

1. Figure 1(a) shows the control system for an excavator, where  $G(s) = \frac{1000}{s(s+10)(s^2+1.2s+1.44)}$ . The oscillatory roots of  $G(s)$  arise from compressibility of the hydraulic fluid and are often referred to as the *oil-mass resonance*. Assume that we use a computer based controller of the form:

$$D(z) = K \left[ 1 + K_v \left( 1 - \frac{1}{z} \right) + K_a \left( 1 - \frac{2}{z} + \frac{1}{z^2} \right) \right]$$

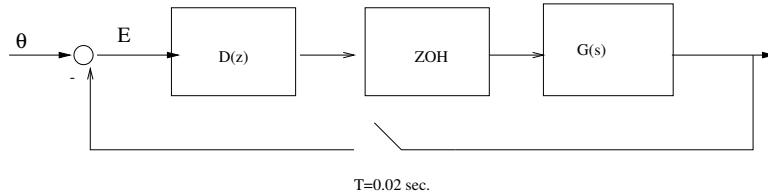


Figure 1: Block Diagram of an Excavator Control System

- (a) Show that the steady state error to a ramp is  $e_{ss}^{\text{ramp}} = \frac{1.44}{K}$ .
  - (b) Determine the highest  $K$  that can be achieved using proportional control (i.e.  $K_v = K_a = 0$ ).
  - (c) Repeat (b) for PD control ( $K_a = 0$ ).
  - (d) Repeat (b) for PD + acceleration control (all  $K_i \neq 0$ ).
2. Consider the system shown in Figure 2:

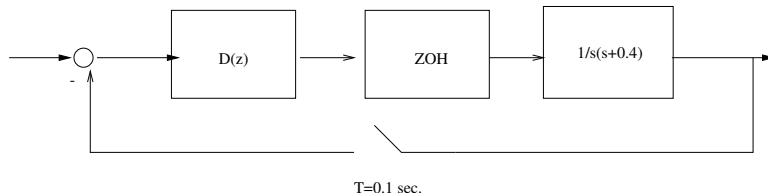


Figure 2: Block Diagram for Problem 2

Design a controller  $D(z)$  so that the closed-loop system step response satisfies:

- (a) Rise Time  $T_r \approx 0.5 \text{ sec.}$
- (b) Overshoot  $\leq 25\%$
- (c)  $e_{ss}^{\text{step}} = 0$

Note: all of the specs above should be achieved *simultaneously*.