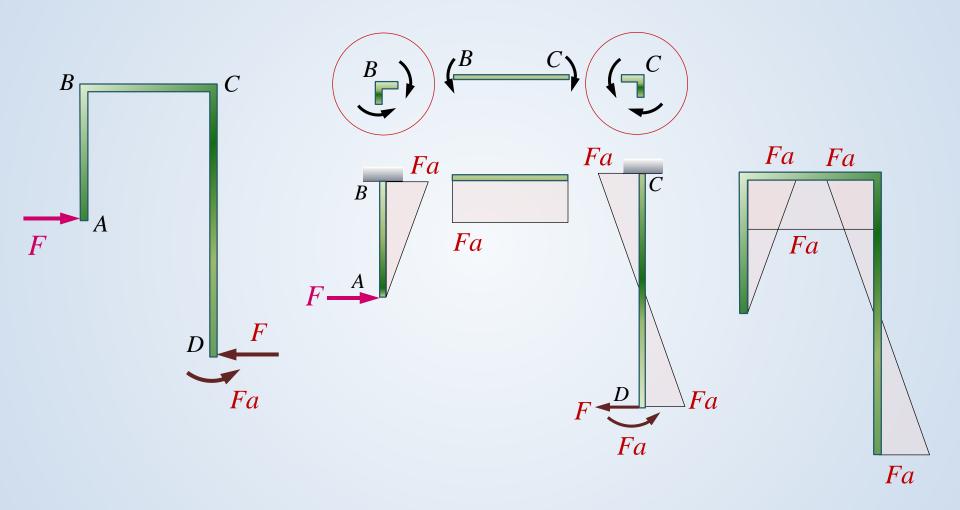




例 19 试画出图示梁的弯矩图。









二. 平面曲杆的弯曲内力

平面曲杆—轴线是一平面曲线

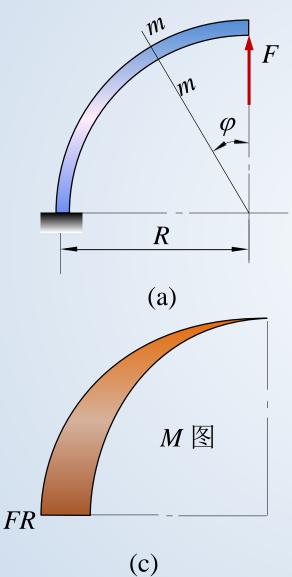
小曲率杆—对小曲率的平面曲杆,其内力的计 算仍采用直梁的方法

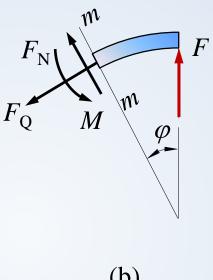
截面法—任意截面切开,设内力 F_N , F_Q , M 根据平衡方程即可列出内力方程

内力符号规定:

 $F_{\rm N}, F_{\rm O}$ 同前; M—使轴线曲率增加为正







(b)

$$F_{\rm N}(\varphi) = F\sin\varphi,$$

$$F_{\mathcal{Q}}(\varphi) = -F\cos\varphi,$$

$$M(\varphi) = -FR\sin\varphi$$



Thank you!