

- Figure 1(a) shows the control system for an excavator, where $G(s) = \frac{1000}{s(s+10)(s^2+1.2s+1.44)}$. The oscillatory roots of $G(s)$ arise from compressibility of the hydraulic fluid and are often referred to as the *oil-mass resonance*. Assume that we use a computer based controller of the form:

$$D(z) = K \left[1 + K_v \left(1 - \frac{1}{z} \right) + K_a \left(1 - \frac{2}{z} + \frac{1}{z^2} \right) \right]$$

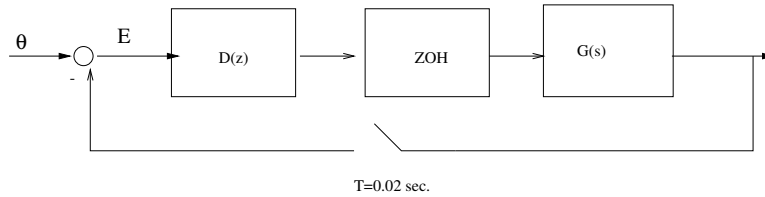


Figure 1: Block Diagram of an Excavator Control System

- Show that the steady state error to a ramp is $e_{ss}^{\text{ramp}} = \frac{1.44}{K}$.
 - Determine the highest K that can be achieved using proportional control (i.e. $K_v = K_a = 0$).
 - Repeat (b) for PD control ($K_a = 0$).
 - Repeat (b) for PD + acceleration control (all $K_i \neq 0$).
- Consider the system shown in Figure 2:

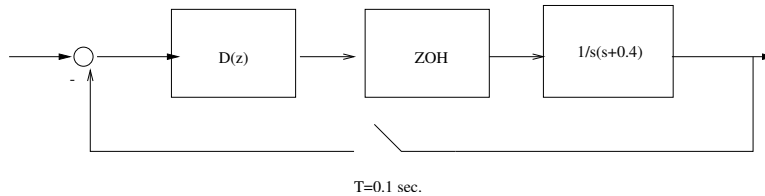


Figure 2: Block Diagram for Problem 2

Design a controller $D(z)$ so that the closed-loop system step response satisfies:

- Rise Time $T_r \approx 0.5 \text{ sec}$.
- Overshoot $\leq 25\%$
- $e_{ss}^{\text{step}} = 0$

Note: all of the specs above should be achieved *simultaneously*.