

Embragues, frenos, coples y volantes de inercia

Embragues y frenos de tambor de expansión interna

$$M_f = \frac{fpabr}{\sin(\theta_a)} \int_{\theta_1}^{\theta_2} \sin(\theta)(r - a \cos(\theta)) d\theta$$
$$M_n = \frac{pabra}{\sin(\theta_a)} \int_{\theta_1}^{\theta_2} \sin^2(\theta) d\theta$$

$$A = \int_{\theta_1}^{\theta_2} \sin(\theta) \cos(\theta) d\theta = \left(\frac{1}{2} \sin^2(\theta) \right) \Big|_{\theta_1}^{\theta_2}$$
$$B = \int_{\theta_1}^{\theta_2} \sin^2(\theta) d\theta = \left(\frac{\theta}{2} - \frac{1}{4} \sin(2\theta) \right) \Big|_{\theta_1}^{\theta_2}$$

$$T = \frac{fpabr^2(\cos(\theta_1) - \cos(\theta_2))}{\sin(\theta_a)}$$

Caso I (fuerza \rightarrow pasador)

$$F = \frac{M_n - M_f}{c}$$

$$R_x = \frac{pabr}{\sin(\theta_a)}(A - fB) - F_x$$
$$R_y = \frac{pabr}{\sin(\theta_a)}(B + fA) - F_y$$

Caso II (pasador \rightarrow fuerza)

$$F = \frac{M_n + M_f}{c}$$

$$R_x = \frac{pabr}{\sin(\theta_a)}(A + fB) - F_x$$
$$R_y = \frac{pabr}{\sin(\theta_a)}(B - fA) - F_y$$

Embragues y frenos de contracción externa

$$M_f = \frac{fpabr}{\sin(\theta_a)} \int_{\theta_1}^{\theta_2} \sin(\theta)(r - a \cos(\theta)) d\theta$$

$$M_n = \frac{pabra}{\sin(\theta_a)} \int_{\theta_1}^{\theta_2} \sin^2(\theta) d\theta$$

$$A = \int_{\theta_1}^{\theta_2} \sin(\theta) \cos(\theta) d\theta = \left(\frac{1}{2} \sin^2(\theta) \right) \Big|_{\theta_1}^{\theta_2}$$

$$B = \int_{\theta_1}^{\theta_2} \sin^2(\theta) d\theta = \left(\frac{\theta}{2} - \frac{1}{4} \sin(2\theta) \right) \Big|_{\theta_1}^{\theta_2}$$

$$T = \frac{fpabr^2(\cos(\theta_1) - \cos(\theta_2))}{\sin(\theta_a)}$$

Caso I (fuerza \rightarrow pasador)

$$F = \frac{M_n + M_f}{c}$$

$$R_x = \frac{pabr}{\sin(\theta_a)}(A + fB) - F_x$$

$$R_y = \frac{pabr}{\sin(\theta_a)}(fA - B) + F_y$$

Caso II (pasador \rightarrow fuerza)

$$F = \frac{M_n - M_f}{c}$$

$$R_x = \frac{pabr}{\sin(\theta_a)}(A - fB) - F_x$$

$$R_y = \frac{pabr}{\sin(\theta_a)}(-fA - B) + F_y$$

Embragues y frenos de banda

$$\frac{P_1}{P_2} = e^{f\phi}$$

$$T = (P_1 - P_2) \frac{D}{2}$$

$$p = \frac{P}{br} = \frac{2P_1}{bD}$$

Embragues axiales de fricción de contacto

Desgaste uniforme

$$F = \frac{\pi p_a d}{2} (D - d)$$

$$T = \frac{\pi f p_a d}{8} (D^2 - d^2)$$

$$T = \frac{Ff}{4} (D + d)$$

Presión uniforme

$$F = \frac{\pi p_a}{4} (D^2 - d^2)$$

$$T = \frac{\pi f p}{12} (D^3 - d^3)$$

$$T = \frac{Ff}{3} \frac{D^3 - d^3}{D^2 - d^2}$$

Frenos de disco

Desgaste uniforme

$$F = (\theta_2 - \theta_1) p_a r_i (r_o - r_i)$$

$$T = \frac{1}{2} (\theta_2 - \theta_1) f p_a r_i (r_o^2 - r_i^2)$$

$$r_e = \frac{r_o + r_i}{2}$$

$$\bar{r} = \frac{\cos(\theta_1) - \cos(\theta_2)}{\theta_2 - \theta_1} \frac{r_o + r_i}{2}$$

Presión uniforme

$$F = \frac{1}{2} (\theta_2 - \theta_1) p_a (r_o^2 - r_i^2)$$

$$T = \frac{1}{3} (\theta_2 - \theta_1) f p_a (r_o^3 - r_i^3)$$

$$r_e = \frac{2 r_o^3 - r_i^3}{3 r_o^2 - r_i^2}$$

$$\bar{r} = \frac{2 r_o^3 - r_i^3}{3 r_o^2 - r_i^2} \frac{\cos(\theta_1) - \cos(\theta_2)}{\theta_2 - \theta_1}$$

Freno de yugo de zapata circular (de botón o de disco)

$$r_e = \delta e$$

$$F = \pi R^2 p_{prom}$$

$$T = f F r_e$$

Embragues y frenos cónicos

Desgaste uniforme

$$F = \frac{\pi p_a d}{2} (D - d)$$

$$T = \frac{\pi f p_a d}{8 \sin(\alpha)} (D^2 - d^2)$$

$$T = \frac{F f}{4 \sin(\alpha)} (D + d)$$

Presión uniforme

$$F = \frac{\pi p_a}{4} (D^2 - d^2)$$

$$T = \frac{\pi f p_a}{12 \sin(\alpha)} (D^3 - d^3)$$

$$T = \frac{F f}{3 \sin(\alpha)} \frac{D^3 - d^3}{D^2 - d^2}$$